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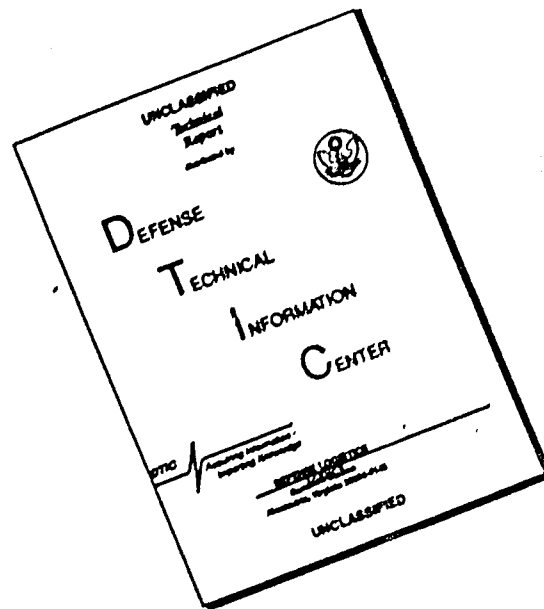
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HERBERT ST. LOUIS MUNICIPAL AIRPORT

DETAILED FINAL REPORT OF RESEARCH ON
HIGH-SPEED ROTARY-SEALED WING AIRCRAFT

VOLUME VII

SAMPLE AIRCRAFT POWER PLANT AND TEST ANALYSIS

OFFICE OF NAVAL RESEARCH, MEMPHIS BRANCH
PROJECT NR 250-001 CONTRACT NR 250-001

Report 1905-1

Serial 15

29 December 1950

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Enclosure (5) to
MAC Letter 2136-70P-1756

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DETAILED FINAL REPORT OF RESEARCH ON
HIGH SPEED ROTARY-FINNED WING AIRCRAFT

VOLUME VII

SAMPLE AIRCRAFT POWER PLANT AND DUCT ANALYSIS

SUBMITTED UNDER Contract Ndonr-64901 to the Office of Naval Research,
Amphibious Branch, Project NR 257-001

PREPARED BY *K. L. Brown*
K. L. Brown

APPROVED BY *J. S. Winter*
J. S. Winter

APPROVED BY *H. H. Ostroff*
H. H. Ostroff

APPROVED BY *C. M. Hurkamp*
C. M. Hurkamp

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MODEL 78

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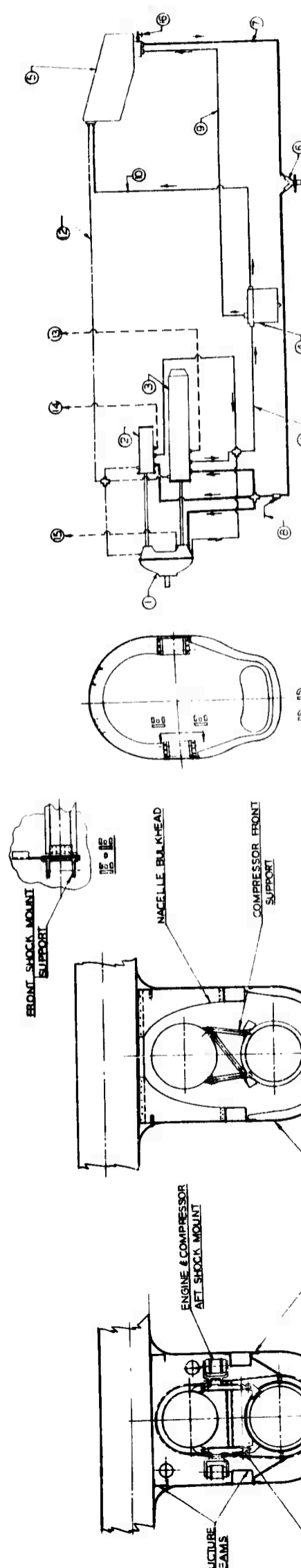
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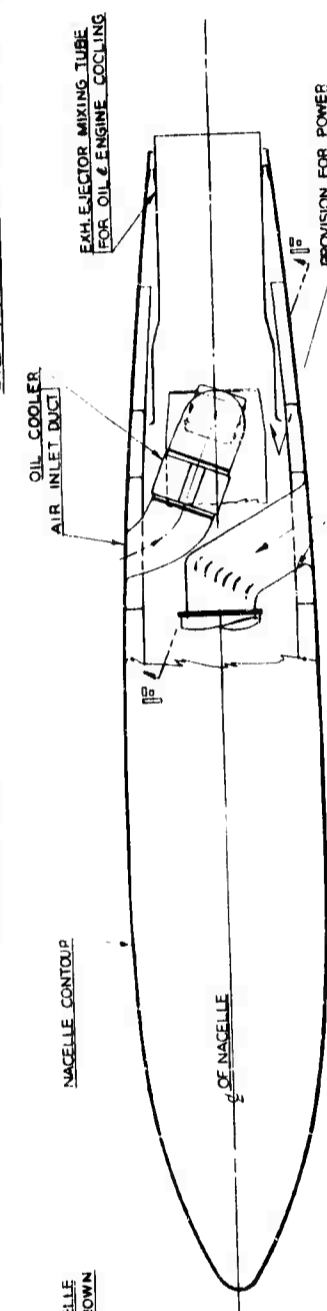
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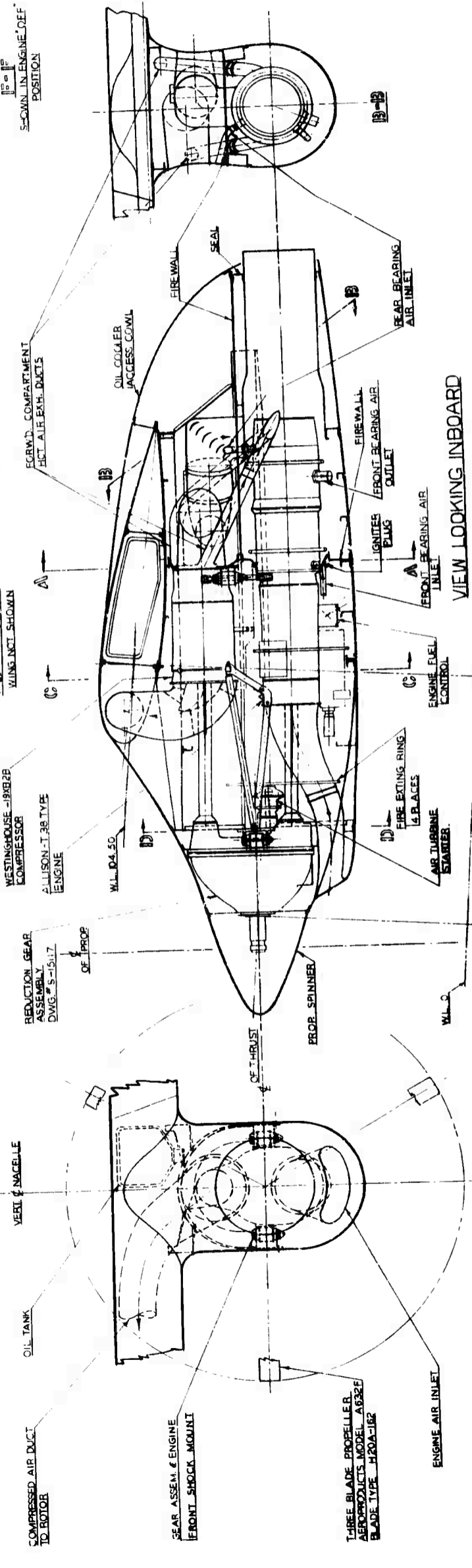
- ① REDUCTION GEAR ASSEMBLY
 - ② WESTINGHOUSE 18XB2B COMPRESSOR
 - ③ ALLISON T-38 TYPE ENGINE
 - ④ OIL COOLER
 - ⑤ OIL TANK
 - ⑥ DRAIN VALVE
 - ⑦ OIL SUPPLY
 - ⑧ OIL IN TEMP
 - ⑨ OIL COOLER BRASS
 - ⑩ RETURN OIL COOLER TO TANK
 - ⑪ HOT OIL RETURN TO COOLER
 - ⑫ OIL TANK VENT
 - ⑬ ENGINE OIL PRESS.
 - ⑭ COMPRESSOR OIL PRESS.
 - ⑮ REDUCTION GEAR OIL PRESS.
 - ⑯ SUMP DRAIN VALVE
- TOTAL CAP. - 17.5 GAL.
OIL CAP. - 4.5 GAL.



OIL SYSTEM - SCHEMATIC



PLAN VIEW



LH NACELLE VIEW LOOKING AFT

NOTES:-
1. FOR CONTINUATION OF COMPRESSED AIR DUCT SEE INBOARD PROFILE DWG. # S-1515.
ROTOR HUB ASSEMBLY DWG. # S-2512

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POWER PLANT INSTALLATION
S-15118

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MODEL 78

1. GENERAL POWER PLANT DESCRIPTION

1.1 General - The MAC Model 78 is provided with two gas turbine engines which drive either compressors for furnishing air to the pressure jet driven rotor or tractor propellers for high speed forward flight. A power unit nacelle is mounted on either side of the fuselage.

1.2 Engine - An Allison Model 301 gas turbine power section is mounted in each nacelle supplying shaft power to the propeller and compressor. A cooling air ejector is fitted to the engine exhaust pipe, drawing cooling air over the engine compressor section, combustion section, and oil radiator.

1.3 Pressure Jet Compressor - A Westinghouse 19XB axial flow compressor is employed to supply compressed air through a duct system to the rotor tip burners. This compressor is mounted directly above the engine with the compressor operated at engine speed by means of a drive shaft connected to the turbine shaft flange at the final discharge stage of the compressor.

1.4 Propeller - An Aeroproducts Model A632F propeller is mounted in the nose of each nacelle. During normal propeller operation, the propeller pitch is controlled by the engine control system. When the compressor is engaged to the engine, the propeller is held at the pitch resulting in minimum power absorption.

1.5 Gear Box - The engine drives the propeller and axial flow compressor by means of a modified Allison XT-38 gear box. The compressor shaft rotates in the opposite direction from the engine drive shaft at the same speed as the engine. Also included in the gear box section is one clutch permitting the engine to be disengaged from the gear box, and a second

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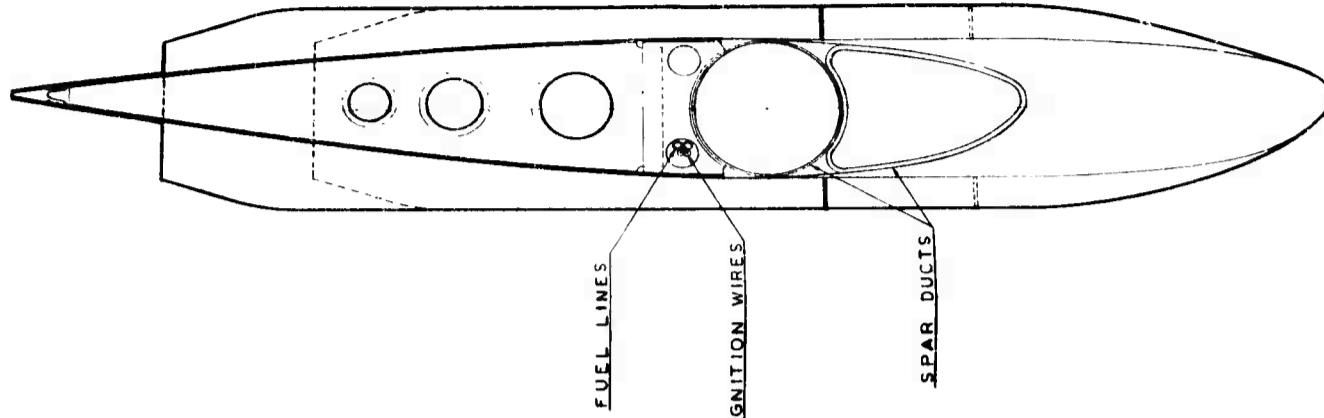
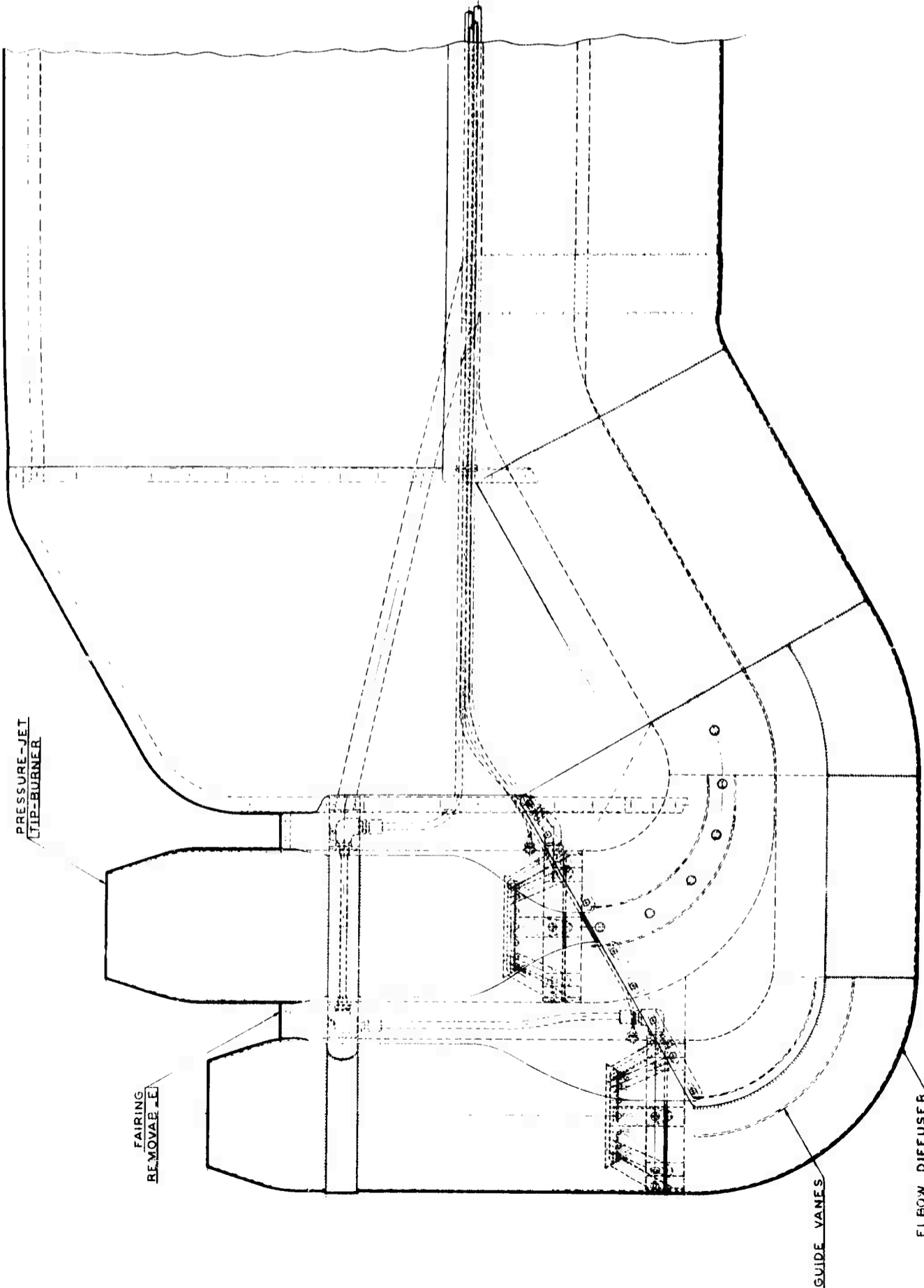
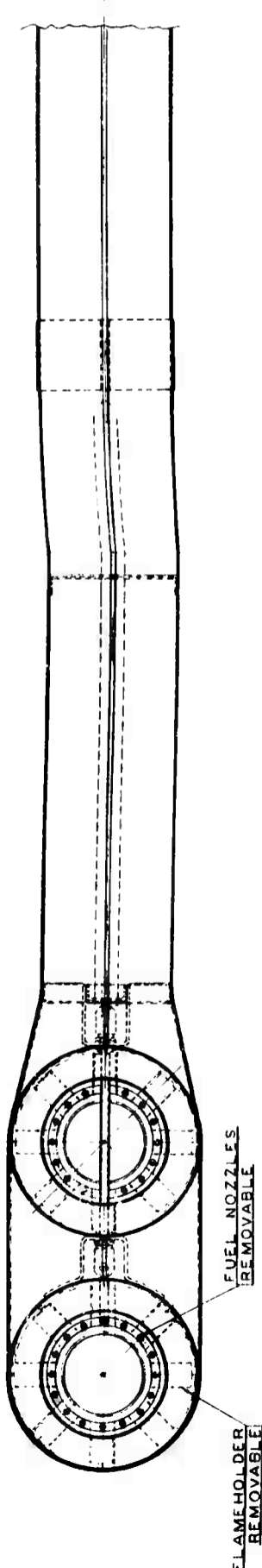
clutch which disengages the compressor. The rear face of the gear box provides the necessary accessory mounting pads and gear drives for the engine starter, generator, hydraulic pump, tachometer, and the connections for the gear box oil system and the propeller brake and governor controls. The starter gear train is connected to the engine drive shaft on the engine side of the clutch permitting the engine to be started while disengaged from the gear box.

1.6 Engine Starter - A single Airesearch air bleed gas turbine compressor is employed to drive a pneumatic starter mounted on each of the two engine gear boxes.

1.7 Rotor Tip Pressure Jets - Two burners are mounted at the tip of each of the three rotor blades. Compressed air supplied from the compressor is delivered to the tip burners through a duct system. Fuel is injected into the tip burners where combustion occurs, initiated by spark ignition.

1.8 Fuel System - The Model 78 fuel system is shown schematically in figure 1. This system was designed in accordance with specification SR-73D. Fuel cells totalling 627 gallons may be filled in a conventional method through the filler necks provided in each wing or by a single point pressure fueling fitting located on the left side of the fuselage. Refueling is made possible through the pressure fueling system by opening a shut-off valve located in the fuselage. This valve is normally closed to prevent inter-cell fuel flow through the pressure fueling system, while check valves prevent outboard fuel flow from the fuselage tank.

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Fuel is delivered to the engine fuel control by two submerged boost pumps located in the sump of the fuselage fuel cell. The fuel to each power section enters through a common selector valve. Tip burner fuel flow is controlled by a rotor driven governor-pump unit. Fuel inlet pressure for this unit is provided by the engine boost pump. When the tip burners are inoperative, fuel to the rotor system is shut off by a valve integral to the rotor governor. For single engine operation, a solenoid valve, located in the rotor hub shuts off the fuel to half of the tip burners.

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operating at the lower speed.

In order to obtain optimum pressure jet performance during single engine operation when the air flow is only one half of the normal flow, it is necessary to reduce the tip burner total exhaust nozzle area by 50 per cent. This is accomplished by closing the butterfly valves located in the aft duct in the root of each rotor blade, thus removing the three inner tip burners from the system. Actuation of this butterfly valve is automatically controlled as explained in section 2.3.

2.3 Pressure Jet Fuel Control - Fuel flow to the rotor tip burners is controlled by a rotor speed governor which meters fuel as necessary to maintain a constant selected rotor speed during all jet powered rotor operation. Thus as the rotor load changes, the pressure jet fuel flow is automatically adjusted to hold rotor speed, completely relieving the pilot of this duty. This type of rotor speed and tip jet fuel control has been performing satisfactorily in flight for a period of three years on the McDonnell XH-20 ram jet powered helicopter.

During periods of single engine operation, the flapper door of the air control valve (see section 2.2) actuates switches which closes a fuel shut-off solenoid valve located in the rotor hub. This solenoid valve shuts off the fuel flow to the three inner tip burners. Also connected to the manifold supplying fuel to the inner set of tip burners are actuating cylinders controlling the butterfly valves located in each rotor blade in the ducts supplying compressed air to the inner set of tip burners (see section 2.2). These cylinders are arranged so as to hold the butterfly valves open when fuel pressure exists in the manifold supplying fuel

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to the inner set of tip burners, and allows the butterfly valves to close when the solenoid valve shuts off the fuel in this manifold.

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SYMBOLS

A	area	ft ² unless otherwise noted
a	speed of sound	ft/sec
c _p	specific heat at constant pressure	BTU/#°R
D	hydraulic diameter	ft
F	thrust	#
F ₀	compressibility factor	$1 - \frac{M^2}{4} + \frac{M^4}{40} + \frac{M^6}{1600} + \dots$
f	wall friction factor	
f/a	fuel-air ratio	
g	acceleration of gravity	32.2 ft/sec ²
H	total head	#/ft ²
HP	horsepower	
H ₂ /E ₀	total pressure recovery	
h _v	lower heating value	18,000 BTU/# for gasoline
J	mechanical equivalent of heat	778 ft·# / BTU
l	length	ft
M	Mach number	
m	mass rate of flow	slugs/sec
P	absolute static pressure	#/ft ² unless otherwise noted
P _T	absolute total pressure	#/ft ² unless otherwise noted
Q	volume rate of flow	ft ³ /sec unless otherwise noted
q	dynamic pressure	#/ft ² unless otherwise noted
q ₀	impact pressure qF ₀	#/ft ² unless otherwise noted
R	gas constant	ft·# / #°R

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RN	Reynolds number		
SH	shaft horsepower		
T	static temperature		OR
T _T	total temperature		OR
V	velocity		ft/sec
V _T	tangential velocity		ft/sec
v	specific volume		ft ³ /#
W	weight rate of flow		#/sec
α	angle of attack		degrees
γ	ratio of specific heats		
Δ	used to represent a change in another quantity		
δ	$P_{T2}/14.7$ when P_{T2} is expressed in PSIA		
θ	$T_{T2}/18.4$		
η	efficiency		
ρ	mass density		slugs/ft ³
ω	rotational velocity		radians/sec

Subscripts

0	free stream	i	inlet
1, 2, etc.	station numbers	j	jet
a	air	ot	total loss
b	burning	p	primary
c	compressor	s	secondary
f	fuel	TR	temperature rise
g	gas - products of combustion		

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3. ANALYSIS OF POWER PLANT DUCTS

3.1 General - The analysis of the Model 78 duct system is presented in accordance with the U. S. Navy Aeronautical Specification **NAVAR 88-150**, "Specification for the Calculation of Duct losses, Turbo-Jet and Gas Turbine Engines".

