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HYDRODYNAMIC FORCES AND MOMENTS ON
A SUBMERGED BODY OF REVOLUTION RE-
SULTING FROM A FAIRWATER AND CONTROL
SURFACES

Joseph James Luckard, Jr.

Massachusetts Institute of Technology

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June 1973

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13. ABSTRACT <p>This thesis involves an investigation of the hydrodynamic forces and moments on a submerged body of revolution, resulting from the addition of an asymmetric fairwater, and hull-control surfaces, and the results of their interactions.</p> <p>This investigation involved both experimental and numerical models. Experimental tests were performed in the M.I.T. Variable Pressure Water Tunnel on a fairwater model, with independently-controlled hydroplanes, and on a submersible model, whose configuration was systematically altered by the addition of a fairwater and control surfaces. Graphical results for both of these models are presented. Flow visualization tests were also performed on the submersible model in order to observe the change in flow past the body of revolution and control surfaces caused by the asymmetric fairwater and its resulting downwash. Photographic results of these tests are presented.</p> <p>A preliminary numerical model (Fortran language) for the prediction of forces and moments, based on potential theory and vorticity distributions, is presented and its results compared with those obtained experimentally.</p>			

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By Joseph James Luckard, Jr.

B.S., United States Naval Academy

1965

Department of Ocean Engineering
Massachusetts Institute of Technology

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Chairman, Departmental Committee
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ABSTRACT

This thesis involves an investigation of the hydrodynamic forces and moments on a submerged body of revolution, resulting from the addition of an asymmetric fairwater, and hull-control surfaces, and the results of their interactions.

This investigation involved both experimental and numerical models. Experimental tests were performed in the M.I.T. Variable Pressure Water Tunnel on a fairwater model, with independently-controlled hydroplanes, and on a submersible model, whose configuration was systematically altered by the addition of a fairwater and control surfaces. Graphical results for both of these models are presented. Flow visualization tests were also performed on the submersible model in order to observe the change in flow past the body of revolution and control surfaces caused by the asymmetric fairwater and its resulting downwash. Photographic results of these tests are presented.

A preliminary numerical model (Fortran language) for the prediction of forces and moments, based on potential theory and vorticity distributions, is presented and its results compared with those obtained experimentally.

Thesis Supervisor: Dr. Damon E. Cummings
Title: Visiting Lecturer

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Thanks are also extended to Dean Lewis, Engineer in charge of the Hydrodynamics Laboratory (water tunnel), for his time, effort and interest expended during the experimental portion of this project; and to Alice Pobuda for her perseverance in the expert typing of this thesis.

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I. INTRODUCTION

In March 1971, a technical proposal was submitted by the Department of Naval Architecture and Marine Engineering, now the Department of Ocean Engineering, Massachusetts Institute of Technology, to the General Hydrodynamics Research Program of the Naval Ship Research and Development Center to investigate hull-control surface interactions on submerged bodies, to include the effects imposed on the body by the fairwater and rudders (1). This proposal was divided into two problem areas: the constant yaw angle case, and the unsteady (time dependent) state case. This thesis will cover investigation performed in the first problem area of the project, that of the steady angle of attack in the horizontal plane. Particular items of interest are:

A. The geometry of the trailing vortex sheet shed from the fairwater when operating at an angle of attack. Where is the sheet with respect to the hull, and control surfaces downstream, such as the rudder? What velocities are induced on the hull and rudder by the trailing vortex sheet, and what forces and moments result from this interaction?

B. The lift generated on the fairwater due to side (sway) velocity when operating at an angle of attack. Besides the external trailing vortex sheet described above in A, an image system of trailing vorticity is required within the hull

surface to satisfy the boundary condition of zero velocity normal to the hull surface, and Kelvin's theorem of conservation of circulation. A net circulation around the hull of a submersible, aft of the fairwater, is implied by this reasoning. The combination of this net circulation and sway velocity leads to a net lift on the hull. This resultant lift resolves itself into a heave force and pitch moment excitation, which must either be compensated for by the vehicle control system, or result in an unanticipated coupling between yaw, heave and pitch motions of the vehicle. Nonsymmetric design of most submersibles, caused by the single fairwater, also combines with this resultant side force (lift component) into a roll moment on the vehicle.

Quantitative assessment of these forces and moments were investigated by experimental, analytical and numerical procedures. Experimental results were necessary to determine actual forces and moments experienced on a submerged body, to visually observe resulting hydrodynamic effects, and to be used as a basis of comparison for numerical results. To satisfy the primary objective of the project, a general and useable motion control prediction model, both analytical and numerical modeling were used.

Experimental results are presented for a fairwater model and a submersible model of different configurations.

Theoretical results, based on another part of this GHR project in which Newman and Rodriguez investigated a linearized low-aspect ratio slender body theory, are compared with the applicable experimental results of this project.

Some numerical results are presented also, but are not final. Continuing modifications are being made to the computer program for the most realistic results, before a final project report is submitted to the Navy in early fall 1973.

Dr. Damon E. Cummings was the project supervisor and Dr. J. N. Newman was a participating faculty member throughout this investigation.

II. BACKGROUND

The design of submersible vehicles has changed drastically since the first successful American military submarine was built in the 19th century by the Holland Torpedo Boat Company. This first vessel, the HOLLAND, had many basic design features which eventually were reinstated into present day research and military submersible designs, especially since the development and design of the ALBACORE in 1950. This latter vessel was designed purposely to maximize submerged features at the expense of surface capabilities, emphasizing high submerged speed and maneuverability.

Particular hydrodynamic points of interest of HOLLAND to this investigation are:

1. A body of revolution hull form;
2. Little superstructure and no fairwater (sail), to minimize submerged resistance;
3. Stern planes and rudder surface located at the vertical centerline; and
4. Forward hydroplanes not employed.

The only features in military submarine and some research submersible design which have not returned to the HOLLAND configuration are the retention of the fairwater (sail) and forward hydroplanes, which are presently located on the fairwater (sail planes) for both military and hydrodynamic per-

formance purposes. Some research and test submersible vehicles have returned to the basic HOLLAND design in all aspects in an attempt to achieve an optimum submersible design.

Drag force components on a submersible resulting from appendages such as control surfaces, fairwater, shafts, and struts are of extreme importance when obtaining propulsion requirements for a particular vessel, but are of relatively minor importance when investigating stability and control. Much effort has been placed in this area for the last two decades to obtain an efficient design. This aspect will not be pursued in this report.

Present day submersible design has put an extraordinary requirement on stability and control. Although the military and research submersibles have vastly different performance capabilities, particularly the speed spectrums, the requirements of precise control and retention of stability remain. In particular, this is true for motions in the vertical plane, where a submersible must have the ability to operate, at slow or high speeds, within a relatively narrow vertical range. Full employment of a vessel's depth capabilities is desirable, but accidental penetration of depths beyond its maximum operating depth might lead to disaster. Nor is accidental broaching of the surface of the water a desirable maneuver in both the military and the research submersibles. Although

the horizontal plane is usually not as narrow ranged as the vertical, horizontal motions are important, especially in restricted waterways.

In the vertical (and horizontal) plane there are basically four performance criteria: (2)

1. Ability to maintain constant depth (course) with minimum plane movement and minimum depth (course) error;
2. Ability to enter into a maneuver as rapidly as possible;
3. Ability to exit from a maneuver as rapidly as possible;
4. Ability to return to equilibrium as quickly as possible when the controls are returned to zero.

An additional performance criterion only in the horizontal plane is the ability to execute a steady-turning maneuver with minimum tactical diameter, advance, transfer, loss of speed, and with minimum cross-coupled motions such as roll.

Most recent submarines have been equipped with fairwater (sail) planes, rather than bow planes, to reduce noise, alleviate the requirement for retraction, and to gain larger span dimensions within the submarine block dimensions. It has been shown on operational submarines that the drag of the fairwater planes compares favorably with previously used bow planes, even though fairwater planes have about 75% more

relative area. This same comparison shows 85% more relative vertical force, although only 20% more moment for fairwater planes than bow planes, because of the reduction in length of the moment arm (2).

For high speed maneuvering, forward hydroplanes are redundant. Depth changes can more readily be obtained by adjusting the angle on the stern planes, rather than applying a force close to the center of gravity. At low speeds, forward hydroplanes do make depth control somewhat easier.

Forward hydroplanes are desirable to help compensate for the nonsymmetrical hull form in the vertical plane, resulting from the fairwater, and to create a hydrodynamic vertical force and hydrodynamic moment in the vertical plane.

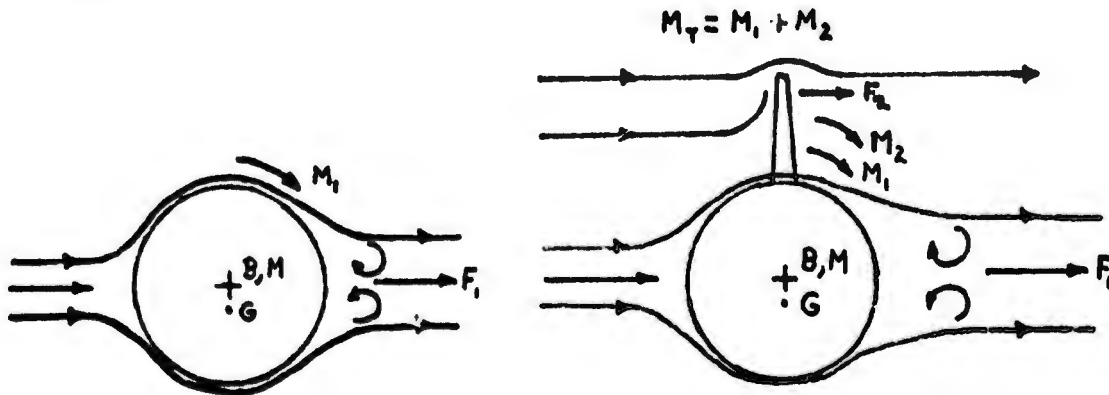
The hydrodynamic effects caused by asymmetry occur even when the hull body axis is parallel to the inflow velocity due to drag on the fairwater. In order to compensate for this effect and maintain constant depth, a hull angle of attack and a stern plane angle must be present, and are referred to as neutral angles of a particular vessel. These angles depend on the size of the fairwater and inflow velocity vector. If inflow velocity is assumed to be maintained horizontal, these angles introduce a pitch angle on the vessel equal to the hull angle of attack. This pitch angle in turn introduces a speed-dependent longitudinal metacentric moment, which is

the reason why the hull angle of attack and stern plane angle are speed-dependent. A critical range occurs at very slow speeds, in which the required angles for constant depth are too great to be accomplished by the stern planes only. Forward planes become a necessity at this time to reduce the magnitude of the requirements on the stern planes.

In the horizontal plane, the performance criterion that a vessel have the ability to execute a steady-turning maneuver with minimum tactical diameter, advance, transfer, loss of speed, and, in particular, minimum crosscoupled motions is of extreme importance and will require the major portion of this investigation for a valid understanding.

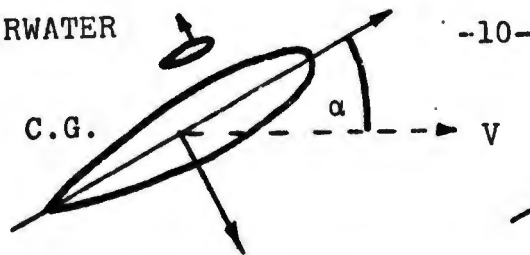
Of particular interest are the effects that a fairwater has on the above performance criterion concerning the steady-turning maneuver. The fairwater causes an increase in roll angle during a turn. One reason for this effect lies in the positions of the center of gravity (G), and center of buoyancy (B) of the vessel with respect to the axis of revolution of the hull body. When a submersible with a body of revolution, whose metacenter (M) is at the axis of symmetry because of wrap-around ballast tanks, is submerged completely, the waterplane area disappears. When this occurs, the location of the center of buoyancy (B) shifts from a position below the center of gravity (G) to one above. The location of the center of gravity (G), which has always been located below

the metacenter for the surfaced situation, is lowered even more by the added ballast. This low center of gravity, although increasing the roll stability of the vessel, introduces an asymmetry when the body of revolution is acted upon by hydrodynamic forces, in particular during a steady-turning maneuver. This occurs whether a fairwater is present or not, although it is more pronounced with a fairwater.

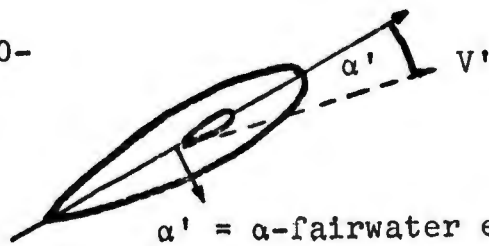


Fairwaters are located forward of the center of gravity; therefore, the hydrodynamic effects resulting in the horizontal plane during a steady turn are both stabilizing and destabilizing. Since the fairwater is essentially a lifting surface, it develops an effective lift force which is directed towards the turning circle center. This force, when combined with the vessel's velocity vector components, tends to decrease the angle of attack, which would be a stabilizing effect.

FAIRWATER



-10-



$\alpha' = \alpha$ -fairwater effect

Yet, when this force is combined with the distance forward of the center of gravity, the resulting moment is destabilizing. It tends to increase the yaw moment on the vessel in the horizontal plane, thus effectively reducing the turning diameter, which is desirable. However, in the roll-heave plane, the point of action of the lift force on the fairwater induces a roll moment on the vessel that is undesirable. The total roll moment experienced by the submersible is not entirely due to the fairwater. Part of this roll moment results from the hydrodynamic side force acting on the body of revolution above the center of gravity (G), which is below the body axis, for stability reasons. Table 5 of reference 2 reports of a model tested with a fairwater at a speed of 20 knots, rudder angle of 35 degrees and in a steady turn, which resulted in an angle of heel of 11.3 degrees. When this same model was tested without the fairwater, the angle of heel was reduced, but still present at 2.5 degrees. It was also shown during these tests that removal of the fairwater increased the turning diameter by about 25 per cent.

Although this investigation is pursuing the hydrodynamic effects at constant angles of attack, a brief statement on

the transient phenomenon of "snap-roll" is of interest. Snap-roll describes what occurs shortly after the initiation of a turn, corresponds to the amplitude of the first half cycle of roll and is believed to be an overshoot phenomenon. After the snap-roll occurs, the roll angle decreases to the steady-roll value. Values for snap-roll in the model tests described in the last paragraph were 39 degrees for the model with fairwater, and 12.5 degrees when the fairwater was removed. As before, the effect of the location of the center of gravity below the axis of symmetry is evident (2).

Since snap-roll is so immediate and of such a large magnitude on high-speed submersibles, control response time and knowledge of a submersible's particular characteristics are of extreme importance. When the snap-roll occurs, in combination with a rudder angle on for the turn, a cross-coupling results in an effective diving attitude for the vessel. Although this effect can be alleviated by judicious handling of available control surfaces, a problem of overshoot can arise. As previously mentioned, snap-roll only lasts for a relatively short period of a turn, after which the steady-turn phase is incurred. As a result, the effective diving attitude decreases. If the correction imposed for snap-roll is not reduced accordingly, the vessel's attitude will be one of decreasing depth, with an extreme result of broaching.

III. METHODOLOGY

The purposes of the experiments reported on in this project were to investigate the hydrodynamic forces and moments that act on a submerged body in a steady flow both on a symmetrical slender body and on one to which appendages were added.

Possible methods were also investigated to redistribute the forces and moments to alleviate undesirable hydrodynamic effects caused by asymmetry, which is present in most submersible designs.

Measurement of Hydrodynamic Effects

Tests were conducted in the Variable Pressure Water Tunnel (see Appendix A) located in the Hydrodynamics Laboratory at M.I.T., see Figure 1. Measurement of the forces and moments on a particular model is accomplished through the application of a dynamometer which has six degrees of freedom, see Figures 2 and 3. These six degrees of freedom can be measured with respect to any point along the height of the test area. A computer program is used to deduce the three forces and three moments on the model being evaluated (see Appendix B). For the model tests covered in this project, coordinate systems were located at the center of the test area, along the axis of revolution of the submersible model,

see Figure 2, and at the tunnel wall, base of the fairwater model, see Figure 3.

Hydrodynamic Notation of Dynamometer Coordinate System

(With respect to model, about end of shaft)

FX - Surge - applied force on longitudinal axis
FY - Heave - applied force on vertical axis
FZ - Sway - applied force on transverse axis
MX - Roll - moment applied about the longitudinal axis
MY - Yaw - moment applied about the vertical axis
MZ - Pitch - moment applied about the transverse axis

Note: FXO, MXO, FZO, MZO, FYO, MYO are simultaneously-computed hydrodynamic effects from "general" dynamometer program for evaluation of model test in which results are desired with respect to water tunnel flow (free stream flow).

Experimental Models

Two different models were designed and built to measure the hydrodynamic forces and moments on a submerged body after symmetry to tunnel water flow is lost by the addition of appendages (fairwaters, control surfaces) and increased angle of attack (angular difference between free stream flow and body of appendage line of symmetry).

The two models include:

A) A streamlined body of revolution (tear-drop shape), with detachable fairwater sail and stern section, see Figures 4a, 5a and 6, to investigate the interaction effects of hydrodynamic forces and moments between a submersible hull, fairwater and control surfaces at various angles of attack and velocities (model hull and fairwater were constructed out of lucite; stern section and control surfaces were constructed out of brass).

The hull was constructed with a length (L) of 24.5 inches, a maximum diameter (D) of 3.5 inches occurring at a distance of 9.8 inches from the bow, resulting in a L/D ratio of 7.0. This slenderness and a fine stern should prevent most separation, and also enhance the applicability of slender body theory. The support shaft was located at a distance, from the bow, of 40 per cent of the model length; and

B) A fairwater with detachable and independent control surfaces, see Figures 4b, 5b, 6 and 7.

Construction of Fairwater and Control Surfaces

In conjunction with both model experiments, foil shapes were required for fairwater and control surfaces. The lucite fairwater (submersible model) and planes (fairwater model) were produced on a manual milling machine, from offsets developed from a computer program for this particular project (see Appendix C). This computer program is in a general form to meet any particular designs of thickness, chord length, taper, setback and foil design (NACA___). This program calculates steps of thickness along the length of chord and span to be milled. These depths of cut are determined by foil geometry, end mill size and step size between milling runs. After milling of the step functions, the foil is finished by hand.

The stern control surfaces (submersible model) were produced on the Guggler Profile machine, located in the Gas Turbine laboratory shop of Building 31 (Sloan Laboratories). This machine requires a 4:1, plus 1/2 inch, scale model of surface to be developed, and is limited in the width of model fed between cam follower surface. Extra 1/2 inch is added on to compensate for abrupt changes in model shape (trailing edge).

Since the rudders used for the submersible were of foil shape, with no taper or setback, a simplified model (cam) was

made of an NACA foil shape, see Figure 8. Extended lengths of the foil shape can be produced and divided into various lengths of span. Cam followers are adjustable, so foil chord and thickness dimensions can be varied within limits, while using the same model (cam).

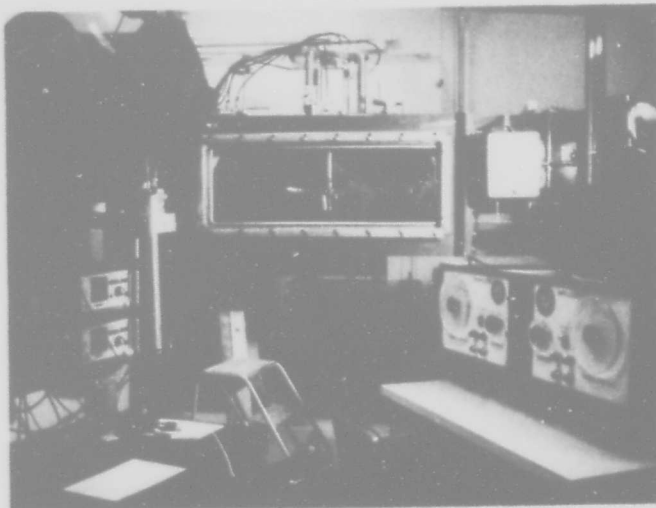


Figure 1 Variable Pressure Water Tunnel control and test section.

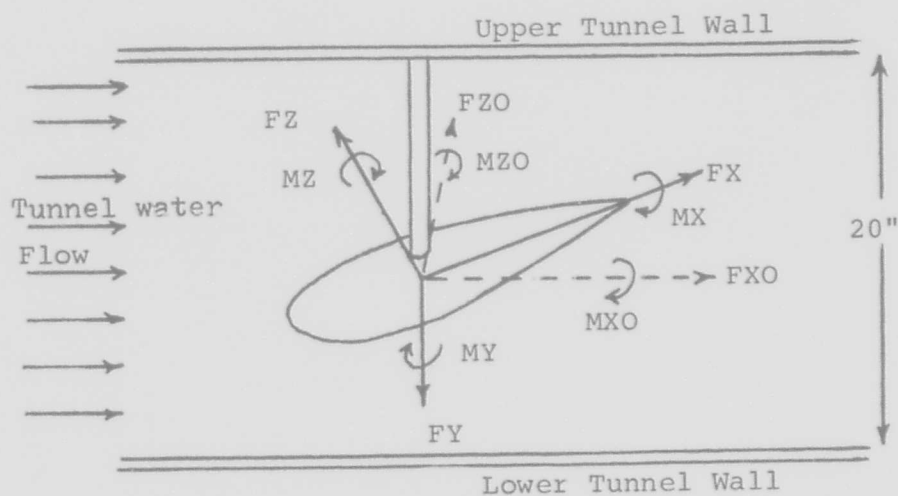
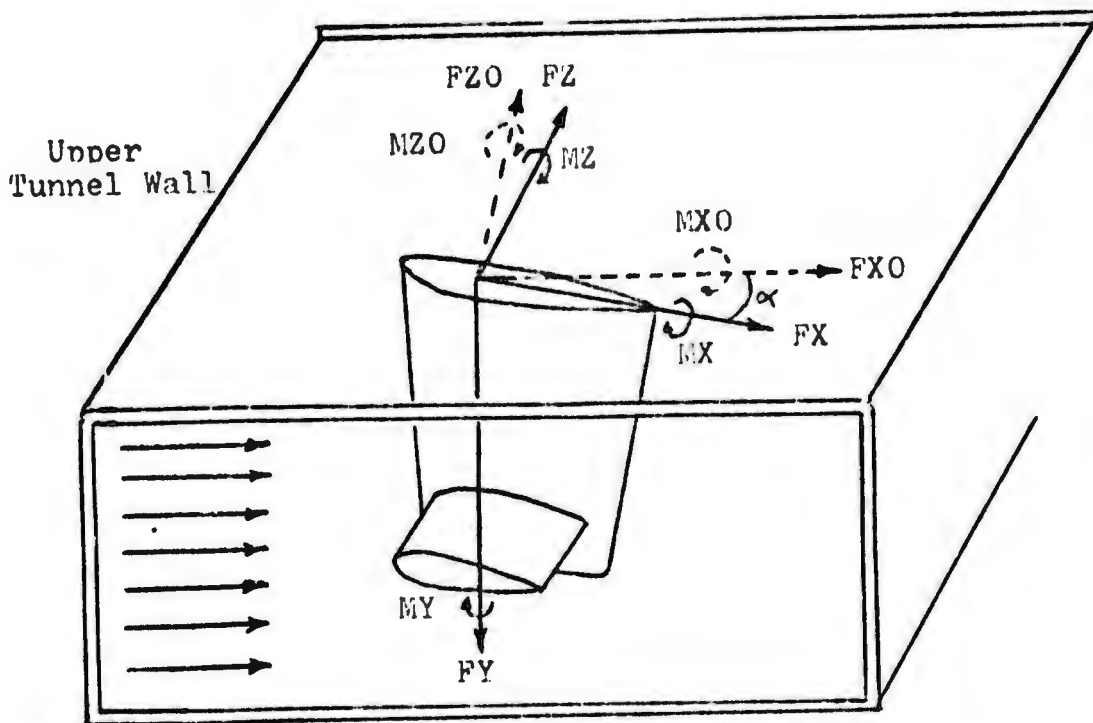
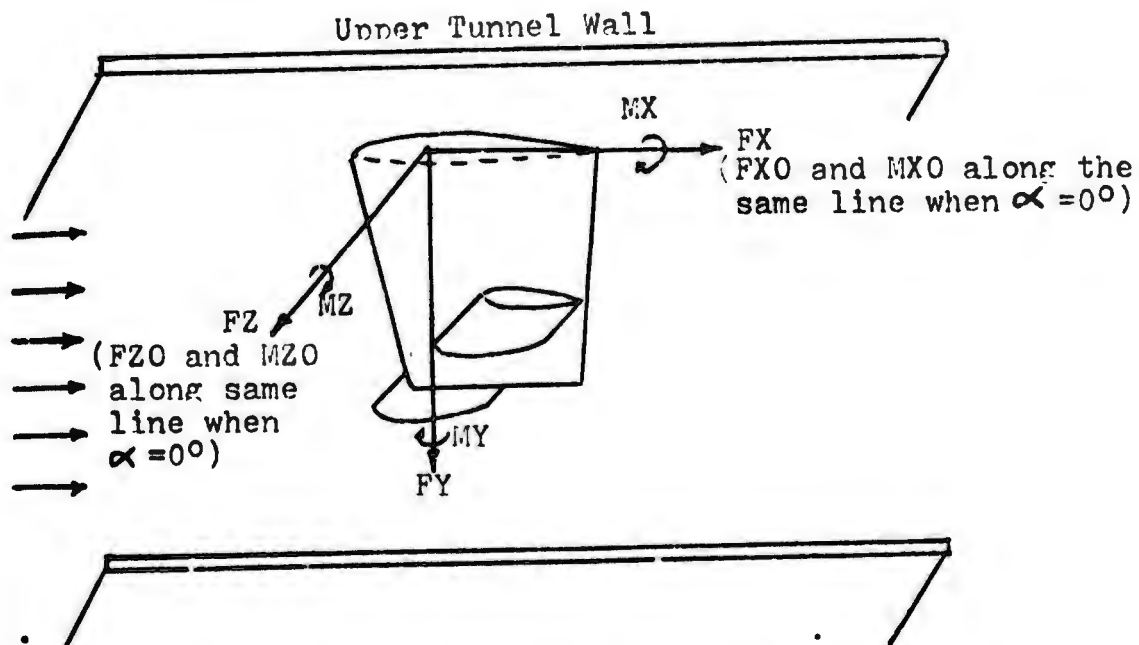


Figure 2 Dynamometer Coordinate System at center of model axis, center of tunnel.

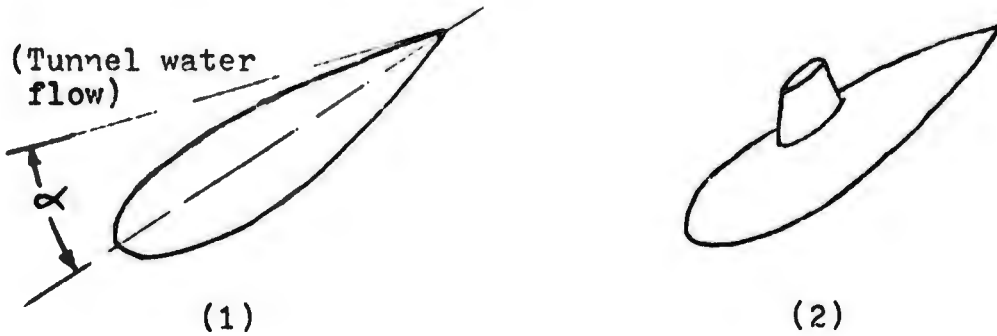


a. Dynamometer Coordinate System at base of model, against upper tunnel wall, at an angle of attack (α).

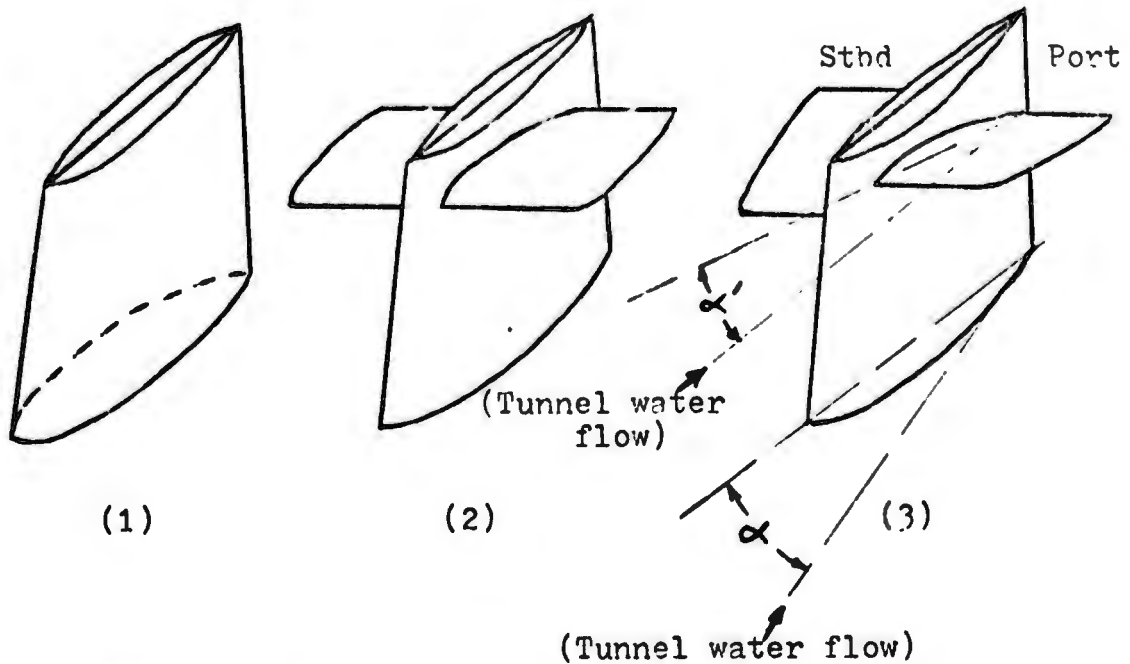


b. Dynamometer Coordinate System at base of model, against upper tunnel wall, parallel to tunnel flow.

Figure 3 Fairwater Coordinate System

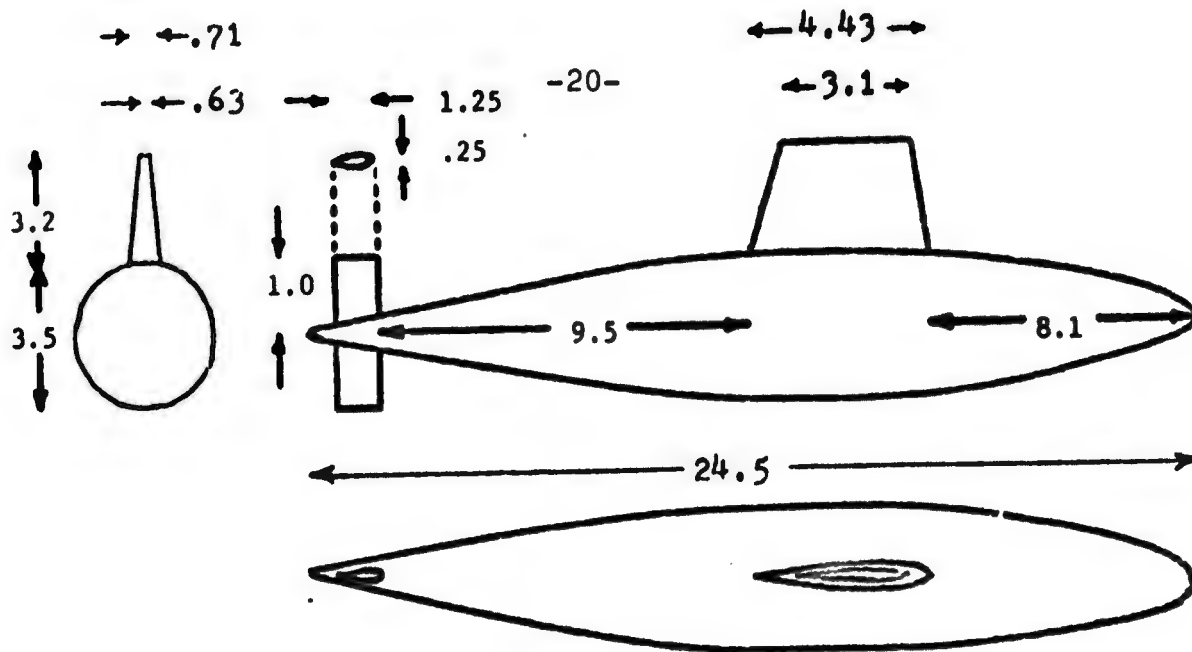


- a. Streamlined body of revolution (tear-drop shape)
(1) without fairwater
(2) with fairwater

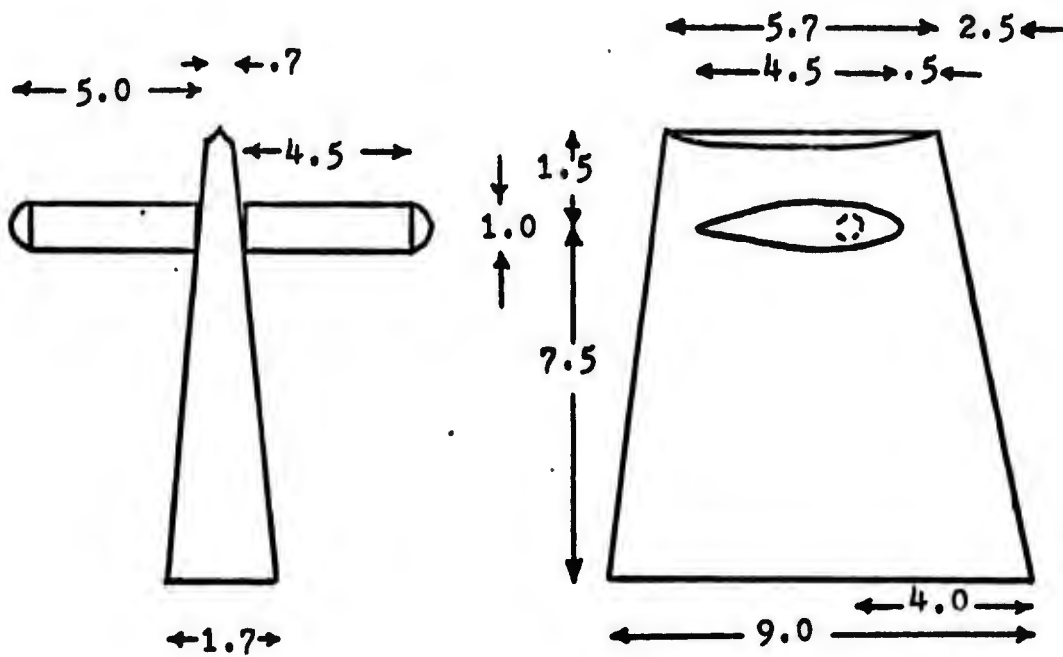


- b. Fairwater design
(1) without control surfaces
(2) with control surfaces, at same angle of attack (α')
(3) with independent control surfaces, port side positive, and starboard(stbd) side negative angle of attack (α').

Figure 4 Model orientation

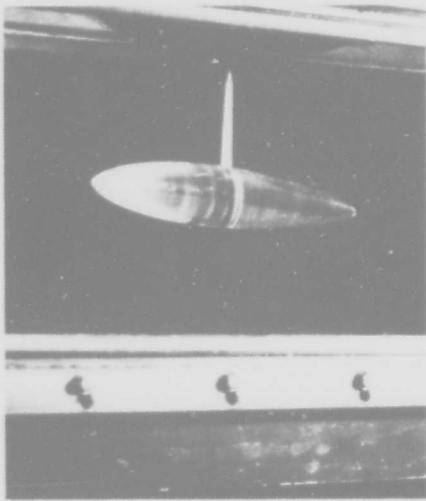


a. Submersible with fairwater

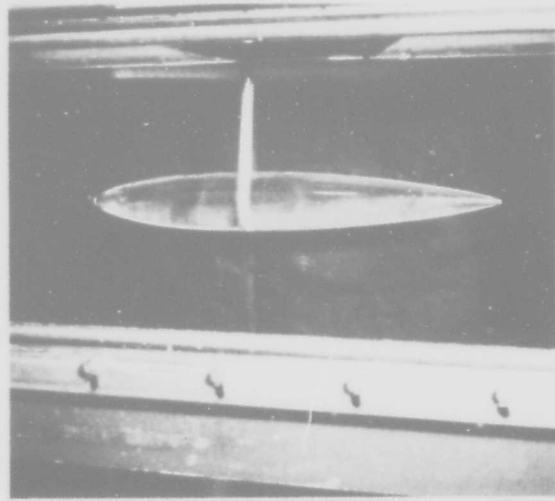


b. Fairwater with control surfaces

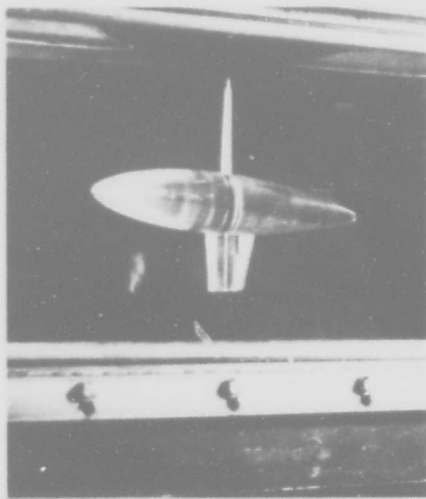
Figure 5 Model dimensions (inches)



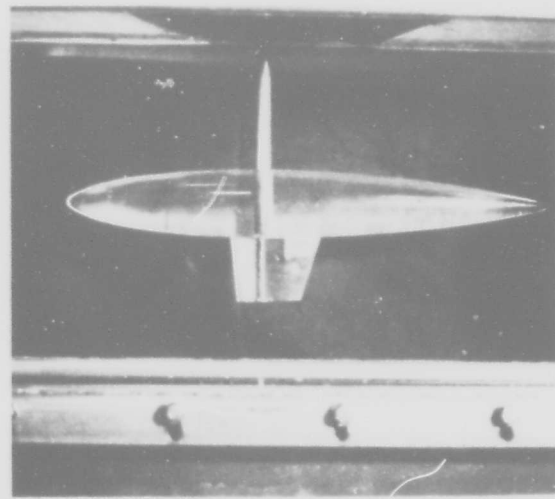
a. Oblique view without fairwater (Clean Hull).



b. Side view without fairwater (Clean Hull).

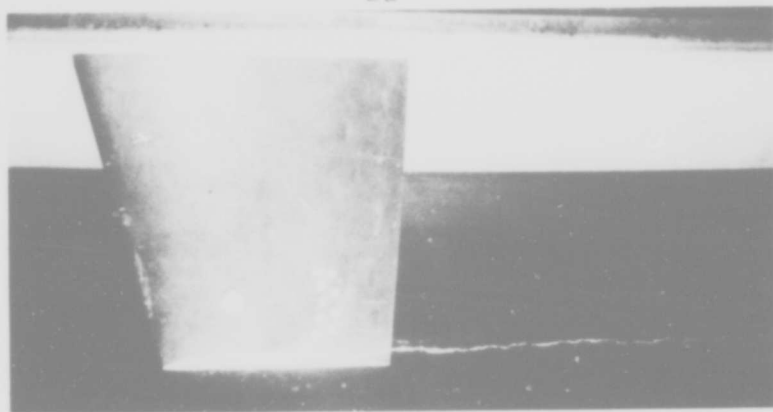


c. Oblique view with fairwater.

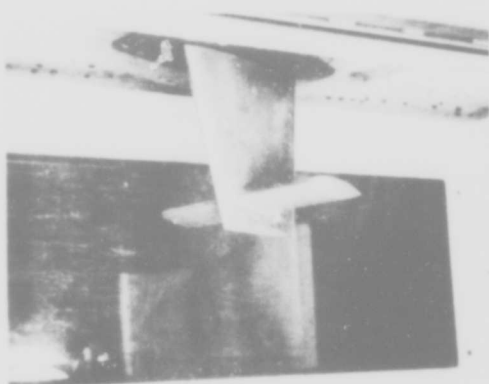


d. Side view with fairwater.

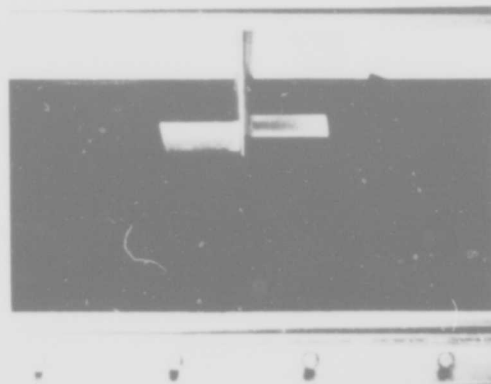
Figure 6 Submersible model



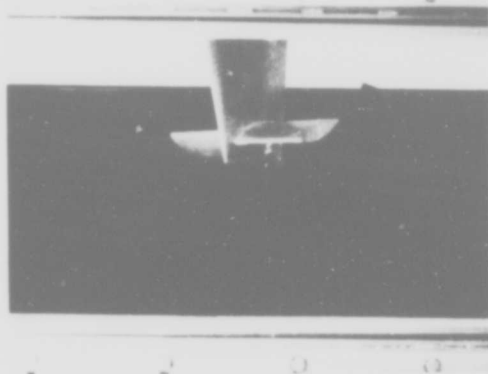
a. Side view of fairwater, without control surfaces. (Note support and guide locations used with control surfaces)



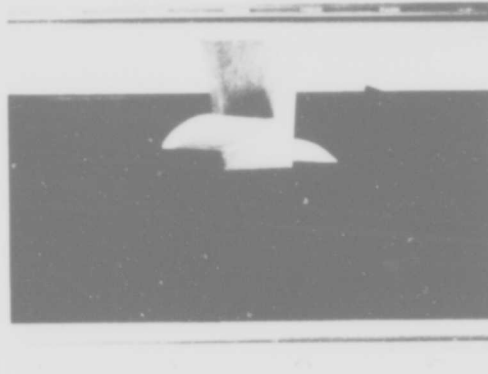
b. Oblique view, with control surfaces at $\alpha' = 0^\circ$.



c. Front view, with control surfaces at $\alpha' = \pm 20^\circ$.

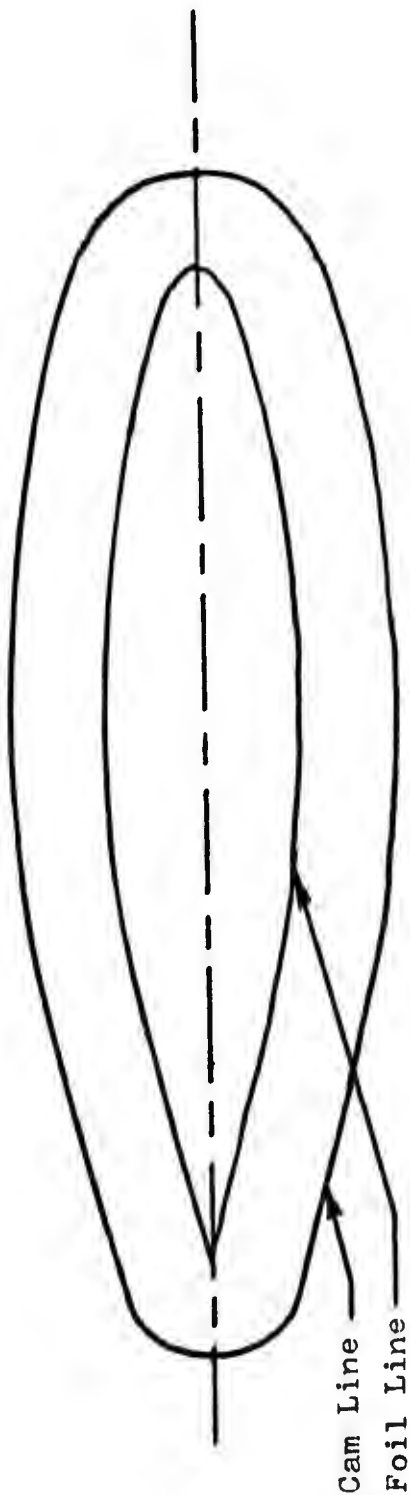


d. Oblique front view, with control surfaces at $\alpha' = \pm 20^\circ$.



e. Oblique rear view, with control surfaces at $\alpha' = \pm 20^\circ$.

Figure 7 Fairwater model



Cam pattern for rudders of submersible model

NACA 66-010

Chord = 1.25 inches

Thickness = .25 inches

Figure 8

IV. SUBMERSIBLE MODEL EXPERIMENTAL RESULTS

To experimentally investigate the hydrodynamic forces and moments on a submerged body of revolution, and how these effects are altered by the addition of appendages, the following different model configurations were tested:

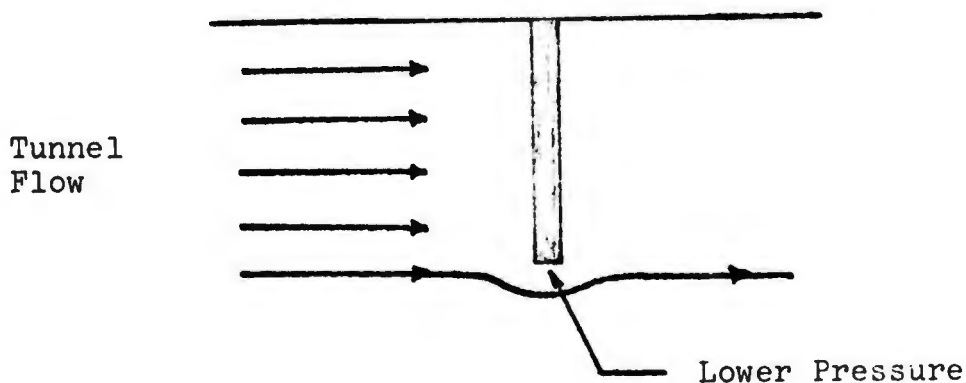
1. clean hull (no appendages)
2. with fairwater
3. with fairwater and upper rudder
4. with fairwater, upper rudder and lower rudder.

Flow velocities of 10, 15 and 20 feet per second were used, and angles of attack in the yaw-sway plane were varied between ± 25 degrees for tests evaluated at 10 feet per second, and ± 15 degrees for tests evaluated at 15 and 20 feet per second.

Caution must be used when interpreting the experimental results obtained during this investigation. All photographs and graphical displays of the results are shown according to the coordinate system previously described. Therefore, it must be kept in mind at all times that the model and results are inverted, since the model is mounted upside down on the top of the tunnel.

The support shaft used during the submersible model testing was evaluated by testing a bare shaft to determine

the forces and moments it would experience in the water tunnel. These effects were then deducted from the model test, to obtain results representing only the model. The interactions caused by the shaft on the hull and control surfaces were not investigated, but must be considered when interpreting the experimental results. Also, the lower pressure at the exposed end of the base shaft, caused by the flow pattern past the shaft, was not considered.



Heave Force (FYO)

1. Shaft Effect

Although the heave force on a bare support shaft, vertical and perpendicular to the flow, should have a constant value with respect to angle of yaw, the results obtained fluctuated and decreased for negative angles, see Figure 9. This is most likely a result of measuring accuracy of the dynamometer load cells used. An approximation was made to obtain a constant value of heave to be deducted from each model results, to alleviate the shaft effect. These values

were:

- 1) 1 lb, for a velocity of 15 ft/sec
- 2) 1.7 lb, for a velocity of 20 ft/sec

2. Model Results

When configuration 1 (clean hull) is tested, an initial negative heave force is registered at zero angle of yaw. According to the coordinate system used, this would appear as a downward force if the model were right side up. As the model is adjusted for both positive and negative angles of yaw, a symmetrical, increasingly more positive result occurs, see Figures 10 and 11. There should not be any heave force on this symmetrical model configuration, but the results could possibly be caused by the model not being perfectly aligned in the tunnel test area, or support shaft interaction with the flow around the body.

Configuration 2 (with fairwater) induces a drastic change in the pattern of the heave force imposed on the model. The results follow that of configuration 1 up to ± 5 degrees yaw. After ± 5 degrees, the slope decreases rapidly, reversing and producing a heave force at ± 10 to ± 15 degrees yaw comparable to that experienced at zero angle of yaw, see Figures 10 and 11. Thus it would appear that the addition of the fairwater and its resulting trailing wake induces a circulation and velocity on the hull body which generates a

decreasing slope of heave force per degree yaw in the coordinate system used. This is comparable to an increasingly large force downward, if the model were in an upright position.

The sensitivity of the results can be seen from Figure 12, in which results of tests 3 and 5, conducted on different days, were compared to see if they were compatible. The magnitude of test 5 varied, but the curve followed the same characteristic pattern as test 3. This difference in magnitude could result from a small difference in model alignment in the water tunnel or variation in the calibration of the dynamometer system on different days.

When configuration 3 (fairwater and upper rudder) and configuration 4 (fairwater plus both rudders) were tested, there appeared to be no effective change from the result of configuration 2, see Figures 13 and 14. The circulation and trailing wake off the rudders did not have any afterbody on which to induce a heave force component.

