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EXPERIMENTAL AND THEORETICAL
STUDIES OF HULL-DECKHOUSE
INTERACTION - 2-LEVEL
DECKHOUSE

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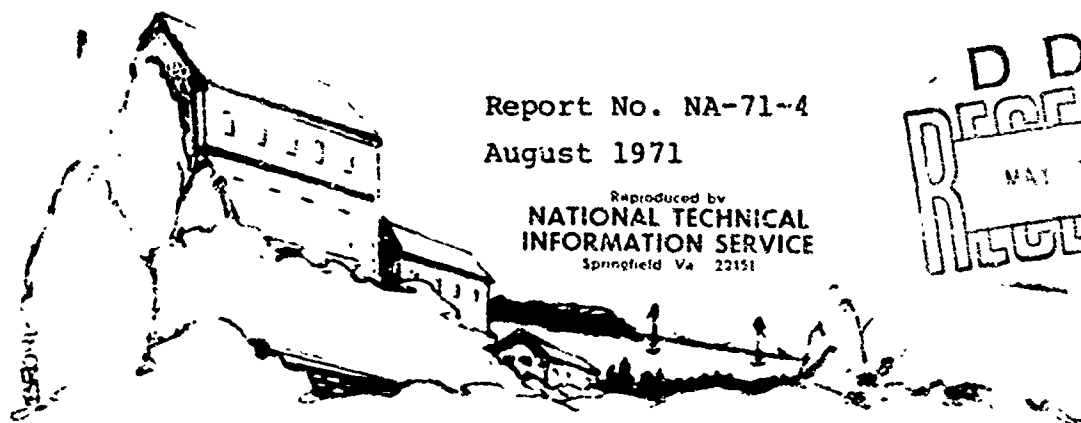
Johannes Buhler
Lawrence J. Levy

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Deflections of hull and deckhouse were measured. Stress distributions along three transverse sections and along the deck to deckhouse joint were calculated from measured strains.

Three bending moment distributions and two deck support arrangements were used in the test program. The experimental results are compared with results from finite element analysis and two other theoretical methods of analysis of hull-deckhouse interaction.

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I. INTRODUCTION

This report presents the results of a series of twelve structural tests performed to study hull-deckhouse interaction. The model, testing apparatus, instrumentation, and testing method are detailed in Section II. The previous series of tests (Ref. [4]) were performed using a single deckhouse, whereas this series of tests was conducted using a two level deckhouse.

The presentation of the experimental data and the computation of the stresses are outlined in Section III.

The finite element analysis is outlined in Section IV.

Two other theoretical methods of stress analysis are described in Section V.

In Section VI, the experimental results are discussed. Also, the different theoretical methods of analysis are evaluated in this section.

II. EXPERIMENTAL EQUIPMENT AND METHODS

1. Structural Model

A detailed description of the ship structural model is given in Ref.[4]. It is a longitudinally framed box girder, 42 feet long, 8 feet wide and 56 inches deep, with an attached 22-foot long deckhouse. For these tests the model was modified by adding a second level to the deckhouse.

A transverse section in way of the deckhouse is shown in Figure 1. A photograph of the interior of the hull model is shown in Figure 2. The model has no conventional transverse bulkheads. The restraint to vertical deformation of the main deck provided by such bulkheads is simulated by a number of stanchions attached to the deck, along the bond between deckhouse side and deck, and extending to the bilge of the model. When it is desired to make a stanchion effective, a spacer is inserted between the overlapping portion of the flanges of the upper and lower parts of the stanchion and the two are firmly bolted together.

The model deckhouse construction plan is shown as Figure 3. The two decks of the two-level deckhouse are bolted to the sides and ends of the deckhouse with 1/4-inch bolts on 3-inch centers. All other structural joints are welded. The diaphragm plate, shown in Figure 3, was installed on one end only. At the other end, the deckhouse was welded directly to the upper deck of the hull girder.

2. Load Application

The test model was subjected to static loads on the Ship Structures Static Testing Machine which is located at the University of California Richmond Field Station and described in Ref.[2]. The model rests on seven bags of equal size which cover its entire bottom area. Each bag has its individual control

valve and can be subjected to air pressures ranging from +12 to -12 psig; this feature allows application of positive and negative loads to each of seven transverse sections of the test model under investigation. When not under tests, the model is suspended at its four corners by means of turnbuckles. During the entire test the pressure in two of the seven bags is adjusted to carry the weight of the model, so that the forces in the turnbuckles decrease to zero. This is continuously checked by recording the forces on a 4-track Sanborn, Model 154, strip chart recorder (Figure 8).

3. Strain Gage Instrumentation

A strain gage map showing the location of the 524 gages applied to the model is shown in Figures 4 and 5. Figure 4 is an expanded view of the deckhouse and the first level deck of the deckhouse. Figure 5 is an expanded view of the hull girder. Gages are distributed along three transverse sections of the hull and deckhouse, and along the joint of the deckhouse sides and ends to the main deck.

Gages are generally applied to both sides of the plate in order to obtain the "heart-of-plate" strain. All gages are of etched foil type, with a resistance of 120 Ω . Three different sizes of these gages have been selected to suit the requirements at different locations. Those used for all 3-gage rosettes were 1/4" x 1/8". In positions 713 to 721, ready-made rosettes with 1/8" x 1/16" gages were installed. In all other locations, 1/2" x 1/4" gages were used. The strain gages were connected to bridge completion boxes (Figure 7). Each box has thirty-six terminals. Thirty-five terminals were used for active gages, the remaining terminal was connected to a dummy gage (resistor). The strains were measured automatically in an electronic data acquisition system (Figure 9) and recorded on magnetic tape. A description of the electronic system is given in Ref.[1].

4. Deflection Instrumentation

A total of seventeen dial gages were used to measure the relative deflection of hull and deckhouse. The distribution of these gages is shown in Figure 6. All gages were attached to two trusses running along the outer edges of the main deck and supported only near the ends of the hull. Gages lettered A to K were used to measure the deflection of the hull along the deck edge. Gages numbered 1 to 6 gave the deflection of the deckhouse as measured at the edge of the deckhouse deck. Gages 5, 6, F, I and J were installed to check the symmetry of the deflections about both axes of the model. Some of the gages can be seen in Figure 7, which is an end view of the deckhouse, looking aft.

The lowest resolution reading of the deflection gages was 0.001 inch.

5. Test Conditions

Load and stanchion support conditions of the twelve tests discussed in this report are summarized in Table 1. The last letter in the designation given to each test (S or H) represents sagging or hogging. Net pressures applied by each of the bags are listed in Table 2. The odd values of pressure result from setting the pressure regulators in units of cm. of Hg rather than psi.

6. Test Methods

Each experiment consisted of reading strains and deflections at each of three load conditions. The first and third load conditions, designated initial and final, have the model afloat on bags 3 and 5. The second, or test step, has the model under pressures corresponding to those listed in Table 2. Bags 3 and 5 have an additional pressure sufficient to float the loaded model. In general the pressure in bags 3 and 5 is approximately the sum of the pressure in either the initial and final step and the nominal

test pressure. Any deviation is due to pull on the model periphery by the strap sealing the vacuum surrounding the pressure bags. A detailed description of the test procedure is given in Ref.[1].

TABLE 1 DESIGNATION AND CONDITION OF TEST

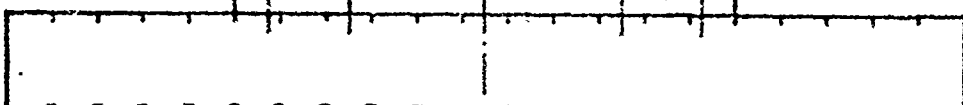
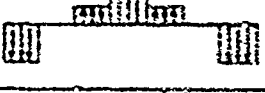
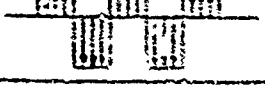
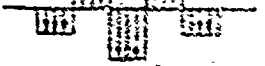
TEST SECTION		1	2	3	4	5																	
																							
FRAME NO.		0	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21
STANCHIONS AT FRAME NOS		NONE											ALL										
LOAD DISTRIBUTION													8AS										
													3AH										
													8BS										
													3BH										
													8CS										
													3CH										

TABLE 2 Pressure Distribution in Tests

Pressure in Bag No. (psig)							Test No.
1	2	3	4	5	6	7	
7.73	0	-7.73	0	-7.73	0	7.73	319 AS
-7.73	0	7.73	0	7.73	0	-7.73	319 AH
7.73	0	-3.865	-7.73	-3.865	0	7.73	319 BS
-7.73	0	3.865	7.73	3.865	0	-7.73	319 BH
0	5.03	10.06	10.06	10.06	5.03	0	319 CS
0	-5.03	10.06	-10.06	10.06	-5.03	0	319 CH

III. EXPERIMENTAL RESULTS

1. Presentation of Deflection Data

As shown in Figure 6, dial gages were distributed port and starboard on both the main deck edge and the deckhouse edge. The gages on the starboard side were installed to check the symmetry of the deflections about the longitudinal axis of the model. The maximum difference between port and starboard readings was $4 \cdot 10^{-3}$ in. The maximum difference between deflection readings for hogging and sagging was $3 \cdot 10^{-3}$ in. For these reasons, only port deflections in hogging were plotted.

Deflection readings are shown in Appendix A. A correction had to be made on the deflection readings because the trusses were supported at 18" from the ends of the main deck and not at its ends. The correction was obtained by extrapolation of the deflection curve to the end of the deck (see Ref.[1]). Corrected values are listed in Table 4 and plotted in Figures 11-16.

2. Computation of Strains and Stresses

The strain data are stored on magnetic tape in binary coded decimal form. During one test of three runs, 1572 values are obtained from 524 strain gages. Each value, called a sample, consists of 6 digits representing a number from 0 to 12000 and a sign. Two computer programs, reproduced in Appendix B, are used to process the strain readings.

A program DATDUMP serves to read the data from tape and to punch them on card in a convenient form. Each card contains the initial, test, and final readings (runs) for two strain gages. Simultaneously, a printout of the data is made.

A second program, called STRESS serves to read and process the data obtained with DATDUMP. Bad data with non-interpretable

digits are changed to zero and listed for identification. The readings are then converted into strain values by multiplication with a calibration factor (Statement 3010). "I" is an index assigned to each strain gage consistent with the chronological order in which the gages are scanned. The strain readings $X(I)$ are then assembled for each position (Statements 14 through 742). Note that the position numbers are identical with the statement numbers. The designations A_1 through A_6 refer to the direction and location of the gages according to the comment cards following statement 2400. For a definition of longitudinal, transverse and diagonal directions, see Appendix B. The values of A_1 to A_6 are printed in the last six columns of Tables 5 to 10.

Stresses are computed for each position number in Statements 2501 to 2505, assuming the following values for the material constants:

$$E = 3 \times 10^6 \text{ lbs/in}^2$$

Modulus of elasticity

$$G = 11 \times 10^6 \text{ lbs/in}^2$$

Shear modulus

$$\nu = 0.3$$

Poisson's constant

The stress values are printed out in the first 5 columns of Tables 5 to 10. Some values are plotted in Figures 11 to 27.

IV. FINITE ELEMENT ANALYSIS

The finite element analysis is based on procedures in which the real continuous structure is approximated by an assemblage of simple structural elements. Conditions of equilibrium of forces and compatibility of displacements are required to be satisfied at discrete locations throughout the approximating structure. This results in the reduction of the problem to the solving of sets of simultaneous equations relating these forces and displacements.

The degree of approximation involved varies, depending on the size of the mesh used in subdividing the structure into elements, but satisfactory convergence to the exact solution is generally obtained if the mesh size is made sufficiently small. One should see Ref.[7] for a more complete description of this type of analysis.

It is necessary to use a very fine mesh when examining regions of high stress concentration, such as the connection of a deckhouse end structure to the hull, resulting in excessive demands on computer capacity. A procedure is outlined in Ref.[7] whereby one may examine closely, by a succession of refinements, small regions of interest in a large structure.

In the analysis used here, one-quarter of the deckhouse (cut along centerline & midships) was used in step one. These results were plotted against the experimental data at the three cross sections, Figures 11-18. In the vicinity of the deckhouse end, two refinements were made in the size of the segment examined, and the results of step three were plotted against the experimental data, Figures 22-27.

The boundaries used in the different steps are shown in Figure 10.

V. OTHER THEORETICAL METHODS

Two other more simplified methods of analysis were used to analyze the stresses at the midship section. The first one was proposed by Kammerer, Ref.[6], and the second one was proposed by Schade, Ref.[10].

Kammerer's method for calculating the stress distribution across hull and deckhouse utilizes semi-empirical results of fullscale experiments to evaluate the effect of differential deflections between deckhouses and their hull girders. These data have been incorporated into an analytical treatment of the problem based on plane stress theory. Shear lag is taken into account in figuring the "equivalent area" of the deckhouses, but is not considered in the hull girder analysis. The design charts given in Kammerer's paper are plotted using full ship dimensions, and it was therefore necessary to extrapolate model scantlings to full-scale. Calculations for one case are shown in Appendix E.

Schade's method for calculating the stress distribution across hull and deckhouse is also based on plane stress theory, but differs from Kammerer's in that shear displacement is accounted for in the sides of the hull and deckhouse and shear lag in the hull girder is included. The last item allows the stress at the deckhouse-hull connection to be different from the stress at the hull side. This method does not depend on empirical data, but instead upon the evaluation of a deck flexibility factor "K". The values of "K" were determined experimentally in Ref.[3] to be 650 psi for the all stanchions loose condition and 34,000 psi for the all stanchions fixed condition.

Fourier coefficients for the expansion of the three bending moments used in the experiments are listed in Appendix D.

The nomenclature for Schade's method is reproduced in Appendix E, along with calculations for one case.

Both of these methods were based on the simplifying assumption that the bending moment could be expanded into one constant term and one sinusoidal term. This is a good assumption in most cases, but will not hold in the case of bending moment "C", a "saddle form" bending moment. It was therefore necessary to return to the basic formulation of the equations and rederive them based on "n" sinusoidal terms. This was done for Schade's methods using Reference [9]. The new equations are shown at the end of Appendix E.

First a new equation for "r" was developed and is shown as Eq. E-8A. Next, using the new assumption for bending moment (Eq. E-9A), a new equation for " p_f " was obtained and is shown as Eq. E-10A. The first four terms of Eq. E-10A are from the homogeneous solution and the last two terms are from the particular solution of the differential equation. The first four terms have coefficients which must be evaluated using the boundary conditions. All six terms must be evaluated for each term of the moment expansion. Because "p" will be different for each term of the moment expansion, all of the "effective" geometric properties, and coefficients calculated from these, have to be recalculated for each term also. Figures 33 and 34 were obtained using a three term expansion.

VI. DISCUSSION

As stated, this report is a continuation of a previous report (Ref.[4]). The main reason for conducting this series of tests was to find what effect the addition of a two-level deckhouse would have on the previous experimental results. Another reason was to further check the various theoretical methods of predicting stress. Only the two extreme stanchion support conditions were used (no stanchions, and all stanchions fixed). Three bending moment distributions were used, instead of five as in the previous work, since the previous variation in moment distribution gave no significant variation in longitudinal direct stress distribution (except for "saddle-form" bending moment "C").

The experimental results followed the same basic form as shown in Part I (Ref.[4]), the only major difference being the reversal of stress in the deckhouse in the "no stanchion" condition. Where, in the one-level deckhouse experiments, the stress would decrease as one moves from the hull girder to the deckhouse top, now the stress decreases and reverses sign below the deckhouse top. This could have been expected from extrapolation of the previous results, but goes against intuition.

In Figures 11-16, the experimental longitudinal stresses and deflections are plotted along with the theoretical finite element values. The two agree very well, although the finite element deflections and stresses are generally slightly less than experimental values for moment distributions "A" and "B", and slightly greater than experimental values for moment distribution "C".

In Figures 17 and 18, the experimental longitudinal stress distributions amidships are compared. They are basically the same as shown in Part I (Ref.[4]), except that the second level of the deckhouse has lessened the effect of moment distribution on deckhouse stresses. The plots were obtained by multiplying the

computed stresses for each moment distribution by the ratio of the midship value of bending moment "C" to the midship value of their respective bending moments.

Figures 19-21 are again similar to those of Part I. These figures show the shear and vertical stresses at the deck-house-hull bond. They show, as expected, highest vertical stresses at the deckhouse end and over the midship stanchions and frames (higher values when stanchions are in place). They show high shear stress at the deckhouse end, continuously decreasing to zero at amidships.

Figures 22-27 compare the experimental values of vertical and shear stress with the theoretical values of the third step finite element analysis. There is generally good agreement between the two values, especially for the vertical stress. These figures again show the difficulty of obtaining accurate values at the corners, due to the finite size of the strain gages. Because of the very steep stress gradient, it is impossible to measure the maximum stresses involved.

Figure 28 shows a comparison of the values obtained by Kammerer's and Schade's methods for a typical bending moment. This shows that Kammerer's method will give results between the two extreme values of Schade's method, but that it gives results much closer to the rigid deck extreme than to the flexible extreme. This should be expected since most naval and passenger ships upon which the empirical data are based have fairly rigid main deck support. For this reason, in Figures 29-34, Kammerer's method is only compared with experiment for the "all stanchions fixed" condition.