This section includes the analysis of the inlet ducts supplying air to the two Allison Model 501 power sections, the inlet ducts supplying air to the two Westinghouse 19XB compressors, and the engine exhaust system including the cooling ejector.

3.2 Description of Ducts - The engine inlet duct is approximately five feet long with a well-rounded lip. The duct increases in cross-sectional area from 160 square inches at the inlet to 186 square inches at the engine face. The annular flow area at the engine compressor face is 160 square inches.

The 19XB compressor inlet duct is about three feet long and has a well-rounded lip. The duct increases in cross-sectional area from 140 square inches to 160 square inches at a station near the compressor face. The annular area at the compressor face is 138 square inches.

The engine exhaust tailpipe is shrouded with an ejector which is split by radial dividers into three parts drawing cooling air through the forward engine compressor compartment, the engine combustion chamber compartment, and the oil cooler.

Figure 3 presents a schematic of the ducting arrangement and cross-sectional areas.

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3.3 Results of Analysis - The total pressure recovery for the engine and the compressor inlets is shown in Figure 4. It is based on a series of wind tunnel and flight tests conducted by MAC on comparable configurations (references 1 and 2).

Tests of similar installations indicate that the engine and compressor inlet ducts should have a total pressure recovery of 92% under sea level static conditions without benefit of propeller ram.

Tables I and II present the detailed description of the inlet and compressor inlet ducts and compressor. The inlet duct has a diameter of 3, a conical angle of 10 degrees, a conical angle of diffusion of 10 degrees, and a normal velocity of flow axial to the axis of 0.5 Mach number. The compressor inlet duct has a diameter of 8 for the engine and a diameter of 10 for the compressor.

The inlet duct (Figure 4) is a simple cylindrical duct with a conical inlet and a conical outlet. The inlet and outlet are both 10 degrees. The inlet duct is 3 in diameter and the outlet duct is 8 in diameter. The inlet duct is 10 in long and the outlet duct is 10 in long. The inlet duct is 10 in long and the outlet duct is 10 in long.

Data presented in reference 1 shows that a secondary air flow of 0.3 l/s/sec is induced by an inlet flow of 0.1 l/s/sec (static sea level, normal power). This ejector action is considered satisfactory for the power plant cooling requirement. Tests of a MAC design ejector burner of 10 in diameter and of ejector inlet diameter of 10 in at low cooling losses may be expected from this test. Cross section area, equivalent conical angle of diffusion and exhaust Mach number for an axial flow burner for the exhaust ducts are presented in Figures 11 and 12.

The analysis of the proposed air duct system for the compressor inlet and the engine inlet, and the inlet ducts, is included in the pressure jet performance report.

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4. ALLISON MODEL 501 LOWER SECTION PERFORMANCE

4.1 Engine Performance - The Allison Model 501 gas turbine engine performance was evaluated by the method presented in the engine specification (reference 5). Reference 5 represents the most recent Navy approved specification for the power section performance. Inlet duct total pressure recovery was based upon the duct analysis presented in section 3. Values of H_2/H_0 from figure 4 were used for the engine performance calculations.

During powered-rotor flight with the propeller in the pitch resulting in minimum power absorption it is assumed that the propeller absorbs 6% of the available engine shaft horsepower. Accordingly the power available to drive the compressor, as presented in figure 13 has been reduced by 6%.

As noted in section 2.1 the engine will be operated at constant normal speed (14,000 rpm) during all periods that it is engaged to the compressor. This necessitates, under certain conditions, that the turbine inlet temperature exceed the normal rated temperature, but under no conditions will it exceed the allowable military temperature. In view of the allowable operating time of 30 minutes at military power this type of operation is considered satisfactory especially as it is accomplished at a reduced engine speed. Figure 13 presents a plot of horsepower available and horsepower required versus altitude for the most critical conditions at 14,000 rpm and a turbine-inlet temperature of 1935°R.

Figures 14 through 22 present shaft horsepower, net jet thrust, and fuel flow for propeller operation of each engine in the anticipated operating ranges of flight velocity and altitude. Performance was determined in accordance with the engine specification (reference 5) using the estimated inlet total pressure recovery presented in figure 4.

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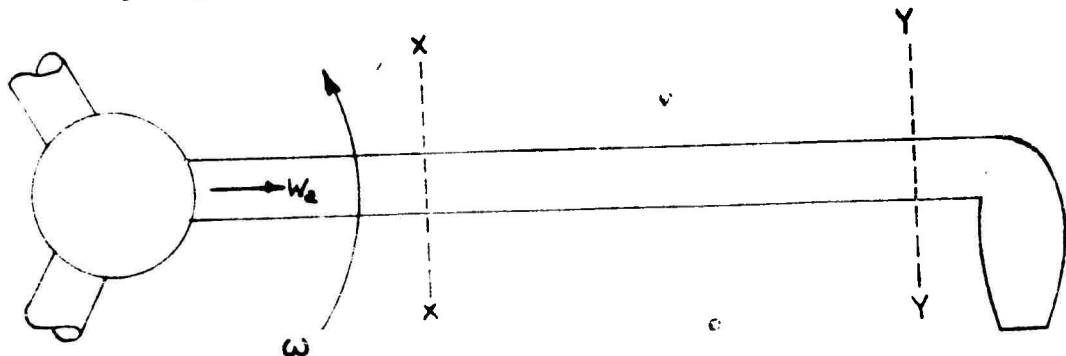
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Considering a rotor blade as shown below with air flowing through the blade passage and being burned in the tip burner, the power required to



accelerate the air from the tangential velocity at station X to the tangential velocity at station Y is:

$$\text{POWER} = \frac{W_a (V_{Ty}^2 - V_{Tx}^2)}{2g} \quad 2$$

also, the power required for isentropic compression of the air from station X to station Y is: (reference 7).

$$\text{POWER} = J C_p T_{Tx} \left[\left(\frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right] W_a \quad 3$$

Equating equation 2 and equation 3

$$\frac{W_a (V_{Ty}^2 - V_{Tx}^2)}{2g} = J C_p T_{Tx} \left[\left(\frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right] W_a$$

$$\left(\frac{P_{Ty}}{P_{Tx}} \right)^{\frac{\gamma-1}{\gamma}} = 1 + \frac{(V_{Ty}^2 - V_{Tx}^2)}{2g J C_p T_{Tx}}$$

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$$\frac{P_{T_y}}{P_{T_x}} = \left[1 + \frac{(V_{T_y}^2 - V_{T_x}^2)}{2gJ C_p T_x} \right]^{\frac{\gamma}{\gamma-1}}$$

Since $J C_p = R(\gamma/\gamma-1)$, the equation for the pressure ratio developed from station X to station Y may now be written:

$$\frac{P_{T_y}}{P_{T_x}} = \left[1 + \frac{(V_{T_y}^2 - V_{T_x}^2)}{2gR T_x (\gamma/\gamma-1)} \right]^{\frac{\gamma}{\gamma-1}}$$

2.2.2 Pressure Losses - To determine the pressure losses in the system due to the flow of air through the ductwork, the tip burner, a combustor, analysis was made for each calculated tip burner per range of γ . The method and data of reference 2 were used to evaluate the pressure losses. Pumping gains due to rotor rotation using the method presented in section

2.1 and momentary pressure loss (estimated from reference 2) were also taken into consideration. After the pressure available in the tip burner for combustion was determined, the tip burner performance was evaluated in the manner discussed below.

2.2.3 Design Condition - A design condition was first selected which was the basis for determining the nozzle exit area of the tip burner. The design condition chosen was a corrected air pressure ratio (P/P_0) of 10,000

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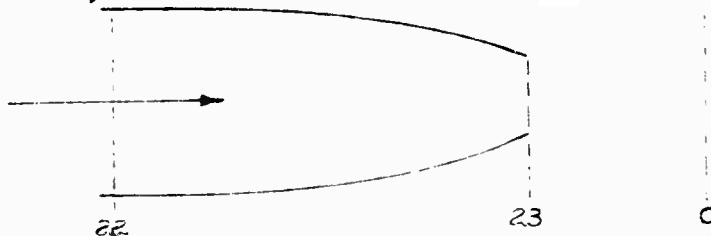
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operation at a ... (p₂₃/p₂₂) ... is
 approximately ... compressor efficiency ...
 This pressure ratio is sufficient to allow the compressor stall to indicate
 a safe design point. Since the ... at this design condition
 indicates the pressure is sufficient to maintain a ... velocity through
 the nozzle exit area, the area was selected on this basis.



$$P_w = RT$$

$$w = \frac{RT}{P} = \frac{AV}{W_g}$$

$$\frac{W_g RT_{22}}{P_{23}} = A_{23} V_{23}$$

where w is velocity at nozzle exit

$$V_{23} = \sqrt{\gamma RT_{23}}$$

6

From critical pressure ratio ...

$$\frac{P_{23}}{P_{T22}} = \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}$$

$$P_{23} = P_{T22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}$$

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Substituting equations 6 and 7 into equation 5 the following expression is obtained for the nozzle area.

$$\frac{W_g RT_{23}}{P_{t22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}} = A_{23} \sqrt{\gamma g RT_{23}} \quad 8$$

$$A_{23} = \frac{W_g RT_{23}}{P_{t22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}} \sqrt{\gamma g RT_{23}}} \quad 9$$

In view of the need for burning efficiency it was desired that temperatures above 4,000° R. be obtained in the combustor chamber. Test data on a similar combustor required a reference indicator burner to generate such a temperature and higher. Therefore the nozzle exit area was determined for the maximum air flow at a total temperature of 4,000° R. Internal boundary layer cooling was assumed over wall. The wall cooling is presumed to be a constant value.

The relation between total temperature and static area temperature at Mach 1 is:

$$T_T = T \left(1 + \frac{\gamma-1}{2} M^2\right) \quad 10$$

SINCE $M = 1.0$

$$T_{23} = 4000 \left(\frac{2}{\gamma+1}\right)$$

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From equation 9, the nozzle area now becomes:

$$A_{23} = \frac{W_g R (4000) \left(\frac{2}{\gamma+1}\right)}{P_{22} \left(\frac{2}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}} \sqrt{\gamma g R (4000) \left(\frac{2}{\gamma+1}\right)}} \quad 11$$

$$\frac{f}{a} = \frac{C_p \Delta T}{h_v \eta_B}$$

Assume a burning efficiency of 90% and a lower heating value of gasoline = 18,000 BTU/#.

$$\frac{f}{a} = \frac{C_p \Delta T}{16,200} \quad 12$$

Since the values of γ and C_p are dependent upon temperature and fuel-air ratio it is now necessary to make successive approximations for fuel-air ratio and cycle through equations 11 and 12 until both equations are satisfied.

P_{22} is determined from the duct analysis as described in section 5.2.2 and a trial and error solution is readily made since there is only one fuel-air ratio that will give a total temperature of 4000°R.

On this basis a total nozzle area of 0.938 square feet was determined. This total nozzle area is divided by the number of tip burners to obtain the area per burner. For the six burners the effective nozzle exit area per burner is 22.5 in.² or 5.35 in. in diameter.

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5.2.4 Performance Calculations - For other corrected compressor speeds and pressure ratios the combustion temperature may be determined since the area has been determined. Transposing equation 8:

$$\frac{T_{23}}{\sqrt{T_{23}}} = \frac{A_{23} \sqrt{\gamma g R} P_{T_{23}} \left(\frac{\gamma}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}}{W_g R}$$

$$T_{23} \left[\frac{A_{23} \sqrt{\gamma g R} P_{T_{23}} \left(\frac{\gamma}{\gamma+1}\right)^{\frac{\gamma}{\gamma-1}}}{W_g R} \right]^2$$

13

Using equation 12 and 13 and determining $P_{T_{22}}$ from the rotor duct analysis T_{23} may be determined for any compressor condition. Although equation 13 applies only when $P_{T_{23}}$ is above the critical pressure ratio, this condition covers most of the system operating range.

Fuel flow is determined by:

$$W_f = \frac{f}{2} W_a = \frac{C_p \Delta T}{16,200} W_a$$

14

Total temperature at the nozzle exit for sonic velocity is

$$T_{T_{23}} = T_{23} \left(\frac{\gamma+1}{2}\right)$$

15

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MODEL 78

The jet velocity is obtained from equation 6. Jet thrust is obtained as follows:

$$F_J = \eta_N \frac{W_a}{g} (V_{23} - V_T) + (P_{23} - P_0) A_{23}$$

16

η_N = nozzle efficiency (assumed .95)

A sample calculation is presented in table 7. The first section of the calculation is the rotor system duct analysis to determine the pressure available for burning. The latter part of the calculation presents the tip burner performance for the available pressure. It was assumed that the division of air flow through the two flow passages in the rotor blade (station 17 through 23) was equal, even though the cross-sectional areas are slightly different (22.1 and 24.3 square inches per blade). The actual division of the air flow will be governed by the back pressure of each tip burner due to burning. The total temperature, fuel flow, and jet thrust were calculated for each of the two tip burners and the average total temperature was used for the total temperature of the gas.

Curves of corrected performance over a range of thrust values are presented in figures 26 and 27. Corrected values are presented in the general curves since they are independent of the compressor inlet conditions. The correction factors used are the standard factors used in correction of jet engine performance. Reference 10 presents the derivation of these factors. These factors are:

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MODEL 78

$$\delta = P_{T_2}/14.7 \text{ where } P_{T_2} \text{ is expressed in PSIA}$$

$$\Theta = T_{T_2}/318.4 \text{ where } T_{T_2} \text{ is expressed in degrees Rankine}$$

Several numerical examples were checked for various altitudes within the performance range of Model 78. Agreement of corrected performance with that determined by using the actual temperature and pressure was obtained.

Actual tip burner thrust and overall fuel consumption are presented in figures 28 through 31 for the anticipated operating range of the tip burners.

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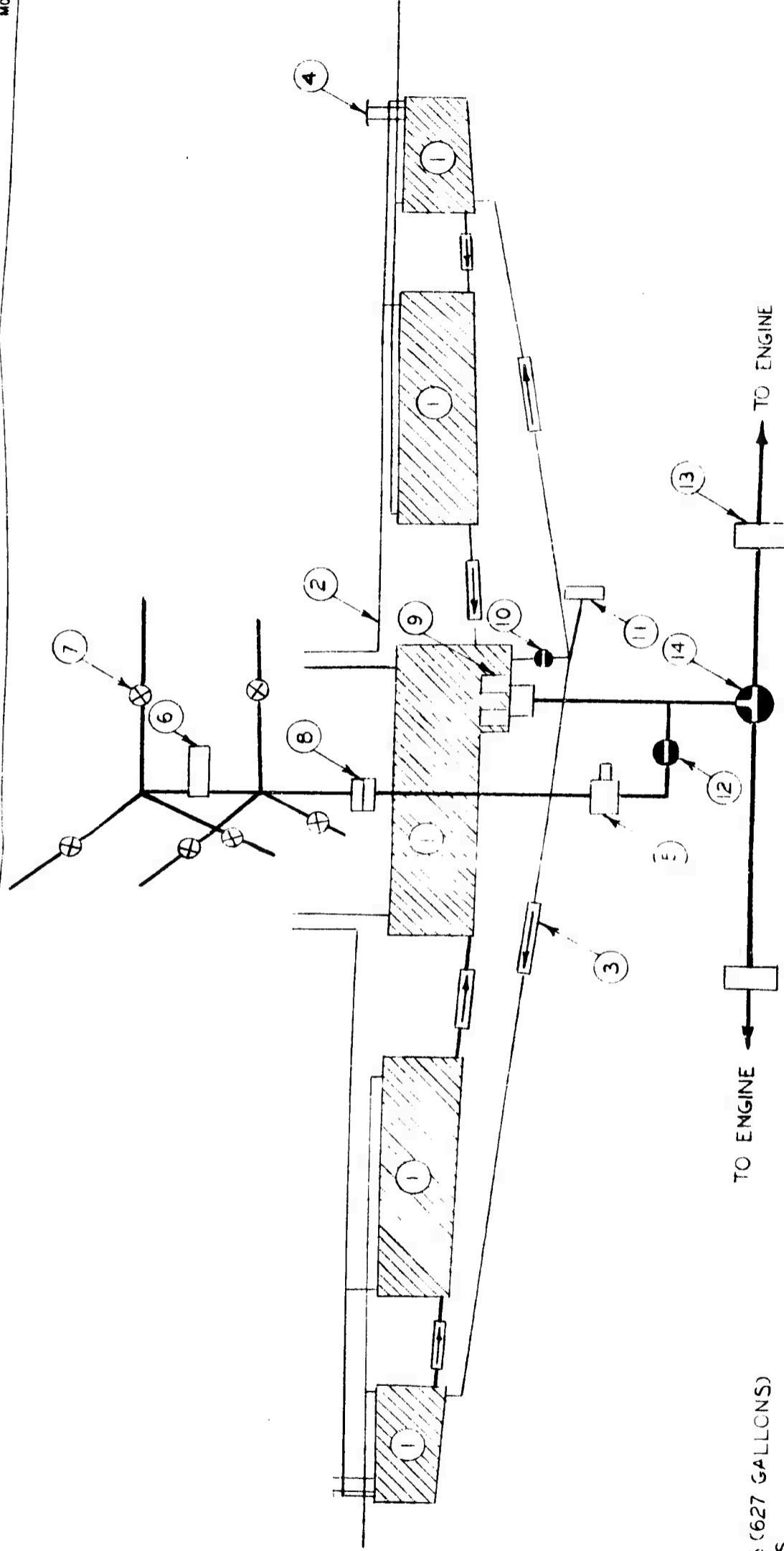
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MODEL 78

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10. Russ, D. G. Correlation Parameters and Correction Factors for Aircraft Turbo Machinery Naval Air Material Center Philadelphia Report No. 49CTMI/1, July 31, 1949.

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- 1 FUEL CELLS (627 GALLONS)
- 2 VENT LINES
- 3 CHECK VALVES
- 4 FILLER NECK
- 5 ROTOR DRIVEN PUMP-GOVERNOR UNIT
- 6 SHUT OFF VALVE (FOR SINGLE ENGINE HELICOPTER FLIGHT ONLY)
- 7 ORIFICE
- 8 RECTARY JOINT
- 9 BOOST PUMP
- 10 SHUT OFF VALVE - NORMALLY CLOSED
- 11 PRESSURE FUELING & DEFUELING FITTING
- 12 SHUT OFF VALVE
- 13 FILTER
- 14 SELECTOR VALVE-LEFT-RIGHT-BOTH-OFF