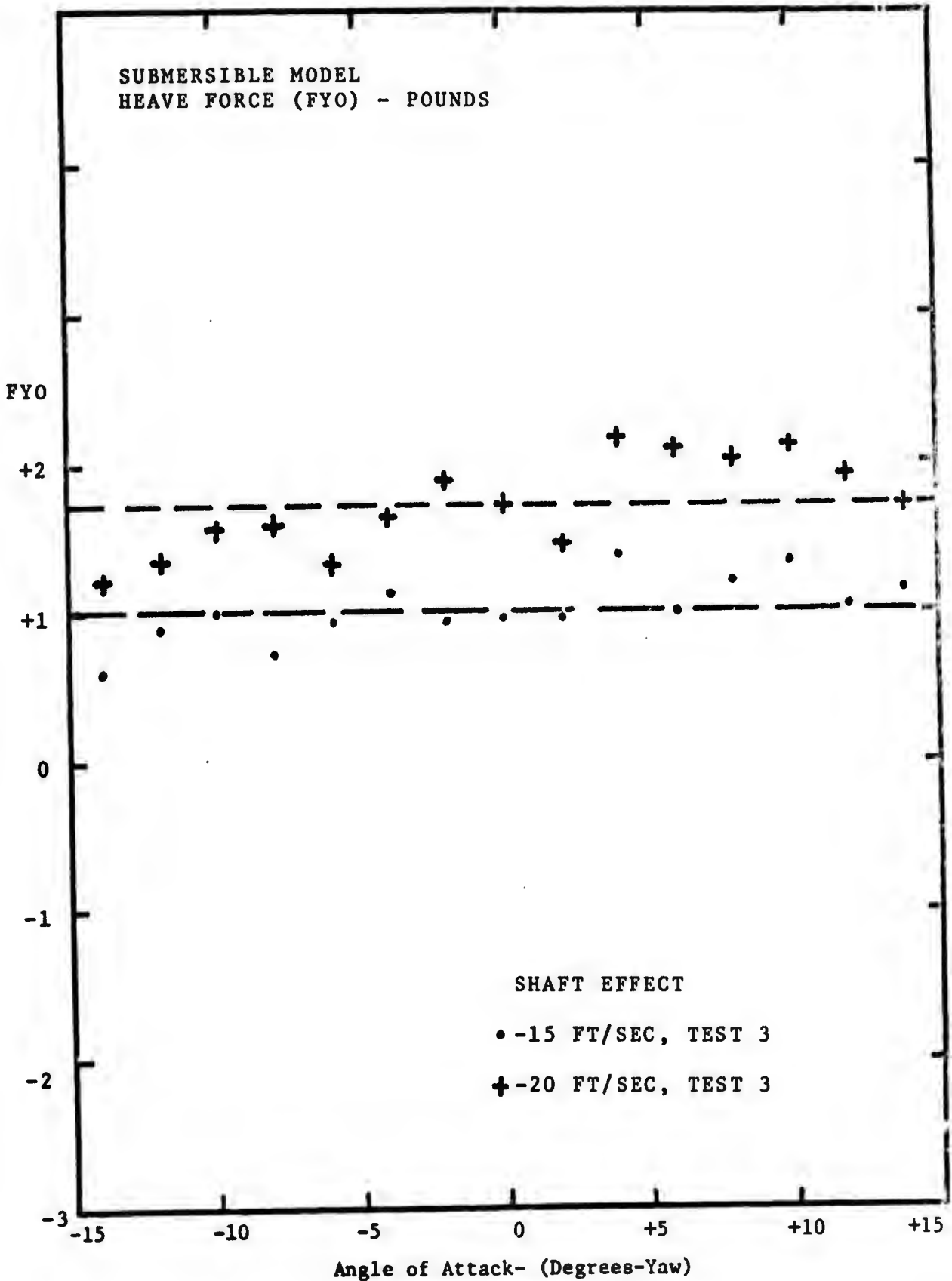


Figure 9

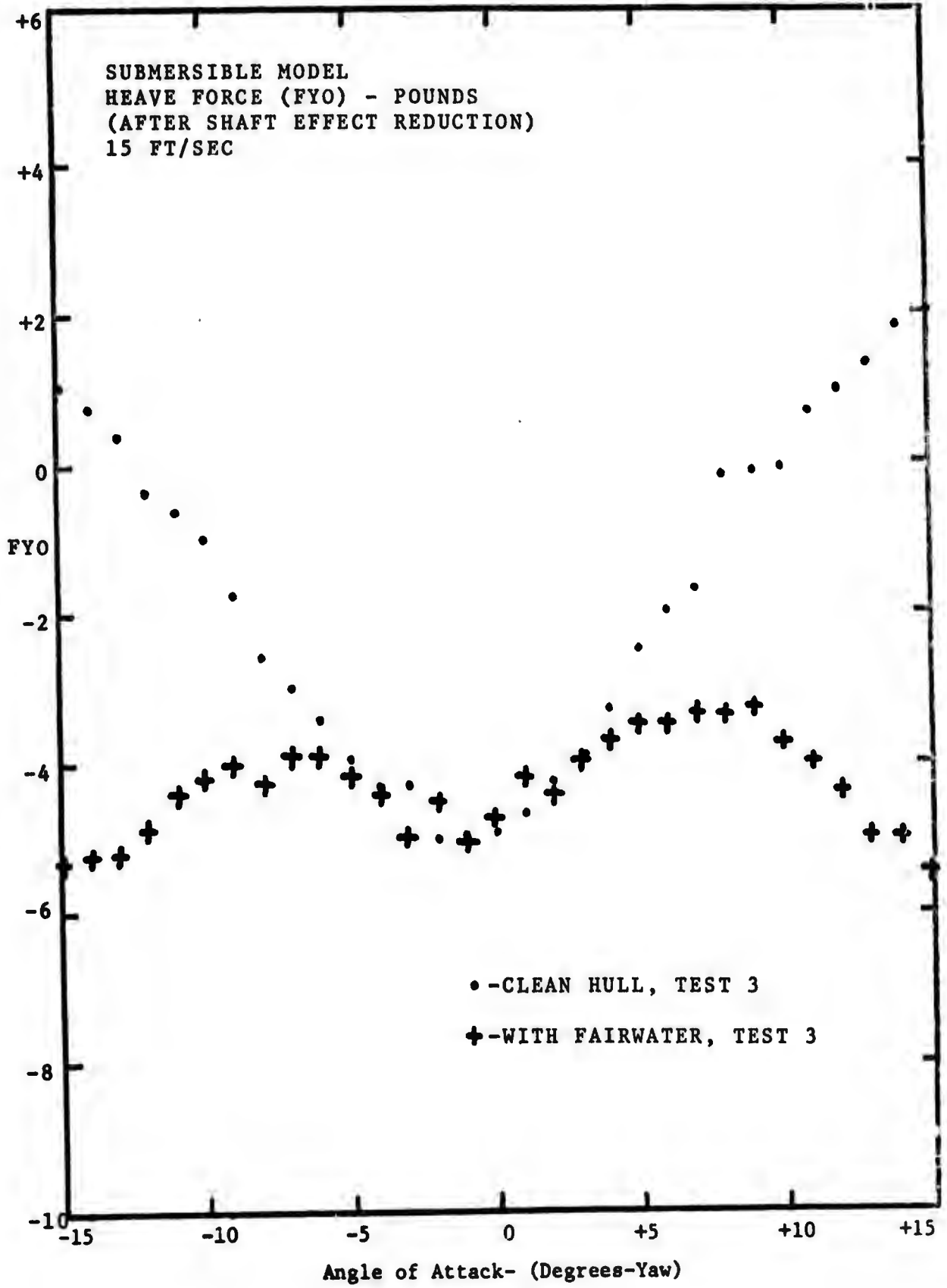


Figure 10

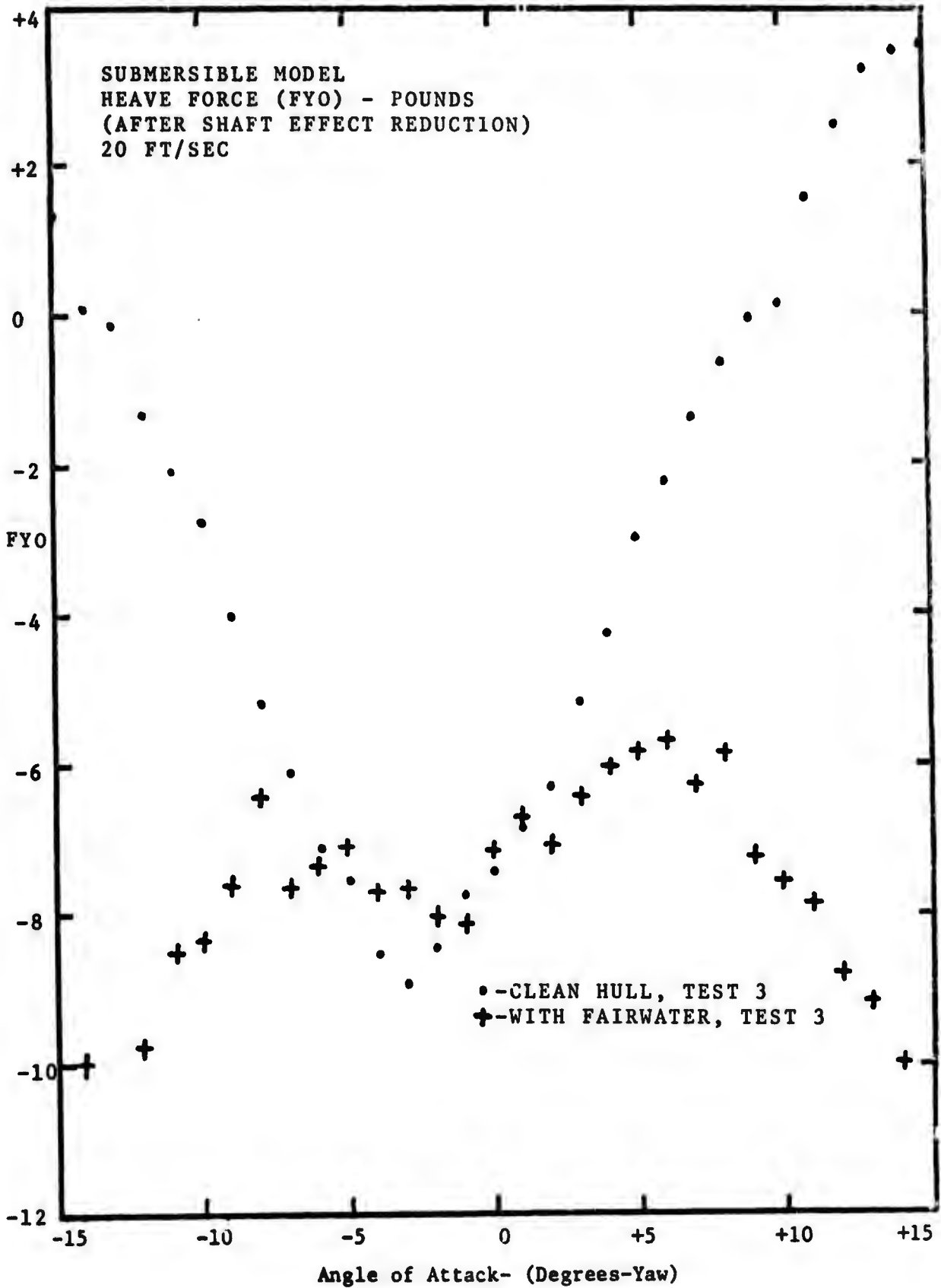


Figure 11

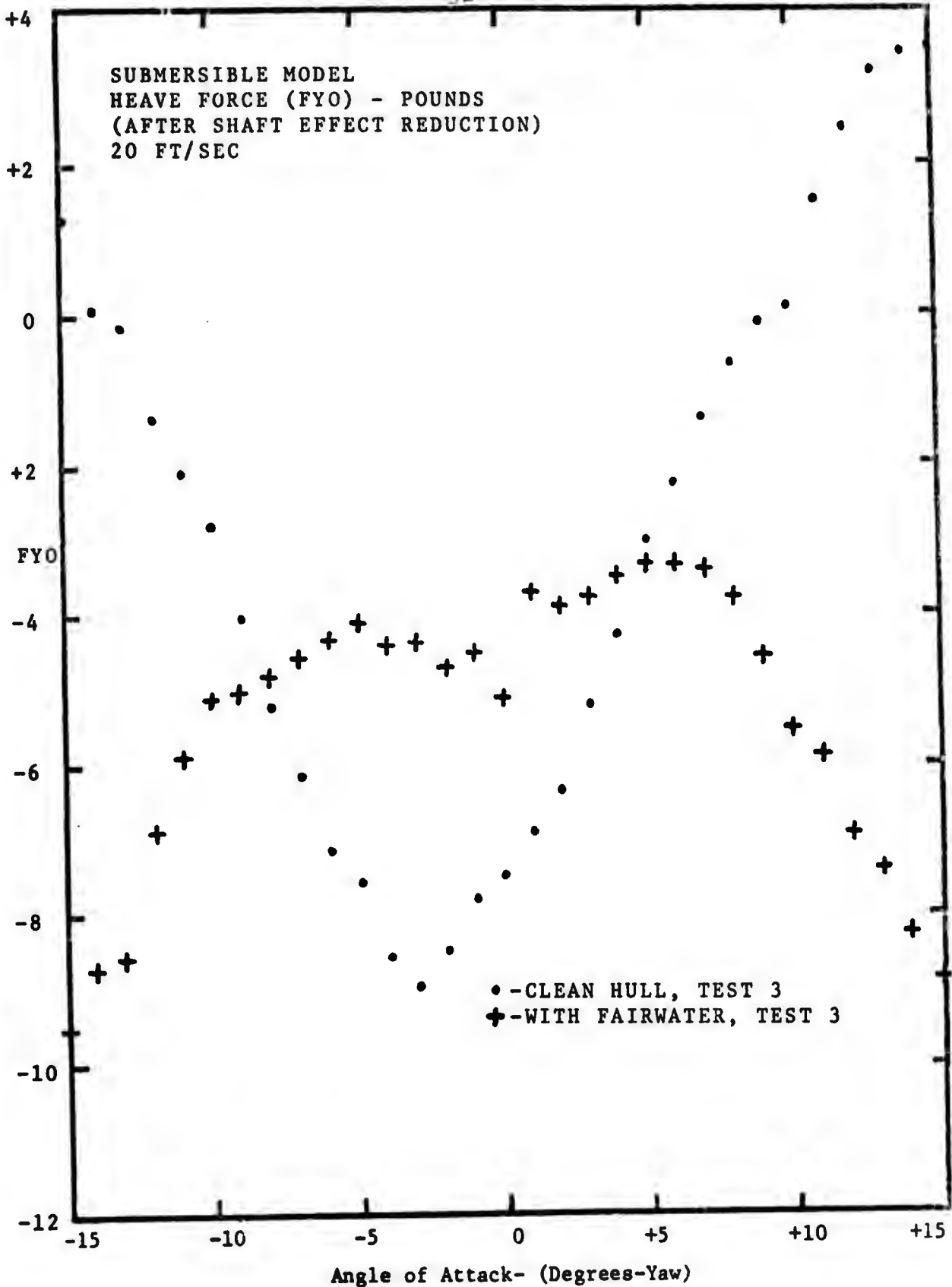


Figure 12

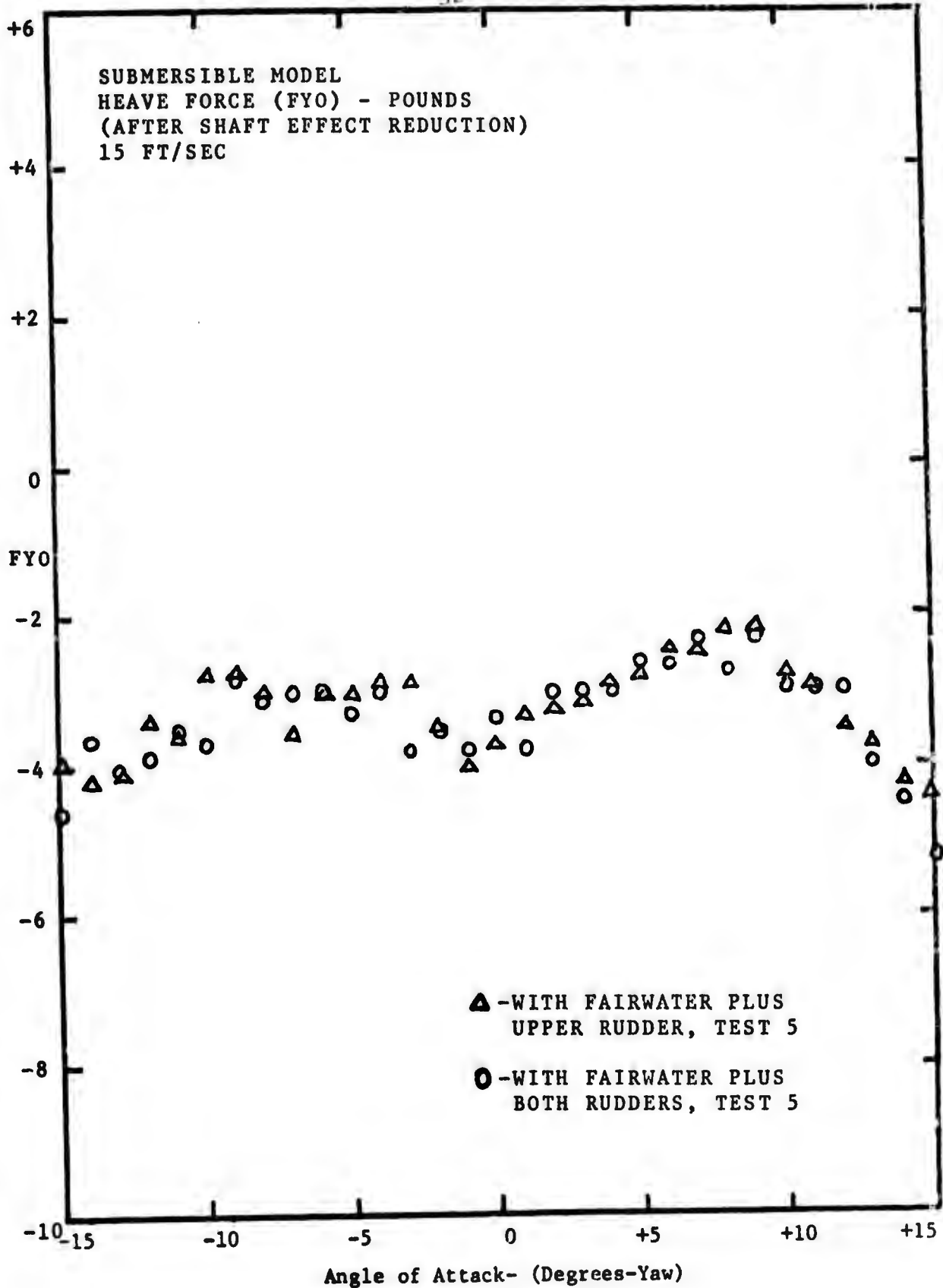


Figure 13

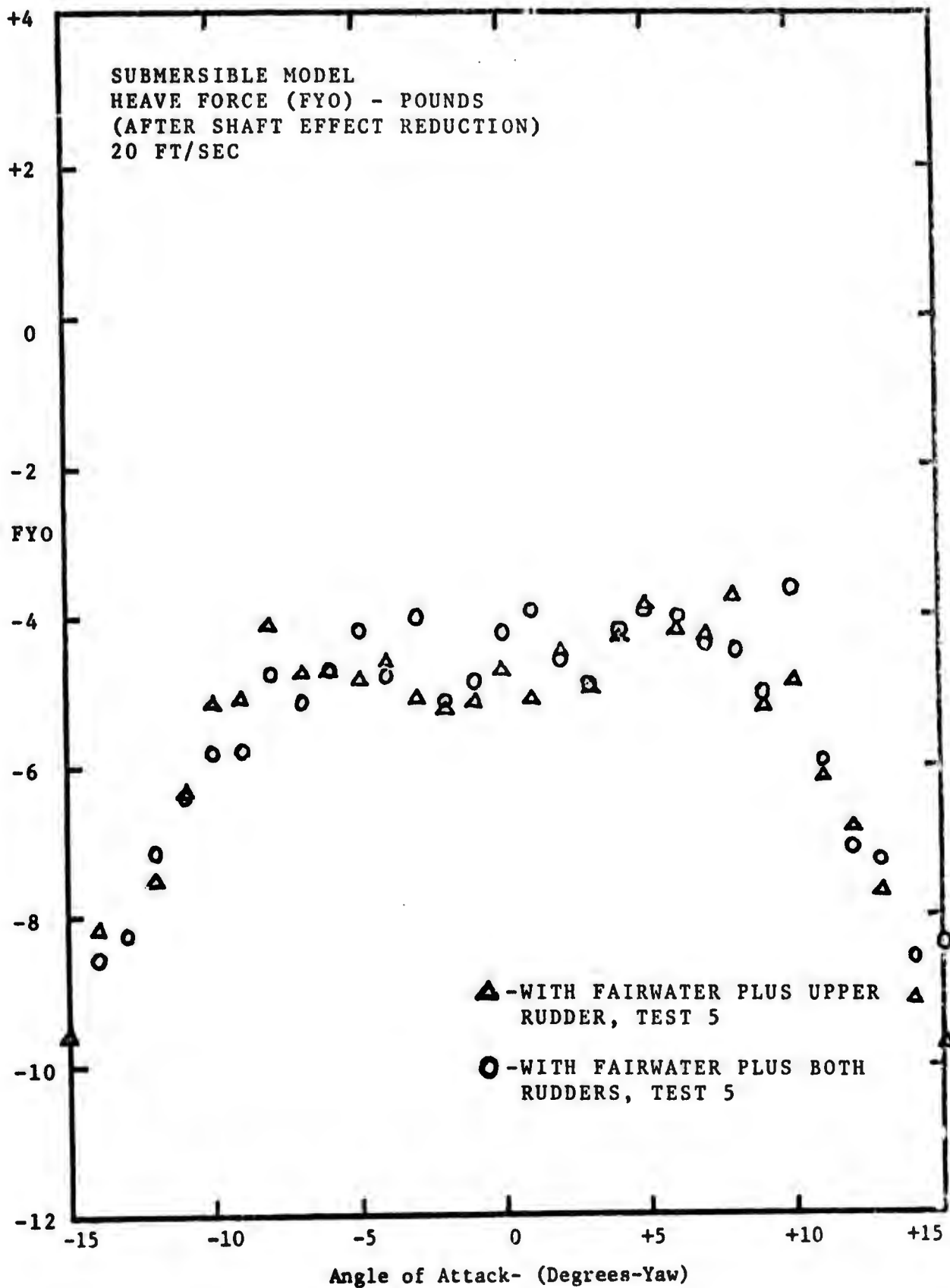


Figure 14

Pitch Moment (MZ)

1. Shaft Effect

Since the pitch moment on the base shaft should decrease with increasing angle of yaw (α), the reduction factor ($\cos \alpha$) can be assumed to be unity for the range of deflection at which the shaft was evaluated (zero to ± 15 degrees). Therefore, the reduction for the shaft effect is assumed to be constant, and a mean value of the experimental results, at different velocities, can be deducted from the model test to arrive at values of pitch moment, for the different model configurations, which approach that of a non-supported model. The resulting mean values were:

- 1) 40 in-lb, for a velocity of 15 ft/sec, and
- 2) 71 in-lb, for a velocity of 20 ft/sec, see Figure 15.

2. Model Results

Pitch moment on the various model configurations followed the same pattern of results obtained for heave force. Configuration 1 (clean hull) produced a negative moment at zero angle of yaw and a steadily increasing moment, with a decreasing slope, as yaw angle is increased, see Figures 16 and 17. The negative moment at zero angle of yaw could be caused by model alignment, or shaft interactions. The flow past the shaft causes a high pressure area at the forward stagnation point and a low pressure area on the aft portion

of the circumference. Fairwater drag would also produce a similar effect, but the magnitude of the negative moment was equivalent to that obtained for configuration 1 (clean hull).

Configuration 2 (with fairwater) produced an increasing moment, which followed the results of configuration 1 between about ± 5 to ± 10 degrees, after which the pitch moment decreased, as fast as it had increased, see Figures 16 and 17. This result would be comparable to a submersible, in an upright position, experiencing an initial bow up pitch moment at zero angle of yaw. This is due to shaft-hull interaction, a minor model misalignment in the vertical plane, or possibly miscalibration of equipment. This is followed, with increasing angle of attack, by a bow down pitch moment, caused by a lift force on the hull aft of the fairwater. This is finally succeeded by a bow up pitch moment, resulting from the drag component of the fairwater, and the circulation and induced velocity from the fairwater on the aft section of the hull body.

As in the heave force results, pitch moment did not seem to be affected by the addition of the upper rudder (configuration 3), see Figures 18, 19 and 20, or when tested with both rudders attached (configuration 4), see Figures 19 and 21. Their induced velocities have very little effect on the hull, which precedes them in the fluid flow, and their wakes have no afterbody to affect either.

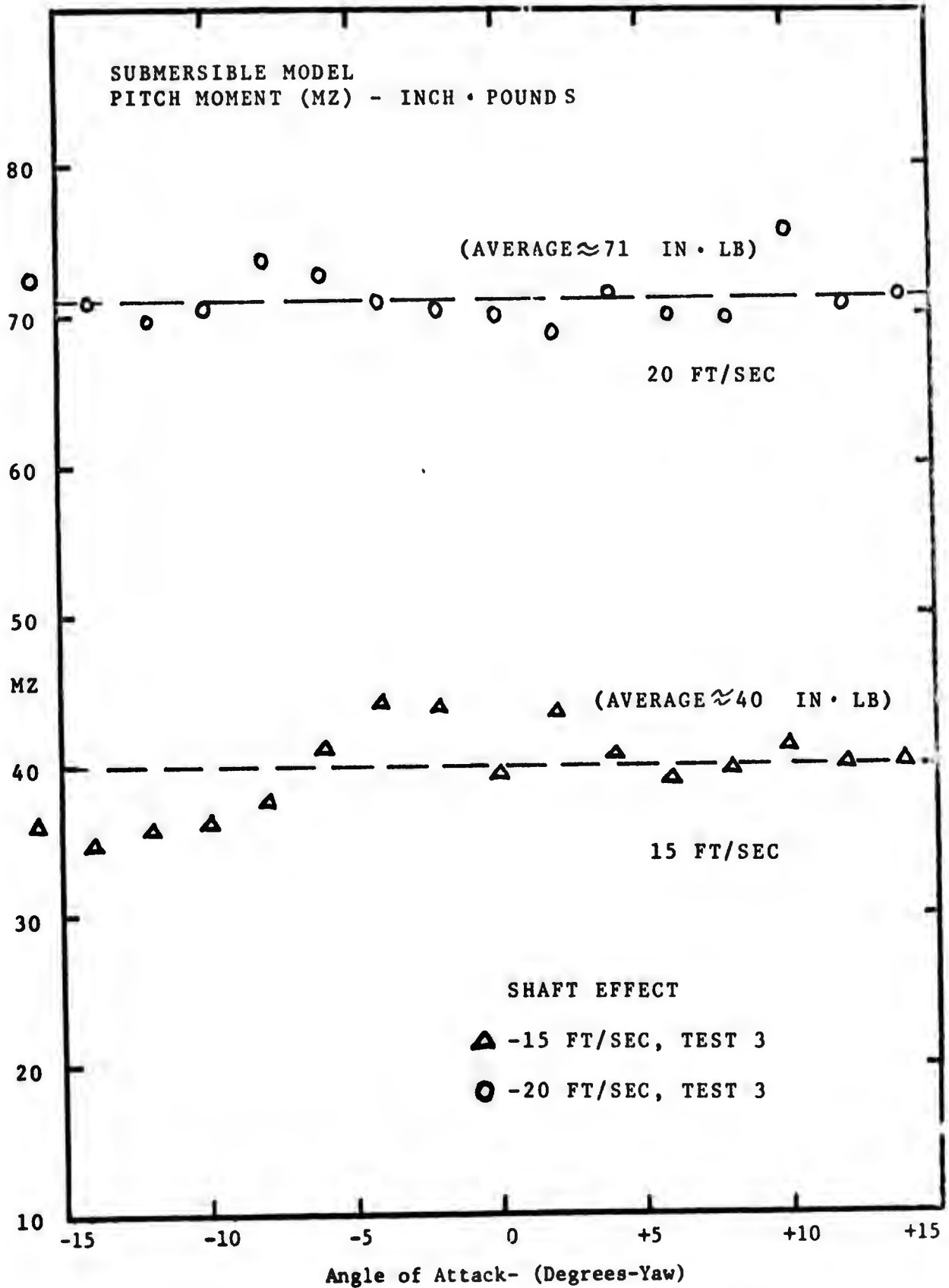


Figure 15

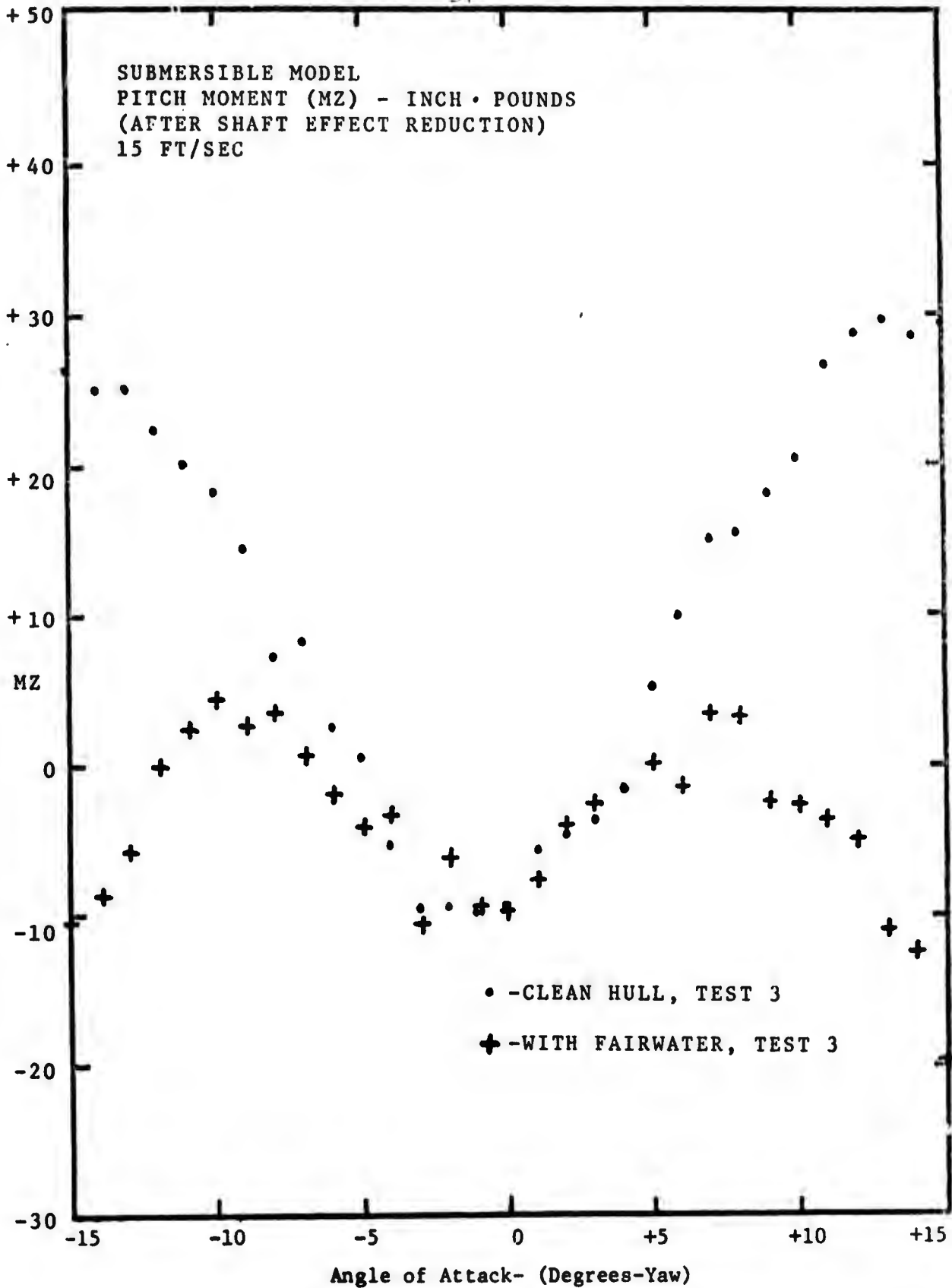


Figure 16

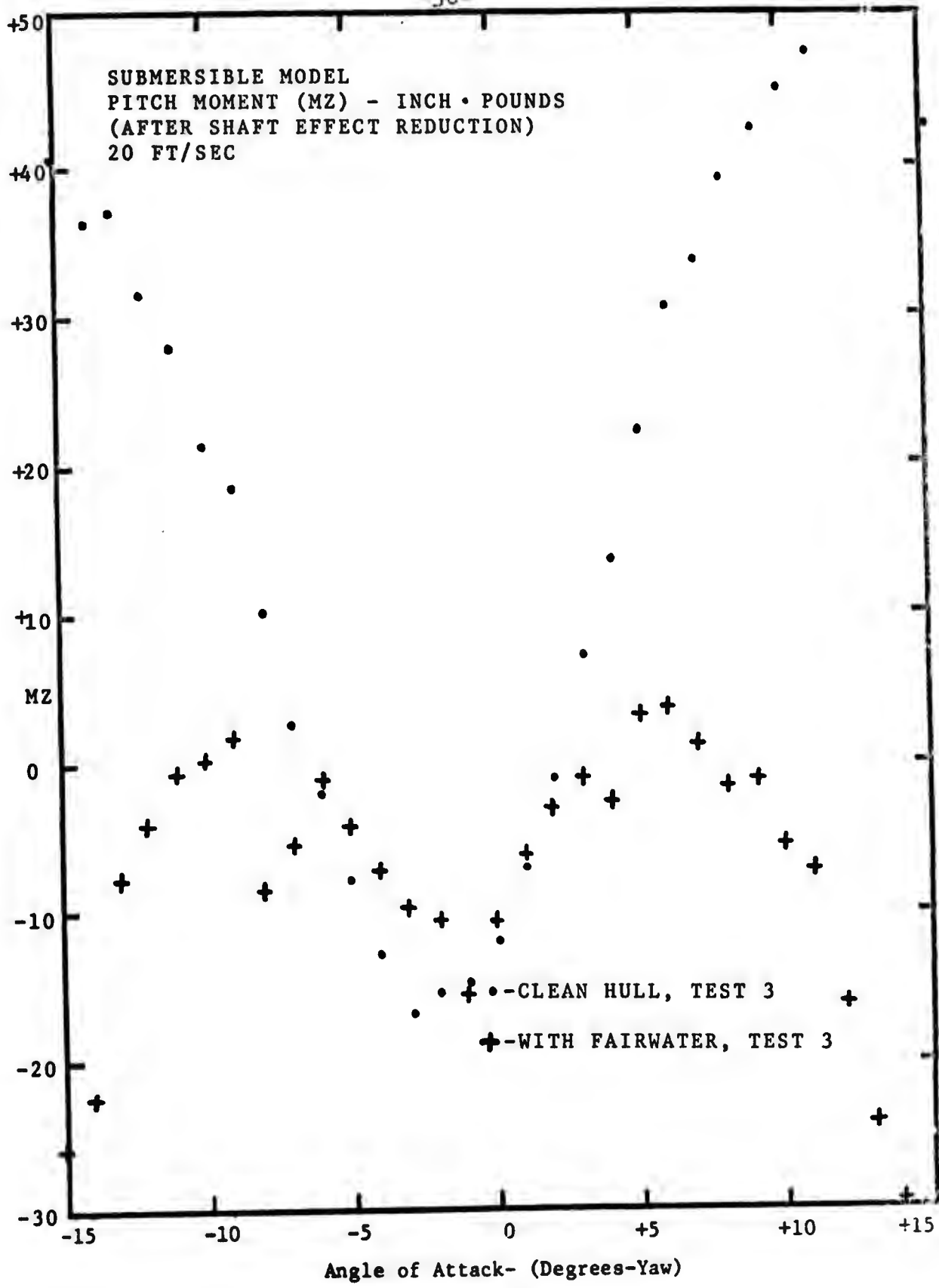


Figure 17

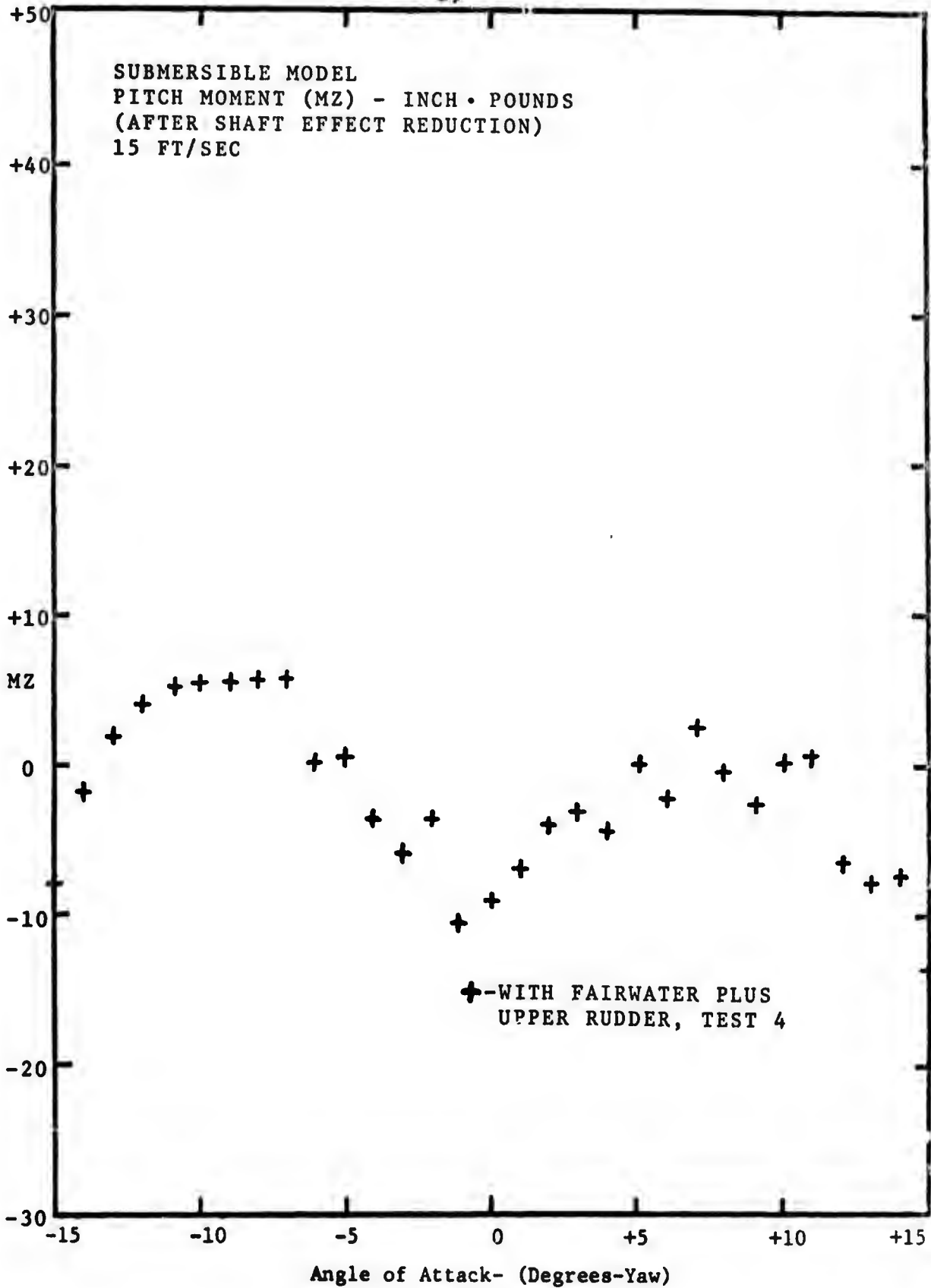


Figure 18

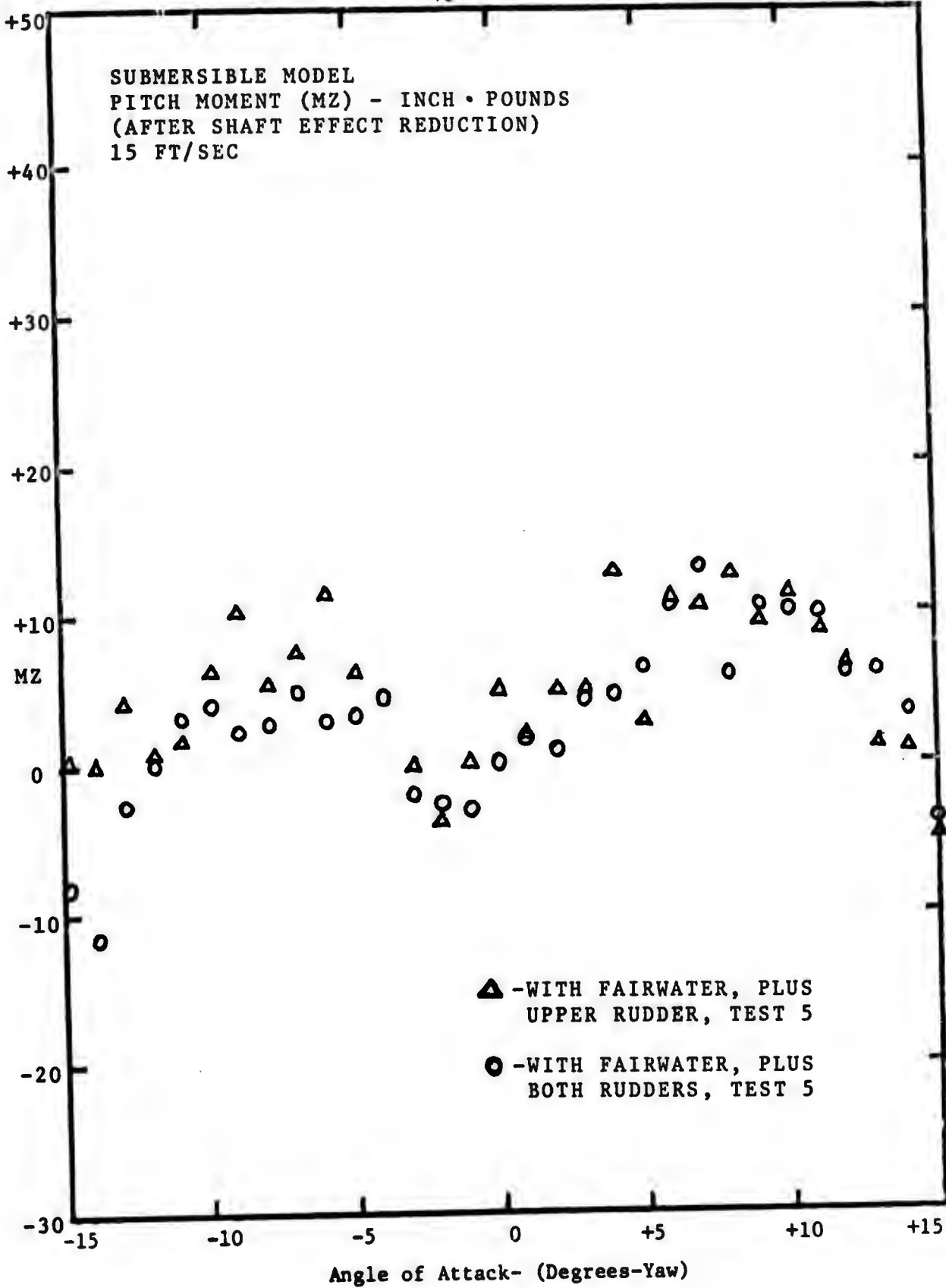


Figure 19

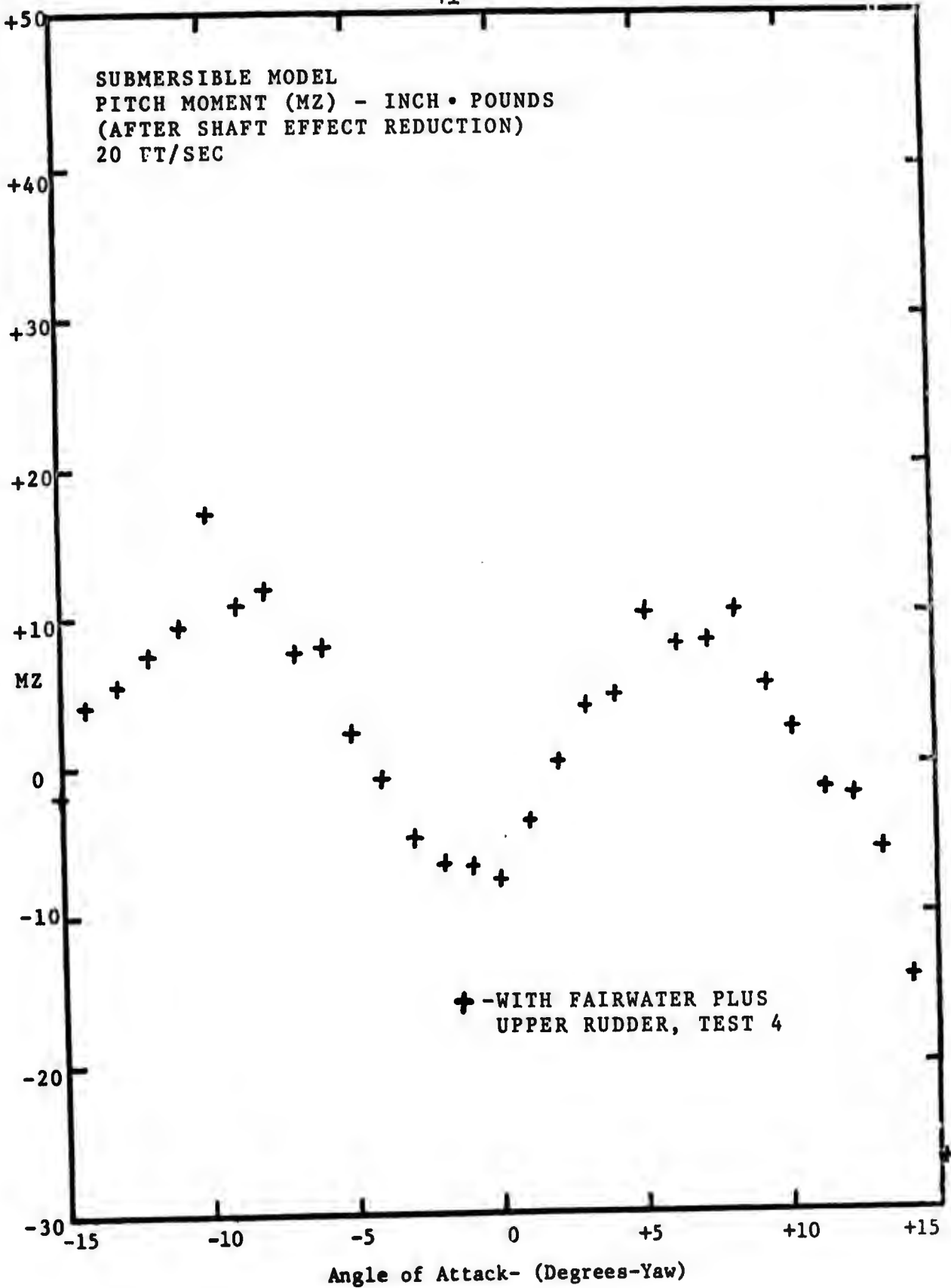


Figure 20

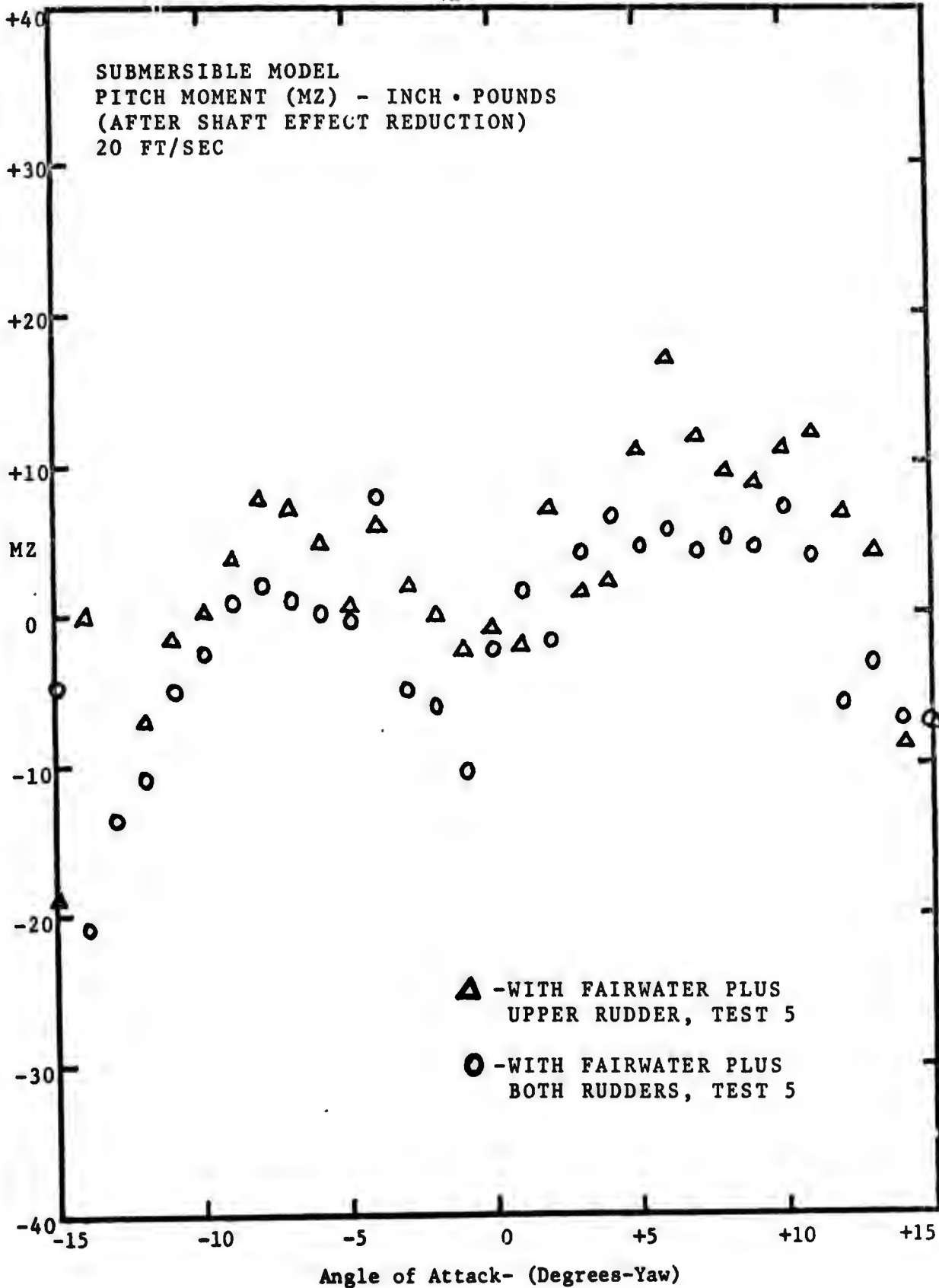


Figure 21

SIDE FORCE (FZ)

1. Shaft Effect

The side force on the bare shaft, although relatively small, was a linear function, and increased with angle of yaw and flow velocity, see Figure 22. The slopes of the lines obtained experimentally were:

A. 1.5 lb/10 degrees = .15 lb/degree, for 15 ft/sec,

B. 2.4 lb/10 degrees = .24 lb/degree, for 20 ft/sec.

An approximation of .05 lb/degree was made for the slope expected for a velocity of 10 ft/sec.