In Figures 29-34, the experimental results at amidships are plotted along with Kammerer's and Schade's methods. The comparison is generally good. As stated in Section V, it was necessary to extend Schade's method in order to obtain reasonable correlation

between experimental and theoretical results for bending moment "C".

Kammerer's method will give a reasonably good approximation of stress when very little information is available on the ship's structure. It is by far the simplest method to apply, but requires that the bending moment be of simple sinusoidal shape.

Schade's method also gives a good approximation of stress, but requires the finding of a deck flexibility factor (which makes it more versatile). It is fairly simple to apply, unless more than one sinusoidal term is required to approximate the bending moment distribution, which complicates the problem somewhat.

The finite element method can, of course, be the most accurate, but also requires the greatest amount of work to solve a problem.

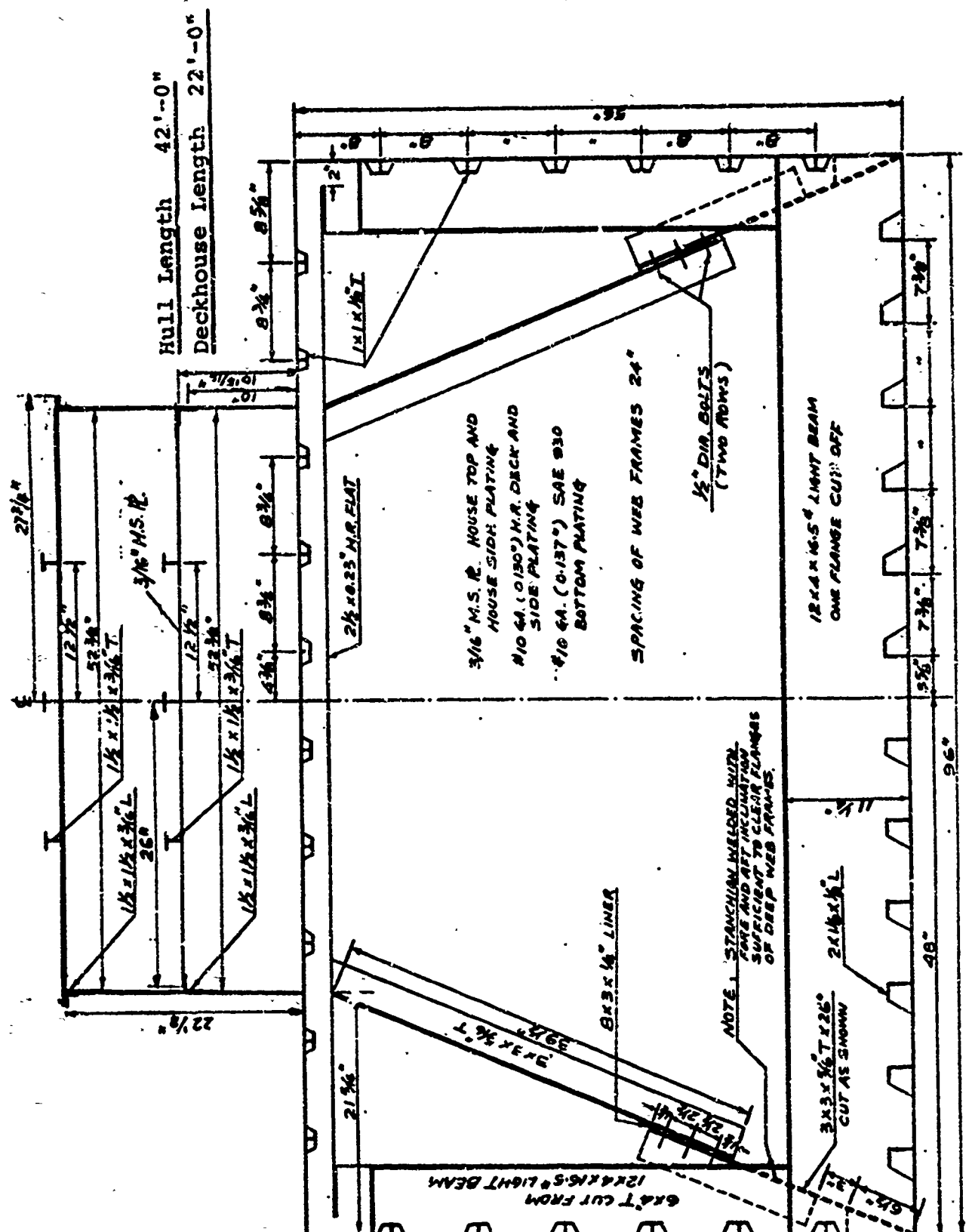


FIGURE 1. MIDSHIP SECTION OF STRUCTURAL MODEL



FIGURE 2. INTERIOR VIEW OF HULL MODEL

CLEARANCE DRILL FOR $\frac{1}{4}$ MACHINE SCREWS ON 3" CENTERS

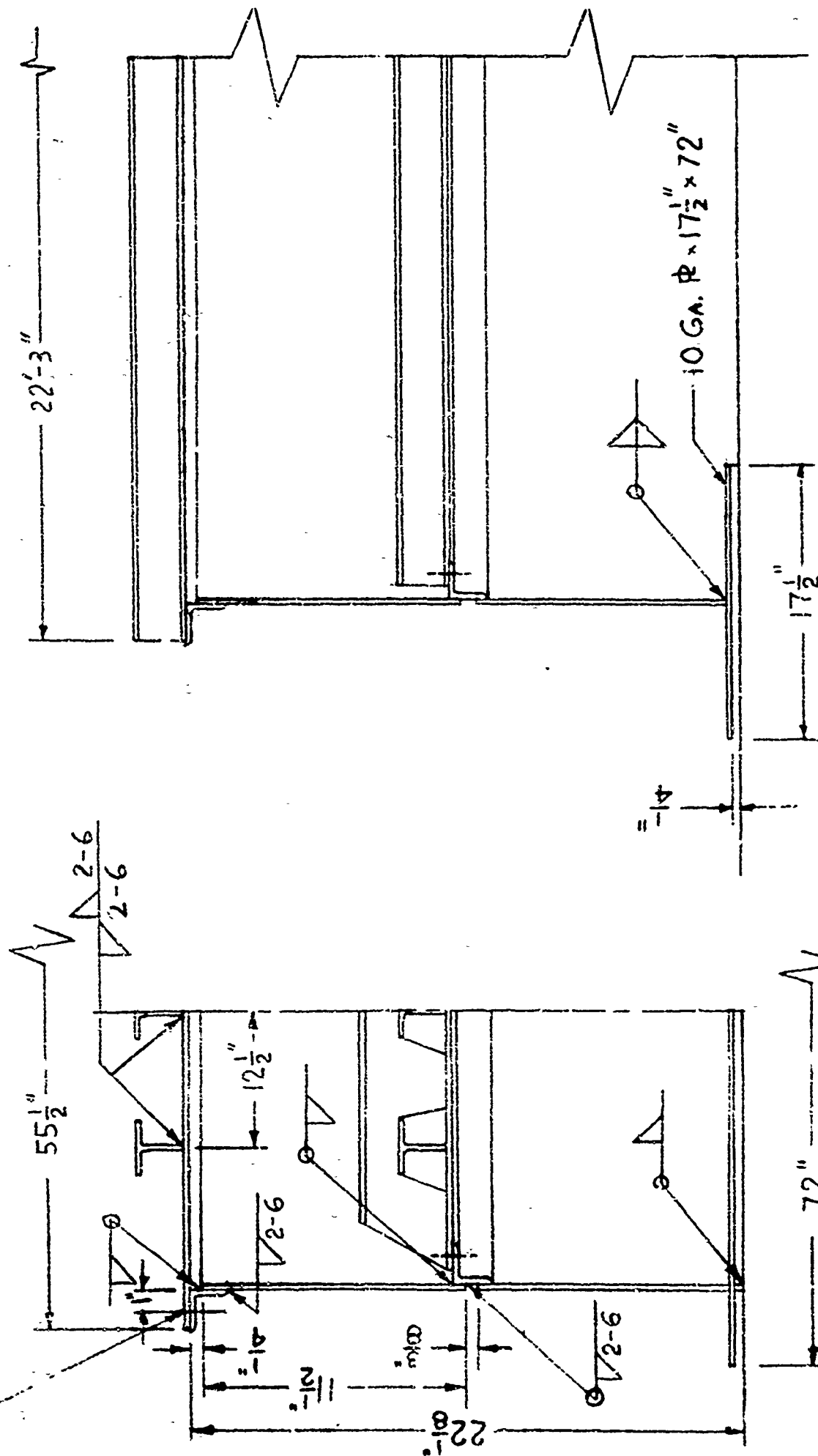


FIGURE 3. DECKHOUSE CONSTRUCTION PLAN

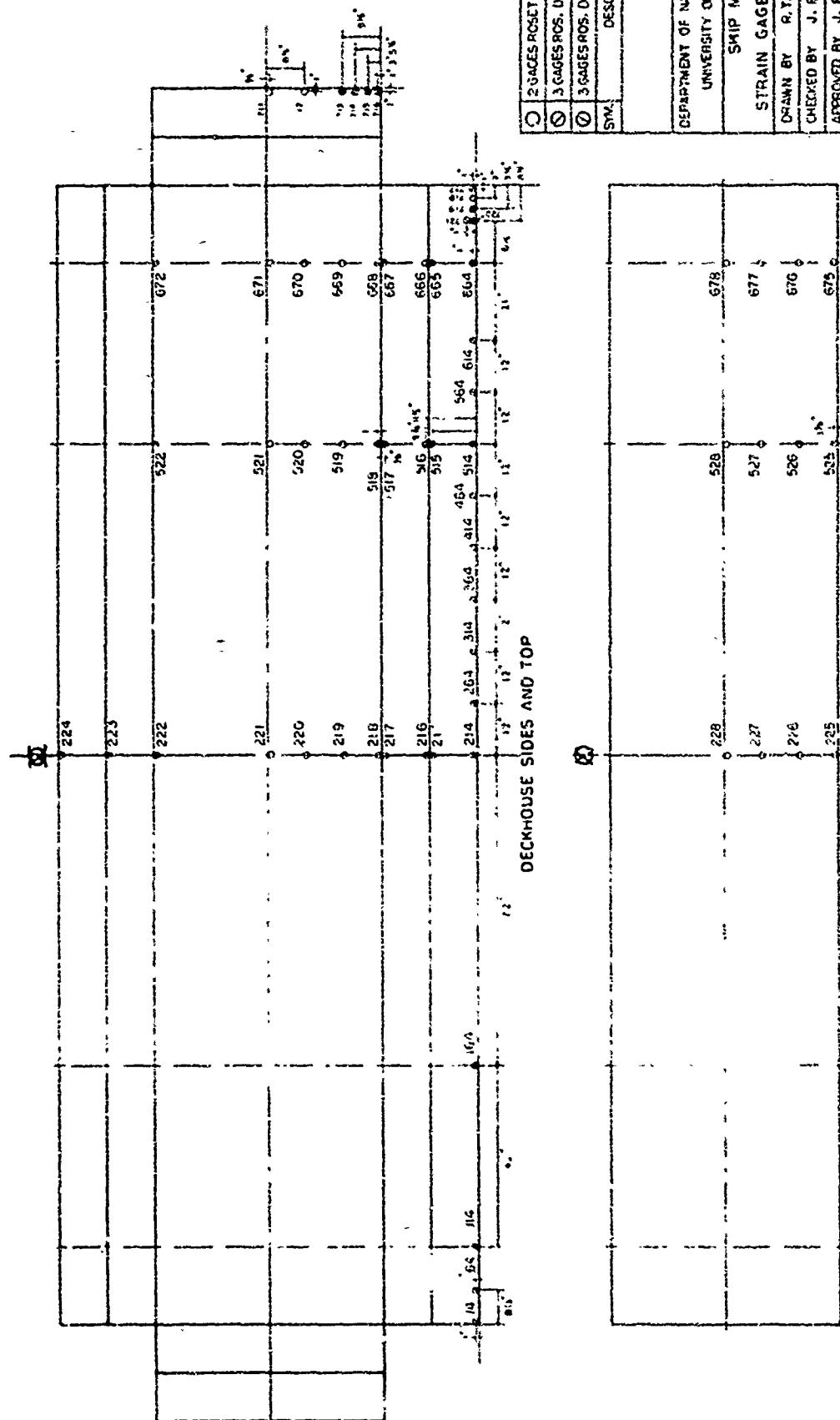


FIGURE 4. STRAIN GAGE LOCATION MAP - DECKHOUSE

2	GAGES ROSETTE
3	GAGES ROS. DIAG. IN C. TON
3	GAGES ROS. DIAG. IN / DIRECTION
SYM.	DESCRIPTION
DEPARTMENT OF NAVAL ARCHITECTURE UNIVERSITY OF CALIFORNIA SHIP MODEL STRAIN GAGE POSITIONS DRAWN BY R. T. HUANG CHECKED BY J. BUMLER APPROVED BY J. BUMLER DWG. NO. TON-01 DATE JULY 1969	