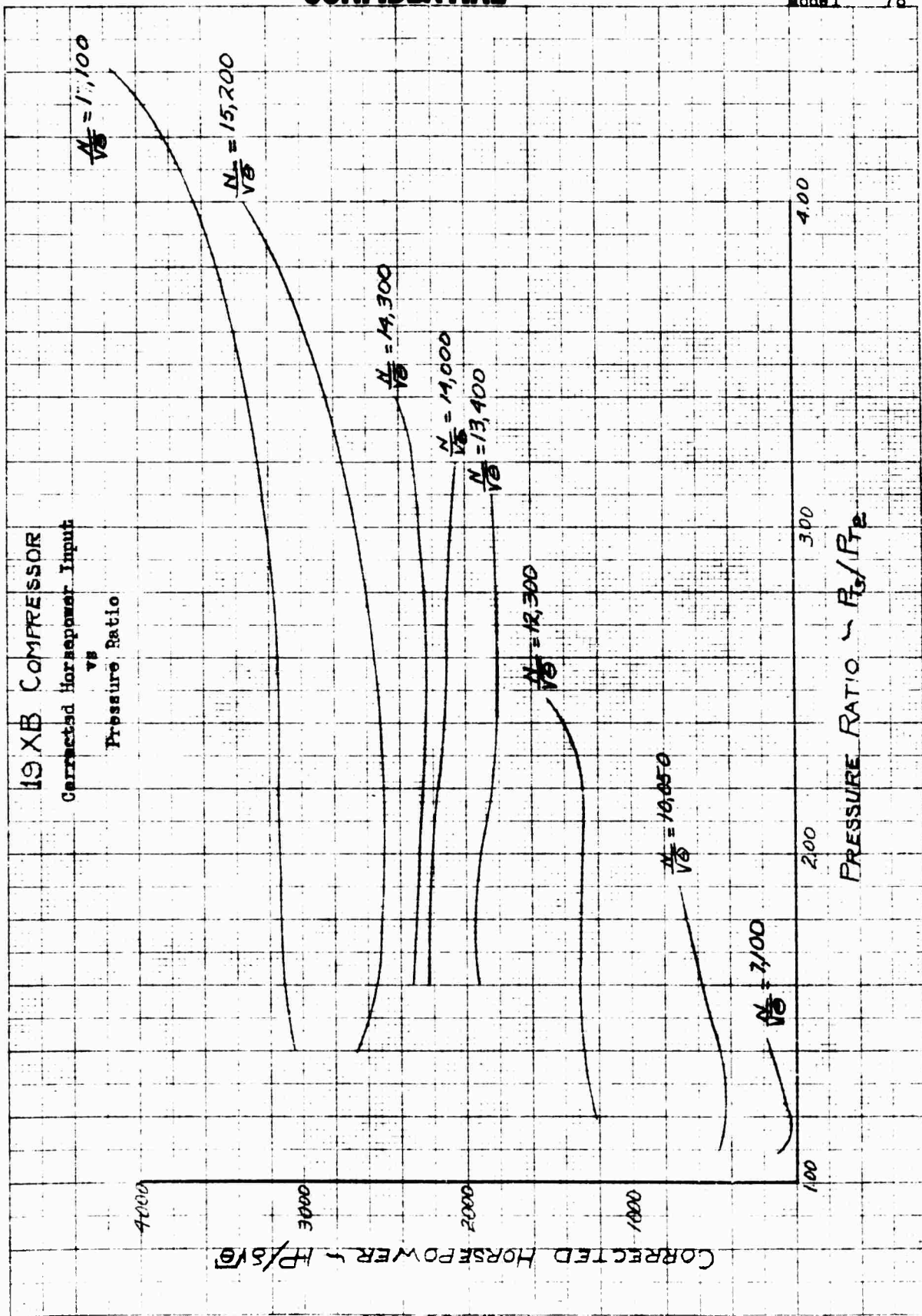
MODEL 78
FUEL SYSTEM SCHEMATIC

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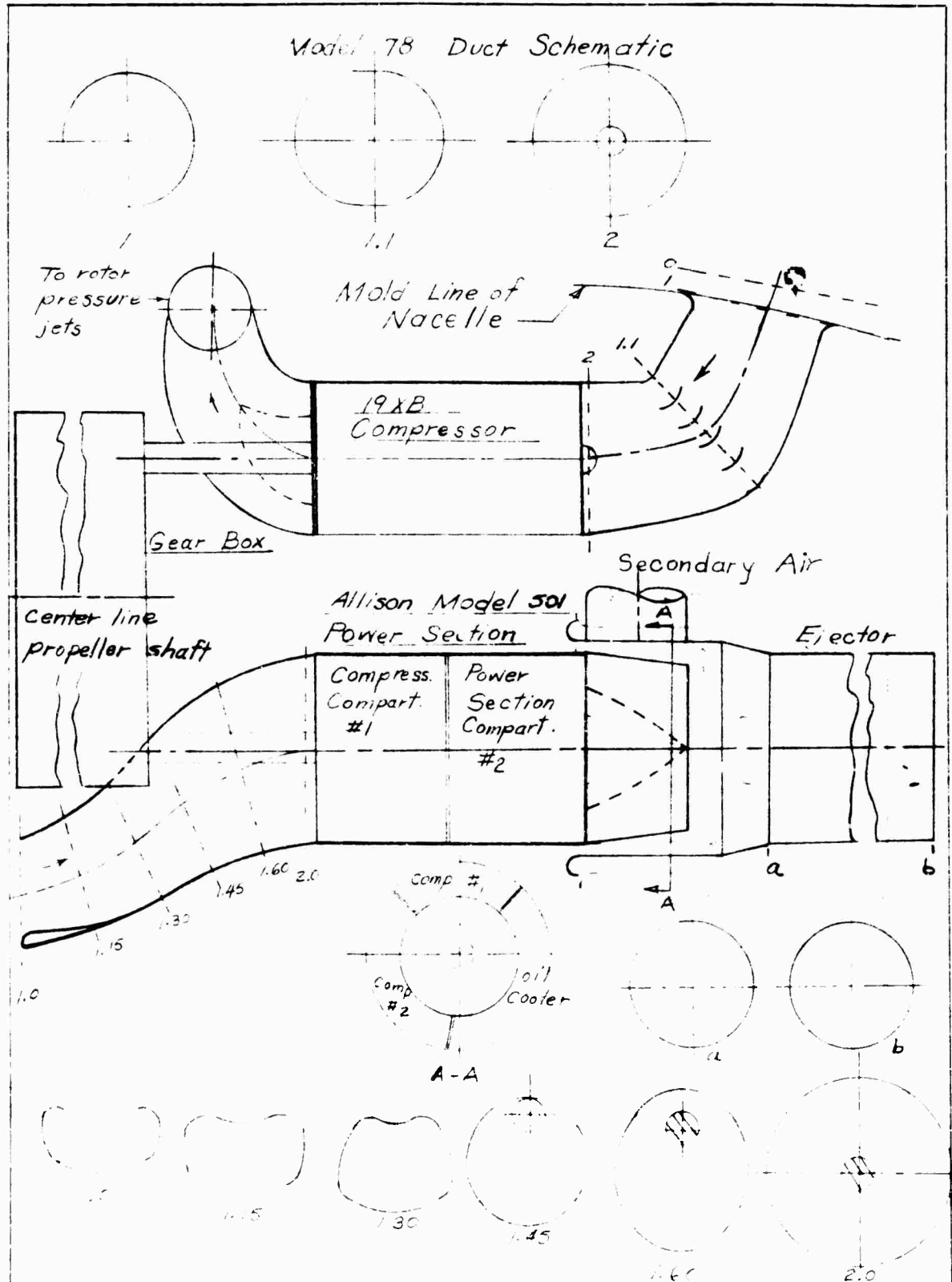
PLUMMER ENGINE CO.

10,000 11, 10 101000 11, 10 101000 11, 10 101000
Engineering 10 10 10

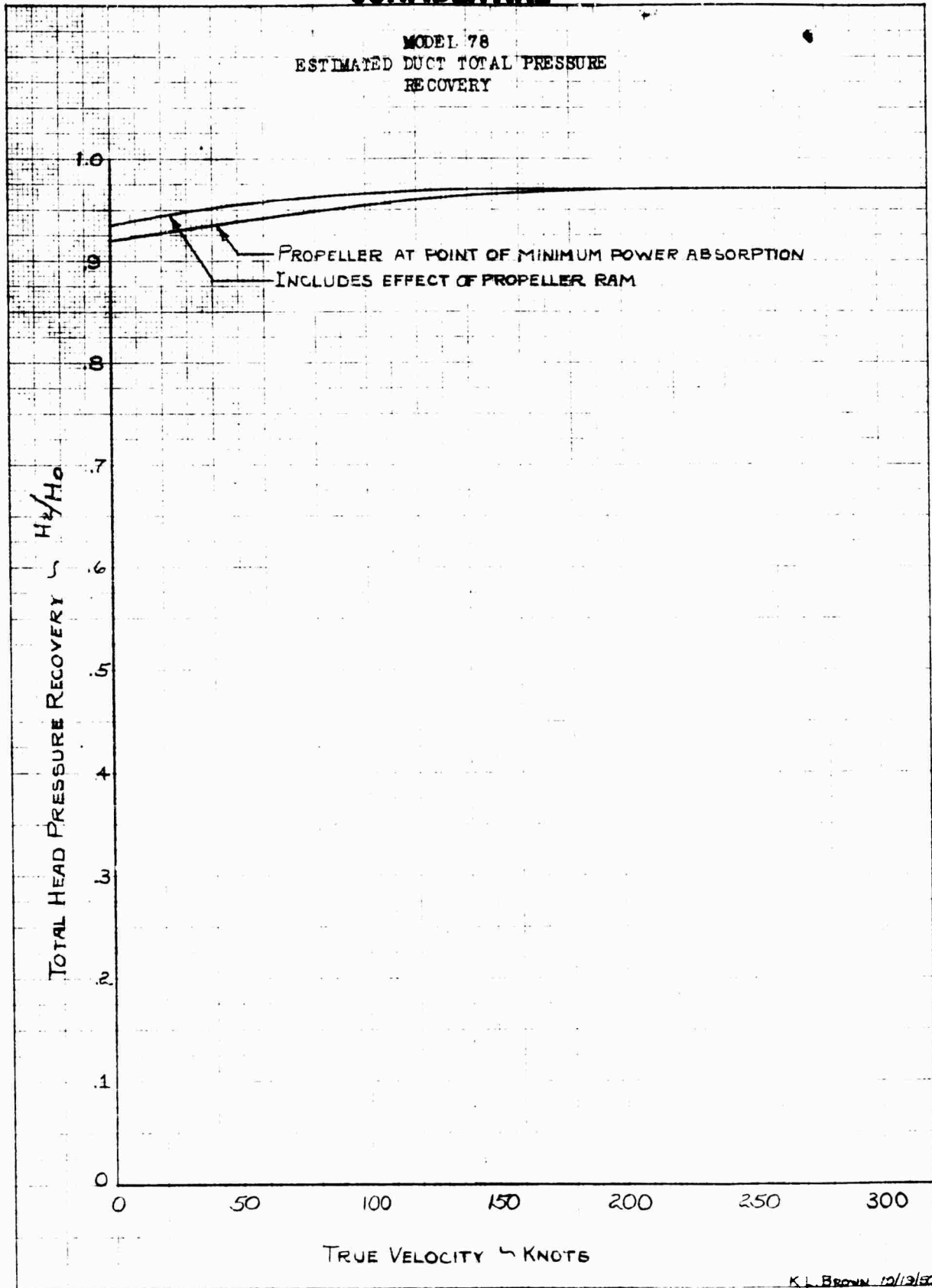


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FIG. 2



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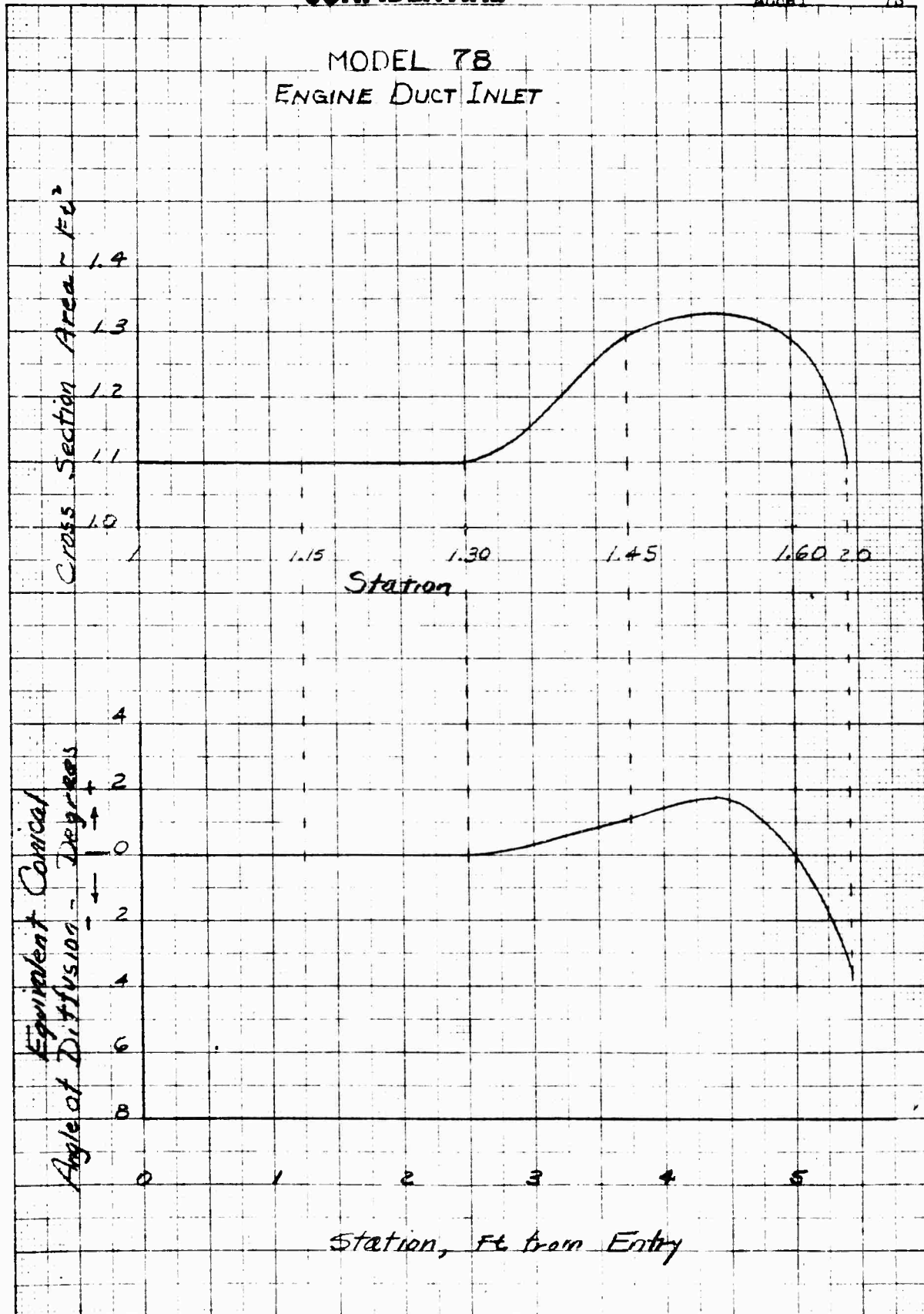


PELLER & ESNER CO.

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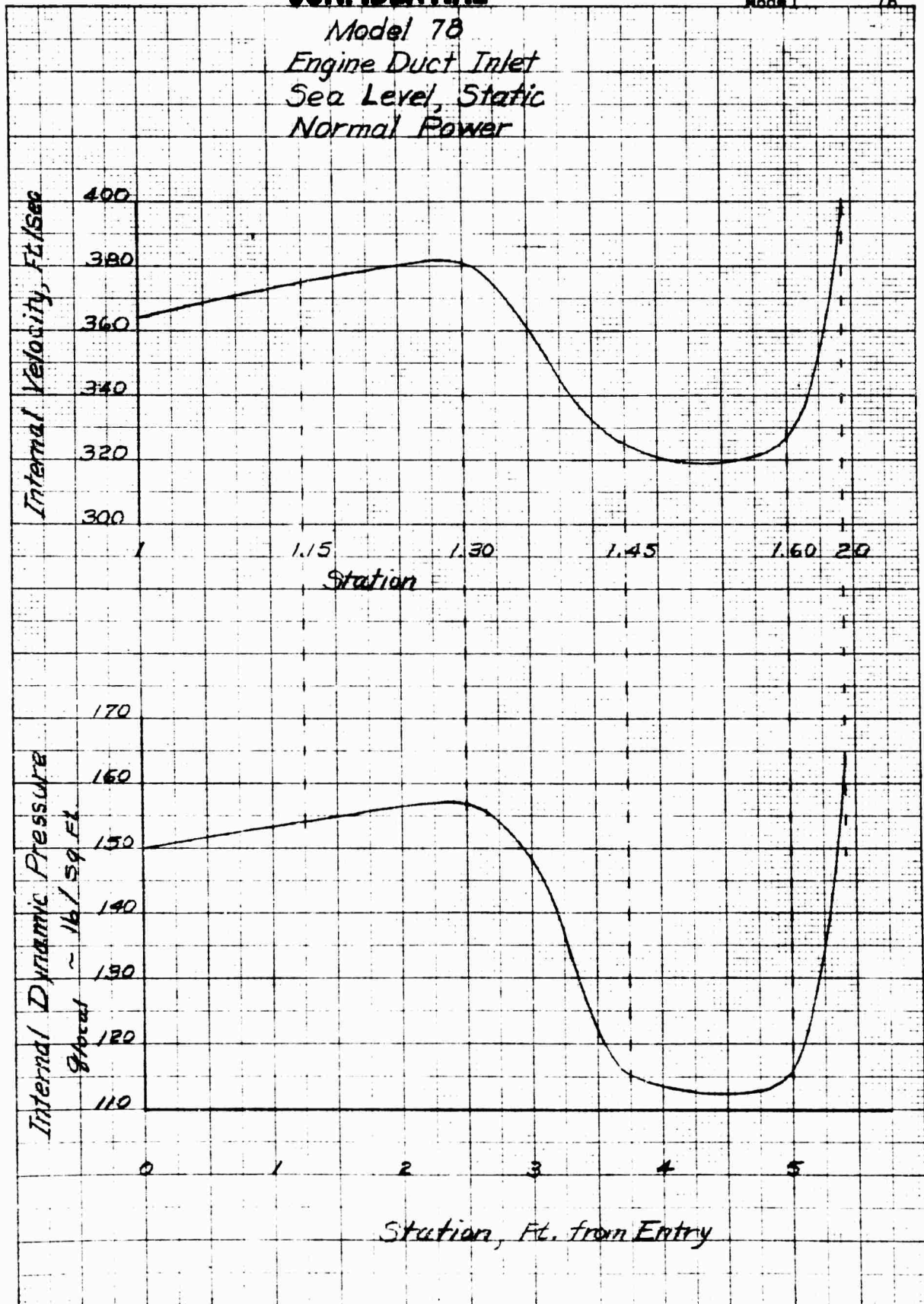
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FIG. 4



REPRODUCED FROM NACA REPORT 1135

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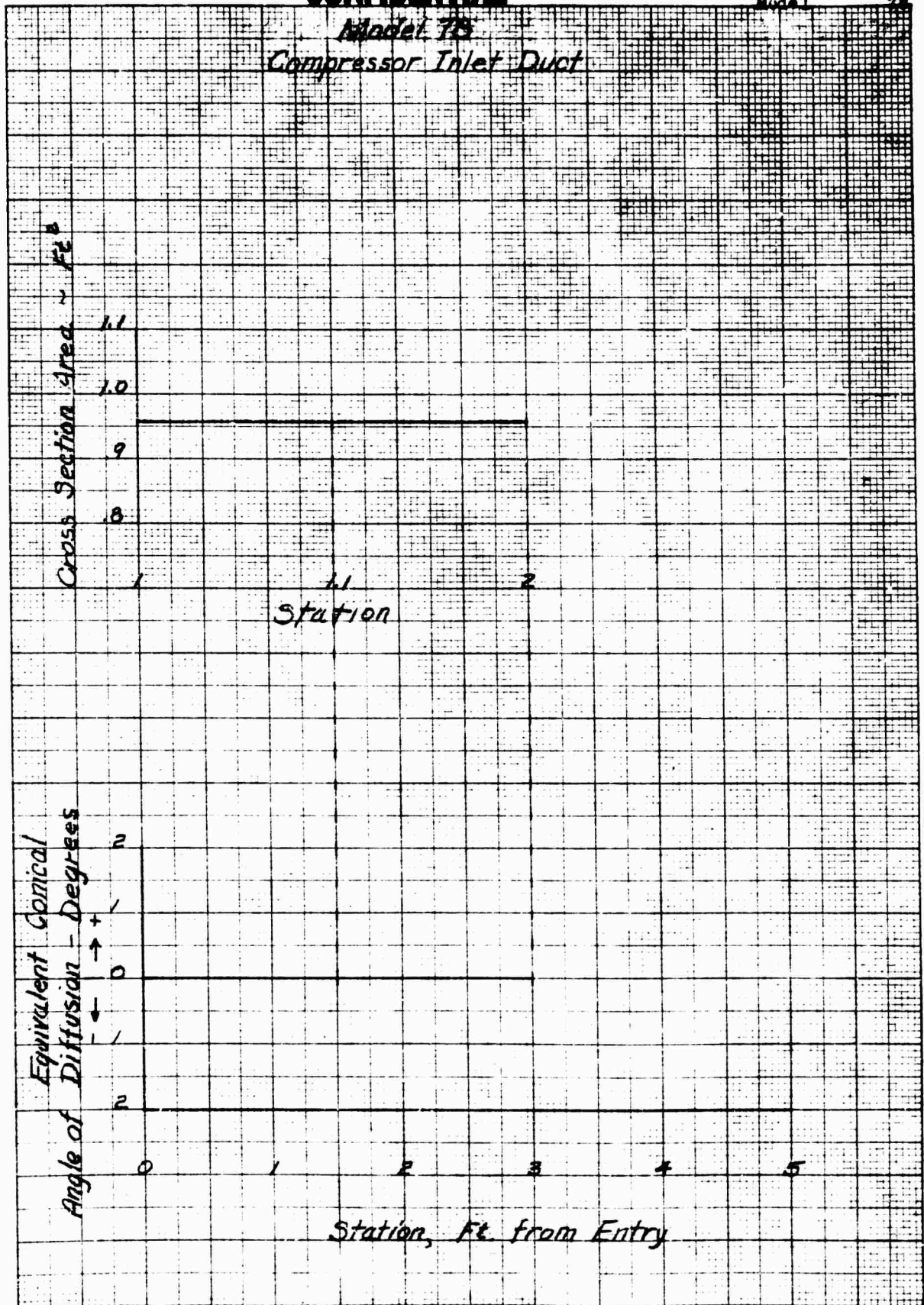
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DAVID S. HARRIS

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FIG 6

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*Model 7B
Compressor Inlet Duct*



REF. FILE IN ENSTR. CO. N. Y. NO. 35821
Date: 12/20/50

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FIG. 7

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MODEL 72

Conditions: Sea Level, Static, Normal Power (14,000 RPM)

$$W = 575 \# \quad \rho = 0.002376 \text{ slugs/ft}^3 \quad \gamma = 0.175$$

$$W_0 = 28.45 \#/\text{sec} \quad W_f = 473 \#/\text{hr} = 1.314 \#/\text{sec}$$

$$W_0/W_f = 0.144 \quad \rho_f = 1.353 \quad T_f = 1295^\circ\text{R}$$

$$F = \gamma \frac{W_0}{\rho} \frac{V_0}{V_f} - W_f V_0 = 0.175 \times \frac{28.45 \times 222.2}{0.002376} - 1.314 \times V_0 = 0 = 575$$

$$V_0 = \frac{575 \times 222.2}{0.175 (28.45)} = 676 \text{ Ft/sec}$$

From the ideal jet velocity equation, solve for the nozzle pressure ratio.

$$V_0 = \sqrt{\frac{2 \gamma}{\gamma - 1} \frac{P_0}{\rho} \left[1 - \left(\frac{P_0}{P_f} \right)^{\frac{\gamma - 1}{\gamma}} \right]}$$

$$\frac{676^2}{2} = \frac{2 \times 0.175 \times 222.2 \times 53.9 \times 1.353 \times 1295}{1.353 - 1} \left[1 - \left(\frac{P_0}{P_f} \right)^{\frac{1.253 - 1}{1.253}} \right]$$

$$\frac{P_0}{P_f} = (4.732)^{3.875} = .90$$

$$P_f/P_0 = \frac{1}{.90} = 1.12$$

The exhaust nozzle pressure ratio is 1.12.

