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HEAT TRANSFER TO SKIN THROUGH THERMALLY-IRRADIATED DRY CLOTH

Technical Report No. 6

from

Fuels Research Laboratory
Massachusetts Institute of Technology
Cambridge, Massachusetts

FC
BAC



Project DSR 7666

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ABSTRACT

↙ An analysis is presented of the problem of heat transmission through dry opaque or diathermanous cloth and an air space into underlying skin, following exposure to thermal radiation. The problem is solved numerically by finite difference approximations, the method being given in detail. It is shown that a technique requiring only the appropriate weighting of temperature-rise data obtained from three curves yields results within 1 to 2% of those obtained by extensive machine calculations in which allowance is made for energy absorption at each of several increments into which the diathermanous cloth is conceptually divided. The analytical work is verified by good agreement with experimental results. ↘

I. INTRODUCTION

An experimental research program carried on in the Fuels Research Laboratory during the past two years has focussed on development of skin simulants and their use to obtain temperature-time-depth data in complex multi-layer systems, such as skin behind various dry or moisture-bearing fabrics. A parallel theoretical program was a study of the unsteady-state heat conduction through such multi-layer systems. The role played by the fabrics in heat and mass transfer when exposed to high intensity thermal radiation is probably very complex due both to the inherent heterogeneity of the structure of all fabrics and to the interaction of energy and mass transfer due to moisture in the cloth. However, it is quite possible that, based on the experimental temperature-time-depth data obtained from skin simulant studies, a set of simplifying assumptions might be made to describe adequately the complex model, so that a mathematical analysis can be carried out. This report is concerned only with the problem of heat transfer through dry cloth to skin. The additional complexity due to moisture in the cloth is presently under active study.

A previous attempt at this type of mathematical analysis was made by Ho and Williams (1) in connection with study of an air-spaced cloth-polyethylene system. They tried to correlate experimentally-determined temperatures of the top cloth surface, the bottom cloth surface, and the polyethylene surface with the respective calculated temperatures. Unfortunately, due to slight errors in their theoretical equations, use of erroneous values of cloth conductivity and effects of uncontrolled moisture and chemical reaction in the experiments, their results were far from expectation, and for a time discouraged further attempts at analysis. Their mathematical analysis was based on the assumption that the cloth layer could be treated as a homogeneous solid, which might be a fair approximation for the underlying polyethylene or other skin simulant, but not for the

cloth layer itself. Therefore the analysis could hardly be expected to predict temperature profiles in the cloth layer. Furthermore, the complexity of the cloth structure makes it impossible to identify accurately the cloth "surface", and consequently the significance of experimentally measured temperatures was ambiguous at best.

Since the main interest is to obtain temperature-time patterns at various depths of skin, and since now it is possible to obtain these data experimentally by the use of skin simulants with depth magnification,⁽²⁾ it seems logical that the theoretical analysis should be directed toward calculating temperature-time data within the simulated skin rather than in the overlying cloth layer. With this purpose in mind, the present work was initiated. The study consists of the correction and modification of Ho and Williams' work, the extension of the analysis to cover both opaque and diathermanous cloths, a proposed modification of the Schmidt technique for calculating unsteady-state heat conduction through heterogeneous systems, the use of a "three-curve" method in solving diathermanous cloth problems, and the use of an IBM-704 computer in calculating the numerical data over the range of interest.

II ANALYSIS OF HEAT TRANSFER IN CLOTH-AIR-SKIN SYSTEMS

A. General Assumptions.

The system under consideration is sketched below.

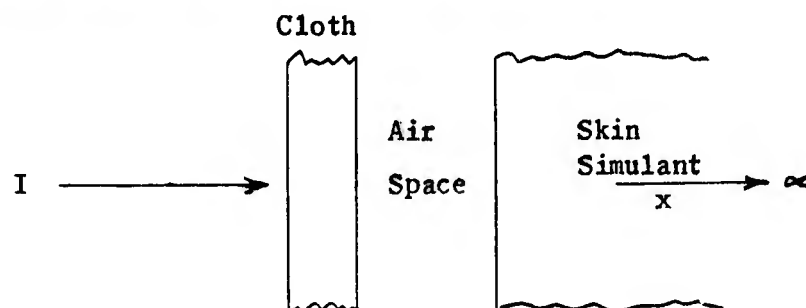


Figure 1.

The following assumptions are made in the analysis:

1. Heat flow is uni-dimensional, occurring in the x-direction, from the cloth through the air space into the skin simulant. The surface of the cloth is irradiated by square wave radiation of uniform intensity.

2. The cloth layer, of thickness L_c is considered to be a homogeneous opaque solid for cloth for which no measurable transmittance has been reported, and for cloth of finite transmittance the radiant energy is assumed to penetrate into the interior of the cloth layer in accordance with the Beer-Lambert Law:

$$I_x = I_0 e^{-\gamma x_c} \quad (1)$$

where I_x = intensity of radiation at a plane distant x_c cm. from the irradiated surface,

I_0 = intensity of radiation entering the cloth layer, cal/cm² sec.,

γ = extinction coefficient, cm⁻¹, a characteristic constant of the cloth for the particular source of radiation in question.

This assumption is made with reference to the particular series of cloths under study in the experimental program at this laboratory. These cloths, ranging in shade from white to deep black, are made from the same white 9-ounce cotton sateen stock, and have measured transmittance values ranging from 0 to 14.1%. Since it is unlikely that a simple dyeing process would change the geometry of the fibers, it is reasonable to assume that the cloth construction provides complete geometrical blockage of incident light and that the transmittance depends on the degree of opacity of the individual cloth fibers. These transmittance values are therefore regarded not as direct transmittance but as diathermancy, as stated in the assumption.

In section V of this report a general method for solving problems of heat transmission through cloth, an air space, and underlying skin is proposed; it

can be applied to either diathermanous cloth samples or to those which allow direct transmission of radiant energy.

3. The physical properties of the cloth layer remain constant throughout the exposure.

4. Any effects due to moisture and/or chemical reaction are negligible.

5. Heat losses from the front surface of the cloth layer, by convection and re-radiation, are characterized by a single overall heat transfer coefficient U_o .

6. Heat transfer from the cloth layer through the air gap to the simulant is similarly characterized by an overall heat transfer coefficient U_i .

7. The skin simulant is considered to be a homogeneous opaque solid, of infinite extent in its x-direction.

B. Derivation of Equations for an Opaque-Cloth, Air, Skin-Simulant System.

Consider the element dx_c in the cloth layer.

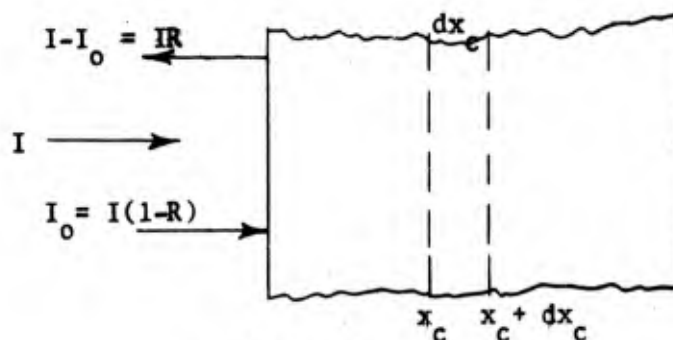


Figure 2.

Per unit area, the rate of heat input at x_c is

$$-k_c \frac{\partial T}{\partial x_c} \quad (2)$$

the rate of heat output at $x_c + dx_c$ is:

$$-k_c \frac{\partial T}{\partial x_c} + \frac{\partial}{\partial x_c} \left(-k_c \frac{\partial T}{\partial x_c} \right) dx_c \quad (3)$$

The heat accumulation in dx_c is:

$$(cp)_c dx_c \frac{\partial T}{\partial \theta_c} \quad (4)$$

Since input minus output equals accumulation, we have:

$$k_c \frac{\partial^2 T}{\partial x_c^2} = (cp)_c \frac{\partial T}{\partial \theta_c} \quad (5)$$

Similarly, for the skin simulant, which is an opaque solid, the heat accumulation is:

$$k_s \frac{\partial^2 T}{\partial x_s^2} = (cp)_s \frac{\partial T}{\partial \theta_s} \quad (6)$$

The boundary conditions are:

1. At the front surface of the cloth,

$$I_0 - U_0 (T_{x_c=0} - T_{surr.}) = -k_c \left. \frac{\partial T}{\partial x_c} \right]_{x_c=0} \quad (7)$$

2. In the air gap between the cloth and the skin simulant, neglecting the heat capacity of the air,

$$-k_c \left. \frac{\partial T}{\partial x_c} \right]_{x_c=L_c} = U_i (T_{x_c=L_c} - T_{x_s=0}) = -k_s \left. \frac{\partial T}{\partial x_s} \right]_{x_s=0} \quad (8)$$

These nonlinear boundary conditions make the exact solutions of the partial differential equations (5) and (6) unobtainable. Hence the method of finite difference approximations commonly known as the Schmidt method is used. Some modifications of the original Schmidt technique have been made to give answers closer to the exact solution, as will be explained in the next section.

III. FINITE DIFFERENCE APPROXIMATION METHOD

Expressing the above-mentioned differential equations and boundary equations in finite increment terms, we have:

$$k_c \frac{\Delta_x (\Delta_x T)}{\Delta x_c^2} = (cp)_c \frac{\Delta \theta^T}{\Delta \theta_c} \quad (5)$$

$$k_s \frac{\Delta_x (\Delta_x T)}{\Delta x_s^2} = (cp)_s \frac{\Delta \theta^T}{\Delta \theta_s} \quad (6)$$

$$I_o - U_o \left(T_{x_c=0} - T_{surr.} \right) = -k_c \left. \frac{\Delta x^T}{\Delta x_c} \right]_{x_c=0} \quad (7)$$

$$\left. -k_c \frac{\Delta x^T}{\Delta x_c} \right]_{x_c=L_c} = U_i \left(T_{x_c=L_c} - T_{x_s=0} \right) = -k_s \left. \frac{\Delta x^T}{\Delta x_s} \right]_{x_s=0} \quad (8)$$

Multiplying equation (5) by $\frac{1}{I_o L_c} \left(\frac{1}{L_c} \right)^2$, we obtain

$$\frac{\Delta_x \left(\frac{\Delta_x T k_c}{I_o L_c} \right)}{\Delta \left(\frac{x_c}{L_c} \right)^2} = \frac{k_c \Delta \theta^T}{I_o L_c} \quad (9)$$

$$\frac{\Delta_x \left(\frac{\Delta_x T k_c}{I_o L_c} \right)}{\Delta \left(\frac{x_c}{L_c} \right)^2} = \frac{\left(\frac{k}{cp} \right)_c \left(\frac{1}{L_c} \right)^2 \Delta \theta_c}{I_o L_c}$$

A. General equations in dimensionless forms.

To make the solution applicable to problems involving the same system but having different physical constants, the basic equations derived in Section II are transformed into dimensionless forms.

It is convenient to define the dimensionless groups as follows:

$$T^o = \frac{T k_c}{I_o L_c}, \quad \text{the dimensionless temperature group;}$$

$$x_c^o = \frac{x_c}{L_c}, \quad \text{the dimensionless distance group, measured from the front surface of the cloth; and}$$

$$\theta_c^o = \left(\frac{k}{cp} \right)_c \frac{\Delta \theta_c}{L_c^2}, \quad \text{the dimensionless time group}$$

Thus, equation (9) can be rewritten as:

$$\frac{\Delta_x (\Delta_x T^0)}{\Delta \left(\frac{x^0}{L_c}\right)^2} = \frac{\Delta_\theta T^0}{\Delta \theta_c^0} \quad (9)$$

Equation (6) multiplied by the term $\frac{k_c}{I_0 L_c} / \left(\frac{1}{L_c}\right)^2$ yields:

$$\frac{\Delta_x \left(\frac{\Delta_x T k_c}{I_0 L_c} \right)}{\Delta \left(\frac{x_s}{L_c}\right)^2} = \frac{\frac{\Delta_\theta T k_c}{I_0 L_c}}{\left(\frac{k}{c\rho}\right)_s \frac{\Delta_\theta s}{L_c^2}} \quad (10)$$

The dimensionless skin-depth group is defined as:

$$x_s^0 = \frac{x_s}{L_c} \cdot \frac{k_c}{k_s} \quad \text{and consequently, from equation (10)}$$

$$\theta_s^0 = \left(\frac{k}{c\rho}\right)_s \cdot \frac{\theta_s}{L_c^2} \cdot \left(\frac{k_c}{k_s}\right)^2 \quad \text{is the definition of the dimensionless time group for the skin layer.}$$

When both sides of equation (10) are multiplied by $\frac{1}{(k_c/k_s)^2}$ it can be re-

$$\text{written as} \quad \frac{\Delta_x (\Delta_x T^0)}{\Delta \left(\frac{x_s^0}{L_c}\right)^2} = \frac{\Delta_\theta T^0}{\Delta \theta_s^0} \quad (10)$$

Boundary equation (7) can be rearranged as

$$\left. \frac{\Delta_x T}{\Delta x_c} \right]_{x_c=0} = - \frac{I_0}{k_c} + \frac{U_0}{k_c} \left(T_{x_c=0} - T_{\text{surr.}} \right)$$

Multiplying both sides by $\left(\frac{k_c}{I_0 L_c}\right) / \left(\frac{1}{L_c}\right)$, we obtain

$$\left. \frac{\Delta_x T \frac{k_c}{I_o L_c}}{\Delta \left(\frac{x_c}{L_c} \right)} \right|_{x_c=0} = -1 + \frac{U_o L_c}{k_c} \left[\frac{k_c T_{x_c=0}}{I_o L_c} - \frac{k_c T_{surr.}}{I_o L_c} \right] \quad (11)$$

If a dimensionless surface-heat-loss group, U_o^o , is defined as $\frac{U_o L_c}{k_c}$,

equation (11) can be rewritten as:

$$\left. \frac{\Delta T^o}{\Delta X_c^o} \right|_{X_c^o=0} = - \frac{\frac{1}{U_o^o} - \left(T_{X_c^o=0}^o - T_{surr.}^o \right)}{\frac{1}{U_o^o}} \quad (11)$$

Similarly, if the air gap group U_i^o is defined in dimensionless terms as $\frac{U_i L_c}{k_c}$

boundary equation (8) can be converted to:

$$\left. \frac{\Delta T^o}{\Delta X_c^o} \right|_{X_c^o=1} = \frac{T_{X_c^o=1}^o - T_{X_s^o=0}^o}{\frac{1}{U_i^o}} = - \left. \frac{\Delta_x T}{\Delta X_s^o} \right|_{X_s^o=0} \quad (12)$$

In summarizing the above general equations, it is found that seven dimensionless groups govern the temperature-time-depth pattern of the system, namely

1. Temperature group,

$$T^o = \frac{k_c T}{I_o L_c}$$

2. Time group for cloth

$$\theta_c^o = \left(\frac{k}{c\rho} \right)_c \left(\frac{\theta_c}{L_c^2} \right)$$

3. Time group for skin

$$\theta_s^o = \left(\frac{k}{c\rho} \right)_s \frac{\theta_s}{L_c^2} \left(\frac{k_c}{k_s} \right)^2$$

4. Depth group for cloth

$$X_c^o = \frac{x_c}{L_c}$$

5. Depth group for skin

$$X_s^o = \frac{x_s}{L_c} \frac{k_c}{k_s}$$

$$6. \text{ Surface heat losses group } U_o^o = \frac{U_o L_c}{k_c}$$

$$7. \text{ Air gap group, } U_i^o = \frac{U_i L_c}{k_c}$$

A relationship between the thermal properties of the cloth and skin would make it possible to eliminate the time and depth groups for skin. In establishing this relationship, the following rules are observed in order to simplify the mathematical manipulation:

1. The modulus, M , defined as $\frac{\Delta(x^o)^2}{\Delta\theta^o}$ is to be the same in both the skin and the cloth, and

2. The actual time θ (in seconds) is to be the same in both the skin and the cloth, i.e., $\theta_c = \theta_s$.

$$\text{Thus } \frac{\Delta(x_c^o)^2}{\Delta\theta_c^o} = \frac{\Delta(x_s^o)^2}{\Delta\theta_s^o} \quad (13)$$

and by substitution

$$\frac{\Delta x_c^o}{\Delta x_s^o} = \frac{\sqrt{\left(\frac{k}{cp}\right)_c \frac{\theta_c}{L_c^2}}}{\sqrt{\left(\frac{k}{cp}\right)_s \left(\frac{k_c}{k_s}\right)^2 \frac{\theta_s}{L_c^2}}} = \sqrt{\frac{(kcp)_s}{(kcp)_c}} \quad (14)$$

From equation (14) it is seen that by using cloth as the reference material, the corresponding size of each increment in the skin simulant is inversely proportional to the square root of the kcp ratio of skin and cloth.

The reason for defining X_s^o as $x_s k_c / L_c k_s$ instead of simply x_s / L_c can be best explained by making a heat balance at the interface of a composite slab, sketched in Figure 3.

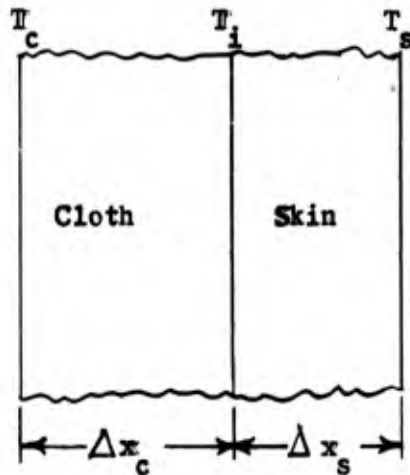


Figure 3.

The boundary condition requires the heat fluxes to be equal at the interface, or:

$$\frac{k_c}{\Delta x_c} (T_c - T_i) = \frac{k_s}{\Delta x_s} (T_i - T_s) \quad (15)$$

Letting $R = \frac{\Delta x_c}{\Delta x_s} \cdot \frac{k_s}{k_c}$, we have,

$$T_i = \left(\frac{1}{1+R} \right) T_c + \left(\frac{R}{1+R} \right) T_s \quad (16)$$

If the dimensionless depth groups are defined in such a way that $\Delta X_c^0 / \Delta X_s^0 = R$, then T_i in equation (16) can be evaluated graphically by drawing a straight line connecting T_c and T_s as shown in Fig. 4. Therefore, when X_c^0 is defined as X_c/L_c , it follows that X_s^0 should be $\frac{X_s}{L_c} \cdot \frac{k_c}{k_s}$ instead of

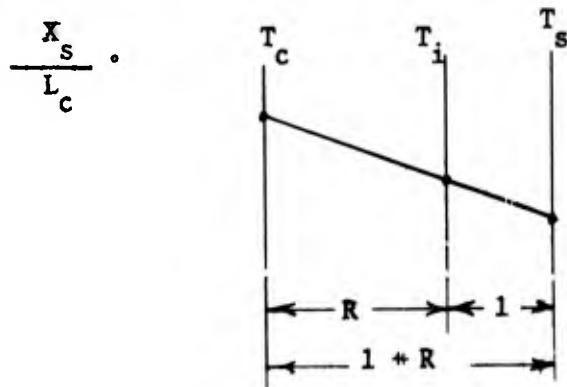


Figure 4.

B. Numerical Solution - Modified Schmidt Technique.

The solution of equations (9)' and (10)' with the boundary conditions set by equations (11)' and (12)' can best be illustrated by working out an actual case.

1. The independent variables U_o^0 , U_i^0 , and the ratio R (equivalent to $\Delta X_c^0 / \Delta X_s^0$ or $\sqrt{\frac{(k\phi)_s}{(k\phi)_c}}$) must be assigned for each particular case. For the example, let us say $U_o^0 = 0.1$; $U_i^0 = 1.0$; $R = 5.0$.

2. With the cloth layer as the reference material, it is convenient to divide the cloth layer into two increments or slices. Each cloth slice, ΔX_c^0 , is therefore 0.5 dimensionless units thick and each skin slice, ΔX_s^0 , is accordingly 0.1 unit.

In the Schmidt technique reference planes for temperature are located at the center of the slices and also half a slice beyond the surfaces of cloth and skin as shown in Figure 5. This procedure, devised by Schmidt and discussed by McAdams (3) yields an improved approximation of the true solution, over that which would be obtained without the concept of half-slices beyond the surfaces.