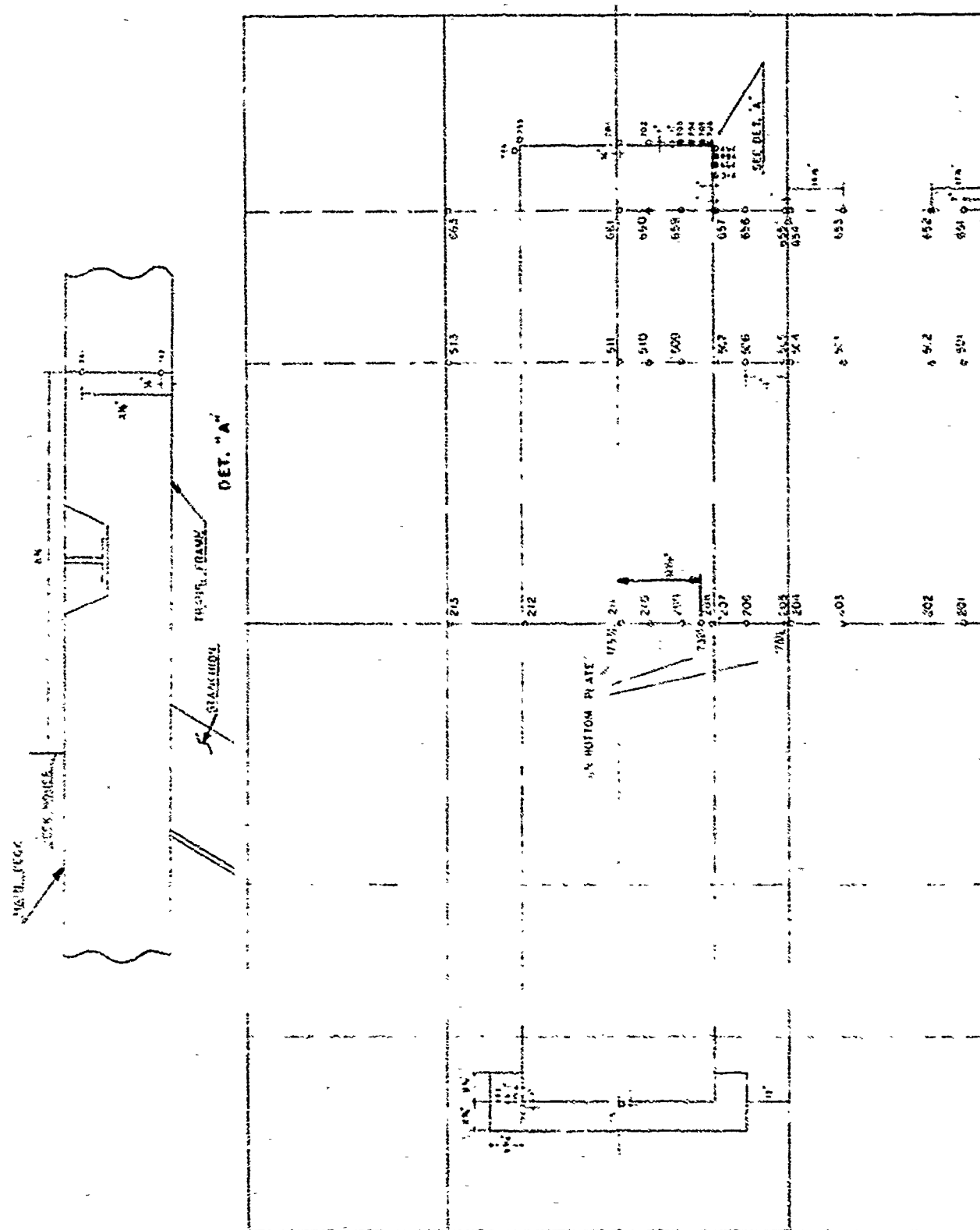


FIGURE 5. STRAIN GAGE LOCATION MAP - HULL GIRDER

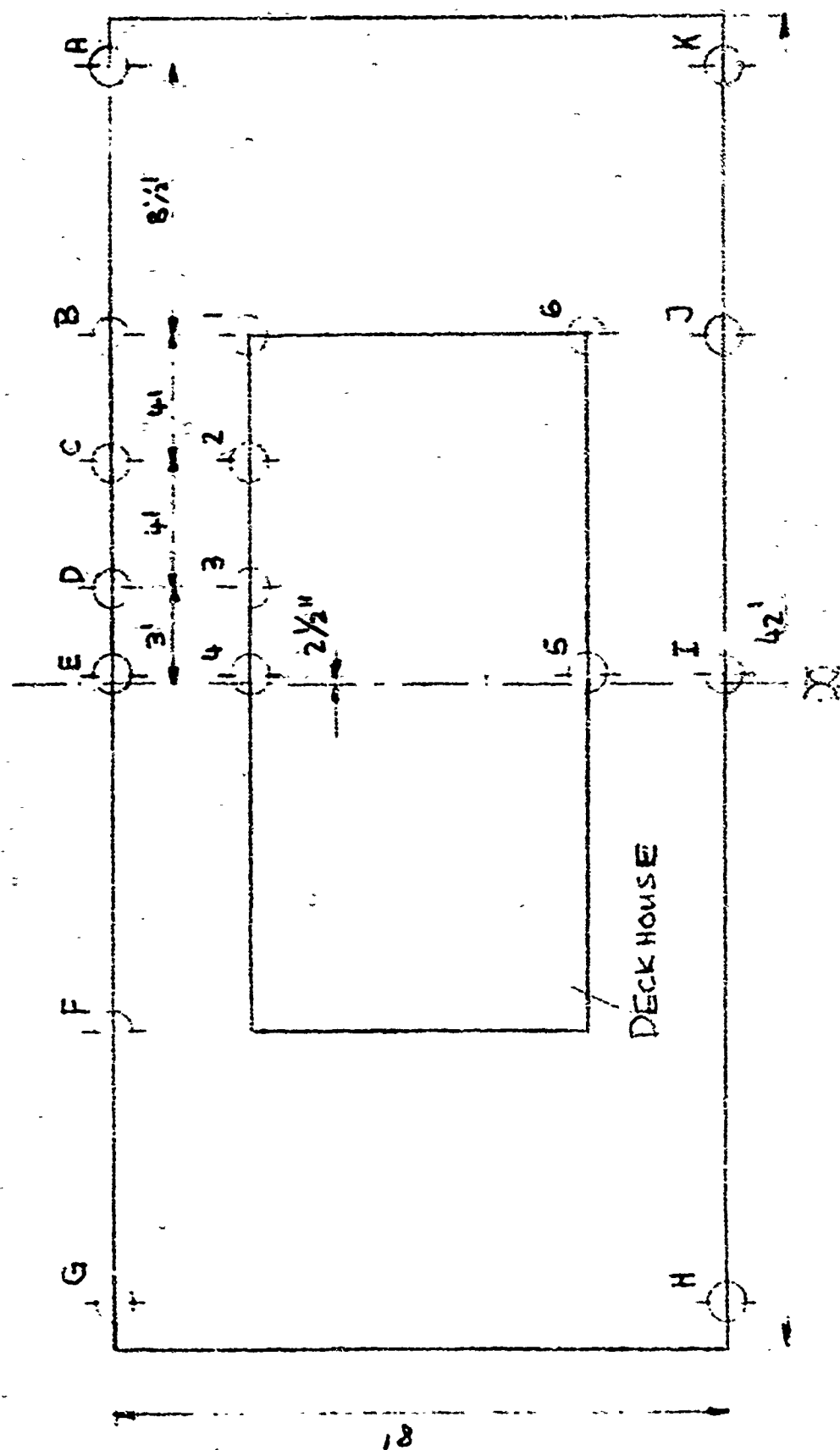


FIGURE 6. LOCATION OF DIAL GAGES

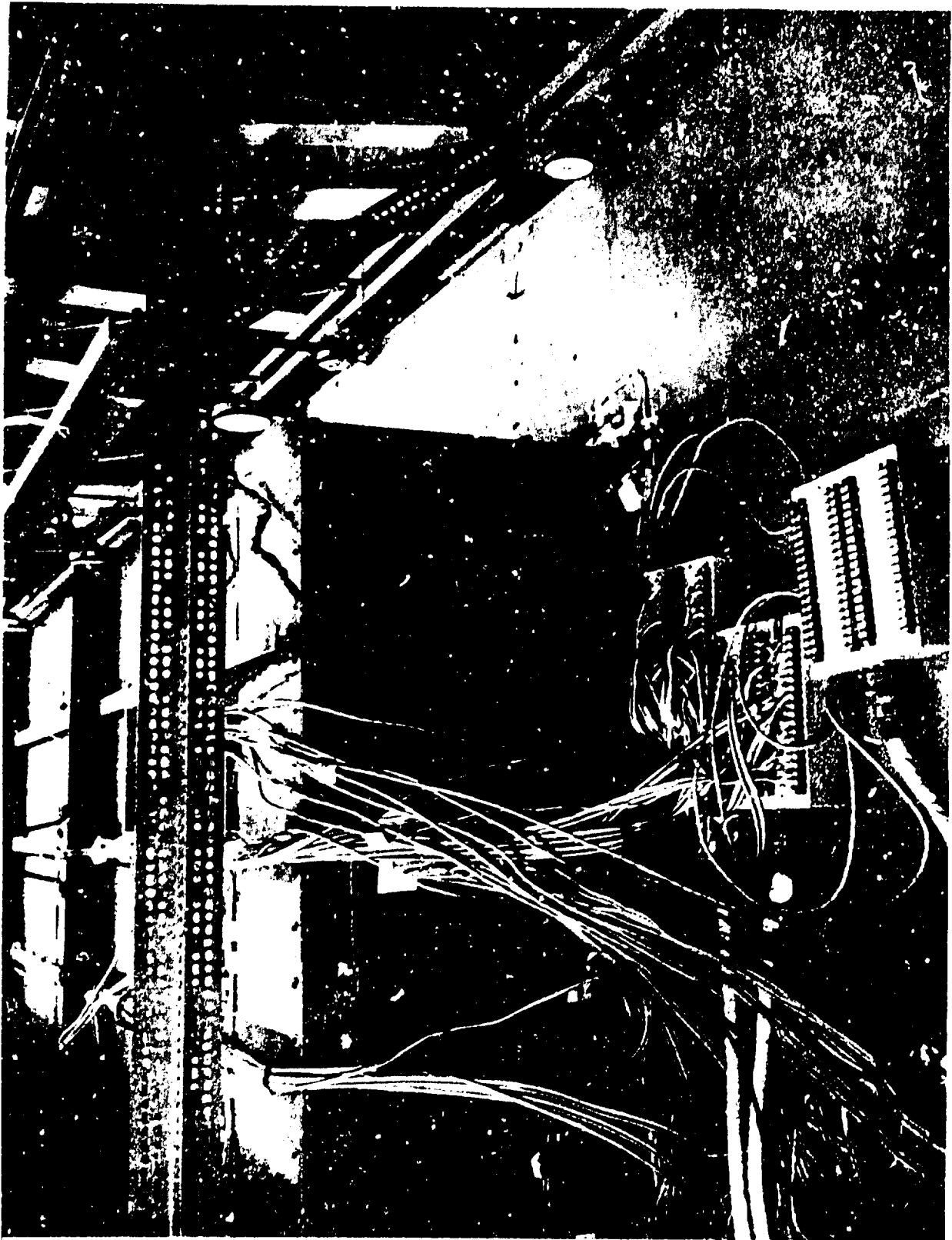


FIGURE 7. DECKHOUSE DEFLECTION INSTRUMENTATION



FIGURE 8 FORCE MONITORING AND PRESSURE CONTROL



FIGURE 9 ELECTRONIC DATA ACQUISITION SYSTEM

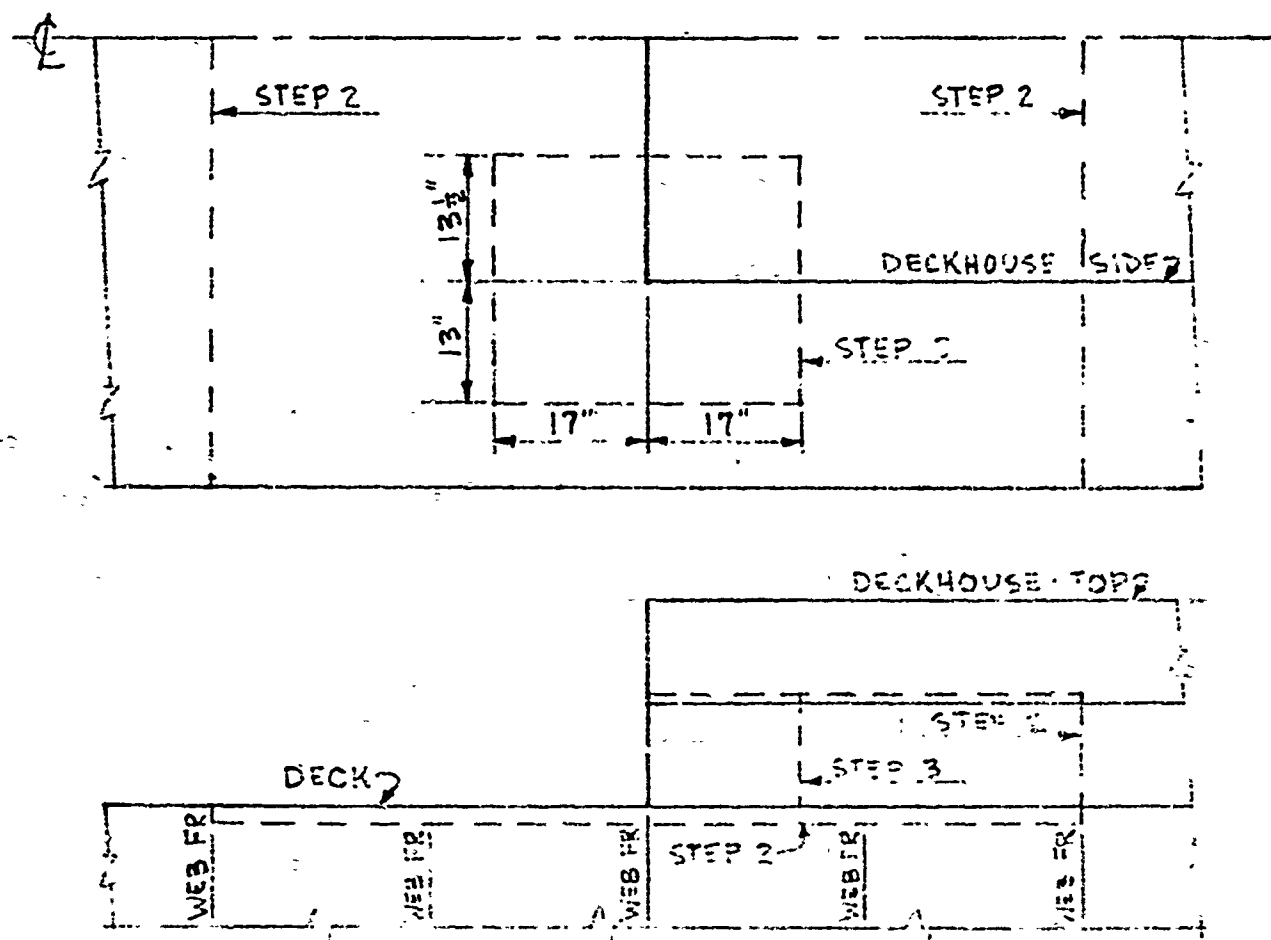


FIGURE 10. FINITE-ELEMENT STEP DESIGNATION

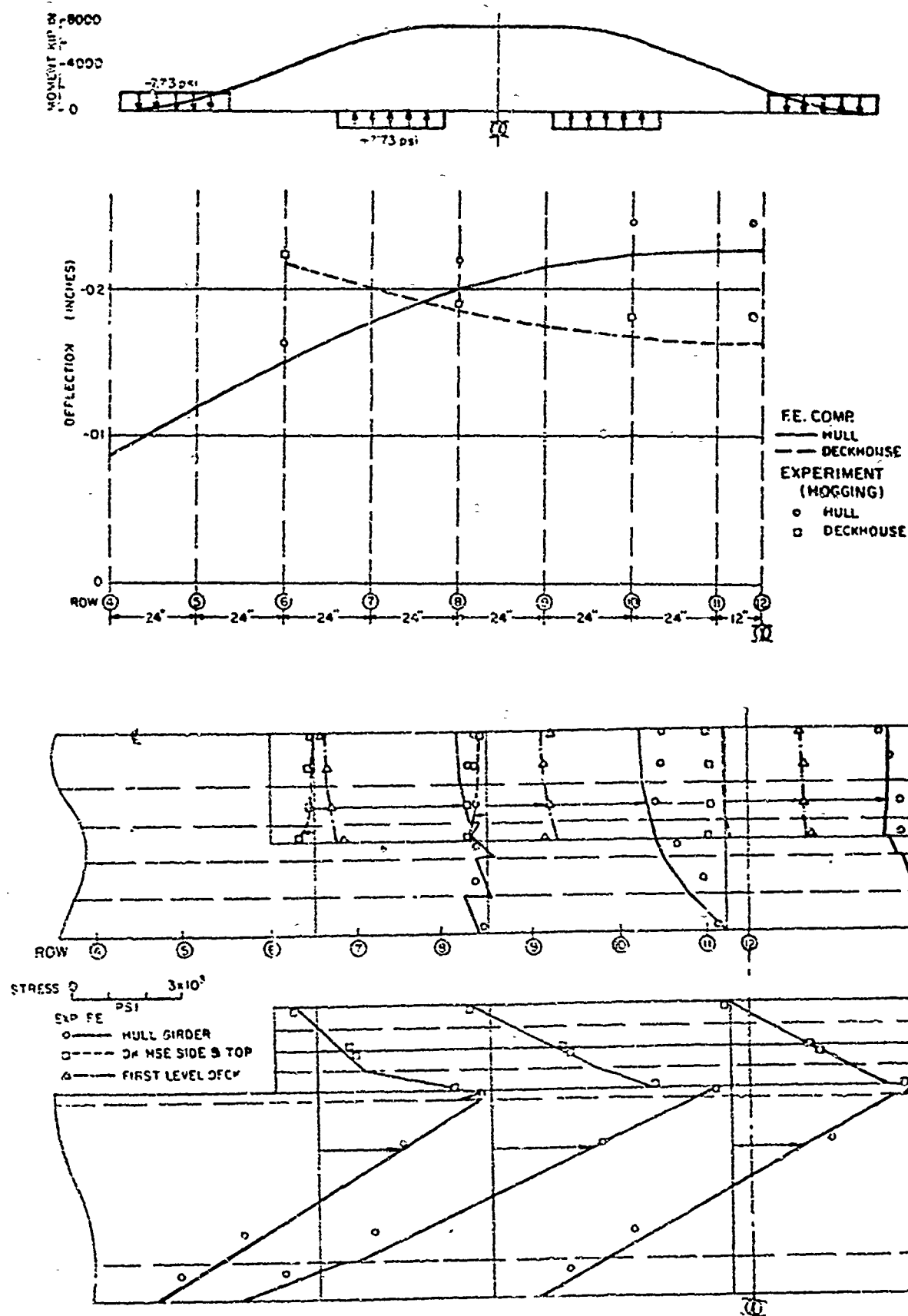


FIGURE 11. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 8AH

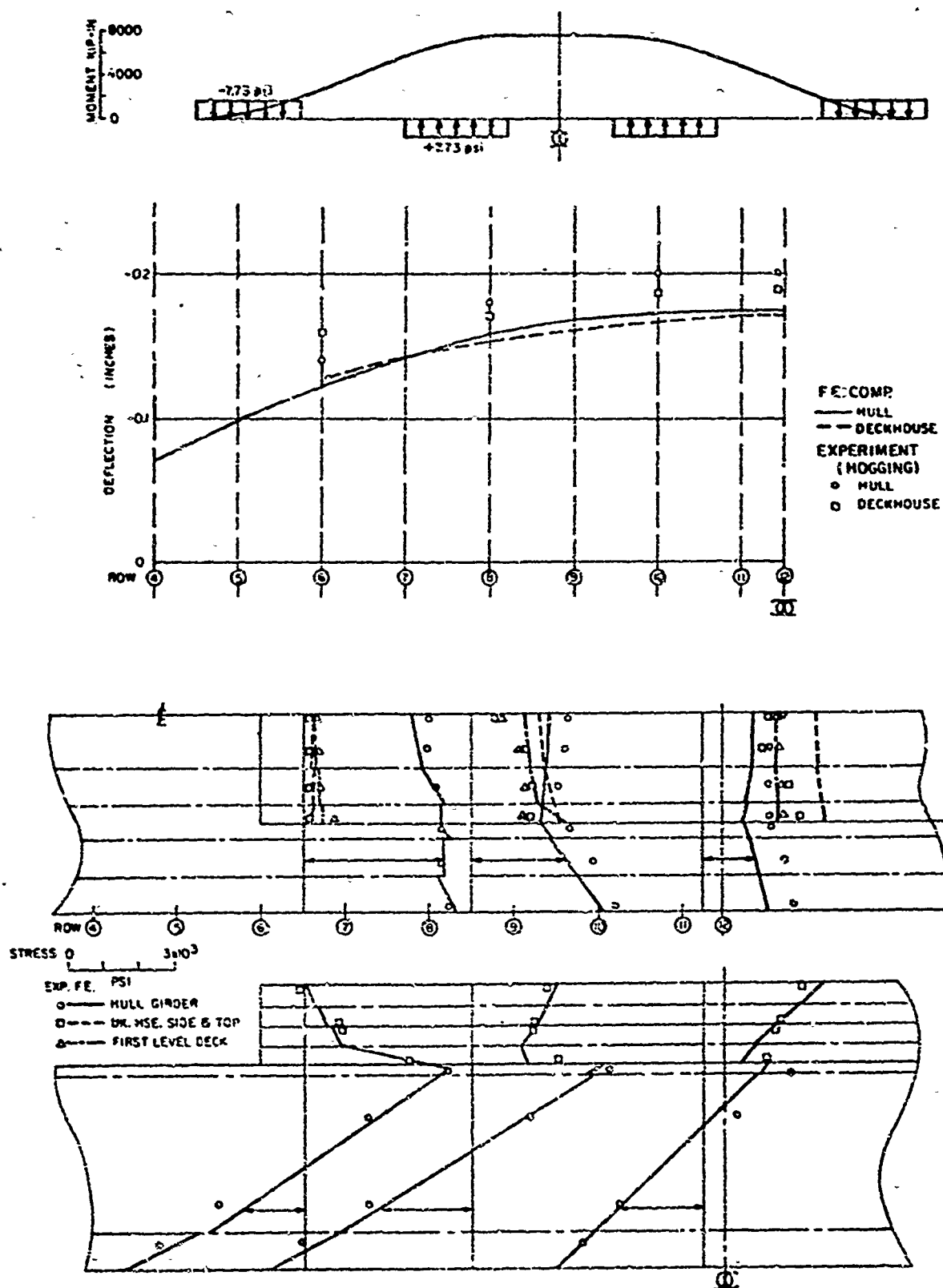


FIGURE 12. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 9AH

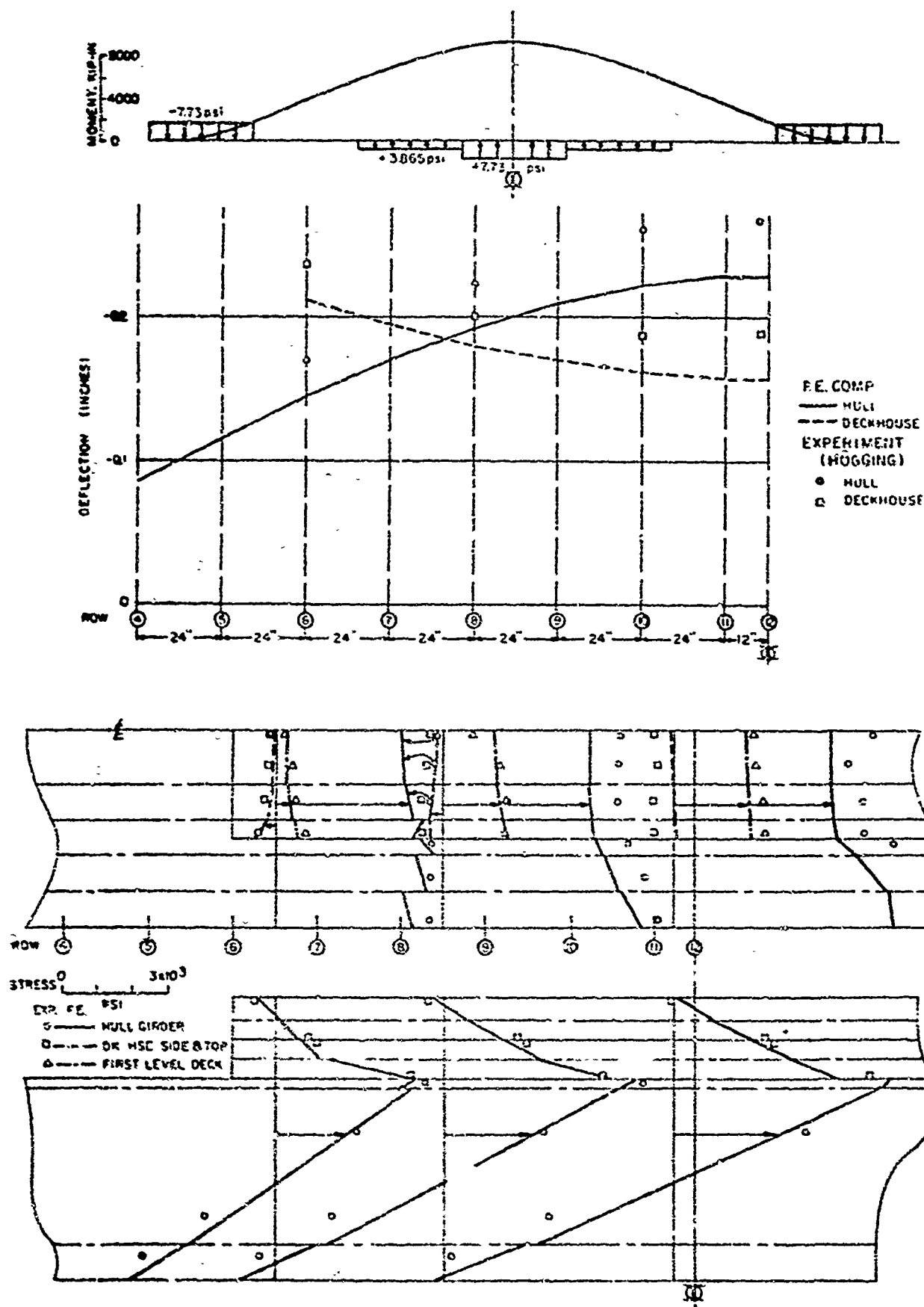


FIGURE 13. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 8BH

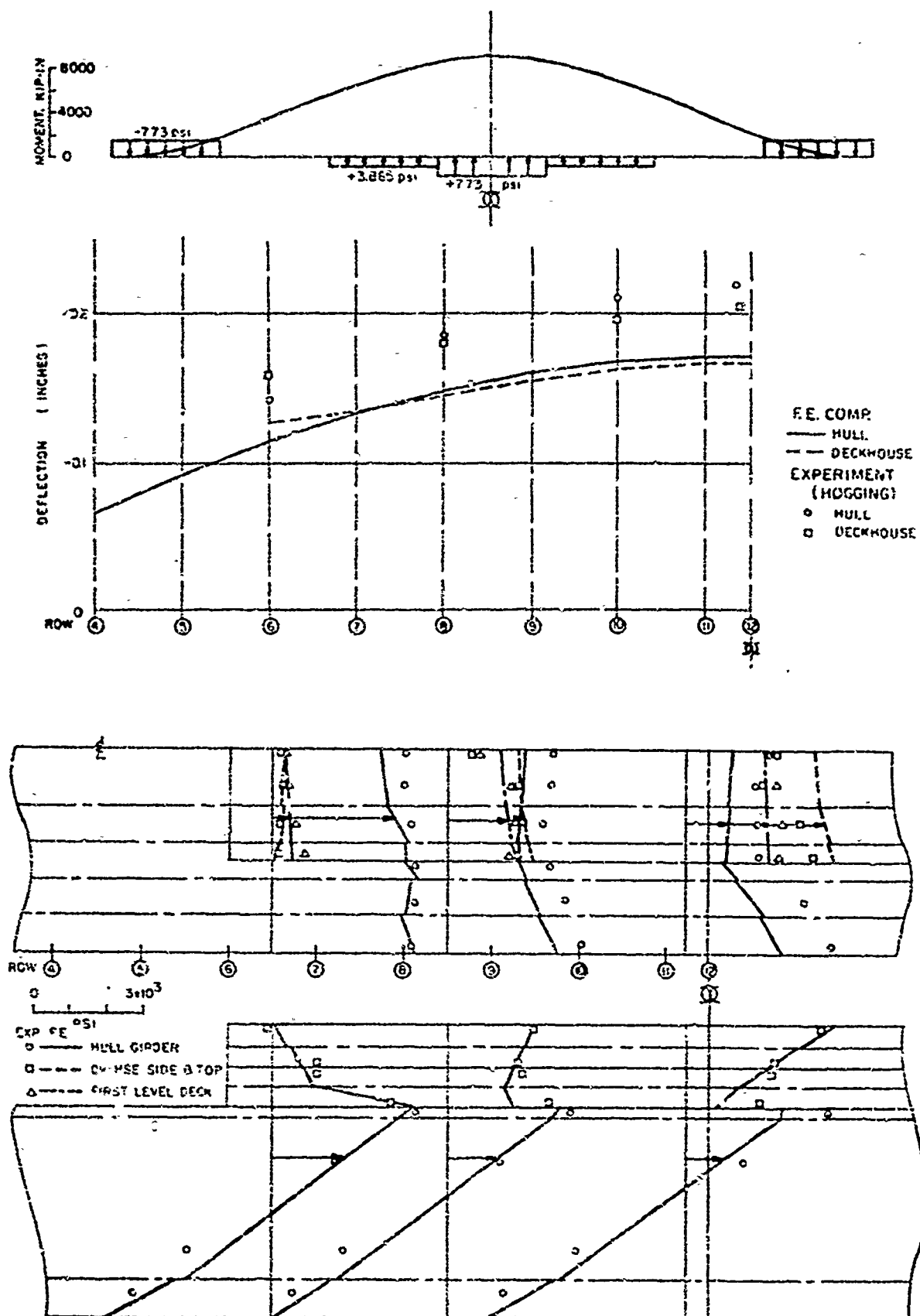


FIGURE 14. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 9BH

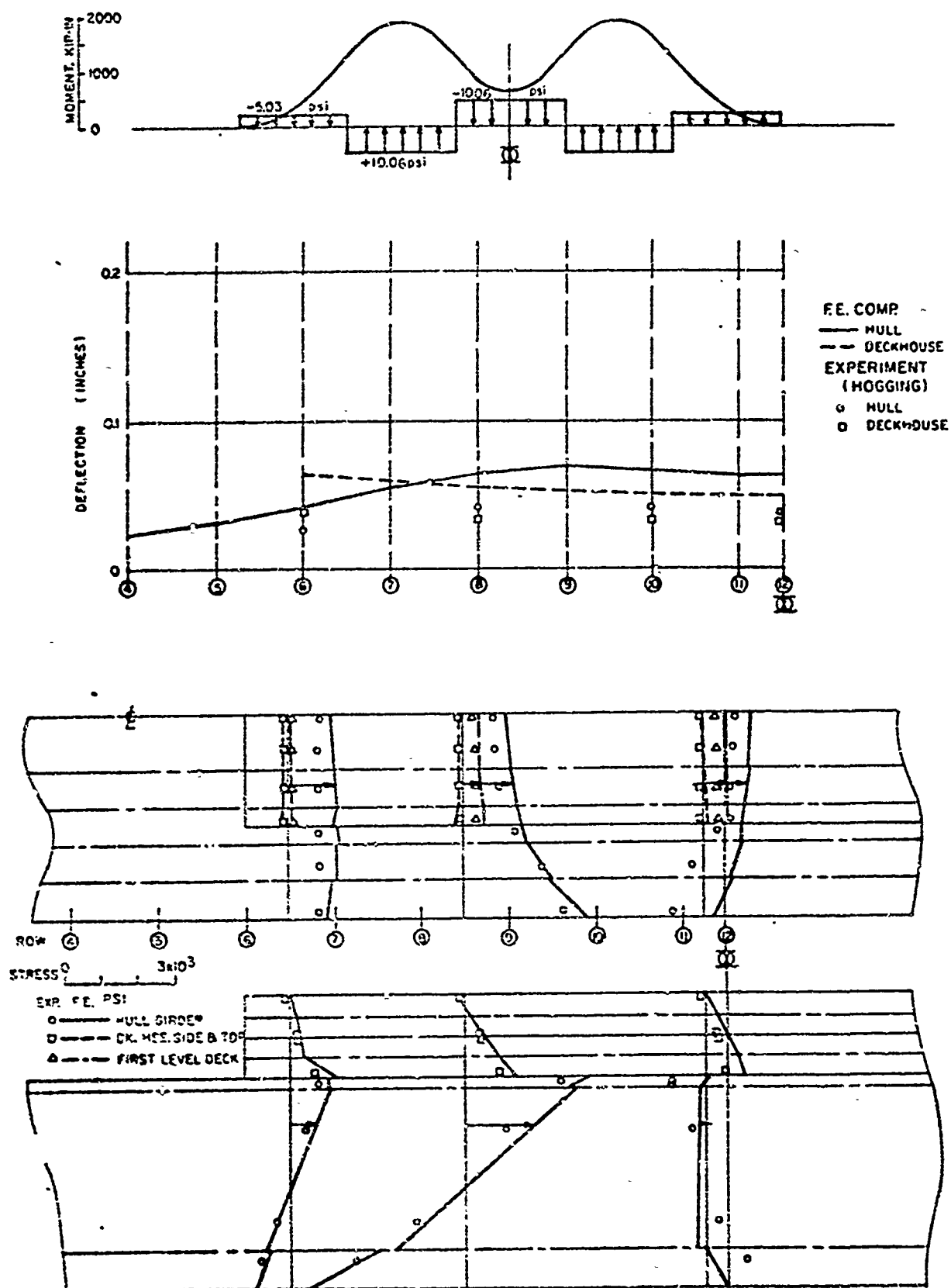


FIGURE 15. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 8CH

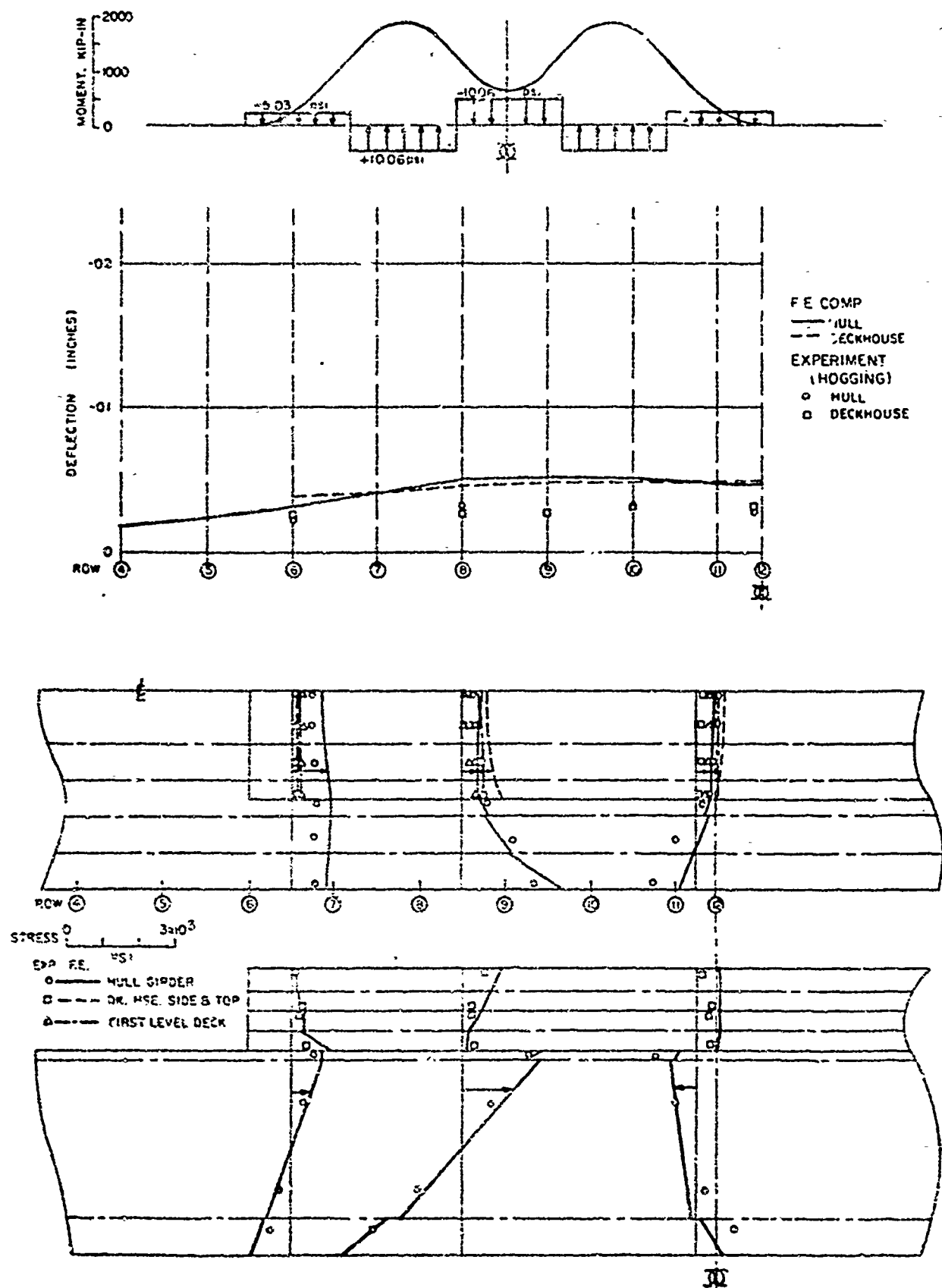


FIGURE 16. DEFLECTIONS AND LONGITUDINAL STRESSES, TEST 9CH