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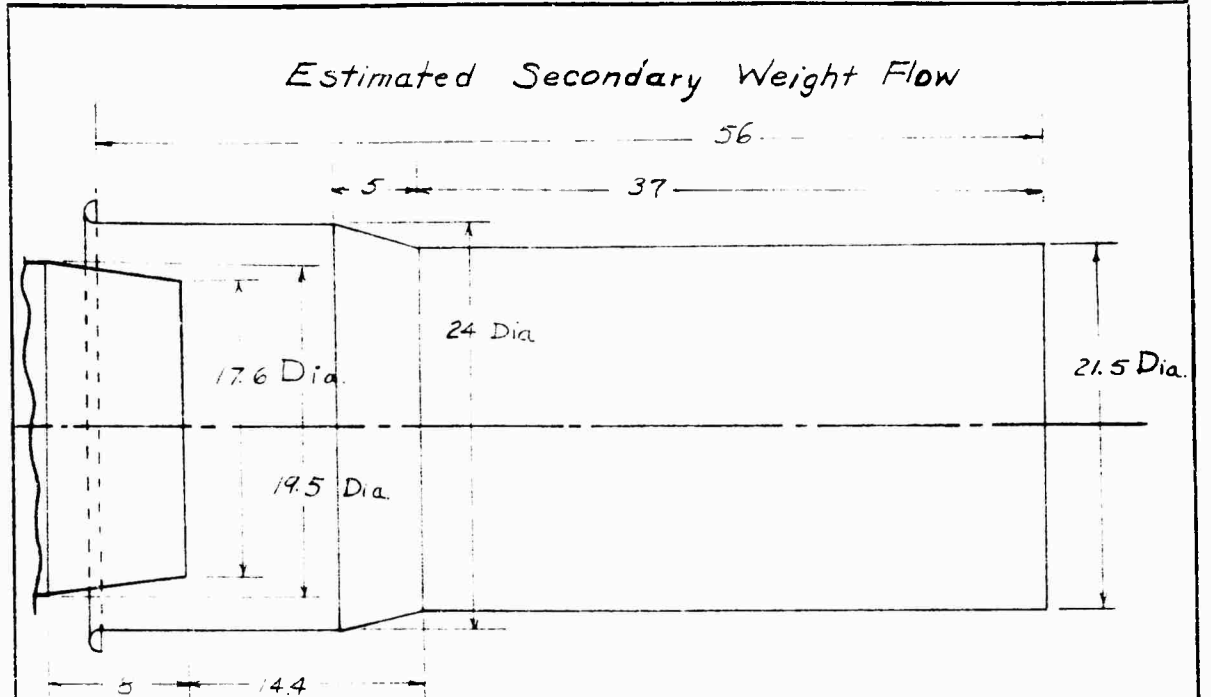
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MODEL 78



The corrected weight flow ratio from reference 4.

$$\frac{W_S}{W_P} \sqrt{\frac{T_S}{T_P}} = .14$$

$$T_S = 520^\circ R$$

$$T_P = 1295^\circ R$$

$$W_S/W_P = .14 \sqrt{\frac{T_P}{T_S}} = .14 \sqrt{\frac{1295}{520}} = .222$$

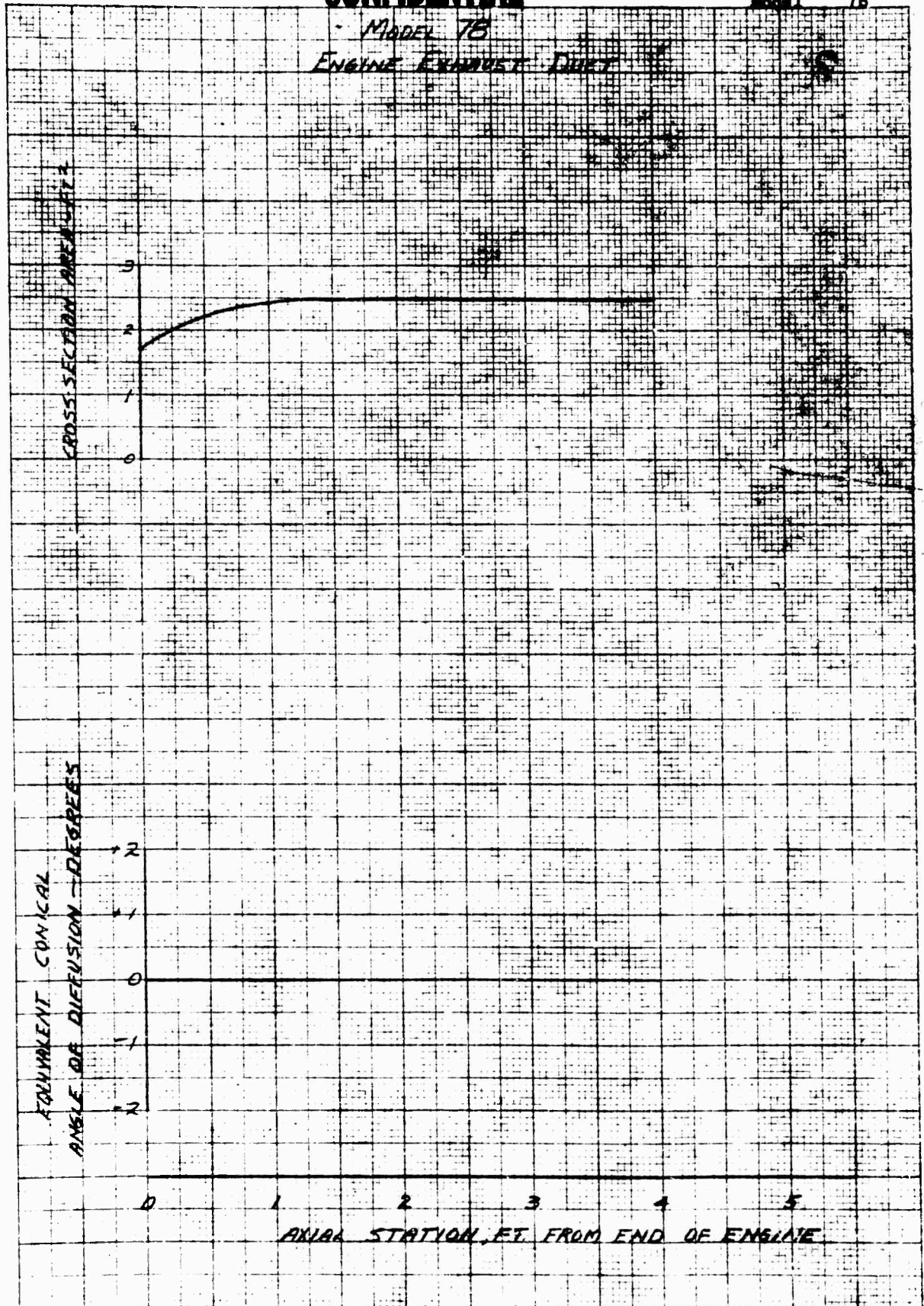
$$W_P = 28.45 \text{ lb/sec}$$

$$W_S = .222 \times 28.45 = 6.28 \text{ lb/sec}$$

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Report 1905
Model 78



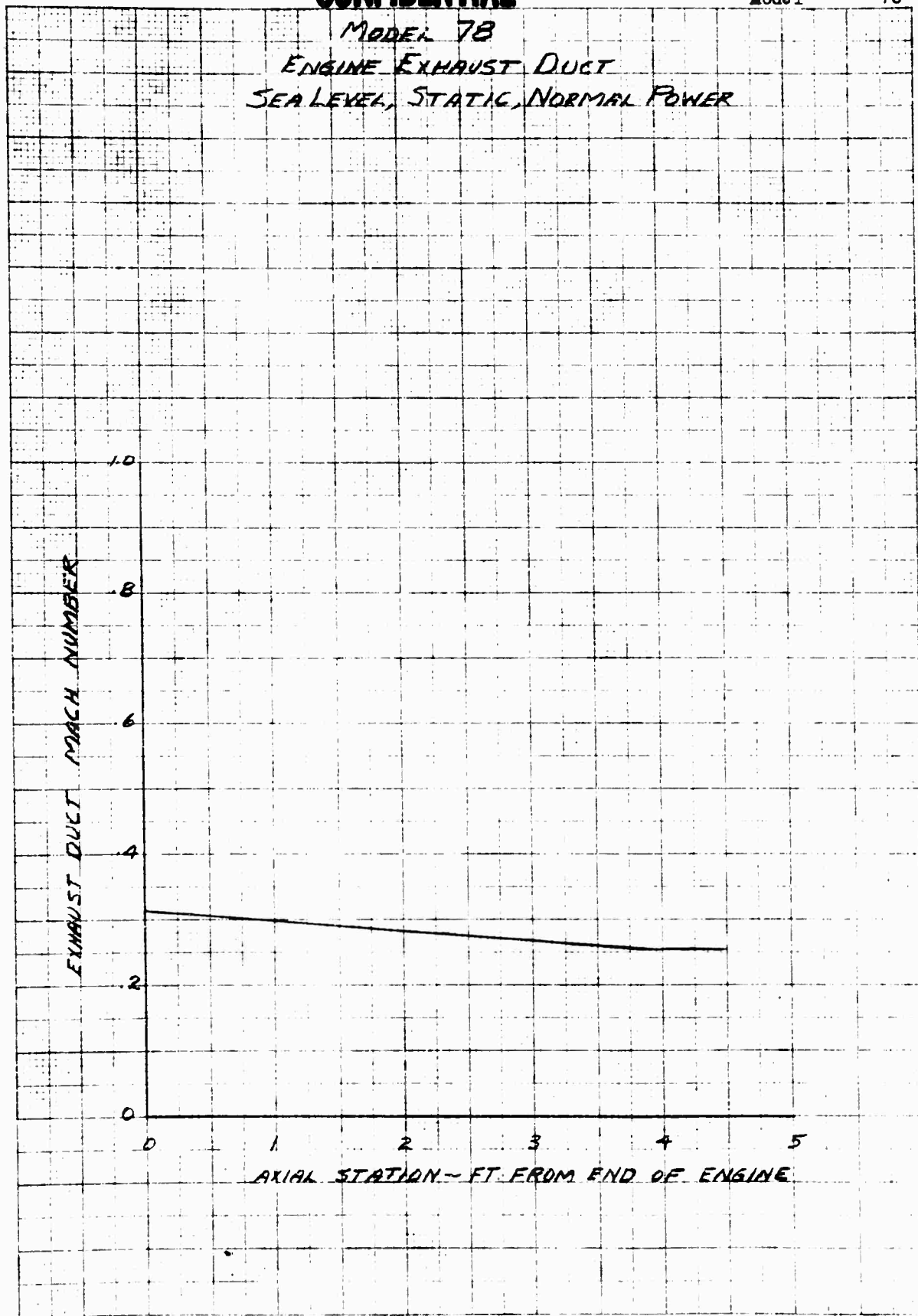
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FIG. 11

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MODEL 78
ENGINE EXHAUST DUCT
SEA LEVEL, STATIC, NORMAL POWER



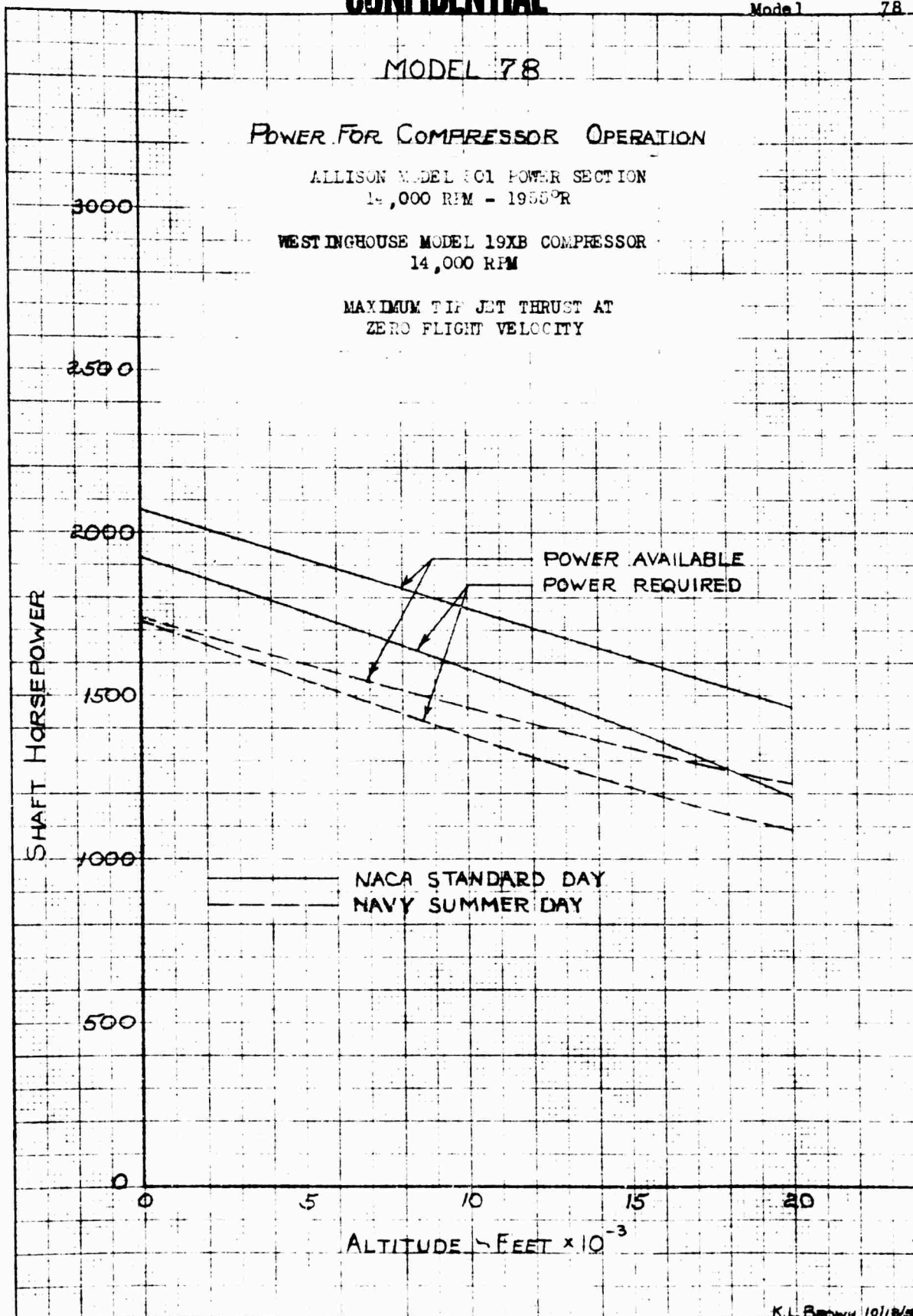
REUPPEL & L. SER CO.

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FIG 12

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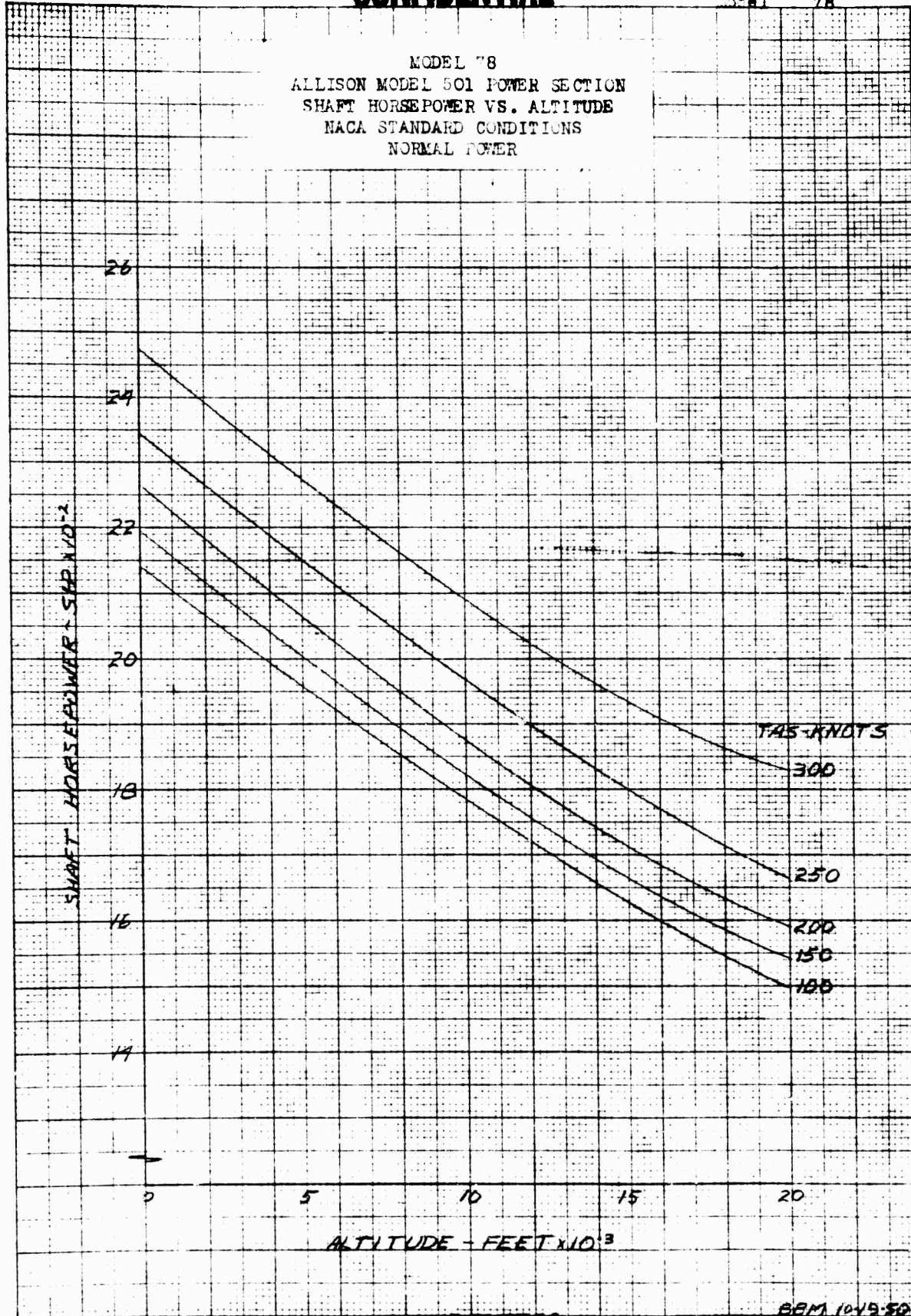


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FIG 13

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KAUFMAN & ESSER CO. N. Y. NO. 358-11
18 X 18 to the 1/4 inch 5th floor amended
U.S. PAT. NO. 2,454,444