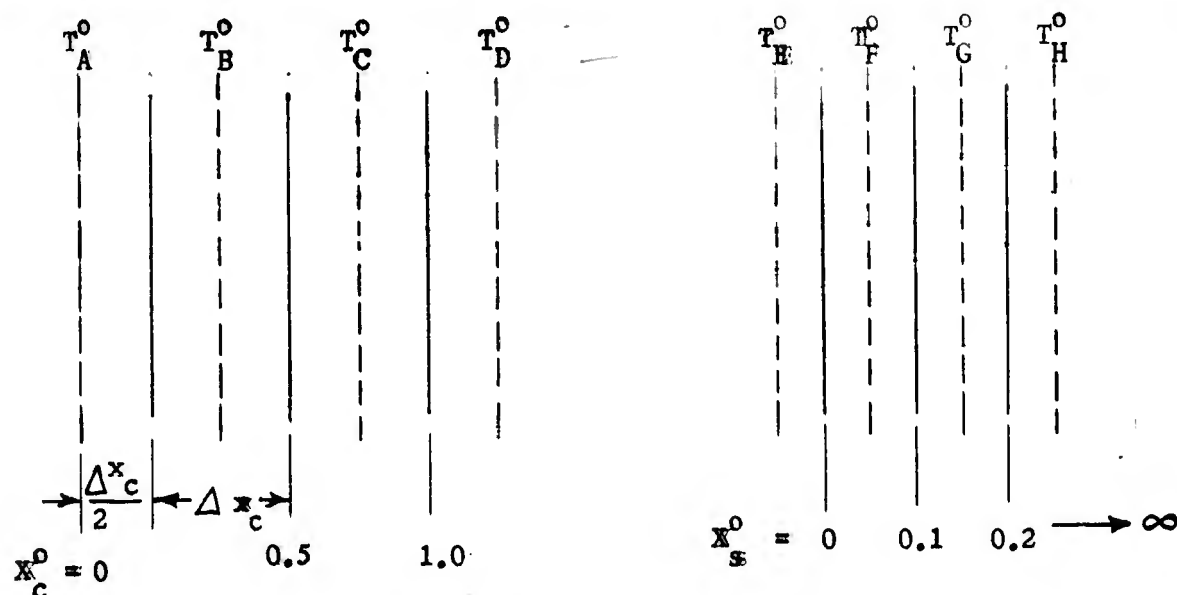


Figure 5.

3. The solution is started by considering boundary equation (11)

$$\left. \frac{\Delta T^0}{\Delta X_c^0} \right|_{X_c^0 = 0} = \frac{\frac{1}{U_o^0} - \left(T_{X_c^0 = 0}^0 - T_{\text{surr.}}^0 \right)}{\frac{1}{U_o^0}} \quad (11)$$

Referring to Figure 5,

$$\text{Let } \frac{T_A^0 + T_B^0}{2} = T_{X_c^0 = 0}^0 = 0; \quad T_{\text{surr.}}^0 = 0; \quad \text{and } - \frac{(T_A^0 - T_B^0)}{\Delta X_c^0} = \left. \frac{\Delta_x T^0}{\Delta X_c^0} \right|_{X_c^0 = 0}$$

Equation (11) can be re-written as:

$$\frac{T_A^0 - T_B^0}{\Delta x_c^0} = \frac{2/U_0^0 - T_A^0 - T_B^0}{2/U_0^0} \quad (17)$$

Rearranging equation (17), we obtain

$$T_A^0 = \frac{\Delta x_c^0 \left(\frac{1}{U_0^0} - T_B^0 \right) + \frac{\Delta x_c^0}{2} T_B^0 + \frac{1}{U_0^0} T_B^0}{\frac{1}{U_0^0} + \frac{\Delta x_c^0}{2}} \quad (18)$$

$$= \frac{\Delta x_c^0 \left(\frac{1}{U_0^0} - T_B^0 \right)}{\frac{1}{U_0^0} + \frac{\Delta x_c^0}{2}} + T_B^0 \quad (18)$$

or in another form,

$$\frac{T_A^0 - T_B^0}{\Delta x_c^0} = \frac{\frac{1}{U_0^0} - T_B^0}{\frac{1}{U_0^0} + \frac{\Delta x_c^0}{2}}$$

Equation (18) is shown graphically in Fig. 6 by connecting the point O of the cloth surface with plane B at the point where $T_B^0 = 0$. This line crosses plane A and establishes the starting T_A^0 at $\theta_c^0 = 0$.

After $T_{A,0}^0$ (the first subscript denotes the temperature plane, and the second a dimensionless time unit) is obtained, the temperature distribution pattern inside the cloth layer is calculated by the use of equation (9),

$$\frac{\Delta_x \left(\Delta_x T^0 \right)}{\Delta (x_c^0)^2} = \frac{\Delta_\theta T^0}{\Delta \theta_c^0} \quad (9)$$

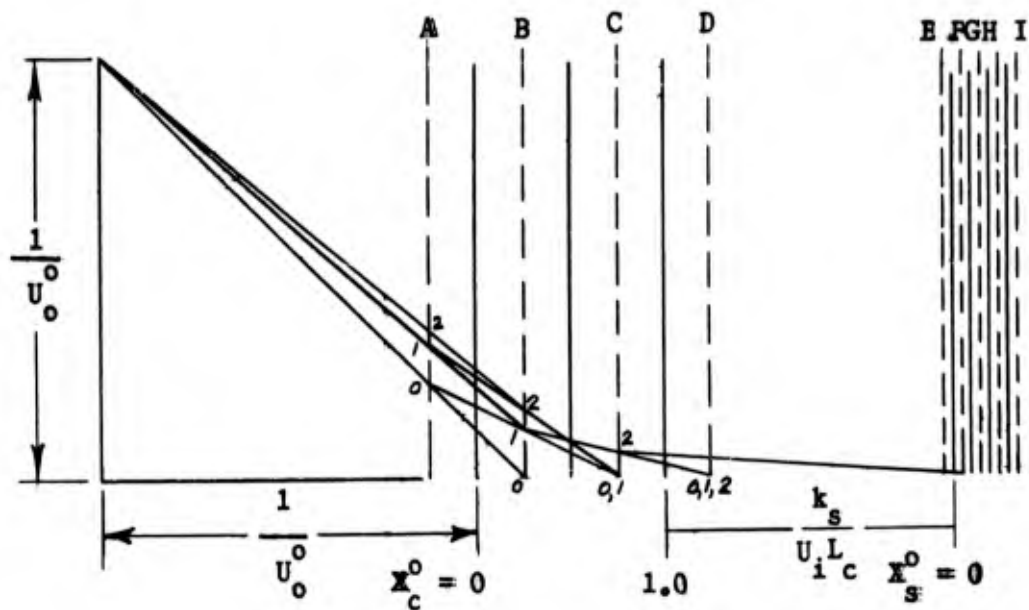


Figure 6.

For plane B, at time unit 1, equation (8)' can be written as

$$(T_{C,0}^0 - T_{B,0}^0) - (T_{B,0}^0 - T_{A,0}^0) = \frac{\Delta(x_c^0)^2}{\Delta\theta_c^0} (T_{B,1}^0 - T_{B,0}^0) \quad (19)$$

Both the dimensionless depth group and the dimensionless time group are independent variables, the choice of their values being entirely arbitrary. But considerable simplification in the arithmetic manipulation is possible if $\Delta(x_c^0)^2 / \Delta\theta_c^0$ is chosen to be 2 as recommended by Schmidt because this causes $T_{B,0}^0$ terms to be cancelled in equation (18), leaving:

$$T_{B,1}^0 = \frac{T_{A,0}^0 + T_{C,0}^0}{2} \quad (20)$$

Equation (20) is represented on Figure 6 as a straight line drawn from $T_{A,0}^0$ on plane A to the datum point on plane C; the line crosses plane B, and establishes $T_{B,1}^0$.

Equation (20) can be written in the general form:

$$T_{n,m+1}^0 = \frac{T_{n-1,m}^0 + T_{n+1,m}^0}{2} \quad (21)$$

This equation can be used alone until heat reaches the back surface of the cloth, i.e., when in this example T_C^0 becomes finite. Then the heat transfer

through the air gap would begin and boundary equation (12) must be considered. Summarizing, the equations used in the calculation of this example can be expressed by the following algebraic forms, from equations (18) and (21):

$$T_{A,m}^0 = 0.04878 (10 - T_{B,m}^0) + T_{B,m}^0 \text{ and}$$

$$T_{n,m+1}^0 = 0.5 (T_{n-1,m}^0 + T_{n+1,m}^0)$$

It is to be noted here that the foregoing starting technique is different from that suggested by Schmidt. The main difference between the methods lies in the way of fulfilling boundary equation (11):

$$\left. \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 0} = - \frac{\frac{1}{U_0^0} - \left(T_{X_c^0}^0 = 0 - T_{\text{surr.}}^0 \right)}{1/U_0^0} \quad (11)$$

Schmidt pointed out that at time 0, $T_{X_c^0}^0 = 0 = T_{\text{surr.}}^0$, so that

$$\left. \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 0} = - \frac{1/U_0^0}{1/U_0^0} = -1 \quad (22)$$

This is represented graphically in Figure 7 by the line which connects the starting point with the point where $X_c^0 = 0$ and $T^0 = T_0^0$. This line establishes T_A^0 for the starting time $\theta^0 = 0$ at the point where it crosses plane A. Schmidt's procedure for continuation to determine T_B^0 , etc., has been used in the present work. Comparing the starting procedures shown in Figures 6 and 7, however, it is clear that our method gives a higher value than Schmidt's for $T_{A,0}^0$. It is also somewhat closer to the true solution, due to a partial balance of compensating errors. In our approximation of the surface boundary condition, equation (11), the heat capacity of the half slice of cloth from plane B to the surface

is neglected, which is the same as assuming for the moment that it equals that of air. The error is partially compensated in the next step, determination of $T_{B,1}^0$, in which the air between plane A and the surface is in effect assigned a heat capacity equal to that of cloth. In the Schmidt starting procedure there is no error in the first step but in the second step all the space between planes A and B is implicitly assumed to have the heat capacity of cloth, an error for which no compensation is made.

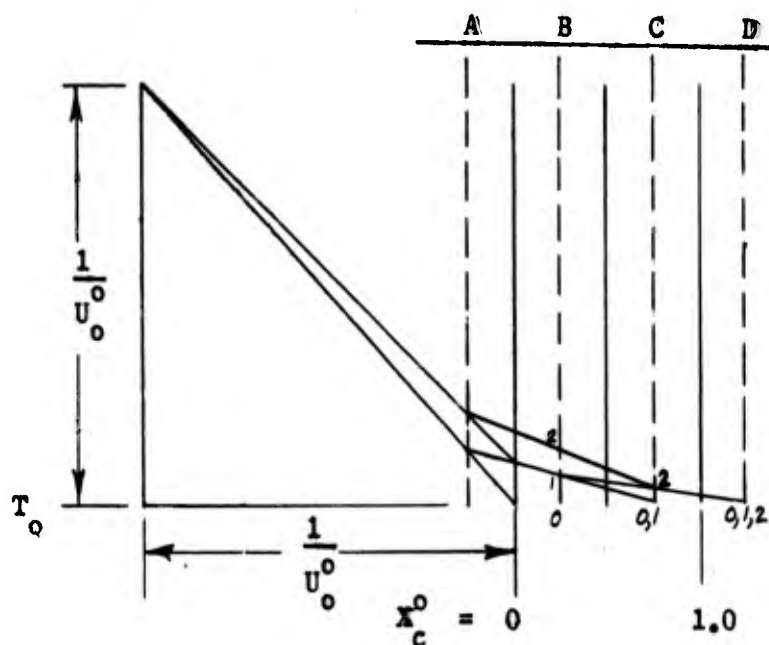


Figure 7.

4. Temperature Rise in Air Gap and Skin Simulant.

From boundary equation (12) and Figure 5,

$$-\left. \frac{\Delta T^0}{\Delta X_c^0} \right]_{X_c^0 = 1} = U_i^0 \left(T_{X_c^0 = 1}^0 - T_{X_s^0 = 0}^0 \right) = - \left. \frac{\Delta T^0}{\Delta X_s^0} \right]_{X_s^0 = 0} \quad (12)$$

Letting $\frac{T_C^0 - T_D^0}{\Delta X_c^0} = - \left. \frac{\Delta T^0}{\Delta X_c^0} \right]_{X_c^0 = 1}$ and $\frac{T_H^0 - T_F^0}{\Delta X_s^0} = - \left. \frac{\Delta T^0}{\Delta X_s^0} \right]_{X_s^0 = 0}$

it follows that

$$\frac{T_C^0 - T_D^0}{\Delta X_c^0} = U_i^0 \left(\frac{T_C^0 + T_D^0}{2} - \frac{T_E^0 + T_F^0}{2} \right) = \frac{T_E^0 - T_F^0}{\Delta X_s^0} \quad (23)$$

From equation (23),

$$T_E^0 = \Delta X_s^0 \left(\frac{T_C^0 - T_D^0}{\Delta X_c^0} + \frac{T_F^0}{\Delta X_s^0} \right) \quad (24)$$

and
$$\frac{T_E^0 + T_F^0}{2} = \frac{\Delta X_s^0}{2} \left(\frac{T_C^0 - T_D^0}{\Delta X_c^0} + \frac{2 T_F^0}{\Delta X_s^0} \right) \quad (25)$$

Substituting equation (25) back to equation (23) and eliminating T_E^0 , we obtain

$$\frac{T_C^0 - T_D^0}{\Delta X_c^0} = U_i^0 \left[\frac{T_C^0 + T_D^0}{2} - \left(\frac{T_C^0 - T_D^0}{\Delta X_c^0} + \frac{2 T_F^0}{\Delta X_s^0} \right) \frac{\Delta X_s^0}{2} \right] \quad (26)$$

By rearranging (26),
$$T_D^0 = \frac{(T_C^0 - T_F^0) \left[\frac{1}{U_i^0} + \frac{\Delta X_s^0 - \Delta X_c^0}{2} \right]}{\frac{1}{U_i^0} + \left(\frac{\Delta X_s^0 + \Delta X_c^0}{2} \right)} + T_F^0 \quad (27)$$

or
$$\frac{T_D^0 - T_F^0}{\frac{1}{U_i^0} + \frac{\Delta X_s^0}{2} - \frac{\Delta X_c^0}{2}} = \frac{T_C^0 - T_F^0}{\frac{1}{U_i^0} + \frac{\Delta X_c^0}{2} + \frac{\Delta X_s^0}{2}} \quad (28)$$

The last equation can also be obtained graphically by proportioning two similar triangles, as shown in Fig. 8. Similarly, T_E^0 can be calculated by

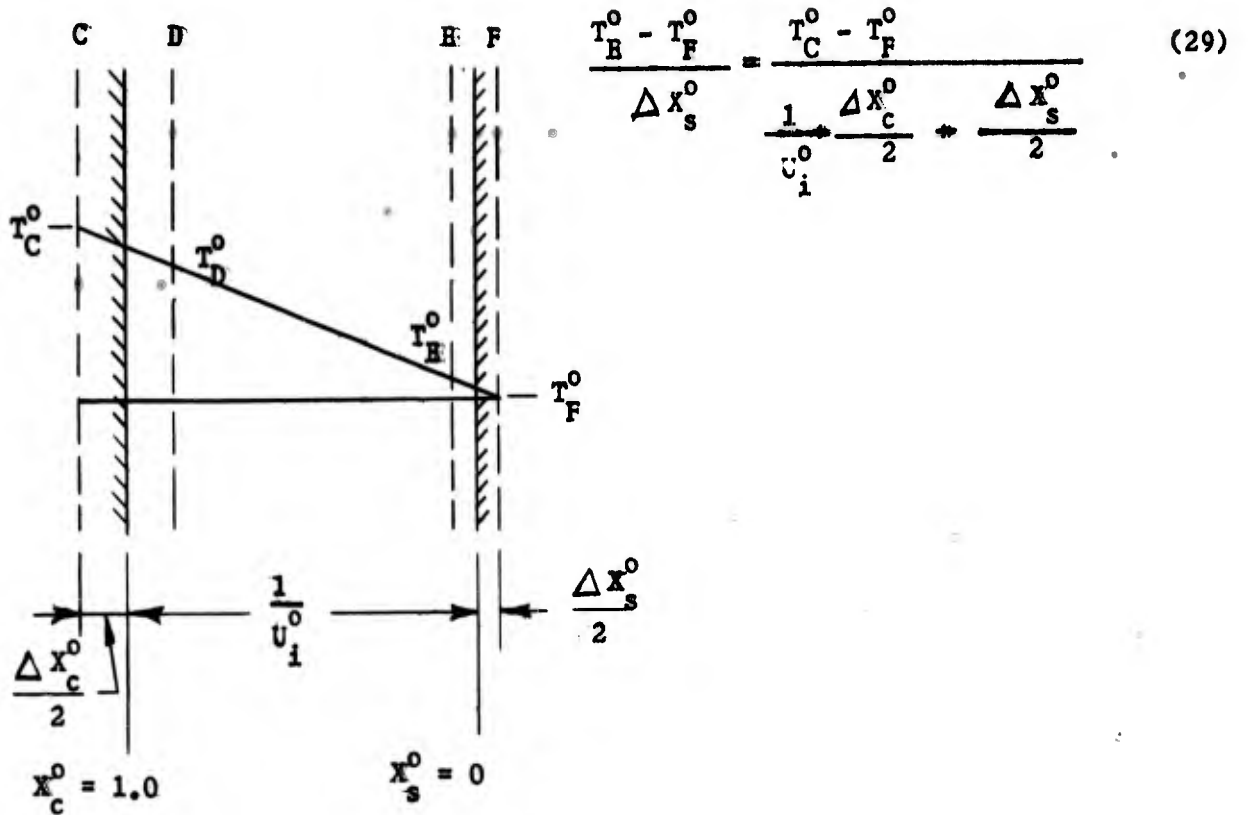


Figure 8.

or
$$T_E^0 = \frac{(T_C^0 - T_F^0) \Delta X_s^0}{\frac{1}{U_i^0} + \frac{\Delta X_c^0}{2} + \frac{\Delta X_s^0}{2}} + T_F^0 \quad (30)$$

definition for

The advantage of the used ΔX_s^0 is revealed in the simplicity of the calculation of heat transfer through the air gap. The calculation can also be made graphically, for as can be seen in Figure 8, if the planes $X_c^0 = 1$ and $X_s^0 = 0$ are separated by a distance corresponding to $\frac{1}{U_i^0}$, equation (12) can be satisfied by a straight line joining plane C and plane F making the heat fluxes at $X_c^0 = 1$, $X_s^0 = 0$ and $U_i^0 \left(\frac{T_{X_c^0=1}^0 - T_{X_s^0=0}^0}{X_c^0 - X_s^0} \right)$ equal to each other. The straight line crosses planes D and E and establishes T_D^0 and T_E^0 respectively.

Summarizing, the general equations for the two temperature planes inside

the air gap at any time are:

$$\frac{T_{D,m}^{\circ} - T_{F,m}^{\circ}}{T_{C,m}^{\circ} - T_{F,m}^{\circ}} = \frac{\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} - \Delta X_c^{\circ}}{2}}{\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} + \Delta X_c^{\circ}}{2}}$$

or $T_{D,m}^{\circ} = \frac{(T_{C,m}^{\circ} - T_{F,m}^{\circ}) \left[\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} - \Delta X_c^{\circ}}{2} \right]}{\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} + \Delta X_c^{\circ}}{2}} + T_{F,m}^{\circ}; \quad (31)$

similarly

$$\frac{T_{E,m}^{\circ} - T_{F,m}^{\circ}}{T_{C,m}^{\circ} - T_{F,m}^{\circ}} = \frac{\Delta X_s^{\circ}}{\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} + \Delta X_c^{\circ}}{2}}$$

or $T_{E,m}^{\circ} = \frac{(T_{C,m}^{\circ} - T_{F,m}^{\circ}) (\Delta X_s^{\circ})}{\frac{1}{U_i^{\circ}} + \frac{\Delta X_s^{\circ} + \Delta X_c^{\circ}}{2}} + T_{F,m}^{\circ}; \quad (32)$

In the example,

$$T_{D,m}^{\circ} = .6154 (T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ}$$

$$T_{E,m}^{\circ} = .07692 (T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ}$$

The final step is conduction of the heat into the interior of the skin simulant. Using equation (10) and setting $\Delta(X_s^{\circ})^2 / \Delta \theta_s^{\circ} = 2$ as in the case of conduction through the cloth layer, the same equation is of course found,

$$T_{n,m+1}^{\circ} = \frac{T_{n-1,m}^{\circ} + T_{n+1,m}^{\circ}}{2} \quad (21)$$

It is plain that equation (21) can be used throughout the skin simulant, which is treated as a semi-infinite solid.

Accuracy of Solution

It has been well established for this particular type of problem that the solution will converge as long as the modulus $M = \Delta(x_c^0)^2 / \Delta\theta_c^0$ is equal to 2 or more, and the only question remaining is the accuracy of the solution. It is generally true that an increase in either the modulus or the fineness of the increment would increase the accuracy of the solution. However, any modulus other than 2 would complicate the arithmetic of the problem; therefore varying the number of slices into which the cloth is conceptually divided is a better way to check on the accuracy of the solution.