2. Model Results

Every model configuration tested resulted in a side force that increased linearly with angle of yaw and flow velocity. The total side force on a configuration was due to appropriate contributions of side force on the symmetric body of revolution, the fairwater and the rudders, see Figure 23. The side force on the body of revolution results from viscous effects, changes in the crossflow pattern along the length of the model, separation of the flow past the body, and interaction with the fairwater. The fairwater and rudders are essentially lifting surfaces that generate a side force component when placed at an angle of incidence to the flow.

The resulting slopes for the configurations tested were:

<u>Configuration</u>	<u>Velocity (ft/sec)</u>	<u>Slope (lb/degree)</u>	<u>Reference Figure Number</u>
1	15	.50	24
1	20	1.05	25
2	15	1.70	24
2	20	3.20	25
3	15	1.90	27
3	20	3.30	28
4	10	.94	26
4	15	2.00	27
4	20	3.40	28

Although the addition of the fairwater (configuration 2) resulted in a step increase of the side force from that of the clean hull (configuration 1), the addition of the upper rudder (configuration 3) and the lower rudder (configuration 4) resulted in relatively no increase of the side force on the body. The reason for the latter result was found by investigation of the flow past the body. A flow visualization method was used in which the hull body, fairwater and

rudders were tufted, in order to observe and photograph the flow at various angles of yaw, see Figures 29, 30, 31 and 32. The flow past the forward portion of the model showed the increasing crossflow component resulting from increasing angle of yaw as seen by the angle of deflection of the tufts from the body axis. Yet, the flow past the aft portion of the model appears as that observed for a reduced angle of yaw. The difference between these two sections is the location of the fairwater. The change in the flow pattern, resulting from the fairwater shedded wake and its induced velocity, reduces the effective angle of yaw on the upper rudder. The upper rudder is defined as that which is on the same side of the submersible as the fairwater.

The lower rudder, on the opposite side from the fairwater, experiences a similar effect due to the presence of the support shaft, but the magnitude is not as great. The flow visualization test was performed at three water tunnel velocities - 10 ft/sec, 15 ft/sec, and 20 ft/sec - and until separation on the fairwater and both rudders occurred. The flow patterns and initiation of separation on the lifting surfaces did not depend on flow velocity, but only on angle of yaw. The initiation of separation occurred at different angles of yaw for each of the three lifting surfaces. Separation on the fairwater was observed at an angle of yaw equal to 16.5 degrees. At this angle, neither of the rudder surfaces showed any

signs of separation. Separation on the lower rudder appeared when the body of the model was placed at 20 degrees yaw angle. The upper rudder still showed no signs of separation. The angle of yaw was increased until separation was observed at a model yaw angle of 29 degrees, see Figure 32c. Figure 32d was included to show the flow pattern along the model at a negative angle of yaw (15 degrees).

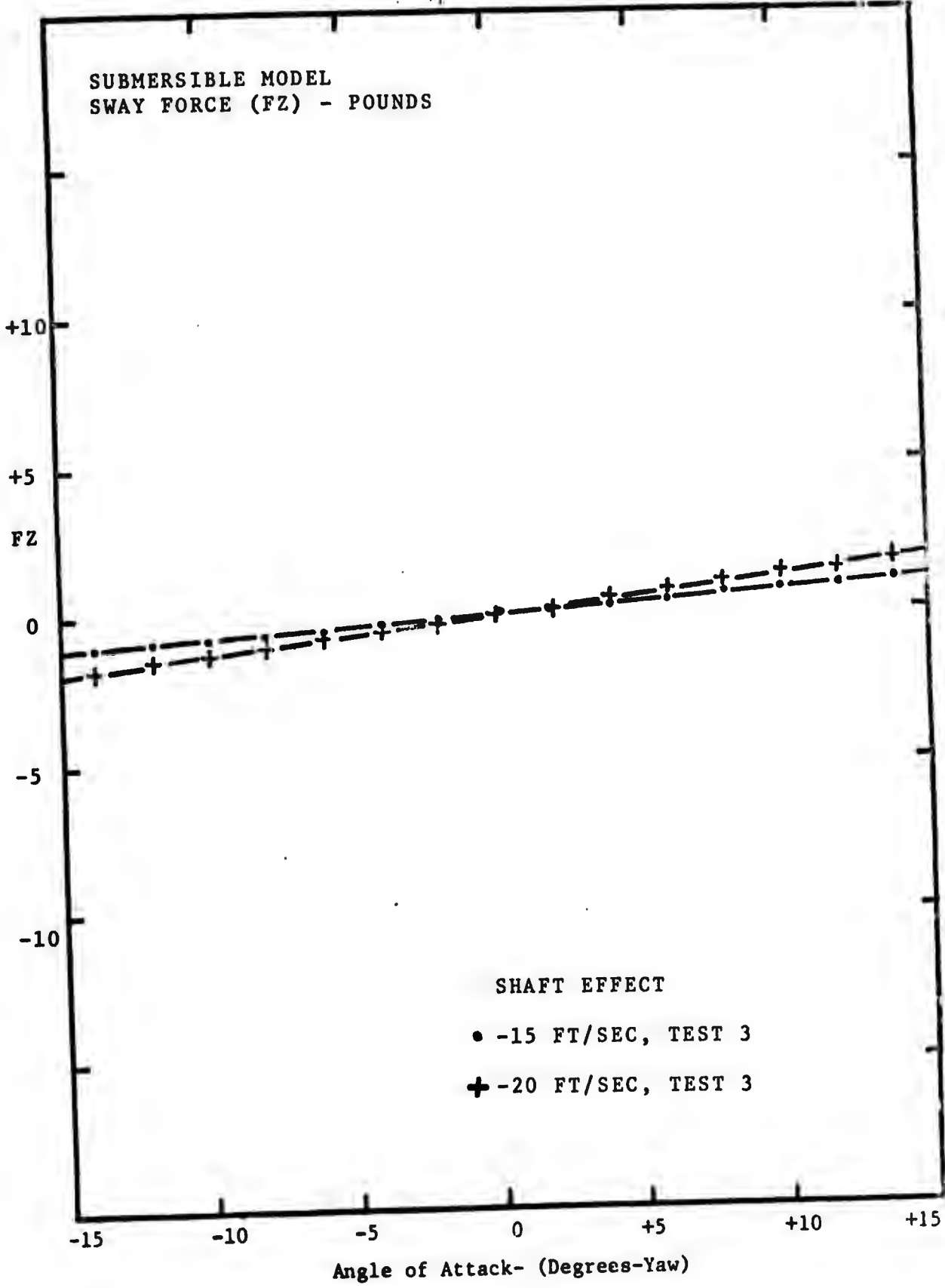
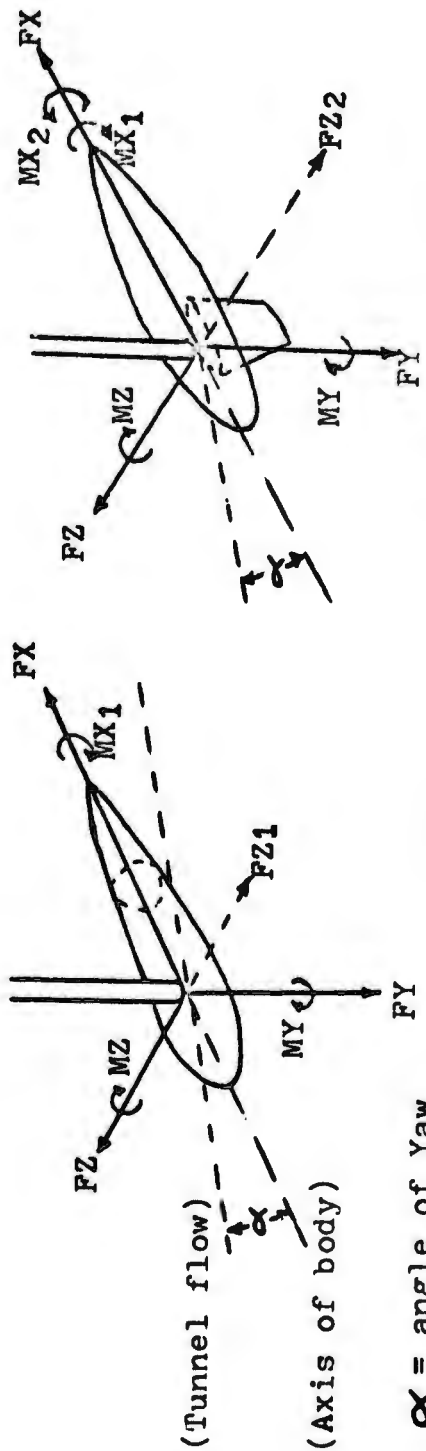


Figure 22



α = angle of Yaw

a. Without fairwater

b. With fairwater

a. MX_1 = Roll moment about body axis, resulting from non-symmetry of streamlined body, (the supporting shaft)

FZ_1 = Side force on the body, due to form drag at angles of Yaw

b. MX_2 = Roll moment about body axis, resulting from non-symmetry of added fairwater, (Net roll moment on model, $MX = \sum MX_i$; in above case, $MX = MX_2 - MX_1$)

FZ_2 = Side force resulting from the addition of the fairwater to the body, (Net side force, $FZ = \sum FZ_i$; in above case, $FZ = FZ_1 + FZ_2$)

Figure 23 Interpretation of hydrodynamic effects for graphic results

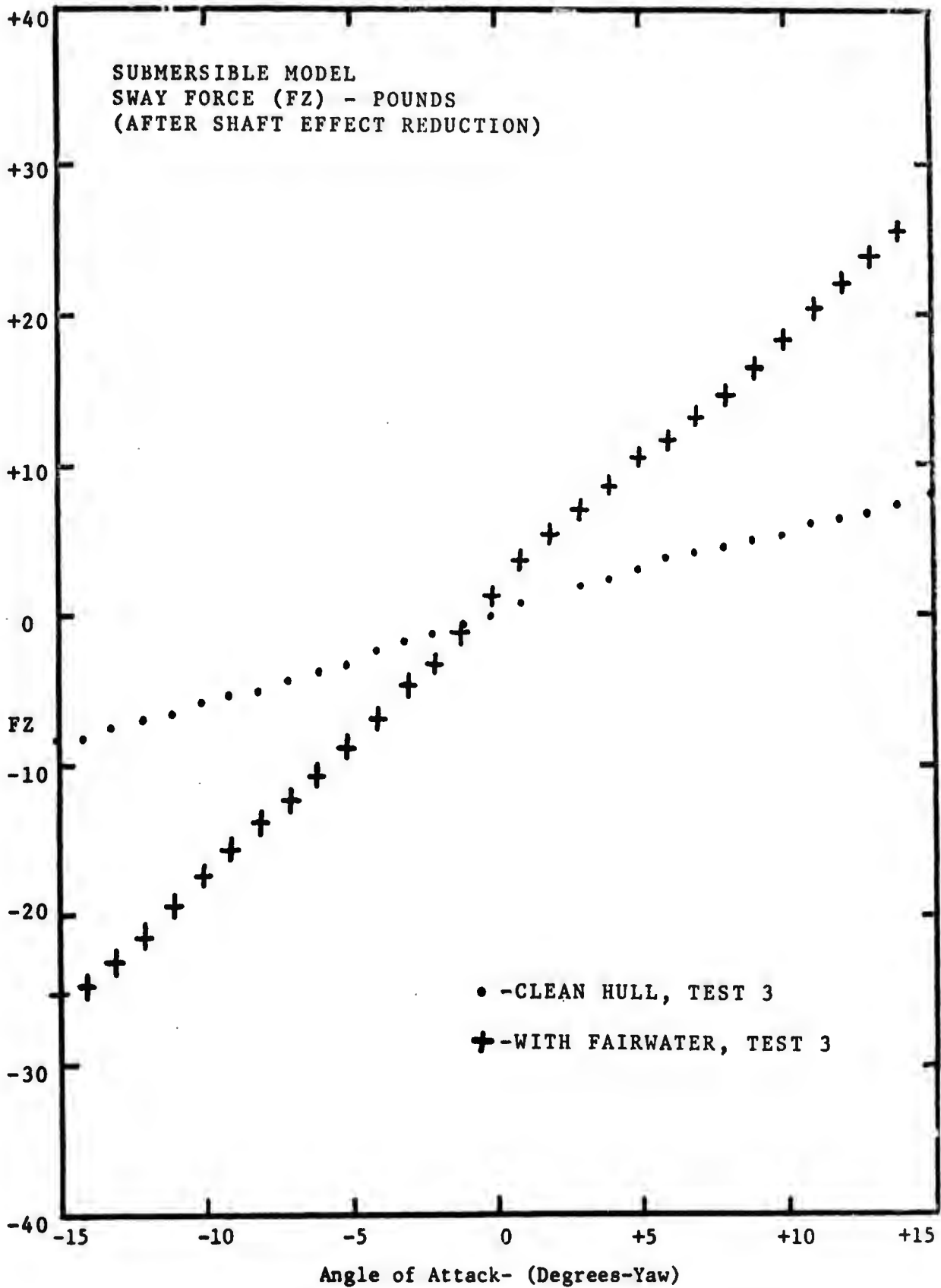


Figure 24

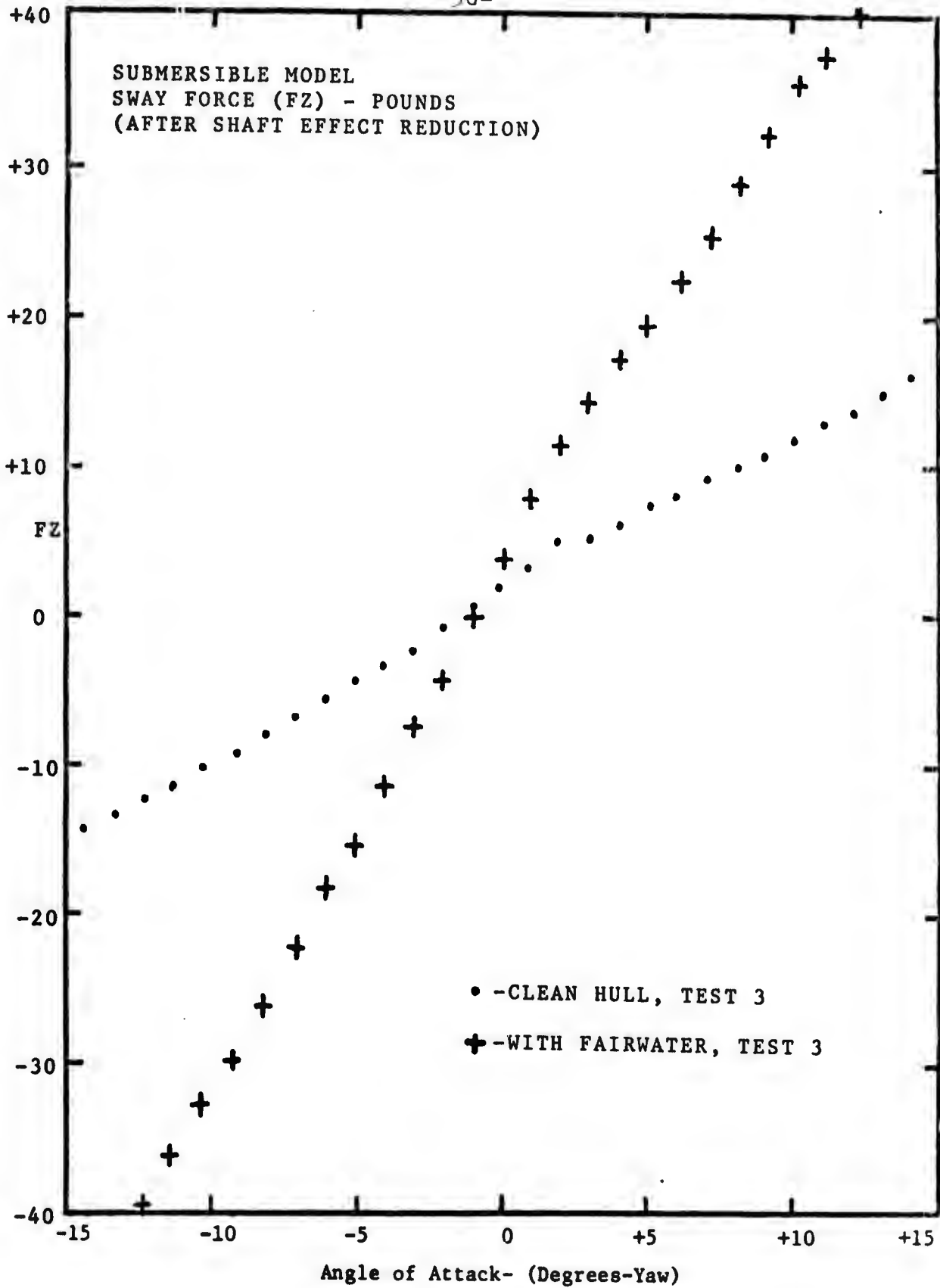


Figure 25

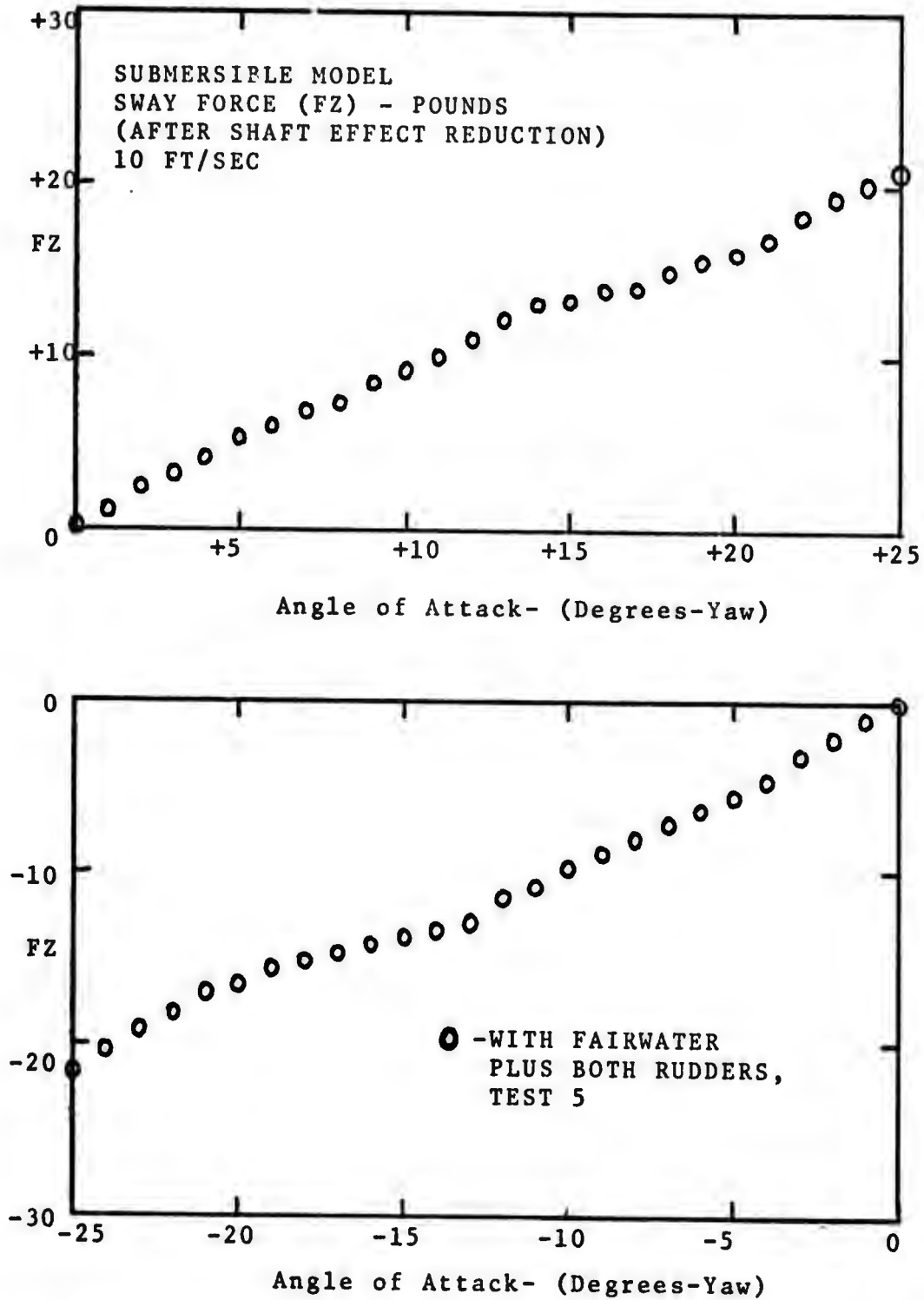


Figure 26

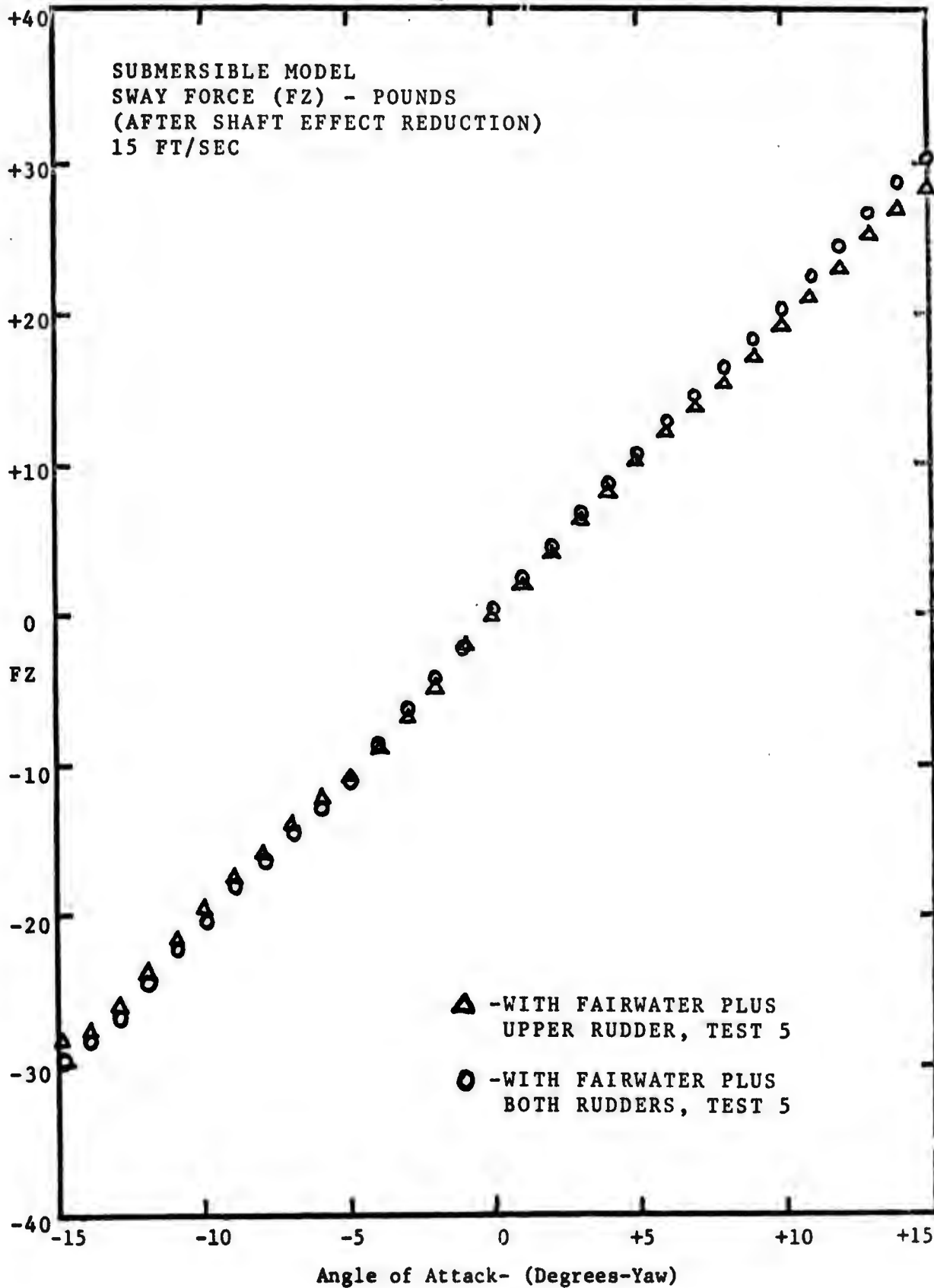


Figure 27

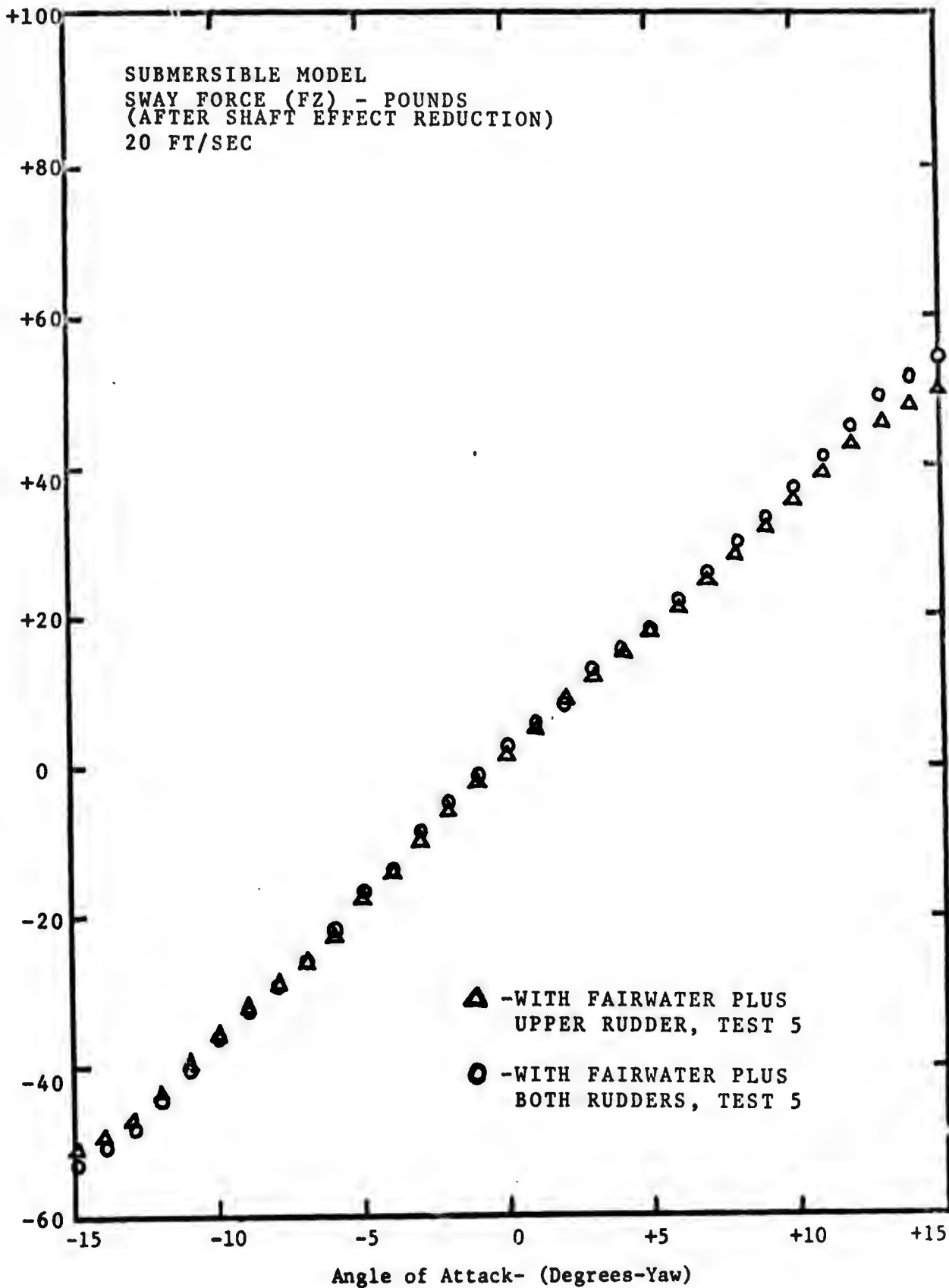
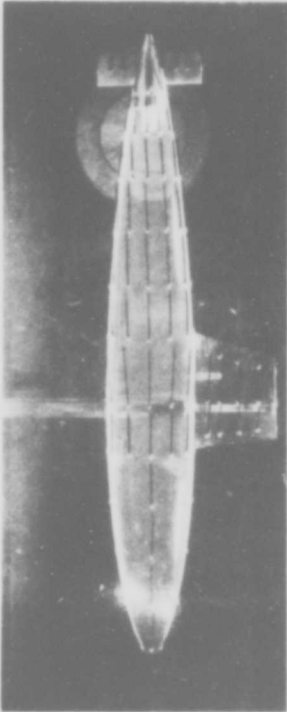
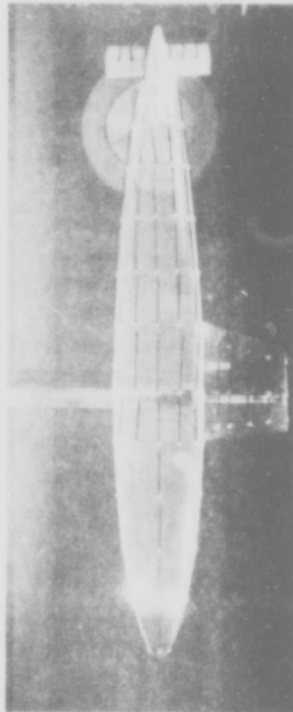


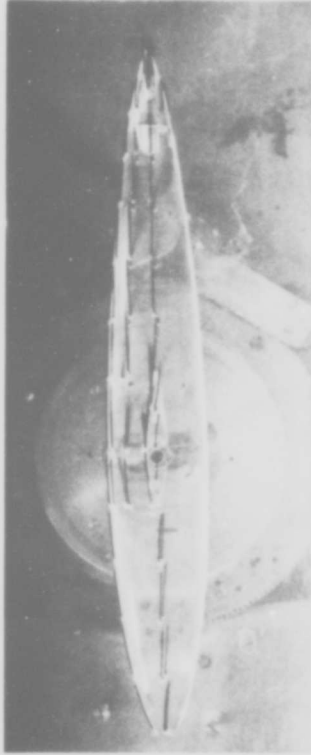
Figure 28



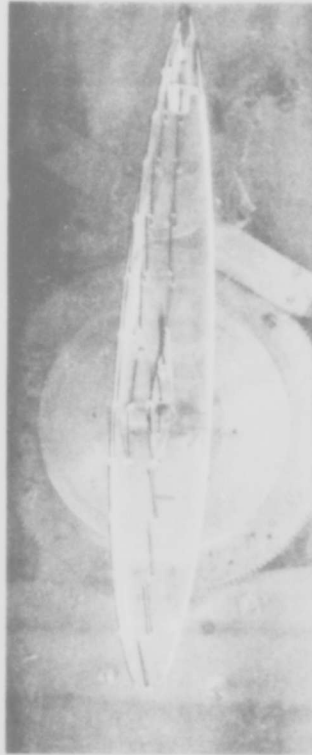
a. When submersible is parallel to water tunnel flow, $\alpha = 0$ degrees



c. When $\alpha = 5$ degrees



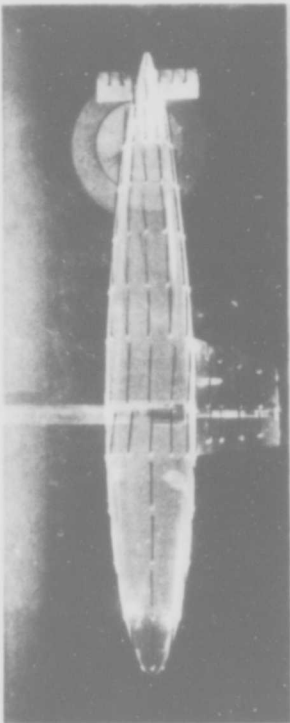
b. When submersible is parallel to water tunnel flow, $\alpha = 0$ degrees



d. When $\alpha = 5$ degrees

(Note: $\alpha =$ Yaw angle)

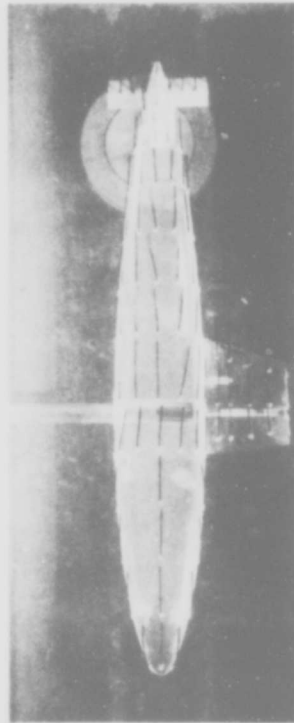
Figure 29 Submersible model with flow visualization (Tufted)



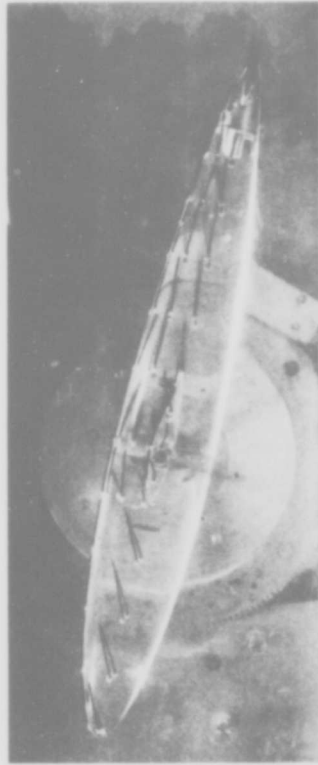
a. When $\alpha = 10$ degrees



b. When $\alpha = 10$ degrees

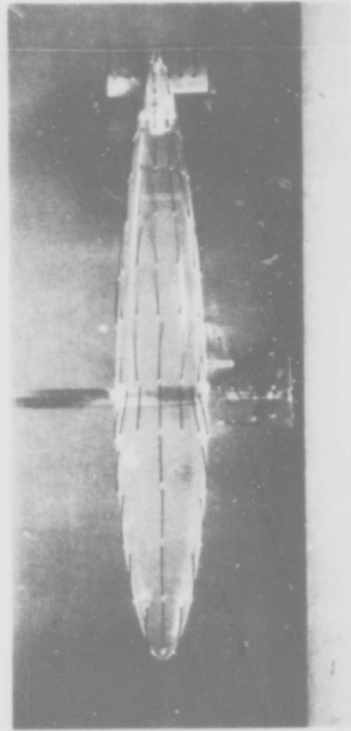


c. When $\alpha = 15$ degrees



d. When $\alpha = 15$ degrees

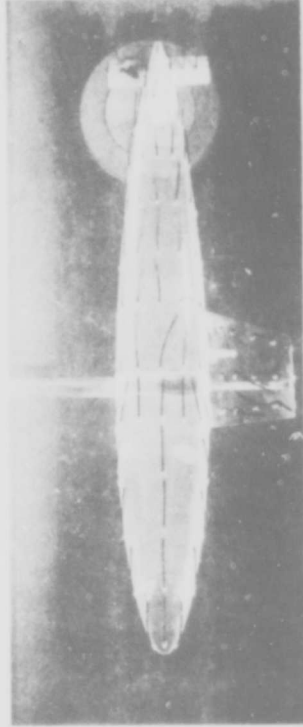
Figure 30 Submersible model with flow visualization (Tufted)



a. Separation on fairwater, but not on either of the rudders, when $\alpha = 16.5$ degrees.



b. Separation on fairwater, but not on either of the rudders, when $\alpha = 16.5$ degrees.

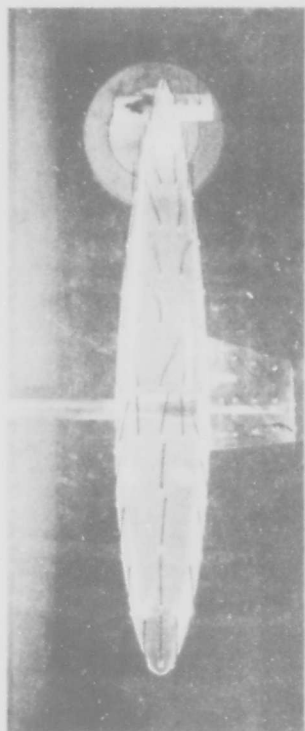


c. Separation on fairwater and one rudder, other rudder still effective, when $\alpha = 20$ degrees.



d. Separation on fairwater and one rudder, other rudder still effective, when $\alpha = 20$ degrees.

Figure 31 Submersible model with flow visualization (Tufted)



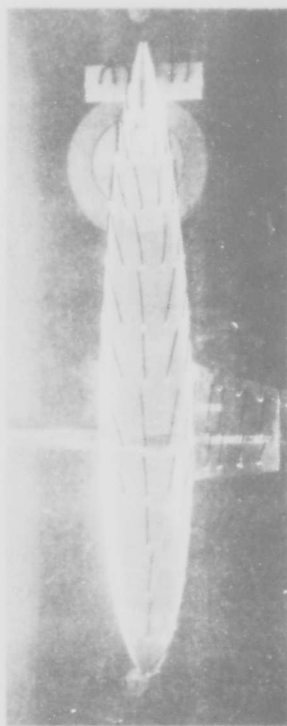
a. Separation on fairwater and one rudder, other rudder still effective, when $\alpha = 25$ degrees.



b. Separation on fairwater and one rudder, other rudder still effective, when $\alpha = 25$ degrees.



c. Separation on all three lifting surfaces is experienced (see backflow), when $\alpha = 29$ degrees.



d. Flow when $\alpha = -15$ degrees; tangle on one rudder resulted from a previous test run.

Figure 32 Submersible model with flow visualization (Tufted)

ROLL MOMENT (MX)

The roll moment on a submerged body of revolution at an angle of yaw, with and without appendages, is a direct result of the side forces (FZ) previously reported and the relative location of their concentrations with respect to a reference point. For an actual submersible, the reference point would be the center of gravity, which would be below the body axis of symmetry. The reference point for the model tested was at the origin of the coordinate system, which is on the body axis of symmetry.

1. Shaft Effect

The roll moment obtained on the base shaft decreased with an increasing angle of yaw and increasing flow velocity in a linear fashion, see Figure 33. When a linear result was approximated, the slopes of the lines formed were:

- A. Slope = $-7 \text{ in. lb}/10 \text{ degrees} = -.7 \text{ in. lb/degree}$,
for a velocity of 15 ft/sec.
- B. Slope = $-13.5 \text{ in. lb}/10 \text{ degrees} = -1.35 \text{ in. lb/degrees}$,
for a velocity of 20 ft/sec.

2. Model Results

Model configuration 1 (clean hull) is symmetrical in all respects, unlike an actual submersible of comparable configuration, whose center of gravity is below the body axis, for

roll stability. This caused an experimental result of zero roll moment for the clean hull at all angles of yaw and all velocities evaluated, see Figures 34 and 35. In actuality, a roll moment would occur on a submersible, due to the asymmetry described above. This fact is reflected in the results of all subsequent configurations, in the form of a difference in total magnitude of roll moment.

With the addition of the fairwater (configuration 2), an increasing roll moment is encountered with both increasing flow velocity and increasing angle of yaw, see Figures 34 and 35. The increase was approximately linear with the following slopes:

- A. Slope = 28 in. lb/10 degrees = 2.8 in. lb/degree,
for a velocity of 15 ft/sec.
- B. Slope = 54 in. lb/10 degrees = 5.4 in. lb/degree,
for a velocity of 20 ft/sec.

A decrease in the magnitude of the roll moment and a reversal of slope was noticeable when the model was placed in a yaw angle of about 15 degrees. This was caused by the separation on the fairwater, and the resulting loss of side force (lift component).

Configuration 3 (with fairwater and upper rudder) produced very little change in the roll moment on the model, see Figures 36 and 37. This is a result of a decrease in the effective angle of yaw on the rudder by the perturbations in

the flow due to the fairwater and its shedded wake, see Figures 29, 30, 31 and 32 for flow visualization. A decrease in moment, caused by separation on the fairwater, was experienced again at about 15 degrees yaw angle.

Configuration 4 (with fairwater and both rudders) decreased the magnitude of the roll moment, but only slightly, see Figures 36 and 37. A greater reduction in roll moment should have been obtained for the lifting surface added, but as in configuration 3, the effective angle of yaw on the lifting surface (rudder) in question was reduced. In this case, the support shaft sheds a wake, which interacts with the in-flow velocity on the lower rudder. A similar loss of lift on the fairwater, as in configurations 2 and 3, was obtained for configuration 4.