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FIG 14

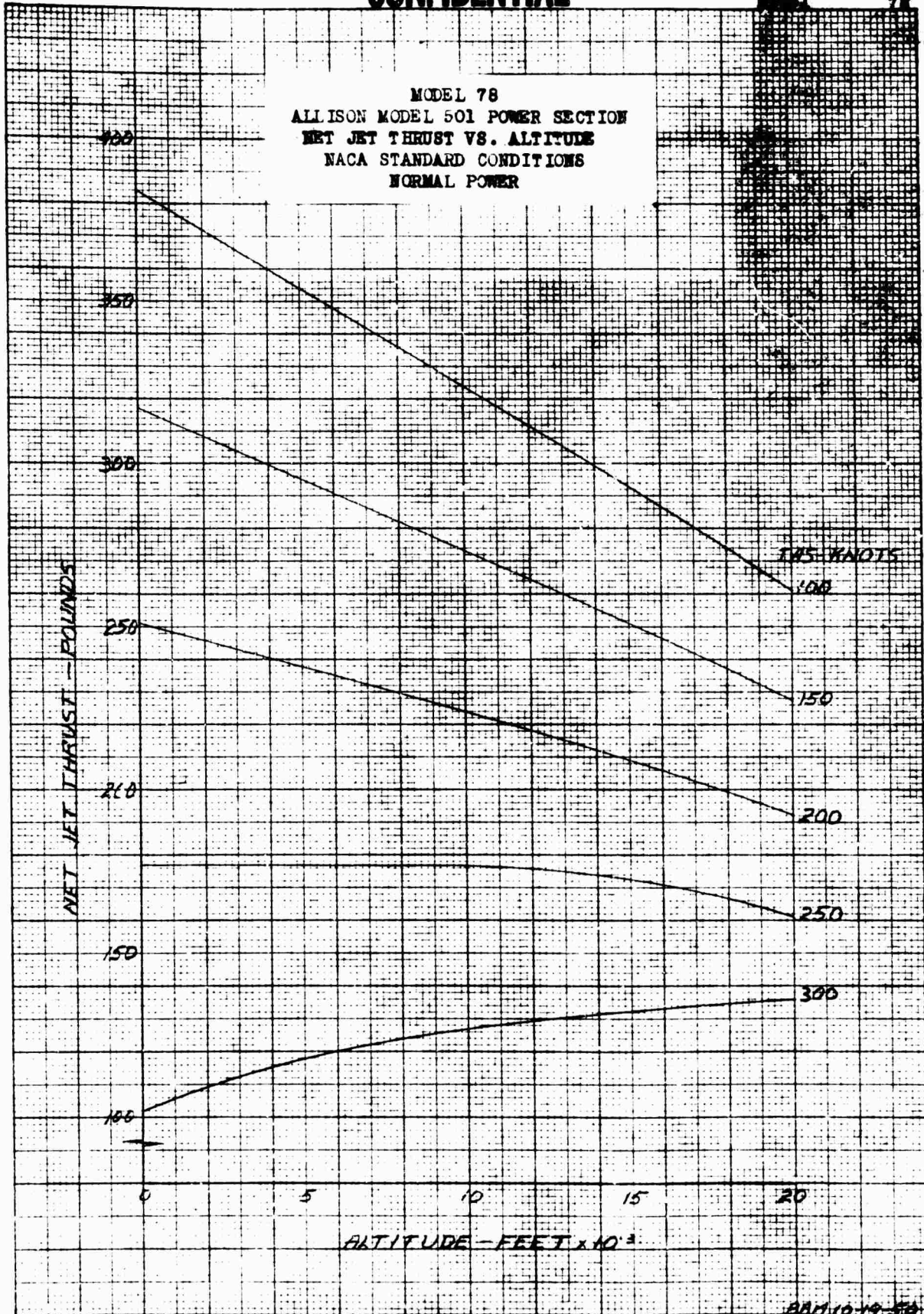
BEM 1049-50

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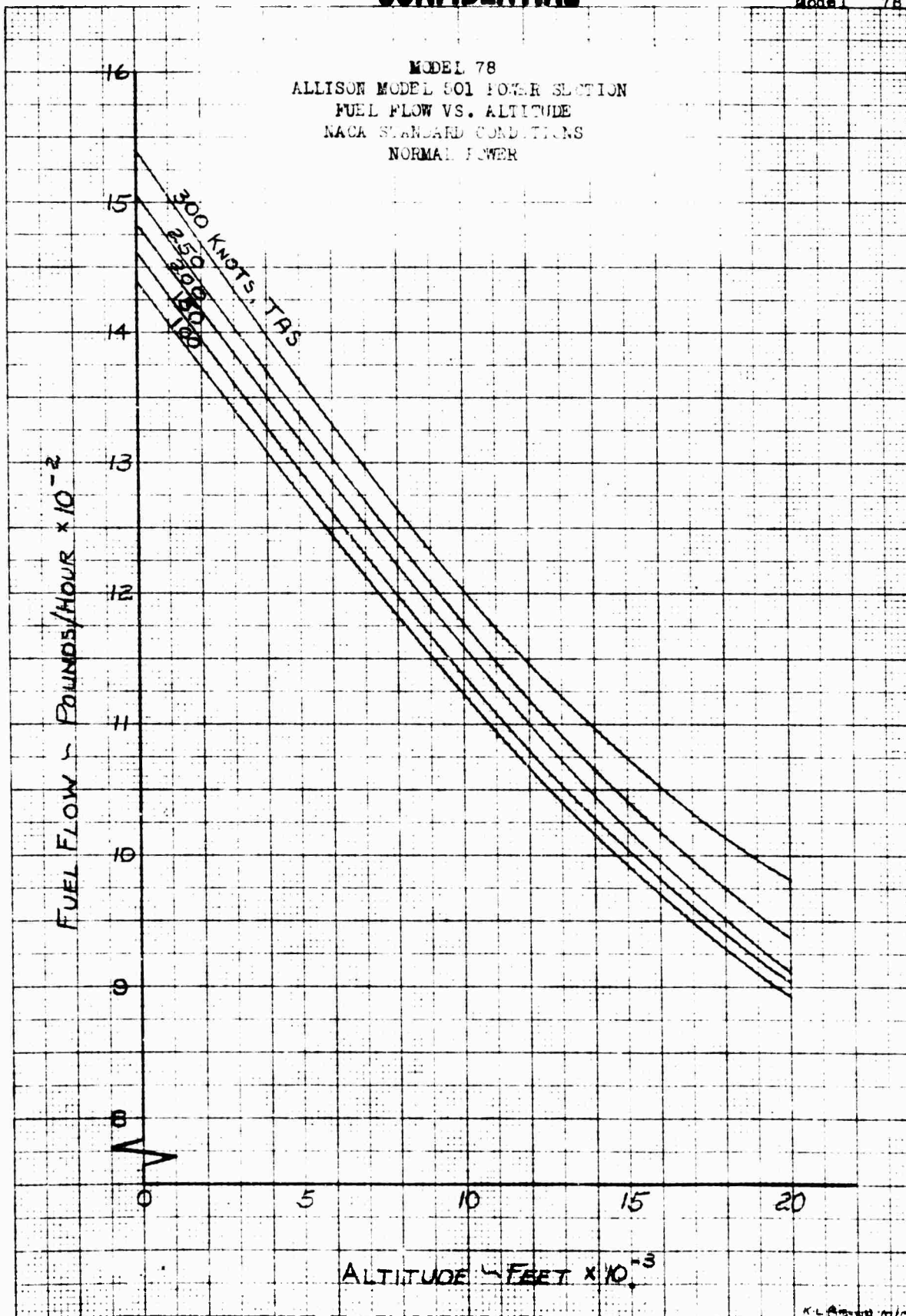


ESSEX CO., N. Y. NO. 388-11
18 X 18 In. 7/8 Inch. Sub. Lines Allowed
MADE IN U.S.A.

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FIG. 15

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NEUFEL & ESSER CO., N. Y. NO. 38941
15-210000-1, 2, 3, 4, 5, 6, 7, 8, 9, 10, 11, 12, 13, 14, 15, 16, 17, 18, 19, 20, 21, 22, 23, 24, 25, 26, 27, 28, 29, 30, 31, 32, 33, 34, 35, 36, 37, 38, 39, 40, 41, 42, 43, 44, 45, 46, 47, 48, 49, 50, 51, 52, 53, 54, 55, 56, 57, 58, 59, 60, 61, 62, 63, 64, 65, 66, 67, 68, 69, 70, 71, 72, 73, 74, 75, 76, 77, 78, 79, 80, 81, 82, 83, 84, 85, 86, 87, 88, 89, 90, 91, 92, 93, 94, 95, 96, 97, 98, 99, 100
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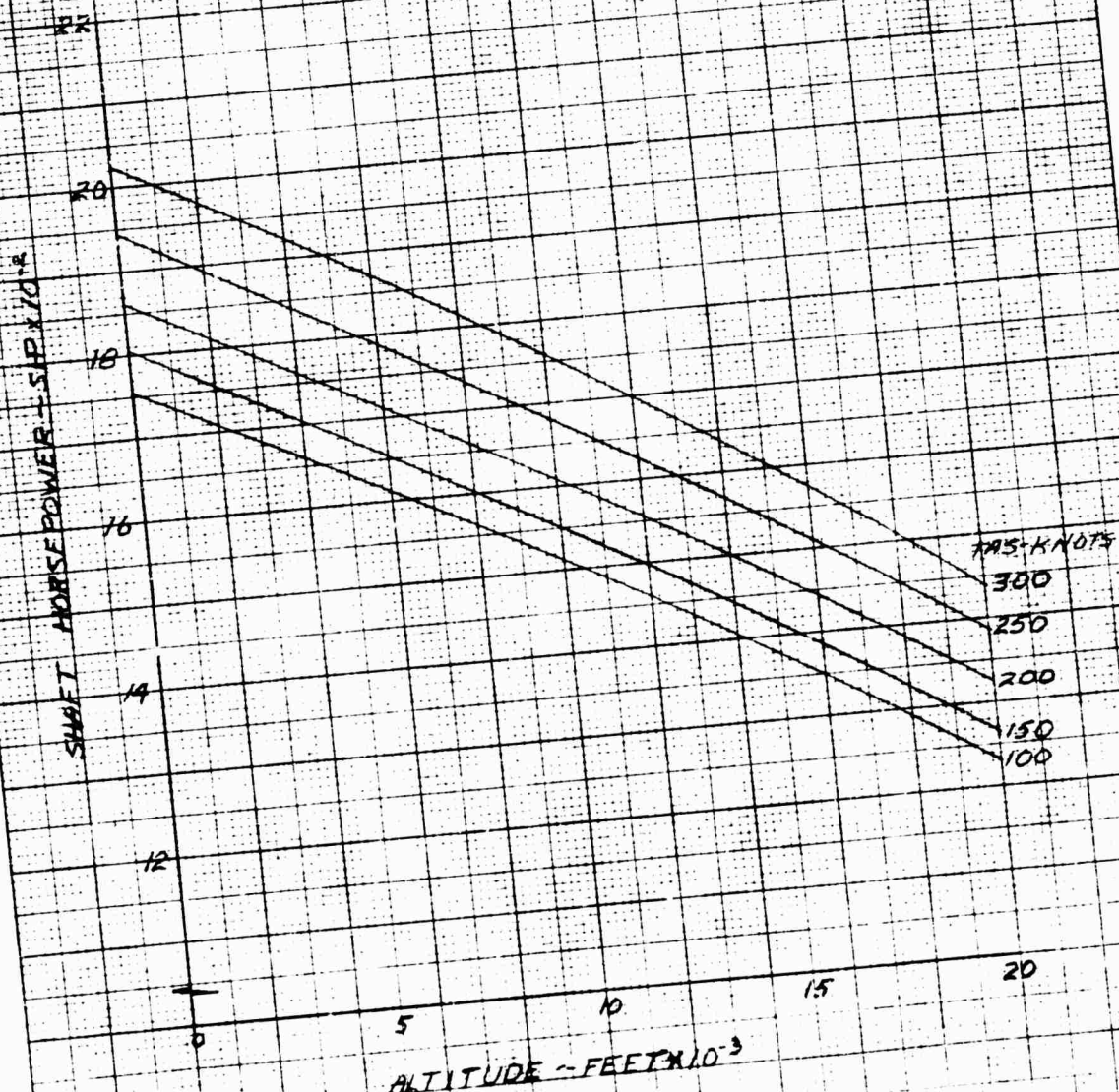
FIG. 16

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MODEL 78
ALLISON MODEL 101 POWER SECTION
SHAFT HORSEPOWER VS. ALTITUDE
DAY, SUMMER DAY
NORMAL POWER



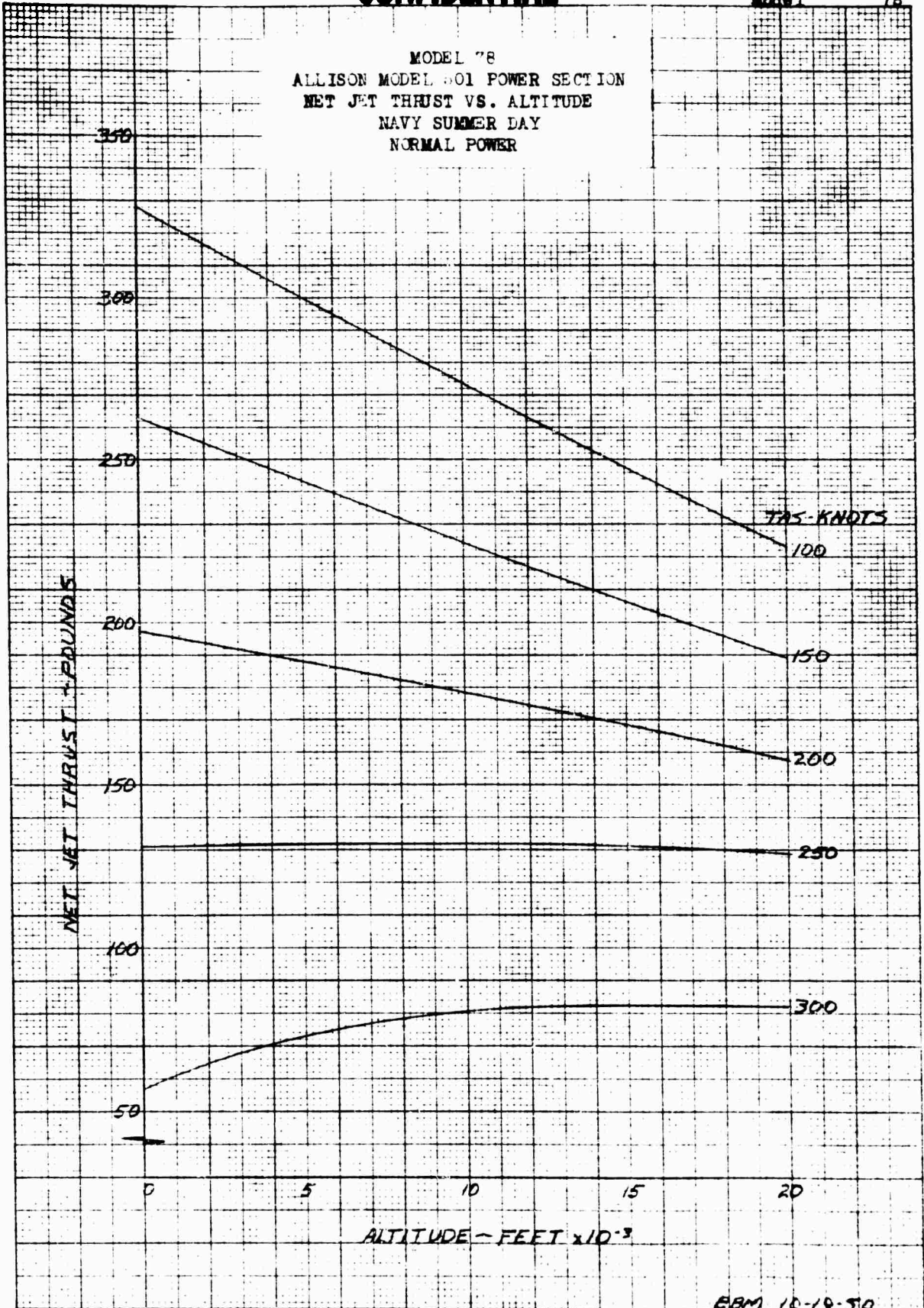
KEUFFEL & ESSER CO., N. Y. NO. 258-11
16 X 18 in. Ink. 5th Edition
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FIG 17.

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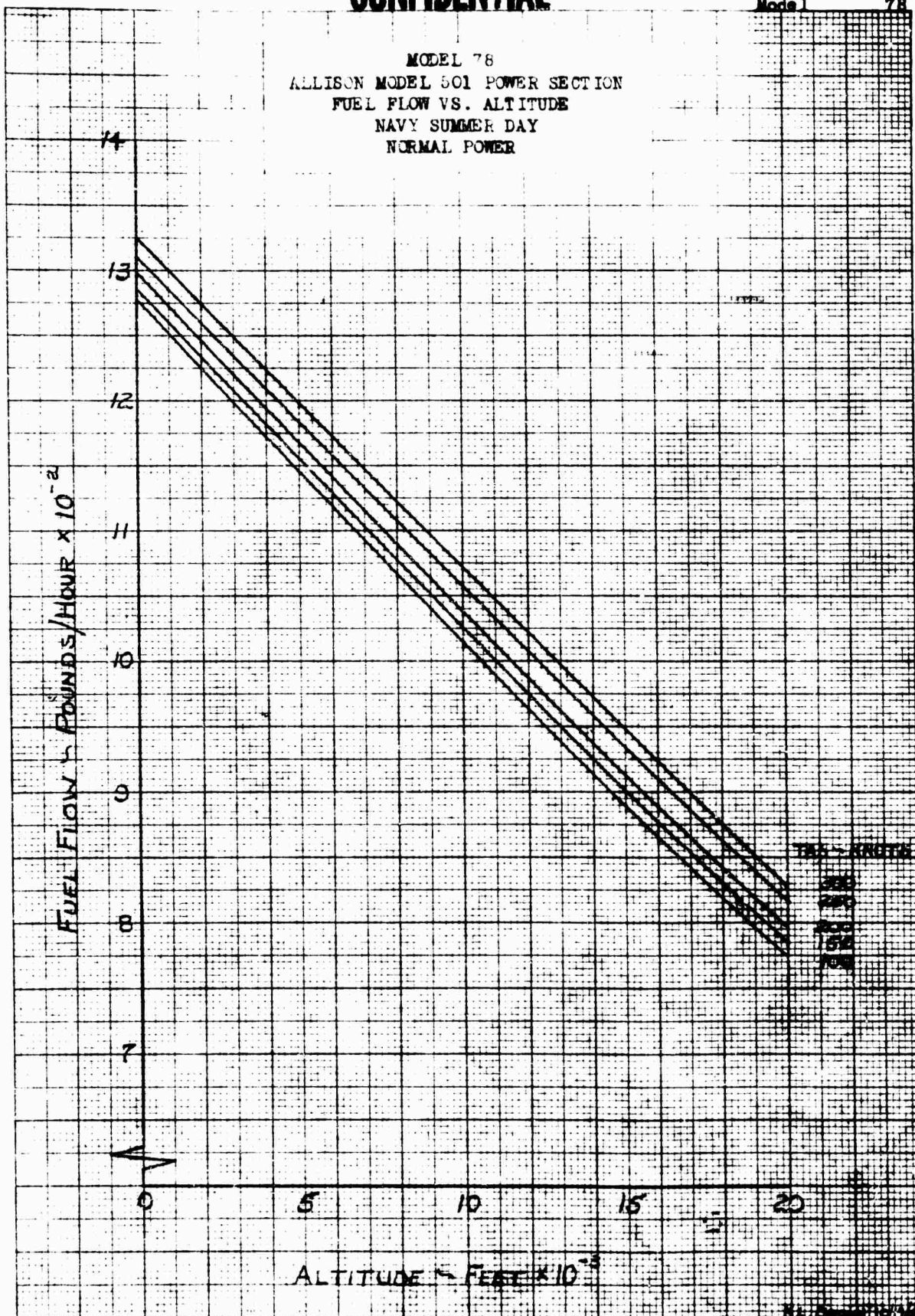
BUFFEL, A. ESCO CO. N. Y. NO. 385-11
10 X 10 to the 1/2 inch 5th lines separated
DATE 12-19-50

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EBM 12-19-50

FIG. 18

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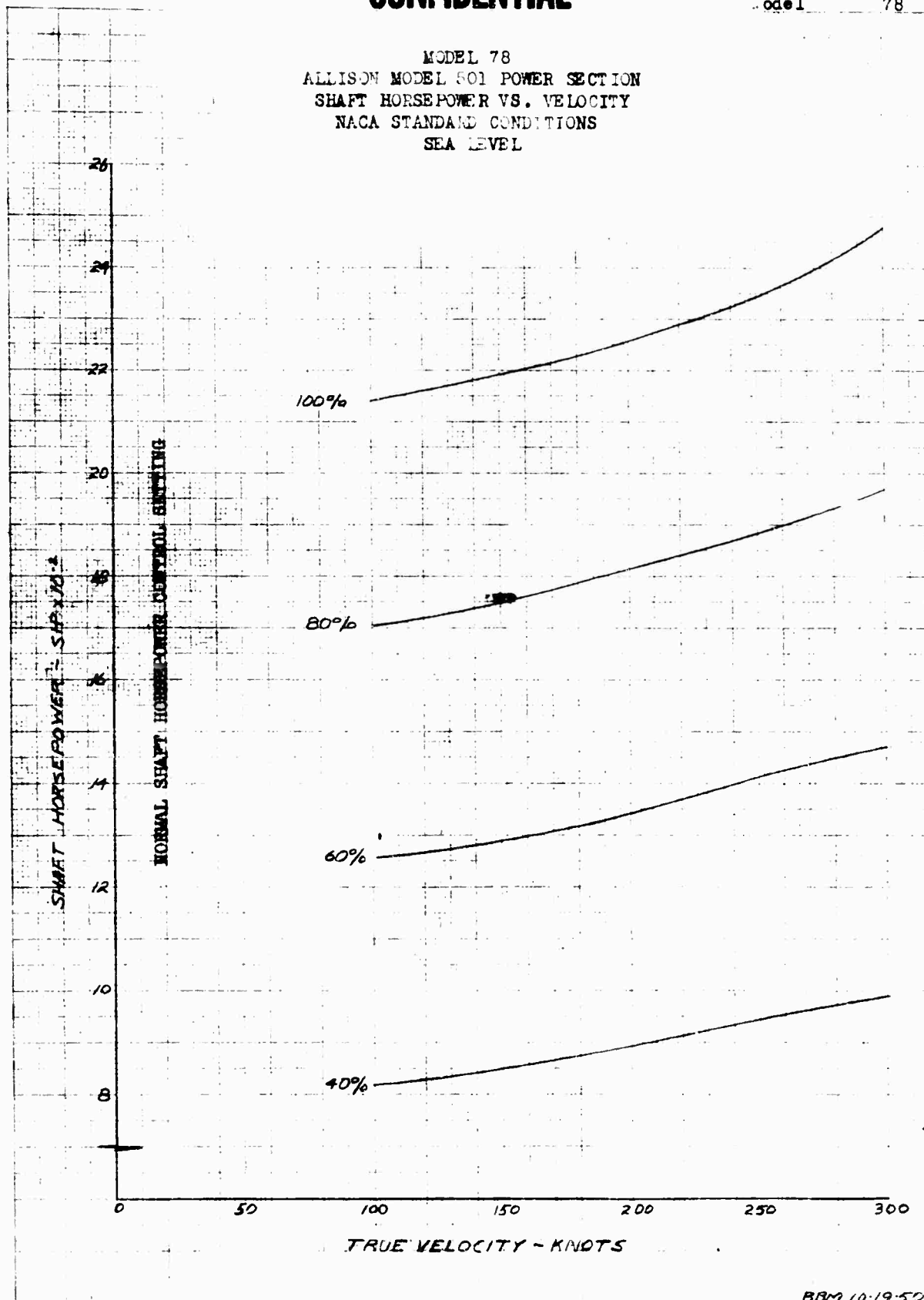
KRUPP & ESSER CO. N. Y. NO 286-12
10 x 10 to the 1/2 inch, 5/8 inch
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FIG 19