It is also obvious that the more slices into which the cloth layer is divided, the closer will be the solution to the true solution. However, the number of dimensionless time increments required to arrive at a given corresponding time in seconds is proportional to the square of the number of slices into which the cloth is divided. There is therefore an optimum point in the division of the cloth into small increments, beyond which the precision gained will be small in comparison with the amount of effort required. Experience in performing the calculations with a desk calculator shows that the work becomes extremely long and tedious if one tries to divide the cloth into more than two slices. Thus, it is important to find out how good a two-slice cloth model is as compared with models using more than 2 slices.

With the aid of the IBM-704 digital computer at the MIT Computation Center, it has been possible to check a two-slice solution with a six-slice solution. The results are shown in Tables I and II. From this it is seen that at $\theta_c^0 = 2.50$, the difference between the two solutions is less than 1%, while at shorter times, such as $\theta_c^0 = 0.5$, the error is over 10%. Using a method commonly given

for estimating the true solution from approximate solutions, and specifically determining T_{∞}^0 from the 2-slice solution, T_2^0 , and the six-slice solution,

T_6^0 :

$$\frac{T_{\infty}^0 - T_2^0}{T_{\infty}^0 - T_6^0} = \left(\frac{6}{2}\right)^2, \quad (33)$$

when T_2^0 is 10% higher than T_6^0 , then

$$T_{\infty}^0 = \frac{9 \cdot (1.0) - 1.1}{8} = .9875, \text{ so that the im-}$$

provement to be expected from an infinite-slice model is less than 1.3%; in other words, very little improvement can be expected if one chooses to use a model having more than 6 cloth increments.

Since $\theta_c^0 = 0.5$ corresponds to a little over one second for the 9-ounce cotton sateen being studied in the experimental program here, use of a two-slice model would introduce a serious uncertainty in the short-time region. A compromise solution which can achieve the accuracy of a 6-slice model while requiring only approximately a quarter of the computation time is a combination of the six-slice and the two-slice models., i.e., the computation is started with a 6-slice model and continued until a relatively stable temperature profile prevails inside the skin layer, followed by a shift to a two-slice model using this pre-established temperature profile as the starting condition.

The data in Table I and Table II are also plotted in Fig. 9, and the estimated true solution (Eq. 33) is shown as a dotted line. It is to be noted that the true solution lies above the approximation at shorter times and below the approximation at longer times.

Finally, to test our modification of the Schmidt technique, mentioned in Section III, a calculation was made in which the cloth layer was assumed to be a semi-infinite solid (this would be valid for a short time after the irradiation is started, i.e., before appreciable heat reaches the back side of the

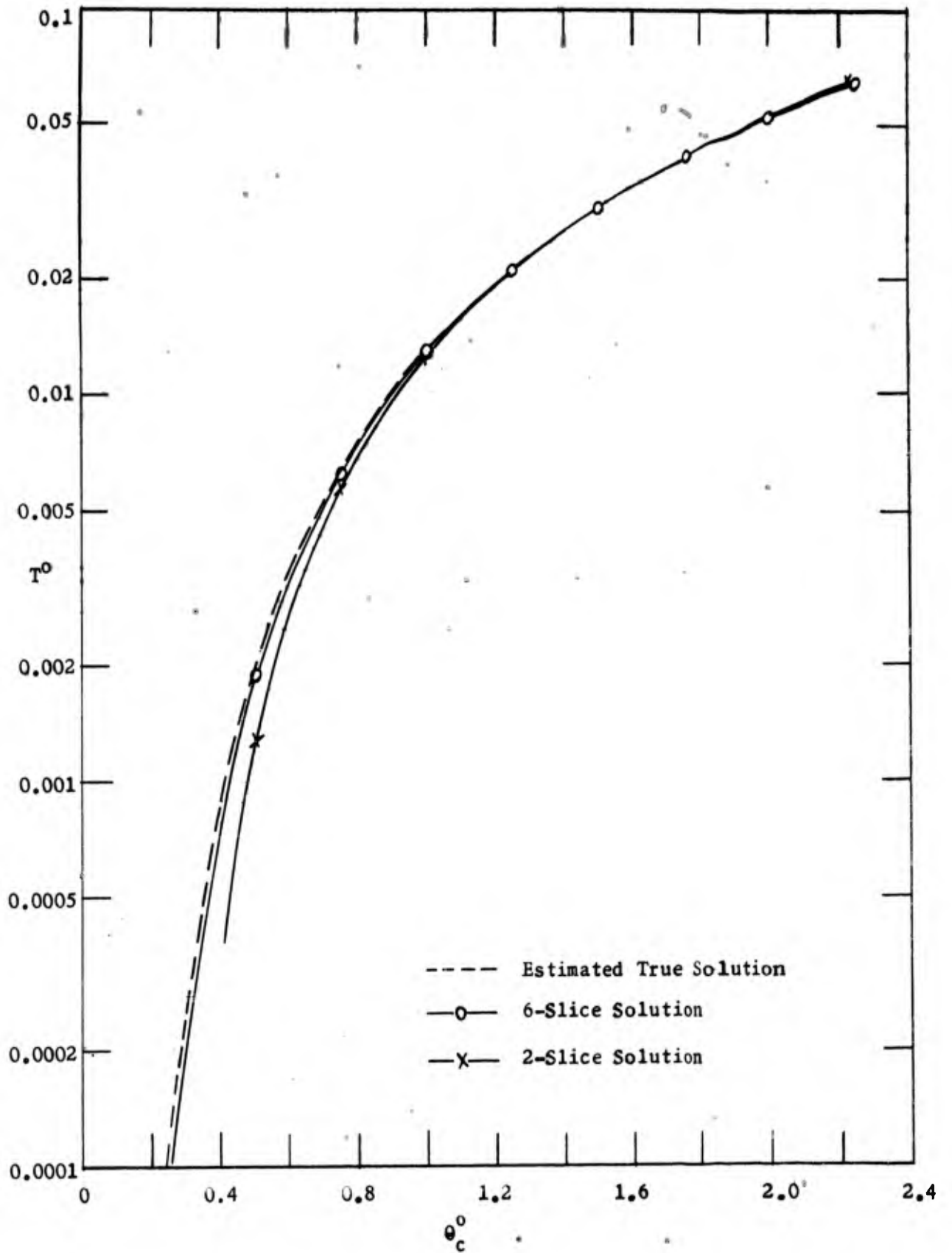


Figure 9. Comparison of Solutions of 2-Slice and 6-Slice Models with Estimated True Solution.

$$X_s^0 = 0.15$$

$$R = 5.0$$

$$U^0 = 0.04$$

$$U_i^0 = 0.5$$

cloth.) From the analytical solution given by Gardon (4) for such a case, the temperature at the time-position values corresponding to the data points of the finite-difference approximation were determined. These data are compared in Table III with the results obtained from the new starting technique and the original Schmidt technique. It is clearly evident the new starting technique is a better approximation of the true solution.

IV. APPLICATION TO DIATHERMANOUS-CLOTH-AIR-SKIN SYSTEMS.

As stated in the general assumptions (Section II, A.), the radiant energy received by a diathermanous cloth layer is assumed to penetrate into the interior of the cloth according to the Beer-Lambert Law. Strictly speaking, this law is applicable to monochromatic radiation only and for most materials, the extinction coefficient, γ , varies with the wave length of the incident radiation. Hence the use of a single extinction coefficient in the present study is merely a first approximation. It is of interest, however, to see how this approximation compares with the experimental results.

With radiation absorbed inside the cloth layer, the differential equation (5) for heat accumulation in the cloth now becomes:

$$-\frac{\partial}{\partial x_c} \left[-k_c \frac{\partial T}{\partial x_c} \right] dx_c - \frac{\partial}{\partial x_c} \left[I_0 e^{-\gamma x_c} \right] dx_c = (cp)_c dx_c \frac{\partial T}{\partial \theta} \quad (5)''$$

and the boundary condition at the front surface of the cloth becomes

$$-U_0 \left(T_{x_c=0} - T_{surr.} \right) = -k_c \left. \frac{\partial T}{\partial x_c} \right|_{x_c=0} \quad (7)''$$

Due to the absorption of part of the incident radiation at the surface of the skin simulant, equations for the boundary conditions in the air gap between the cloth and skin simulant have to be rewritten as follows:

(a) Between the back surface of the cloth and the air gap:

$$\left. -k_c \frac{\partial T}{\partial x_c} \right]_{x_c=L_c} = U_i \left(T_{x_c=L_c} - T_{x_s=0} \right) \quad (8)$$

(b) Between the air gap and the surface of the skin simulant:

$$I_0 e^{-\gamma L_c} - U_i \left(T_{x_s=0} - T_{x_c=L_c} \right) = -k_c \left. \frac{\partial T}{\partial x_s} \right]_{x_s=0} \quad (8)''$$

Before proceeding to solve the above system, it is useful to examine the Beer-Lambert Law graphically:

From eq. (1) $I_x = I_0 e^{-\gamma x_c}$ and

$$\ln \frac{I_x}{I_0} = -\gamma L_c \cdot \frac{x_c}{L_c}$$

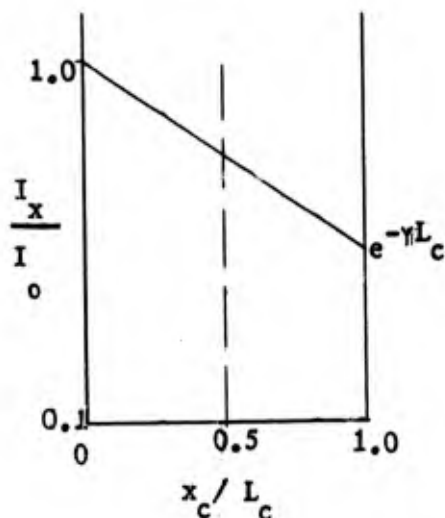


Figure 10.

Therefore, when $\ln I_x/I_0$ is plotted on semi-log paper against x_c/L_c , a straight line should result, with $I_x/I_0 = 1$ at $x_c/L_c = 0$, and $I_x/I_0 = e^{-\gamma L_c}$ at $x_c/L_c = 1$, as shown in Fig. 10. It should also be noted that $e^{-\gamma L_c}$ is equal to the fraction of the incident radiation absorbed at the

surface of the skin, i.e., the measured "transmittance" value divided by one minus the measured reflectance of the cloth or $e^{-\gamma L} = \tilde{T}/(1-R)$.

In using the finite-difference approximation method to solve this problem, it is assumed that the amount of radiation absorbed in the center of each finite increment or slice of cloth is distributed according to the above law. For cloth of high diathermancy, the incident radiation intensity as it travels through the cloth decreases nearly linearly with distance and thus justifies

this assumption; but for cloth of low diathermancy, the intensity of radiation drops sharply to a low value at the front part of the cloth. For the latter, it would be a better approximation to assume that the fraction of incident radiation that is absorbed in the first increment of the cloth layer is absorbed at the surface of the cloth (as in the case of an opaque cloth) rather than at the center of that increment.

With the above understanding, Equations (5)', (7)", (8) and (8)" can be re-written in dimensionless finite-increment terms, using the procedures outlined in Section III, to yield the following equations:

1. For the cloth layer:

$$\frac{\Delta_x (\Delta_x T^o)}{\Delta (X_c^o)^2} - \frac{\Delta_x e^{-\gamma L_c X_c^o}}{\Delta X_c^o} = \frac{\Delta_\theta T^o}{\Delta \theta_c^o} \quad (34)$$

2. For the skin simulant:

$$\frac{\Delta_x (\Delta_x T^o)}{\Delta (X_s^o)^2} = \frac{\Delta_\theta T^o}{\Delta \theta_s^o} \quad (10)$$

3. For the boundary condition at the front cloth surface:

(a) High diathermancy

$$\left. \frac{\Delta_x T^o}{\Delta X_c^o} \right]_{X_c^o = 0} = U_o^o \left(T_{X_c^o}^o = 0 - T_{\text{surr.}}^o \right) \quad (35)$$

(b) Low diathermancy

$$\left. \frac{\Delta_x T^o}{\Delta X_c^o} \right]_{X_c^o = 0} = - \frac{\frac{1}{U_o^o} \left(\Delta_x e^{-\gamma L_c X_c^o} \right) - \left(T_{X_c^o}^o = 0 - T_{\text{surr.}}^o \right)}{\frac{1}{U_o^o}} \quad (36)$$

and consequently, the equation for the first increment differs from equation (35) by the absence of the radiation term, i.e.,

$$\frac{\Delta_x (\Delta_x T^0)}{\Delta (X_c^0)^2} = \frac{\Delta_\theta T^0}{\Delta \theta_c^0} \quad (9)'$$

4. Boundary conditions in the air gap:

(a) Between the back surface of the cloth and the air gap.

$$\left. - \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 1} = U_i^0 \left(T_{X_c^0}^0 = 1 \quad -T_{X_s^0}^0 = 0 \right) \quad (12)'$$

(b) Between the air gap and the surface of the skin simulant

$$e^{-\gamma L} - U_i^0 \left(T_{X_s^0}^0 = 0 \quad -T_{X_c^0}^0 = 1 \right) = - \left. \frac{\Delta_x T^0}{\Delta X_s^0} \right]_{X_s^0 = 0}$$

$$\text{or } \left. \frac{\Delta_x T^0}{\Delta X_s^0} \right]_{X_s^0 = 0} = - \frac{\frac{1}{U_i^0} \left(e^{-\gamma L} \right) - \left(T_{X_s^0}^0 = 0 \quad -T_{X_c^0}^0 = 1 \right)}{\frac{1}{U_i^0}} \quad (37)$$

The numerical solution for this problem is exactly the same as that of an opaque cloth system, except that in the present case, the independent variable γL_c must also be fixed for each particular case. Using the same example as in Section III B, and assigning a value of 2.82 to γL_c (equivalent to $e^{-\gamma L_c} = .06$, as for 9-ounce Medium Gray Cotton Sateen), the above equations become:

$$T_{A,0}^0 = \frac{\Delta X_c^0 \left(\frac{1}{U_o^0} \Delta_x e^{-\gamma L_c X_c^0} - T_{B,0}^0 \right)}{\frac{1}{U_o^0} + \frac{\Delta X_c^0}{2}} + T_{B,0}^0 \quad (38)$$

$$T_{B,1}^0 = \frac{T_{A,0}^0 + T_{C,0}^0}{2} \tag{39}$$

$$T_{C,1}^0 = \frac{T_{B,0}^0 + T_{D,0}^0}{2} - \frac{\Delta X_c^0 e^{-\gamma L_c}}{2} \tag{40}$$

$$T_{D,0}^0 = \frac{(T_{F,0}^0 - T_{C,0}^0) \Delta X_c^0 + \frac{\Delta X_s^0 \Delta X_c^0 \cdot e^{-\gamma L_c}}{2}}{\frac{1}{U_i^0} + \frac{\Delta X_s^0}{2} + \frac{\Delta X_c^0}{2}} + T_{C,0}^0 \tag{41}$$

$$T_{E,0}^0 = \frac{(T_{F,0}^0 - T_{C,0}^0) \left(\frac{1}{U_i^0} + \frac{\Delta X_c^0}{2} - \frac{\Delta X_s^0}{2} \right) + \frac{\Delta X_c^0 \Delta X_s^0 \cdot e^{-\gamma L_c}}{2} + \frac{\Delta X_s^0 \cdot e^{-\gamma L_c}}{U_i^0}}{1} + T_{C,0}^0 \tag{42}$$

$$T_{F,1}^0 = \frac{T_{E,0}^0 + T_{G,0}^0}{2}, \text{ etc.}$$

To summarize, the algebraic equations used in numerical examples for both the opaque cloth and diathermanous cloth systems are as follows:

1. Size of cloth increment, $\Delta X_c^0 = 0.5$
2. Size of skin increment, $\Delta X_s^0 = 0.1$
3. $U_o^0 = 0.1, U_i^0 = 1.0,$
4. The ratio $\sqrt{\frac{(k\varphi)_s}{(k\varphi)_c}} = 5.0$
5. $e^{-\gamma L_c} = \infty$ for opaque cloth; $e^{-\gamma L_c} = 0.06,$ for diathermanous medium grey cloth.

6. Opaque Cloth System

$$T_{A,m}^{\circ} = .04878(10 - T_{B,m}^{\circ}) + T_{B,m}^{\circ}$$

$$T_{B,m+1}^{\circ} = 0.5(T_{A,m}^{\circ} + T_{C,m}^{\circ})$$

$$T_{C,m+1}^{\circ} = 0.5(T_{B,m}^{\circ} + T_{D,m}^{\circ})$$

$$T_{D,m}^{\circ} = .6154(T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ}$$

$$T_{E,m}^{\circ} = .0769(T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ}$$

$$T_{F,m+1}^{\circ} = 0.5(T_{E,m}^{\circ} + T_{G,m}^{\circ})$$

Diathermanous Cloth System

$$T_{A,m}^{\circ} = .04878(7.55 - T_{B,m}^{\circ}) + T_{B,m}^{\circ}$$

$$T_{B,m+1}^{\circ} = 0.5(T_{A,m}^{\circ} + T_{C,m}^{\circ})$$

$$T_{C,m+1}^{\circ} = 0.5(T_{B,m}^{\circ} + T_{D,m}^{\circ}) + .04625$$

$$\begin{aligned} T_{D,m}^{\circ} &= .3846(T_{F,m}^{\circ} - T_{C,m}^{\circ}) + T_{C,m}^{\circ} + .001154 \\ &= .6154(T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ} + .001154 \end{aligned}$$

$$\begin{aligned} T_{E,m}^{\circ} &= .9231(T_{F,m}^{\circ} - T_{C,m}^{\circ}) + T_{C,m}^{\circ} + .005769 \\ &= .0769(T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ} + .005769 \end{aligned}$$

$$T_{F,m+1}^{\circ} = 0.5(T_{E,m}^{\circ} + T_{G,m}^{\circ})$$

(43 a, b, c, d, e, f)

V. SIMPLIFIED GENERAL SOLUTION FOR SYSTEMS HAVING DIATHERMANOUS CLOTH

From the foregoing discussion, it is seen that the method just introduced would require a complete set of calculations for each value of γL_c . A closer comparison of the equations for the diathermanous cloth with that for the opaque cloth shows that the differences between these two sets of equations involve only the constant terms. Therefore, a general solution for any diathermanous cloth should be possible, if in each of a number of calculations it is assumed that a unit of incident radiant energy is absorbed in one particular increment of the cloth layer. Thus, if the cloth is divided into two increments, only three sets of calculations, and if the cloth is divided into six increments, a total of seven sets of calculations would be necessary. To

get the solution for a cloth having a particular ηL_c value, it is only necessary to sum up with proper weighting the solutions of each set of calculations; the weighting is determined by absorption distribution given by the Beer-Lambert Law.

The proposed "three curve" method is an example of this technique. The method requires three sets of solutions, called curves I, II, and III, which differ only in the assumption regarding the location of radiant-energy absorption. Curve I represents the case of a unit of incident radiant energy absorbed at the cloth surface. This is equivalent to the opaque-cloth solution. Curve II represents the case of incident radiant energy absorbed at $X_c^0 = 0.75$. Curve III represents the case of incident radiant energy absorbed at the surface of the skin simulant.

Thus it is seen that together with the solution for the opaque cloth system, only two additional sets of solutions are necessary for the general solution of all diathermanous cloth systems. The last two curves are calculated in exactly the same fashion as the first curve, except that in Curve II, the radiant energy is absorbed in the increment whose mid-plane lies on dimensionless depth $X_c^0 = 0.75$.