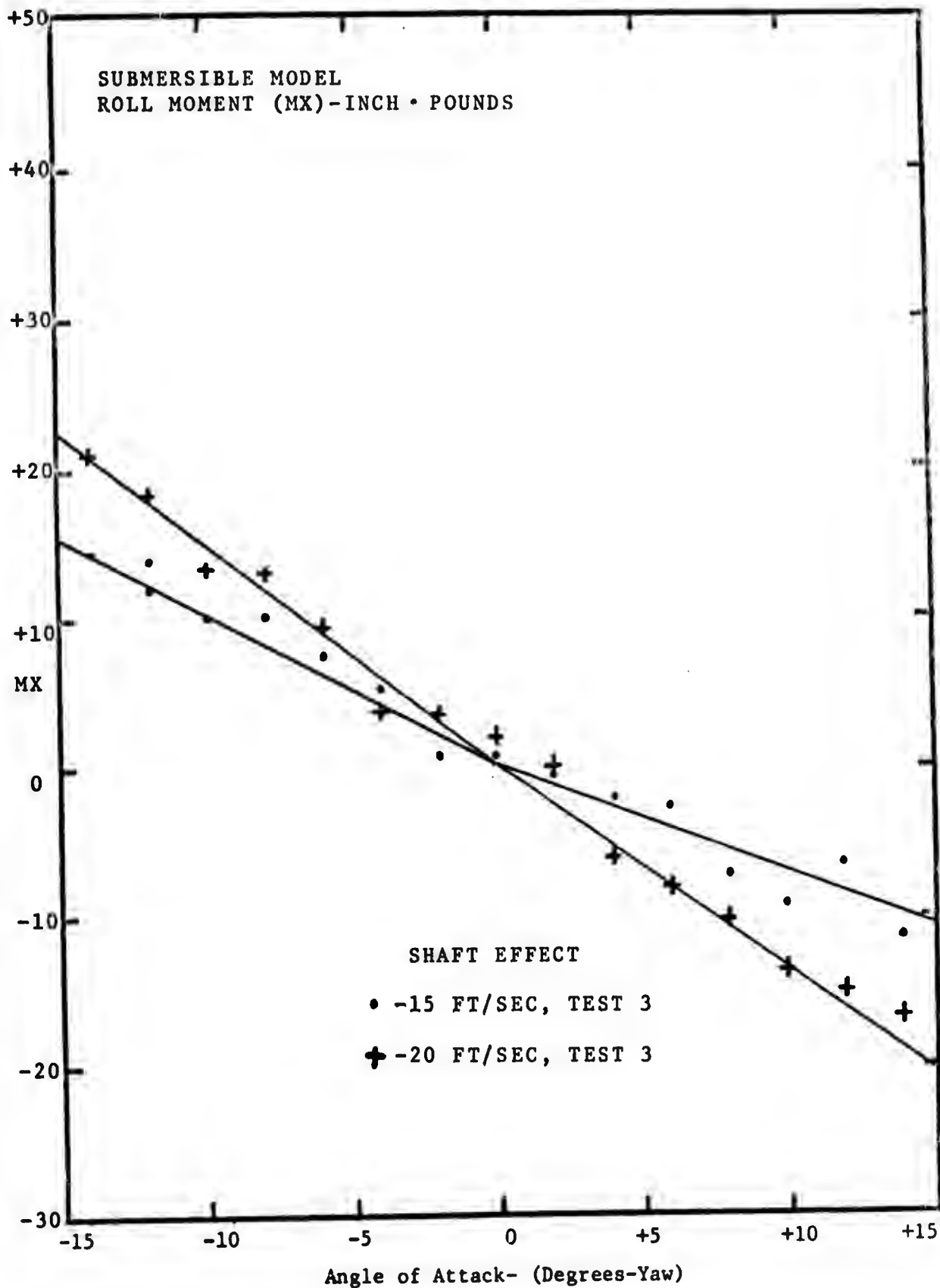


Figure 33

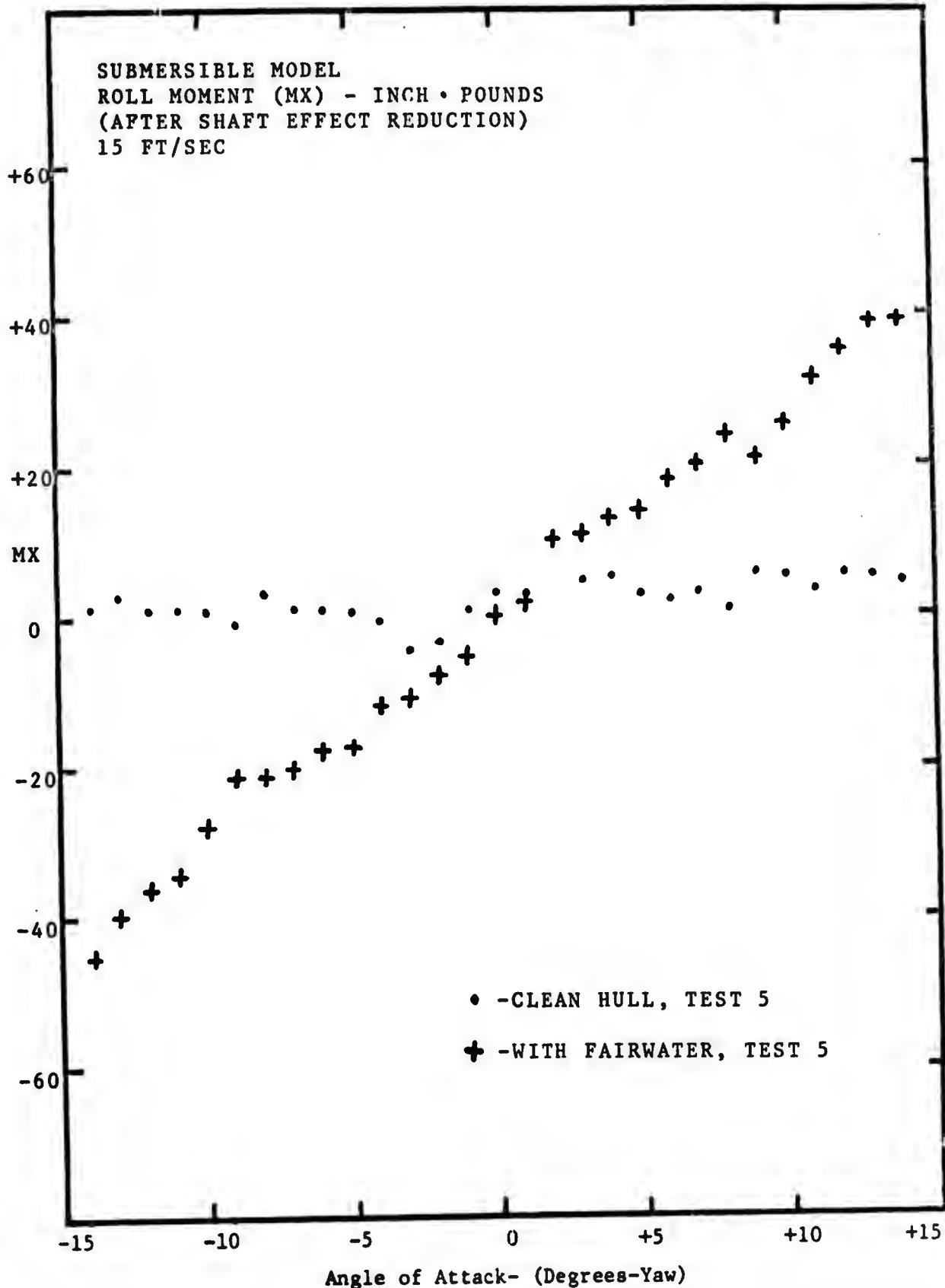


Figure 34

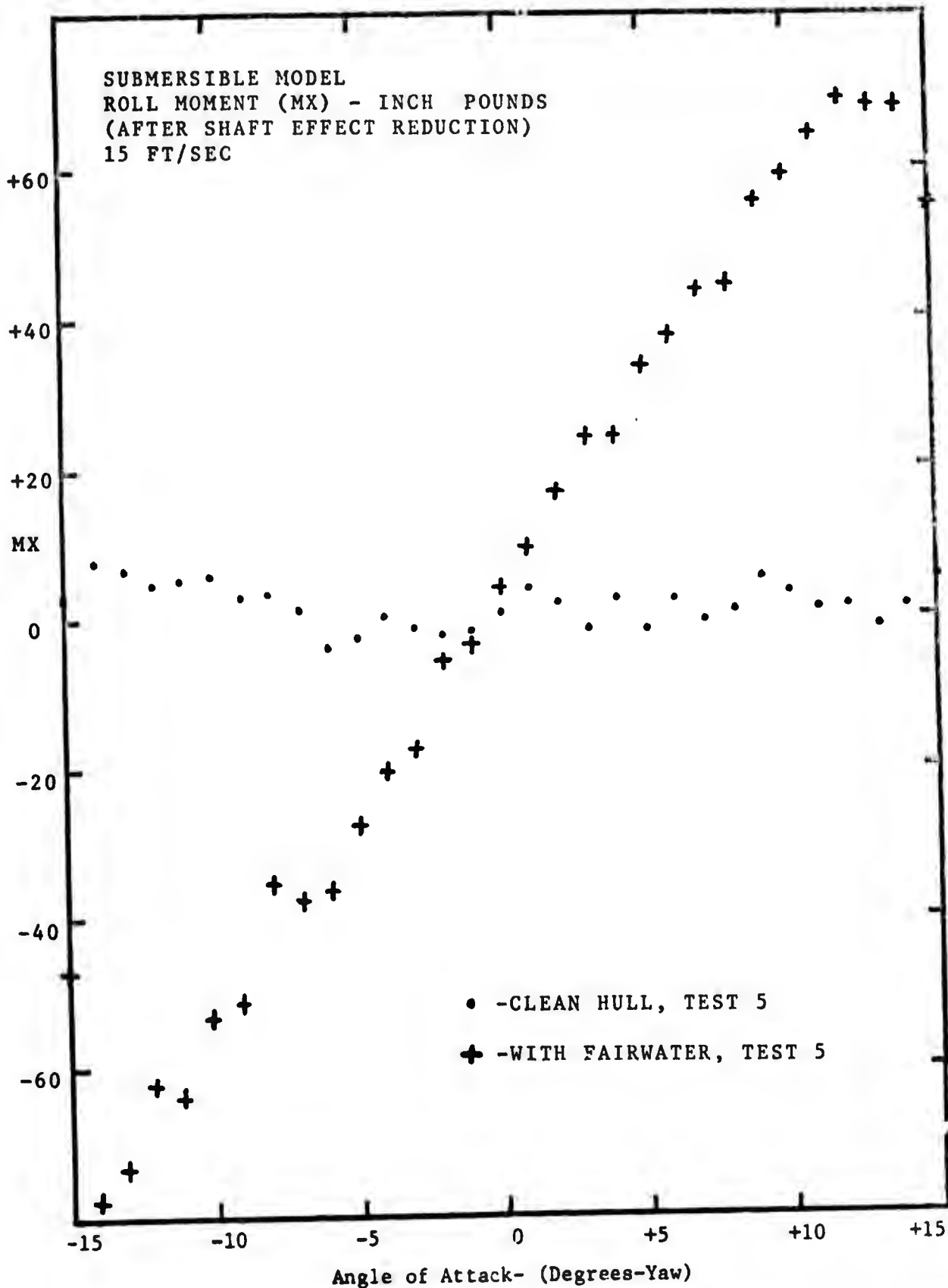


Figure 35

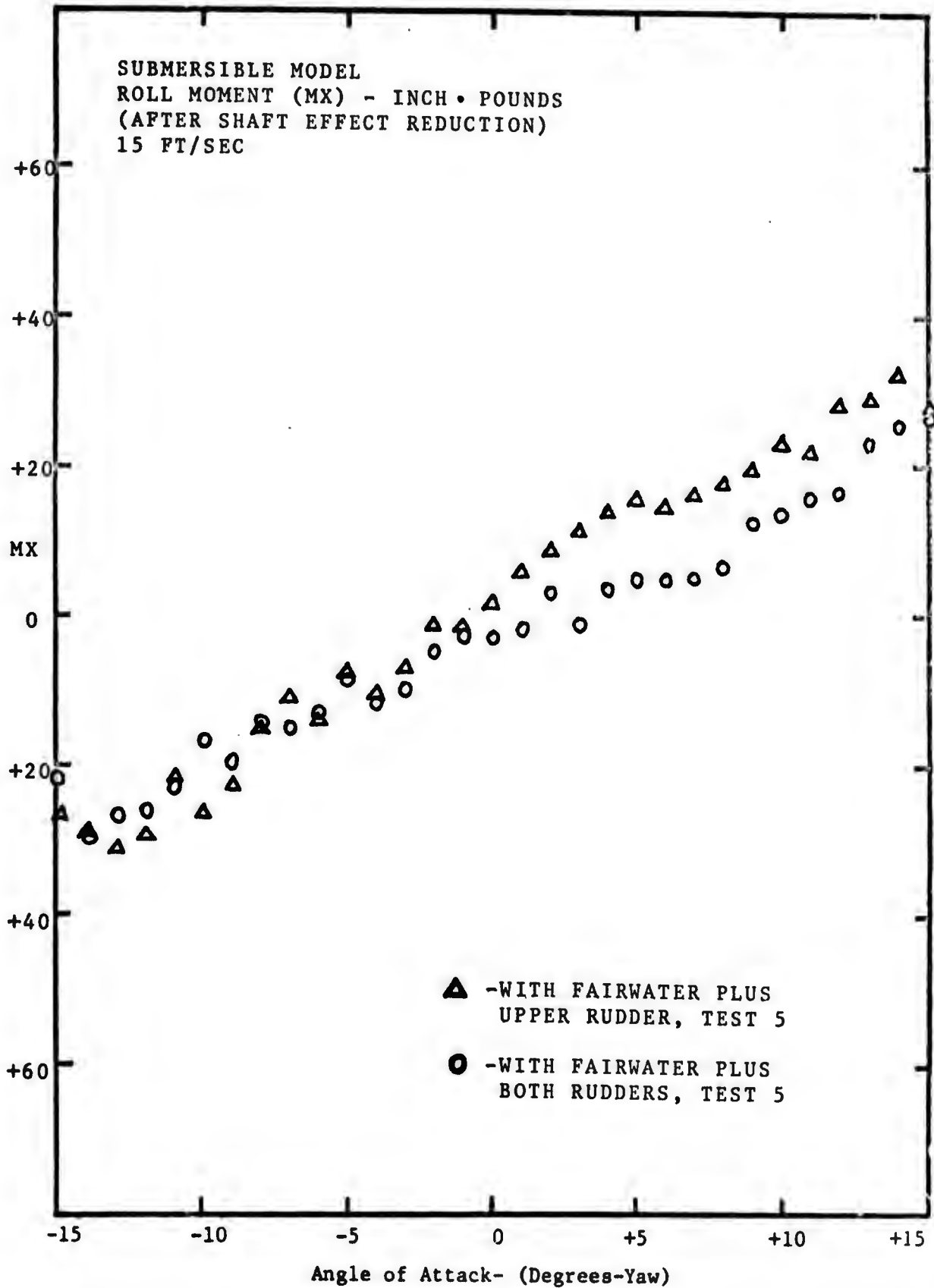


Figure 36

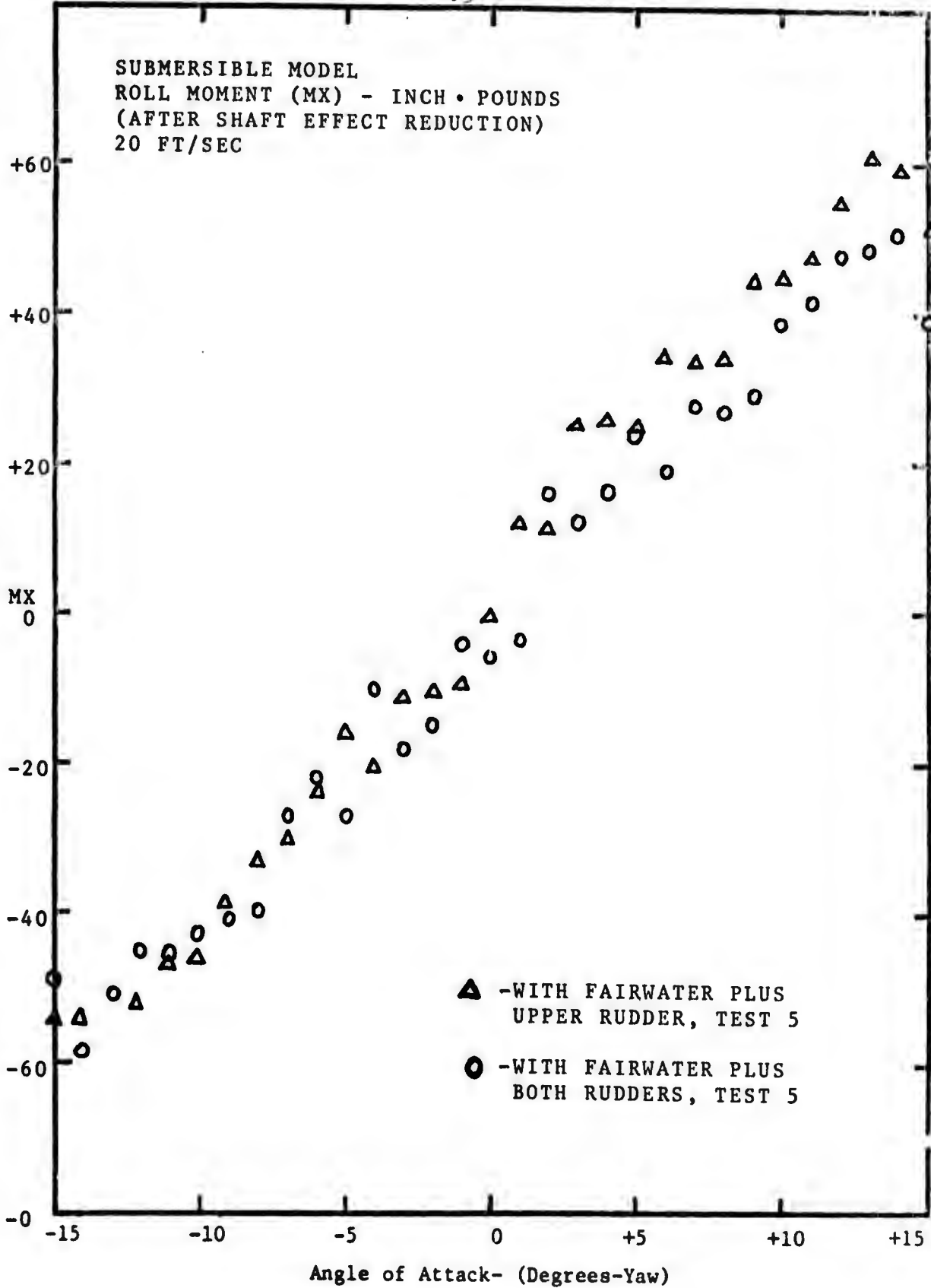


Figure 37

YAW MOMENT (MYO)

1. Shaft Effect

No shaft effect was, or should be, obtained for yaw moment, as this moment is measured about the shaft axis of symmetry.

2. Model Results

Configuration 1 (clean hull) produced a linear result that increased with increasing angle of yaw and flow velocity, see Figure 38. Yaw moment components resulted from form drag and changes in the cross flow pattern along the length of the model. The resulting slopes for the two velocities evaluated were:

- A. Slope = 43 in. lb/10 degrees = 4.3 in. lb/degree,
for 15 ft/sec,
- B. Slope = 80 in. lb/10 degrees = 8.0 in. lb/degree,
for 20 ft/sec.

When the fairwater (configuration 2) was added, there appears to be no effect until a yaw angle of 4 degrees is reached. Then, the yaw moment increases at a greater rate until the fairwater incurs separation (about 15 degrees yaw angle) and the yaw moment decreases until it reaches the magnitude obtained for the clean hull at that angle of yaw, see Figure 39.

When configuration 3 (with fairwater and upper rudder)

was tested, the magnitude of the yaw moment was decreased slightly due to the opposing yaw moment component from the upper rudder, see Figure 40. The yaw component from the rudder would have been larger if the fairwater had not caused any interaction, as previously described, to reduce the side force on the rudder. This configuration follows the same pattern as configuration 2; it incurs a rapid change in slope when the fairwater separates, but at a lower magnitude.

Configuration 4 (with fairwater and both rudders) produced a very powerful effect on reduction of yaw moment, see Figure 41. The side force developed by the addition of the lower rudder, combined with its moment arm, produces a yaw component that eventually supercedes that caused by the fairwater and forward portion of the body of revolution. When this model configuration was tested at an extreme yaw angle of 25 degrees, in both directions, and a flow velocity of 10 ft/sec, the resulting yaw moment was zero. The result for positive angles of yaw can be seen in Figure 42. Negative angles of yaw produced symmetrical results.

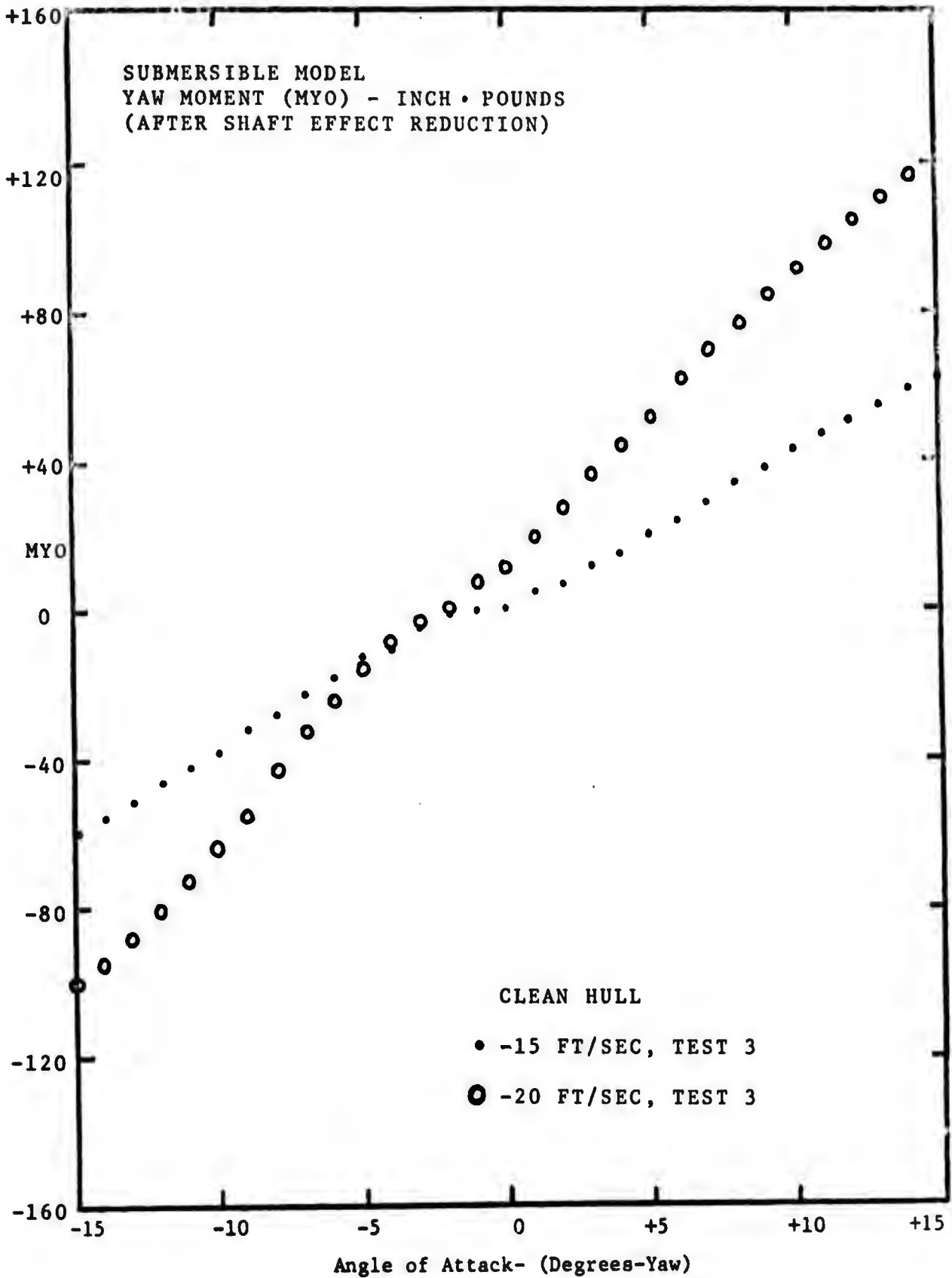


Figure 38

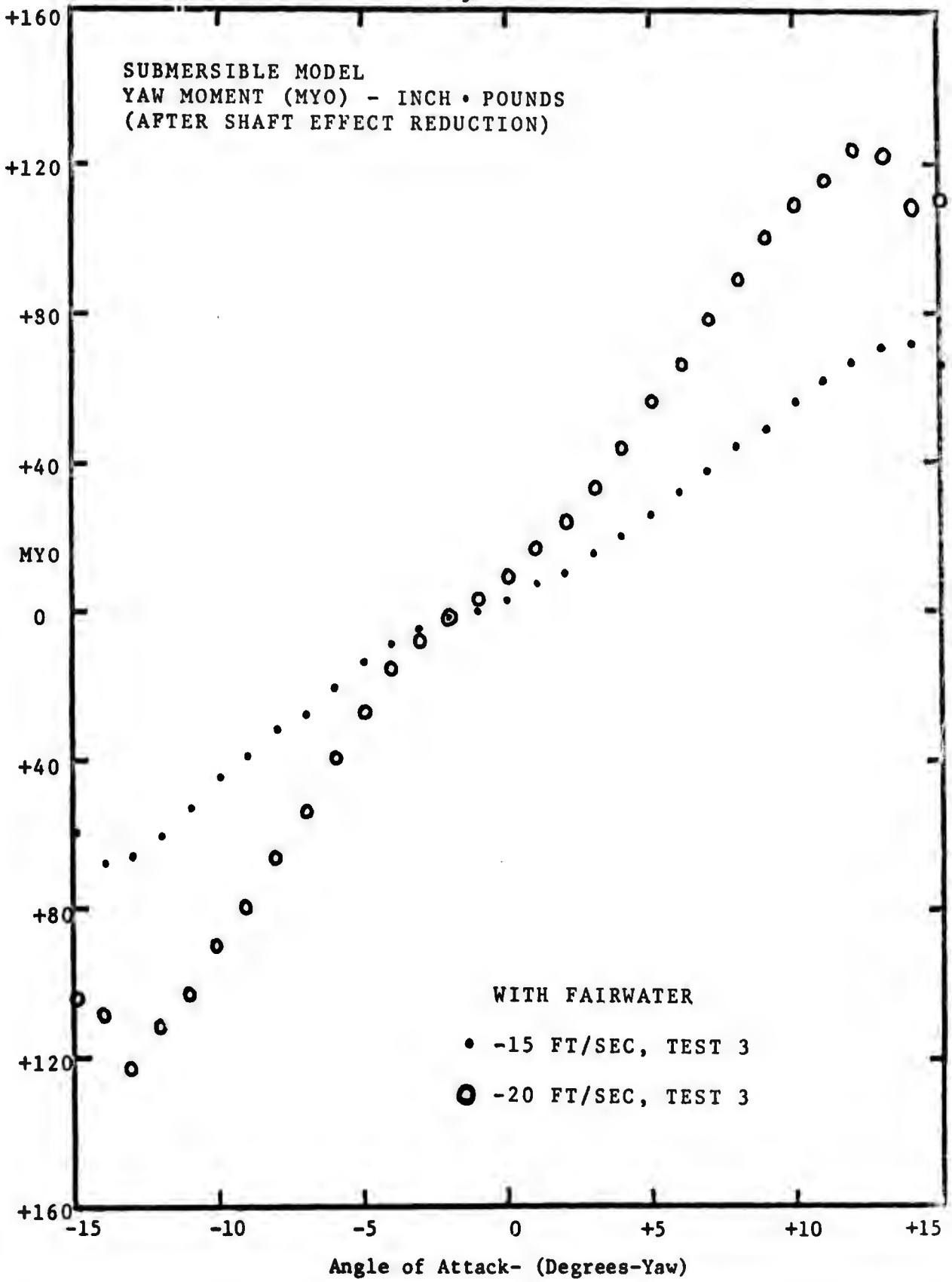


Figure 39

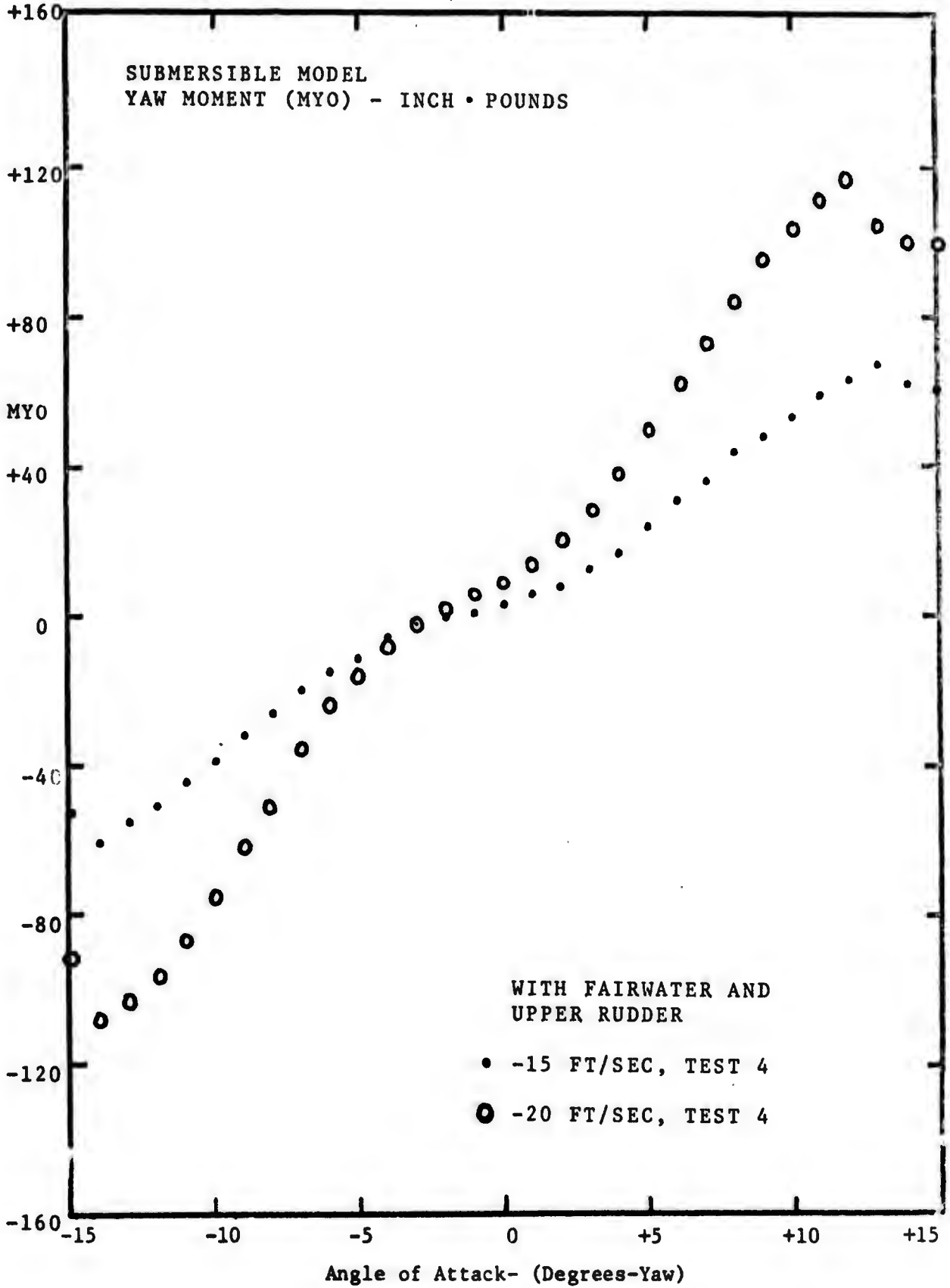


Figure 40

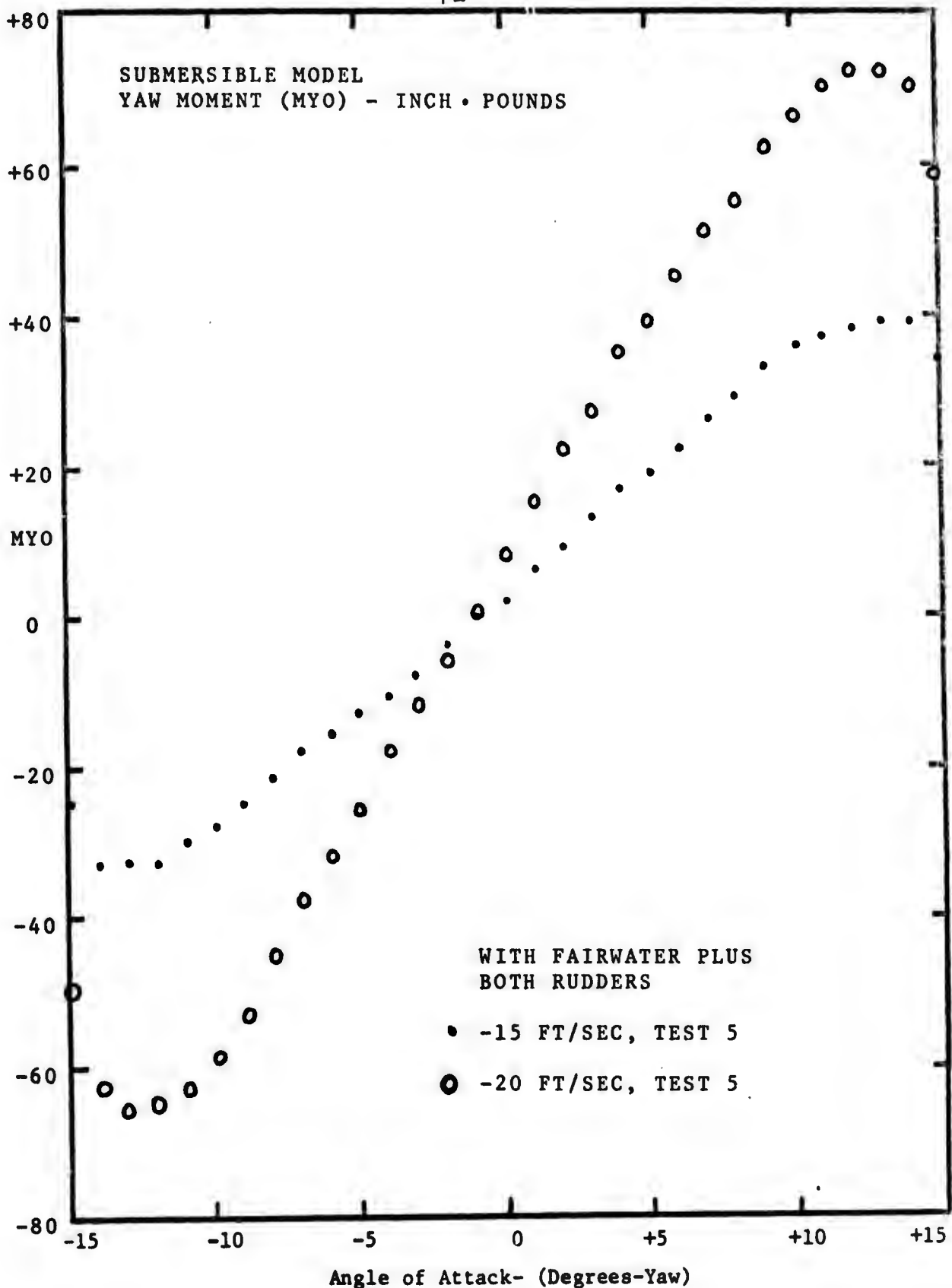


Figure 41

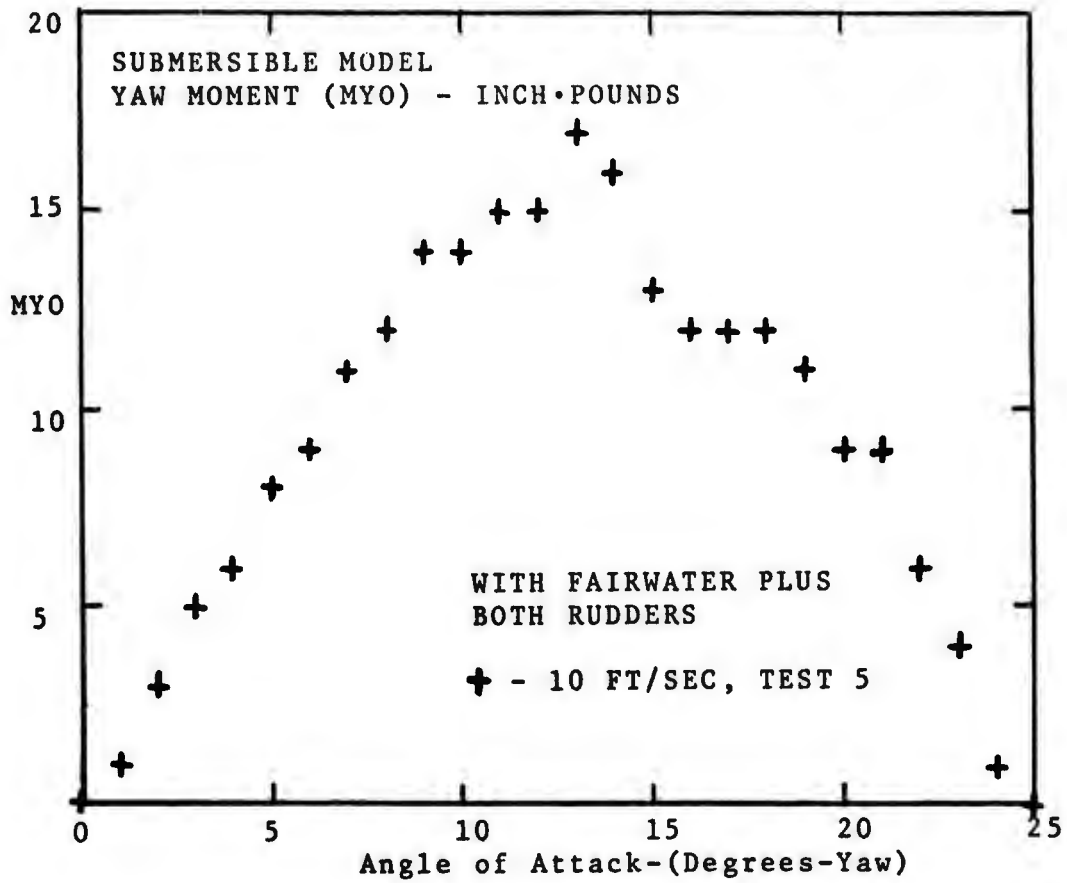


Figure 42

V. FAIRWATER MODEL EXPERIMENTAL RESULTS

The second set of experiments conducted in the M.I.T. Variable Pressure Water Tunnel consisted of testing the fairwater model, without control surfaces and then with control surfaces attached. The control surfaces were attached to the fairwater by means of a shaft through both the control surfaces and the fairwater. The shaft was used for transferring hydrodynamic forces and moments on the control surfaces to the fairwater, in effect, to the body of the submersible. To control and maintain particular angles of attack of the control surfaces, a guide pin was installed on each of the two separate control surfaces, between the location of the shaft and trailing edge. Guide holes were placed on each side of the fairwater for angles of attack to be evaluated, see Figure 7a. The particular angles of attack of the control surfaces (α') tested were: 0° , $\pm 5^\circ$, $\pm 10^\circ$, $\pm 15^\circ$, and $\pm 20^\circ$.

Six different tests (configurations) were conducted on the above model, and consisted of:

Test 1 - fairwater clean, no control surfaces, see Figure 7a;

Test 2 - fairwater, with control surfaces at $\alpha' = 0^\circ$, see figure 7b;

Test 3 - fairwater, with control surfaces at opposite angles of attack, $\alpha' = \pm 5^\circ$;

Test 4 - fairwater, with control surfaces at opposite angles of attack, $\alpha' = \pm 10^\circ$;

Test 5 - fairwater, with control surfaces at opposite angles of attack, $\alpha' = \pm 15^\circ$;

Test 6 - fairwater, with control surfaces at opposite angles of attack, $\alpha' = \pm 20^\circ$, see Figures 7c - e.

Each test considered fairwater angles of attack (α) from 0° through stall on both sides of the fairwater, to cover the full spectrum of possible configurations, see Figure 43.

The purpose of these tests was to determine the hydrodynamic effect of using fairwater control surfaces independent of each other to generate side forces and moments to help alleviate the roll-moment (MZ) and side forces (FZ) caused by angle of attack (α) in yaw of the fairwater, which is the same as that of the submersible. These control surfaces are located as close as possible to the fairwater tip, contrary to usual practice on current submersible design.

These tests were developed into more meaningful results by transforming the side forces (FZ) and roll moments (MZ) of the fairwater to the center of an imaginary submersible, to evaluate only the effects of a fairwater and control surfaces on a submersible.

An imaginary submersible moment arm, in the vertical plane, was obtained by scaling the fairwater model with that of an actual submersible. The resulting roll moment:

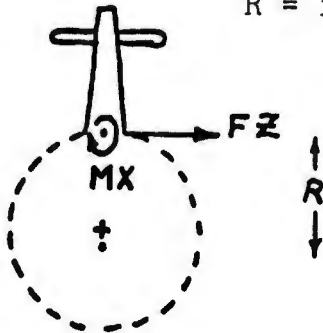
Net Roll Moment = $MX + (FZ) \cdot (R)$,

where:

MX = roll moment at fairwater base

FZ = side force at fairwater base

R = imaginary submersible moment arm = 6.3".



The results were: for an increasing angle of attack (α') of the control surfaces, the effective angle of attack (α) of the fairwater (submersible) in yaw is reduced. For a maximum angle of attack (α') of $\pm 20^\circ$, the reduction is approximately 4.5° for the first series of model test, see Figures 44 - 49.

This first series of tests of the fairwater used control surfaces with non-tapered tips. A second series of tests was performed with tapered end tips added, to alleviate some of the possible drag of the flat-ended configuration of the control surfaces. The results of this second series of tests showed an increase in desired effect, or a decrease in effective angle of attack of the fairwater from the first series of tests. This difference in the second test was approximately 2° beyond the first, see Figures 50 and 51.

Compatible results for lift (FZ_0 - component perpendicular to tunnel flow) and drag (FX_0 - component parallel to

tunnel flow) on the fairwater model for the flat tip and tapered tip fairwater control surfaces are given in Figures 52 - 59, and Figures 60 - 63 respectively.