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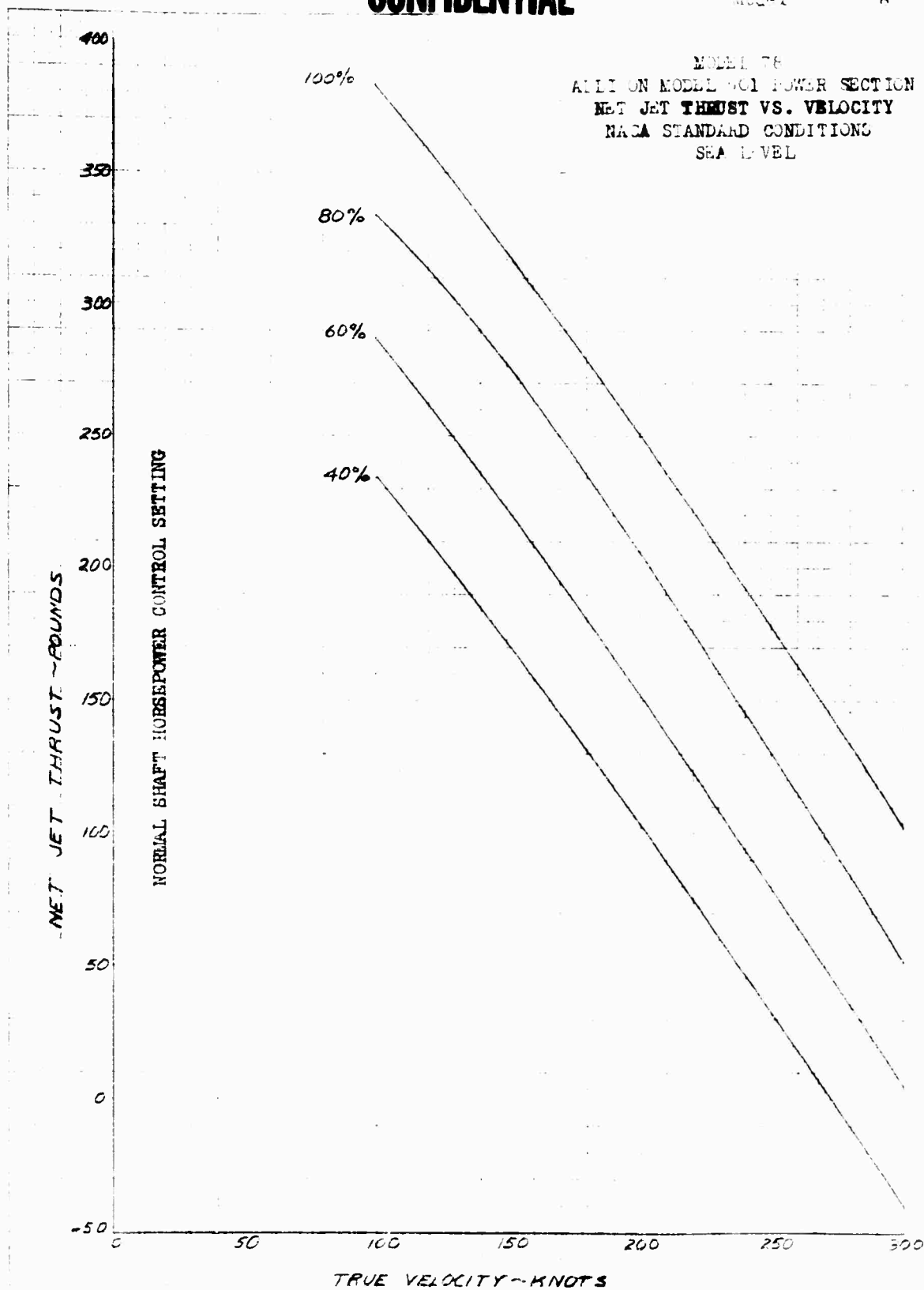
MODEL 78
ALLISON MODEL 501 POWER SECTION
SHAFT HORSEPOWER VS. VELOCITY
NACA STANDARD CONDITIONS
SEA LEVEL



REFUEL & EJECTOR
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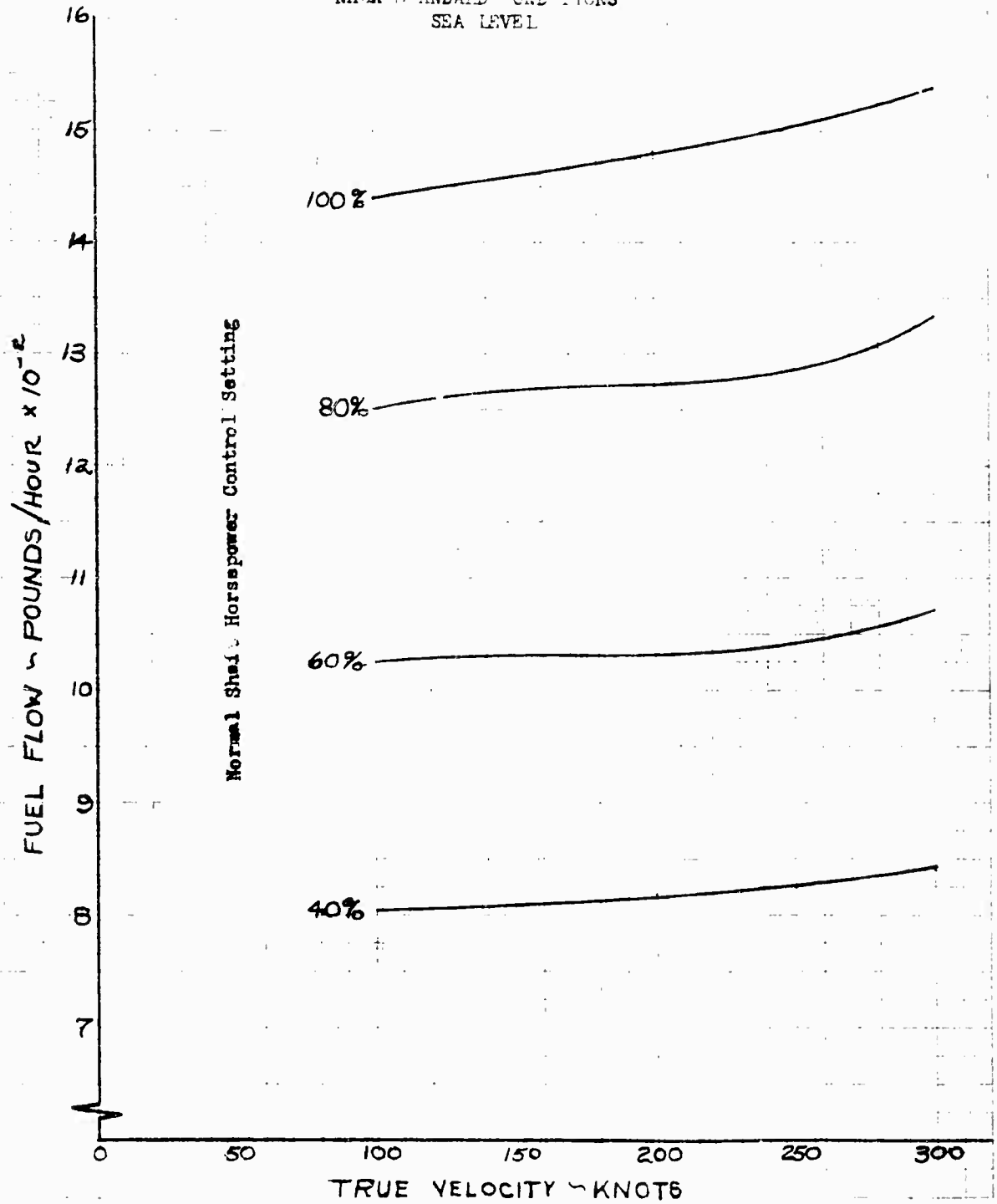
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MODEL 78
ALLISON MODEL 501 POWER SECTION
FUEL FLOW VS. VELOCITY
NACA STANDARD CONDITIONS
SEA LEVEL

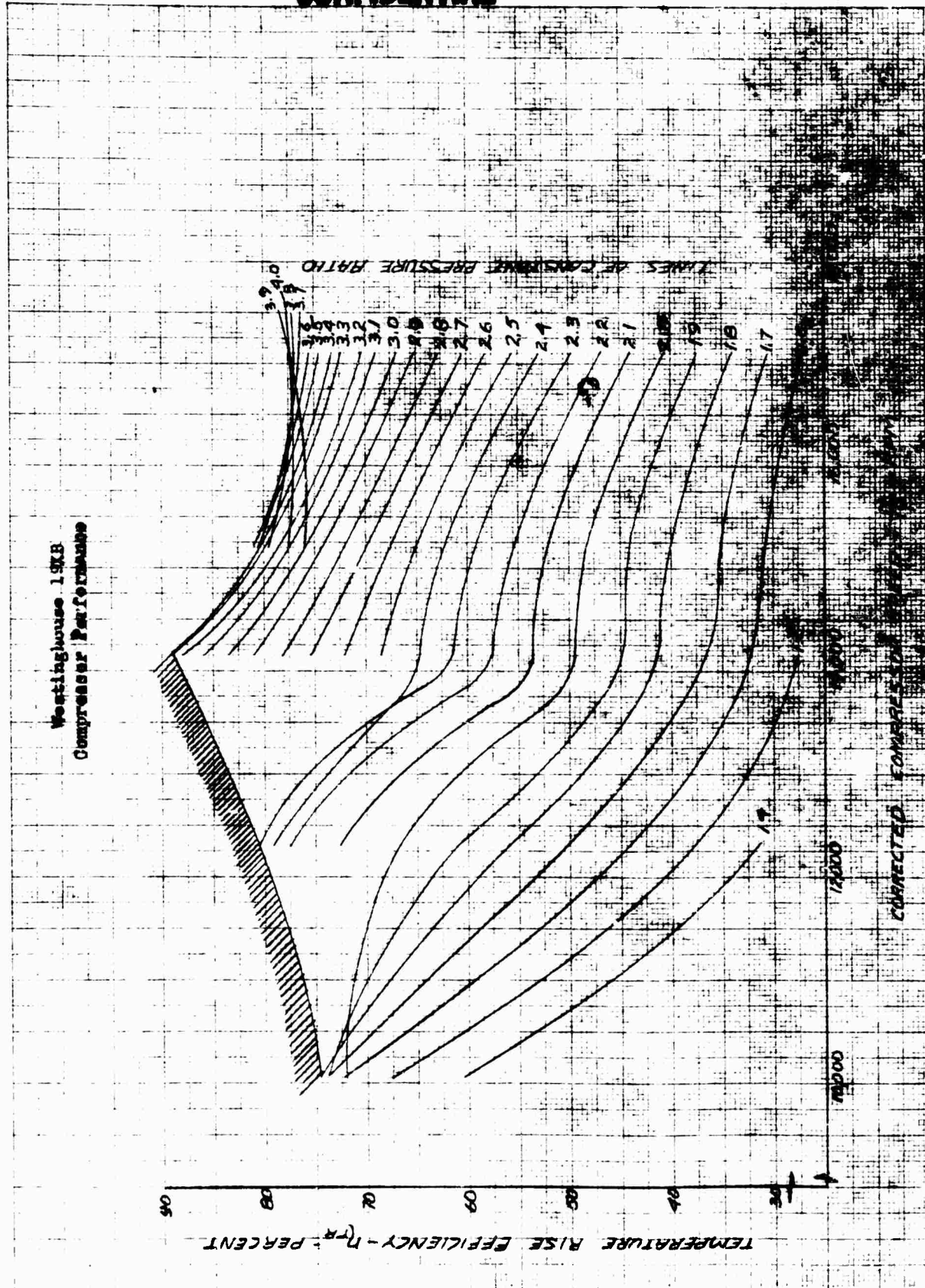


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FIG. 22

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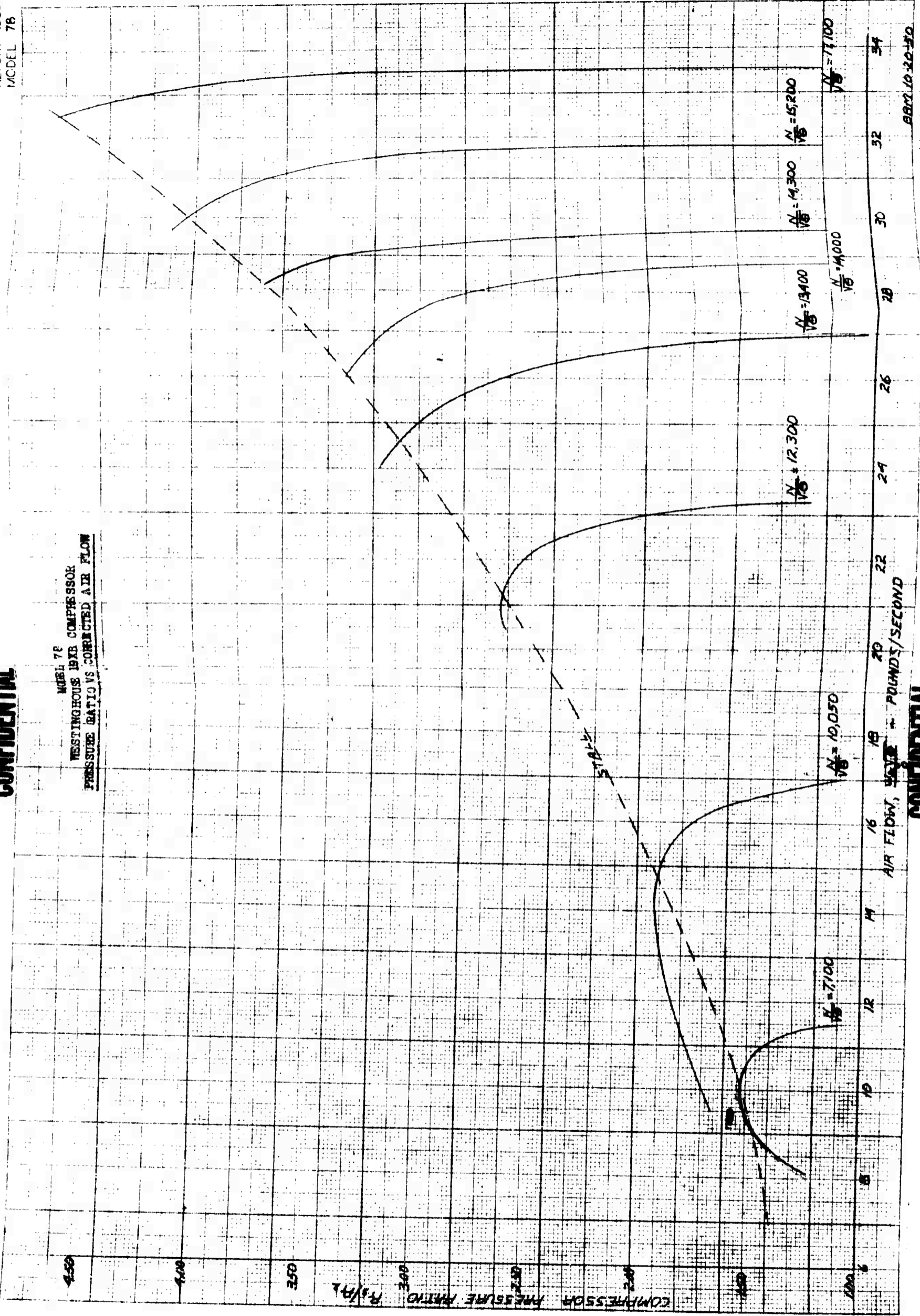
FIG. 23

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MODEL 78

MODEL 78
WESTINGHOUSE 19XB COMPRESSOR
PRESSURE RATIOS CORRECTED AIR FLOW



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FIG 24

DATE 20 DECEMBER 1950

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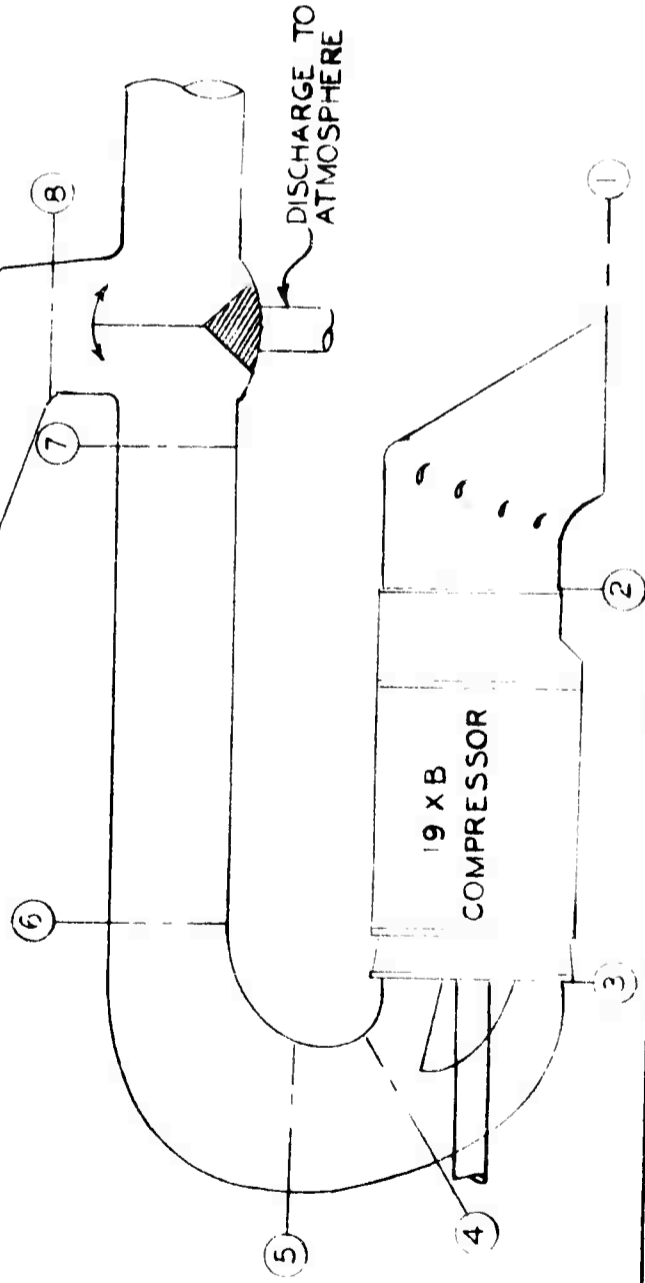
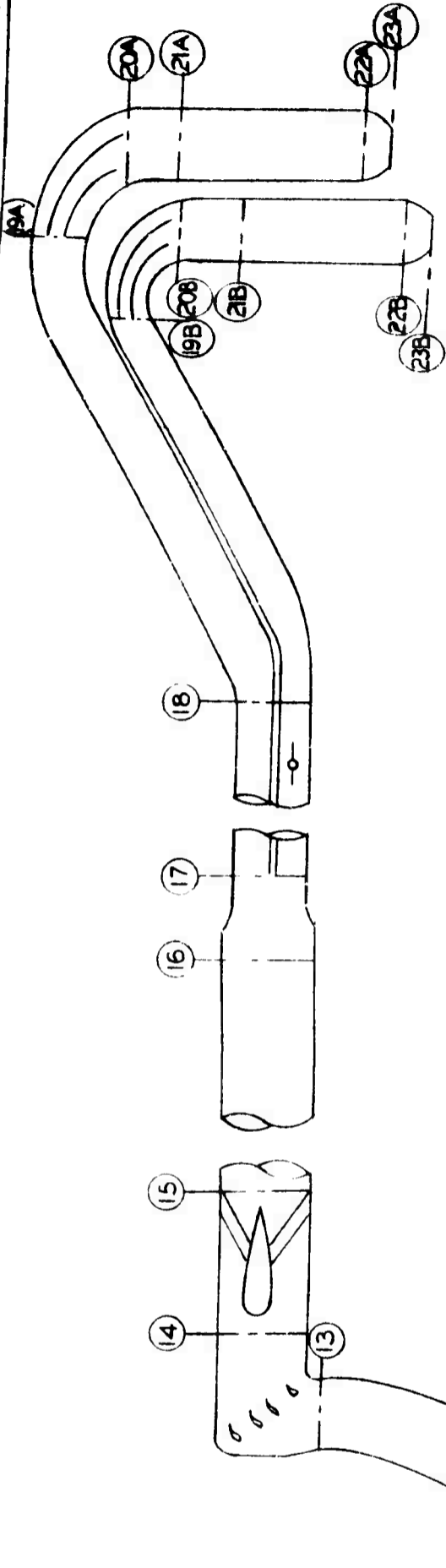
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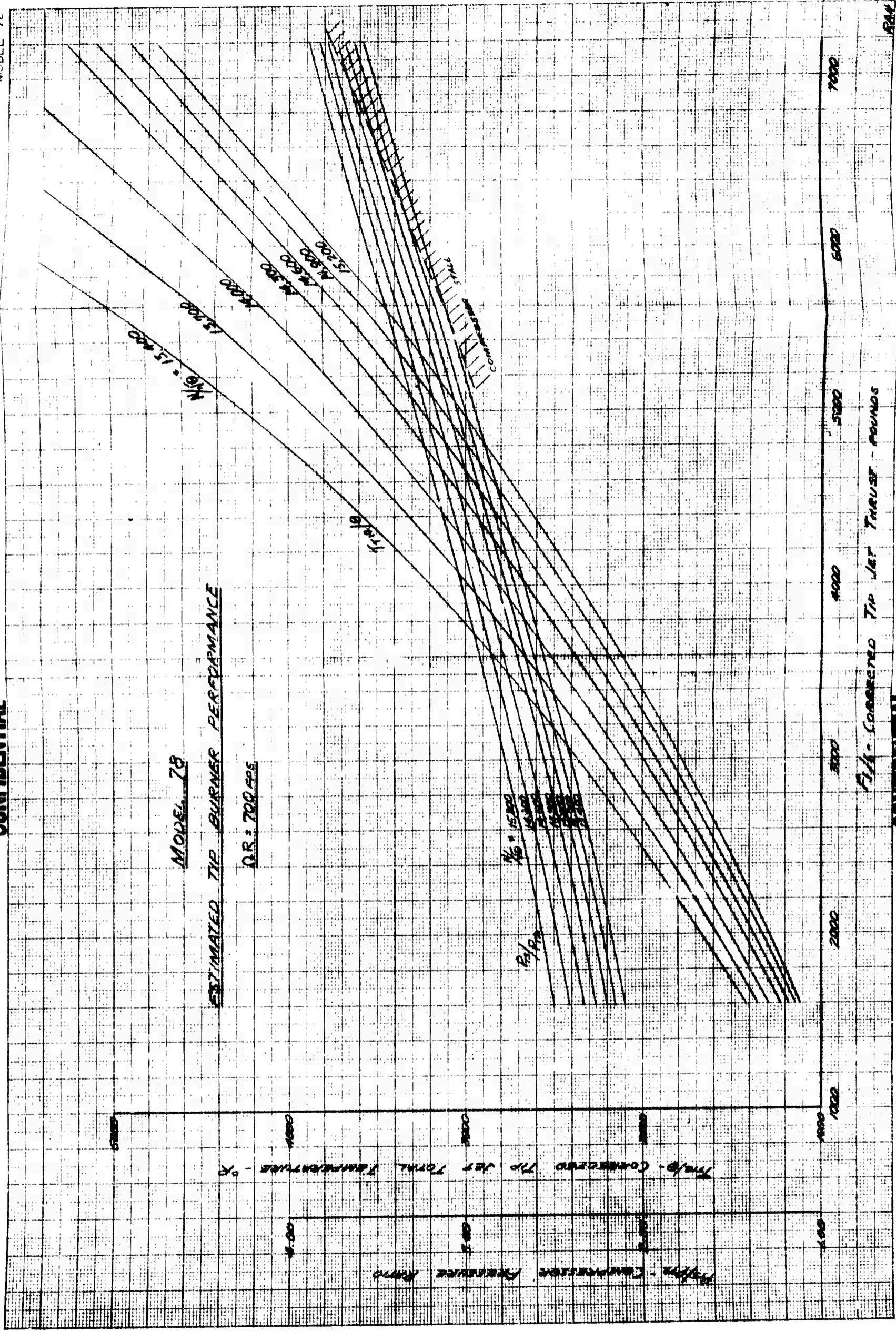
MODEL 78
PRESSURE JET DUCT SCHEMATIC