The basic equations in finite difference form become:

1. For the cloth layer,

(a) Any increment other than the increment associated with $X_c^0 = 0.75$,

$$\frac{\Delta_x (\Delta_x T^0)}{\Delta (X_c^0)^2} = \frac{\Delta \theta_c T^0}{\Delta \theta_c^0} \quad (9)$$

(b) For the cloth increment whose centerline is at $X_c^0 = 0.75$,

$$\frac{\Delta_x(\Delta_x T^0)}{\Delta(X_c^0)^2} + \frac{1}{\Delta X_c^0} = \frac{\Delta_\theta T}{\Delta \theta_c^0} \quad (44)$$

2. For the skin simulant

$$\frac{\Delta_x(\Delta_x T)}{\Delta_x(X_s^0)^2} = \frac{\Delta_\theta T}{\Delta \theta_s^0} \quad (10)'$$

3. Boundary conditions in the air gap,

$$-\left. \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 1} = U_i^0 \left(T_{X_c^0}^0 = 1 - T_{X_s^0}^0 = 0 \right) = -\left. \frac{\Delta_x T^0}{\Delta X_s^0} \right]_{X_s^0 = 0} \quad (12)'$$

Similarly in curve III, the radiant energy is absorbed at the surface of the skin simulant, and the basic equations are:

1. For the cloth layer,

$$\frac{\Delta_x(\Delta_x T^0)}{\Delta(X_c^0)^2} = \frac{\Delta_\theta T}{\Delta \theta_c^0} \quad (9)'$$

2. For the skin simulant,

$$\frac{\Delta_x(\Delta_x T^0)}{\Delta(X_s^0)^2} = \frac{\Delta_\theta T^0}{\Delta \theta_s^0} \quad (10)'$$

3. Boundary conditions at the front cloth surface,

$$\left. \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 0} = U_o^0 \left(T_{X_c^0}^0 = 0 \sim T_{\text{surr.}}^0 \right) \quad (35)$$

4. Boundary conditions in the air gap.

(a) Between the back surface of the cloth and the air gap,

$$-\left. \frac{\Delta_x T^0}{\Delta X_c^0} \right]_{X_c^0 = 1} = U_i^0 \left(T_{X_c^0}^0 = 1 - T_{X_s^0}^0 = 0 \right) \quad (12)'$$

(b) Between the air gap and the surface of the skin simulant

$$1 - U_i^0 \left(T_{X_s^0}^0 = 0 - T_{X_c^0}^0 = 1 \right) = - \left. \frac{\Delta_x T^0}{\Delta X_s^0} \right]_{X_s^0 = 0}$$

or

$$-\left. \frac{\Delta_x T^0}{\Delta X_s^0} \right]_{X_s^0 = 0} = \frac{\frac{1}{U_i^0} - \left(T_{X_s^0}^0 = 0 - T_{X_c^0}^0 = 1 \right)}{\frac{1}{U_i^0}} \quad (45)$$

Using the same material as before and comparing with equations (43a) to (43 f) inclusive, the equations for curve II are as follows:

$$T_{A,m}^0 = .9512 T_{B,m}^0$$

$$T_{B,m}^0 = 0.5 (T_{A,m}^0 + T_{C,m}^0)$$

$$T_{C,m}^0 + 1 = 0.5 (T_{B,m}^0 + T_{D,m}^0) + .2500 \quad (46 \text{ a, b, c, d, e, f})$$

$$T_{D,m}^0 = .6154 (T_{C,m}^0 - T_{F,m}^0) + T_{F,m}^0$$

$$T_{H,m}^0 = .0769 (T_{C,m}^0 - T_{F,m}^0) + T_{F,m}^0$$

$$T_{F,m}^0 + 1 = 0.5 (T_{H,m}^0 + T_{G,m}^0)$$

Similarly for curve III, the radiant energy is absorbed at the surface of the skin simulant, and the equations are:

$$T_{A,m}^{\circ} = .9512 T_{A,m}^{\circ}$$

$$T_{B,m+1}^{\circ} = 0.5 (T_{A,m}^{\circ} + T_{C,m}^{\circ})$$

$$T_{C,m+1}^{\circ} = 0.5 (T_{B,m}^{\circ} + T_{D,m}^{\circ}) \quad (47 \text{ a, b, c, d, e, f})$$

$$T_{D,m}^{\circ} = .6154 (T_{C,m}^{\circ} - T_{F,m}^{\circ}) + T_{F,m}^{\circ} + .01923$$

$$T_{E,m}^{\circ} = .0769 (T_{C,m}^{\circ} - T_{F,m}^{\circ}) + .09615 + T_{F,m}^{\circ}$$

$$T_{F,m+1}^{\circ} = 0.5 (T_{E,m}^{\circ} + T_{G,m}^{\circ})$$

Finally, the solution for any diathermanous-cloth problem can be obtained by summing up these three curves in their proper proportion. These proportionality constants can be obtained either graphically as shown in Fig. 10, or directly from the Beer-Lambert Law. Using the numerical example in which $e^{-\gamma L} = .06$, we have

$$T_{0.06}^{\circ} = 0.755 (T_I^{\circ}) + 0.185 (T_{II}^{\circ}) + 0.06 (T_{III}^{\circ}) \quad (48)$$

It can be shown that the solution obtained by this method is exactly the same as that calculated by the method described in Section IV.

For highly diathermanous cloth, as mentioned in Section IV, a better approximation would be based on the assumption that radiation is absorbed in the center of each cloth increment. This would seem to necessitate a new curve which can be called Curve I, in place of curve I, which represents surface absorption. However, a closer examination of the two approximations shows that

Curve I and Curve I' are really interrelated, and can be converted from one to the other. The conversion can be best explained by reviewing the basic equations used, as follows:

Figure 11 shows the three temperature planes being considered at the front part of the cloth:

For curve I, we have:



$$T_{A,0}^0 = \frac{\Delta X_c^0 \left(\frac{1}{U_0^0} - T_{B,0}^0 \right)}{\frac{1}{U_0^0} + \frac{\Delta X_c^0}{2}} + T_{B,0}^0$$

$$= \frac{\frac{\Delta X_c^0}{U_0^0} + \left(\frac{1}{U_0^0} - \frac{\Delta X_c^0}{2} \right) T_{B,0}^0}{\frac{1}{U_0^0} + \frac{\Delta X_c^0}{2}} \quad (49)$$

$$T_{B,1}^0 = \frac{T_{A,0}^0 + T_{C,0}^0}{2}$$

Combining these two equations, we obtain:

$$T_{B,1}^0 = \frac{\left(\frac{1}{U_0^0} - \frac{\Delta X_c^0}{2} \right) T_{B,0}^0}{2 \left(\frac{1}{U_0^0} + \frac{\Delta X_c^0}{2} \right)} + \frac{T_{C,0}^0}{2} + \frac{\frac{\Delta X_c^0}{2}}{2 + \frac{\Delta X_c^0}{U_0^0}} \quad (50)$$

For Curve I', we have,

$$T_{A,0}^0 = \frac{\left(\frac{1}{U_0^0} - \frac{\Delta X_c^0}{2} \right) T_{B,0}^0}{\frac{1}{U_0^0} + \frac{\Delta X_c^0}{2}} \quad (51)$$

$$T_{B,1}^0 = \frac{T_{A,0}^0 + T_{C,0}^0}{2} + \frac{\Delta X_c^0}{2}$$

Combining these two equations, we obtain,

$$T_{B,1}^0 = \frac{\left(\frac{1}{U_0^0} - \frac{\Delta X_c^0}{2}\right) T_{B,0}^0}{2\left(\frac{1}{U_0^0} + \frac{\Delta X_c^0}{2}\right)} + \frac{T_{C,0}^0}{2} + \frac{\Delta X_c^0}{2} \quad (52)$$

The factor for converting Curve I into Curve I' is thus simply the term

$\frac{2 + \Delta X_c^0 U_0^0}{2}$, by which the last term of equation (50) must be multiplied to yield the last term of equation (52). From this it is seen that the two approximation methods would be identical if U_0^0 were zero, as in the case of no heat losses from the cloth surface; furthermore, the smaller the cloth increment, the smaller the conversion factor.

VI PRESENTATION AND DISCUSSION OF RESULTS

A. Analytical Results.

With the aid of the IBM 704 computer the complete "3-curve" solutions covering the following values of the independent variables have been determined;

$$\sqrt{\frac{(kcp)_s}{(kcp)_c}} = 5.0$$

$$U_i^0 = 0.5, 1.0, 2.0, \text{ and } 5.0$$

$$U_0^0 = 0, 0.04, 0.1, \text{ and } 0.2.$$

Each U_i^0 value was matched successively with each U_o^0 value for each of the three curves which represent energy absorption at different planes in the system having diathermanous cloth; thus 48 sets of calculations were required. The results are presented in Figures 12 to 26, inclusive, as plots of dimensionless temperature rise, ΔT^0 vs. dimensionless time, $\Delta \theta_c^0$, with the third parameter on each curve being a particular combination of U_o^0 and U_i^0 . The curves extend up to 21.25 dimensionless time units and 1.05 dimensionless skin-depth units, which should make it possible with these data to solve problems of heat transfer through dry cloth to skin for radiation times ranging from 1 to 30 seconds.

Of the 15 final plots each of the first 14 represents the temperature rise at one of seven planes beneath the skin (or skin simulant) surface, for absorption of the radiant energy impulse at one of the first two arbitrarily chosen planes in the cloth-air-skin system. Because the curves for different combinations of U_o^0 and U_i^0 lie so close together some of them have not been drawn in but are instead represented by interpolation points. For Curve III the lines for various $U_o^0 - U_i^0$ combinations for different planes are sufficiently separated to permit plotting the data for all seven skin-simulant planes on one sheet, which is Figure 26. The solution of any given problem, as described in Section V, is obtained by weighting the temperature-rise values given by the three curves, in accordance with the Beer-Lambert Law and the known diathermancy of the cloth.

The chief advantage of using the "3-curve method" in obtaining a general solution for any diathermanous cloth system is its simplicity. It is granted that better accuracy could be obtained if the general solution were made up by use of a larger number of curves, each of which would represent absorption of radiant energy in a particular increment. However, the small gain in accuracy would not justify the large increase in effort required, and the purpose of get-

ting a relatively simple general solution would be defeated.

Thus the justification of such a method depends almost entirely on the accuracy of the solution as against those calculated by the more complicated method discussed in Section IV. Test solutions have therefore been obtained, using the 6-slice model which was shown in Section III, above, to be close to the "true" solution.

The test consists of the following calculations:

(1) "Rigorous" solutions were obtained for two specific γL values. One corresponds to the low diathermancy, medium-grey cloth with $e^{-\gamma L}_c = 6\%$, and one to the high diathermancy, white cloth with $e^{-\gamma L} = 42.8\%$. A 6-slice cloth model was used, and the procedure outlined in Section V was followed, assuming appropriate fractions of the radiation energy are absorbed in each of the six cloth slices and at the skin surface in accordance with the Beer-Lambert Law. The solution was carried out in 180 time increments corresponding to a dimensionless time value of 2.5.

(2) Solutions by the 3-curve method were then obtained, again using a six-slice model, with the radiation absorbed at the surface, at $X_c^0 = 0.75$, and at the skin surface, respectively. Proper proportioning of these three curves gave the final solutions for the two types of cloth. (When $e^{-\gamma L} = .06$, the proportioning of the 3 curves is, I:II:III = 0.7555:0.185:0.06; and when $e^{-\gamma L} = .428$, I:II:III = .347:.225:.428.)

Results obtained by the two procedures are given in Tables IV and V for the cases of low and high diathermancy, respectively. It was not considered necessary to carry the solution beyond a dimensionless time of 2.50, since the percentage difference between the two results decreases with increasing time. It is evident from the tabulation that the use of the three-curve method is amply justified by the good results it gives.

B. Experimental Verification of Analytical Work.

If the physical and thermal properties of the cloth samples used in experiments were accurately known, comparison of experimental and theoretical results would be relatively straightforward. However, the properties are not all well known. As pointed out in Quarterly Status Report No. 14 of this program, literary values for thermal conductivity of cotton fabrics vary considerably and none are presently available for the 9-ounce cotton sateen samples which have been supplied by the Quartermaster R & D Command for the experimental work. Even if the conductivity values as usually determined were available, however, it is questionable that they could properly be applied for experiments involving radiant as well as conductive heat transmission. This difficulty arises from the non-homogeneity of cloth; the measured thickness used in the conventional conductivity determination would not be the proper effective thickness for a case involving radiation. Because of the ambiguity in the determination of an effective thickness, L_c , it has been found necessary to combine this property with other properties in the correlations. It is believed that the thermal conductance, k_c/L_c , should be the same for all six of the cotton sateen samples, regardless of their optical properties, and the comparison of experimental with theoretical data has become, in effect, a procedure for determining this property by trial and error.

Since the density of the cloth also depends upon how the thickness, L_c , is measured or defined, the specification of thickness has been avoided by use of the product $(\rho L)_c$, the value of which can be nicely determined under the same conditions as those prevailing when the cloth is used in thermal radiation experiments. To determine the density-thickness product duplicate samples were stretched over 5-inch embroidery rings and then cut out along the inside rim of the rings with a razor blade. The discs were then dried and weighed. The area

included by the rings (which are not perfect circles) was determined by measuring with a planimeter the area of paper samples similarly stretched over the rings and cut out. The $(\rho L)_c$ product so determined for four cloth samples (of different colors) was $0.0273 \pm 2\%$ gm/cm². Literature values for the heat capacity of cotton fabrics vary between about 0.32 (5) and 0.35 (1); a value of 0.335 cal/gm°C has been used in this work. Thus the volumetric heat capacity $(\rho L)_c$, of the cloth is between 0.0088 and 0.0095, or $0.0092 \pm 8.7\%$ cal/sq. cm°C.

The physical and thermal properties of human skin are of course also variable. For the design of skin simulants used in experiments here the properties of average white skin agreed upon at an AFSWP conference (6) have been used. The agreed value for k_s is 1.065×10^{-3} cal/cm sec.°C, and that of $(k\rho)_s$ is 8.5×10^{-4} cal²/cm⁴ sec (°C)². While these values were used in its design, the performance of simulant No. 10 showed (by procedures previously reported (2)), that for this simulant the actual k_s is 1.035×10^{-3} and $(k\rho)_s$ is 8.65×10^{-4} .

We consider next the heat transfer coefficients U_o and U_i for surface heat loss and for transmission through the gap between cloth and simulant, respectively. These appear in Figures 12 to 26 in the non-dimensional forms, $U_o^0 = U_o L_c/k_c$ and $U_i^0 = U_i L_c/k_c$. The value of U_o can be fairly well estimated from knowledge of heat transmission by natural convection. The value of U_i is more difficult to determine, especially for the experiments in which the cloth and skin simulant are in contact, so that the effective gap width is not measurable. In the procedure to be followed for comparing experimental and analytical results, (the determination of $(k/L)_c$ by trial), two courses are possible in the handling of these two heat transfer coefficients. The simpler way is to let U_o^0 and U_i^0 remain constant with each selection of $(k/L)_c$. Since this implies, unrealistically, that the coefficients U_o and U_i are variables, the more rigorous

procedure of keeping U_0 and U_i fixed while letting U_0^0 and U_i^0 vary will be followed. This requires interpolation to new values on the T^0 vs. θ_s^0 charts with each new selection of $(k/L)_c$.

As an example of the procedure, a comparison will be made between experimental and theoretical temperature-time-depth data within skin simulant No. 10 upon its exposure to thermal radiation through an overlying and contacting layer of 9-ounce cotton sateen designated as light gray, or QM5. The intensity of test exposures ranged from 0.5 to 4.0 cal/sq.cm.sec. and the duration from 12 to 4 seconds. As measured by the Naval Material Laboratory (7), the transmittance, T , and reflectance, R , determined by integration over the solar spectral range of 0.4μ to 2.4μ , were 7.4% and 56.0%, respectively, for the light gray sateen. Thus the fraction of unreflected radiant energy which passes through the back surface of the cloth, due to its diathermancy, is $e^{-\gamma L} = T/(1-R) = 0.1682$. This latter value requires that T^0 values read from Figures 12 through 26 must be weighted in the ratio $0.5899:0.2419:0.1682$ for Curve I, Curve II and Curve III data. While the T^0 values can be read from the charts for any cloth-simulant systems, for the Q.M. cloth samples specifically, these values have been IBM-calculated with the weighting factors for diathermancy included. Copies of these tabulated data can be obtained by interested parties by arrangement with the M.I.T. Fuels Research Laboratory.

The simplest approach to comparing experimental and theoretical data is to make the first choice of $(k/L)_c$ be $1/266$, the value which corresponds to a value of 5.0 for R , the square root of the ratio $(k\rho)_s / (k\rho)_c$. Having made this choice, the first step is to plot values of T^0 vs. X_s^0 for several values of dimensionless time, θ_s^0 . The second step is to determine values of X_s^0 corresponding to the thermocouple locations in Skin Simulant No. 10.

Since $X_s^0 = \frac{x_s}{k_s} \cdot \frac{k_c}{L_c}$, this transformation is simply as tabulated below:

Thermocouple Number	x_s	$\frac{x_s}{k_s}$	$X_s^0 = \frac{x_s}{k_s} \cdot \frac{1}{266}$
1	0.029 cm	28.0	0.1053
2	0.037	35.7	0.1342
4	0.068	65.7	0.2470
5	0.130	125.6	0.4720

For the X_s^0 locations listed above, values of T^0 are read and tabulated with the corresponding θ_s^0 values.

The third step is to transform T^0 and θ_s^0 values into the form $\Delta T U_o / I_o$ and $\Delta \theta U_o^2 / (kcp)_s$ which eliminates $(k/L)_c$. The transformation for temperature is simply:

$$\frac{\Delta T U_o}{I_o} = \frac{\Delta T k_c}{I_o L_c} \cdot \frac{U_o L_c}{k_c} = \frac{\Delta T U_o^0}{I_o};$$

and in our present example $\frac{\Delta T U_o^0}{I_o} = (0,1) \Delta T^0$.

The conversion for time is:

$$\frac{\Delta \theta U_o^2}{(kcp)_s} = \frac{\Delta \theta k_c}{(cp)_c L_c^2} \cdot \frac{(kcp)_c}{(kcp)_s} \cdot \left(\frac{U_o L_c}{k_c} \right)^2 = \theta_c^0 \left(\frac{1}{R} \right) \left(U_o^0 \right)^2,$$

which in this case is

$$\theta_c^0 \left(\frac{1}{25} \right) \left(0.01 \right) = 0.0004 \theta_c^0$$

Step four of the procedure is to plot the new values of $\Delta T U_o / I_o$ vs. $\Delta \theta U_o^2 / (kcp)_s$, for comparison with the experimental data. This comparison is conveniently made by plotting the experimental data as $\Delta T / I_o$, $^{\circ}\text{C sq cm sec/cal}$, vs. θ , seconds, on a sheet of transparent vellum which can be laid over the

theoretical curves. Alignment of the two curves along the y-axis is made by matching $\Delta T/I_0$ equal to unity for the experimental data with the point where the theoretical $\Delta TU_0/I_0$ is equal to U_0 , or $0.1/266 = 3.76 \times 10^{-4}$. Alignment along the x-axis is made by matching the experimental time of 1.0 second with the theoretical-data value of $U_0^2/(kcp)_s$, or $(3.76 \times 10^{-4})^2/8.65 \times 10^{-4} = 1.633 \times 10^{-4}$.

A plot of $\Delta T U_0/I_0$ vs. $\Delta \theta U_0^2/(kcp)_s$ is shown in Fig. 27, for the four thermocouples of Simulant No. 10. Superimposed on or near the curves are short vertical lines which represent the experimental data, the length of each of these lines indicating the estimated precision of the measurement. In order to test the assumption regarding $(k/L)_c$ and to determine how sensitive the correlation may be to this parameter, it is necessary to go through the procedure with another value of $(k/L)_c$.

For this second trial, $(k/L)_c$ is arbitrarily chosen as 1/240, consequently,

$$\begin{aligned} (kcp)_c &= (cpL)_c \cdot \left(\frac{k}{L}\right)_c \\ &= (0.0092 \pm 8.7\%) \left(\frac{1}{240}\right) = 3.836 \times 10^{-5} \pm 8.7\%, \end{aligned}$$

and

$$\begin{aligned} R &= \sqrt{(kcp)_s / (kcp)_c} = \sqrt{8.65 \times 10^{-4} / 3.836 \times 10^{-5}} \\ &= 4.75 \pm 2.95\% \end{aligned}$$

Since the theoretical curves are based on $R = 5.0$, values of T^0 read therefrom should be increased by a factor,

$$\left(\frac{5.0}{4.75}\right)^{0.89}, \text{ which has been found empirically to be a satis-}$$

factory way to correlate calculations based on different "R" values. (This transformation makes it possible to use a single value of R for all theoretical curves, rather than a number of values, as are required for the parameters U_0^0 and U_i^0 .) Also from the new value of $(k/L)_c$, we find

$$U_0^0 = 0.1 \frac{240}{266} = 0.0902$$

$$\text{and } U_i^0 = 2.0 \frac{240}{266} = 1.802.$$

To make up a new plot of T^0 vs. X_s^0 , corresponding to these new heat transfer parameters, it is necessary to perform a double interpolation, using T^0 data for U_0^0 values of 0.04 and 0.10 and U_i^0 values of 1.0 and 2.0. The 10% reduction in U_0^0 makes a small upward change in T^0 values for dimensionless times of 2.0 and more; for shorter times the effect is nearly negligible. The decrease in T^0 caused by a 10% decrease in U_i^0 is significant for all the dimensionless times of this example (i.e. to 6.0). Finally, the effect of the correction for change in the ratio R against increases T^0 , so that the net effect of all three factors is an increase in T^0 which is negligible for short time intervals and increasingly important for longer times.