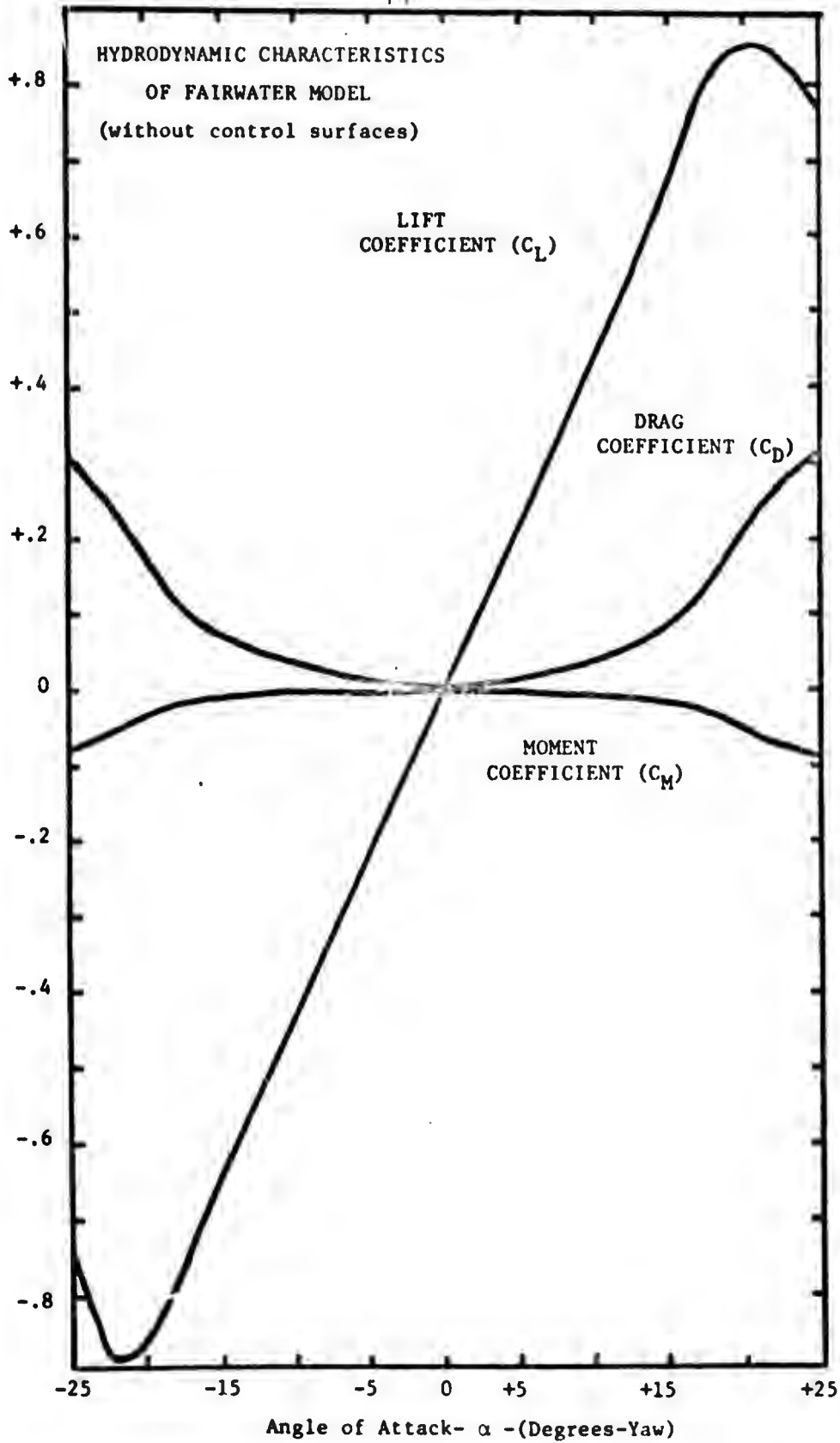


Figure 43

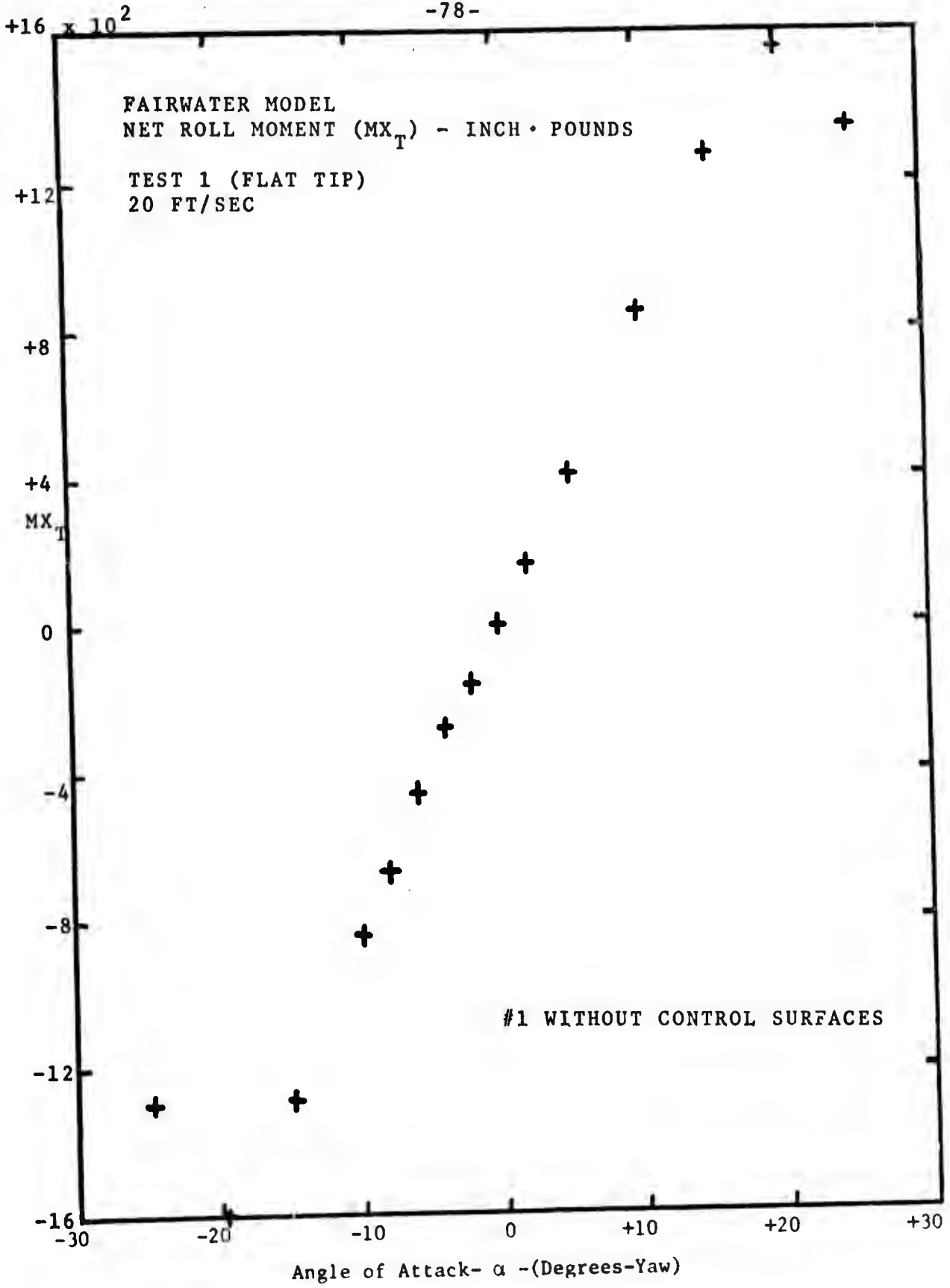


Figure 44

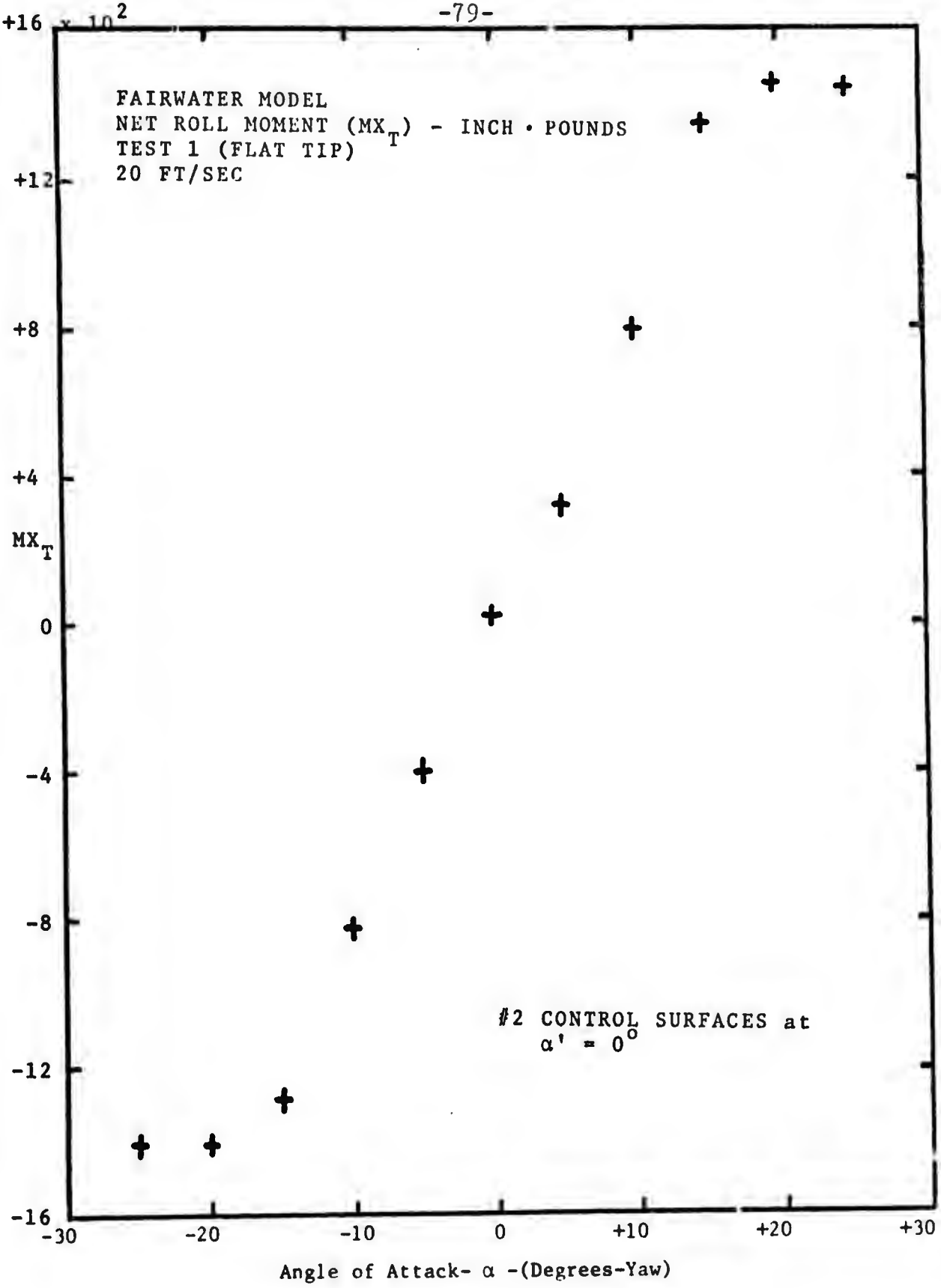


Figure 45

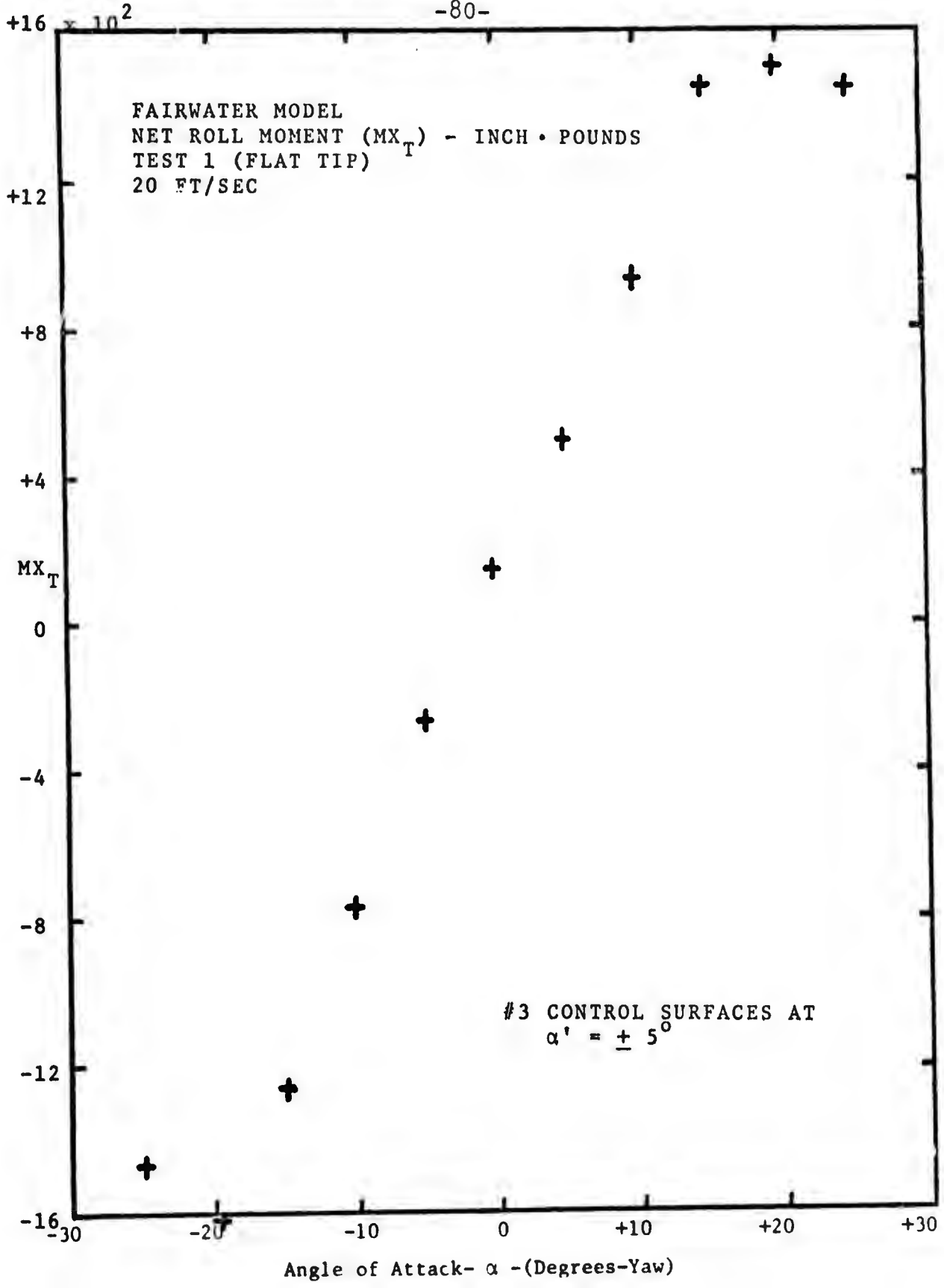


Figure 46

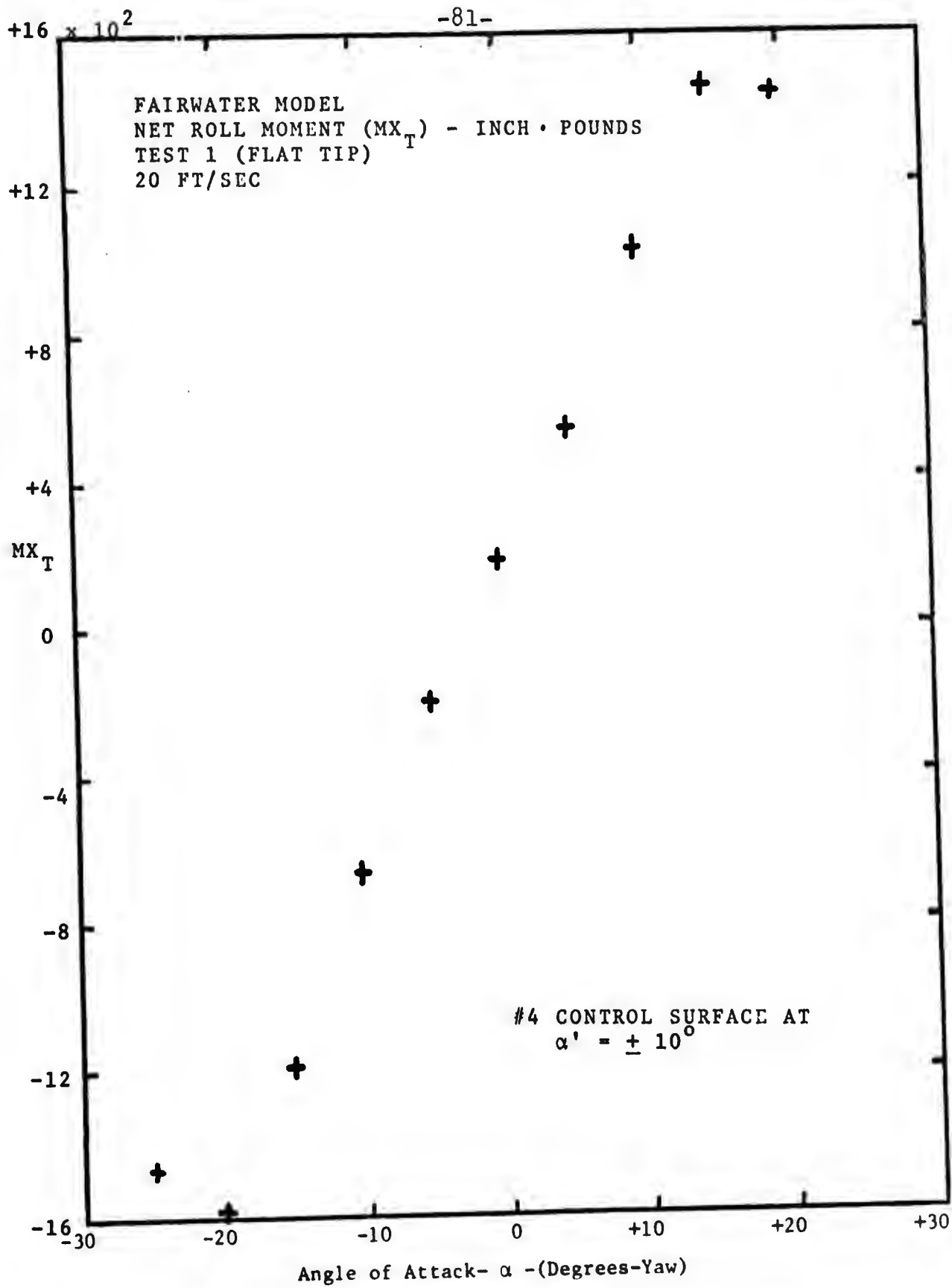


Figure 47

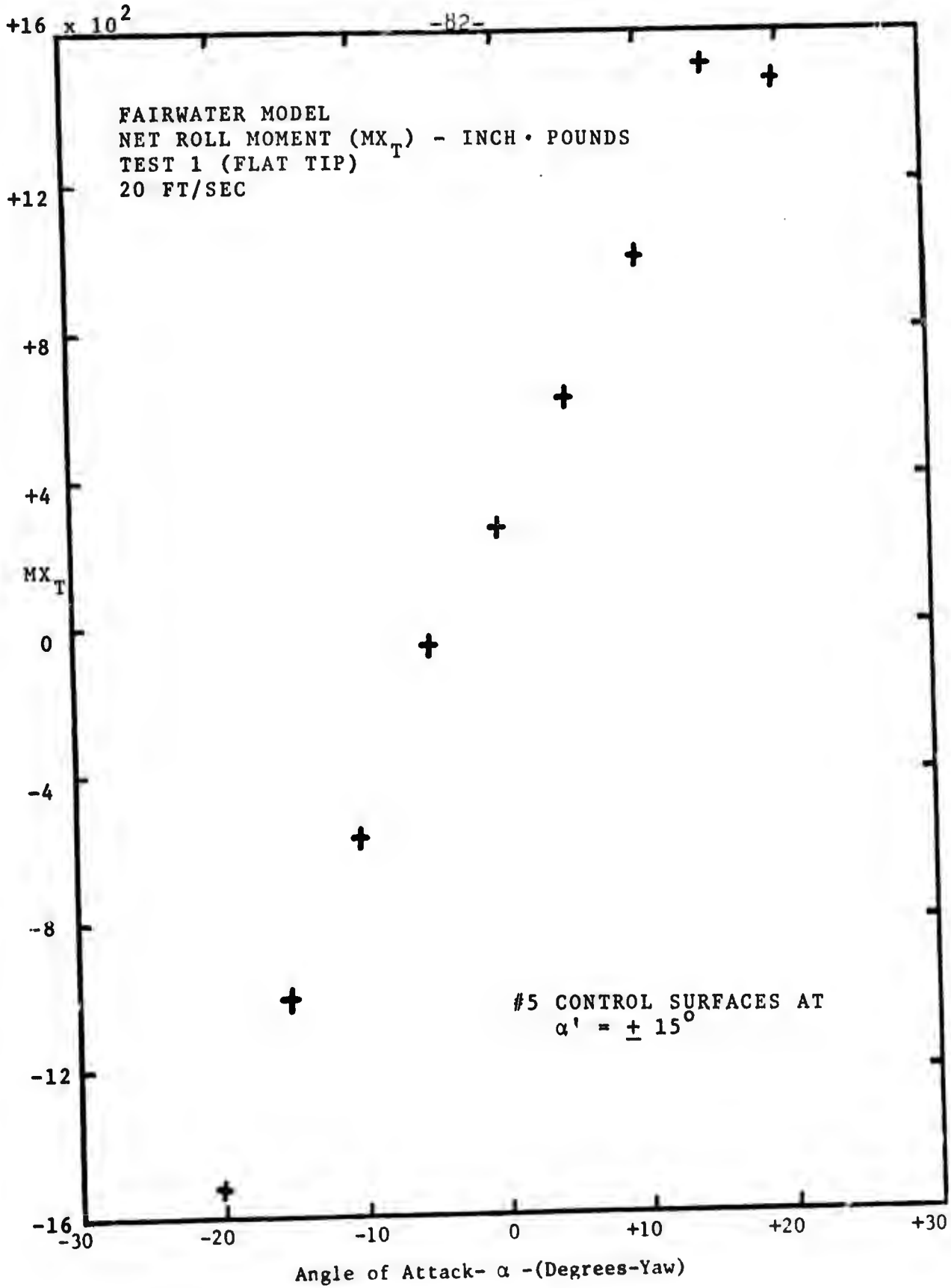


Figure 48

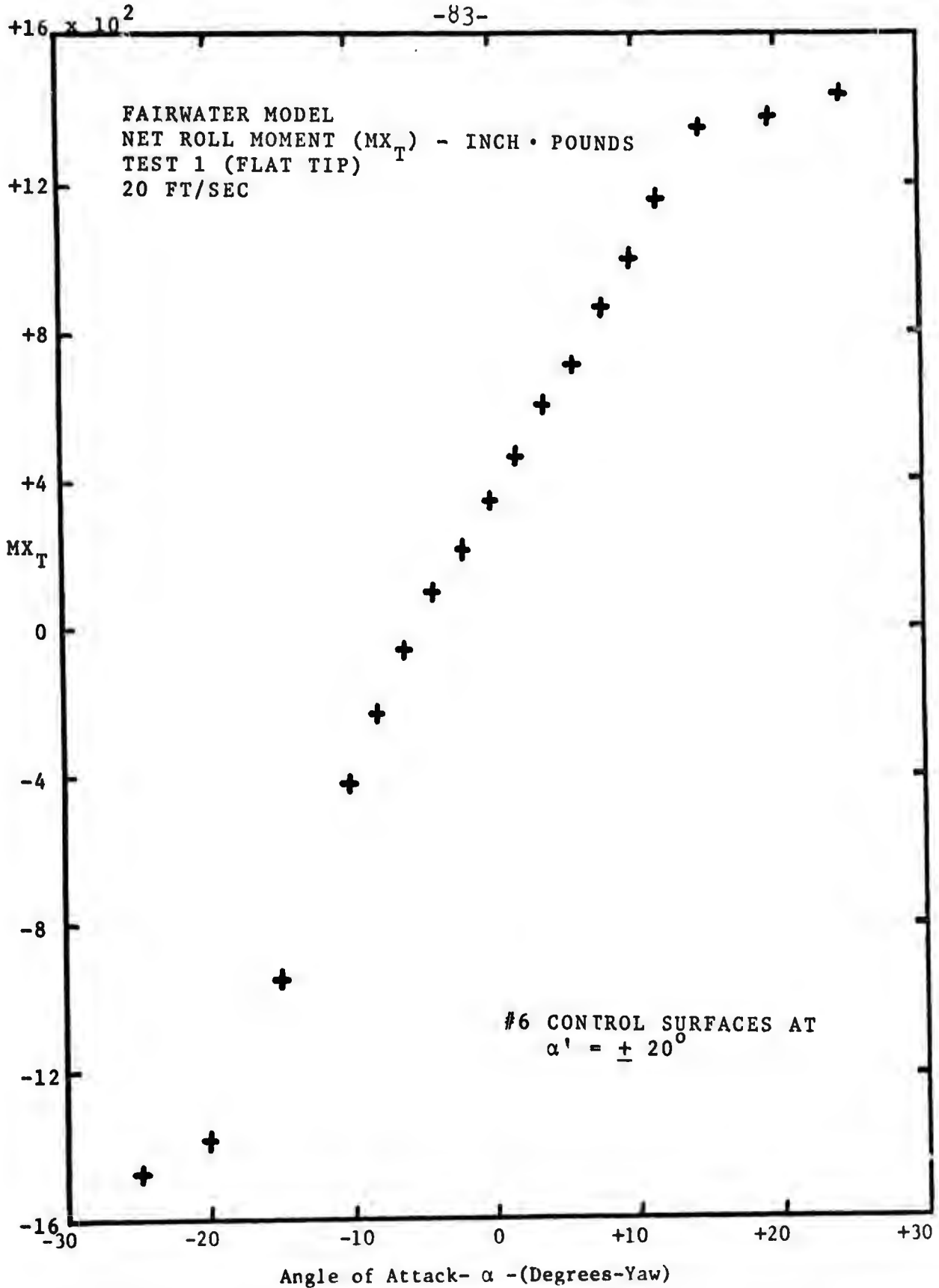


Figure 49

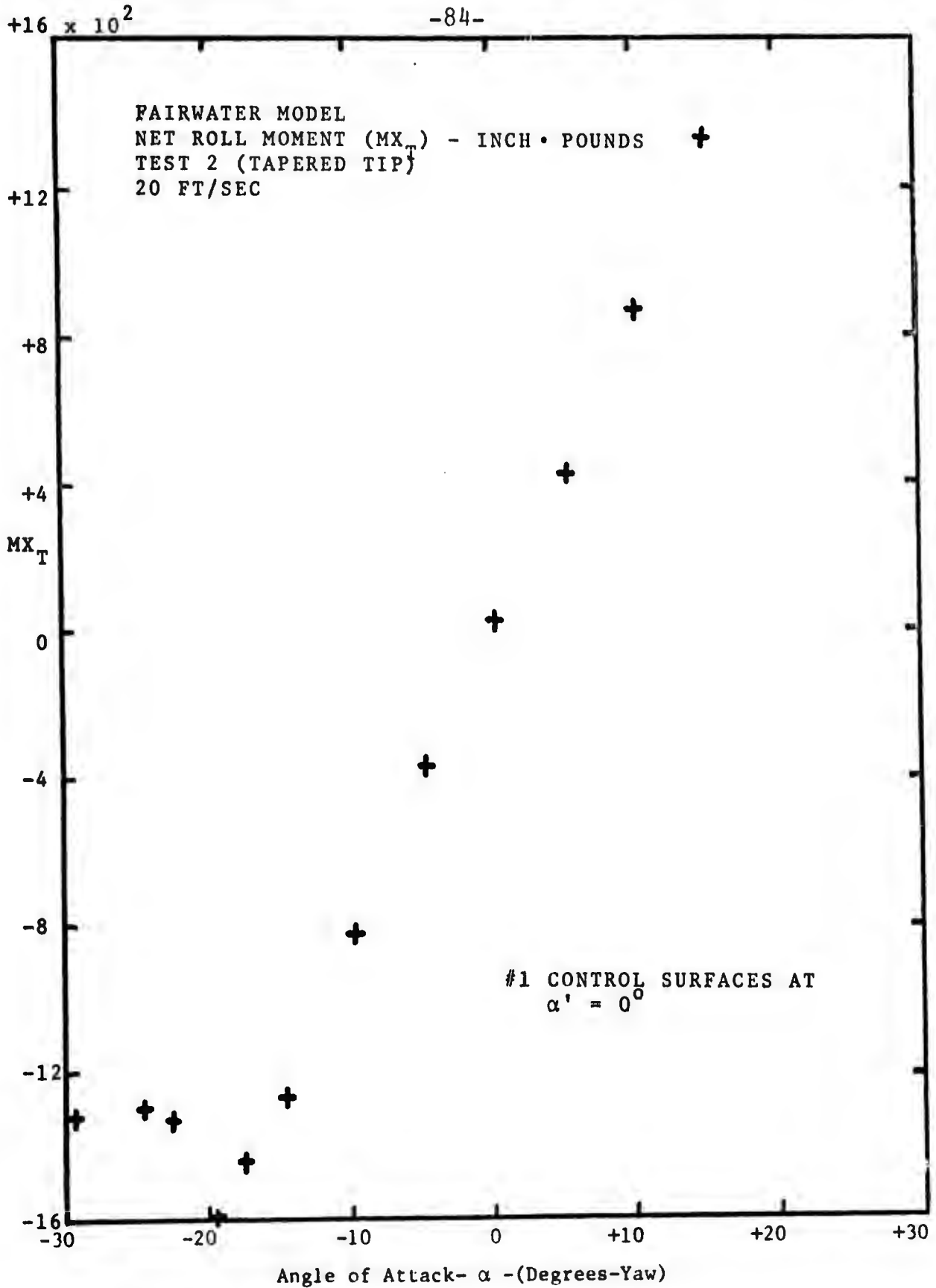


Figure 50

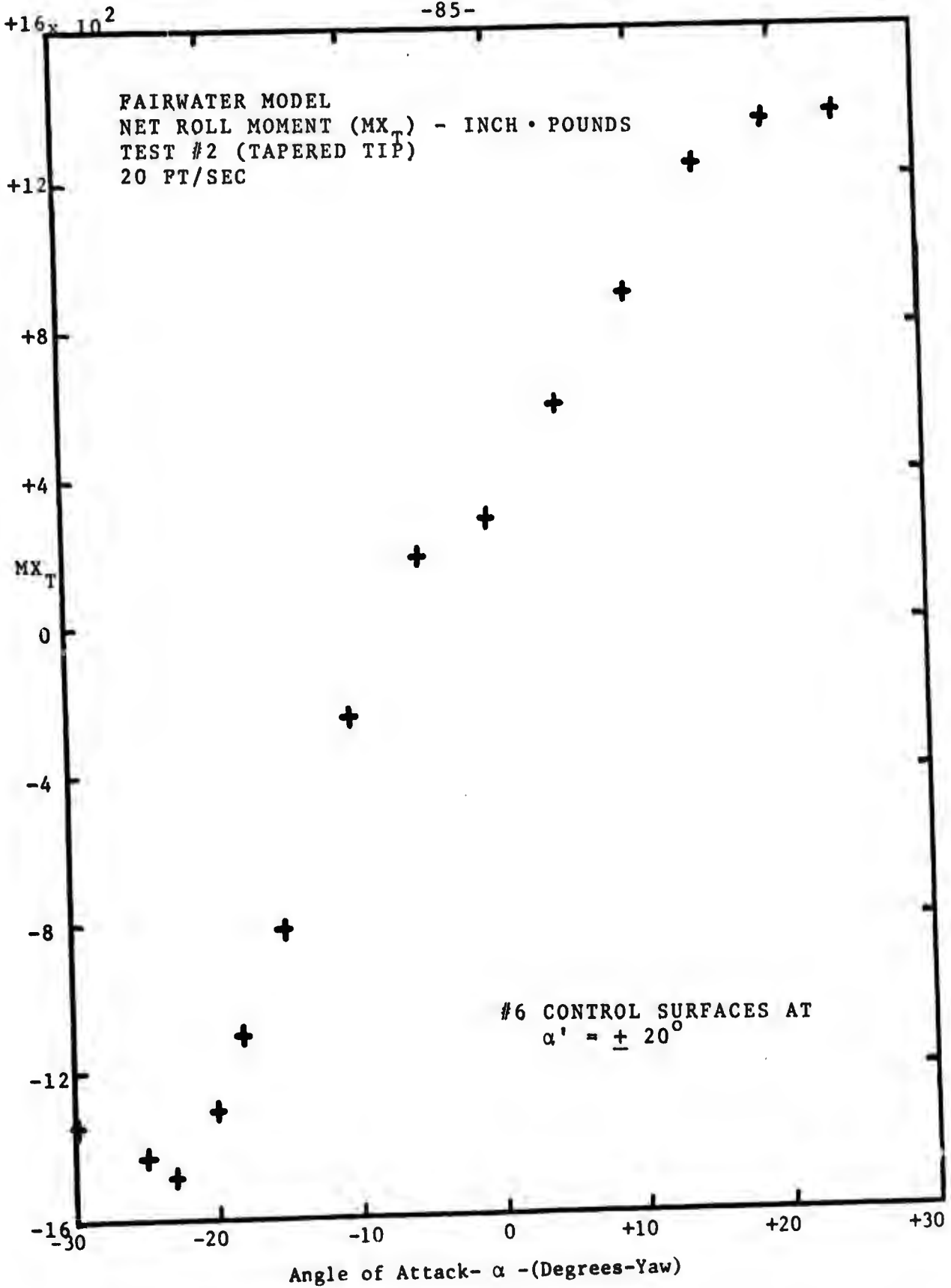


Figure 51

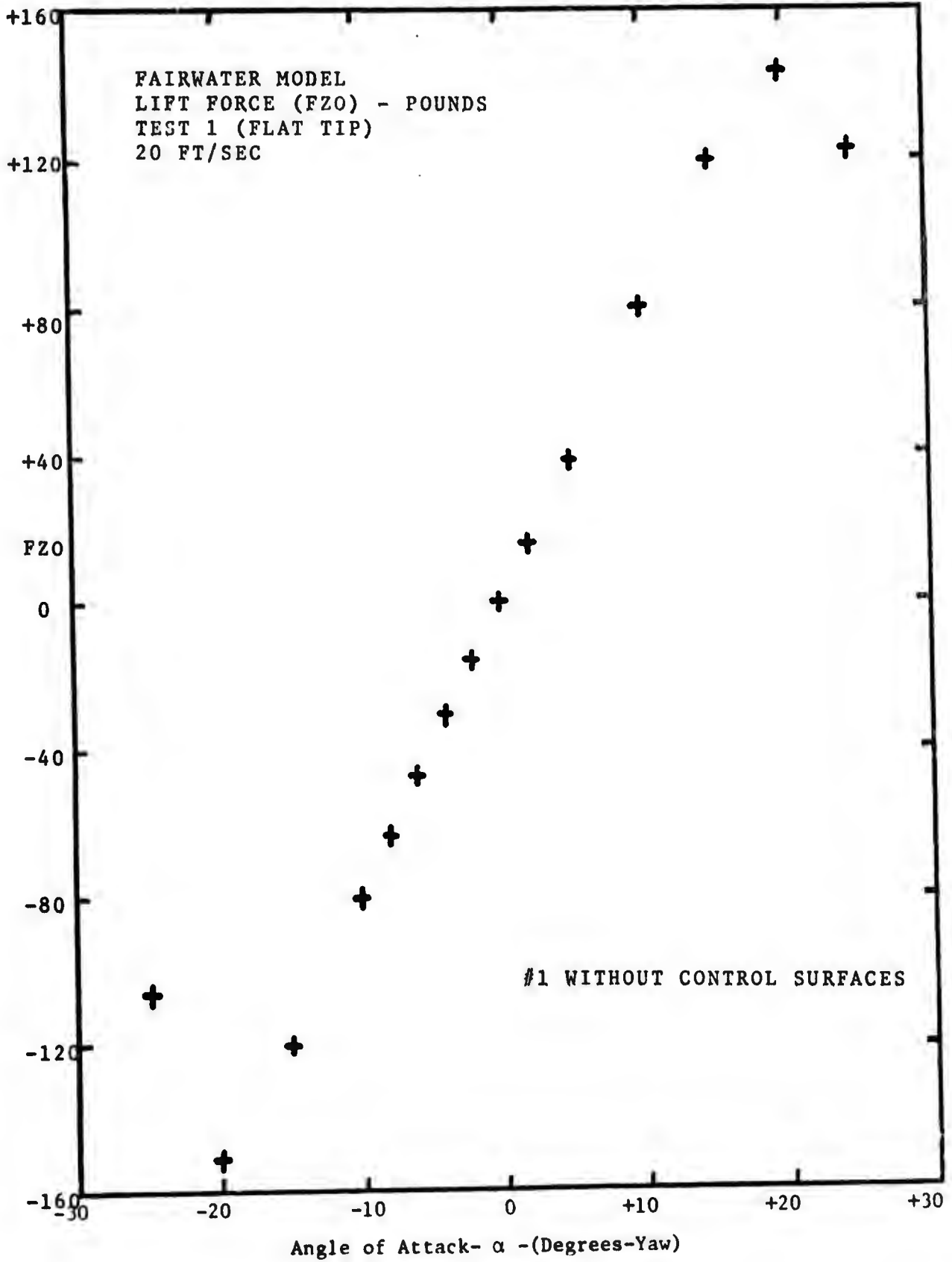


Figure 52

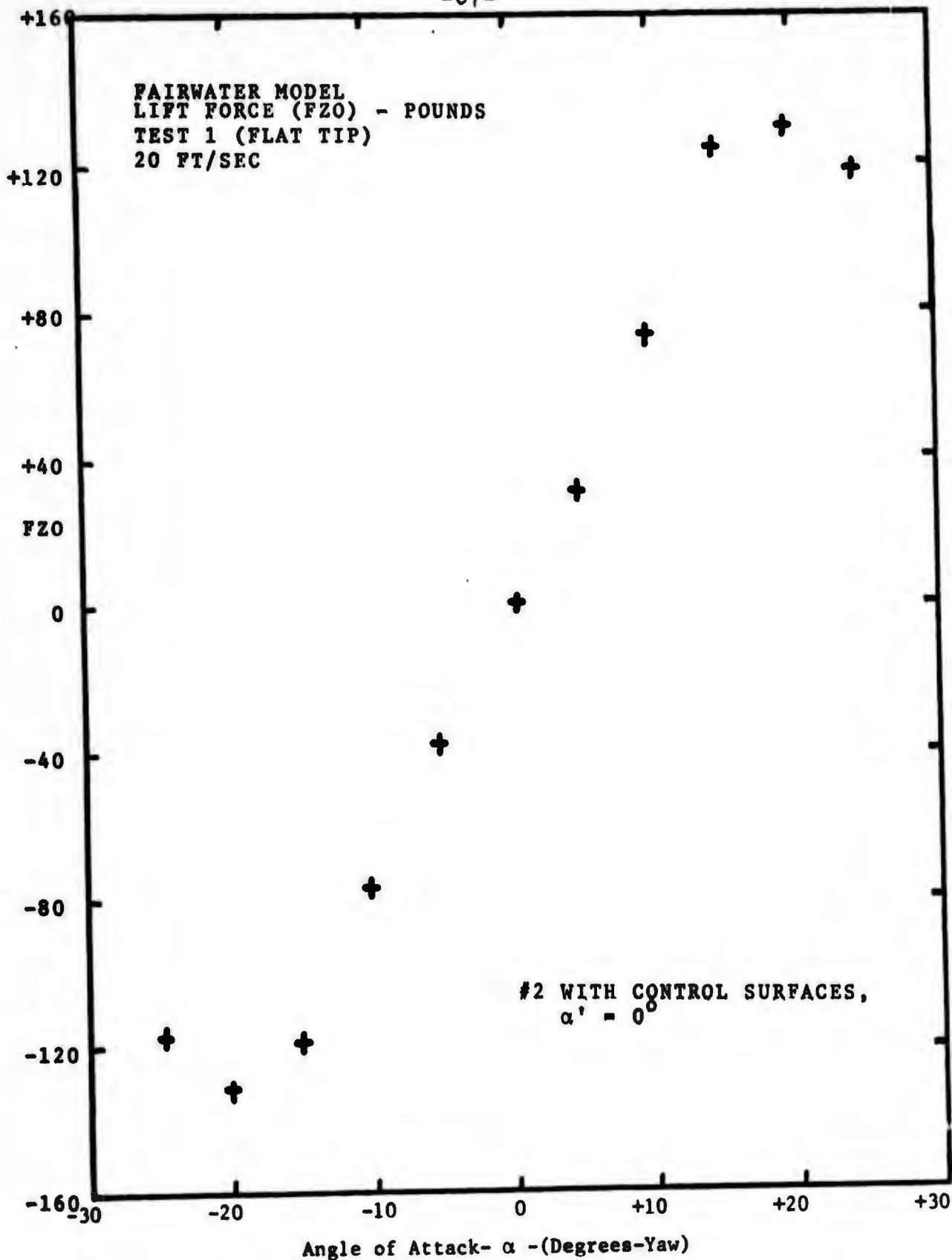


Figure 53

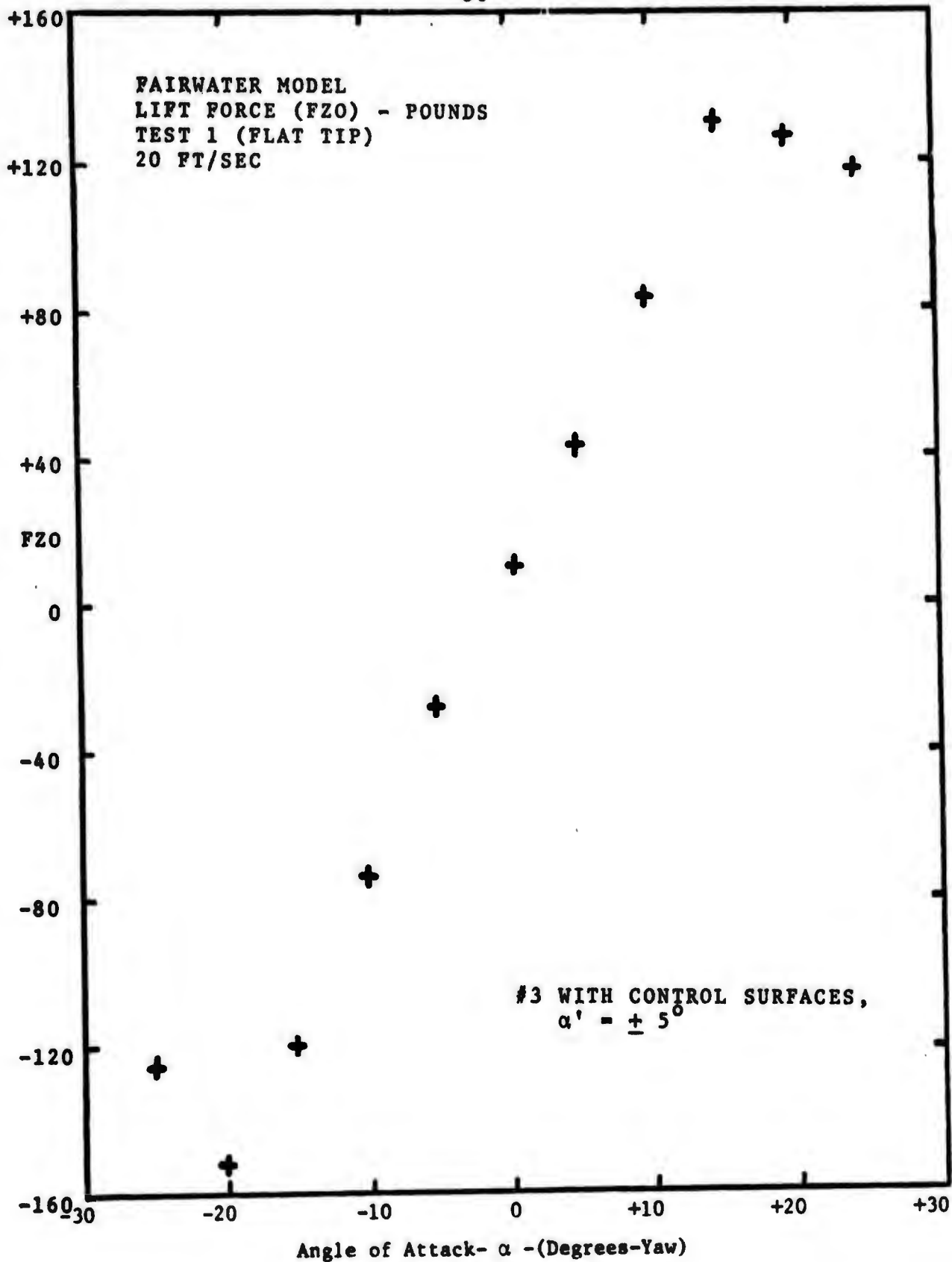


Figure 54

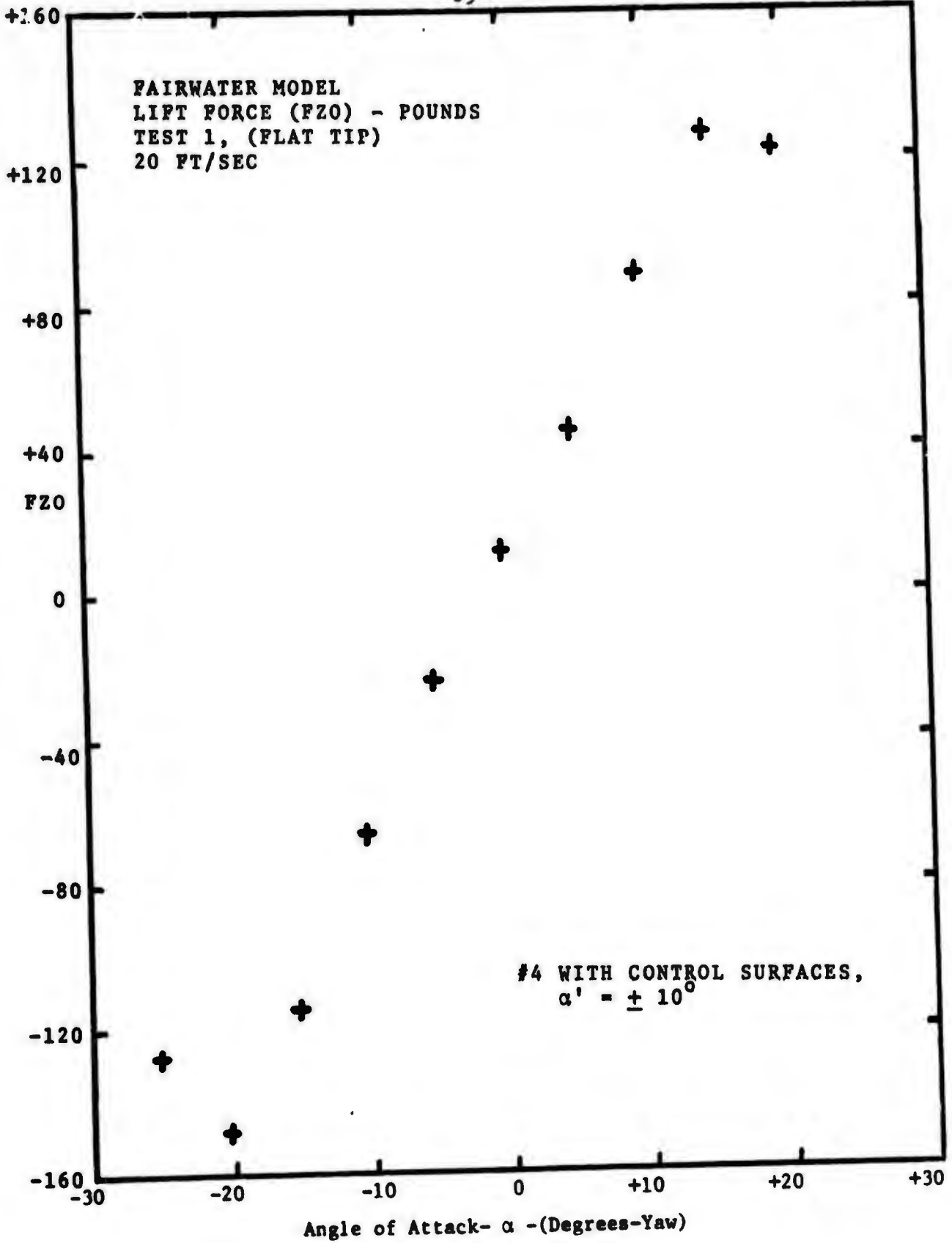


Figure 55

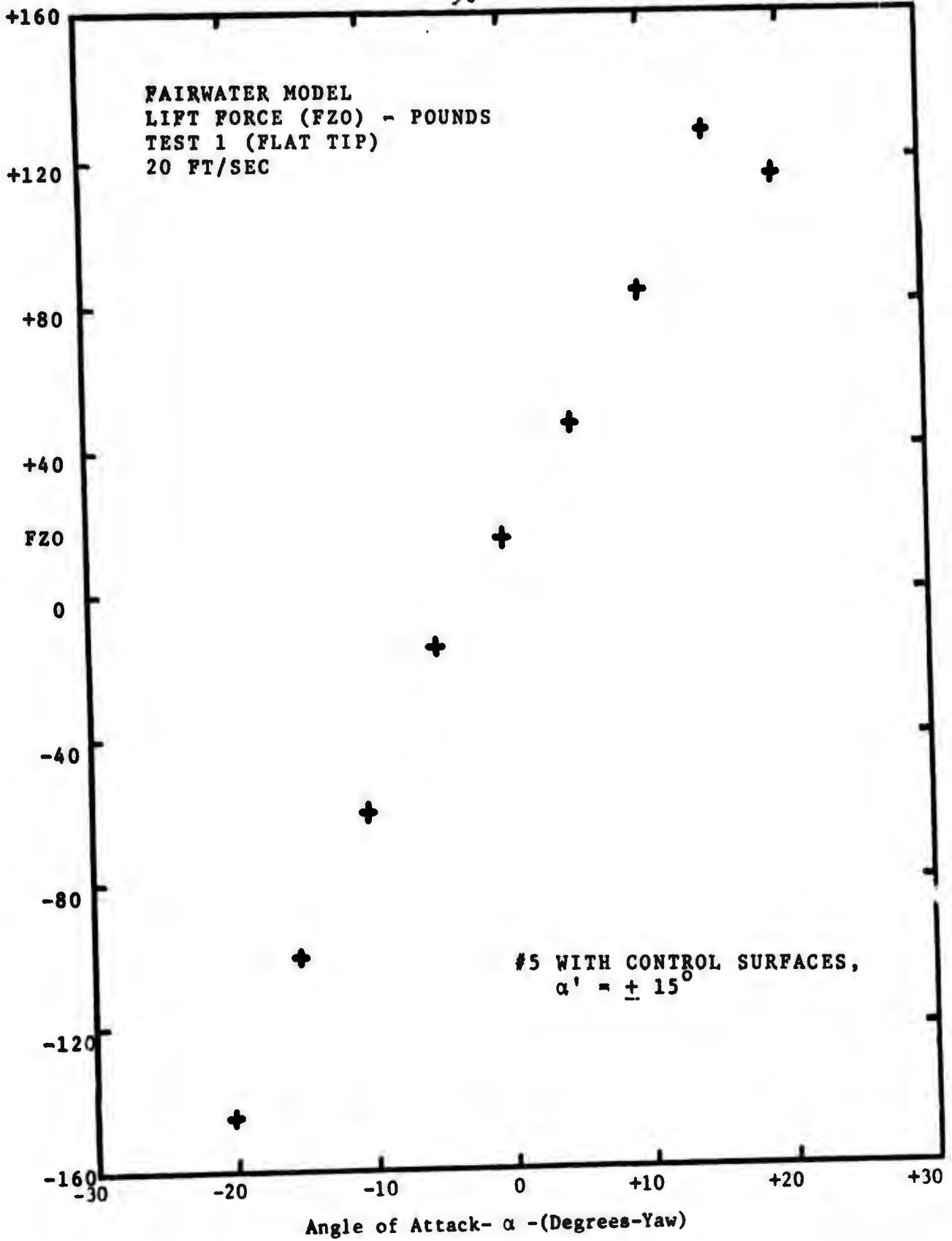


Figure 56

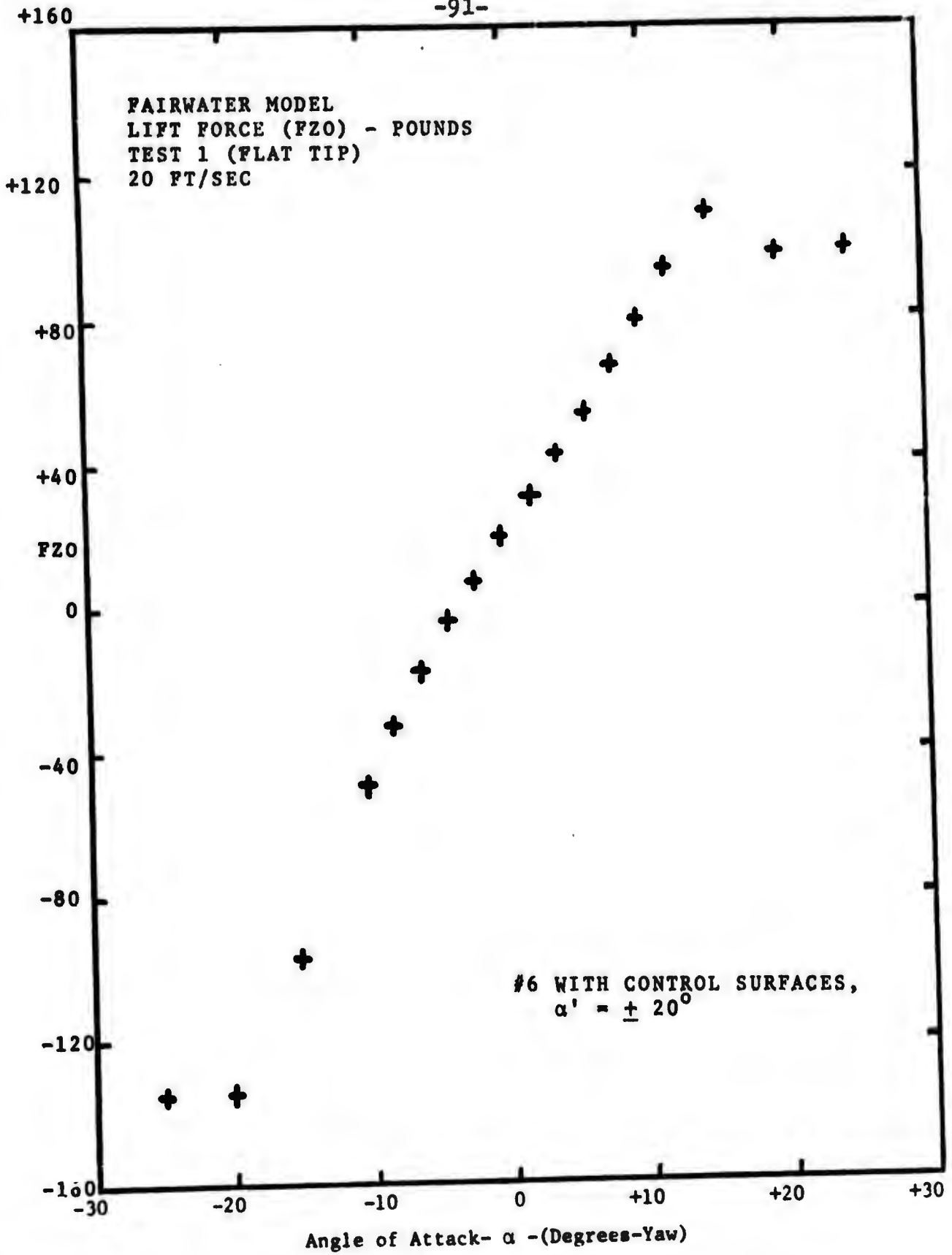


Figure 57

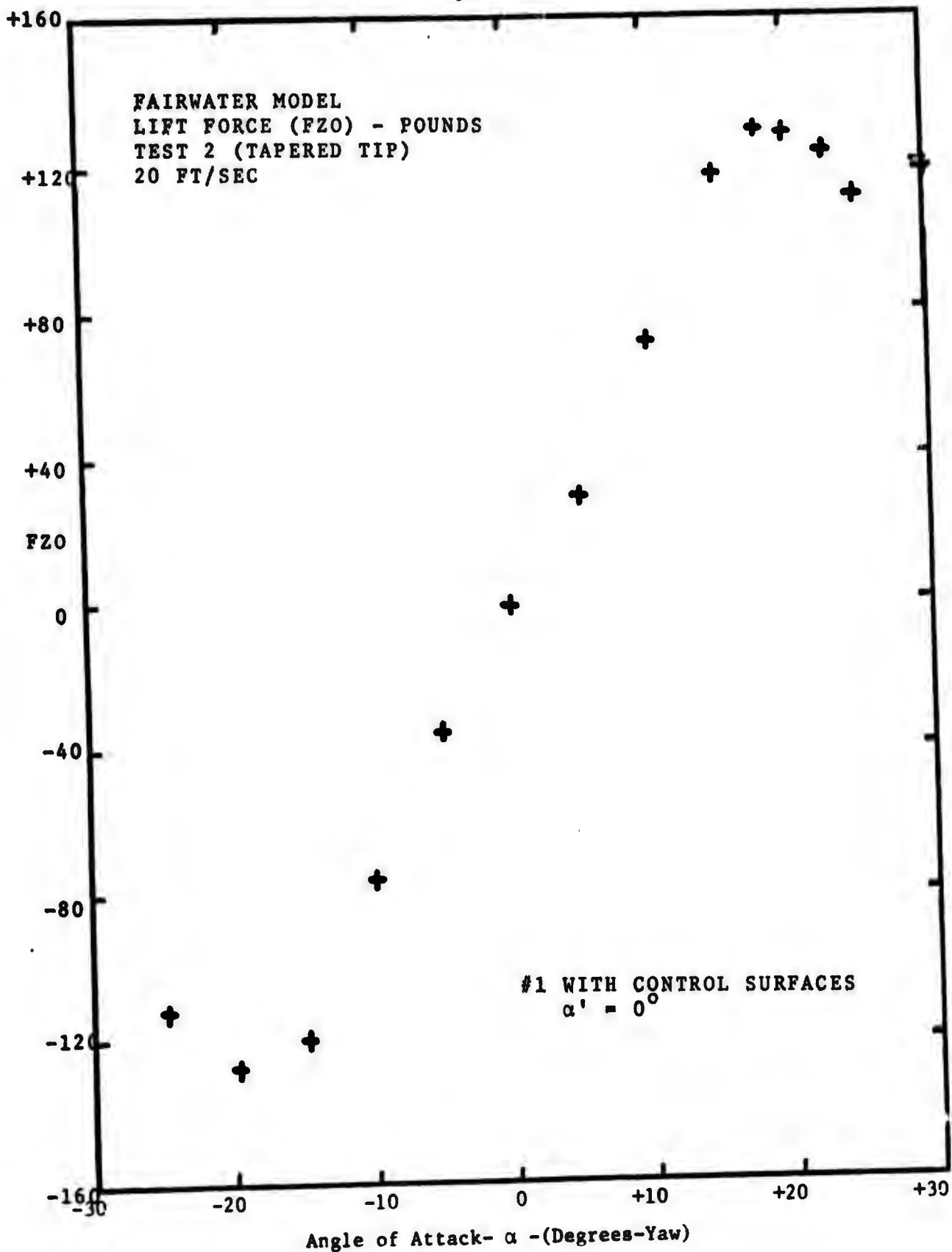


Figure 58

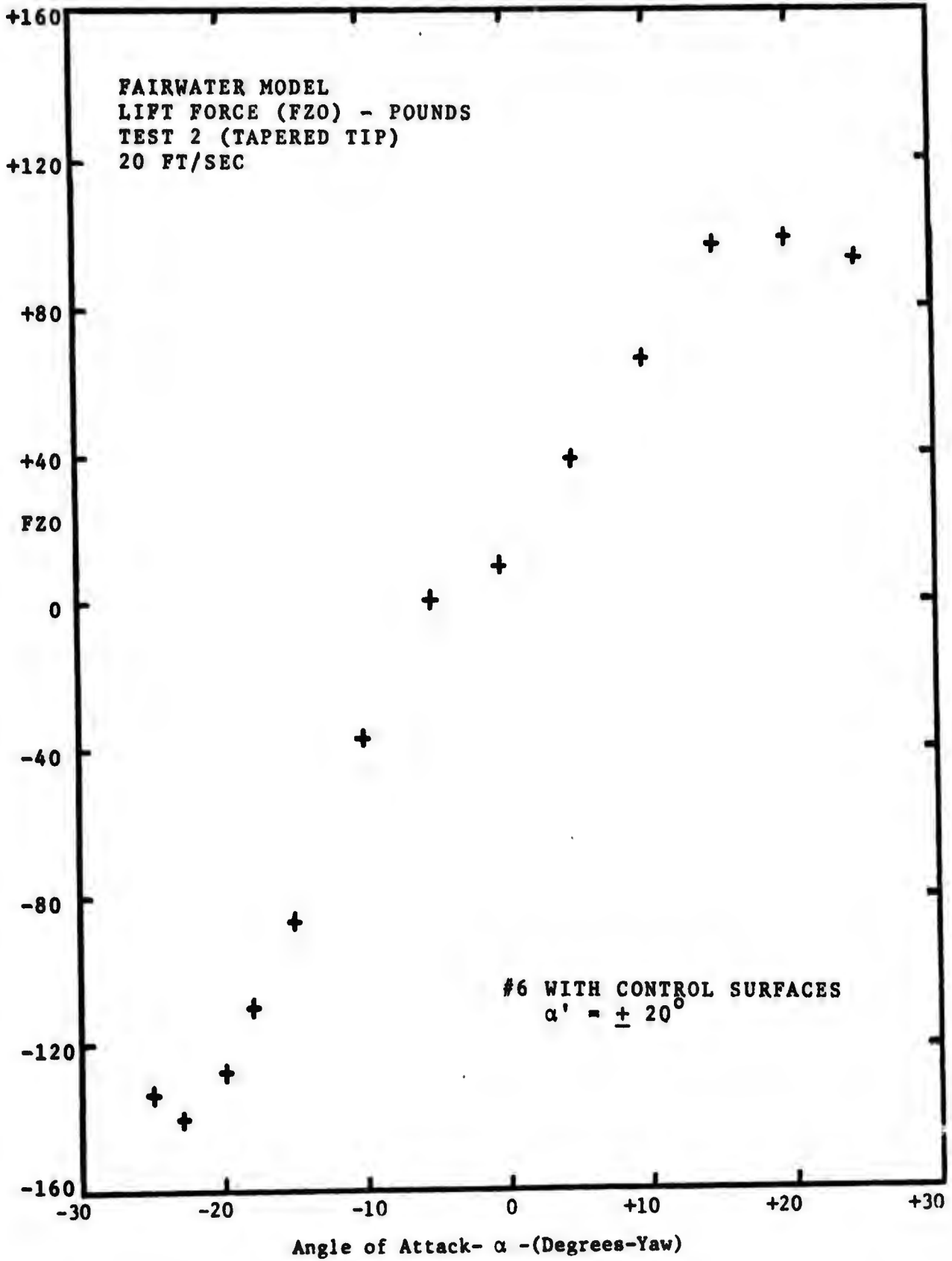


Figure 59

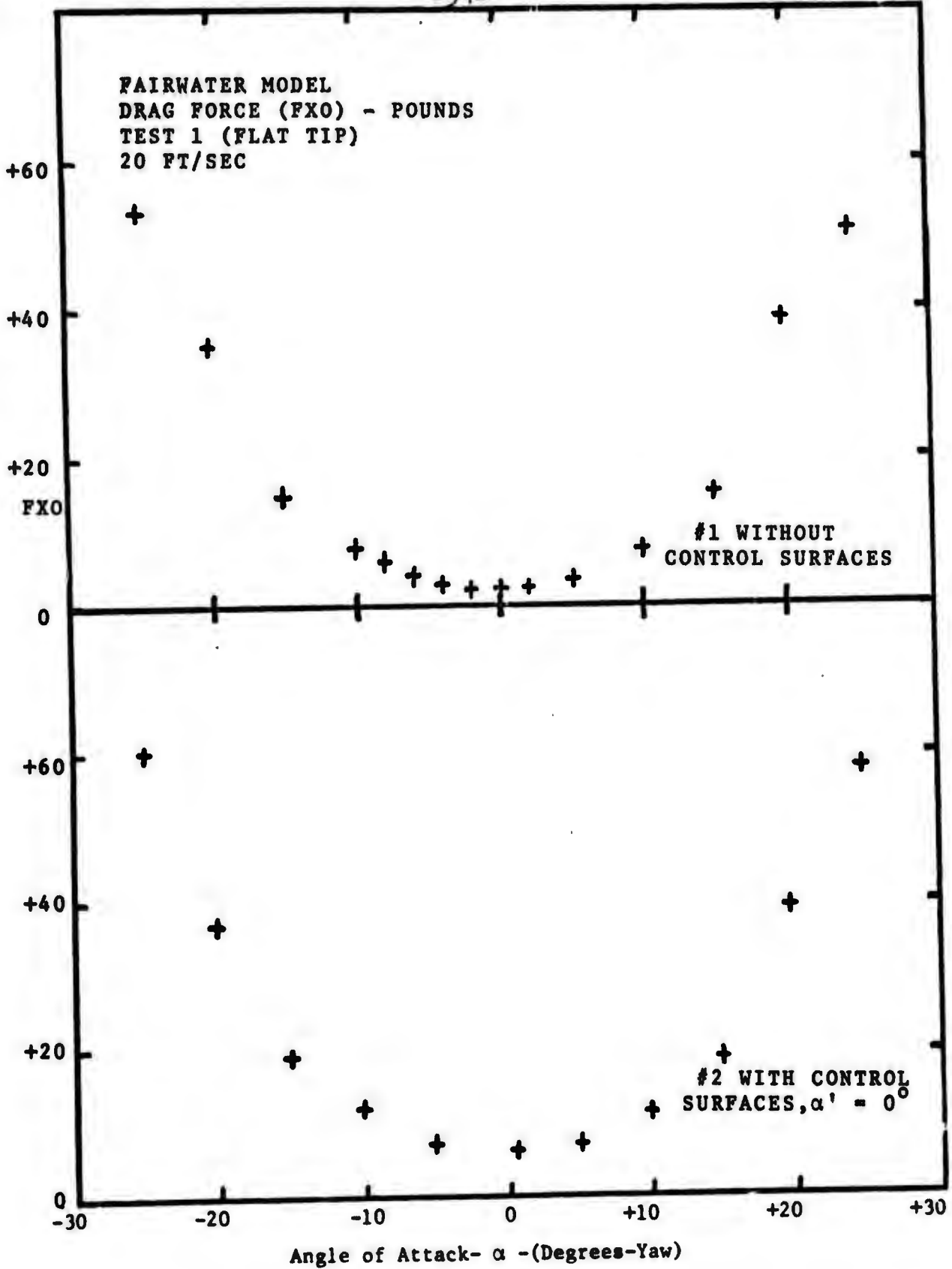


Figure 60

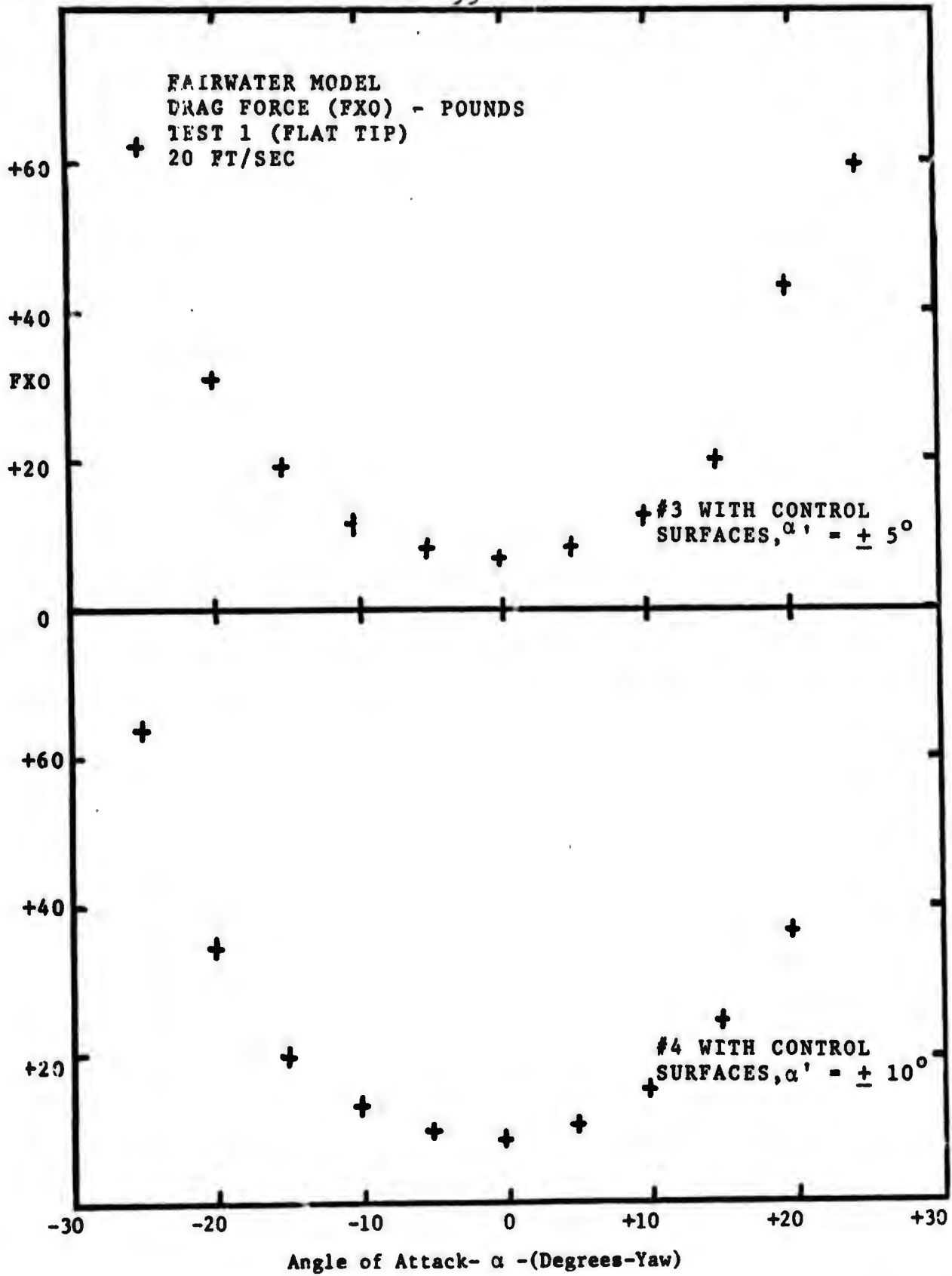


Figure 61

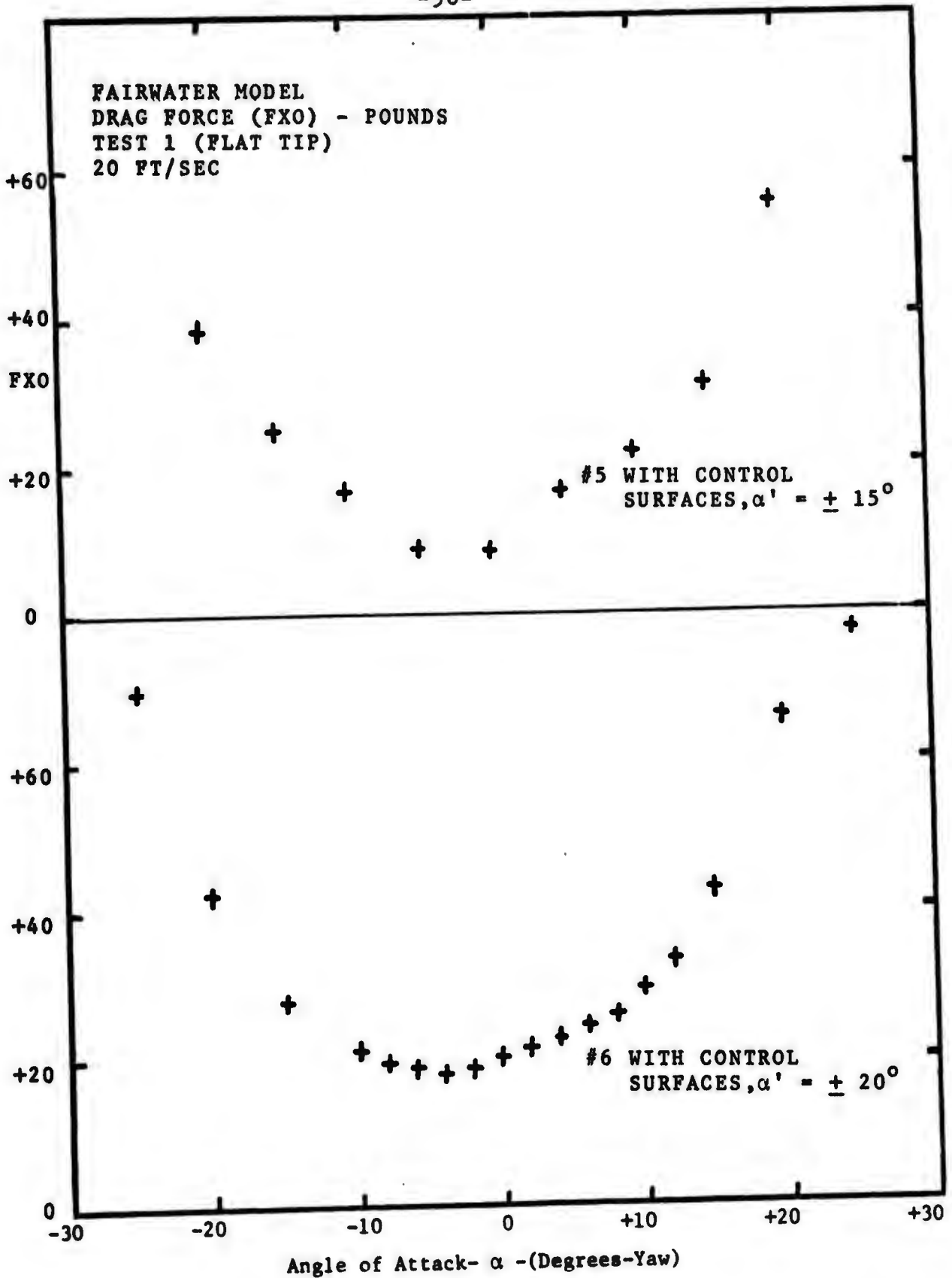


Figure 62

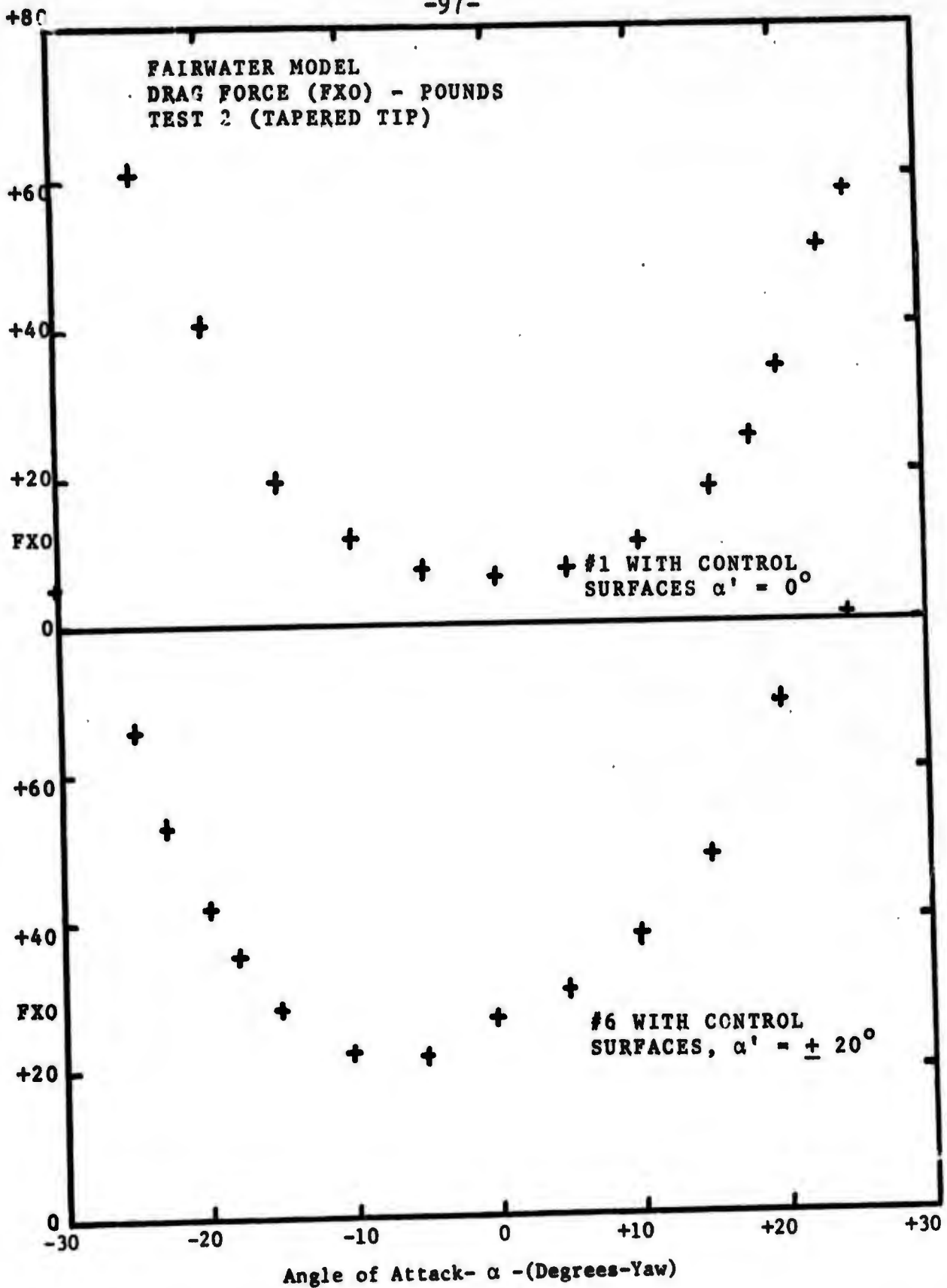
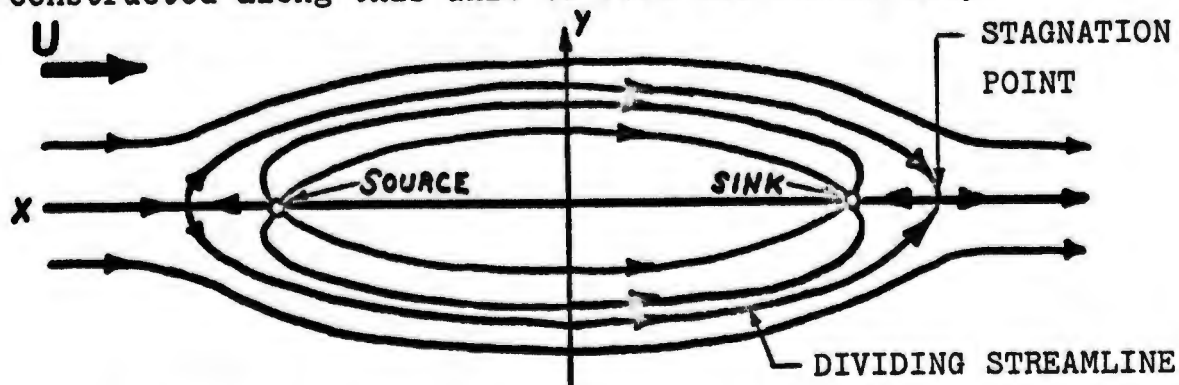


Figure 63

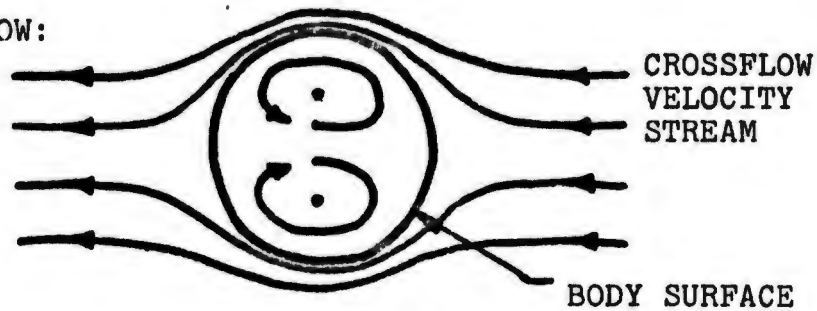
VI. THEORETICAL BASIS FOR NUMERICAL MODEL

In developing a numerical method for predicting the flow characteristics, and eventually the forces and moments, on a submerged body of revolution, it was necessary to use potential flow as the basic theory. Therefore, simplifying assumptions of an ideal fluid were initially required of the flow: inviscid, incompressible, homogeneous, and irrotational. The submerged body, in this case a slender body of revolution, is approximately represented by the combination of two different appropriate distributions. To model the flow along the longitudinal body axis, a distribution of sources and sinks was constructed along this axis to form the closed body.



In order to satisfy the crossflow along the body, resulting from an angle of incidence between the body axis and free stream, a distribution of doublets was similarly placed along the body axis.

IDEAL FLUID FLOW:



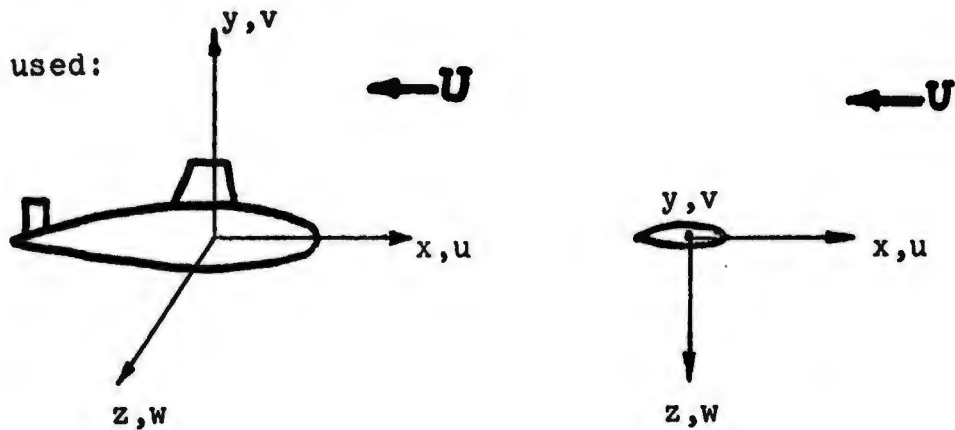
The basis of the approach used for lifting surface calculations was the linearized two-dimensional hydrofoil theory for a thin foil and the law of Biot-Savart, which is a governing relationship for an induced velocity in space by a three dimensional vortex.

Using the linearized two-dimensional hydrofoil theory for a thin foil with the required assumptions of no separation, and the Kutta condition imposed on the trailing edge, the lift force and moment resulting from the hydrodynamic pressure can be obtained. This result, Kutta-Joukowski theorem, states that for a two-dimensional body, moving with constant velocity in an unbounded inviscid fluid, the hydrodynamic pressure force (lift) is directed normal to the velocity vector and is equal to the product of the fluid density, velocity, and the circulation around the body.

$$L = \rho U \Gamma = \text{lift/unit length of span}$$

Circulation (Γ) is defined as the integrated tangential velocity around the closed contour, in this case a lifting surface, in a fluid. With the standard hydrodynamic reference

orientation used:



Circulation, $\Gamma = \oint u dx$, where

u = horizontal perturbation velocity component

U = free stream velocity component

The hydrodynamic moment about the y-axis is represented by the pressure integral:

$$M = \rho U \int u dx$$

The coordinate system used during the experimental portion of this project required the above system to be rotated 180 degrees in the horizontal plane, to be compatible with the water tunnel construction and dynamometer program.

In order to satisfy the presence and the effects of lifting surface appendages (fairwater and rudders) and the discontinuity in pressure, pressure loading across these surfaces was modeled by a distribution of vorticity. The axis for each distribution of vorticity is on the center plane of the

representative lifting surface and normal to the incoming flow, and this vorticity is referred to as a bound vorticity distribution. To fulfill continuity of vorticity, an additional distribution of vorticity, whose axis is on the representative lifting surface and parallel to the incoming flow, was required. This latter distribution is referred to as trailing vortices, or more commonly, trailers. Trailing vortices extend to infinity in the wake formed behind each respective lifting surface, while the effects of the bound vortices are only experienced on the lifting surface itself.

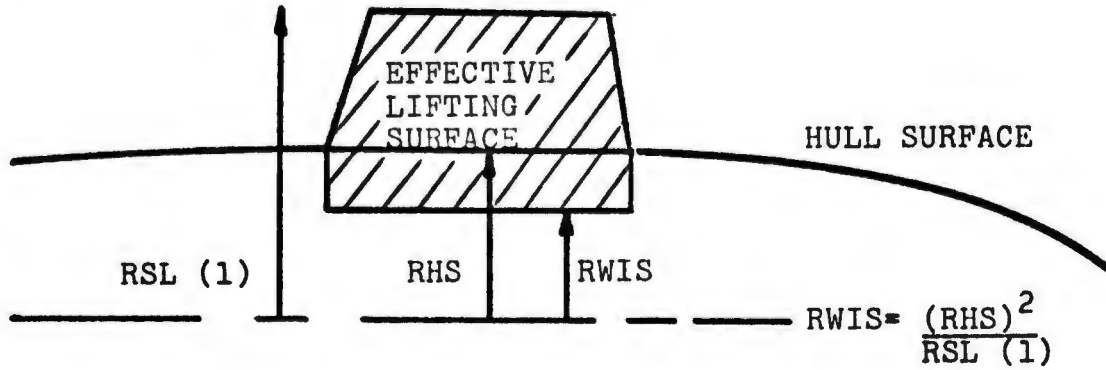
The thickness of the lifting surfaces described above could be represented by a distribution of sources and sinks of appropriate strength, but were defined as being of zero thickness.

Effective Span of Lifting Surfaces

Since the lifting surfaces are each connected to the hull, essentially a curved ground board, an effective span length must be calculated for a valid modeling. This was accomplished using a relationship from Milne-Thomson (3), in which a distance from the body axis to the new effective base is generated. This distance is equal to the ratio of the actual hull radius, at the mid-point of the actual base of lifting surface, squared, divided by the maximum radial distance from

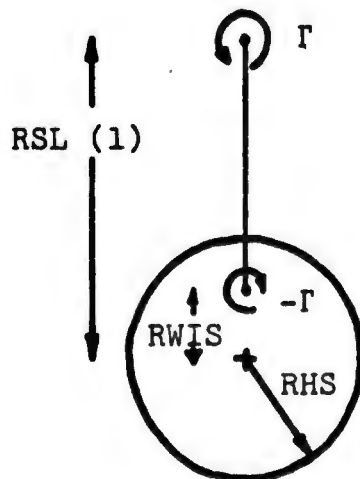
the body axis to the lifting surface tip.

FAIRWATER EXAMPLE:



Effective Span of Fairwater = SPNS = $RSL(1) - RWIS$

The image system is based on the theory that when a pair of two dimensional vortices of equal and opposite strength are located on the same radial line, there is no normal induced velocity on a circle of radius RHS, if RSL is equal to the radius of the outer vortex, and RWIS the radius of the image vortex.



By using this image system for the fairwater and rudder(s), the solid boundary condition of the hull is satisfied.

The technique used, distribution of vorticity, is similar to a method developed by Falkner for lift distribution of wings (4, 5, 6, 7, 8). A continuous distribution of bound and trailing vortices is replaced by a discrete element approximation, in which a lattice of horse-shoe shaped discrete vortex lines are used. The lifting surface is divided into a series of finite rectangular elements in which the lift is considered to be constant within each element, and each element is a horse-shoe of the total lattice, see Figure 64. As the number of elements chordwise and spanwise is increased, the approximation to the actual surface is refined. The trailing vortex sheet formed is assumed, as a first approximation, to remain a flat sheet to infinity downstream while the sheet angle to the vehicle longitudinal axis is adjustable. A selected minimum number of control points are then systematically located chordwise and spanwise midway between vortices on each lifting surface. At the control points, the induced velocity by each horse-shoe lattice element is determined by integration using the Biot-Savart law:

$$(u,v,w) \quad \vec{w} = \frac{\Gamma}{4\pi} \int \frac{d\vec{l} \times \vec{S}}{S^3}$$

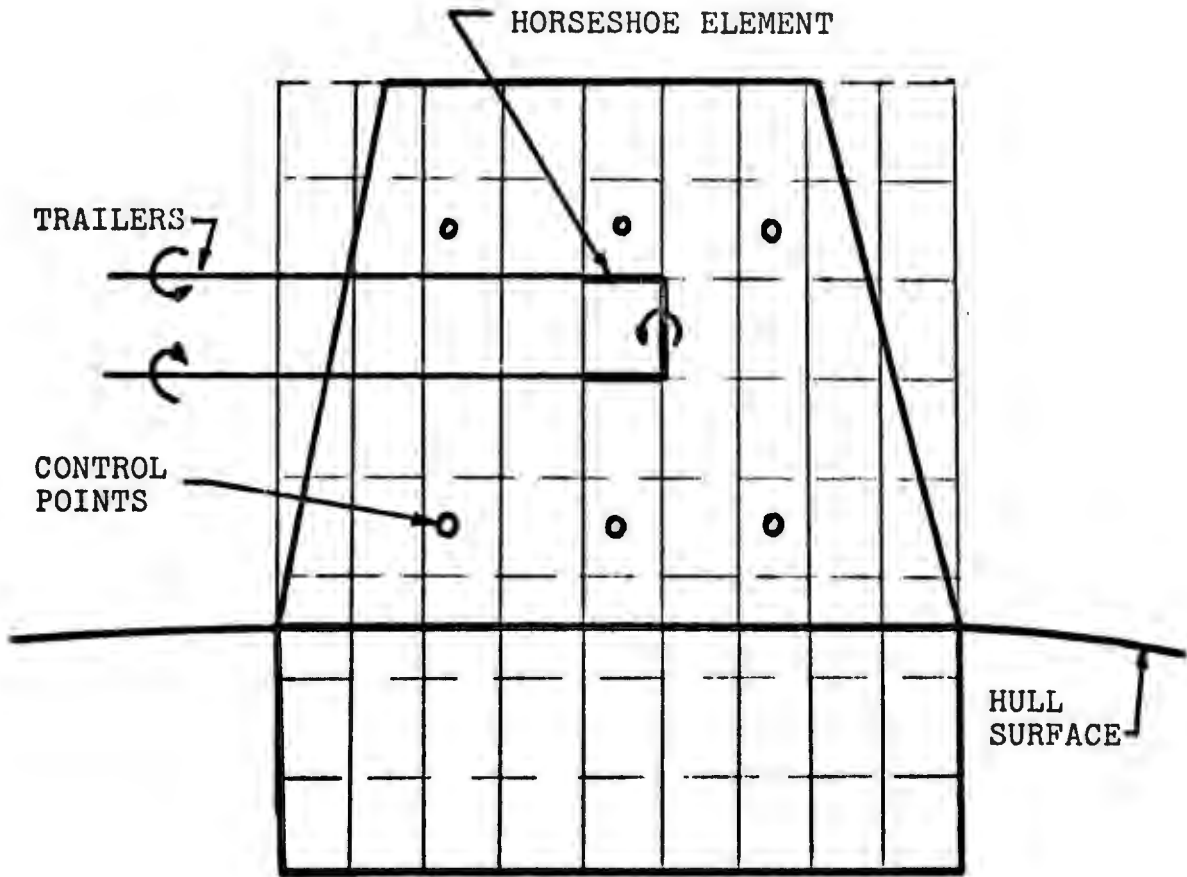


Figure 64 Vortex Lattice

where Γ = vortex strength

\bar{S} = vector distance from vortex element to control point

$d\bar{l}$ = vector element of distance along the vortex

\bar{w} = vector-induced velocity of an element.

The bound vortex strength of each element is first numerically solved using 2 modes of distribution spanwise and 2 modes chordwise; one of a flat plate and the other for camber over the various chord lengths along the length of the effective span.

To insure zero slope, but not necessarily zero value of the circulation at the actual hull surface, an approximated series form for the bound circulation was taken from Kerwin and Leopold (9). This series:

$$\text{nondimensional circulation} = G(q) = (1 - q^2)^{\frac{1}{2}} \sum_{j=1}^J a_{j,q} (2j - 2)$$

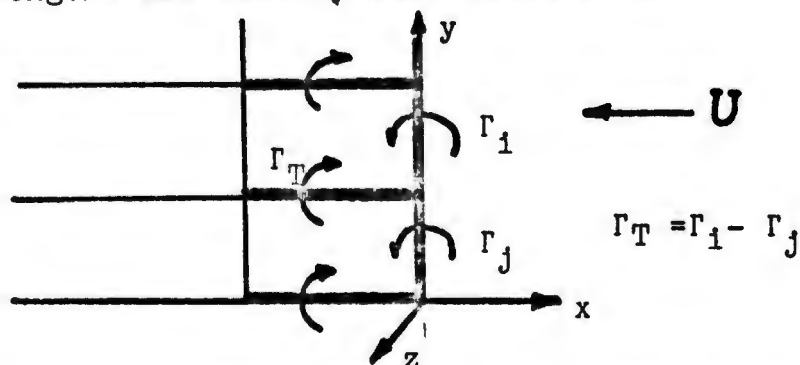
$$\text{where } q = \frac{r - r_h}{1 - r_h}$$

r = radius of image vortex (RWIS, RWIR)

r_h = radius of hull (RHS, RHR)

a_j = unknown coefficient to
be determined from
modal distributions

Since the strength of a trailing vortex of an element is equal to the difference in bound circulation between adjacent bound vortices, as a vortex may not end in a fluid, the vortex strengths are readily calculated also.



Using the law of Biot-Savart, and the bound vortex strengths just calculated, the induced velocities of each element on each of the control points of the lifting surface are calculated. Then, the same method is repeated for the induced velocities resulting from the trailing vortices.