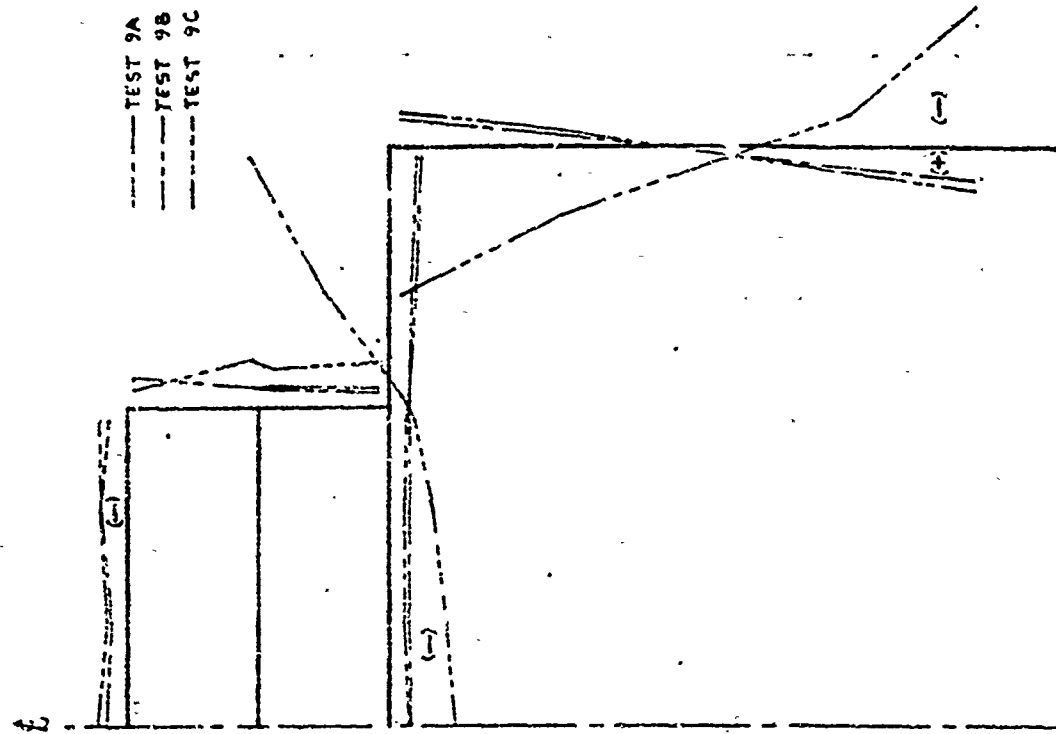


FIGURE 17. STANCHIONS LOOSE

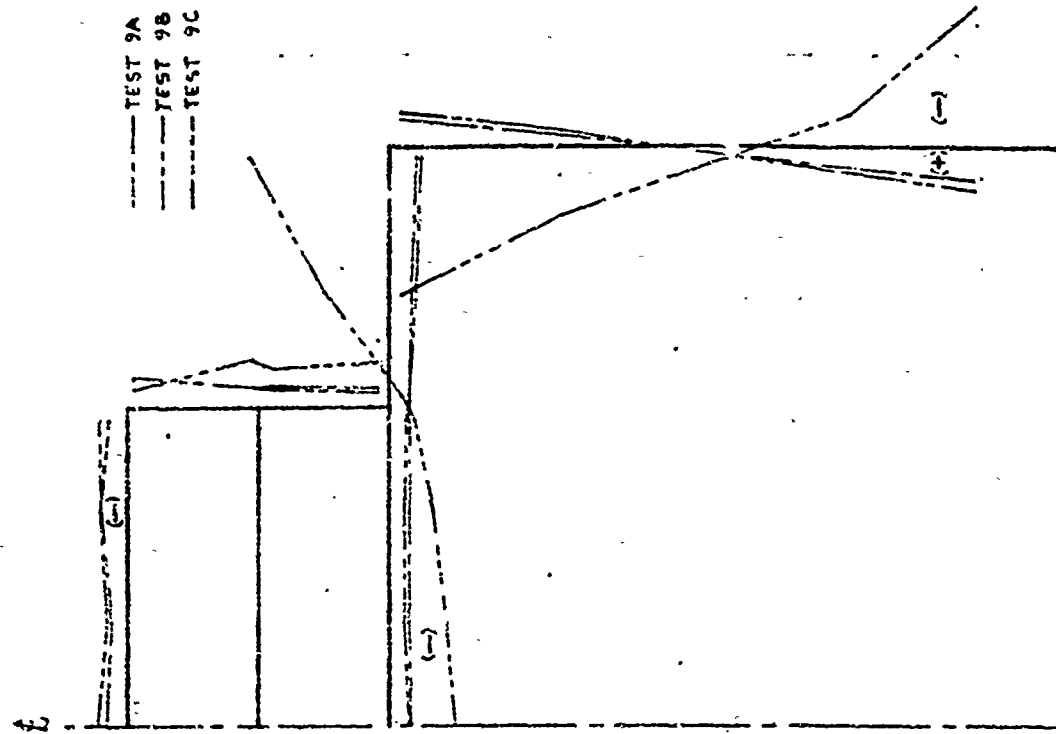


FIGURE 18. STANCHIONS FIXED

LONGITUDINAL STRESS DISTRIBUTION AMIDSHIPS

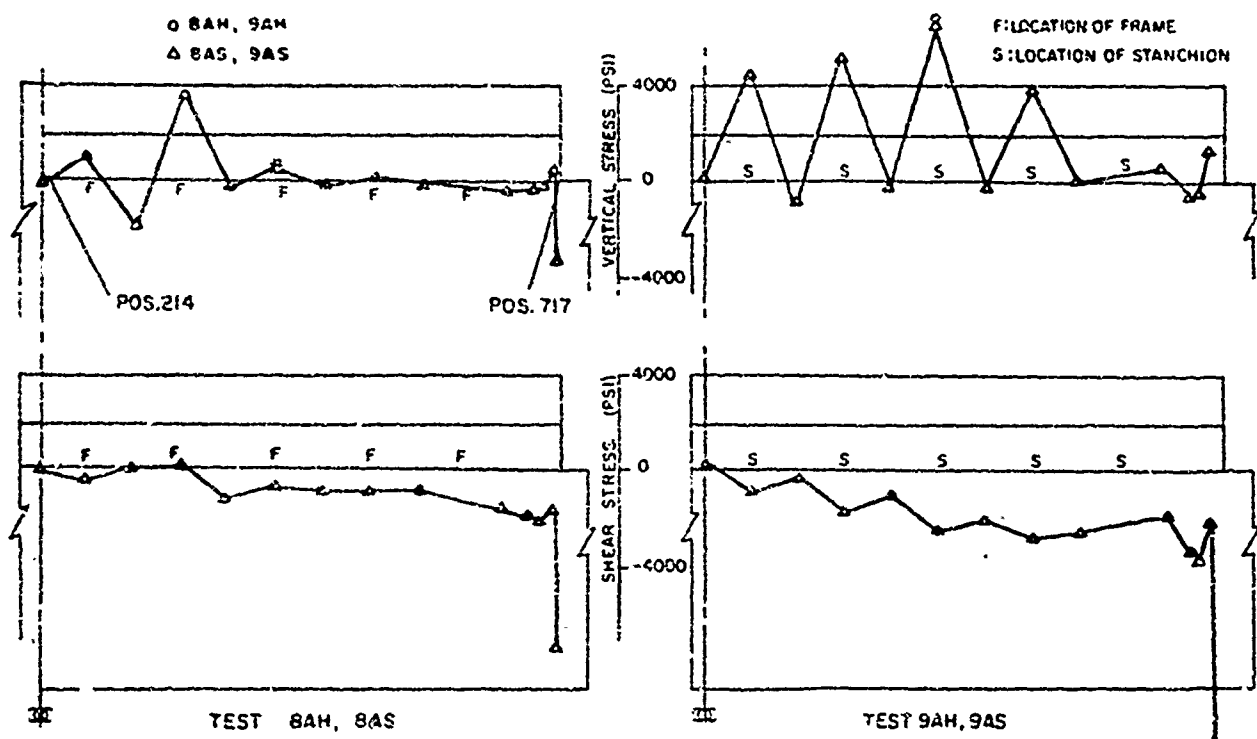


FIGURE 1. SHEAR AND VERTICAL STRESSES AT BOND, TESTS 8A & 9A

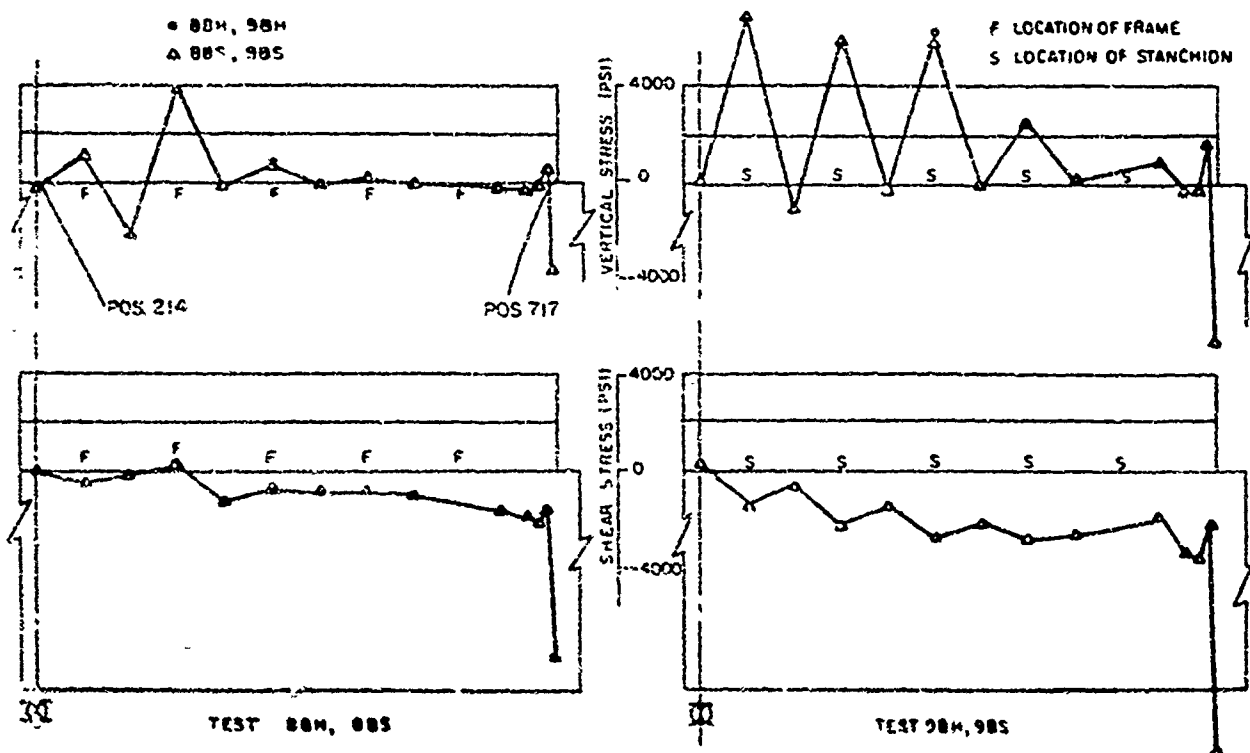


FIGURE 20. SHEAR AND VERTICAL STRESSES AT BOND, TESTS 8B & 9B

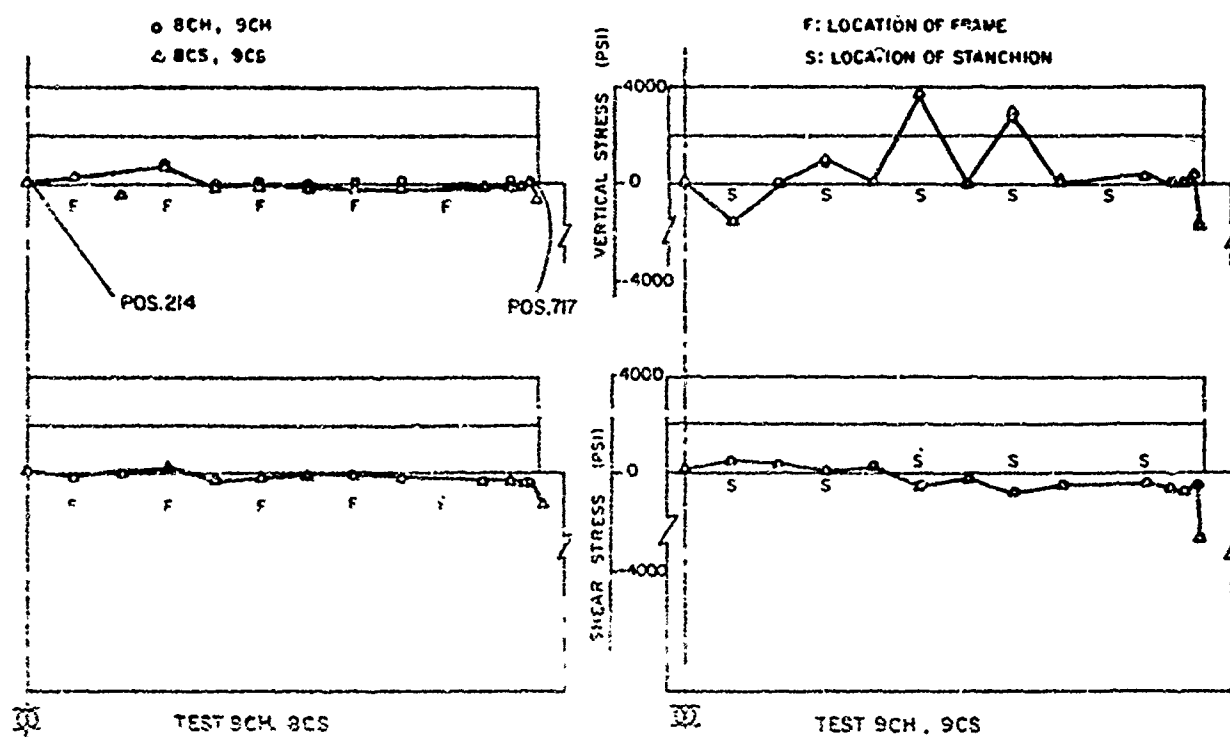


FIGURE 21. SHEAR AND VERTICAL STRESSES AT BOND, TESTS 8C & 9C

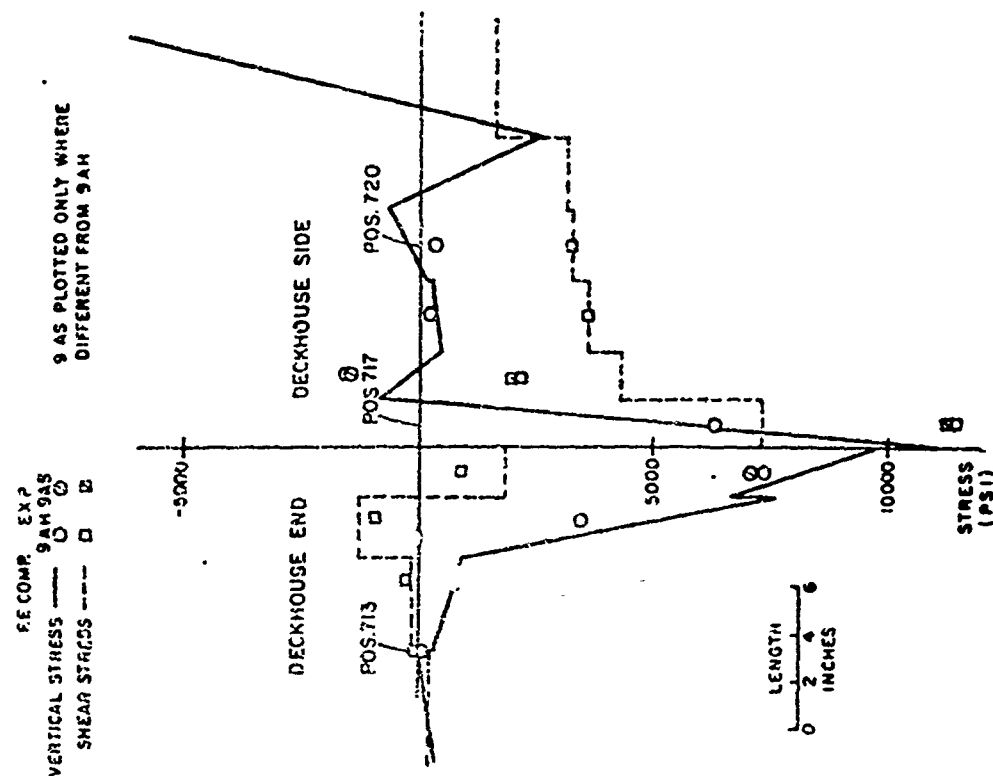


FIGURE 23. TESTS 8AH & 9AS

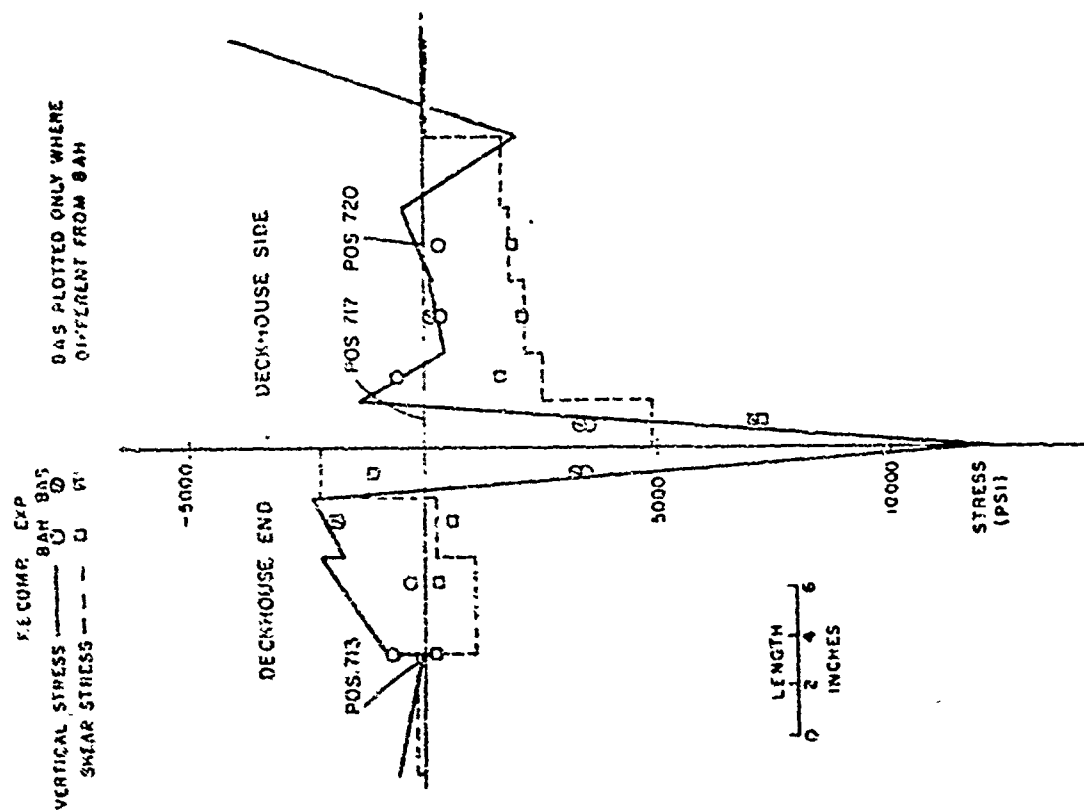


FIGURE 22. TESTS 8AH & 8AS

SHEAR AND VERTICAL STRESSES AT DECKHOUSE END

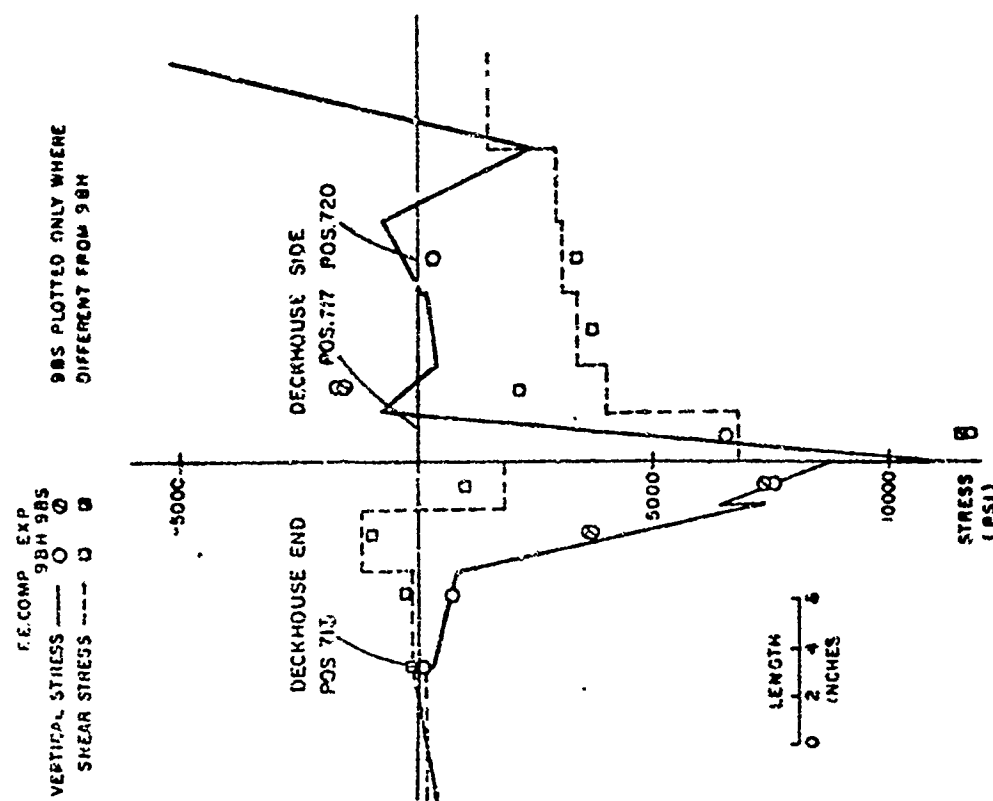


FIGURE 25. TESTS 9BH & 9BS

SHEAR AND VERTICAL STRESSES AT DECKHOUSE END

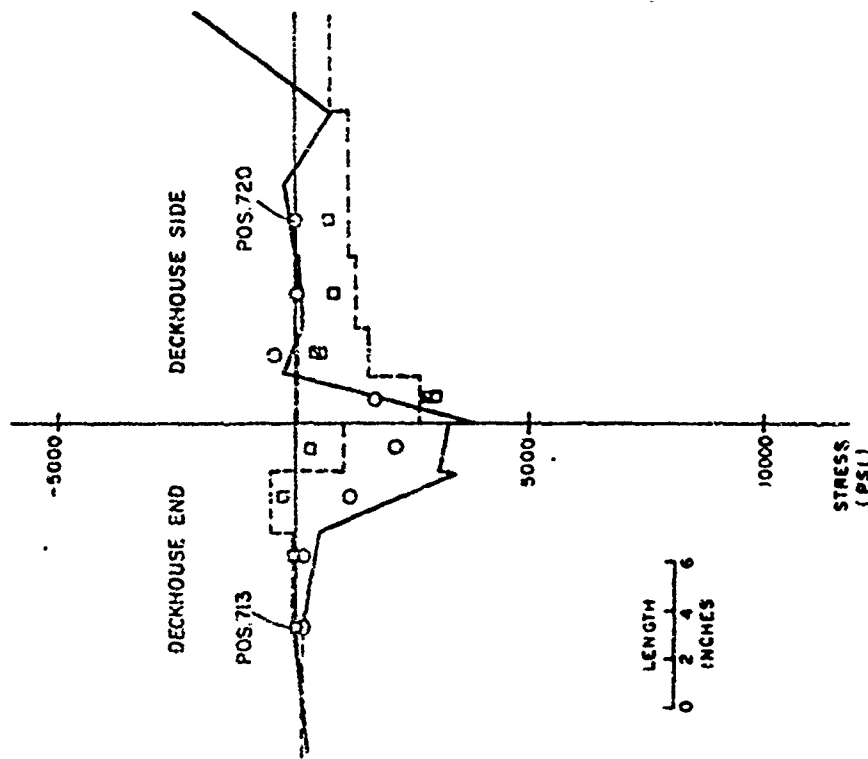


FIGURE 27. TESTS 9CH & 9CS

SHEAR AND VERTICAL STRESSES AT DECKHOUSE END

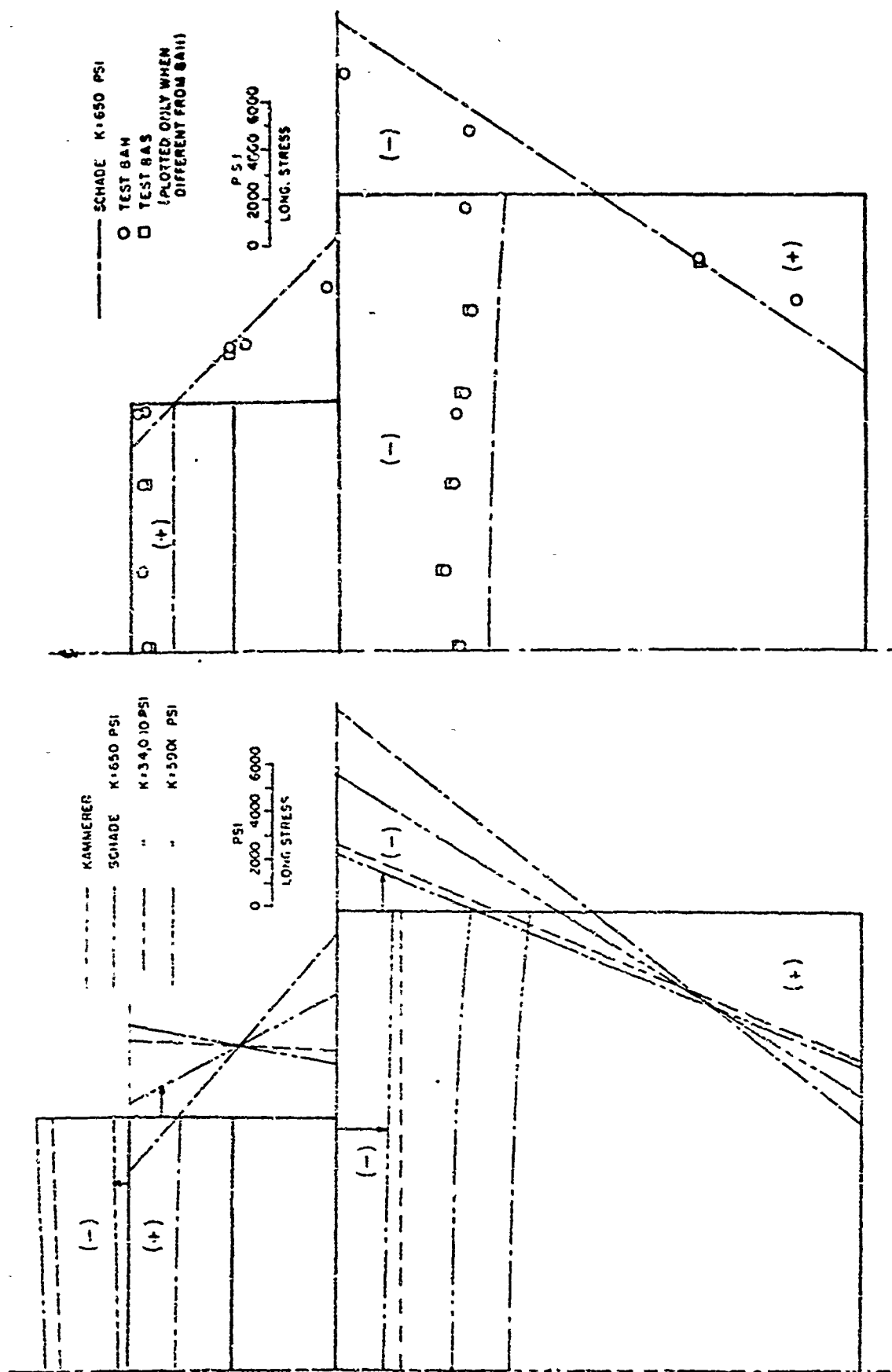


FIGURE 28. COMPARISON OF KAMMERER'S AND SCHADE'S ANALYSIS FOR A TWO-LEVEL DECKHOUSE UNDER BENDING MOMENT B

FIGURE 29. COMPARISON OF TESTS BAH-S AND THEORY AT MIDSHIPS