The depth parameter, X_s^0 , is also affected by changes in $(k/L)_c$ and the consequent change in R . Since

$$\frac{\Delta X_c^0}{\Delta X_s^0} = \sqrt{\frac{(k\varphi)_s}{(k\varphi)_c}} = R,$$

and ΔX_c^0 is fixed at 0.5, a change of R from 5.0 to 4.75 causes ΔX_s^0 to be changed to 0.1053. For this new case, therefore, the positions of planes F, G, H, etc. in the skin simulant are made deeper by the ratio, 0.1053/0.1. In other words, the dimensionless temperature values calculated and plotted for the case of $R = 5$, now apply for greater depths, because $R = 4.75$.

Again, the position of skin-simulant thermocouples in terms of X_s^0 are also changed when $(k/L)_c$ is changed. The new depths are as follows:

Thermocouple	1	2	4	5
x_s/k_s	28.0	35.7	65.7	125.6
$X_s^0 = \frac{x_s}{k_s} \frac{1}{240}$	0.1167	0.1487	0.274	0.523

The next step is to read values of T^0 from the new T^0 vs. X_s^0 plot, tabulate them and make a conversion of the dimensionless time values to those from which $(k/L)_c$ is eliminated. The conversion this time is:

$$\begin{aligned} \frac{\Delta \theta U_0^2}{(k\varphi)_s} &= \left(\frac{4\theta k_c}{(\varphi)_c L_c^2} \right) \frac{(k\varphi)_c}{(k\varphi)_s} \left(\frac{U_0 L_c}{k_c} \right)^2 \\ &= \theta_c^0 \left(\frac{1}{R} \right)^2 \left(U_0^0 \right)^2 \\ &= \left(\frac{1}{22.55} \right) (0.0081)^2 \theta_c^0 = 3.59 \times 10^{-4} \theta_c^0 \end{aligned}$$

Finally, a plot can be made of the theoretical data, corrected and converted as required by the choice of $(k/L)_c = 1/240$. The results are represented by the dashed-line curves of Figure 27. In the example chosen the dependence of results on $(k/L)_c$ is so small as to be barely perceptible when this parameter is changed by 10%. The reason that thermal conductance of the cloth is of so little importance is that a sizable fraction of the incident radiant energy passes through without dependence on conductance. Thus the temperature rise in skin behind a diathermanous cloth covering is rapid regardless of the conductance of the cloth. In this comparison of the experimental and theoretical data, the light gray cloth was chosen simply because the case of a diathermanous cloth is a more exacting test of the theoretical analysis. The comparison procedure is, of course, the same regardless of cloth type. The results shown in Figure 27 indicate that agreement between the experimental and theoretical work is remarkably good. The agreement for relatively large values of dimensionless time, however, is not as good as desired, but could be improved by choice of a somewhat higher value for U_0 . It is inevitable that a single comparison is not adequate for showing both (a) the applicability of the theory to cases involving use of diathermanous cloth

and (b) a nice determination of $(k/L)_c$. Determination of $(k/L)_c$ was not the original purpose of this work. Rather the aim was simply to learn whether heat transfer in such a complex multi-component system could be adequately treated by theoretical analysis of a simplified model. This has been conclusively demonstrated, not only for opaque cloth systems but also for diathermanous cloth systems. In addition, application of the analysis to opaque cloth systems offers a good way of determining the effective value of thermal conductance of cloth $(k/L)_c$, when the heat source is radiant thermal energy. Results to date show that values of 1/260 to 1/270 are good for the cotton sateen samples.

VIII CONCLUSIONS

1. An analytical study has been made of heat transmission through dry cloth and air space to underlying skin, following exposure of the system to radiant thermal energy.
2. Both the simple case involving absorption of all the radiant energy at the radiated surface of opaque cloth and the more complicated case in which some of the energy is transmitted through diathermanous cloth can be satisfactorily solved numerically by finite-difference approximation.
3. For systems involving cloth of known diathermancy a technique which requires the use of only three curves, each representing absorption at a different plane in the system, gives results (in terms of temperature rise) within 1 to 2% of those obtained by machine computation for two specific cases wherein allowance was made for energy absorption at each of six slices in the cloth. These two cases represented cloths of very high and very low diathermancy, thus permitting the conclusion that use of the 3-curve method is a satisfactory substitute for machine computation of specific problems.

4. The curves necessary for solution of any problem involving heat transmission through dry cloth and an air space to skin have been determined by machine computation for the range of exposure times up to about 30 seconds.

5. Adequacy of the analysis and computation procedures have been verified by good agreement with the experimental results obtained by exposure of cloth-air-skin simulant systems to thermal radiation.

IX NOMENCLATURE

c	Heat capacity, cal/gm $^{\circ}\text{C}$
I	Intensity of incident radiation, cal/cm ² -sec.
I_0	Intensity of unreflected radiation, cal/cm ² -sec.
k	Thermal conductivity, cal/cm-sec. $^{\circ}\text{C}$
L	Thickness of cloth, cm
M	Modulus, $(X_c^0)^2 / \Delta \theta_c^0$
R	Ratio = $\Delta X_c^0 / \Delta X_s^0 = \sqrt{(kcp)_s / (kcp)_c}$
R	Reflectance of cloth
T	Temperature, $^{\circ}\text{C}$
U	Overall heat transfer coefficient, cal/sq. cm.,-sec. $^{\circ}\text{C}$
x, X	Depth, cm

Greek

γ	Extinction coefficient, cm ⁻¹
Δ	Finite increment
θ	Time, sec.
ρ	Density, gm/cm ³
τ	Transmittance of cloth

Subscripts

c	Cloth
i	Air gap between cloth and skin simulant
o	Cloth surface
s	Skin simulant
A, B, ... n, n+1	Depth planes
0, 1.... m, m+1	Time increments

Superscript

o	Dimensionless group.
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TABLE I.
COMPARISON OF 6-SLICE MODEL WITH 2-SLICE MODEL

TEMPERATURE PROFILE IN OPAQUE CLOTH

$\Delta\theta_c^\circ$	x_c°								
	-1/4	+1/4	3/4	1 1/4	2-slice 6-slice % diff.	2-slice 6-slice % diff.			
.2500	.8566	.3688	.3493	5.6	.1238	.1202	3.0	.0969	-
.5000	1.0873	.6042	.5898	2.4	.3365	.3256	3.3	.2647	-
.7500	1.2926	.8136	.7994	1.8	.5271	.5153	2.3	.4161	-
1.000	1.4761	1.0009	.9865	1.5	.6979	.6859	1.8	.5524	-
1.250	1.6405	1.1686	1.1543	1.2	.8512	.8391	2.4	.6753	-
1.500	1.7880	1.3191	1.3051	1.1	.9890	.9770	1.2	.7863	-
1.7500	1.9206	1.4544	1.4407	0.9	1.1131	1.1013	1.1	.8867	-
2.0000	2.0401	1.5762	1.5629	0.8	1.2251	1.2135	1.0	.9776	-
2.2500	2.1478	1.6861	1.6733	0.8	1.3262	1.3150	0.9	1.0602	-
2.5000	2.2451	1.7854	1.7731	0.7	1.4178	1.4070	0.8	1.1352	1.1271*

Note: R = 5.0 $\Delta x_c^\circ = 1/2$ and $1/6$

$U_o^\circ = .04$ * extrapolated value

$U_i^\circ = 0.5$

TABLE II.
COMPARISON OF 6-SLICE MODEL WITH 2-SLICE MODEL:
TEMPERATURE PROFILE INSIDE SKIN SIMULANT

$\Delta\theta_c^\circ$.05		.15		.25		.35		.55	
	2 slice	6 slice % diff.	2 slice	6 slice % diff.	2 slice	6 slice % diff.	2 slice	6 slice % diff.	2 slice	6 slice % diff.
.2500	0	.0009 -	0	.0001 -	0	0 -	0	0 -	0	0 -
.5000	.0063	.0068 - 7.4	.0013	.0019 -31.6	0	.0004 -	0	.0001 -	0	0 -
.7500	.0163	.0164 - 0.6	.0058	.0063 - 7.9	.0016	.0021 -23.8	.0003	.0006 -50.0	0	0 -
1.000	.0283	.0283 0	.0126	.0130 - 3.1	.0049	.0054 - 9.3	.0016	.0020 -20.0	.0001	.0002 -50.0
1.2500	.0417	.0416 + 0.2	.0211	.0213 - 0.9	.0096	.0101 - 4.9	.0040	.0045 -11.1	.0004	.0007 -42.9
1.5000	.0565	.0559 + 1.1	.0309	.0309 - 0	.0157	.0161 - 2.5	.0074	.0079 - 6.3	.0012	.0015 -20.0
1.7500	.0715	.0708 + 1.0	.0415	.0414 + 0.2	.0227	.0231 - 1.7	.0117	.0122 - 4.1	.0025	.0029 -13.8
2.000	.0869	.0861 + 0.9	.0529	.0527 + 0.4	.0306	.0309 - 1.0	.0169	.0173 - 2.3	.0043	.0047 - 8.5
2.2500	.1024	.1015 + 0.9	.0647	.0644 + 0.5	.0392	.0393 - 0.3	.0227	.0231 - 1.7	.0066	.0070 - 5.7
2.5000	.1179	.1170 + 0.8	.0770	.0765 + 0.7	.0483	.0483 0	.0292	.0295 - 1.0	.0093	.0098 - 5.1

Note: Numbers in body of table indicate T°

TABLE IV.

DIMENSIONLESS TEMPERATURE RISE IN LOW-DIATHERMANCY
 CLOTH ($e^{-\gamma L} = 0.06$) AND UNDERLYING SKIN: COMPARISON
 OF SIX-SLICE "EXACT" AND "THREE-CURVE" SOLUTIONS.

$U_o^0 = 0.04, \quad U_i^0 = 0.5, \quad R = 5.0$

$\Delta \theta_c^0$	$X_c^0 = 0.25$		$X_c^0 = 0.75$		$X_s^0 = 0.05$		0.15		0.25	
	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve
0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
0.1250	0.1584	0.1635	0.0664	0.0675	0.0027	0.0027	0.0004	0.0004	0.0000	0.0000
0.2500	0.2905	0.2912	0.1575	0.1630	0.0059	0.0060	0.0015	0.0015	0.0003	0.0003
0.3750	0.4047	0.4047	0.2555	0.2614	0.0099	0.0103	0.0032	0.0033	0.0009	0.0009
0.5000	0.5109	0.5104	0.3503	0.3563	0.0148	0.0153	0.0056	0.0058	0.0018	0.0019
0.6250	0.6106	0.6099	0.4408	0.4467	0.0203	0.0209	0.0085	0.0087	0.0032	0.0033
0.7500	0.7049	0.7040	0.5266	0.5324	0.0264	0.0271	0.0119	0.0122	0.0049	0.0050
0.8750	0.7941	0.7931	0.6080	0.6136	0.0328	0.0336	0.0158	0.0162	0.0069	0.0071
1.0000	0.8786	0.8774	0.6852	0.6907	0.0396	0.0404	0.0200	0.0205	0.0094	0.0096
1.1250	0.9587	0.9574	0.7584	0.7638	0.0467	0.0475	0.0246	0.0251	0.0121	0.0124
1.2500	1.0346	1.0332	0.8278	0.8331	0.0539	0.0548	0.0295	0.0300	0.0152	0.0155
1.3750	1.1066	1.1050	0.8937	0.8990	0.0613	0.0623	0.0346	0.0352	0.0185	0.0188
1.5000	1.1749	1.1733	0.9563	0.9615	0.0689	0.0699	0.0400	0.0406	0.0220	0.0224
1.6250	1.2397	1.2380	1.0158	1.0209	0.0765	0.0776	0.0455	0.0462	0.0258	0.0262
1.7500	1.3013	1.2995	1.0723	1.0773	0.0842	0.0854	0.0512	0.0520	0.0298	0.0303
1.8750	1.3598	1.3579	1.1261	1.1310	0.0920	0.0932	0.0570	0.0578	0.0339	0.0344
2.0000	1.4153	1.4134	1.1772	1.1821	0.0998	0.1010	0.0630	0.0638	0.0382	0.0388
2.1250	1.4682	1.4662	1.2259	1.2307	0.1076	0.1089	0.0690	0.0699	0.0427	0.0433
2.2500	1.5185	1.5164	1.2722	1.2770	0.1155	0.1167	0.0752	0.0761	0.0472	0.0479
2.3750	1.5663	1.5642	1.3163	1.3211	0.1233	0.1246	0.0814	0.0823	0.0519	0.0526
2.5000	1.6118	1.6097	1.3583	1.3631	0.1311	0.1324	0.0876	0.0886	0.0567	0.0574

θ_c^0	$X_s^0 = 0.35$		0.45		0.55		0.65	
	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve
0.	0.	0.	0.	0.	0.	0.	0.	0.
0.1250	0.	0.	0.	0.	0.	0.	0.	0.
0.2500	0.0000	0.0000	0.0000	0.	0.	0.	0.	0.
0.3750	0.0002	0.0002	0.0000	0.0000	0.0000	0.0000	0.0000	0.
0.5000	0.0005	0.0005	0.0001	0.0001	0.0000	0.0000	0.0000	0.
0.6250	0.0010	0.0011	0.0003	0.0003	0.0001	0.0001	0.0000	0.0000
0.7500	0.0018	0.0018	0.0006	0.0006	0.0002	0.0002	0.0000	0.0000
0.8750	0.0028	0.0029	0.0010	0.0011	0.0003	0.0003	0.0001	0.0001
1.0000	0.0041	0.0042	0.0016	0.0017	0.0006	0.0006	0.0002	0.0002
1.1250	0.0056	0.0057	0.0024	0.0024	0.0009	0.0010	0.0003	0.0003
1.2500	0.0073	0.0075	0.0033	0.0034	0.0014	0.0014	0.0005	0.0006
1.3750	0.0093	0.0095	0.0044	0.0045	0.0020	0.0020	0.0008	0.0008
1.5000	0.0115	0.0117	0.0057	0.0058	0.0026	0.0027	0.0012	0.0012
1.6250	0.0139	0.0142	0.0071	0.0073	0.0035	0.0035	0.0016	0.0016
1.7500	0.0165	0.0168	0.0087	0.0089	0.0044	0.0045	0.0021	0.0021
1.8750	0.0193	0.0196	0.0105	0.0107	0.0055	0.0056	0.0027	0.0028
2.0000	0.0222	0.0226	0.0124	0.0126	0.0066	0.0068	0.0034	0.0035
2.1250	0.0254	0.0258	0.0145	0.0147	0.0079	0.0081	0.0042	0.0043
2.2500	0.0286	0.0290	0.0167	0.0170	0.0094	0.0095	0.0050	0.0051
2.3750	0.0320	0.0325	0.0190	0.0193	0.0109	0.0111	0.0060	0.0061
2.5000	0.0355	0.0360	0.0215	0.0218	0.0126	0.0128	0.0071	0.0072

TABLE V.

DIMENSIONLESS TEMPERATURE RISE IN HIGH-DIATHERMANCY CLOTH ($e^{-\gamma L} = 0.428$) AND UNDERLYING SKIN: COMPARISON OF SIX-SLICE "EXACT" AND "THREE-CURVE" SOLUTIONS.

$U_0^0 = 0.04,$

$U_i^0 = 0.5,$

$R = 5.0$

$\Delta\theta_c^0$	$X_c^0 = 0.25$		$X_c^0 = 0.75$		$X_s^0 = 0.05$		0.15		0.25	
	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve
0.	0.	0.	0.	0.	0.	0.	0.	0.	0.	0.
0.1250	0.0793	0.0814	0.0651	0.0612	0.0172	0.0172	0.0025	0.0025	0.0001	0.0001
0.2500	0.1552	0.1543	0.1267	0.1256	0.0308	0.0310	0.0090	0.0091	0.0018	0.0018
0.3750	0.2241	0.2228	0.1893	0.1882	0.0424	0.0427	0.0164	0.0165	0.0049	0.0050
0.5000	0.2895	0.2878	0.2492	0.2482	0.0532	0.0536	0.0237	0.0239	0.0089	0.0090
0.6250	0.3515	0.3497	0.3065	0.3053	0.0634	0.0639	0.0311	0.0313	0.0133	0.0134
0.7500	0.4106	0.4086	0.3610	0.3597	0.0732	0.0738	0.0383	0.0386	0.0181	0.0182
0.8750	0.4669	0.4648	0.4130	0.4115	0.0828	0.0834	0.0456	0.0460	0.0230	0.0232
1.0000	0.5205	0.5182	0.4625	0.4609	0.0922	0.0928	0.0529	0.0533	0.0281	0.0283
1.1250	0.5716	0.5692	0.5097	0.5080	0.1013	0.1021	0.0601	0.0606	0.0333	0.0336
1.2500	0.6203	0.6177	0.5547	0.5529	0.1104	0.1112	0.0673	0.0678	0.0387	0.0390
1.3750	0.6667	0.6640	0.5977	0.5958	0.1193	0.1201	0.0745	0.0751	0.0441	0.0444
1.5000	0.7109	0.7081	0.6387	0.6367	0.1281	0.1290	0.0817	0.0823	0.0496	0.0499
1.6250	0.7531	0.7502	0.6778	0.6757	0.1368	0.1377	0.0889	0.0894	0.0551	0.0555
1.7500	0.7934	0.7904	0.7151	0.7130	0.1454	0.1463	0.0960	0.0966	0.0607	0.0611
1.8750	0.8318	0.8288	0.7508	0.7486	0.1539	0.1548	0.1031	0.1037	0.0664	0.0668
2.0000	0.8685	0.8654	0.7849	0.7827	0.1623	0.1632	0.1101	0.1108	0.0720	0.0725
2.1250	0.9036	0.9004	0.8176	0.8152	0.1706	0.1716	0.1172	0.1179	0.0777	0.0782
2.2500	0.9371	0.9339	0.8488	0.8464	0.1788	0.1798	0.1241	0.1249	0.0834	0.0839
2.3750	0.9692	0.9659	0.8786	0.8762	0.1869	0.1879	0.1311	0.1318	0.0892	0.0897
2.5000	0.9998	0.9965	0.9072	0.9047	0.1949	0.1959	0.1380	0.1388	0.0949	0.0955
	0.35		0.45		0.55		0.65			
θ_c^0	Exact	3-curve	Exact	3-curve	Exact	3-curve	Exact	3-curve		
0.	0.	0	0.	0.	0.	0.	0.	0.		
0.1250	0.	0	0.	0.	0.	0.	0.	0.		
0.2500	0.0002	0.0002	0.0000	0.0000	0.	0.	0.0000	0.		
0.3750	0.0011	0.0011	0.0002	0.0002	0.0000	0.0000	0.0000	0.		
0.5000	0.0027	0.0028	0.0007	0.0007	0.0001	0.0001	0.0000	0.0000		
0.6250	0.0050	0.0050	0.0015	0.0016	0.0004	0.0004	0.0001	0.0001		
0.7500	0.0075	0.0076	0.0028	0.0028	0.0009	0.0009	0.0002	0.0002		
0.8750	0.0105	0.0106	0.0043	0.0043	0.0016	0.0016	0.0005	0.0005		
1.0000	0.0137	0.0138	0.0061	0.0062	0.0025	0.0025	0.0009	0.0009		
1.1250	0.0172	0.0173	0.0082	0.0082	0.0036	0.0036	0.0014	0.0014		
1.2500	0.0208	0.0209	0.0104	0.0105	0.0048	0.0049	0.0021	0.0021		
1.3750	0.0246	0.0248	0.0129	0.0130	0.0063	0.0064	0.0029	0.0029		
1.5000	0.0285	0.0287	0.0155	0.0156	0.0079	0.0080	0.0038	0.0038		
1.6250	0.0326	0.0328	0.0183	0.0184	0.0097	0.0098	0.0048	0.0049		
1.7500	0.0367	0.0370	0.0212	0.0213	0.0116	0.0117	0.0060	0.0061		
1.8750	0.0410	0.0413	0.0242	0.0244	0.0136	0.0137	0.0073	0.0074		
2.0000	0.0453	0.0456	0.0273	0.0275	0.0158	0.0159	0.0087	0.0088		
2.1250	0.0497	0.0500	0.0306	0.0308	0.0181	0.0182	0.0102	0.0103		
2.2500	0.0542	0.0545	0.0339	0.0341	0.0204	0.0206	0.0118	0.0119		
2.3750	0.0587	0.0591	0.0373	0.0376	0.0229	0.0230	0.0135	0.0136		
2.5000	0.0632	0.0636	0.0408	0.0411	0.0254	0.0256	0.0153	0.0154		