The number of control points used should be a minimum, yet retain accuracy, to reduce the size of the resulting matrix of induced velocities that is eventually developed for each lifting surface, to reduce computation time. Although accuracy should be improved by a greater number of control points and smaller horse-shoe element size, a point of diminishing return will eventually occur, and care must be

taken not to allow the matrix, a set of simultaneous linear equations, to become too nearly singular to be solved numerically.

Since the bound and trailing vortices do not yield the total induced velocity on the lifting surface, the trailing wake of the surface in question, the hull, and other adjacent lifting surface wakes must also be considered and their effects combined.

The induced velocity on the control points from the wake of the surfaces is determined by the Biot-Savart law with respect to the control points. The induced velocity on the lifting surfaces from the hull results from the singularities used in developing the hull model and the potential flow around it. The sources and sinks do not cause any induced velocities, but the crossflow compensating doublet distribution along the body axis does contribute induced velocity normal to the lifting surfaces. These velocities are developed from the doublets distributed at the corresponding locations of the lifting surfaces with respect to the body axis. Their effects on the previously-designated control points are deduced by an equation which is a function of the crossflow velocity, and the ratio of body radius and perpendicular control point distance from body axis at corresponding points on the body axis. This relationship:

$$WDS = W(x) \frac{R^2}{YCP^2}$$

where:

WDS = induced velocity of doublet

R = body radius at XCP (long. location of control point)

YCP = vertical location of control point

W(x) = crossflow velocity

is the component normal to the lifting surface of the velocity due to a doublet in a uniform crossflow, and is found by differentiating the potential of the doublet (10).

The wake of each lifting surface connected to the body also produces induced velocities on all other lifting surfaces and the hull. The magnitude and relative importance of each induced velocity is based on the relative distance and location of the wakes with respect to the other surfaces, and is determined by the Biot-Savart law. In the numerical procedure developed for this project, (see Appendix D), the effect of the rudder wake on the fairwater is assumed to be negligible, because of its relative location, and was not considered.

The combined induced velocities at each control point of a particular lifting surface result in a set of simultaneous linear equations, a matrix of induced velocities, which relate:

- (1) the modal vortex strength of the lattice elements to the lifting surface shape;
- (2) wake effects;
- (3) applicable effects of other lifting surfaces (wake, surface); and,
- (4) the induced velocity of the hull doublet distribution, based on inflow velocity and yaw angle.

The different mode strengths for each lifting surface are then generated for subsequent development of force and moment calculation.

The hydrodynamic forces and moments on the hull result from induced velocity contributions of each lifting surface wake, the crossflow velocity, source and sink distribution, and doublet distribution.

As before, the induced velocity imposed by each wake on the hull is found by the Biot-Savart law, at each station along the body axis. This externally-induced velocity is combined with the crossflow, or sway, velocity to give a total externally-induced velocity at each station on the centerline.

The induced forces and moments at each station are derived using the above externally-induced velocities as a major governing parameter. Using a relationship from McCreight (11, 12) based on Lagally's theorem, the resulting force on a source distribution of constant strength per unit length along the body axis, when subjected to the above

externally-induced velocity, is equal to:

$$U\rho \sum_{1}^{NSTA} S'(M)W(M)$$

where:

U = free stream velocity

ρ = density of fluid

S'(M) = slope of hull surface at input point M;

M = station number

W(M) = externally induced velocity

NSTA = number of stations

The slope of the hull surface (S'), being a measure of the change in area per unit length, is also the source strength at the station in question. The moment due to the source distribution is found from the numerically-integrated product of the force at a station and distance from axial reference point.

Since the doublet distribution does not contribute a force when an induced velocity is imposed, we are only interested in its moment contributions. Using a method introduced

by von Karman (1930) for crossflow past airships and the assumptions:

- 1) uniform crossflow in the transverse plane; and,
- 2) body and flow radius do not change drastically along the body axis;

the crossflow effect can be represented by the flow past a circular cylinder in which the doublet strength is equal to:

$$\bar{\gamma}(x) = -\frac{1}{2} r^2(x)W(x)$$

where:

$r(x)$ = body radius at cross-section

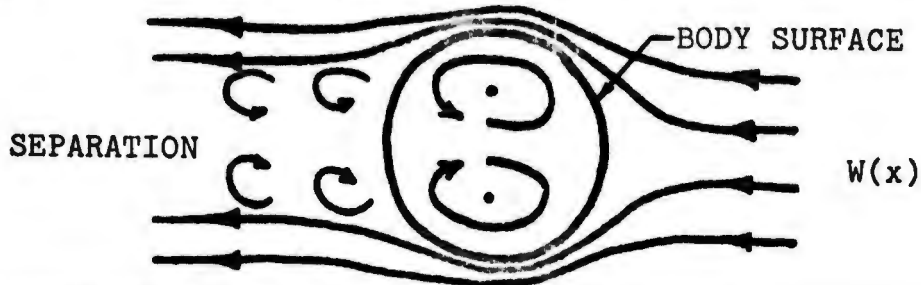
$W(x)$ = externally induced velocity (crossflow)

Since the doublets with a vertical axis are in a horizontal uniform velocity stream U , the moment is equal to $-4\pi\rho\bar{\gamma}(x)$. When this moment is integrated over the length of the body axis the resulting integral is formed:

$$M = 2 \pi\rho U \int_x r^2(x)W(x)dx$$

This is numerically approached by calculating the surface area at each station and knowing the crossflow velocity $W(x)$ at each station.

Since the above two models, of sources and doublets, do not account for viscous effects - separation of flow around the body and crossflow drag -



consideration and provisions must be made to compensate for this failure, as the effects are significant with respect to some of the hydrodynamic forces and resulting moments acting on a submerged body in the steady state case. A first approximation is made by adjusting with a linear distribution factor (FAC) along the portion of the body length aft of the coordinate system reference point. This adjustment is accomplished by a factor which increases the force and moment effect of the sinks, and decreases the moment due to the doublet distribution aft.

The forces on the lifting surfaces are found by numerically integrating the four spanwise modal circulation effects to obtain the circulation along the effective span, and, in accordance with the Kutta-Joukowski theorem, determine the lift (side force) caused by the fairwater and rudder(s) separately.

The moments resulting from these lifting surfaces are

determined from the numerically integrated product of the spanwise modal circulation effects and the appropriate chordwise moment arm, based on the center of pressures for the flat plate and camber modes along the chord.

The total force and moment experienced on the submerged body is obtained from the vector summation of each individual component calculated.

$$\begin{aligned} \text{Total Side Force} &= \text{Rudder(s) Side Force} + \text{Fairwater Side} \\ &\quad \text{Force} + \text{Source Side Force} \\ \text{Total Moment} &= \text{Rudder(s) Moment} + \text{Fairwater Moment} + \\ &\quad \text{Source Moment} + \text{Doublet Moment} \end{aligned}$$

An attempt to model heave force and pitch moment is made by taking the induced circulation around the hull caused by the strength of the trailer-forming wake off the external portion of the actual fairwater span. The Kutta-Joukowski theorem again is used to determine the heave force and pitch moment.

VII. COMPARISON OF EXPERIMENTAL SUBMERSIBLE RESULTS

A. NSRDC Model Test

A model test was conducted at the Naval Ship Research and Development Center (NSRDC) to investigate the forces and moments on a submerged body of revolution for various rudder aspect ratios. The final results of this investigation have not been completed, and only preliminary results of the clean hull configuration were comparable with these experimental results. Lateral force coefficients and yawing moment coefficients were calculated and plotted for all submersible model configurations for comparison when the final NSRDC results are available, see Figures 65 - 71. The clean hull results obtained for this experiment were about 40 per cent higher for lateral coefficients, but showed only minor differences for yawing moment coefficients.