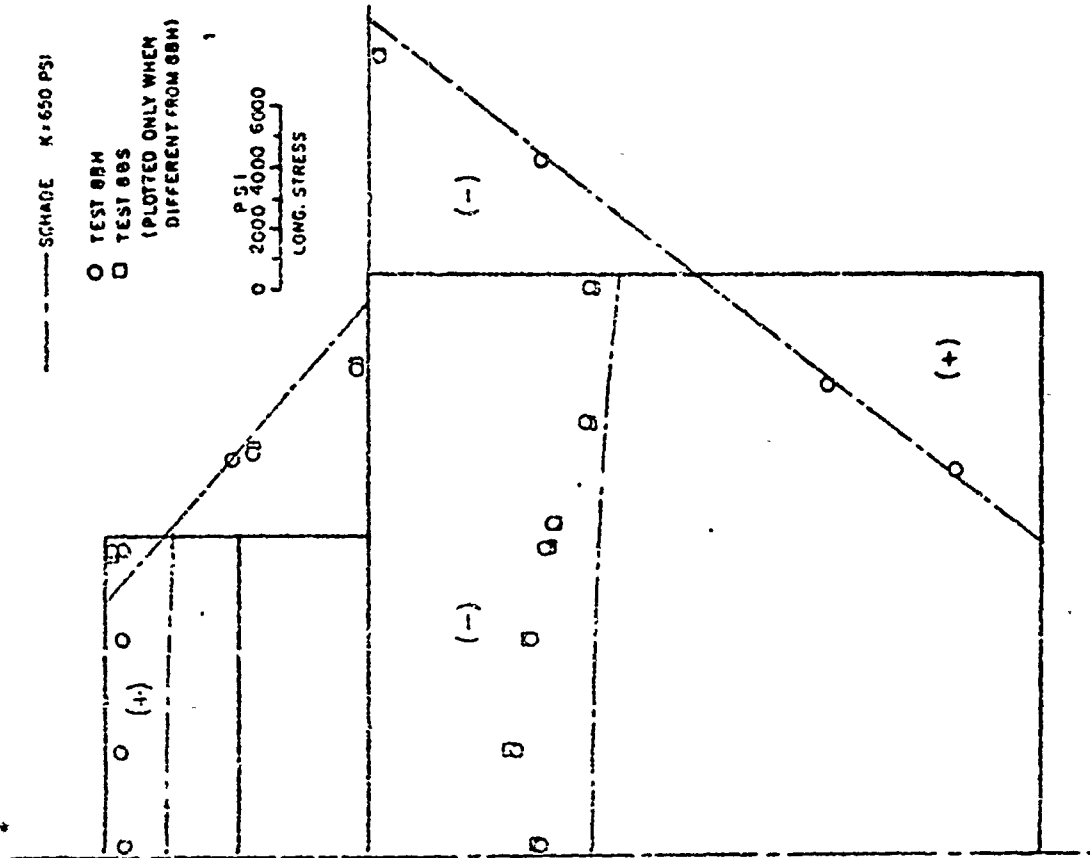


FIGURE 31. COMPARISON OF TESTS 8BH-S AND THEORY AT MIDSHIPS

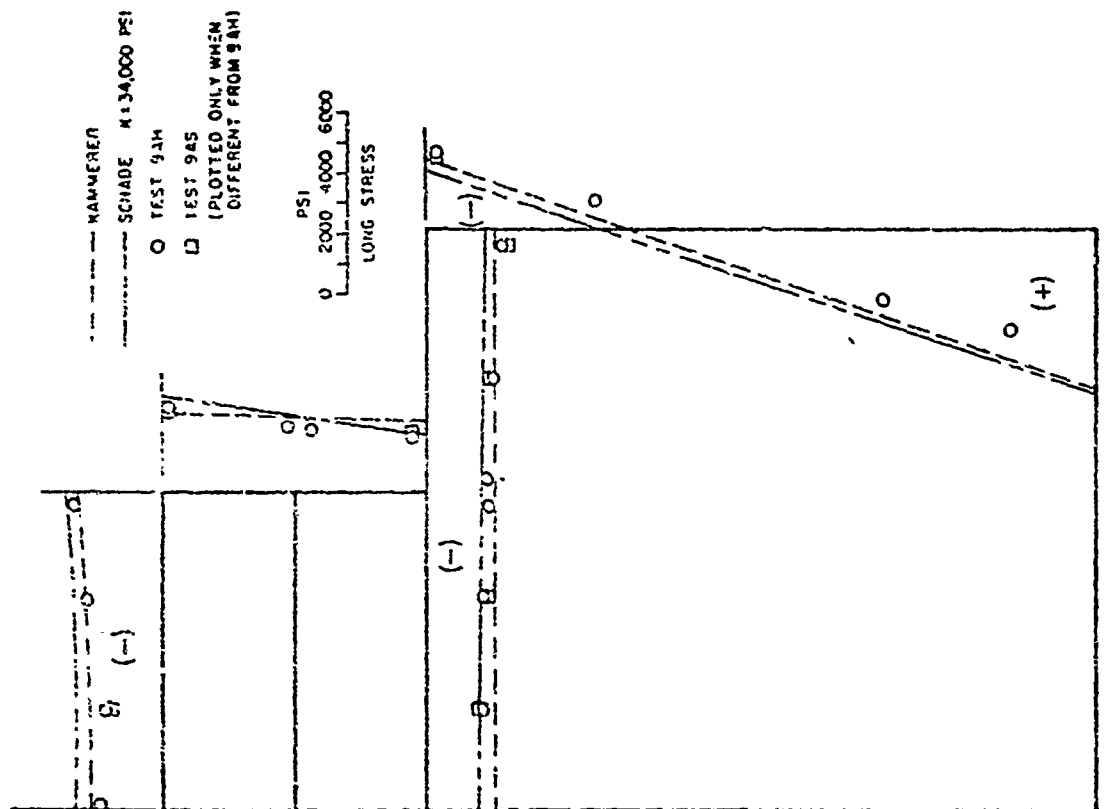


FIGURE 30. COMPARISON OF TESTS 9AH-S AND THEORY AT MIDSHIPS

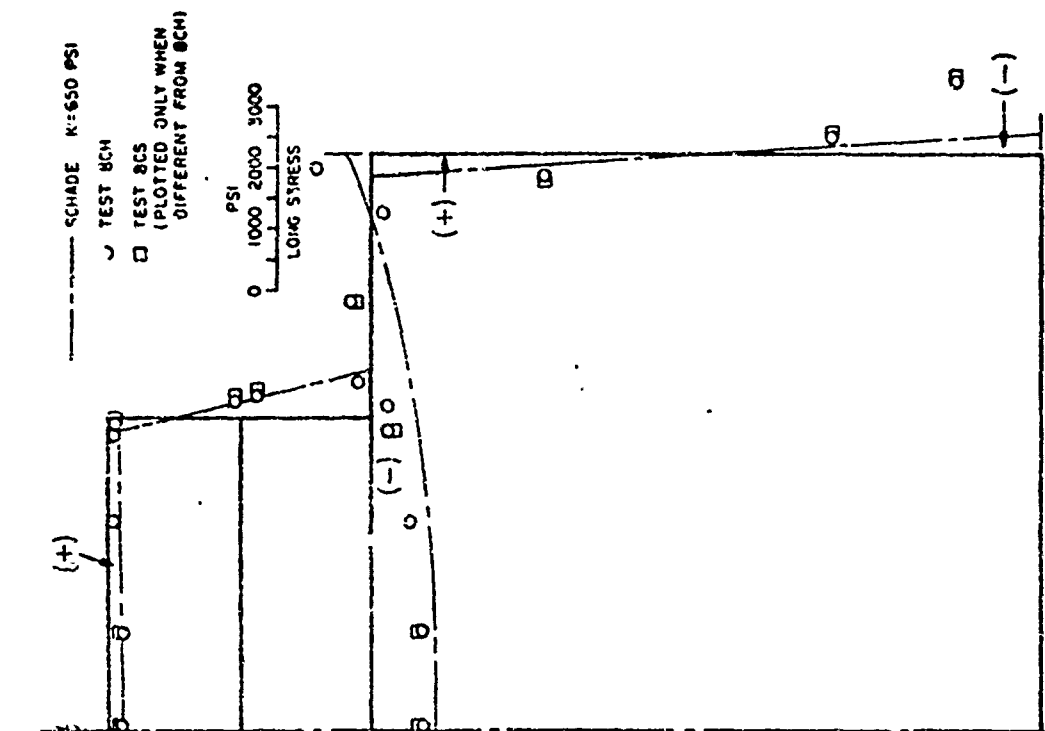


FIGURE 32. COMPARISON OF TESTS 9BH-S AND THEORY AT MIDSHIPS

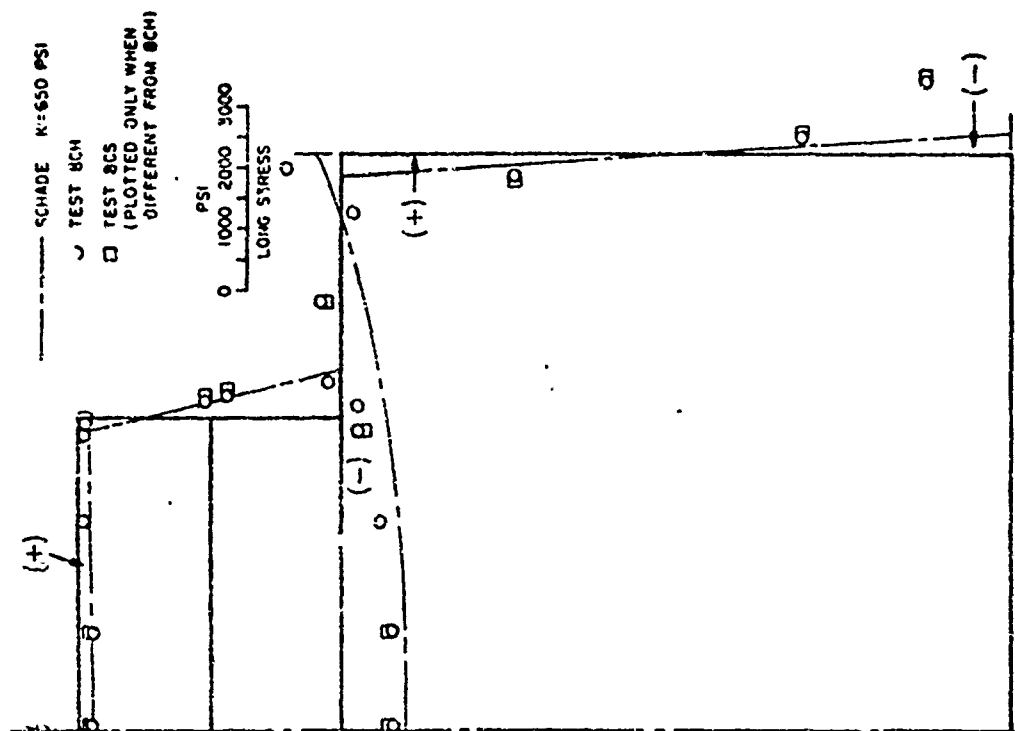


FIGURE 33. COMPARISON OF TESTS 8CH-S AND THEORY AT MIDSHIPS

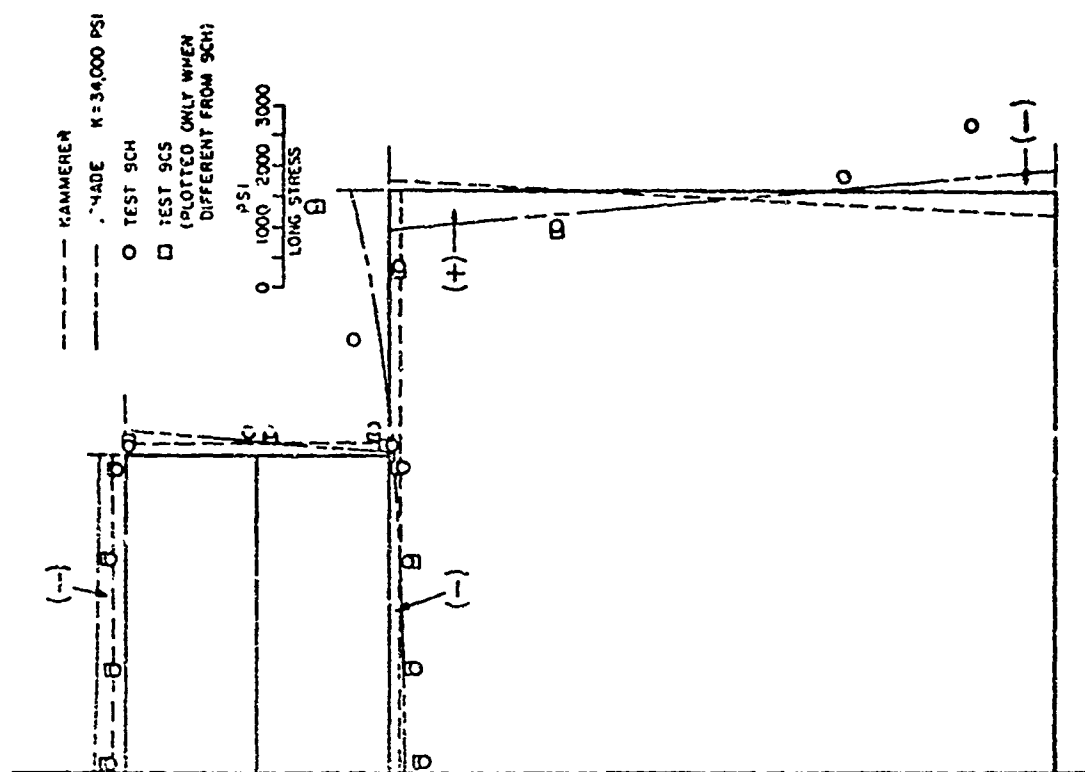


FIGURE 34. COMPARISON OF TESTS
9CH-S AND THEORY AT
MIDSH..PS

ACKNOWLEDGEMENTS

The authors are obliged to Professor J. R. Paulling for the guidance and help given throughout the performance of the project. The following students were actively involved in the performance of the tests and the evaluation of the results: Kwang June Bai, Rong Tsao Huang, and Theodore R. Wise. Their assistance is gratefully acknowledged.

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APPENDIX A

TABLE 3

DEFLECTION DATA, in $\times 10^{-3}$

GAGE	TEST											
	8AH	8AS	8BH	8BS	8CH	8CS	9AH	9AS	9BH	9BS	9CH	9CS
A	11	12	10	13	1	1	8	10	8	1	1	1
B	147	150	154	157	23	22	126	126	129	129	19	18