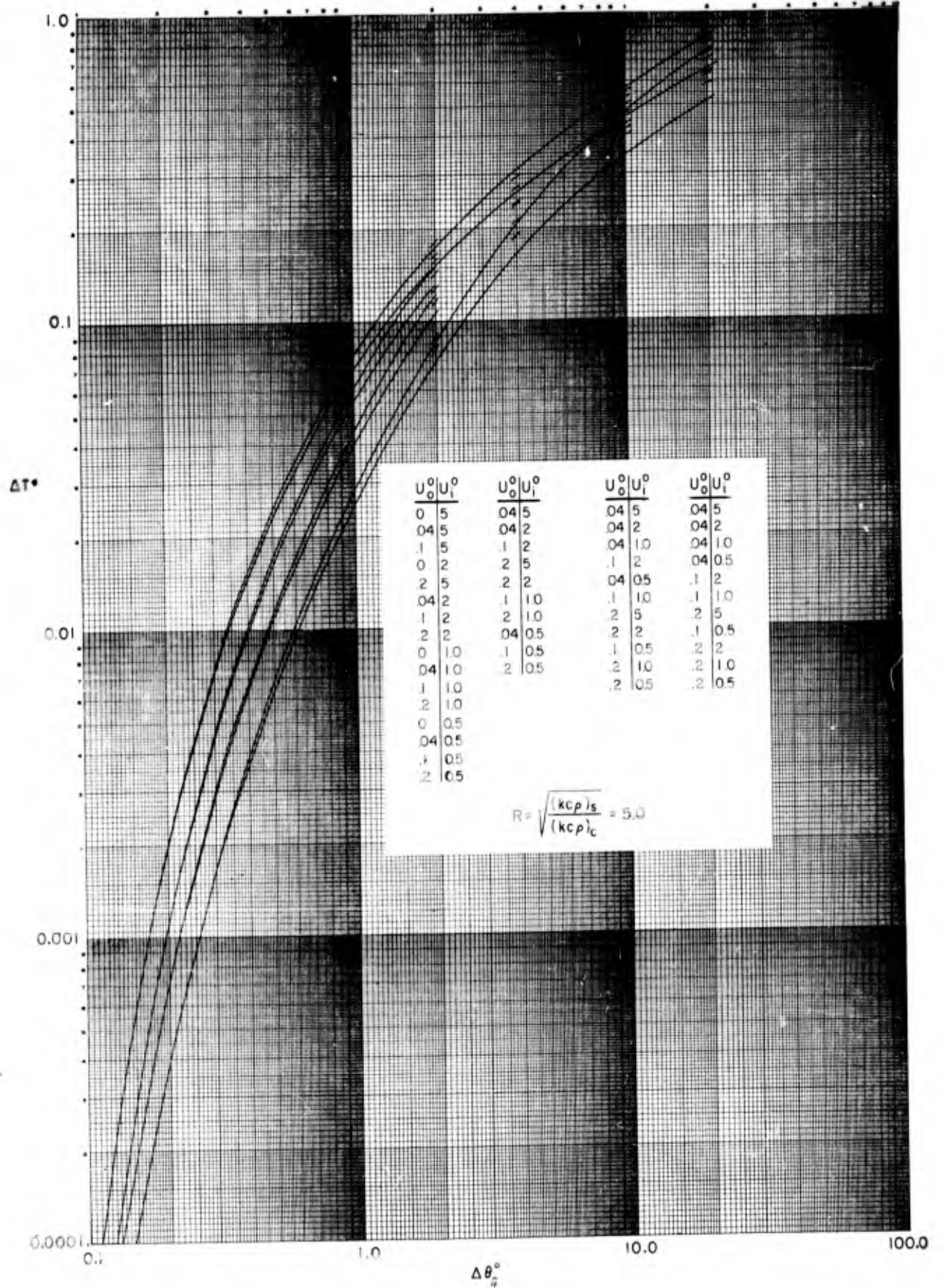


FIG.12 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.05$
 (Curve I - All radiation absorbed at $X_c^0 = 0$)

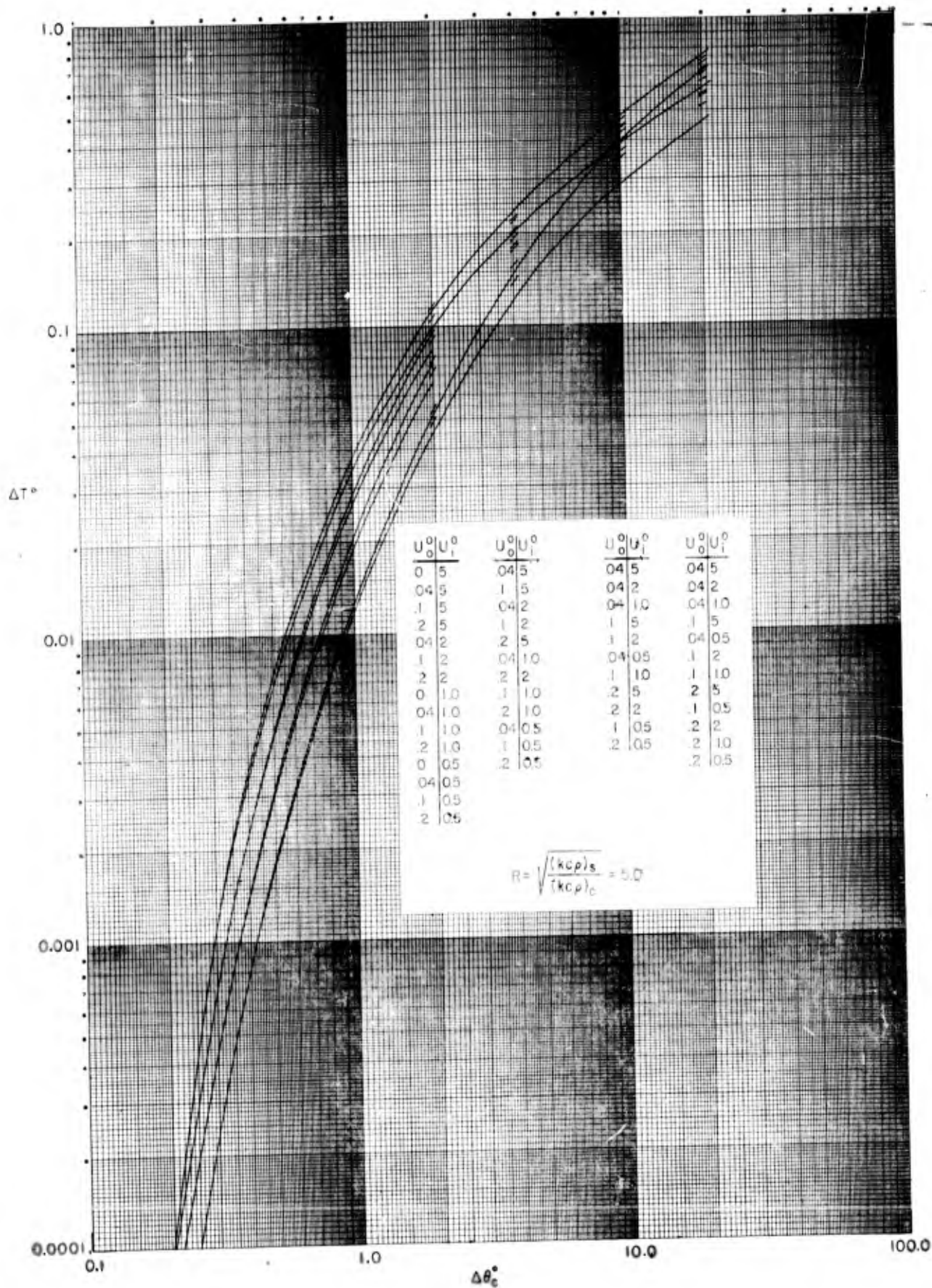


FIG.13- TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.15$
 (Curve I- All radiation absorbed at $X_c^0 = 0$)

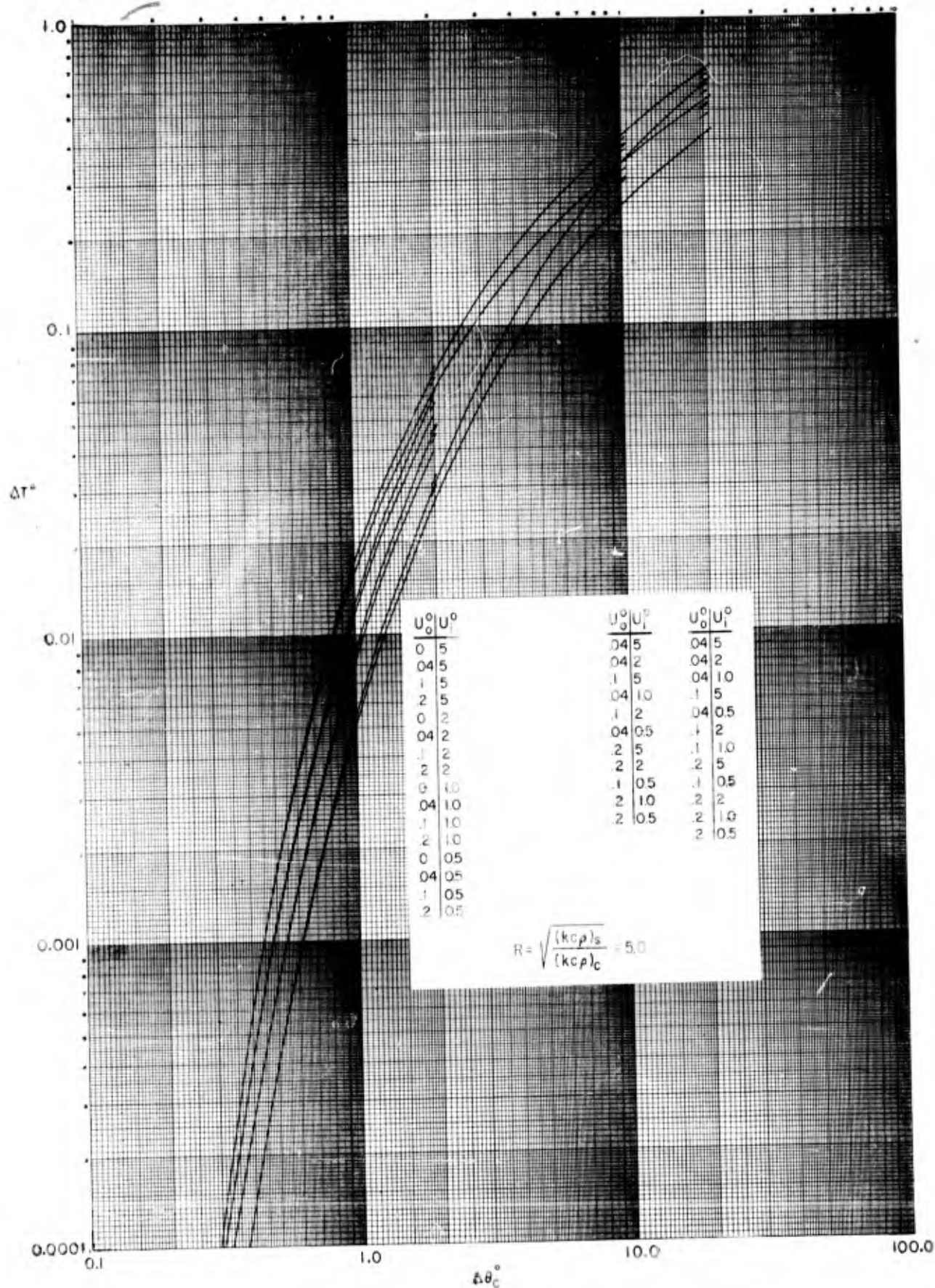


FIG. 14 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_S^\circ = 0.25$
 (Curve I - All radiation absorbed at $X_C^\circ = 0$)

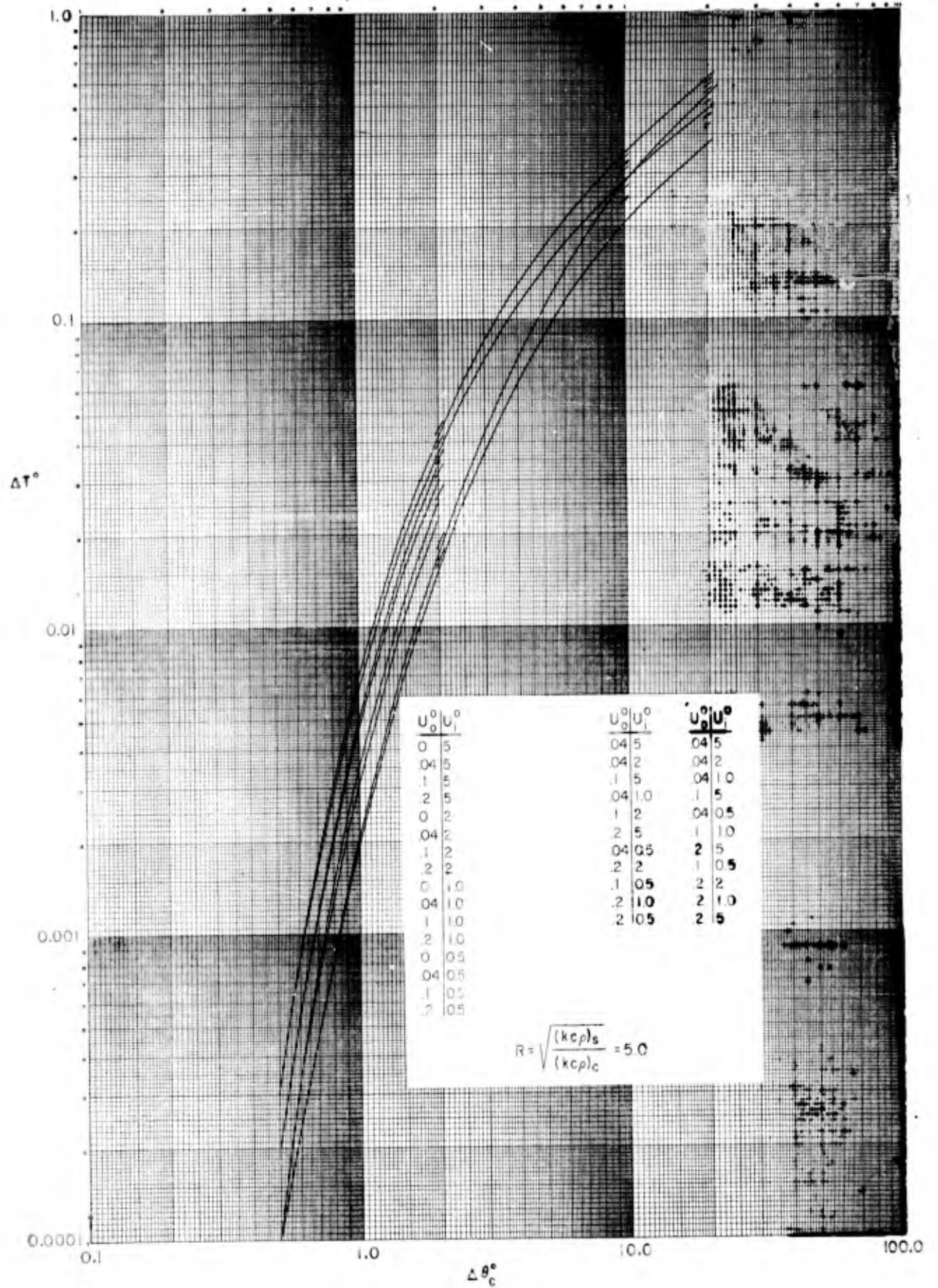


FIG. 15 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.35$
 (Curve I - All radiation absorbed at $X_c^0 = 0$)

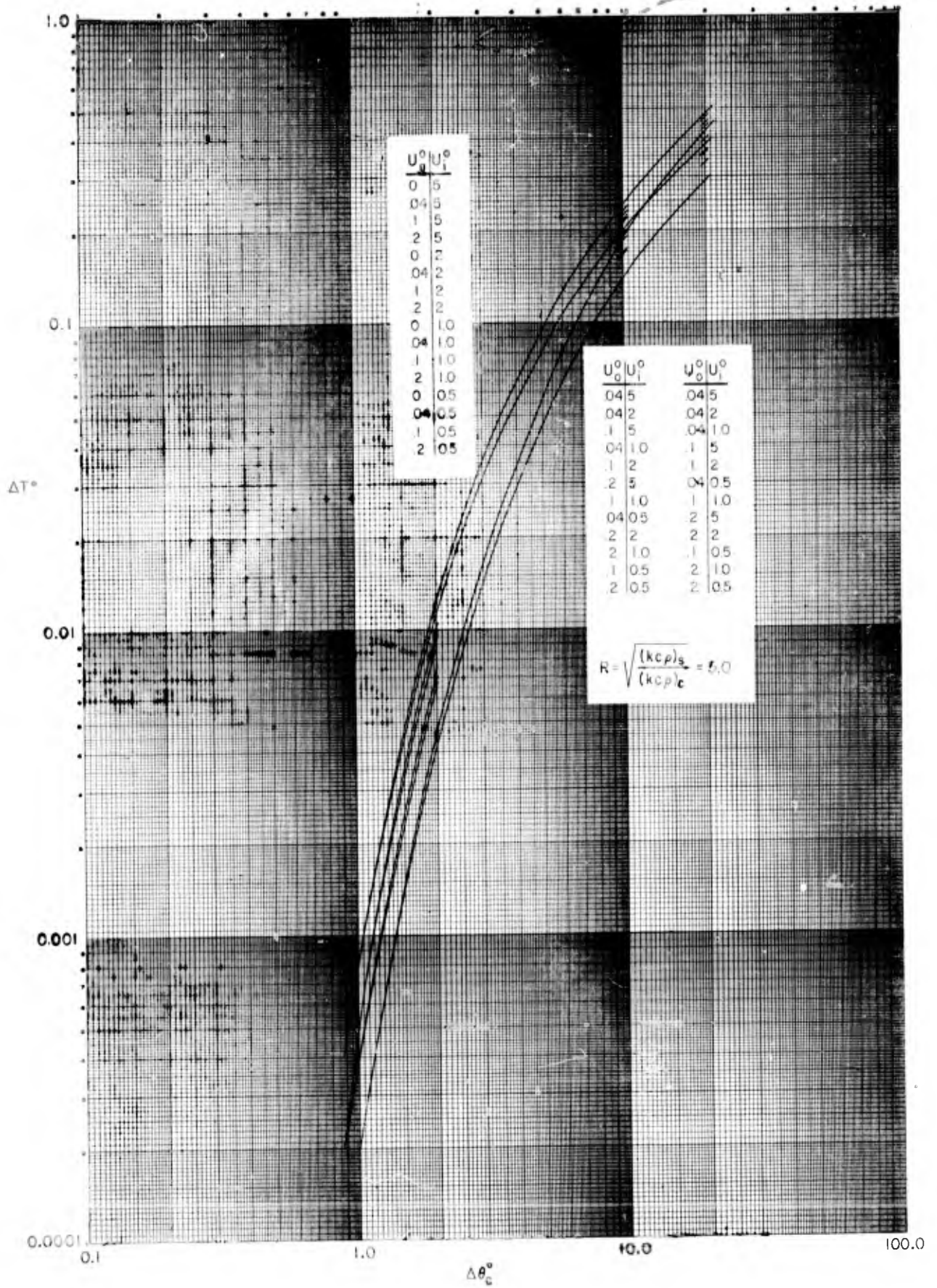


FIG.16- TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.55$
 (Curve I- All radiation absorbed at $X_c^0 = 0$)