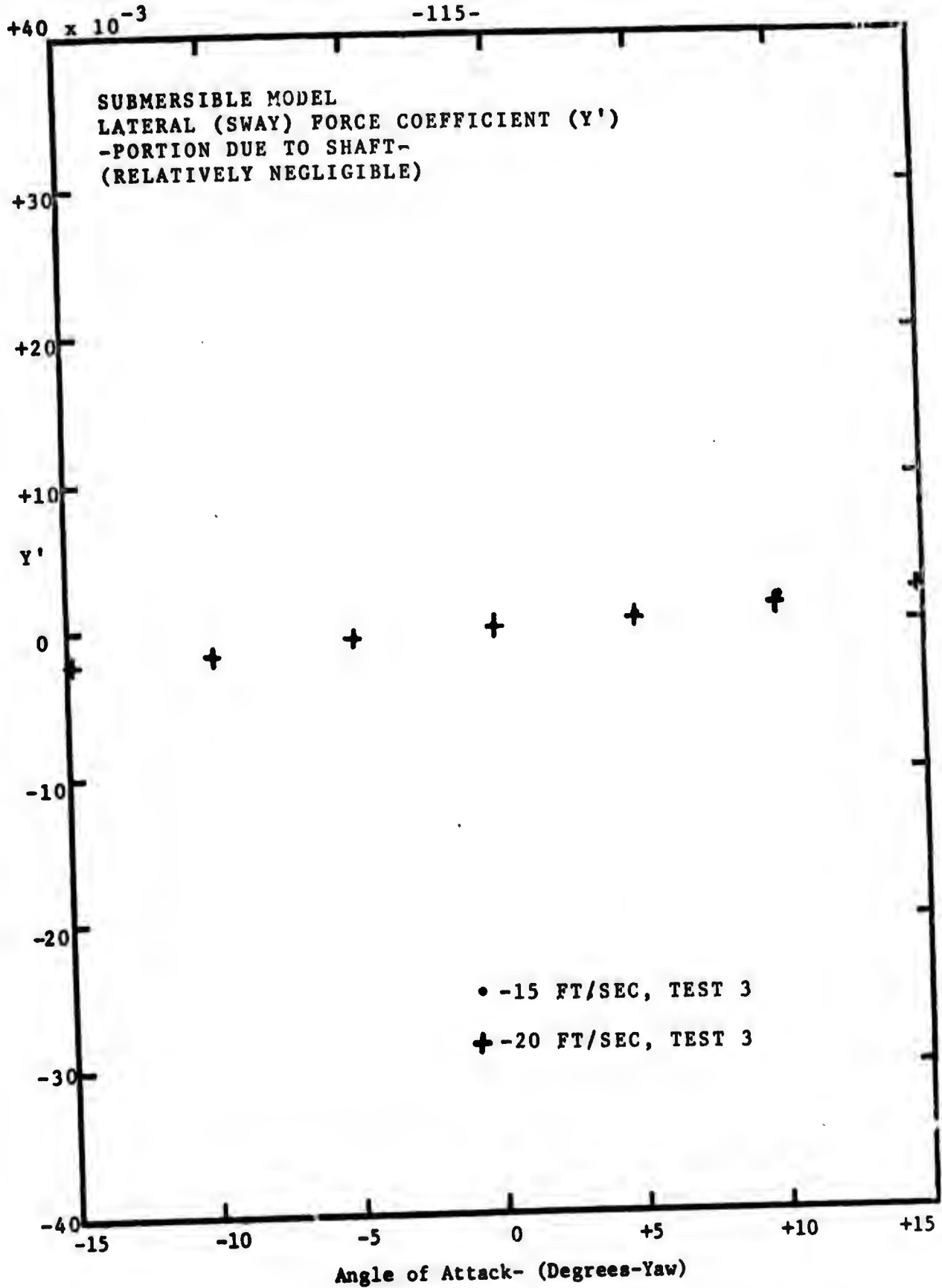


Figure 65

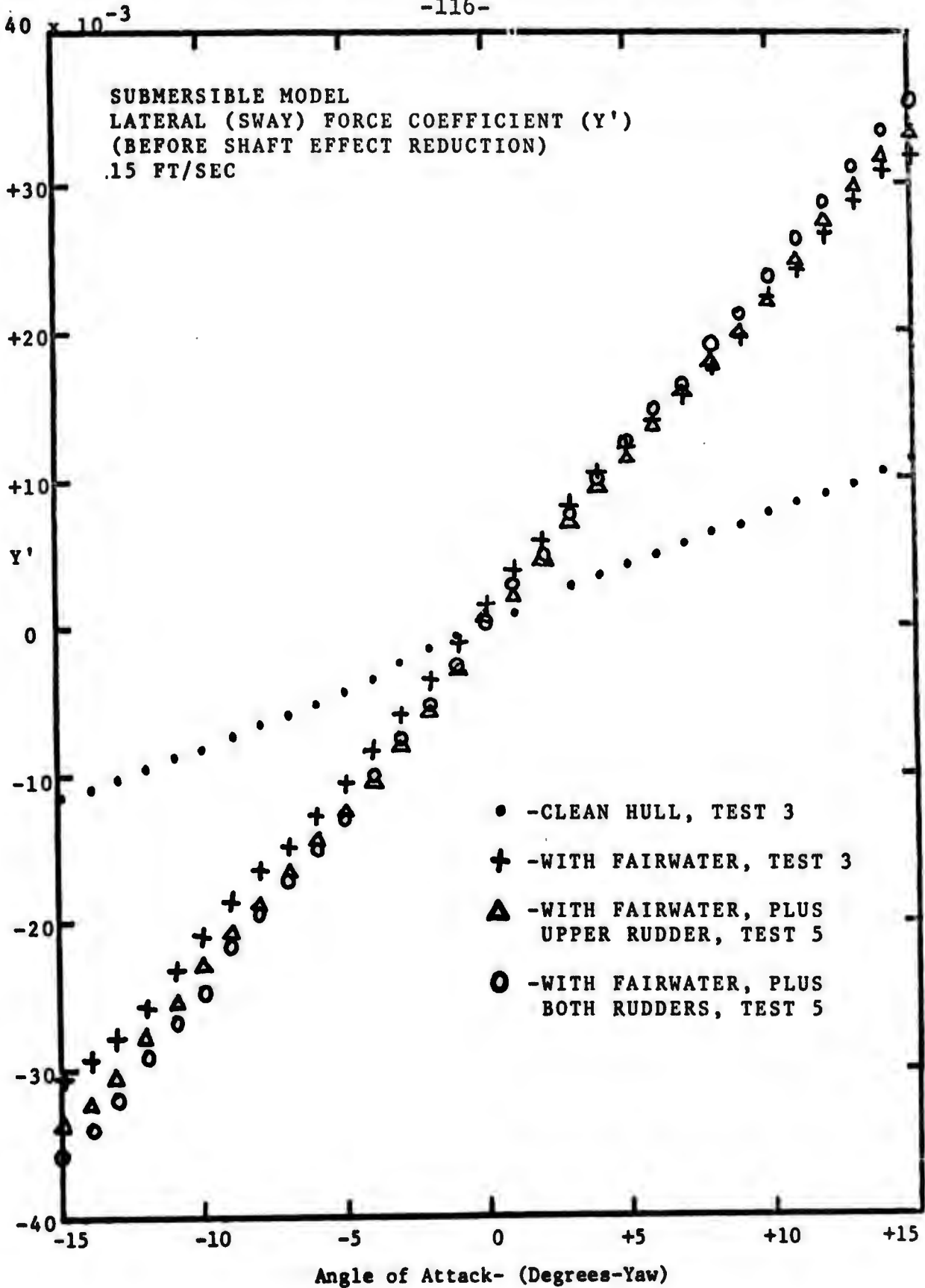


Figure 66

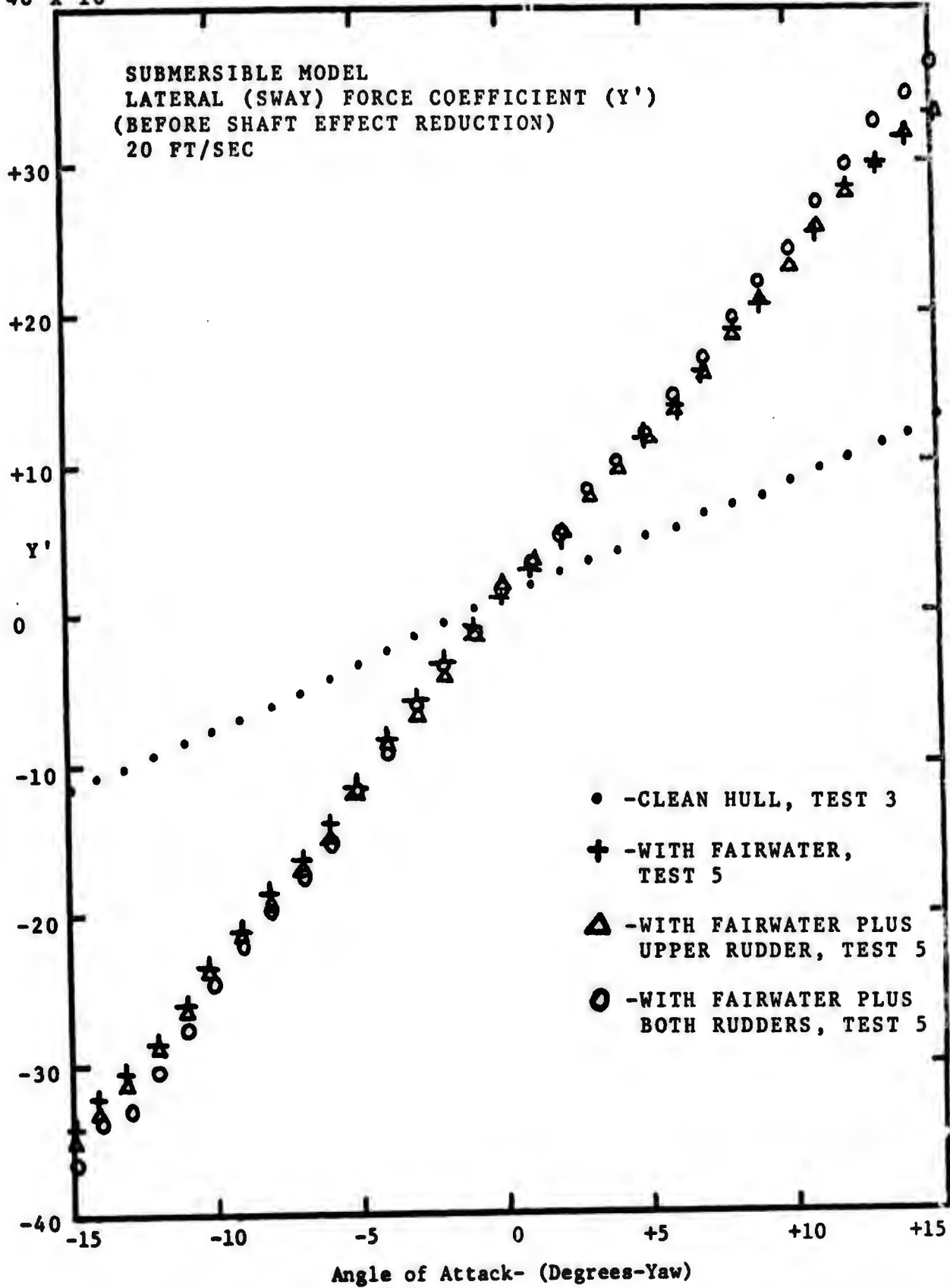


Figure 67

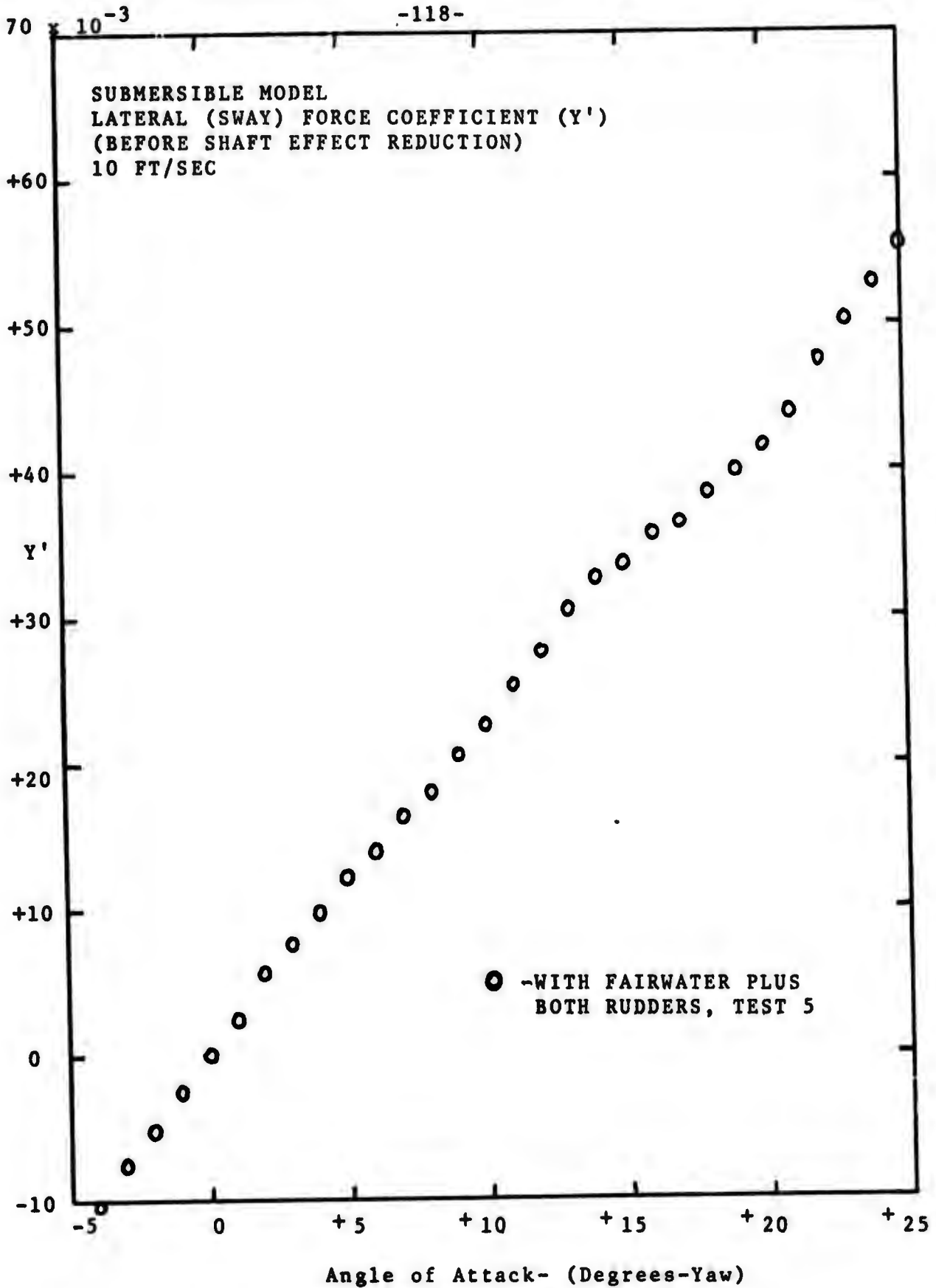
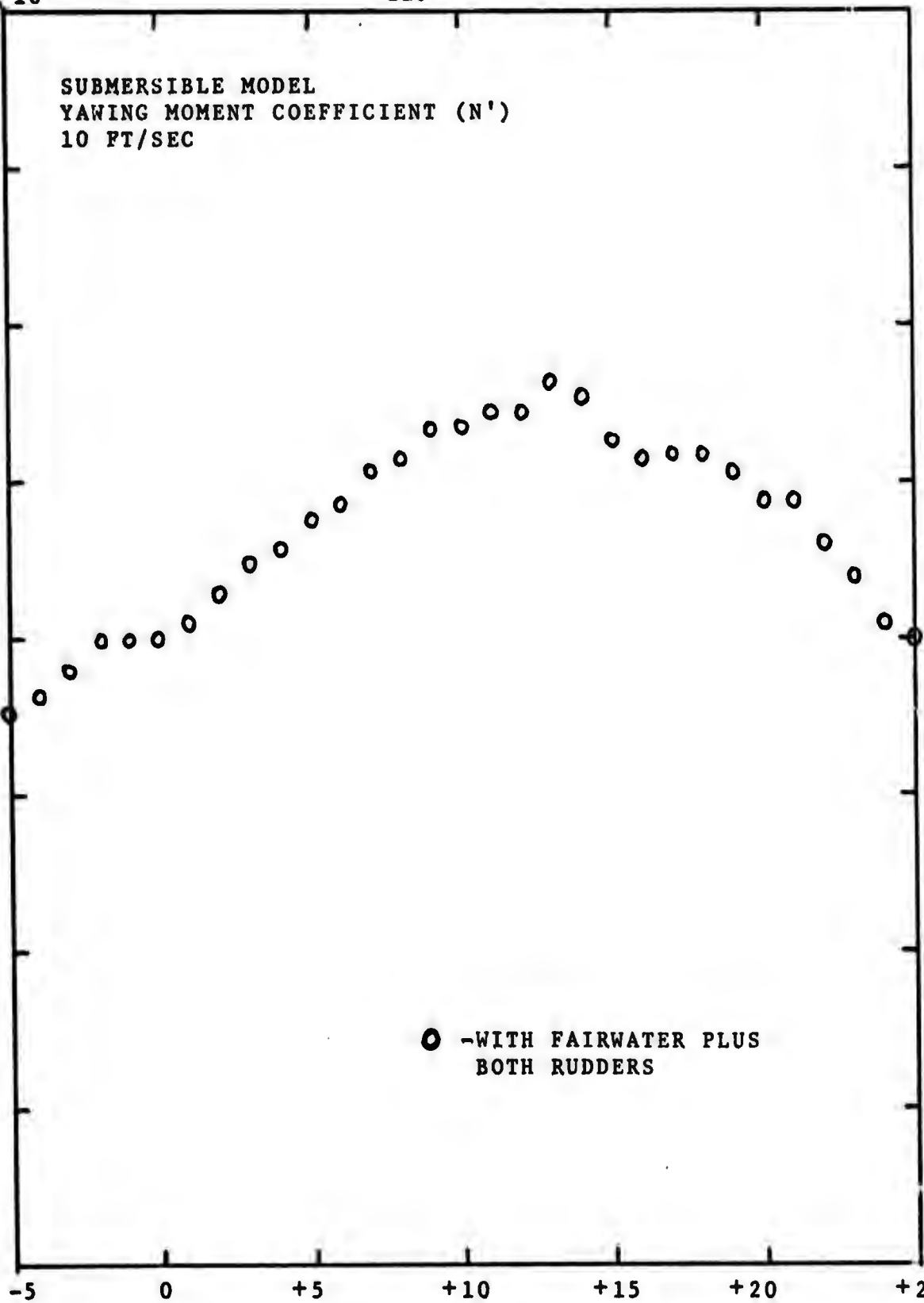


Figure 68

4 x 10⁻³

SUBMERSIBLE MODEL
YAWING MOMENT COEFFICIENT (N')
10 FT/SEC

+3
+2
+1
N'
0
-1
-2
-3
-4



○ -WITH FAIRWATER PLUS
BOTH RUDDERS

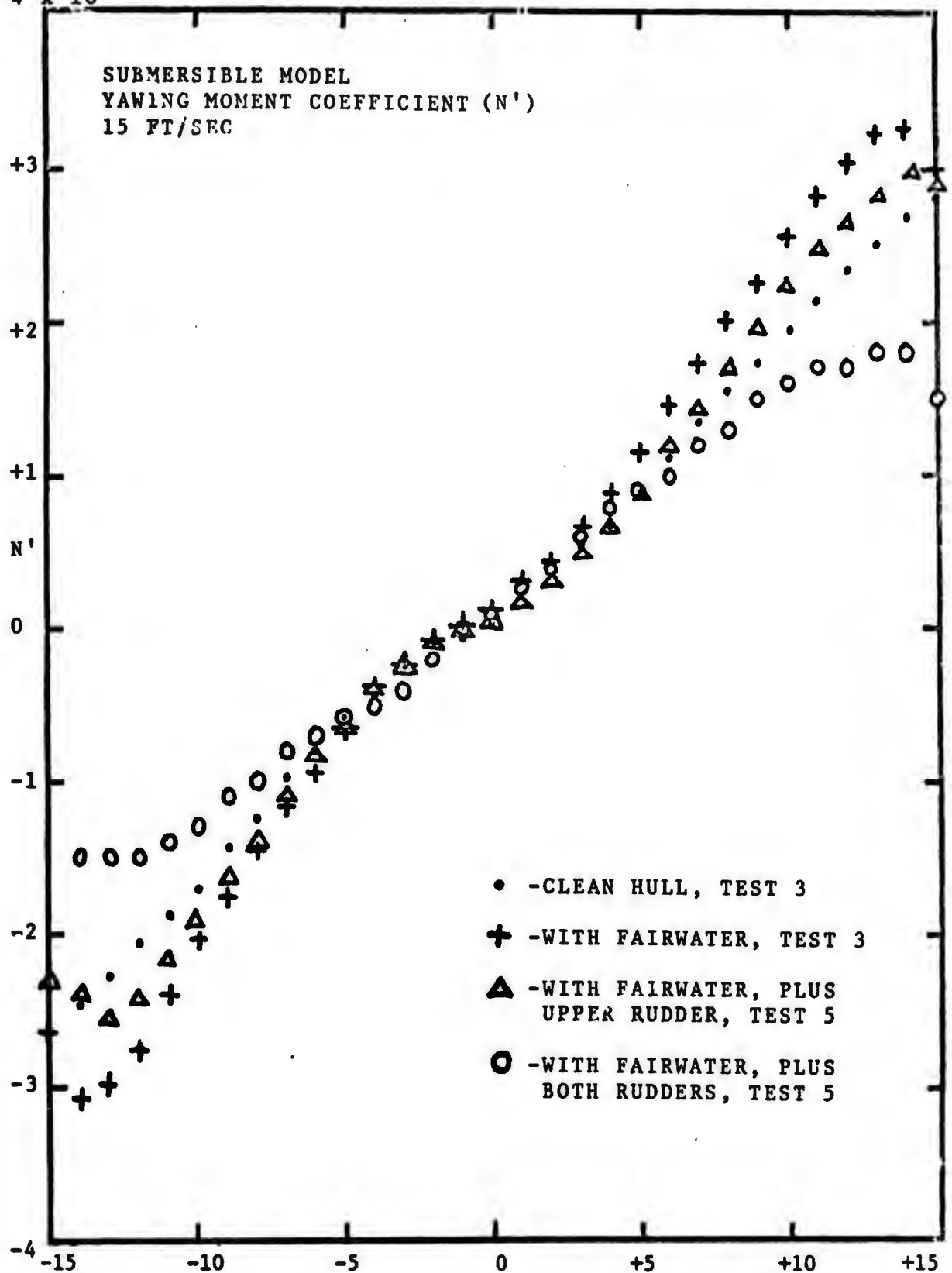
Angle of Attack - (Degrees-Yaw)

Figure 69

4 x 10⁻³

-120-

SUBMERSIBLE MODEL
YAWING MOMENT COEFFICIENT (N')
15 FT/SEC



- -CLEAN HULL, TEST 3
- + -WITH FAIRWATER, TEST 3
- ▲ -WITH FAIRWATER, PLUS UPPER RUDDER, TEST 5
- -WITH FAIRWATER, PLUS BOTH RUDDERS, TEST 5

Angle of Attack- (Degrees-Yaw)

Figure 70

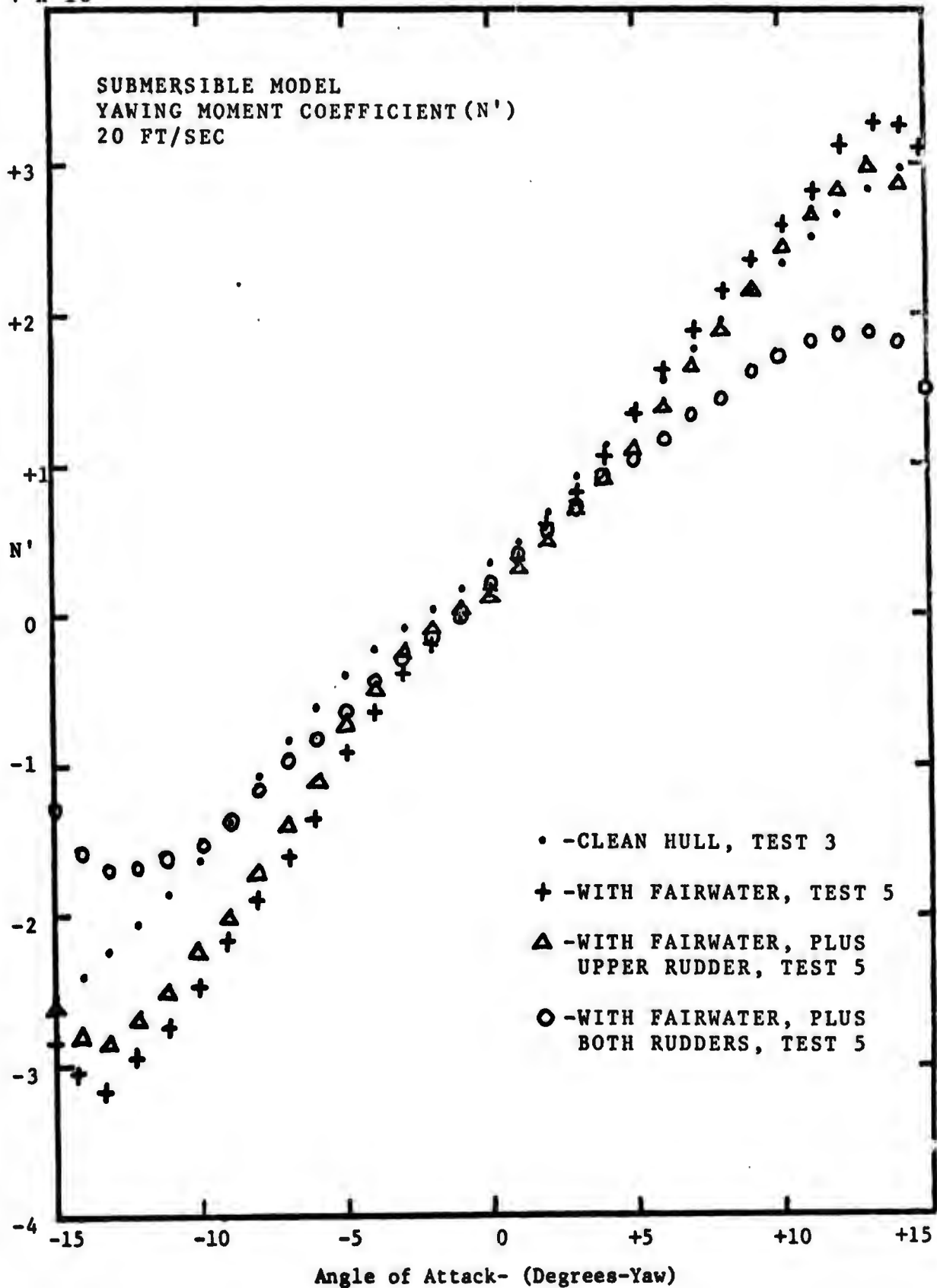


Figure 71

B. Newman-Rodriguez Results

Theoretical results obtained by Dr. J. N. Newman and Neptune Rodriguez from an investigation of a linearized low aspect ratio slender body theory are presented for comparison with experimental results, see Figures 72 and 73, taken from (13).

$$\text{Lift Coefficient} = C_L = \frac{L}{\rho b_o^2 U^2 \alpha \pi}$$

where:

L = lift force, perpendicular to flow

ρ = fluid density

b_o = radial distance to tip of fairwater

U = flow velocity

α = yaw angle.

Configuration 2 (with fairwater)

$$\frac{r_o}{b_o} = .354$$

$$\frac{b_t}{b_o} = 0$$

C_L (experimental) = .433, using Figure 74

C_L (Newman-Rodriguez) = .375

Configuration 3 (with fairwater and upper rudder)

$$\frac{r_o}{b_o} = .354$$

$$\frac{b_t}{b_o} = .417$$

C_L (experimental) = .445, using Figure 75

C_L (Newman-Rodriguez) = .270

Configuration 4 (with fairwater and both rudders)

$$\frac{r_o}{b_o} = .354$$

$$\frac{b_t}{b_o} = .417$$

C_L (experimental) = .497, using Figure 76

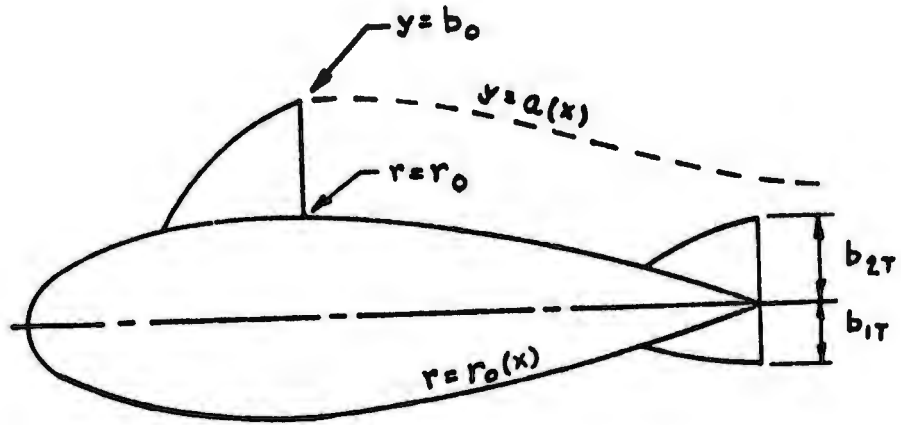
C_L (Newman-Rodriguez) = .475

The greater difference in lift coefficients for configuration 3 results from the fact that the downwash from the fairwater, although decreasing the effective angle of attack of the rudder, does not cause the flow to become parallel to the body axis. In the linear theory, the trailing vortex sheet passes directly over the upper rudder, thereby decreasing the

rudder angle of attack. Experimentally and in the numerical model, the trailing vortex system misses the rudder.

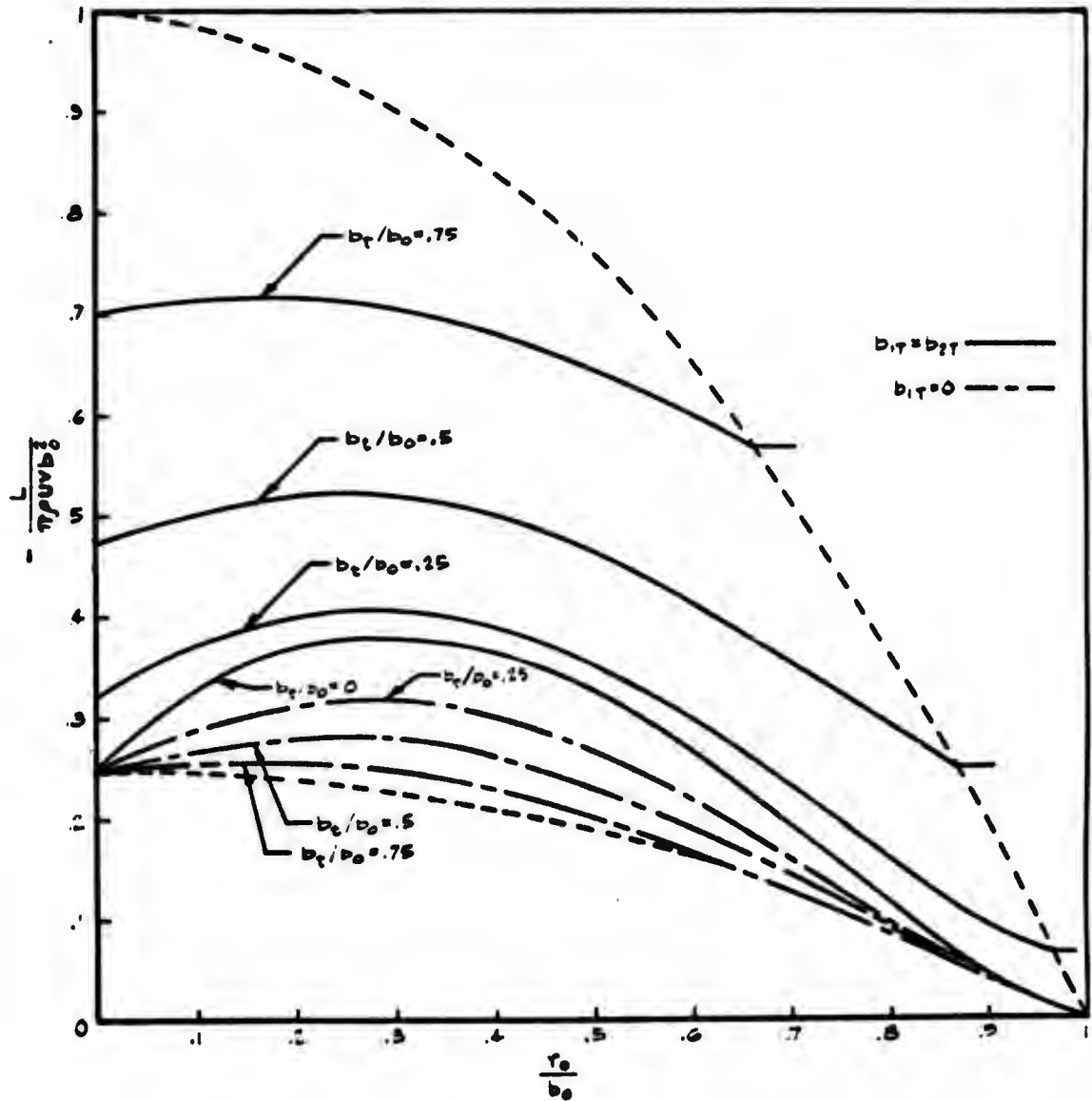
Therefore, more lift is generated by the rudder than would appear in the linearized theory used by Newman and Rodriguez.

The experimental and theoretical values of C_L are actually in better agreement than presented, since the experimental lift coefficients calculated included a minor component of lift resulting from the support shaft, which was not deducted before calculation.



The axi-symmetric body with asymmetric tail fins and one upstream fin.

Figure 72



Lift coefficient of the axi-symmetric body with asymmetric fins shown in Figure 72. The upper family of curves (—) are for a symmetric tail configuration ($b_{1T} = b_{2T}$), and the lower family of curves (---) are for a single upper tail fin ($b_{1T} = 0$). The curve $b_{2T}/b_0 = 0$ is for a body without tail fins. Note that the symmetric tail fin carries a positive lift force, whereas the upper tail fin experiences a negative lift force due to the effects of downwash.

Figure 73

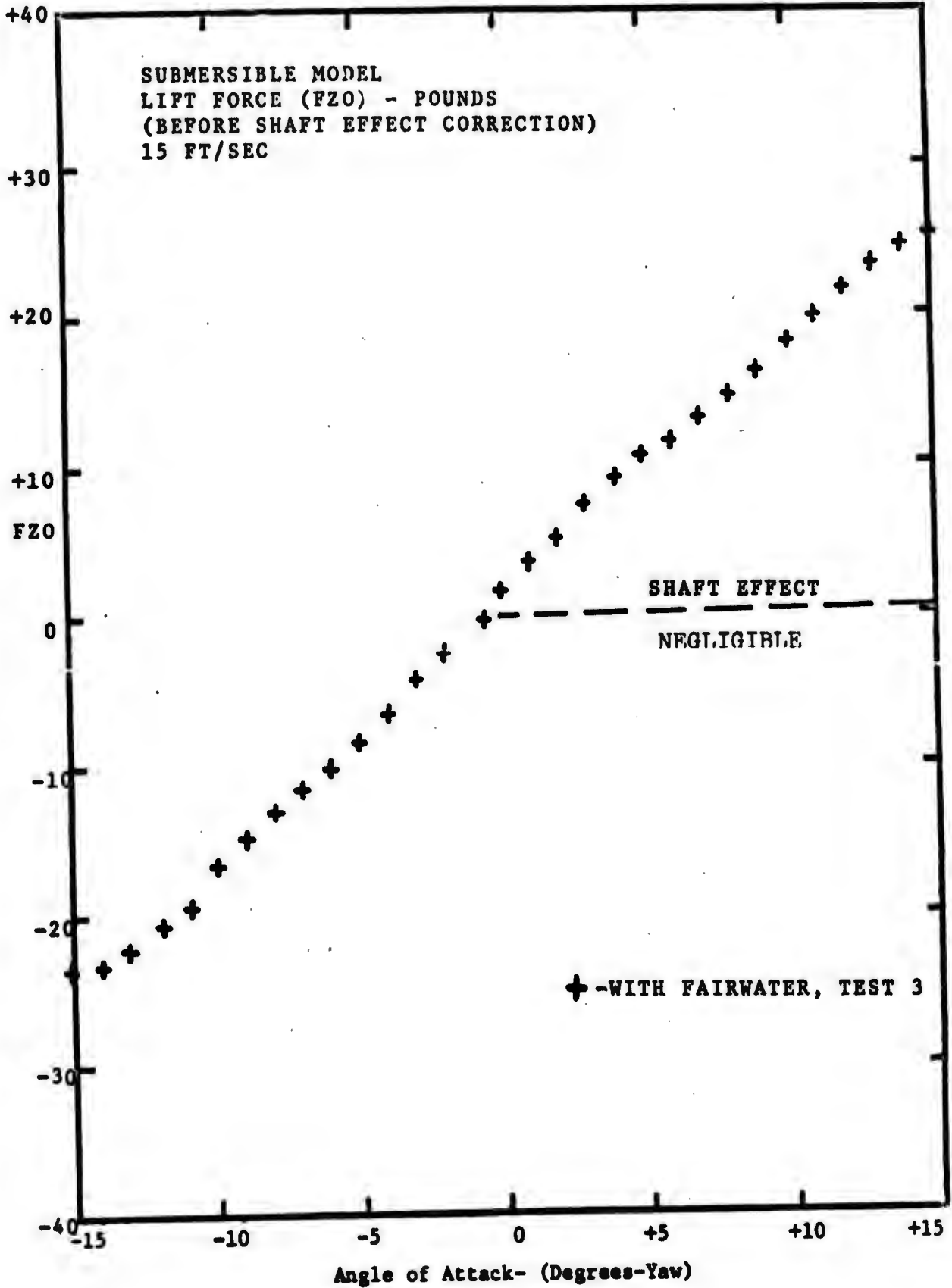


Figure 74

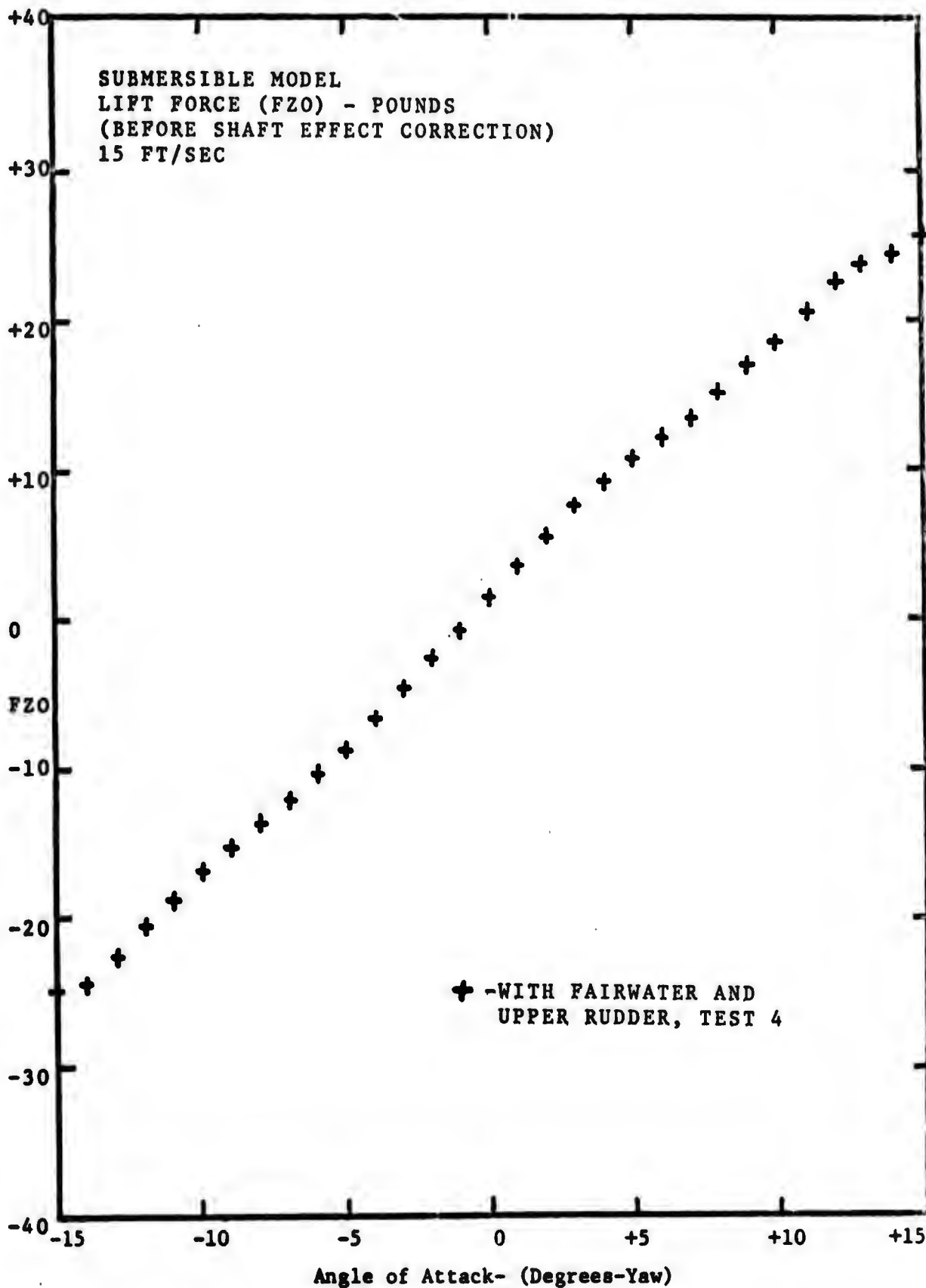


Figure 75

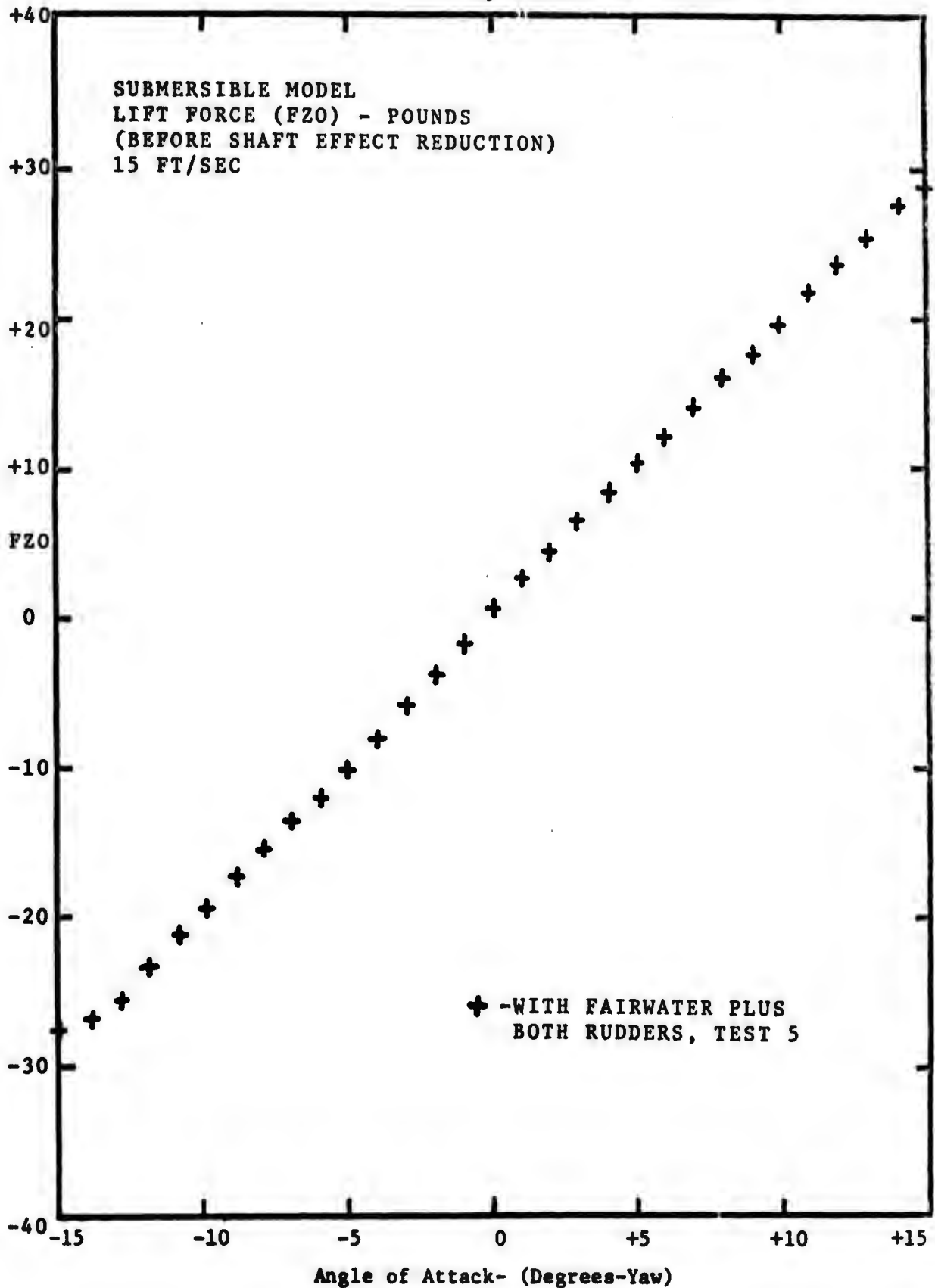


Figure 76

C. Numerical Prediction

Numerical procedure results for a submersible

with:

1. fairwater and upper rudder
2. Angle of yaw = 10 degrees
3. flow velocity = 15 ft/sec; therefore, free stream velocity = 14.77 ft/sec, side velocity = 2.6 ft/sec
4. wake angle = 10 degrees (free stream)

were:

Rudder side force	= 1.025 lb
Fairwater side force	= 11.754 lb
<u>Source side force</u>	<u>= 1.394 lb</u>
Total side force	= 14.173 lb

Rudder yaw moment	= 1.0947 ft·lb
Fairwater yaw moment	= -0.8960 ft·lb
Source yaw moment	= 7.2101 ft·lb
<u>Doublet yaw moment</u>	<u>= -9.0152 ft·lb</u>
Total yaw moment	= -1.064 ft·lb

Heave force on body	= -5.0780 lb
Pitch moment on body	= 3.5585 ft·lb

Comparable averaged experimental results were:

<u>Test 4 results</u>	<u>Test 5 results</u>	
18.0	19.0	Side force (FZ) lb
-3.75	-4.0	Yaw moment (MYO) ft·lb
-1.7	-2.7	Heave force (FYO) ft
3.0	7.0	Pitch moment (MZ) ft·lb

Numerical results were also compared with preliminary results from NSRDC for the base hull configurations. Bare hull configuration for the numerical program was accomplished by reducing the span length of the fairwater and rudder, measured from the body axis of symmetry to the radius of the hull surface at the appropriate points on the hull. The results were:

Rudder side force	= 0.012 lb
Fairwater side force	= 0.003 lb
<u>Source side force</u>	<u>= 2.990 lb</u>
Total side force (FZ)	= 3.004 lb

No side force should result from fairwater or rudder; therefore,

Total side force (FZ) = 2.99 lb

Rudder yaw moment	= 0.0134
Fairwater yaw moment	= 0.0000
Source yaw moment	= 8.5134
<u>Doublet yaw moment</u>	<u>= -9.7262</u>
Total yaw moment	= -1.1994

No moment should result from fairwater or rudder; therefore,

Total yaw moment (MYO) = -1.2128 ft lb

Since

$$Y' = \frac{-FZ}{\frac{1}{2} \rho U^2 L^2} = \text{Lateral Force Coefficient}$$

$$N' = \frac{MYO}{\frac{1}{2} \rho U^2 L^3} = \text{Yaw Moment Coefficient}$$

where ρ = fluid density

U = free stream velocity

L = model length

$$Y'(\text{numerical}) = -3.41 \times 10^{-3}$$

$$N'(\text{numerical}) = -.679 \times 10^{-3} \quad (\text{needs a greater doublet strength correction aft of the fairwater})$$

Comparable results for NSRDC and this investigation were:

NSRDC

Experimental

$$Y' = -3.5 \times 10^{-3}$$

$$Y' = -5.92 \times 10^{-3}$$

$$N' = -1.9 \times 10^{-3}$$

$$N' = -1.8 \times 10^{-3}$$

VIII. CONCLUSION

The purpose of this project was to investigate hull-control surface interactions on submerged bodies, to include the effects imposed on the body by the fairwater and rudders, when the body is at a constant yaw angle.

It has been shown through experimental and numerical models, and flow visualization, that the addition of a non-symmetric appendage (fairwater) on a symmetrical body of revolution has a significant effect on the hull and the control surfaces (rudders) located downstream.

The fairwater, besides being a lifting surface and generating a side (sway) force when at an angle of yaw, also induces a circulation around the hull of a submersible. When this circulation and a sway velocity are combined, a lift is generated on the hull. These asymmetric forces then produce moments around each of the three coordinate axes and cross-coupling moments. The fairwater was also instrumental in decreasing the effective angle of attack of the upper rudder, located on the same side and downstream of it. The trailing vortex sheet, shed from the fairwater, induced velocities on this rudder and effectively changed the direction of fluid flow meeting the rudder. This change in flow direction was readily seen during the flow visualization experiments.

The upper and lower rudders produced relatively minor

effects on the forces and moments at small angles of yaw. Only at high angles of yaw, when the fairwater and support-shaft downwashes were least effective on the rudders, did the side (sway) force of the rudders combine with the extended moment arm to produce a major effect. Investigation of the effects of the rudders on the body of revolution, without the fairwater, is needed for a more complete analysis.

The numerical model considered the effects of the fairwater and although requiring refinement, the initial results are comparable with experimental results obtained during this investigation.

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APPENDIX A

THE M.I.T.

VARIABLE PRESSURE WATER TUNNEL

APPENDIX A

THE M.I.T. VARIABLE PRESSURE WATER TUNNEL (14)

The M.I.T. Variable Pressure Water Tunnel was built in 1938 - 39, and provided a significant advance in propeller and model test facilities at that time. Since then, several modifications have been made to the tunnel to extend its useful life, to make it more convenient to use, and to give it additional capabilities in keeping with current research needs.

General Features

The tunnel is a closed-return type and is approximately a 24-foot diameter square in shape. See Figure A-1. It occupies parts of the first and second floors of Building 3 on the M.I.T. campus. The smallest internal diameter is approximately 30 inches, but this gradually widens to a 60-inch maximum before narrowing again to enter the test section. The tunnel cross-section is circular except at the test section and in the tapered transition section. The transition section narrows the 60-inch circular cross-section to the 20-inch square test section.

Testing Facilities

The tunnel can provide water velocities in the test

section up to 33 ft/sec and maintain any selected internal pressure from atmospheric (760mm Hg) down to approximately 140 mm Hg.

The test section is accessible via any of four identical removable plexiglas windows. The windows are 2.0 inches thick by 13 7/16 in. by 41 15/16 in. high and cover access holes measuring 41 15/16 by 13 7/16 in. The windows are readily cut or drilled and are thus ideal for mounting experiments. The dynamometer apparatus used allows measuring six-degrees-of-freedom forces on lifting surfaces, such as rudders or hydrofoils. Fittings are installed in the windows to accept Pitot tubes at several locations.

The tunnel may also be operated with a free surface to test-surface piercing hydrofoils or to simulate a miniature towing tank for small ship model testing.

Impeller Drive Systems

A 100 HP motor drives three DC generators. One of these, a 60 KW shunt-wound generator, provides power for the 75 HP DC impeller drive motor. The second, a 30 KW shunt-wound DC generator, provides power for the 40 HP propeller drive motor. The third is a 3 KW compound-wound DC generator and provides excitation for the other two DC generators.

A variable resistance vernier control system provides close control of impeller RPM.

A device attached to the impeller drive shaft provides a 60 pulse per revolution output. The signal is fed to a Hewlett-Packard 5212A electronic counter. By using 60 pulses per revolution, the counter reads directly in RPM instead of RPS.

Water Flow

A four-bladed bronze impeller is used to provide water flow. It is located in the lower level horizontal run in the 30-inch diameter section. It is immediately followed by straightening vanes to remove rotation from the water.

Each 90° bend has several thin stainless steel turning vanes with a non-symmetrical cross-section developed from airfoil theory. The vanes have a low power loss and the resolving turbulence generated by rounding the corners is fine-patterned and on the order of the turning vane spacing.

The 30-inch diameter widens out to 60 inches to slow the water velocity and further reduce turbulence. After rounding a 90° bend, the water is passed through a honeycomb of one-inch diameter plastic tubes and 2 sections of metal screens.

The 60-inch diameter is then smoothly constricted to flow through the 20-inch square test section. The section area reduction is 7.07. The constriction speeds up the flow velocity and at the same time stretches out any remaining turbulence. A smooth flow pattern results.

The Variable Pressure System

Any pressure in the range of atmospheric to 140 mm Hg can be maintained by the vacuum pump. The lower limit is dictated by the vapor pressure of water and small air leaks in the tunnel. Water vapor pressure at 60° F is 135 mm Hg.

Suction from the vacuum pump is taken at three points on the tunnel: (1) from the vacuum dome; (2) from the manhole cover over the 60-inch horizontal section; and (3), at the last 90° bend before entering the honeycomb.

If the tunnel is completely filled with water, some water may be sucked into the vacuum line to the pump. This is trapped in a receiver tank. Suction by the pump is taken from the top of the tank, while a drain line is provided at the bottom of the tank. The drain line drops 35 feet and is kept below water level in the building basement.

The vacuum system is the primary means of removing dissolved gases from the tunnel water. Dissolved oxygen is chemically removed for corrosion protection of the mild steel walls, and other gases are removed by keeping the tunnel at reduced pressure for several hours. If this is not done, dissolved gases form bubbles at reduced pressure and obscure the viewing of experiments in the test section.