C	201	201	208	211	40	38	166	166	171	170	32	31
D	224	225	244	246	39	36	184	182	196	196	30	30
E	226	226	249	251	35	32	186	185	203	202	27	26
F	149	149	156	156	24	23	130	128	134	132	20	19
G	11	11	12	11	1	1	10	9	10	9	1	1
H	9	9	9	9	1	1	7	6	7	7	1	1
I	227	227	-	251	35	32	185	183	201	199	27	26
J	147	146	153	153	24	22	124	123	127	126	18	18
K	11	12	12	12	2	2	9	9	9	10	1	1
1	204	203	220	218	34	33	146	146	145	150	24	23
2	171	173	185	183	30	28	157	157	165	161	27	26
3	160	160	173	171	28	26	171	170	182	180	29	28
4	162	162	174	174	28	26	177	175	189	187	29	27
5	159	162	72	173	27	26	174	-	187	-	30	28
6	203	203	217	217	37	35	142	142	146	145	23	22

The sign of the deflections is shown reversed for sagging.

TABLE 4

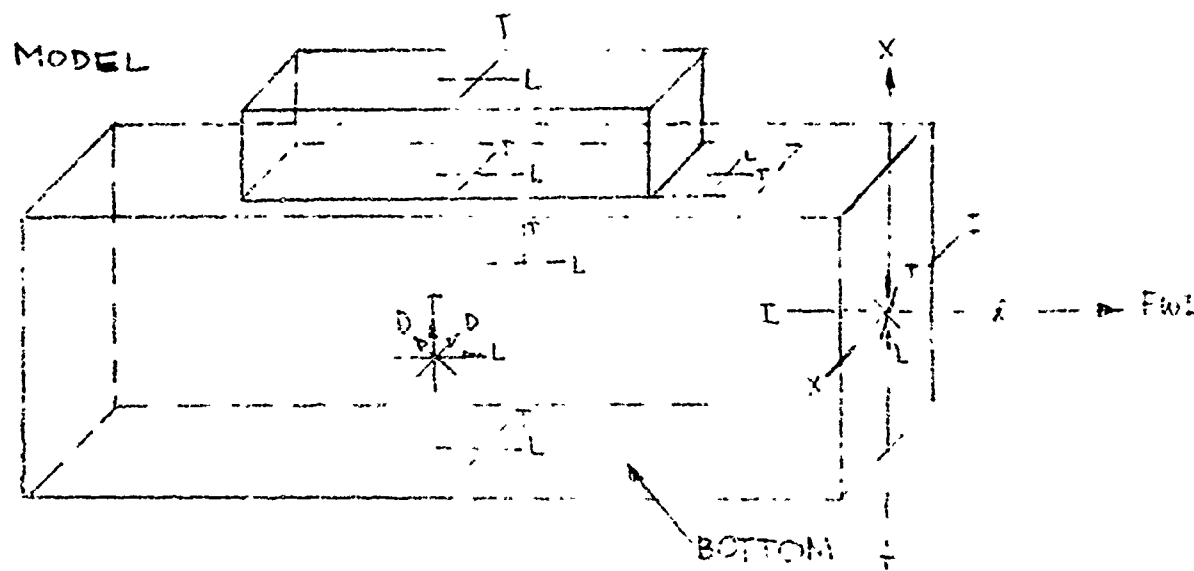
CORRECTED DEFLECTION DATA, in $\times 10^{-3}$

GAGE	TEST											
	8AH	8AS	8BH	8BS	8CH	8CS	9AH	9AS	9BH	9BS	9CH	9CS
A	31	32	25	28	3	3	23	25	23	26	2	2
B	167	170	169	172	25	24	141	141	144	144	20	19
C	221	221	223	226	42	40	181	181	186	185	33	32
D	244	245	259	261	41	38	199	197	211	211	31	31
E	246	246	264	266	37	34	201	200	218	217	28	27
F	169	169	171	171	26	25	145	143	149	147	21	20