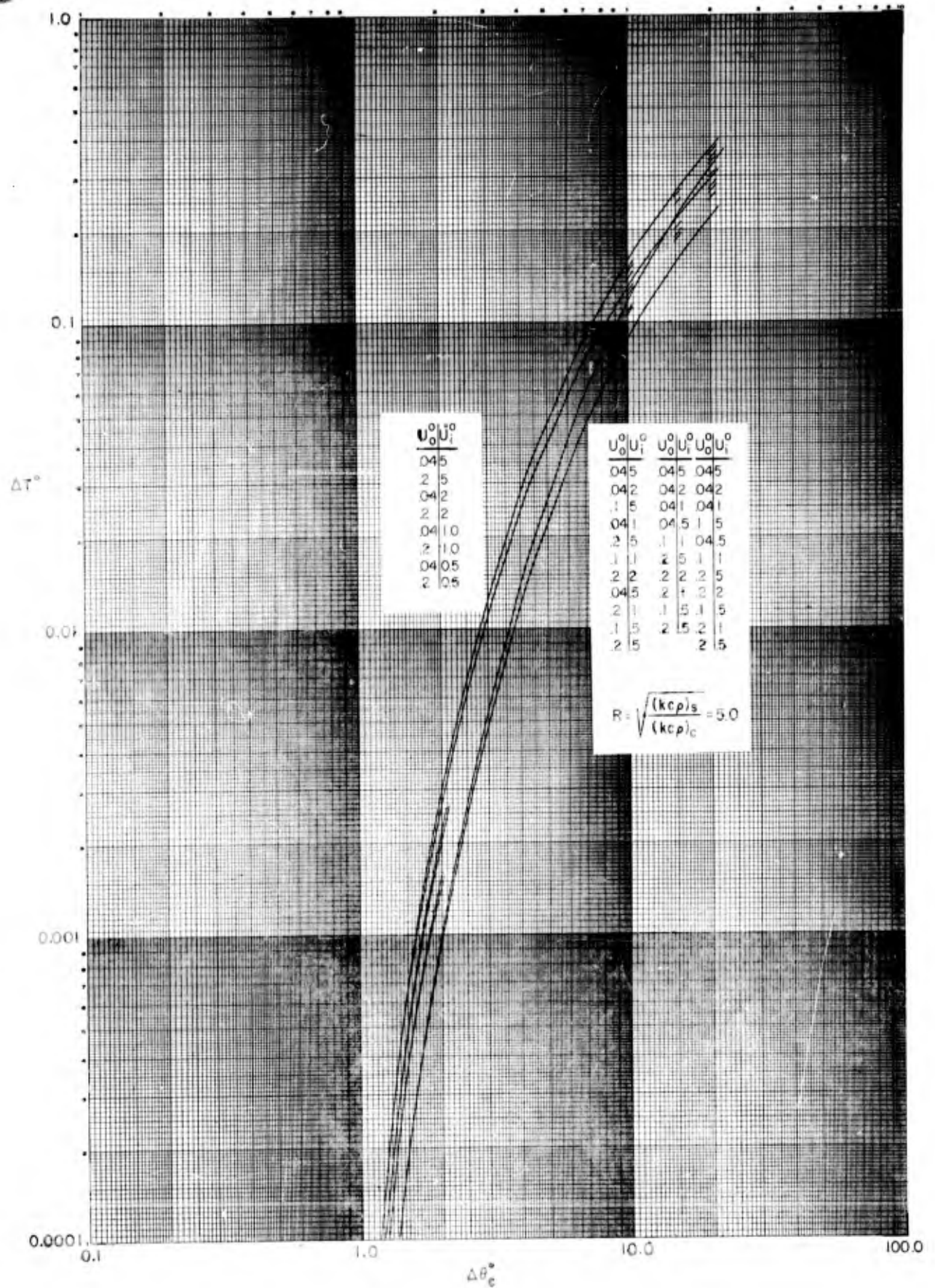


FIG. 17 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.75$
 (Curve I - All radiation absorbed at $X_c^0 = 0$)

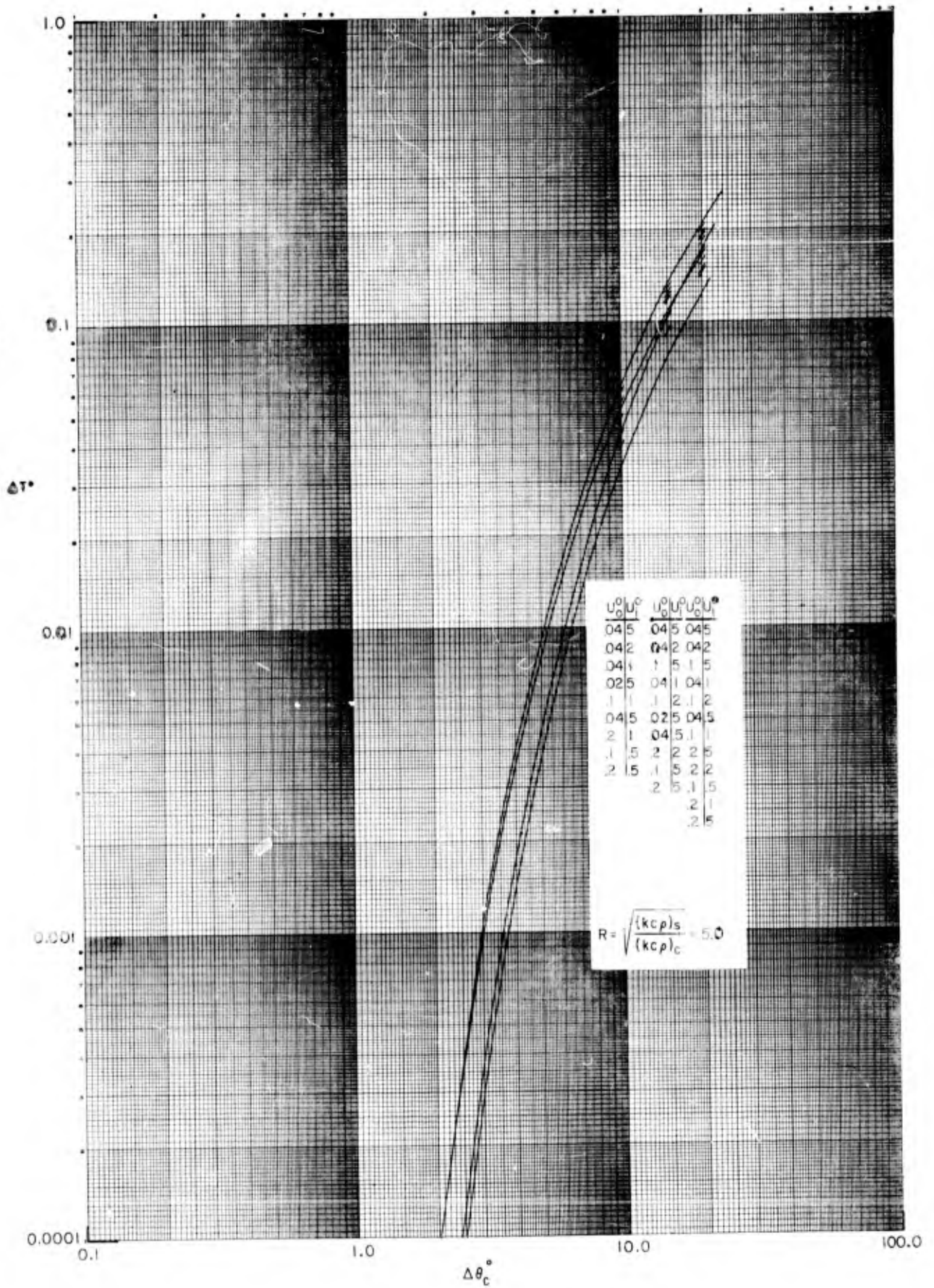


FIG. 18- TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 1.05$
 (Curve I- All radiation absorbed at $X_c^0 = 0$)

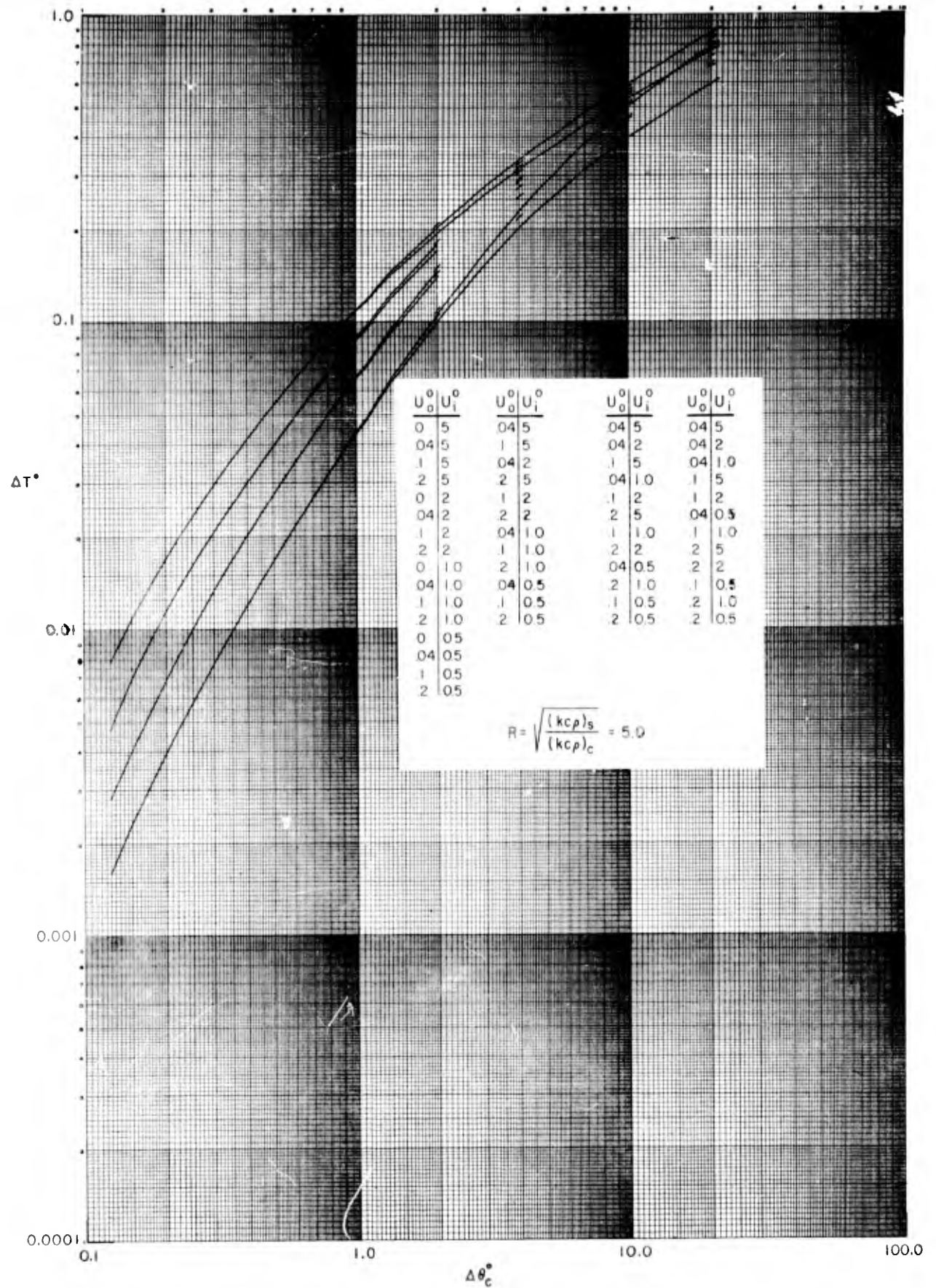


FIG. 19 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.05$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

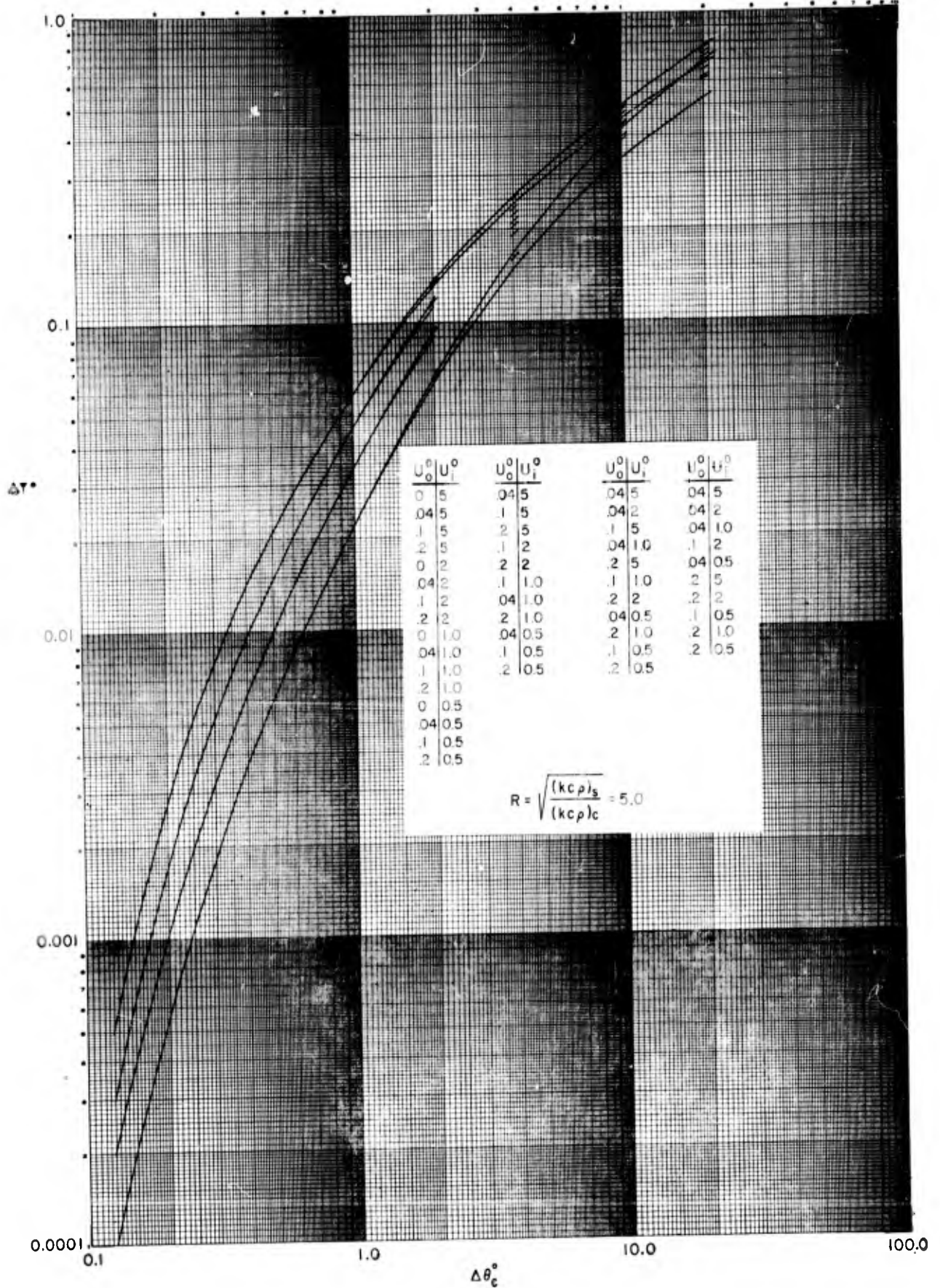


FIG.20 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.15$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

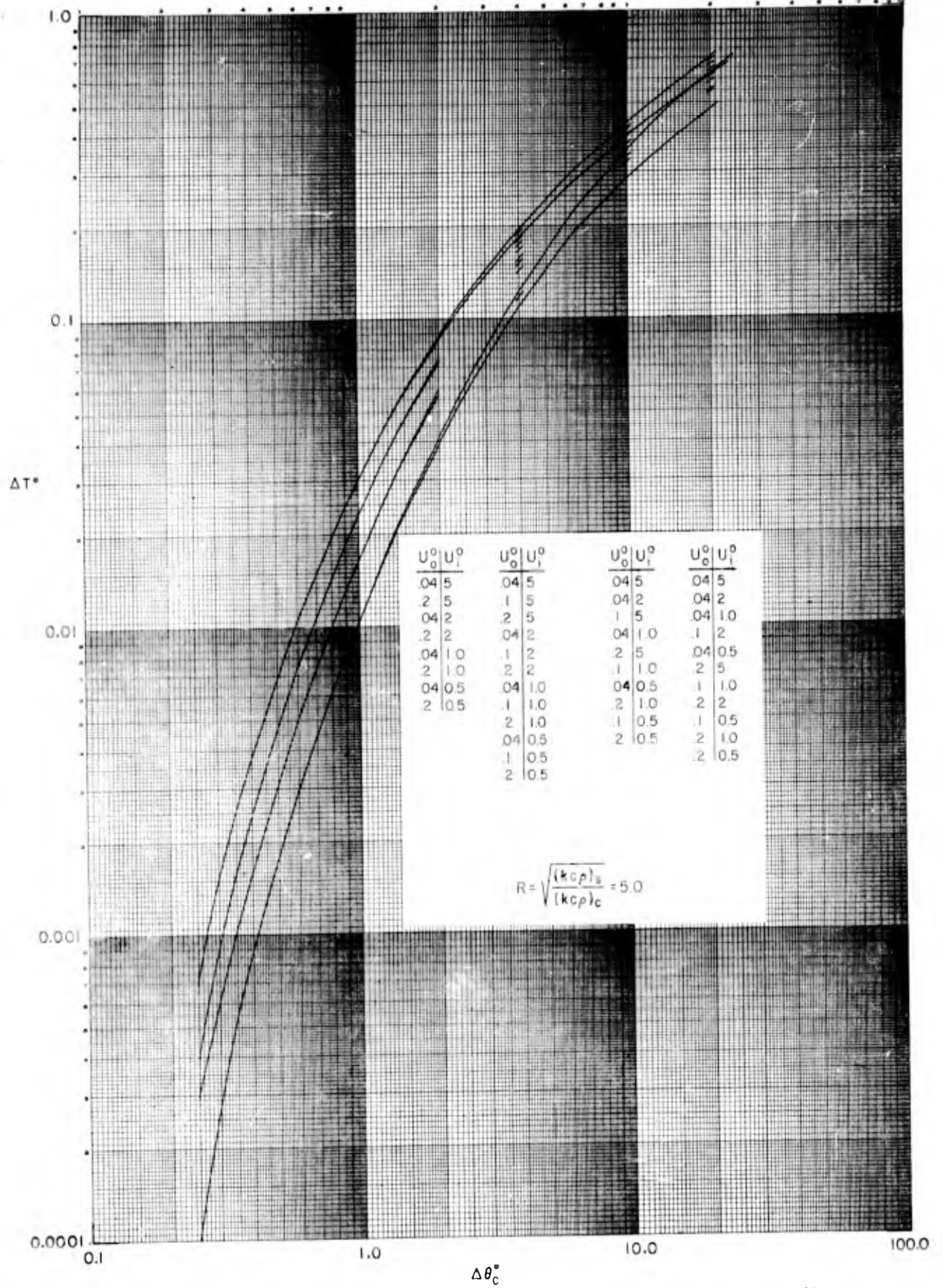


FIG.21 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.25$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

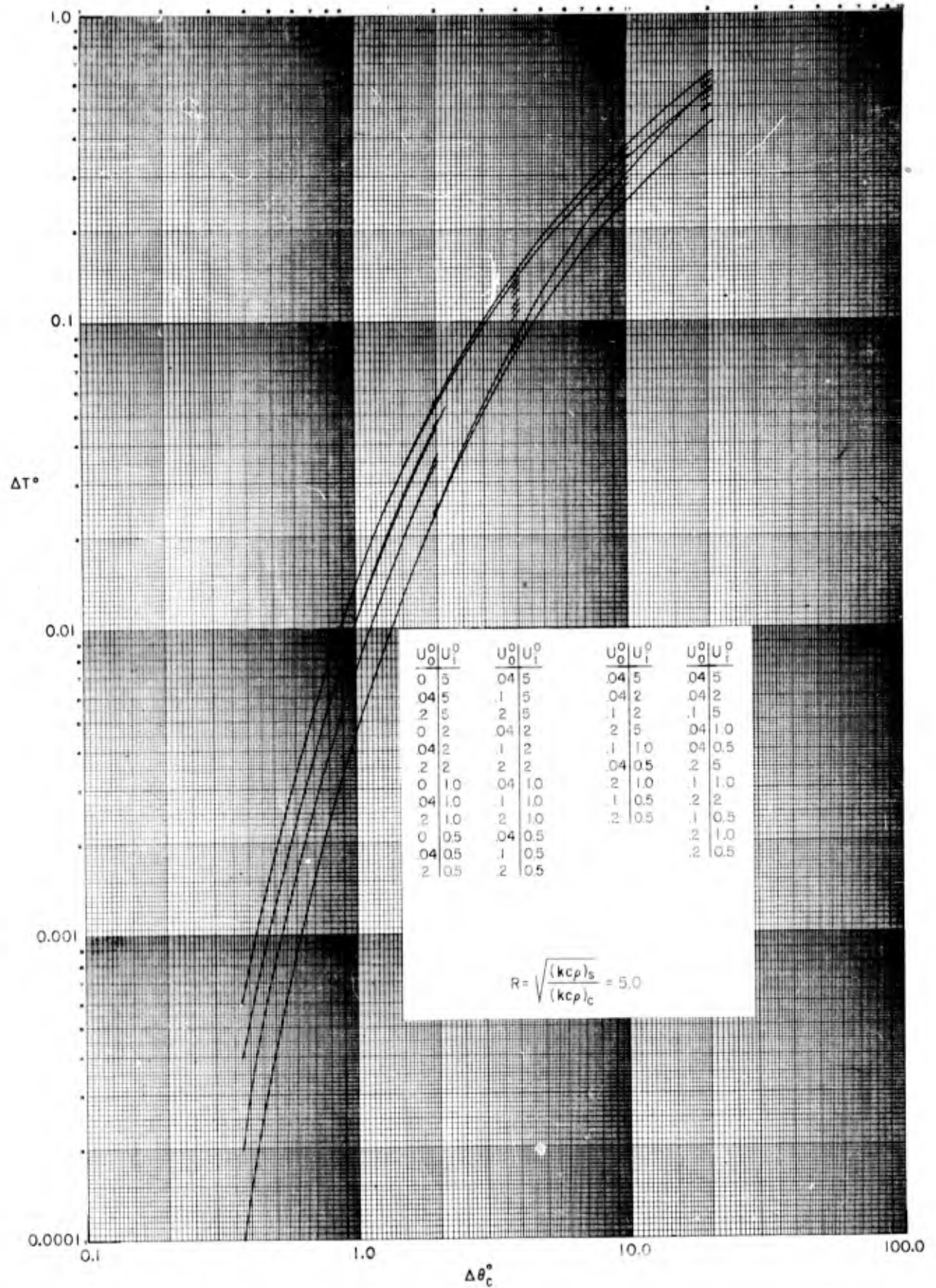


FIG.22- TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.35$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