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MAC 3187 (REV. 6-6-49)

FIG. 25

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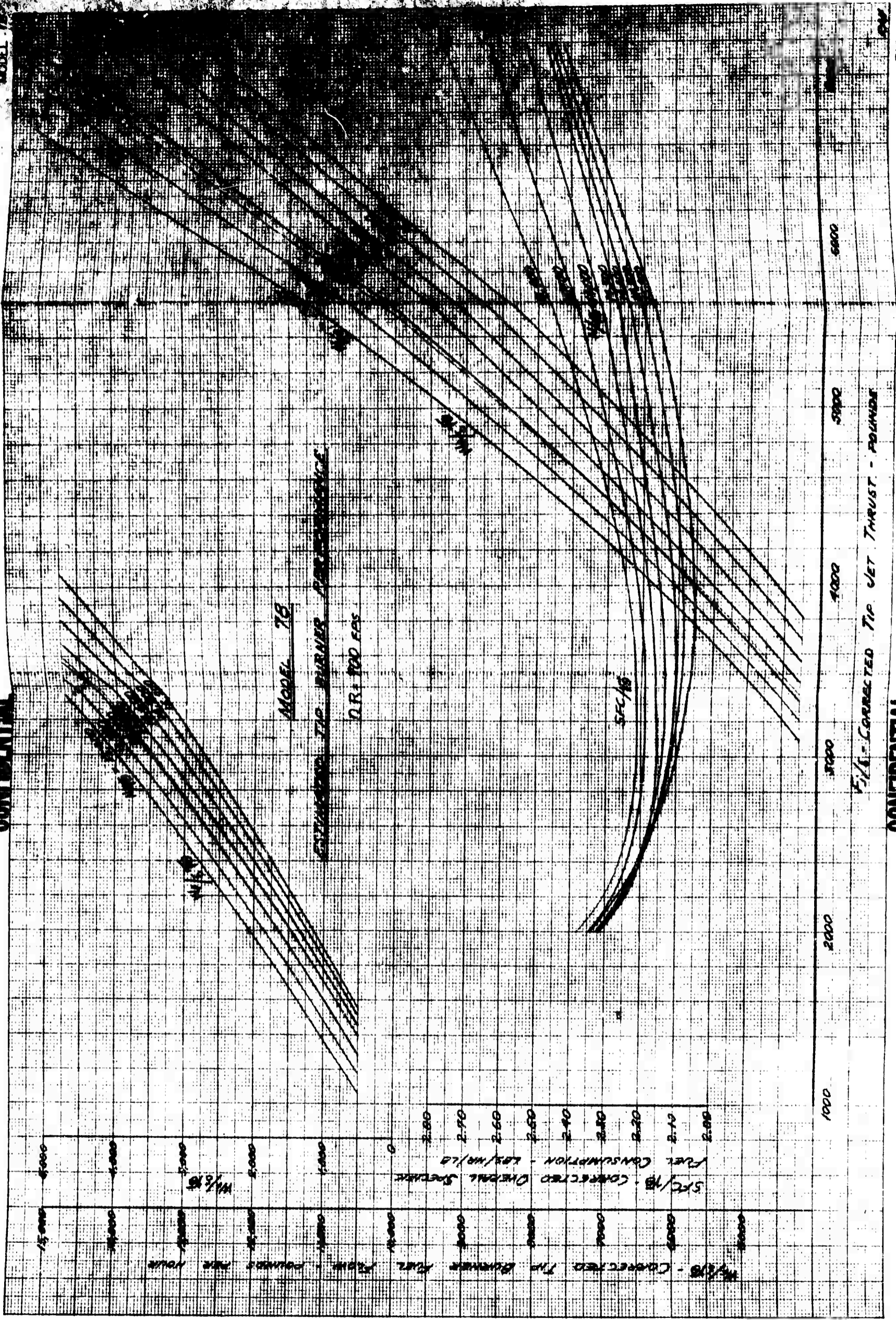
Thrust-Compressor Tip Jet Thrust - POUNDS

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FIG 26

RAUPR & BERN CO., N. Y. NO. 178-141
Millimeters, 5 mm lines spaced on 10mm base
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FIG. 27

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SERIALS SECTION
WASHINGTON, D.C. 20340

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MODEL 78

TIP BURNER PERFORMANCE

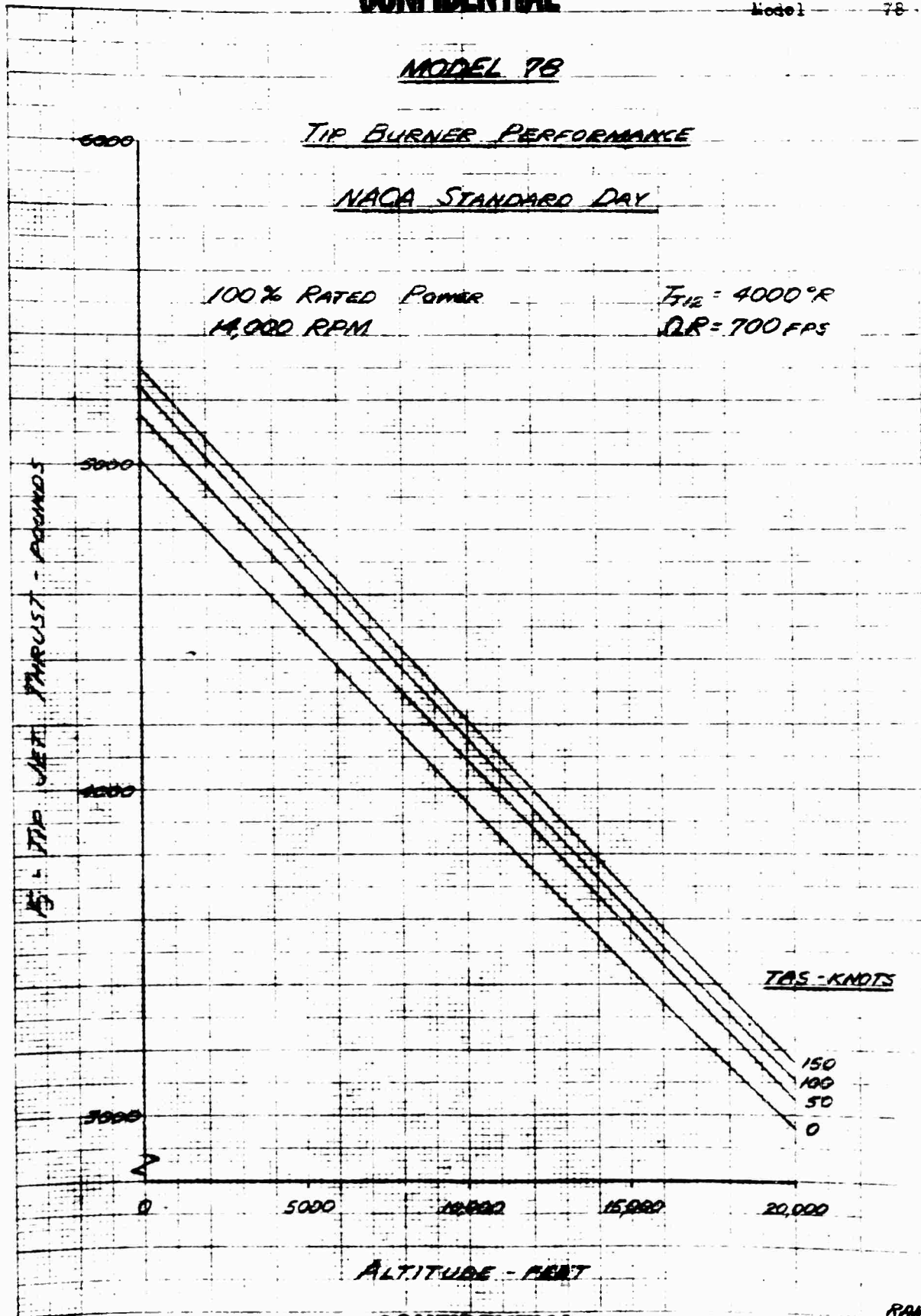
NACA STANDARD DAY

100% RATED POWER
14,000 RPM

$T_{12} = 4000^{\circ}R$
 $Q_R = 700 \text{ FPS}$

KEUFFEL & ESSER CO.

NO. 2011 10 1/2 x 1/2 inch 16 lines centered
Fig. 28 10 1/2 x 1/2 inch



ALTITUDE - FEET

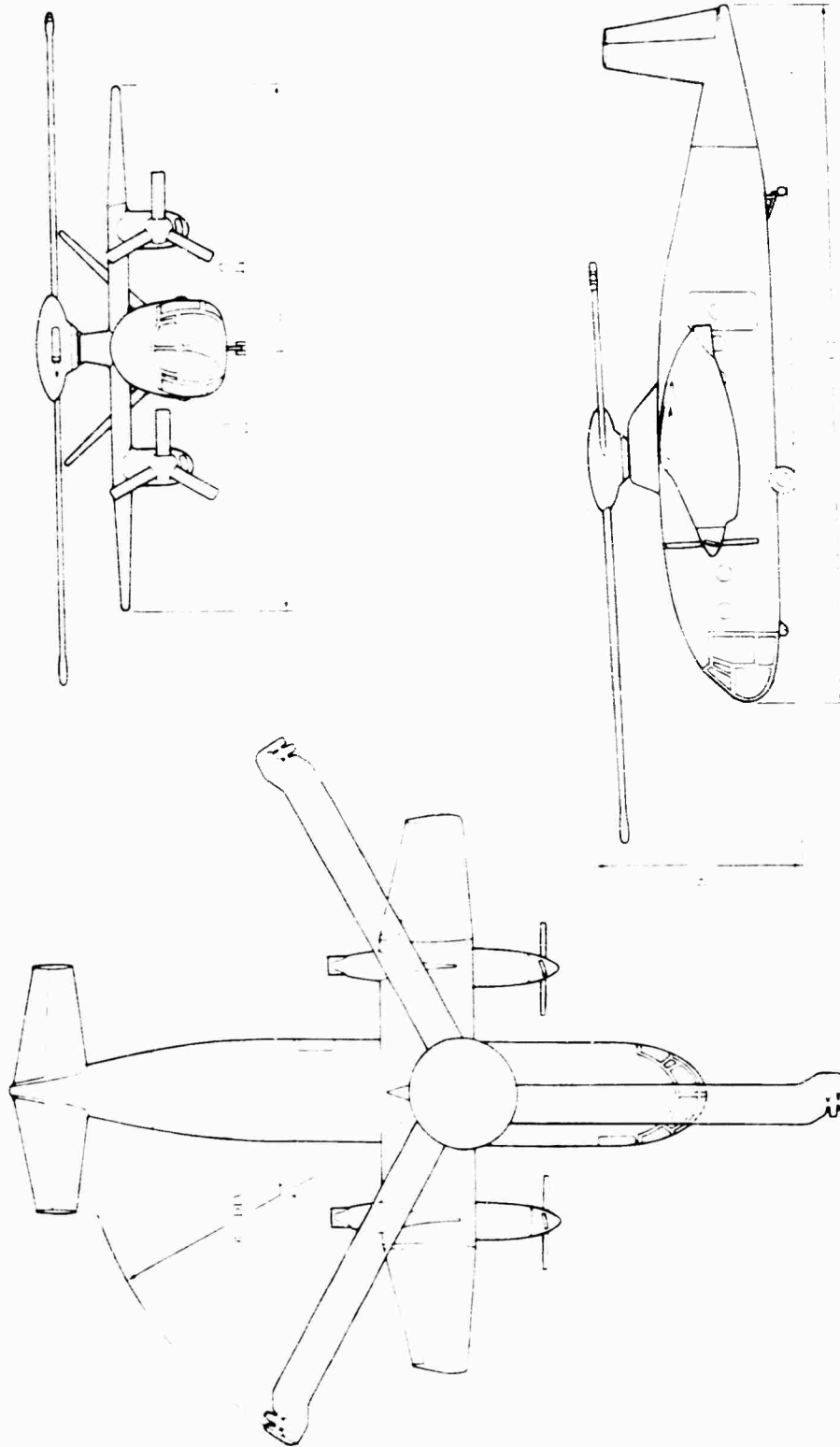
TAS - KNOTS

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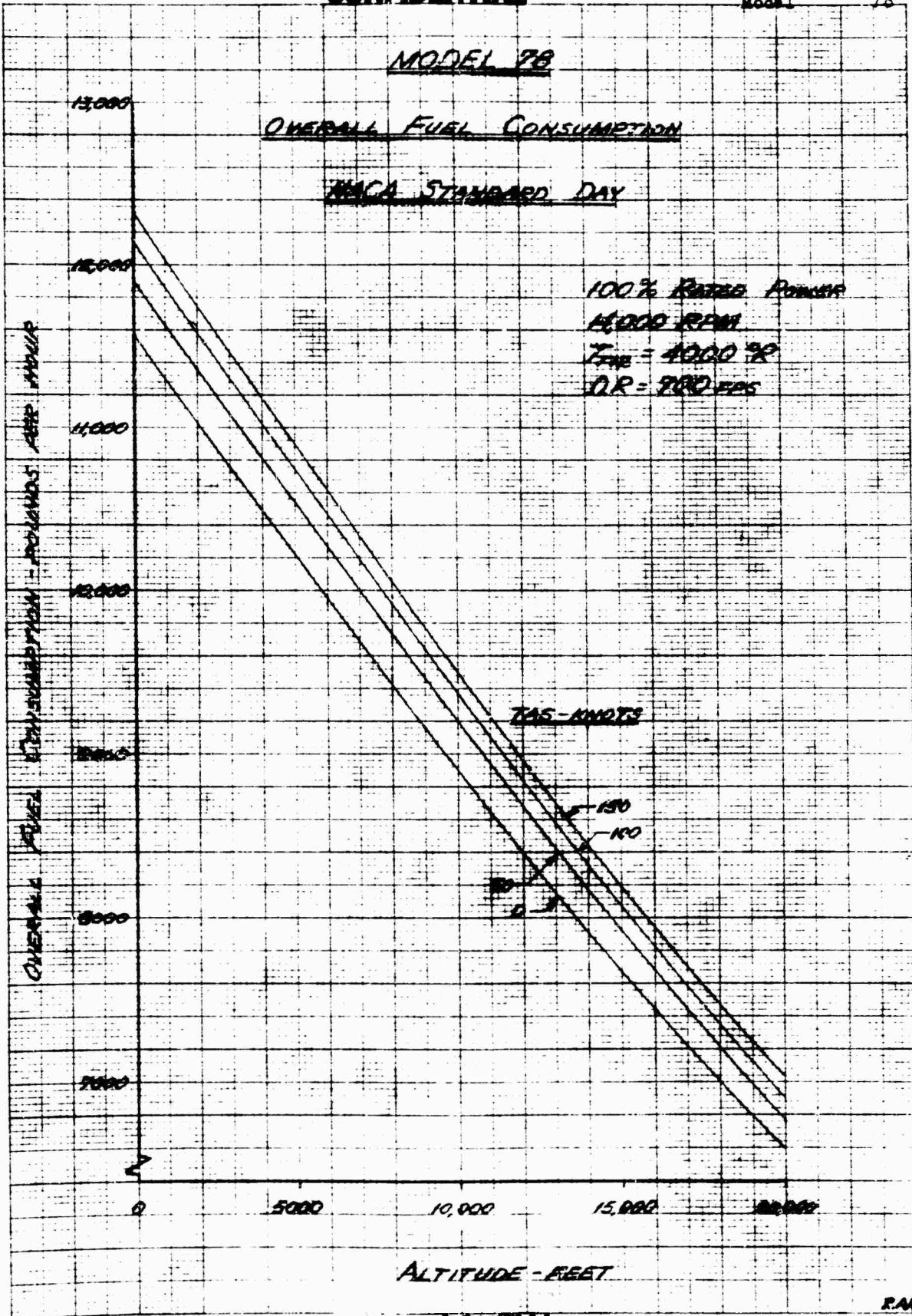
Report 1905
Model 78



MODEL 78 GENERAL ARRANGEMENT

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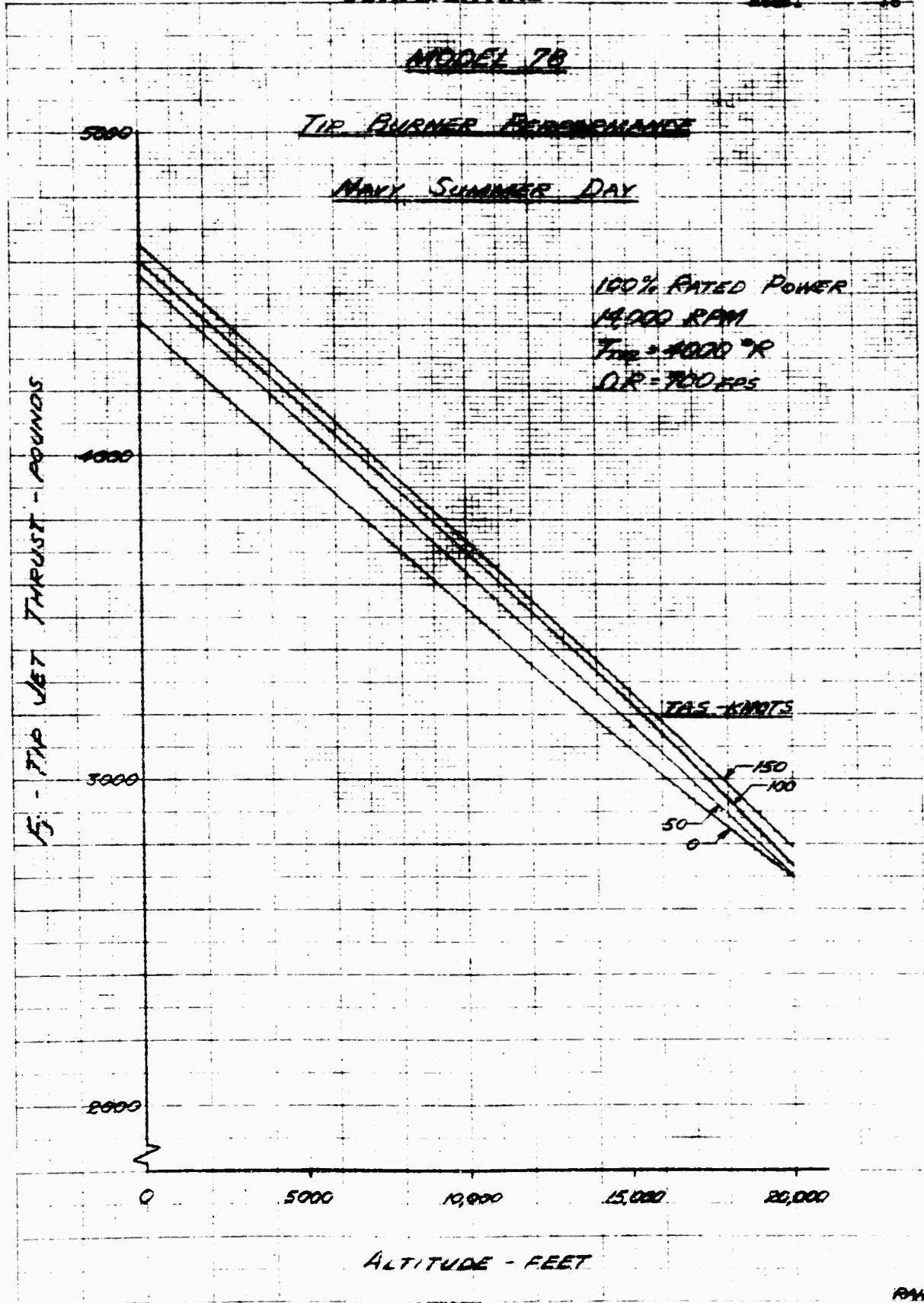
KEUFFEL & ESSER CO