G	31	31	27	26	3	3	25	24	25	24	2	2
1	224	223	235	233	36	35	161	161	160	165	25	24
2	191	193	200	198	32	30	172	172	180	179	28	27
3	180	180	188	186	30	28	186	185	197	195	30	29
4	182	182	189	189	30	28	192	190	204	202	30	28
CORREC- TION	+20	+20	+15	+15	+2	+2	+15	+15	+15	+15	+1	+1

The sign of the deflections is shown reversed for sagging.

APPENDIX B

CODING AS USED IN THE COMPUTER PROGRAMS
AND IN TABLES 5 THROUGH 10

Location of Gage

X: Top, Starboard or Forward Side.

I: Bottom, Port or Aft Side.

Direction of Gage

L: Longitudinal

T: Transversal

D: Diagonal

Abbreviations

STRS: Stress

STR: Strain

D: Direct

B: Bending

Conventions

Stresses are given in units of psi, strains in μ in/in.

Tensile stresses appear with negative sign.

Gages located in marked area have different convention as shown.

AMERICAN NATIONAL INPUT, INPUT, PUNCH)

INDEX1 IS A REAL VARIABLE REPRESENTING THE SET OF STRAIN GAGE HEADINGS IN A COMPLETE TEST. INDEX1 MULTIPLIES THE STRAIN GAGE INDEX 1 BY THE TOTAL NUMBER OF STRAIN GAGE ON THE MODEL. INDEX 2 INDICATES THE NUMBER OF RUNS IN ONE TEST. MAXJ INCLUDES THE INITIAL AND FINAL RUNS.