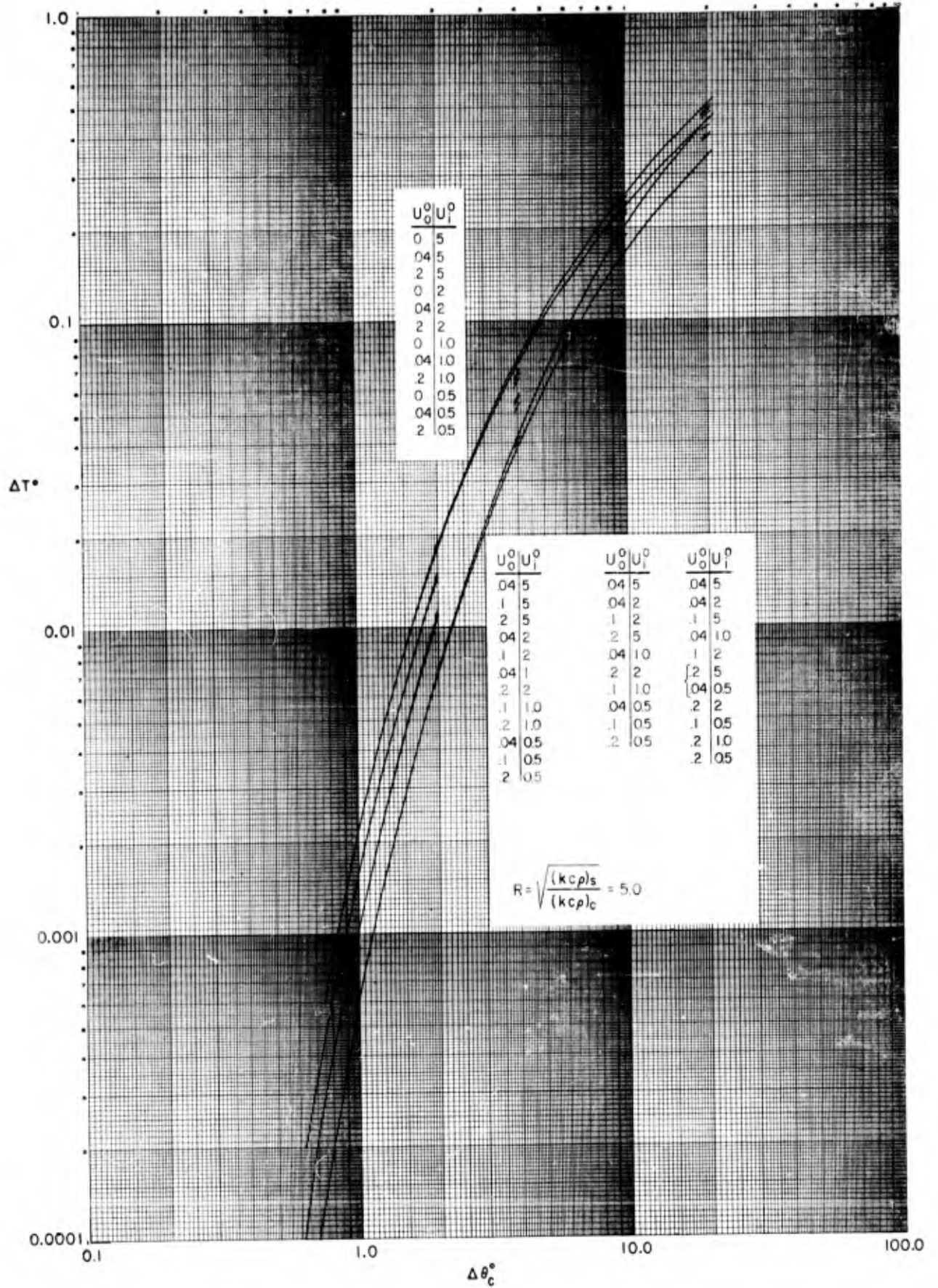


FIG.23 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 0.55$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

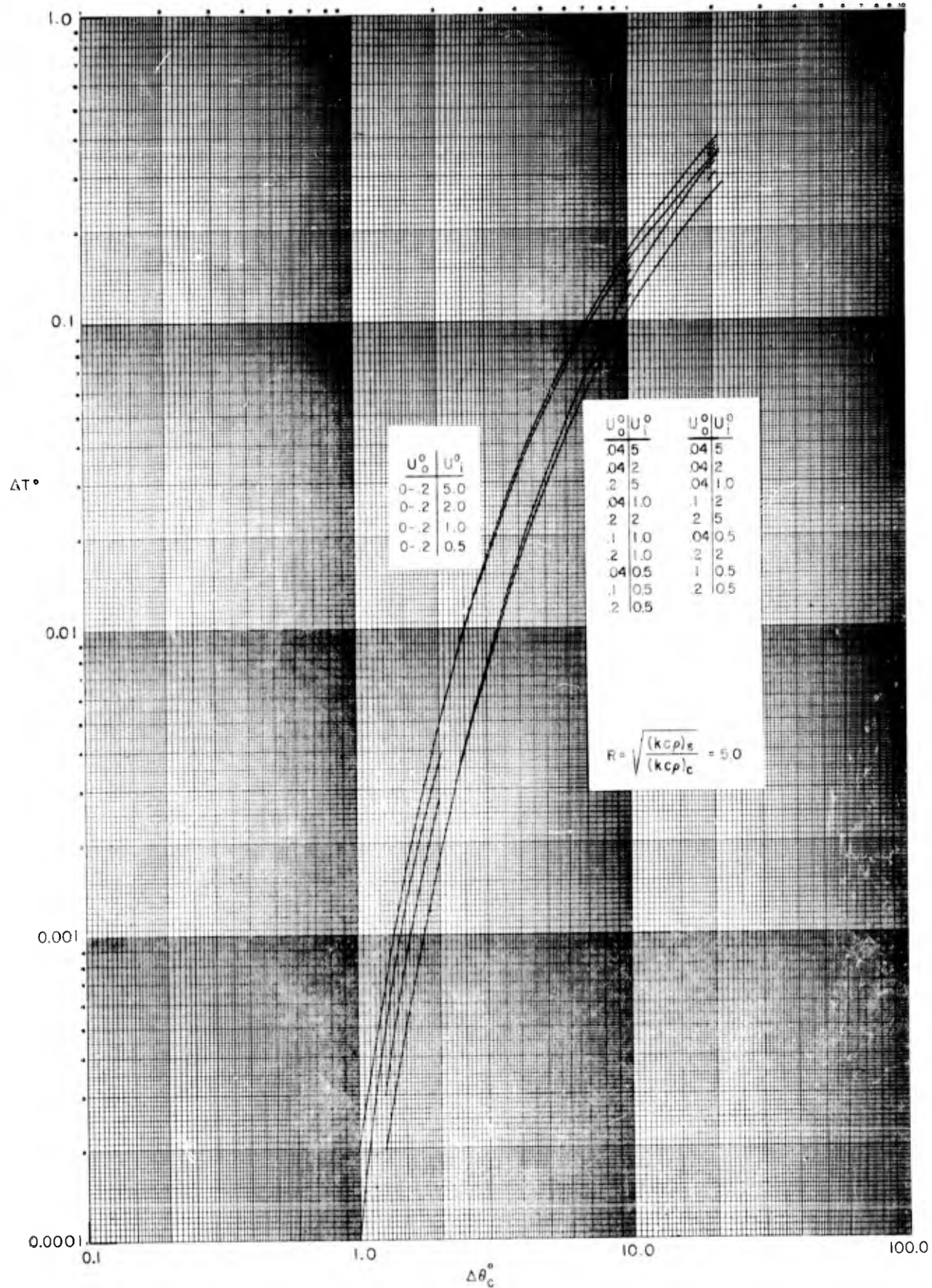


FIG.24 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_S^0 = 0.75$
 (Curve II - All radiation absorbed at $X_C^0 = 0.75$)

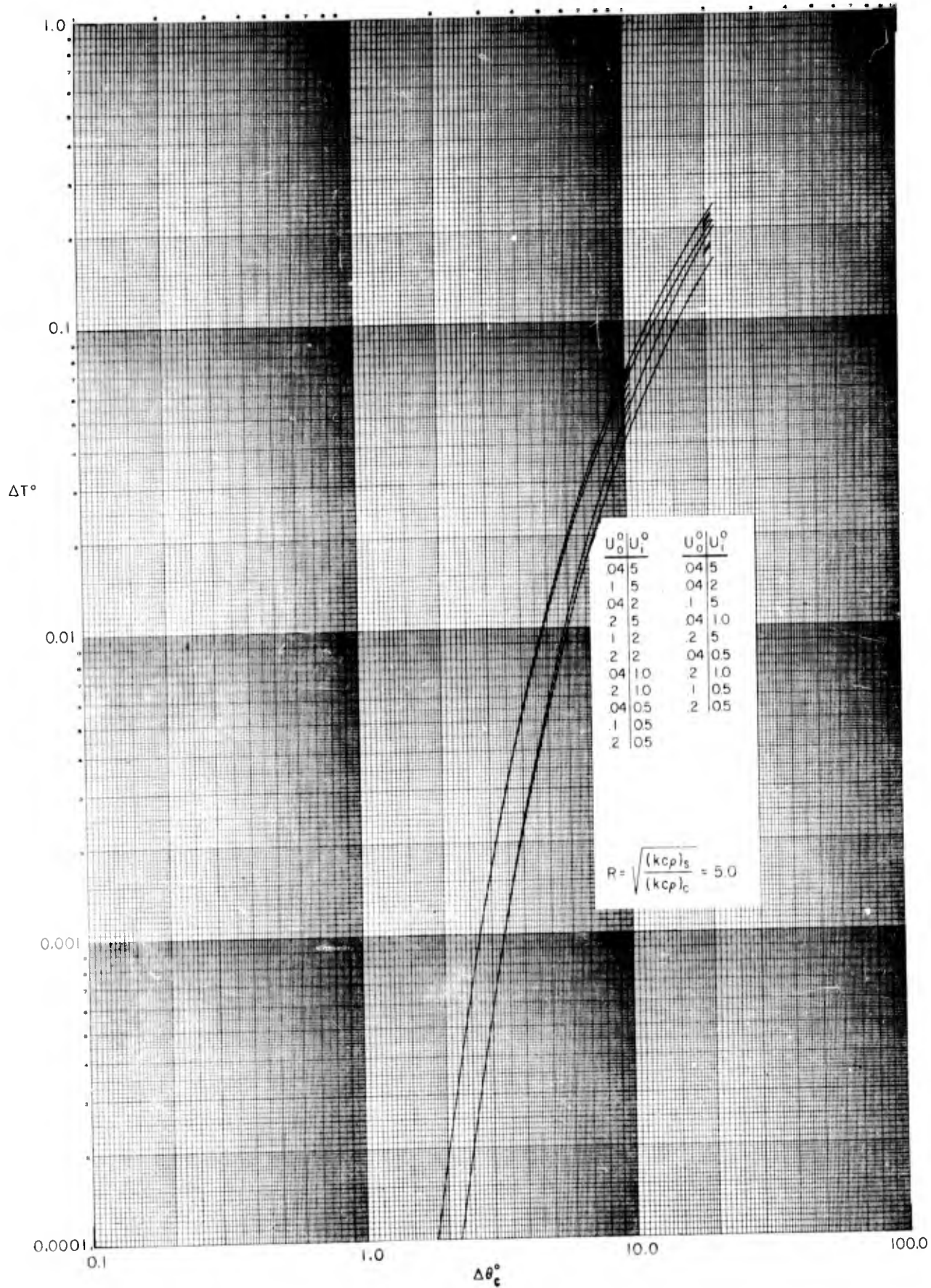


FIG. 25 - TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER. $X_s^0 = 1.05$
 (Curve II - All radiation absorbed at $X_c^0 = 0.75$)

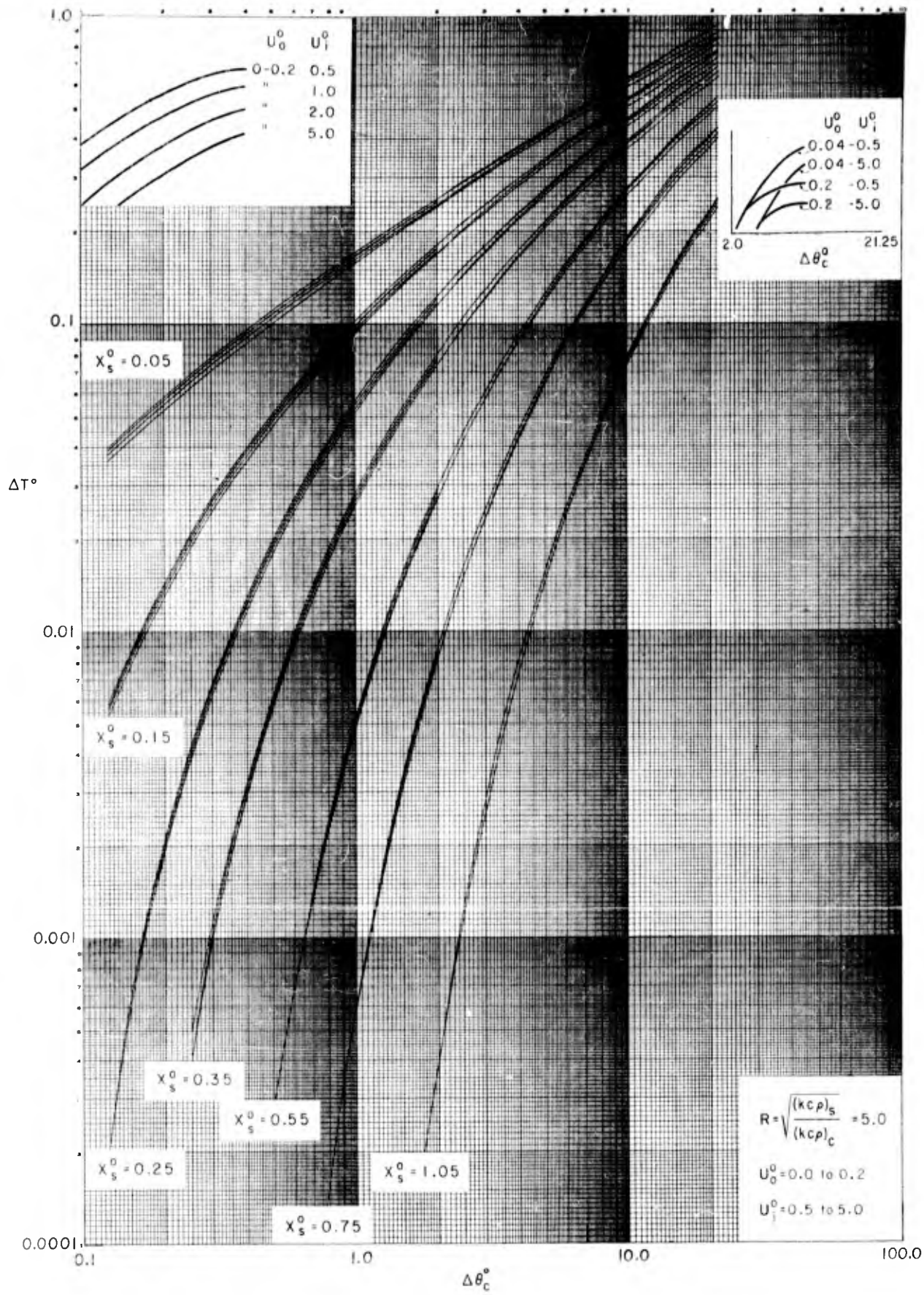


FIG.26- TEMPERATURE HISTORY WITHIN AN OPAQUE SEMI-INFINITE SOLID (SKIN) FOLLOWING THERMAL IRRADIATION THROUGH AN OVERLYING CLOTH LAYER.
 (Curve III- All radiation absorbed at skin surface)

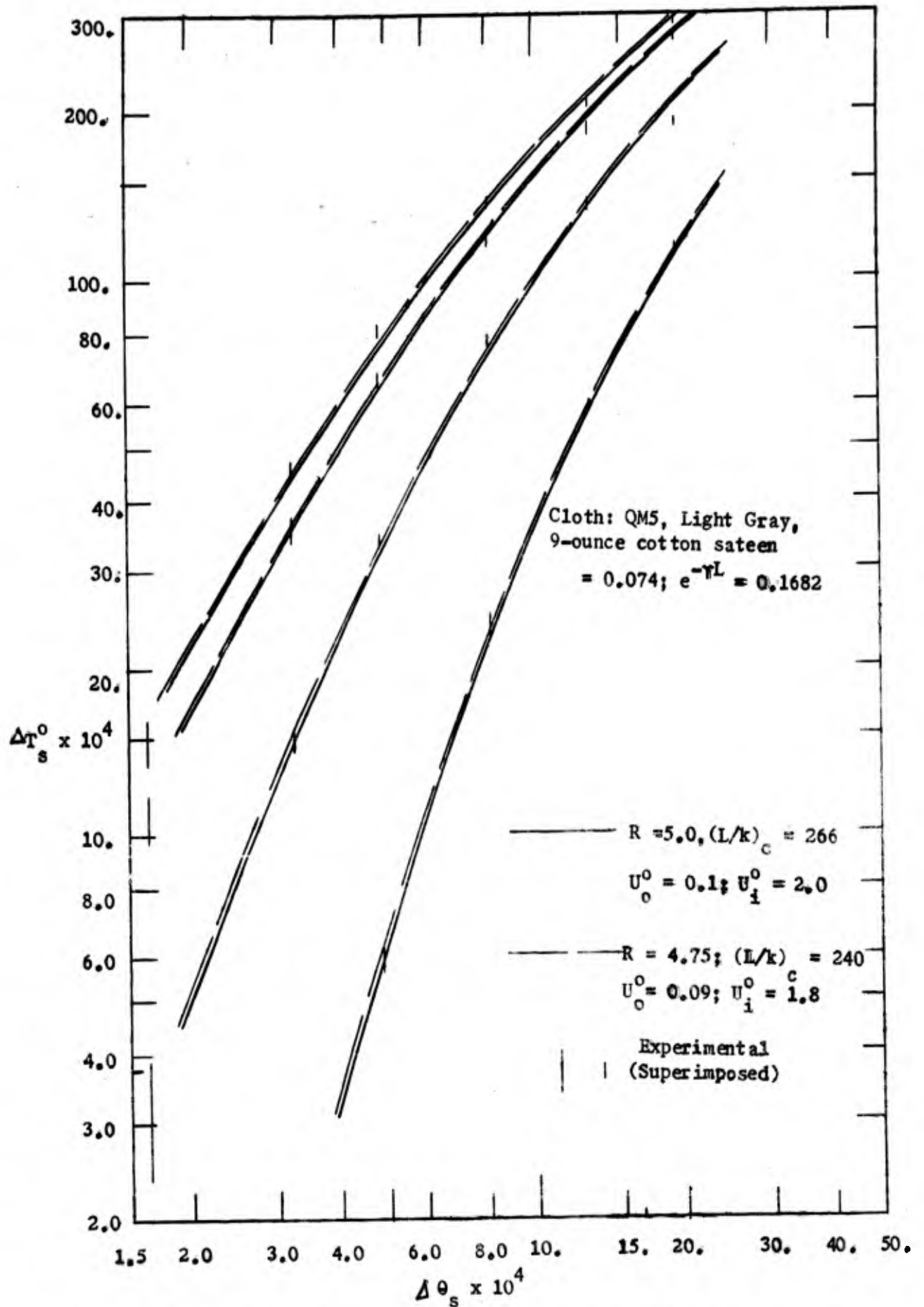


Fig. 27. Comparison of Theoretical and Experimental Temperature Histories Within a Skin Simulant Following Thermal Irradiation through Overlying Diathermanous Cloth.

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Technical Report No. 6

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