NO. 359 11. 10 x 10 to the left of the horizontal centerline
Engineering. 10 in

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FIG 29

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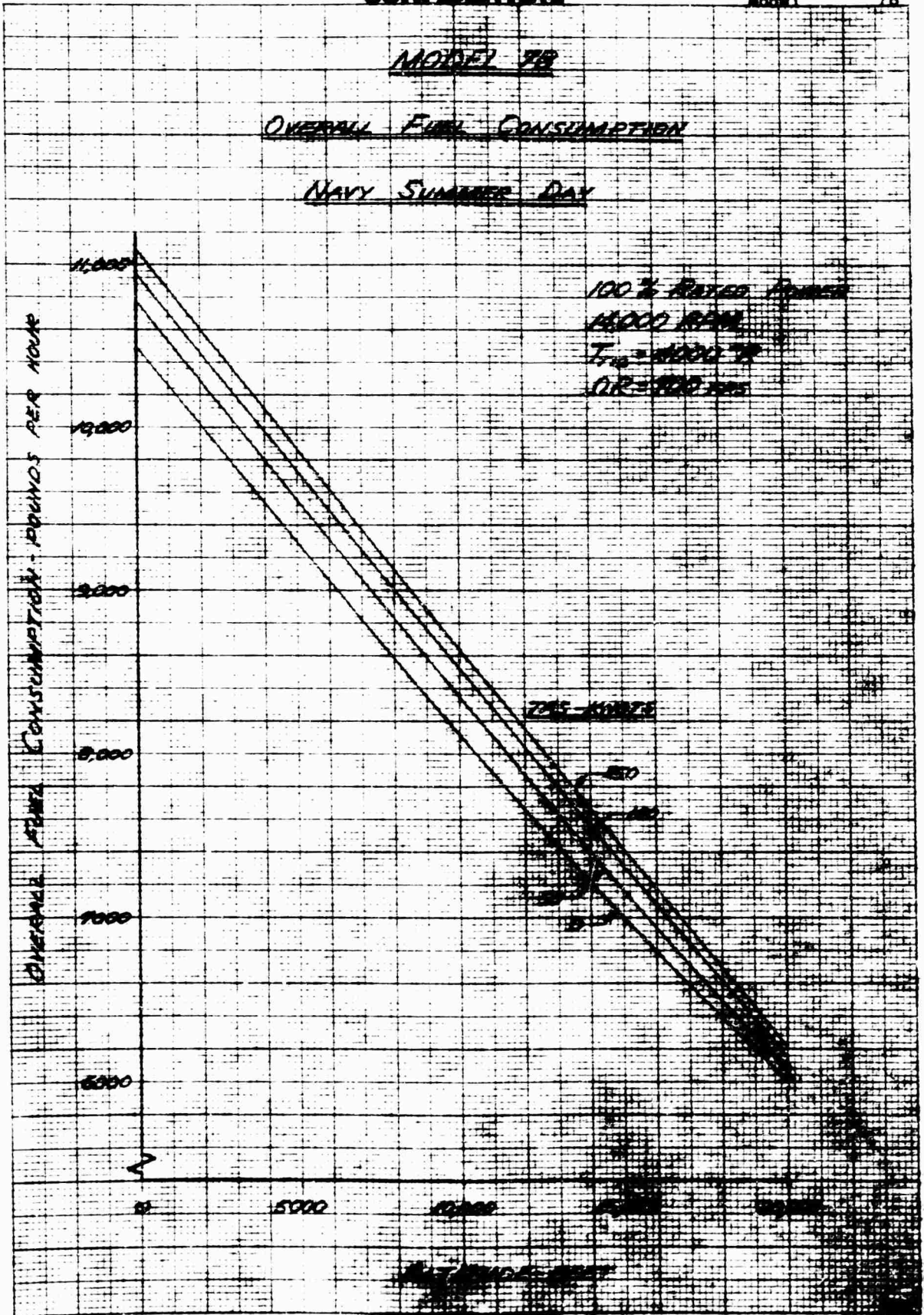


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FIG. 31

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SECTION INLET DUCT ANALYSIS

Station 2

PARAMETER	STATION 1	STATION 2	STATION 3	STATION 4	STATION 5
Total temperature, $^{\circ}R$	1.30	1.30	1.4	1.40	2
Static pressure, l./sq. ft.	1.30	1.30	1.4	1.40	1.864
Density, slug/cu. ft.	0.0021	0.0021	0.0021	0.0021	0.00207
Mass flow of air, lb./sec	0.91	0.91	0.91	0.91	0.91
Volume flow of air, cu. ft./sec	0.67	0.67	0.67	0.67	0.67
Cross sectional area sq. ft.	1.11	1.11	1.23	1.23	1.11
Velocity, ft./sec	37	37	37	37	37
$\frac{1}{2} \rho V^2$ l./sq. ft.	1.7	1.7	1.7	1.7	1.7
Velocity of sound, ft./sec	1110	1110	1110	1110	1110
Mac. Number	0.33	0.33	0.33	0.33	0.33
Compressibility factor	1.03	1.03	1.02	1.02	1.034
Impact pressure, l./sq. ft.	1.9	1.9	1.9	1.9	1.70
Total pressure, l./sq. ft.	2044	2028	2020	2014	1910
Change in total pressure, l./sq. ft.	-11	-11	-6.1	-6	-6
Change in static pressure, l./sq. ft.	-10	-10	30	-7	-110
Change in total temperature, $^{\circ}R$	0	0	0	0	0
Pressure-loss coefficient	0.1	0.1	0.0	0.0	0.08
Velocity parameter	0.31	0.31	0.31	0.31	0.31
Static temperature, $^{\circ}R$	0.07	0.07	0.10	0.10	0.06
Air flow rate, lb./sec	29.64	29.64	29.64	29.64	29.64
Velocity, ft./sec	37	37	37	37	37
Temperature ratio	1.021	1.021	1.016	1.016	1.024
Pressure ratio P/P_0	1.07	1.07	1.07	1.07	1.087

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MODEL 78

ALLISON MODEL 501 ENGINE SECTION INLET DUCT ANALYSIS

TABLE 3

Conditions: $V_{\infty} = 87$ Knots, Sea Level, NACA Standard Day, Normal Power

Quantity	STATION					
	0	1	1.1	1.30	1.45	2
Total temperature, $^{\circ}R$	200.2	200.2	200.2	200.2	200.2	200.2
Static pressure, $l./sq. in.$	211	198.0	190.1	188.3	191.9	184.4
Density, $lb./cu. ft.$.002306	.0021	.00219	.00218	.00219	.00212
Mass flow of air, $lb./sec.$.014	.014	.019	.019	.019	.019
Volume flow of air, $cu. ft./sec.$.06	.06	.09	.09	.09	.09
Cross-sectional area, $sq. in.$	1.11	1.11	1.11	1.11	1.29	1.11
Velocity, $ft./sec.$	369	378	383	383	320	390
$\frac{1}{2} \rho V^2$, $l./sq. in.$	20.9	1.7	1.7	1.6	1.16	1.61
Velocity of sound, $ft./sec.$	1120	1110	1110	1108	1112	1108
Mach number	.331	.340	.340	.340	.292	.342
Compressibility factor	1.000	1.000	1.003	1.001	1.022	1.032
Impact pressure, $l./sq. in.$	1.8	1.1	1.5	1.5	1.19	1.66
Total pressure, $l./sq. in.$	214.0	214.0	205.2	204.6	203.6	201.0

Change in total pressure, $l./sq. in.$

Change in static pressure, $l./sq. in.$

Change in total temperature, $^{\circ}R$

Pressure-loss coefficient

Velocity, per meter

Static temperature, $^{\circ}C$

Air flow rate, $lb./hr.$

Velocity, $ft./sec.$

Temperature ratio

Pressure ratio

ΔH	0	-3	-16	-6	-6	-22
ΔP	1.04	-82	-16	38	-6	-69
ΔT_t	0	0	0	0	0	0
$\Delta H/q_c$.	.1	.30	.00	.00	.180
W_{∞}/W_1	.100	.100	.100	.100	.101	.102
T	100	99	100	112	11	108
WA	29.0	29.0	29.0	29.0	29.0	29.0
V	310	310	320	320	320	320
T_t/T	1.02	1.002	1.024	1.016	1.017	1.024
P_t/P	1.072	1.079	1.067	1.077	1.070	1.065

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McDONNELL MODEL 78 INLET SECTION INLET DUCT ANALYSIS

TABLE 2

Conditions: V = 240 KNOTS, SEA LEVEL, MACH STANDARD DA, Normal Lower

Quantity	STATION	
	0	1
Total temperature, °R	1.10	1.30
Static pressure, lb./sq. in. abs.	1.8	1.8
Density, slugs/cu. ft.	0.00233	0.00229
Mass flow of air, slugs/sec.	40	385
Volume flow of air, cu. ft./sec.	40	426
Cross sectional area, sq. ft.	1.11	1.11
Velocity, ft./sec.	360	326
$\frac{1}{2} \rho V^2$ lb./sq. ft.	194	124
Velocity of sound, ft./sec.	1120	1120
Mach number	0.320	0.344
Compressibility factor	1.023	1.031
Impact pressure, lb./sq. ft.	190	127
Total pressure, lb./sq. ft.	231	220
Change in total pressure, lb./sq. ft.	-63	-88
Change in static pressure, lb./sq. ft.	-90	-39
Change in total temperature °p	0	0
Pressure-loss coefficient	0.05	0.05
Velocity parameter	0.654	0.590
Static temperature, °R	518.4	522.6
Air flow rate, lb./sec.	31.0	31.5
Velocity, ft./sec.	370	328
Temperature ratio	1.021	1.010
Pressure ratio	1.094	1.061

TABLE 4
SUMMARY OF ALLISON MODEL 501 POWER SECTION INLET DUCT ANALYSIS
Sea Level, Normal Power

Condition	Static	$V_0 = 87$ Knots	$V_0 = 240$ Knots
Velocity ratio V_1/V_0	∞	2.11	0.903
Angle of Attack α	2.0	0	0
Estimated H/q_1	1.11	0.816	1.54

Estimate of losses through the duct

Condition	Static	$V_0 = 87$ Knots						$V_0 = 240$ Knots						
Section	0-1	1-1.10	1.3-1.45	1.45-1.6	1.6-2	0-1	1-1.15	1.15-1.3	1.3-1.45	1.45-1.6	1.6-2	1.3-1.45	1.45-1.6	1.6-2
$\Delta H/q_{local}$.4800	.1040	.0522	.0520	.0518	.0516	.1025	.0506	.0518	.190	.0510	.0518	.0517
q_{local}/q_c	∞	.9150	.9570	.7010	.7070	.982	.95	.968	.982	.721	1.05	.909	.919	.675
$\Delta H/q_c$.4390	.0494	.0356	.3900	.0497	.4920	.0994	.0497	.0373	.1370	.448	.0476	.0346
Total H_{ot}/q_c		1.0126											1.3791	

Overall pressure loss and effect upon engine performance

Condition	Static	$V_0 = 87$ Knots	$V_0 = 240$ Knots
$\Delta H_{ot}/q_c$	1.0126	.8101	1.3791
$\Delta H_{ot}/H_{ot}$.0784	.0611	.11
$\Delta SHP/SEHP$.126	.0662	.0602
$\Delta F/F$.122	.0791	.0769
$\Delta W_F/W_F$.0644	.0337	.0272

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TABLE 5

WESTINGHOUSE 19XP COMPRESSOR - LITTON AIRCRAFT

Condition: Static, Sea Level, ISA Standard Day, 15,000 ft.

Quantity	Symbol	0	1	1.1	2
1 Total temperature, °	T_T	518.4	518.4	518.4	518.4
2 Static pressure, lbs./sq.ft. abs.	P	2116	1780	1780	1780
3 Density, slug/cu.ft.	ρ	.001875	.001875	.001875	.00207
4 Mass flow of air, slug/sec.	m	.714	.714	.714	.714
5 Volume flow of air, cu.ft./sec.	Q		330	330	330
6 Cross-sectional area, sq.ft.	A	.938	.938	.938	.938
7 Velocity, ft. sec.	V		330	330	330
8 ρV^2 , lbs./sq.ft.	q		130	143	130
9 Velocity of sound, ft. sec.	a	1116	1116	1116	1116
10 Mach number	M		.314	.314	.314
11 Compressibility factor	P_c		1.027	1.0	1.031
12 Impact pressure, lbs./sq.ft.	$q P_c$		134.5	143	134.5
13 Total pressure, lbs./sq.ft.	P ₀	2116	2116	2044	1980
14 Change in total pressure, lbs./sq.ft.	ΔP_0			-72	-136
15 Change in static pressure, lbs./sq.ft.	ΔP			-38	-136
16 Change in total temperature, °F	ΔT_T				
17 Pressure-loss coefficient	$\Delta h/q$				
18 Velocity parameter	$R_{u/10^4}$.647	.647	.647	.647
19 Static temperature, °F	T	518.4	507	507	518.4
20 Airflow rate, lb./sec.	W	24.5	24.5	24.5	24.5
21 Velocity, ft. sec.	V		330	330	330
22 Temperature ratio	T/T_0		1.017	1.031	1.014
23 Pressure ratio	P/P_0		1.0	1.0	1.0

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		TABLE 7 - ANALYSIS OF PRESSURE JET SYSTEM						
		$N/\sqrt{e} = 14,000$					$T_3/P_2 = 2.40$	
Station	Unit	0	3	4	5	6	7	8
W_a	#/sec		28.45	28.45	28.45	28.45	28.45	56.90
T_T	°R	518.4	740	740	740	740	740	740
P_T	PSIA	14.7	35.28	35.18	34.96	34.77	34.59	34.37
A	in ²		187	100	100	100	100	200
$RW_a/P_T A$.230	.431	.434	.436	.438	.441
V	ft/sec		170	328	331	333	334	337
T	°R		737.62	731.13	730.97	730.86	730.81	730.75
γ			1.3942	1.3944	1.3944	1.3944	1.3944	1.3945
$\gamma/\gamma - 1$			3.5368	3.5355	3.5355	3.5355	3.5355	3.5349
T_T/T			1.00323	1.01213	1.01235	1.01251	1.01258	1.01266
P_T/P			1.01147	1.04360	1.04440	1.04487	1.04512	1.04540
P	PSIA		34.88	33.71	33.47	33.28	33.10	32.88
q_0	PSI		0.40	1.47	1.49	1.49	1.49	1.49
f	ft		2.95*	2.95	2.95	2.95	2.95	4.767
W_a/EP			.300	.300	.300	.300	.300	.371
RN			2.45×10^6	2.45×10^6	2.45×10^6	2.45×10^6	2.45×10^6	3.1×10^6
f					.0031			
$1/D$					10.15			
$\Delta H/q$.25	.15	.128	.12	.12	.05
ΔH			.10	.22	.19	.13	.22	.07
V_{Tx}								
V_{Ty}								
$V_{Ty} - V_{Tx}$								
P_{Ty}/P_{Tx}								
ΔH								

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TABLE 7 Continued

Qty.	Station Unit	9	10	11	12	13	14	15
W_a	#/sec	56.90	56.90	56.90	56.90	18.97	18.97	18.97
T_T	$^{\circ}R$	740	740	740	740	740	740	740
P_T	PSIA	34.30	34.15	33.87	33.83	33.80	33.53	33.23
A	in ²	200	209	410	251	78.4	44.2	44.2
$RW_a/P_T A$.442	.425	.218	.357	.382	.682	.688
V	ft/sec	338	323	162	268	288	551	557
T	$^{\circ}R$	730.58	731.40	737.84	734.08	733.17	715.96	714.41
γ		1.3945	1.3944	1.3941	1.3943	1.3944	1.3950	1.3950
$\gamma/\gamma - 1$		3.5349	3.5355	3.5374	3.5361	3.5355	3.5316	3.5316
T_T/T		1.01289	1.01176	1.00293	1.00806	1.00932	1.03358	1.03582
P_T/P		1.04632	1.04215	1.01040	1.02876	1.03334	1.1238	1.1323
P	PSIA	32.78	32.77	33.52	32.88	32.71	29.84	29.35
qc	PSI	1.52	1.38	0.35	0.95	1.09	3.69	3.88
ρ	ft	4.767			14.45			1.96
W_a/gp		.371			.122			.300
RN		3.1×10^6			1.1×10^6			2.45×10^6
f					.0035			.0031
1/D					1.90			11.3
$\Delta H/q$.10	.20	.10	.027	.25	.10	.14
ΔH		.15	.28	.04	.03	.27	.37	.54
V_{T_x}							0	75
V_{T_y}							75	220
$V_{T_y} - V_{T_x}$							5625	42,775
P_{T_y}/P_{T_x}							1.00223	1.01698
ΔH								.07

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TABLE 7 Continued

Station	Units	16	17a	18a	19a	20a	21a	22a
W _a	#/sec	18.97	9.48	9.48	9.49	9.48	9.48	
T _T	°R	740	740	740	740	740		
P _T	PSIA	33.25	35.25	36.01	36.85	36.84	36.54	36.22
A	in ²	44.2	24.3	24.3	24.3	48.4		
RW _a /P _T A		.688	.625	.577	.564	.284		
V	ft/sec	557	498	453	441	212		
T	°R	714.41	719.55	723.08	723.97	736.30		
γ		1.3950	1.3948	1.3947	1.3947	1.3943		
γ/γ - 1		3.5316	3.5329	3.5336	3.5336	3.5361		
T _T /T		1.03582	1.02842	1.02340	1.02214	1.00503		
P _T /P		1.1323	1.1040	1.0850	1.08035	1.01790		
P	PSIA	29.37	30.12	33.19	34.11	36.19		
q _c	PSI	3.88	3.13	2.82	2.74	.65		
p	ft		17.08					
W _a /q _p			.172					
RN			1.45x10 ⁶					
f			.0035					
1/D			44.0					
ΔH/g		.001	.58	.15	.15	2.0		
ΔH		0	1.82	.42	.41	1.30	.32	
V _{Tx}			220	618	680			
V _{Ty}			618	680	700			
V _{Ty} - V _{Tx}			333,524	80,476	27,600			
P _{Ty} /P _{Tx}			1.1378	1.0350	1.01089			
ΔH		.56	0	4.58	1.26	.40		

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TABLE 7 Continued

Station	Units	23a	17b	18b	19b
Qty.	Units				
W _a	#/sec		9.48	9.48	9.48
T _a	°R	1885	740	740	740
P _T	PSIA	35.22	33.25	35.89	36.62
A	in ²	22.51	22.1	22.1	22.1
RW _a /P _T A			.688	.637	.624
V	ft/sec	1932	557	507	.497
T	°R	1594	714.41	718.80	719.63
γ			1.3950	1.3949	1.3948
γ/γ - 1			3.5316	3.5323	3.5329
T _T /T			1.03582	1.02949	1.02831
F _T /P			1.1323	1.1060	1.1034
P	PSIA		29.37	32.39	33.19
q _c	PSI		3.88	3.50	3.43
ρ	ft		1.39		
W _a /EP			2.11		
RM			1.65x10 ⁶		
f			.0032		
1/D			39.34		
ΔH/q			.50	.15	.15
ΔE			1.94	.53	.51
V _{Tx}					
V _{Ty}					
V _{Ty} - V _{Tx}					
P _{Ty} /P _{Tx}			1.1378	1.0350	1.01089
ΔH			4.08	1.25	.40

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TABLE 7 Continued

<u>Station</u>		<u>20b</u>	<u>21b</u>	<u>22b</u>	<u>23b</u>
Qty.	Unit				
W _a	#/sec	9.48	9.48		
T _T	°R	740			1865
P _T	PSIA	36.81	35.21	34.91	34.91
A	in ²	48.4			22.51
RW _a /F _T A		.283			
V	ft/sec	213			1922
T	°R	756.26			1576
γ		1.3943			
γ/γ-1		3.8361			
T _T /T		1.00508			
P _T /P		1.01808			
P	PSIA	35.86			
q ₀	PSI	.65			
P	ft				
W _a /q _F					
RN					
1/D					
ΔH/q			2.0		
ΔH			1.30	.30	
V _{Tx}					
V _{Ty}					
V _{Ty} -V _{Tx}					
P _{Ty} /P _{Tx}					
ΔH					

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TABLE 7 Continued

Duct		a	b	Total Model 78
Qty.	Unit			
A	ft ²	.1563	.1503	469
P _{T22}	PSIA	35.22	34.91	
γ		1.3649	1.3664	
γ/γ - 1		3.7475	3.7293	
(2/γ + 1) ^{γ-1}		.5345	.5375	
P ₂₃	PSIA	18.83	18.66	
W _E	#/sec	9.67	9.61	
T ₂₃	°R	1594	1576	
T _T	°R	1885	1865	1875
C _{pav}		.2575	.2571	
ΔT	°R	858	840	
W _F	#/sec	.12931	.12637	276704
W _F	#/hr	468.5	454.9	2761
F/A		.01364	.01333	
V	ft/sec	1932	1922	
$\frac{Wq}{g} \Delta V$	#	350	348	
(ΔP)A	#	93	89	
F _J	#	443		2634
H _{req}				4290
HP _{out put/engine}				2283
W _{FE} /engine				1480
W _{FE} total				2960
W _{FE} + W _{FJ}				3721
Overall SFC				2.17