READ 1.000
1 FORMAT(15)
READ 6.0001.000
4 FORMAT(1215)

```

1-1
J-1
GO 3 111210M
HEAD 190437
* FORMAL (1210)
IF (COT-97) 10, JUV
DO 1110MECA, 991 3130
DO 101 1AME MLCURD SAMO
* EXAMINED IN BL USU AS JAV
30 CO 7 M-1012
A. (J-1012)
1-1
1-1
1-1
J-1
1-1
9 1-1 (J-1012) 30 Y0 1
3 CONTINUE

```

```

10 PRINT I,MAXI,MAXJ
11 FORMAT(1H1,NUMBER OF STRAIN GAGES ON MODEL
12 1ER OF TESTING RUNS
13 0.15)
14 PRINT 15
15 FORMAT(1H2,0.0,LINE X(1,0) ARRAY)
16 PRINT 20,1(1(1,0),J=1,MAXJ),1(1,MAXI)
17
18 FORMAT(1/2, (A1X,0A3)/1)
19
20 PUNCH 21,1(1(1,0),J=1,MAXJ),1(1,MAXI)
21
22 FORMAT(1H3,0,0A3)
23
24 STOP
25
26 PRINT 32,11
27
28 FORMAT(1H4,11 IN RECORD,0.15)
29
30 GO TO 34
31
32 END

```

Sample Output, gases 1-50

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611		PAGE 612		PAGE 613		PAGE 614		PAGE 615		PAGE 616		PAGE 617		PAGE 618		PAGE 619		PAGE 620		PAGE 621		PAGE 622		PAGE 623		PAGE 624		PAGE 625		PAGE 626		PAGE 627		PAGE 628		PAGE 629		PAGE 630		PAGE 631		PAGE 632		PAGE 633		PAGE 634		PAGE 635		PAGE 636		PAGE 637		PAGE 638		PAGE 639		PAGE 640		PAGE 641		PAGE 642		PAGE 643		PAGE 644		PAGE 645		PAGE 646		PAGE 647		PAGE 648		PAGE 649		PAGE 650		PAGE 651		PAGE 652		PAGE 653		PAGE 654		PAGE 655		PAGE 656		PAGE 657		PAGE 658		PAGE 659		PAGE 660		PAGE 661		PAGE 662		PAGE 663		PAGE 664		PAGE 665		PAGE 666		PAGE 667		PAGE 668		PAGE 669		PAGE 670		PAGE 671		PAGE 672		PAGE 673		PAGE 674		PAGE 675		PAGE 676		PAGE 677		PAGE 678		PAGE 679		PAGE 680		PAGE 681		PAGE 682		PAGE 683		PAGE 684		PAGE 685		PAGE 686		PAGE 687		PAGE 688		PAGE 689		PAGE 690		PAGE 691		PAGE 692		PAGE 693		PAGE 694		PAGE 695		PAGE 696		PAGE 697		PAGE 698		PAGE 699		PAGE 700		PAGE 701		PAGE 702		PAGE 703		PAGE 704		PAGE 705		PAGE 706		PAGE 707		PAGE 708		PAGE 709		PAGE 710		PAGE 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B.2. Program STRESS

[illegible]

APPENDIX C

TABLE 5. STRESSES AND STRAINS

[illegible]

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SRG TEST

[illegible]

TABLE 9. STRESSES AND STRAINS

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APPENDIX D

A. Coefficients for Expansion of B.M. over Length of Hull

B.M.	Fourier Coef. icients (Sine Series) KIP-IN							
	a ₁	a ₃	a ₅	a ₇	a ₉	a ₁₁	a ₁₃	a ₁₅
A	7332.9	-949.01	-533.85	-0.0342	-91.857	-19.473	3.4403	2.2414
B	7808.1	-1385.1	-249.56	-141.85	-42.920	-28.414	3.6958	2.4009
C	1253.7	-252.29	-460.84	368.70	-79.319	-5.1688	0.5841	0.3867

B. Coefficients for Expansion of B.M. over Length of Deckhouse

B.M.	Fourier Coefficients (Constant + Sine Series) KIP-IN								
	m	a ₁	a ₃	a ₅	a ₇	a ₉	a ₁₁	a ₁₃	a ₁₅
A	4482.3	3625.5	398.64	-37.444	-5.4501	-1.8476	-12.944	-1.0843	-2.2294
B	4482.3	4580.9	-37.637	26.376	-5.5713	-7.4400	-0.0113	-4.3631	-2.2753
C	554.95	939.12	770.88	-137.18	-21.705	-5.8589	-33.633	-3.4325	-8.9022

TABLE 11

APPENDIX E

Computations for Comparison of Kammerer's and Schade's Methods of Deckhouse Analysis with Experimental Data

Basic Nomenclature (Kammerer - Ref. 6)

- A_{DE} = equivalent area of deckhouse
- A_H = area of hull
- B = beam of ship
- C_E = distance from the strength deck to the effective
neutral axis of hull area and equivalent deckhouse
area
- C_H = distance from the strength deck to hull neutral axis
- D_H = depth of hull
- I_E = effective moment of inertia of hull and deckhouse
- I_H = moment of inertia of hull
- K_3 = shear lag factor
- L = length of ship
- M = bending moment generated by external forces
- M_{DE} = statical moment of the equivalent area of the deck-
house about the effective neutral axis
- b = width of deckhouse
- h = height of deckhouse
- l = length of deckhouse
- l_E = effective length of deckhouse
- σ_D = longitudinal stress at the strength deck
- σ_K = longitudinal stress at the keel
- σ_T = longitudinal stress at top of deckhouse

Basic Nomenclature (Schade - Ref. 10)

The external loading is applied to the primary member (the hull) identified by the subscript 1; the "following" member (deckhouse or superstructure) is identified by the subscript f.

σ_1 and σ_f	longitudinal stress in sides (x-direction)
$\bar{\sigma}_1$ and $\bar{\sigma}_f$	longitudinal stress in sides at deck (x-direction)
p_1 and p_f	average longitudinal stress (x-direction)
M_1 and M_f	stress moment
S_1 and S_f	section modulus referred to deck
E_1 and E_f	material modulus - normal
G_1 and G_f	material modulus - shear
A_1 and A_f	effective section area including webs and flanges ($A = A_1 + A_f$)
a_1 and a_f	section area (webs only)
I_1 and I_f	section moment of inertia
q_1 and q_f	vertical load/unit length ($q = q_1 + q_f$)
w_1 and w_f	vertical deflection (+ down)
Q_1 and Q_f	vertical shear force ($Q = Q_1 + Q_f$)
e_1 and e_f	distance from bond to individual N.A. ($e = e_1 + e_f$)
z_1 and z_f	vertical coordinate distance from individual N.A. (+ down)
2b	beam of hull
y	transverse coordinate distance from centerline
k	spring constant of hull at bond
$\bar{N}$	longitudinal shear flow in deckhouse at bond
(k)	shear deflection factor
M	bending moment generated by external forces
m	constant component of bending moment

- a amplitude of sinusoidal component of bending moment
- F_x longitudinal external force on hull
- 2λ length of deckhouse
- ρ effectiveness ratio

Parameters Used for Computations

$$B \equiv \frac{A_1 r S_f}{(A_1 + r A_f) S_1 S_f + A_1 A_f (e_f S_1 + e_1 r S_f)}, \text{ in.}^{-3} \quad [E-1]$$

$$C \equiv \frac{A_1 (r e_1 + e_f)}{(A_1 + r A_f) (I_1 + I_f) + A_1 A_f e (r e_1 + e_f)}, \text{ in.}^{-3} \quad [E-2]$$

$$\delta \equiv 1 + \frac{\eta}{\omega^2} \left(\frac{\pi}{2\lambda} \right)^2 + \frac{1}{4\omega^4} \left(\frac{\pi}{2\lambda} \right)^4 \quad [E-3]$$

$$\theta \equiv 1 + \frac{E}{r e_1 + e_f} \left[\frac{I_1 e_f}{a_1 G} + \frac{r I_f e_1}{a_f G} \right] \left(\frac{\pi}{2\lambda} \right)^2 + \frac{E}{r e_1 + e_f} \frac{r I_f e_1}{k} \left(\frac{\pi}{2\lambda} \right)^4 \quad [E-4]$$

$$\omega \equiv \sqrt{\frac{k}{4r I_f} \frac{(r e_1 + e_f)}{E e_1} \frac{B}{C}}, \text{ in.}^{-1} \quad [E-5]$$

$$\eta = \frac{1}{2G} \left(\frac{1}{a_1} + \frac{1}{a_f} \right) \sqrt{\frac{k E r I_f e_1}{(r e_1 + e_f) \frac{C}{B}}} \quad [E-6]$$

$$J^2 = \frac{\omega^2}{\eta} = \frac{1}{\frac{E}{G} \left(\frac{1}{a_1} + \frac{1}{a_f} \right)} \frac{(A_1 + A_f r) (I_1 + I_f) + A_1 A_f e (r e_1 + e_f)}{(A_1 + A_f r) I_1 I_f + A_1 A_f (I_1 e_f^2 + I_f e_1^2 r)}, \text{ in.}^{-2} \quad [E-7]$$

$$r = \frac{1}{2} \frac{1}{\cosh \frac{\pi b}{2\lambda}} \left[\frac{\pi Y}{2\lambda} \sinh \frac{\pi Y}{2\lambda} + 2 \cosh \frac{\pi Y}{2\lambda} - \frac{\pi b}{2\lambda} \tanh \frac{\pi b}{2\lambda} \cosh \frac{\pi Y}{2\lambda} \right] \quad [E-8]$$

ψ, ϕ Parameters for numerical solution for p_f .

Equations

$$M = m + a \sin \frac{\pi x}{2\lambda} \quad [\text{E-9}]$$

$$P_f = -C\psi m + [(B\delta - C\theta)\phi - B \sin \frac{\pi x}{2\lambda}]a \quad [\text{E-10}]$$

$$\bar{\sigma}_1 = - \frac{1}{S_1 - rS_f} \left[\left(\frac{A_f}{A_1} S_1 + S_f + A_f e \right) P_f + M \right] \quad [\text{E-11}]$$

$$\bar{\sigma}_f = r \bar{\sigma}_1 \quad [\text{E-12}]$$

$$\sigma_1 = \left(1 + \frac{z_1}{e_1} \right) p_1 - \frac{z_1}{e_1} \bar{\sigma}_1 \quad [\text{E-13}]$$

$$\sigma_f = \left(1 - \frac{z_f}{e_f} \right) p_f + \frac{z_f}{e_f} \bar{\sigma}_f \quad [\text{E-14}]$$

$$p_1 = - \frac{A_f}{A_1} p_f \quad [\text{E-15}]$$

DETAILS OF CALCULATION

A. Application of Kammerer's Method to the Model

The hull girder of the model was designed to simulate the midship portion of a Mariner and constructed on a 1:9.5 scale. The corresponding full-scale dimensions are:

$$\begin{aligned} L &= 528' \\ B &= 76' \\ D_H &= 44.5' \end{aligned}$$

The geometric properties of the hull girder, considered as a simple beam, are:

$$\begin{aligned} A_H &= 50.35 \text{ in}^2 \\ I_H &= 29,553 \text{ in}^4 \\ C_H &= 29.49 \text{ in } (C_H = 23.35' \text{ in full-scale}) \end{aligned}$$

The corresponding full-scale dimensions of the deckhouse are:

$$\begin{aligned} \lambda &= \lambda_E = 209.15' \\ b &= 41.76' \\ h &= 17.76' \\ \ell/L &= 0.396 \\ b/\lambda &= 0.200 \\ \sigma_T/\sigma_D &= 1.33 \text{ for } C_H = 15' \text{ (Kammerer's Fig. 7)} \\ \sigma_T/\sigma_D &= 1.33 \times 0.842 = 1.13 \text{ for } C_H = 23.35' \\ &\quad \text{(Kammerer's Fig. 8)} \\ K_3 &= 0.943 \text{ (Kammerer's Fig. 10)} \end{aligned}$$

The geometric properties of the deckhouse are:

Item	Actual Area [in ²]	Factors	Equiv. Area [in ²]	Lever [in]	Moment [in ² -in]
2nd Level Dk	10.41	1.13 x 0.943	11.09	22.3	247.3
T-Bars	1.58	1.13 x 0.943	1.68	23.0	38.6
Angles	1.05	1.13	1.19	21.8	25.9
Side Plating	4.31	1.09	4.70	16.1	75.7
1st Level Dk	9.75	1.06 x 0.943	9.75	11.0	107.2
T-Bars	1.58	1.06 x 0.943	1.58	11.7	18.5
Angles	1.05	1.06	1.11	10.5	11.7
Side Plating	3.75	1.03	3.86	5.0	19.3

$$A_{DE} = 34.96$$

$$*C_E = 17.4 \quad \underline{608.3}$$

$$M_{DE} = 1152.5$$

$$*C_E = \frac{A_H C_H}{A_H + A_{DE}} = \frac{50.35 \times 29.49}{50.35 + 34.96} = 17.4 \text{ in}$$

$$I_E = I_H + A_H (C_H - C_E)^2 + M_{DE} C_E$$

$$= 29,553 + 50.35(29.5 - 17.4)^2 + 1152.5(17.4)$$

$$I_E = 56,979 \text{ in}^4$$

For bending moment B, $M = 9.123 \times 10^6$ lbs-in at midship.

Therefore:

$$\sigma_D = \frac{-MC_E}{I_E} = \frac{-9.123 \times 10^6 \text{ lb-in} (17.4 \text{ in})}{56,979 \text{ in}^4} = -2786 \text{ psi}$$

$$\sigma_T = \left(\frac{\sigma_T}{\sigma_D} \right) \sigma_D = 1.13(-2786 \text{ psi}) = -3148 \text{ psi}$$

$$\sigma_K = \frac{M(D_H - C_E)}{I_E} = \frac{9.123 \times 10^6 \text{ lb-in} (56.0 - 17.4) \text{ in}}{56,979 \text{ in}^4} = +6180 \text{ psi.}$$

Above results are plotted in Figure 28.

B. Application of Schade's Method to the Model

$$m = 4.482 \times 10^6 \text{ lbs-in, } a = 4.641 \times 10^6 \text{ lbs-in (moment B)}$$

$$k = 6.50 \times 10^2 \text{ psi (all stanchions loose)}$$

$$k = 3.40 \times 10^4 \text{ psi (all stanchions fixed)}$$

Effective Geometric Properties:

		For Constant B.M.	For Sinusoidal B.M.
r	[Eq.E-8]	1.00 [Ref.5]	0.81
ρ_1	[Ref. 8]	1.00	0.82
a_1	in^2	17.56	17.56
A_1	in^2	50.35	44.45
e_1	in	29.49	29.39
I_1	in^4	29,553	25,008
S_1	in^3	1002.1	850.9
ρ_f	[Ref. 8]	0.93	0.93
a_f	in^2	10.16	10.16
A_f	in^2	31.85	31.85

Effective Geometric Properties: (cont.)

		For Constant B.M.	For Sinusoidal B.M.
e_f	in	15.34	15.34
I_f	in^4	1221	1221
S_f	in^3	79.6	79.6

Computation Parameters

B	[Eq.E-1]	1.146×10^{-4}	1.106×10^{-4}
C	[Eq.E-2]	3.923×10^{-4}	4.026×10^{-4}
J	[Eq.E-7]	2.996×10^{-2}	3.021×10^{-2}
k		6.50×10^2	3.40×10^4
ω	[Eq.E-5]	6.663×10^{-3}	1.792×10^{-2}
$2\lambda\omega$		1.76	4.73
n	[Eq.E-6]	0.049	0.358
δ	[Eq.E-3]		3.657
θ	[Eq.E-4]		1.921
ψ	[Fig. 7 Ref.10]	0.112	0.879
ϕ	[Fig. 8 Ref.10]		0.091

Therefore the stresses at the midlength of the deckhouse are:

a) for constant B.M. alone:

	$k = 6.50 \times 10^2 \text{ psi}$	$k = 3.40 \times 10^4 \text{ psi}$
$p_f [= -C\psi m]$	-197 psi	-1545 psi
p_1 [Eq.E-15]	+125 "	+978 "
$\bar{\sigma}_1$ [Eq.E-11]	-4385 "	-1266 "
$\bar{\sigma}_f$ [Eq.E-12]	-4385 "	-1266 "
σ_1 (keel) [Eq.E-13]	+4179 "	+2995 "
σ_f (top) [Eq.E-14]	+1704 "	-1672 "

b) for sinusoidal B.M. alone:

	<u>$k = 6.50 \times 10^2$ psi</u>	<u>$k = 3.40 \times 10^4$ psi</u>
P_f	-669 psi	-1798 psi
P_1	+480 "	+1288 "
$\bar{\sigma}_1$	-4097 "	-1069 "
$\bar{\sigma}_f$	-3319 "	-866 "
σ_1 (keel)	+4621 "	+3421 "
σ_f (top)	+534 "	-2221 "

Superimposing (a) and (b), we obtain the total stresses for the combination bending moment:

$\bar{\sigma}_1$	-8482 psi	-2335 psi
$\bar{\sigma}_f$	-7704 "	-2132 "
σ_1 (keel)	+8800 "	+6416 "
σ_f (top)	+2238 "	-3893 "

Above results are plotted in Figure 28.

Equations for Schade's Method Using
"n" Terms of Moment Expansion

$$r_n = \frac{1}{2} \frac{1}{\cosh \frac{n\pi b}{2\lambda}} \left[\frac{n\pi y}{2\lambda} \sinh \frac{n\pi y}{2\lambda} + 2 \cosh \frac{n\pi y}{2\lambda} - \frac{n\pi b}{2\lambda} \cosh \frac{n\pi y}{2\lambda} \tanh \frac{n\pi b}{2\lambda} \right]$$

[E-8A]

$$M = m + \sum a_n \sin \frac{n\pi x}{2\lambda}$$

[E-9A]

$$p_f = \alpha_1 \cos \gamma x \cosh \beta x + \alpha_2 \cos \gamma x \sinh \beta x + \alpha_3 \sin \gamma x \cosh \beta x \\ + \alpha_4 \sin \gamma x \sinh \beta x - Cm - \sum C_n \frac{\theta_n}{\delta_n} a_n \sin \frac{n\pi x}{2\lambda}$$

[E-10A]

where,

$$\gamma = \omega \sqrt{1 - \eta}$$

$$\beta = \omega \sqrt{1 + \eta}$$

[E-16A]

$$\delta_n \equiv 1 + \frac{\eta}{\omega^2} \left(\frac{n\pi}{2\lambda} \right)^2 + \frac{1}{4\omega^2} \left(\frac{n\pi}{2\lambda} \right)^4$$

[E-3A]

and

$$\theta_n \equiv 1 + \frac{E}{r_n e_l + e_f} \left[\frac{I_l e_f}{a_l G} + \frac{r_n I_f e_l}{a_f G} \right] \left(\frac{n\pi}{2\lambda} \right)^2$$

$$+ \frac{E}{r_n e_l + e_f} \frac{r_n I_f e_l}{k} \left(\frac{n\pi}{2\lambda} \right)^4$$

[E-4A]

all other nomenclature, parameters and equations remain unchanged.