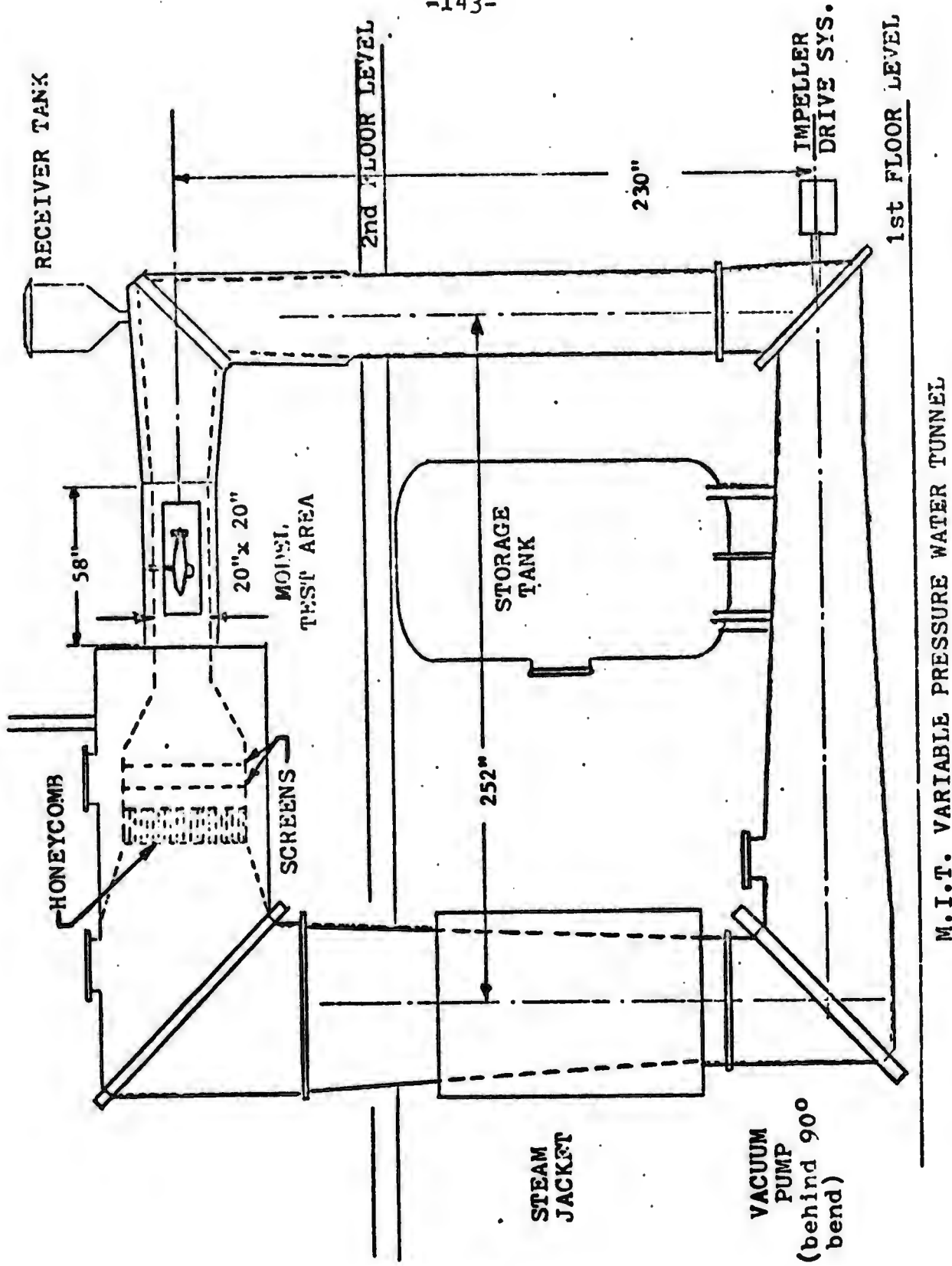


Figure A-1

M.I.T. VARIABLE PRESSURE WATER TUNNEL

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APPENDIX B

DYNAMOMETER PROGRAM USED

FOR

EXPERIMENTAL MODEL TESTS

```

C DYNAMOMETER PROGRAM USED TO EVALUATE THE FORCES AND MOMENT UN THE
C SUBMERSIBLE AND FAIRWATER MODELS (EXPERIMENTAL)
C MK4 RUDDER DYNAMOMETER DATA REDUCTION PROGRAM KERWIN/LEWIS MAY 72
  INTEGER RS,S,BLANK
  REAL MX,MYU,MYU,MZ,MZC,MFLAP
  DIMENSION IDEN(18),ZM(2),ZI(6,2,2),ZF(2),NTAP(32),INFLD(32),ANUM(3
  12),ANGU(32),S(32,6),R(32,7),DZI(6,2),CU(32),CM(32),CPL(32),
  2CY(32),CMF(32),VC(5),VE(5),FM(2),C(6),CLU(32),KN(32),SI(5
  24),CC(32)
  DATA VC/4.11447,2.60088,C.9016,0.6214,4.73017,VE/.49353,.5,.4724,0
  1.20000,.5/,RS/'R',BLANK/' '
  READ(2,104)(C(N),N=1,6),TWIST,SHAFT
  104 FORMAT(6F10.5)
  1 READ(2,103)(IULN(N),N=1,18)
  103 FORMAT(18A4)
  READ(2,100) DF,NRT,NTI,AREA,SPAN,AC,XAC,ZAC,FMC
  100 FORMAT(F5.1,F5.1,F10.2,F5.1,F5.3,F5.3,F10.2,F9.5)
  IF(AREA.LL.0) GO TO 99
  RMU=1.0074-0.00028*NRT
  SCALL=1.000552-13.3E-6*NRT
  FM(1)=1.0070-0.00071*NRT
  FM(2)=24.5004-0.00236*NRT
  WRITE(5,200)(IDENT(N),N=1,18)
  200 FORMAT(18,17X,'FLAPPED RUDDER INPUT DATA',1X,18A4/5X,'DF',5X,'TR',
  1,7X,'TI',5X,'AREA',6X,'SPAN',4X,'C-MAC',4X,'X-MAC',14X,'ZAC',6X,'F
  ZMC')
  WRITE(6,100) DF,NRT,NTI,AREA,SPAN,AC,XAC,ZAC,FMC
  READ(5,101)(ZM(K),I(ZI(M,L,K),L=1,2),M=1,6),ZF(K),K=1,2)
  101 FORMAT(F5.0,12X,F5.0,F4.0,F5.0,F4.0,F5.0,F4.0,F5.0,F4.0,F5.0,F4.0,
  1F5.0,F4.0,F9.0)
  WRITE(6,201)
  201 FORMAT(10,201)
  1 2-R 3-N 3-K 4-N 4-R 4-K 5-N 5-K 6-N 6-K 6-LAP)
  WRITE(6,204)(ZM(K),I(ZI(M,L,K),L=1,2),M=1,6),ZF(K),K=1,2)
  204 FORMAT(12(F4.0,7(2X,F4.0),2(1X,F4.0),2(2X,F4.0),1X,F6.0/))
  WRITE(6,212)(C(N),N=1,6),TWIST,SHAFT

```

```

P0M10001
P0M10002
P0M10003
P0M10004
P0M10005
P0M10006
P0M10007
P0M10008
P0M10009
P0M10010
P0M10011
P0M10012
P0M10013
P0M10014
P0M10015
P0M10016
P0M10017
P0M10018
P0M10019
P0M10020
P0M10021
P0M10022
P0M10023
P0M10024
P0M10025
P0M10026
P0M10027
P0M10028
P0M10029
P0M10030
P0M10031
P0M10032
P0M10033
P0M10034
P0M10035
P0M10036

```



```

K(J,M)=K(J,M)-ZI(M,1,1)-BUG*ZLI(M,1)
UU TC 7
K(J,M)=-K(J,M)+ZI(M,2,1)+BUG*ZLI(M,2)
CONTINUE
K(J,7)=K(J,7)-ZF(1,1)-BUG*ZLF
WRITE(0,205)(NIAP(J),NFLD(J),ANUM(J),ANGL(J),(K(J,M),M=1,6),R(J,7)
1,J=1,JI)
205  FURMAT(//, INPUT DATA CORRECTED FOR ZERO READINGS AND SIGNS'/(I
13,11,F0.0,F7.2,2X,6F9.2,F7.0))
WRITE(0,207)(IDENT(N),N=1,18)
207  FURMAT(//, **FLAPPED RUBBER DATA REDUCTION**//IX,16A4//3X, ALP
IHA FX-LB FZ-LB FX-INLB MZ-INLB FXJ-LB FYU-LB FZU-LB
2 MAX-INLB MYU-INLB MZG-INLB MYGFLAP-INLB VEL-FPS')
DEAR=AC/(12*3.9739*EXP(67.6032/NIT))
UU 9 J=1,JI
I=NIAP(J)
IF(1.0LT.1.0GR.1.0I.5) GO TO 99
N=NFLD(J)
IF(K.LI.1.0K.K.GI.2) GO TO 99
JUG=VE(1)
V=(ANUM(J)*FM(K)*SCALE/(VC(1)*RHO)**BLG
FYHAFI=0.00327*SHAFT#2*V**2
FBUX1=C(1)*K(J,1)
FBUX2=C(2)*K(J,2)
FBUX3=C(3)*R(J,3)
FBUX4=C(4)*R(J,4)+FYHAFI#0.41477
FBUX5=C(5)*K(J,5)+FYHAFI#0.41477
FBUX6=C(6)*K(J,6)+FYHAFI#0.17045
FA=-FBUX3
FY=-FBUX4-FBUX5-FBUX6
FZ=-FBUX1-FBUX2
MX=14.725*(FBUX1+FBUX2)+6.002*(FBUX5-FBUX4)
MY=-18.0*FBUX2
MZ=17.034*FBUX6-14.725*FBUX3-5.5*(FBUX4+FBUX5)
FAU=FX*CU(J)+FZ*SI(J)
FYU=FY

```

PGMLJ073
PGMLJ074
PGMLJ075
PGMLJ076
PGMLJ077
PGMLJ078
PGMLJ079
PGMLJ080
PGMLJ081
PGMLJ082
PGMLJ083
PGMLJ084
PGMLJ085
PGMLJ086
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PGMLJ093
PGMLJ094
PGMLJ095
PGMLJ096
PGMLJ097
PGMLJ098
PGMLJ099
PGMLJ100
PGMLJ101
PGMLJ102
PGMLJ103
PGMLJ104
PGMLJ105
PGMLJ106
PGMLJ107
PGMLJ108

```

FZU=FZ*CU(J)-FX*SI(J)
MXU=MX*CU(J)+MZ*SI(J)
MYU=MY
MZU=MZ*CU(J)-MX*SI(J)
MFLAP=R(J,7)*FMC
ANGL(J)=ANGL(J)+MYU/TWIST
WRITE(6,206) ANGL(J),FX,FZ,MX,MZ,FXL,FYU,FZU,MXC,MYU,MZU,MFLAP,V
  FORMAT(11F9.2,F13.2,F8.2)
200 KVS=U.D*RHCF*AREA*V**2/144.0
    CL(J)=FZJ/KVS
    CLSG(J)=CL(J)**2
    CU(J)=FAU/KVS
    CLD(J)=CL(J)/CD(J)
    CM(J)=(MYU+FZ*XAC)/KVS/AC
    CPL(J)=(MXU/FZU)/SPAN
    CY(J)=FYU/KVS
    KN(J)=V*DLAK
    CMF(J)=MFLAP/KVS/AC
    WRITE(6,207)(IDENT(N),N=1,13)
207  FORMAT('1' **FLAPPED KUDDER DATA IN NON-DIMENSIONAL FORM**//3X104
1477' ALPHA CL CE CM CPL CY L/D CMF C
2L04 KN*10**-6')
WRITE(6,209)(ANGL(J),CL(J),CD(J),CM(J),CPL(J),CY(J),CLD(J),CMF(J),
1CLS4(J),KN(J),J=1,JT)
209  FORMAT(13.2,F7.3,F8.4,5F7.3,F8.4,F8.3)
DU 15 J=1,JT
ANGL(J)=ANGL(J)+0.01562*AREA*CL(J)
CU(J)=CU(J)+0.0003425*AREA*CL(J)**2
CLD(J)=CLD(J)/CD(J)
WRITE(6,211)
211  FORMAT(//' ABOVE DATA CORRECTED FOR TUNNEL INTERFERENCE'
2L04 KN*10**-6')
WRITE(6,209)(ANGL(J),CL(J),CD(J),CM(J),CPL(J),CY(J),CLD(J),CMF(J),
1CLS4(J),KN(J),J=1,JT)
DU TC 1

```

```

PGM10109
PGM10110
PGM10111
PGM10112
PGM10113
PGM10114
PGM10115
PGM10116
PGM10117
PGM10118
PGM10119
PGM10120
PGM10121
PGM10122
PGM10123
PGM10124
PGM10125
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PGM10127
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PGM10135
PGM10136
PGM10137
PGM10138
PGM10139
PGM10140
PGM10141
PGM10142
PGM10143
PGM10144

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PUMIUI45
PUMIUI46

-149-

PAGE 5

99 STOP
END

C SAMPLE DATA DECK FOR SUBMERSIBLE MODEL
 SUB WITH FAIRWATER, UPPER & LOWER RUDDERS 10 FT/SEC
 0.0 75 62.06 3.5 24.600 0.0 97.60
 00 100 100 50 50 100 100 50 50
 00 100 100 50 50 100 100 50 50
 11 N 110 N 50 R 77 N 156 R 49
 026 96.6 77 148 52
 022 95.6 77 160 55
 024 94.6 76 175 58
 020 93.6 76 185 60
 027 92.6 76 200 63
 026 91.6 75 210 66
 027 90.6

PGM10001
 PGM10002
 PGM10003
 PGM10004
 PGM10005
 PGM10006
 PGM10007
 PGM10008
 PGM10009
 PGM10010
 PGM10011
 PGM10012

Sample Results

FLAPPED RUDDER INPUT DATA
 SUR WITH FAIRWATER, UPPER & LOWER RUDDERS 10 FT/SEC
 DE TR YI AREA SPAN C-MAC X-MAC ZAC FMC
 0-0 75 81 62.06 3.5 24.600 0.7 97.67 0-0
 ZERO READINGS BEFORE AND AFTER
 ANGLE 1-N 1-R 2-N 2-R 3-N 3-R 4-N 4-R 5-N 5-R 6-N 6-R FLAP
 0. 100. 100. 50. 50. 50. 50. 100. 100. 50. 50. 50. 50. 0.
 0. 100. 100. 50. 50. 50. 51. 100. 100. 50. 50. 50. 50. 0.

CELL LMS/COUNT 1=0.79926 2=0.35000 3=0.20030 4=0.10050 5=0.48970 6=0.20490
 TWIST= 323.7 SHAFT DIA.= 0.75 IN.

INPUT DATA AS RECORDED

TF	ANOM	ANGLE	S	1	S	2	S	3	S	4	S	5	S	6	FLAP
11	676.	96.6	N	117.	N	50.	R	77.	N	136.	R	49.	R	64.	0.
00	675.	95.6		121.		51.		77.		148.		52.		63.	0.
00	624.	94.6		130.		53.		77.		160.		58.		63.	0.
03	626.	91.6		142.		55.		76.		175.		58.		62.	0.
00	627.	92.6		153.		56.		76.		185.		60.		62.	0.
00	626.	91.6		162.		56.		76.		207.		63.		62.	0.
00	627.	90.6		169.		58.		75.		210.		66.		61.	0.

INPUT DATA CORRECTED FOR ZERO READINGS AND SIGNS

11	626.	-1.00		10.00		0.0		-27.00		36.00		1.00		-14.00	0.
11	625.	-2.77		21.00		1.07		-26.83		48.79		-2.00		-13.00	0.
11	624.	-3.00		30.00		3.00		-26.67		60.00		-5.00		-13.00	0.
11	626.	-4.00		42.00		5.00		-25.50		75.00		-8.70		-12.00	0.
11	627.	-5.00		53.00		6.00		-25.33		85.00		-10.00		-12.00	0.
11	626.	-6.00		62.00		6.00		-25.17		100.00		-13.00		-12.00	0.
11	627.	-7.00		69.70		8.70		-24.00		110.77		-16.00		-11.00	0.

FLAPPED RUDDER DATA REDUCTIONS

SUB WITH FAIRMATER, UPPER & LOWER RUDDERS 10 FT/SEC

ALPHA	FX-LB	FZ-LB	MX-INLR	MZ-INLR	FXD-LB	FYD-LB	FZD-LB	MXO-INLR	MVO-INLR	MZO-INLR	MYOFLAP-INLR	VEL-FPS
-1.00	5.41	-0.99	-4.35	16.39	5.46	-1.43	-0.67	-5.31	0.0	16.11	0.0	10.30
-2.00	5.37	-2.13	-3.75	20.31	5.52	-1.38	-1.72	-5.30	0.0	19.77	0.0	10.29
-3.01	5.34	-3.13	-5.34	20.74	5.61	-1.11	-2.61	-7.27	-7.00	20.15	0.0	10.28
-4.01	5.11	-4.42	-4.37	20.66	5.57	-1.36	-3.82	-6.65	-4.00	20.04	0.0	10.31
-5.02	5.07	-5.56	0.41	20.08	5.75	-1.38	-4.86	-2.18	-5.00	19.96	0.0	10.31
-6.02	5.04	-6.45	-4.48	19.45	5.93	-1.42	-5.65	-7.27	-5.00	18.59	1.1	10.30
-7.02	4.81	-7.25	-7.77	21.12	5.93	-1.16	-6.37	-11.12	-7.00	19.57	0.0	10.31

FLAPPED RUDDER DATA IN NON-DIMENSIONAL FORM**

SUB WITH FAIRWATER, UPPER & LOWER RUDDERS 10 FT/SEC

ALPHA	CL	CD	CM	CPL	CY	L/D	CMF	CLSQ	RM1000--6
-1.00	-0.015	0.1234	0.0	2.265	-0.032	-0.123	0.0	0.0002	2.304
-2.00	-0.039	0.1251	0.0	0.882	-0.031	-0.311	0.0	0.0015	2.302
-3.00	-0.059	0.1273	-0.002	2.795	-0.025	-0.465	0.0	0.0035	2.300
-4.00	-0.086	0.1299	-0.004	0.497	-0.031	-0.696	0.0	0.0075	2.304
-5.00	-0.110	0.1299	-0.005	0.128	-0.031	-0.846	0.0	0.0120	2.306
-6.00	-0.128	0.1341	-0.005	0.368	-0.032	-0.553	0.0	0.0163	2.304
-7.00	-0.144	0.1338	-0.006	0.499	-0.026	-1.074	0.0	0.0207	2.306

ABOVE DATA CORRECTED FOR TUNNEL INTERFERENCE

ALPHA	CL	CD	CM	CPL	CY	L/D	CMF	CLSQ	RM1000--6
-1.00	-0.015	0.1234	0.0	2.265	-0.032	-0.123	0.0	0.0002	2.304
-2.00	-0.039	0.1251	0.0	0.882	-0.031	-0.311	0.0	0.0015	2.302
-3.00	-0.059	0.1274	-0.002	0.795	-0.025	-0.465	0.0	0.0035	2.300
-4.00	-0.086	0.1261	-0.004	0.497	-0.031	-0.696	0.0	0.0075	2.304
-5.00	-0.110	0.1300	-0.005	0.128	-0.031	-0.846	0.0	0.0120	2.306
-6.00	-0.128	0.1344	-0.005	0.368	-0.032	-0.553	0.0	0.0163	2.304
-7.00	-0.144	0.1342	-0.006	0.499	-0.026	-1.071	0.0	0.0207	2.306

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APPENDIX C

NUMERICAL PROGRAM TO GENERATE OFFSETS
FOR
CONSTRUCTION OF FOIL SHAPES

06M20037
 06M20038
 06M20039
 06M20040
 06M20041
 06M20042
 06M20043
 06M20044
 06M20045
 06M20046
 06M20047
 06M20048
 06M20049
 06M20050
 06M20051
 06M20052
 06M20053
 06M20054
 06M20055
 06M20056
 06M20057
 06M20058
 06M20059
 06M20060
 06M20061
 06M20062
 06M20063
 06M20064
 06M20065
 06M20066
 06M20067
 06M20068
 06M20069
 06M20070

```

3  IF (K .EQ. 2) STOP
    K=3
    YCEM=F/2.0
    XCEM=0.0+5/7.0
    WRITE(6,103)
103  FORMAT(//, ' BACK HALF')
    GO TO 1
  END
C  CALL SUBROUTINE PARABOLIC INTERPOLATION
C  SENDS Y(X) FROM TABLE OF
C  AB(M) AND DB(M) CONTAINING M POINTS,
C  FUNCTION FILED IN(X,AB,CR,NO)
C  DIMENSION AB(2),DB(2)
C  ANTPA(X1,X2,X3,X,Y1,Y2,Y3)=Y1-(X-X2)/((X1-X2)-(X1-X3))+
1  Y2-(X-X1)/((X2-X1)-(X2-X3))+Y3-(X-X1)/((X3-X1)-
2  (X3-X2))
3  IF(X-AB(1)) 1,3,2
   Y=DB(1)
   GO TO 99
1  Y=ANTPA(AB(1),AB(2),AB(2),X,CR(1),CR(2),CR(3))
2  GO TO 99
3  IF(X-AB(2))1,6,5
   Y=DB(2)
   GO TO 99
4  GO TO 1+3,10
   M=1
5  IF(X-AB(1))3,6,7
   Y=DB(1)
   GO TO 99
6  CONTINUE
7  M=ANTPA(AB(M-2),AB(M-1),AB(M),X,CR(M-1),CR(M))
8  CALL M=1
9  RETURN
END

```

C	SAMPLE DATA	DECK FOR	MACA (S(MOD)	8	A=0	CAMPER			
.1	.45	.000	1.0000	.0000	.0000	.2500	.1	.1	05020002
.0	.005	.0075	.0125	.025	.05	.075	.1	.1	05020003
.15	.2	.25	.3	.35	.4	.45	.5	.5	05020004
.55	.6	.65	.7	.75	.8	.85	.9	.9	05020005
.65	.075	1.0							05020006
.1	.655	.912	.1044	.1665	.2065	.2525	.2907	.2907	05020007
.3521	.600	.6363	.4637	.4832	.4952	.5	.4862	.4862	05020008
.4347	.652	.4393	.4335	.3612	.3111	.2532	.1377	.1377	05020009
.1147	.0763	.0333							05020010

APPENDIX D
NUMERICAL MODEL OF A SUBMERSIBLE
BODY OF REVOLUTION
WITH
LIFTING SURFACE APPENDAGES

```

C NUMERICAL MODEL OF A SUBMERSIBLE BODY CF REVOLUTION WITH LIFTING
C SURFACE APPENDAGES, TO DETERMINE THE HYDRODYNAMIC FORCES AND
C MOMENTS AND THEIR INTERACTIONS.
COMMON ALPHA,ALR,AUX(12),AR(5),AUXZ(6),C,CHOR(20),CSA,CMSSI(7),CHD
IKR1(33),CMSK1(7)
COMMON DELX,DELTA(2),DBT1,DELCL,DBCP,DCX,DBDK1
COMMON GS(30,3),G1,G2,GKKS1(31),GTR(31),GTF(31),GBVSI(31,20,3,2),G
IK(30,3),GBVK1(30,20,3,2)
COMMON I,IPIV(6),IPI(3),IHCL,J,JM,L(6),IER,EPS,ISP(35)
COMMON NSTA,NTS,NBSC,NCPS,NCPC,NBS,NHS,NHLFT,NMIN,PI,PIINV,UPIINV,
1W
COMMON KRU,KIM(21),RSL(2),RKUD(2),RHS,RW1,5,RHR,RHS2,RHR2,KWIR,RF,R
1F,RMSD(32),RILHS(32,7),RFR,KTR,KIRHS(32)
COMMON SIMV(21),SPNS,SPNR,SLEF,SLEF,SGC,SA,SY,SZ,S32,SIA,SILHS(32,
17),SPNR(21),SLEK,SLEK
COMMON INCL,IVSSI(31,20,3,2),IEN(32,4),IVSK1(31,20,3,2)
COMMON UJ,VZ(6,4,3,2),VZ1(8,4,3,2),VZK(6,4,3,2),VZBK1(6,4,3,2),VZ
1TK1(6,4,3,2),VZWK1(8,4,3,2),VZ
COMMON WFS,WBS1(6,4),WKSIR1(31),WDSR1(6,4),WTK1(6,4),WTF(31,21),
1TK(31,21),W(21)
COMMON XIR(21),XLEI(2),XLEP(2),XILE(2),XLEIR(2),XLEBR(2),X
1LEIR(2),XLEBR(2),XEVSI(20),XLE,XLE,XCPSI(6,4),X6,XBVR1(20),XCPR1(8
2,4)
COMMON YIKSI(31),YPSI(30),YCPSI(8),Y,YWKS1(31,21),YDUMK(31),YTRK1(
131),YBR1(30),YCPR1(6),ZWKSI
REAL LENGTH
FURMAT(14,5X,3F10.4)
101 FURMAT(16F10.5)
102 FURMAT(5F10.5)
103 FURMAT(6F10.3)
104 FURMAT(2F10.4)
105 FURMAT(413)
C INPUT HULL AND FLOW PROPERTIES
NSA=2
KLAU (5,100)NSTA,UU,WFS,RHO
KLAU (5,101)(XIN(J),J=1,NSTA)

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PUM30001
PUM30002
PUM30003
PUM30004
PUM30005
PUM30006
PUM30007
PUM30008
PUM30009
PUM30010
PUM30011
PUM30012
PUM30013
PUM30014
PUM30015
PUM30016
PUM30017
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PUM30031
PUM30032
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PUM30034
PUM30035
PUM30036

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PGM30037
 PGM30038
 PGM30039
 PGM30040
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 PGM30042
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 PGM30048
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 PGM30067
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 PGM30069
 PGM30070
 PGM30071
 PGM30072

```

1  READ (5,1011)(RIN(J),J=1,NSTA)
   DELX=XIN(1)-XIN(2)
   PI=3.14159
   P1INV=1./PI
   PPIINV=0.25*P1INV
   DO 1 J=1,NSTA
1  SINV(J)=PI*KIN(J)*#2
   INPUT FIN PROPERTIES
   DO 2 J=1,2
2  READ (2,102) RSL(J),XLET(J),XLEB(J),XLET(J),XTEB(J)
   INPUT RUDDER PROPERTIES
   DO 3 J=1,2
3  READ (3,103) RKUD(J),XLETR(J),XLEBR(J),XTEIK(J),XTEBR(J),DELTA(J)
   INPUT WAKE SHEDDING ANGLES
   READ (5,104) ALPHA,ALK
   IF(ALPHA.LQ.0.9999) ALPHA=WFS/UD
   IF(ALK.LQ.0.9999) ALR=WFS/UD
   INPUT GRID SIZES
   READ (2,105) NIS,NBSC,NCPS,NCPC
   CALCULATE SPAN OF FIN
   DUMX=C.5*(ALEB(1)+XTEB(1))
   RNS=FILL1 (DUMX,XIN,RIN,NSTA)
   NNS2=NNS#RNS
   RNS=NNS2/NSL(1)
   SPNS=KSL(1)-RNS
   CALCULATE RUDDER SPAN
   DUMX=C.5*(XLEBR(1)+XTEBR(1))
   RNR=FILL1 (DUMX,XIN,KIN,NSTA)
   NNR2=NRN#RNR
   RNR=NRN2/RKUD(1)
   SPNR=KNUD(1)-RNR
   INAILING VORTEX RADII ON FIN 1
   DBT1=SPNS/FLUAT(NIS-1)
   YTKS1(1)=RNS
   DO 4 N=2,NTS
4  YTKS1(N)=YTKS1(N-1)+DBT1
  
```

```

120 WRITE(6,120)(YTRSI(N),N=1,NTS)
C   FORMAT(' TRAILER RADII ON SAIL',/(16F7.3))
C   CALCULATE MIDPOINT RADII ON SAIL 1
YBS1(1)=YTRSI(1)+DBTI*0.5
NBS=NIS-1
DO 2 N=2,NBS
YBS1(N)=YBS1(N-1)+DBTI
C   CALCULATE X VALUE OF BOUND VORTEX LINES ON SAIL 1
XBS1(1)=XLEB(1)
NBS=NBS-1
DELCL=(ALLO(1)-XTEB(1))/FLOAT(NBS)
DO 3 M=2,NBS
XBS1(M)=XBS1(M-1)-DELCL
WRITE(6,121)(XBS1(M),M=1,NBS)
121 FORMAT(' X LOCATION OF BOUND VORTEX LINES ON SAIL',/(16F7.2))
C   LOCATE SAIL 1 CONTROL POINTS SPANWISE
DOCP=(RSL(1)-RHS)/FLOAT(NCPS+1)
YCP1(1)=RHS+DOCP
DO 7 N=1,NBS
N=N
IF (YCP1(1).GT.YBS1(N).AND.YCP1(1).LE.YBS1(N+1)) GO TO 8
CONTINUE
YCP1(1)=YBS1(N)
NHLF=NBS-N
IHUL=FLOAT(NHLF)/FLOAT(NCPS)
DO 9 N=2,NCPS
YCP1(N)=YCP1(N-1)+IHUL*DBTI
CONTROL POINT SPACING CHECKWISE
KF=FILLI (XLEB(1),XIN,RIN,NSTA)
KF=FILLI (XTEB(1),XIN,RIN,NSTA)
SLEF=(XLEB(1)-XLET(1))/(RSL(1)-RF)
SLEF=(XLEB(1)-XLET(1))/(RSL(1)-RT)
DO 10 N=1,NCPS
Y=YCP1(N)
ALL=ALLO(1)-SLEF*(Y-RF)
ATL=ATEB(1)-SLEF*(Y-RT)

```

PGM3J73
 PGM3J74
 PGM3J75
 PGM3J76
 PGM3J77
 PGM3J78
 PGM3J79
 PGM3J80
 PGM3J81
 PGM3J82
 PGM3J83
 PGM3J84
 PGM3J85
 PGM3J86
 PGM3J87
 PGM3J88
 PGM3J89
 PGM3J90
 PGM3J91
 PGM3J92
 PGM3J93
 PGM3J94
 PGM3J95
 PGM3J96
 PGM3J97
 PGM3J98
 PGM3J99
 PGM3J100
 PGM3J101
 PGM3J102
 PGM3J103
 PGM3J104
 PGM3J105
 PGM3J106
 PGM3J107
 PGM3J108

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11 C=ALE-XTE
12 DCA=C/FLUAT(NCPC+1)
13 XCPST(N,1)=XTE+DCX
14 DO 11 M=1,NHS
15 M=M
16 IF(XCPST(N,1).LT.XBVS1(M).AND.XCPST(N,1).GE.XBVS1(M+1))GO TO 12
17 CONTINUE
18 XCPST(N,1)=XBVS1(M)-0.5*DELCL
19 DO 12 M=2,NCPC
20 XCPST(N,M)=XCPST(N,M-1)+DCX
21 DO 12 JM=1,NHS
22 JM=JM
23 IF(XCPST(N,M).LT.XBVS1(JM).AND.XCPST(N,M).GE.XBVS1(JM+1))XCPST(N,M
24 )=XBVS1(JM)-0.5*DELCL
25 CONTINUE
26 CONTINUE
27 DO 123 N=1,NCPS
28 WRITE(9,122)(YCPST(N),(XCPST(N,M),M=1,NCPC))
29 FORMAT(9,YCP=9,F10.3,XCP=9,F7.3)
30 DO 14 N=1,NBS
31 DO 14 M=1,NBS
32 XLE=XLEB(1)-SLEF*(YBS1(N)-RF)
33 IF(XLE.GT.XLEB(1))XLE=XLEB(1)
34 XLE=XTEB(1)-STEF*(YBS1(N)-RT)
35 IF(XLE.LT.XLEB(1))XLE=XTEB(1)
36 CHUR(N)=ALE-XTE
37 W=(YBS1(N)-RHS)/(RSL(1)-RHS)
38 SWW=(1.0-W**2)**0.5
39 DO 15 N=1,NBSM
40 YS(N,NBS)=SWW**2*(2*NBS-2)
41 GSUM1=0.0
42 GSUM2=0.0
43 G1=0.0
44 G2=0.0
45 DO 14 M=1,NBS
46 AU=XLE+.5*DELCL

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PGM30109
 PGM30110
 PGM30111
 PGM30112
 PGM30113
 PGM30114
 PGM30115
 PGM30116
 PGM30117
 PGM30118
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 PGM30139
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 PGM30142
 PGM30143
 PGM30144

```

IF(XBVSI(M) .GT. XD) GO TO 16
XI=XTE-.5*DELCL
IF( XBVS1(M).LT. XT ) GO TO 16
XI=XTE+.5*DELCL
IF( XBVS1(M) .LT. XI) GO TO 951
GO TO 952
THEIA=PI
GO TO 955
952 CONTINUE
GO TO 17
16 G1=0.0
G2=J.0
GO TO 16
17 COSX= (1.-2.*(XLE-XBVS1(M))+DELCL*.5)/CHUK(N))
THEIA=ARCCOS(COSX)
FUNKAT(0, COSX=*,F20.8,*, THEIA=*,F20.5)
G1=(THEIA+SIN(THEIA))*PIINV -GSUM1
G2=2.*PIINV*(THEIA*.5-SIN(.5*THEIA)*.25)-GSUM2
GSUM1=GSUM1+G1
GSUM2=GSUM2+G2
DO 14 NSP=1,NSM
GBVSI(N,M,NSP,1)=G1*GS(N,NSP)
GBVSI(N,M,NSP,2)=G2*GS(N,NSP)
DO 124 NSP=1,NSM
WRITE(6,125) NSP,(GS(N,NSP),N=1,NGS)
FUNKAT(0, MUDEL=,I2,(16F7.3))
MUJL=FIX(FLOAT(NTS)/2.)
WRITE(6,126) MUJL,(GBVSI(NCUT,M,1,1),M=1,NBSC)
FUNKAT(0, N=*,I2,CIRC MUDE 1,1,(12F7.5))
CALCULATE TRAILER STRENGTH UN SAIL 1
NRAIN=NTS-1
DO 19 NSP=1,NSM
DO 19 MCR=1,2
TVSS1(1,1,NSP,MCR)=GBVSI(1,1,NSP,MCF)
DO 20 M=2,NBSC
TVSS1(1,M,NSP,MCK)=GBVSI(1,M,NSP,MCF)+TVSS1(1,M-1,NSP,MCR)

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PGM30143
PGM30140
PGM30147
PGM30148
PGM30149
PGM30150
PGM30151
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PGM30157
PGM30158
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20 CONTINUE
   DO 21 N=2,NMIN
      TVSSI(N,1,NSP,MCR)=GBVSI(N,1,NSP,MCR)-GBVSI(N-1,1,NSP,MCR)
   DO 21 M=2,NBSC
      TVSSI(N,M,NSP,MCR)=TVSSI(N,M-1,NSP,MCR)+GBVSI(N,M,NSP,MCR)-GBVSI(N
1-1,M,NSP,MCR)
21 CONTINUE
   TVSSI(NIS,1,NSP,MCR)=-GBVSI(NBS,1,NSP,MCR)
   DO 19 M=2,NBSC
      TVSSI(NIS,M,NSP,MCR)=TVSSI(NIS,M-1,NSP,MCR)-GBVSI(NBS,M,NSP,MCR)
   WRITE(6,128) TVSSI(1,1,1,1),TVSSI(1,1,1,1),TVSSI(NIS,1,1,1),I
1VSSI(NIS,NBSC,1,1)
128 FORMATT(1) TRAILERS FOR UNIT MODE(1,1) AT BASC(LE,IE) AND TIP(LE,IE)
1,4F10.4)
C VELOCITY UN SI DUE TO SI BOUND VORTEX SYSTEM
   DO 22 NCP=1,NCPS
   DO 22 MCP=1,NCPC
   DO 22 NSP=1,NISM
   DO 22 MCR=1,2
      VZB(NCP,MCP,NSP,MCR)=0.0
      VZT(NCP,MCP,NSP,MCR)=0.0
      VZw(NCP,MCP,NSP,MCR)=0.0
      VZBRI(NCP,MCP,NSP,MCR)=0.0
      VZTRI(NCP,MCP,NSP,MCR)=0.0
      VZwRI(NCP,MCP,NSP,MCR)=0.0
   DO 23 NCP=1,NCPS
   DO 23 MCP=1,NCPC
   DO 23 N=1,NBS
   DO 23 M=1,NBSC
      SA=XBVSI(N)-XCP51(NCP,MCF)
      SY=YB51(N)-YCP51(NCP)
      S3Z=SBK1(SX*SX+SY*SY)**3
      VL=QPIINV/S3Z*(DBTI*SA)
   DO 23 NSP=1,NISM
   DO 23 MCR=1,2
      VZB(NCP,MCP,NSP,MCR)=VZB(NCP,MCP,NSP,MCR)+VZ*GBVSI(N,M,NSP,MCR)

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PUM3J161
PUM3J182
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NMID=IFIX(FLOAT(NCPS)/2.)
WRITE(6,129)XCPSI(NMID,NCPC),YCPSI(NMID),VZB(NMID,NCPC,1,1)
FORMAT(' VEL AT CP X=0,F7.3, Y=0,F7.3,F20.4, ' DUE TO MODE 1,1 80
129 1000 VORTICES')
C
VELOCITY ON SAIL 1 DUE TO SAIL 1 TRAILERS
DO 24 NCP=1,NCPS
DO 24 MCP=1,NCPC
DO 24 N=1,NTS
SY=YTRSI(N)-YCPSI(NCP)
DO 24 M=1,NHS
SX=0.5*(XBVSI(M)+XBVSI(M+1))-XCPSI(NCP,MCP)
SD2=SQR1(SX*5X+SY*SY)**3
VZ=PIIRV/SD2*SY*(XBVSI(M+1)-XBVSI(M))
DO 24 NSP=1,NSM
DO 24 MCR=1,2
VZ1(NCP,MCP,NSP,MCR)=VZ1(NCP,MCP,NSP,MCR)+VZ*TVSSI(N,M,NSP,MCR)
C
VELOCITY ON SAIL 1 DUE TO SAIL 1 WAKE
SIA=5IN(ALPHA)
CSA=CES(ALPHA)
X0=ADVSI(NBSC)
DO 25 NCP=1,NCPS
DO 25 MCP=1,NCPC
DO 25 N=1,NTS
KS=((XCPSI(NCP,MCP)-XB)*SIA)**2+(YTRSI(N)-YCPSI(NCP))**2
K=SQR(KS)
QS=(XCPSI(NCP,MCP)-XB)**2+(YTRSI(N)-YCPSI(NCP))**2
W=SQR(QS)
QV=PIIRV/K*((XB-XCPSI(NCP,MCP))/Q*CSA+1.0)
IF(YTRSI(N).GT.YCPSI(NCP)) QV=-QV
VZ=QV*CSA
DO 25 NSP=1,NSM
DO 25 MCR=1,2
VZW(NCP,MCP,NSP,MCR)=VZW(NCP,MCP,NSP,MCR)+VZ*TVSSI(N,NBSC,NSP,MCR)
C
WRITE(6,130)VZ1(NMID,NCPC,1,1),VZW(NMID,NCPC,1,1)
FORMAT(' VEL AT SAME POINT DUE TO TRAILERS ON BLADE=0,F15.4, ' AND
130 WAKE=0,F15.4, ' DUE TO UNIT MODE 1,1')

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C      VELOCITY UN SAIL 1 DUE TO HULL
      UU 26 NCP=1,NCPS
      UU 26 MCP=1,NCPC
      K=FULL(XCPSI(NCP,MCP),XIN,KIN,NSTA)
      WDSI(NCP,MCP)=WFS*K*R/(YCPSI(NCP)**2)
      WRITE(6,131)WDSI(NMID,NCPC)
      FORMAT(' VELOCITY AT SAME POINT DUE TO HULL=',F15.5)
C
      GET KFS FOR EACH CP
      I=1
      UU 27 NCP=1,NCPS
      UU 27 MCP=1,NCPC
      RHSS1(I)=-WFS-WDSI(NCP,MCP)
      WRITE(6,JUL) I,RHSS1(I)
      FORMAT(14,F40.10)
      I=I+1
      IMAX=NCPS*NCPC
      GET LEFT HAND SIDE FOR EACH CP
      I=1
      UU 28 NCP=1,NCPS
      UU 28 MCP=1,NCPC
      J=1
      UU 89 MCR=1,2
      UU 39 NSP=1,NSM
      SILHS(I,J)=VZM(NCP,MCP,NSP,MCR)+VZT(NCP,MCP,NSP,MCR)+VZB(NCP,MCP,N
      ISP,MCR)
      WRITE(6,802) I,J,SILHS(I,J)
      FORMAT(214,F40.10)
      J=J+1
      I=I+1
      UU 900 K=1,IMAX
      SILHS(K,5)=RHSS1(K)
      EPS=.0001
      CALL GLSQ(SILHS,CMSSI,ISP,IMAX,4,BUG,EPS,EPS)
C
      WRITE(6,132) (CMSSI(J),J=1,4)
      FORMAT(' MODE STRENGTHS ON SAIL=',6F10.5)
  
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C      VELOCITY AT RL DUE TO S1 WAKE
      UU 29 MS=1,NSTA
      MS=MS
      IF(XIN(MS).LT.XTEB(1)) GC TO 30
      CONTINUE
      NSTS=MS
      UU 31 MS=NSTS,NSTA
      MS=MS
      IF(XIN(MS).LT.XLEBR(1)) GO TO 52
      CONTINUE
      NSTE=MS
      UU 32 N=1,NTS
      YWKS1(N,NSTIS)=YTRSI(N)
      NSTEM=NSTE-1
      UU 34 MS=NSTIS,NSTEM
      CONST=.5*PIINV*(SINV(MS)-SINV(MS+1))
      UU 34 N=1,NTS
      YWKS1(N,MS+1)=YWKS1(N,MS)+CONST/YWKS1(N,MS)
      IF(YWKS1(N,MS+1).LT.RIN(MS+1))YWKS1(N,MS+1)=RIN(MS+1)*YTRSI(N)/RHS
      CONTINUE
      WRITE(6,133)(YWKS1(NDUT,MS),MS=NSTS,NSTEM)
      FORMAT(' TYPICAL TRAILER CONTRACTION TRAJECTORY ',L2F7.3)
      ZWKS1=ALPHA*(XTEB(1))-XLEER(1)
      UU 35 N=1,NTS
      WKS1(N)=Z*U
      J=1
      UU 35 MCR=1.2
      UU 35 NSP=1,NSM
      WKS1(N)=WKS1(N)+CMSSI(J)*1VSSI(N,NBSC,NSP,MCR)
      J=J+1
      WRITE(6,134)(GWKS1(N),N=1,NTS)
      FORMAT(' REAL WAKE TRAILER STRENGTHS FROM SAIL 1',/, '(IX,16F7.3)')
      UU 36 N=1,NBS
      YUWKN(N)=.5*(YWKS1(N,NSTE)+YWKS1(N+1,NSTE))
      WWSIK1(N)=U*U
      UU 36 NV=1,NTS
  
```

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K=5*DKI(ZWKSI**2+(YDUMR(N)-YWKSI(NV,NSTE))**2)
VEL=6*WKSI(NV)*.5*PI*INV/R
WWSIKI(N)=WWSIRI(N)+VEL*(YDUMR(N)-YWKSI(NV,NSTE))/R
WRITE(6,135)(WWSIRI(N),N=1,NBS)
FORMAT(' RAW VELOCITY ON RUDDER DUE TO SAIL',/, (1X,16F7.3))
FIT WITH POLYNOMIAL
KMIN=FULL(XLEBR(1),XIN,KIN,NSTA)
DU 028 N=1,NBS
NU=N
IF(YDUMR(N).GT.KMIN) GO TO 629
CONTINUE
NR=N-NBS-NU+1
DU 37 N=1,NRN
TEN(N,1)=1.0
TEN(N,2)=YDUMR(N+NU-1)
TEN(N,4)=WWSIRI(N+NU-1)
TEN(N,5)=TEN(N,2)**2
L2=1
EPS=.0001
CALL GLEM(TEN,AR,ISP,NRN,3,BUG,.0001,.0001)
WRITE(6,136)(AR(J),J=1,3)
FORMAT(' POLYNOMIAL COEFFICIENTS FOR RUDDER VEL DUE TO SAIL',3F10.
10)
C GRAD SETUP ON R1
DBTRI=SPNR/FLUAT(NBS)
YIKRI(1)=KWIR
DU 38 N=2,NTS
YIKRI(N)=YIKRI(N-1)+DBTRI
WRITE(6,140)(YIKRI(N),N=1,NTS)
FORMAT(' TRAILER RADII ON R1',/, (16F7.3))
YDRI(1)=YIKRI(1)+0.5*DBTRI
DU 39 N=2,NBS
YDRI(N)=YDRI(N-1)+DBTRI
XDVKI(1)=XLEBR(1)
DELCKI=(XLEBR(1)-XTEBR(1))/FLOAT(NBS)
DU 40 N=2,NBS

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PGM3J356
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PGM3J358
PGM3J359
PGM3J360

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40 XBVK1(M)=XBVK1(M-1)-DELCRI
WRITE(0,141)(XBVK1(M),M=1,NBSC)
141 FURNAT( X LUCATION OF BOUND VUKTEX LIALS UN RI',/, (16F7.2))
DUCPSK=(KRUD(1)-KTR)/FLCAT(NCPS+1)
YCPRI(1)=RHR+DUCPSK
DO 41 N=1,NBS
N=1
IF(YCPRI(1).GT.YBR1(N).AND.YCPRI(1).LE.YBR1(N+1)) GO TO 42
CONTINUE
41 YCPRI(1)=YBR1(N)
NHLF=INDS-N
INDL=FLCAT(NHLF1)/FLOAT(NCPS)
DO 43 N=2,NCPS
YCPRI(N)=YCPRI(N-1)+INDL*DBTRI
KTR=FULLI(XLEBK(1),XIN,KIN,NSIA)
KTR=FULLI(XTEBK(1),XIN,KIN,NSIA)
SLEK=(ALEBK(1)-XLETR(1))/(KRUD(1)-KTR)
SLEK=(ATEBK(1)-XTETR(1))/(KTR(1)-KTR)
DO 44 N=1,NCPS
Y=YCPRI(N)
ALLK=XLEBK(1)-SLEK*(Y-KTR)
XLEK=XLEBK(1)-SLEK*(Y-KTR)
CR=XLEK-XTR
DCKR=CR/FLCAT(NCPC+1)
XCPRI(N,1)=XLEK+DCKR
DO 45 MC=1,NHS
MC=MC
IF(XCPRI(N,1).LT.XBVK1(MC).AND.XCPRI(N,1).GE.XBVK1(MC+1))GO TO 46
CONTINUE
42 XCPRI(N,1)=(XBVK1(MC)+XBVK1(MC+1))*0.5
DO 44 M=2,NCPC
XCPRI(N,M)=XCPRI(N,M-1)+DCKR
DO 47 MC=1,NHS
MC=MC
IF(XCPRI(N,M).LT.XBVK1(MC).AND.XCPRI(N,M).GE.XBVK1(MC+1))GO TO 44
CONTINUE
47

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P0M30361
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 P0M30364
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44  XCPRI(N,M)=0.5*(XBVRI(MC)+XBVRI(MC+1))
    DU 143 N=1,NCPS
143  WRITE(6,142)(YCPRI(N),(XCPRI(N,M),M=1,NCPC))
142  FORMAT(' YCP=',F10.3,' XCP=',4F7.2)
    BUUND VURTEX STRENGTHS CN KI FOR UNIT MODES
    DU 46 N=1,NBS
    XLEK=XLEBR(1)-SLLEK*(YBK1(N)-RFR)
    IF(XLEK.GT.XLEBR(1)) XLEK=XLEBR(1)
    XTER=XTLBR(1)-SLTER*(YBK1(N)-RTR)
    IF(XLEK.LT.XTER(1)) XTER=XTER(1)
    CHOKRI(N)=XLEK-XTER
    W=(YBRI(N)-RHR)/(RKUD(1)-RHR)
    SWW=SWRT(1.0-Q*W)
    DU 47 NSP=1,NSM
49  UR1(N,NSP)=SWW*J*(2*NSP-2)
    G1=0.0
    G2=3.0
    USUM1=0.0
    USUM2=0.0
    DU 48 M=1,NBSC
    XD=XLEK+.5*DELCK1
    IT(ABVRI(M).GT.XD) GO TO 50
    XI=XTER-.5*DELCK1
    IF(XBVRI(M).LT.XI) GO TO 50
    XI=XTER+.5*DELCK1
    IF(XBVRI(M).LT.XI) GO TO 953
    GO TO 954
953  THETA=PI
    GO TO 955
954  CONTINUE
    GO TO 51
50  G1=0.0
    G2=0.0
    GO TO 52
51  COSX= (1.0-2.0*(XLEK-XPVRI(M)+.5*DELCK1)/CHOKRI(N))
    THETA=ARCUS(COSX)

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950 G1=(THETA+SIN(THETA))*PI*INV -GSUM1
    G2=2.0*PI*INV*(THETA*0.5-SIN(2.0*THETA)*0.25) -GSUM2
    GSUM1=GSUM1+G1
    GSUM2=GSUM2+G2
    DU 40 NSP=1,NSM
    GBVRI(N,M,NSP,1)=G1*GR(N,NSP)
    GBVRI(N,M,NSP,2)=G2*GR(N,NSP)
    DU 144 NSP=1,NSM
    AKTIL(6,145) NSP,(GR(N,NSP),N=1,NBS)
    FORKAI(' MDE',12,(16F7.3))
    AKTIE(6,146) NCUT,(GBVRI(NCUT,M,1,1),M=1,NBSC)
    FJMAI(' N=,I2,' CIRC MCDE 1,1',(12F7.5))
    DU 55 NSP=1,NSM
    DU 50 MCR=1,2
    TVSRI(1,1,NSP,MCR)=GBVRI(1,1,NSP,MCR)
    DU 54 M=2,NBSC
    TVSRI(1,1,M,NSP,MCR)=TVSRI(1,M-1,NSP,MCR)+GBVRI(1,M,NSP,MCR)
    DU 55 N=2,AMIN
    TVSRI(N,1,NSP,MCR)=GBVRI(N,1,NSP,MCR)-GBVRI(N-1,1,NSP,MCR)
    DU 55 M=2,NBSC
    TVSRI(M,M,NSP,MCR)=TVSRI(M,M-1,NSP,MCR)+GBVRI(M,M,NSP,MCR)-GBVRI(M-1,M,NSP,MCR)
    TVSRI(NIS,1,NSP,MCR)=-GBVRI(NBS,1,NSP,MCR)
    DU 55 M=2,NBSC
    TVSRI(NIS,M,NSP,MCR)=TVSRI(NIS,M-1,NSP,MCR)-GBVRI(NBS,M,NSP,MCR)
    AKTIE(6,148) TVSRI(1,1,1),TVSRI(1,1,NBSC,1,1),TVSRI(NIS,1,1,1),IVS
    IRI(NIS,NBSC,1,1)
    FURMAI(' TRAILERS FOR UNIT MUDE(1,1) AT BASE(LE,TE) AND TIP(LE,TL)
    1,4F10.4)
    VELOCITIES ON KI
    DU 50 MCP=1,NCPS
    DU 50 MCP=1,NCPC
    DU 50 N=1,NBS
    DU 50 M=1,NBSC
    SX=XGBVRI(M)-XCPR1(NCP,MCP)
    SY=YGBVRI(1)-YCPR1(NCP)

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50 JJZ=SQR(X*SX+SY*SY)**3
VZ=QPIINV/532*(DBTR1*SX)
UU 50 NSP=1,NSM
UU 50 MCR=1,2
VZBRI(NCP,MCP,NSP,MCR)=VZBRI(NCP,MCF,NSP,MCR)+VZ*GBVRI(N,M,NSP,MCR
1)
WRITE(6,145)XCPRI(NMID,NCPC),YCPRI(NMID),VZBRI(NMID,NCPC,1,1)
149 FORMAT(1,VEL AT CP X=,F7.2, Y=,F7.3,F20.4, DUE TO MODE 1,1 BOU
AND VORTICES')
UU 57 NCP=1,NCPS
UU 57 MCP=1,NCPC
UU 57 N=1,NIS
SY=YTRRI(N)-YCPRI(NCP)
UU 57 N=1,NHS
SX=U.5*(XBVRI(N)+XBVRI(N+1))-XCPRI(NCP,MCP)
JJZ=SQR(X*SX+SY*SY)**3
VZ=QPIINV/532*SX*(XBVRI(N+1)-XBVRI(N))
UU 57 NSP=1,NSM
UU 57 MCR=1,2
VZTRI(NCP,MCP,NSP,MCR)=VZTRI(NCP,MCF,NSP,MCR)+VZ*TVSRI(N,M,NSP,MCR
1)
SIAR=SIN(ALR)
CSAR=COS(ALR)
ASR=XBVRI(NBSC)
UU 50 NCP=1,NCPS
UU 50 MCP=1,NCPC
UU 50 N=1,NIS
KSR=((XCPRI(NCP,MCP)-XBR)*SIAR)**2+(YTRRI(N)-YCPRI(NCP))**2
KR=SQR(KSR)
ASR=(XCPRI(NCP,MCP)-XBR)**2+(YTRRI(N)-YCPRI(NCP))**2
WR=SQR(WR)
VVR=QPIINV/RR*((XBR-XCPRI(NCP,MCP))/CR*CSAR+1.0)
I=(YTRRI(N).GT.YCPRI(NCP))CVR=-QVR
VZ=JVR*CSAR
UU 50 NSP=1,NSM
UU 58 MCR=1,2

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50  VZWR1(NCP,MCP,NSP,MCR)=VZWR1(NCP,MCP,NSP,MCR)+VZ*TVSK1(N,NB5C,NSP,
1MCR)
WRITE(6,15C)VZTR1(NMID,NCPC,1,1),VZWR1(NMID,NCPC,1,1)
150  FORMAT(' VEL AT SAME POINT DUE TO TRAILERS ON BLADE=',F15.4,' AND
1 WAKE=',F15.4,' DUE TO UNIT MODUL 1,1')
C  VELOCITY ON R1 DUE TO HULL
DU 59 MCP=1,NCPS
DU 59 MCP=1,NCPC
K=ELLI(XCPRI(NCP,MCP),XIN,KN,NSIA)
WDR1(NCP,MCP)=WFS*R**7/(YCPRI(NCP)**2)
59  WRITE(6,151)WDR1(NMID,NCPC)
151  FORMAT(' VELOCITY AT SAME POINT DUE TO HULL=',F15.5)
C  VELOCITY AT R1 DUE TO SI AT CONTROL POINTS
DU 60 MCP=1,NCPS
R=YCPRI(NCP)
U=AK(1)+AK(2)*R+AK(3)*R**K
DU 60 MCP=1,NCPC
WDR1(NCP,MCP)=U
C  CALCULATE LHS FOR RUDDER
I=1
DU 61 MCP=1,NCPS
DU 61 MCP=1,NCPC
J=1
DU 62 MCR=1,Z
DU 62 NSP=1,NSM
KILHS(I,J)=VZWR1(NCP,MCP,NSP,MCR)+VZTR1(NCF,MCP,NSP,MCR)+VZUR1(NCP
1,MCP,NSP,MCR)
WRITE(6,602) I,J,KILHS(I,J)
62  J=J+1
61  I=I+1
C  CALCULATE RHS FOR RUDDER
I=1
DU 63 MCP=1,NCPS
DU 63 MCP=1,NCPC
KIKHS(I)=DELTA(1)*DU-WFS-WDIR1(NCP,MCP)-WDR1(NCP,MCP)
WRITE(6,601) I,KIKHS(I)

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PUM30000
PUM30006
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PUM30040

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63      I=I+1
        LPS=.0001
        DO 901 K=1,IMAX
          KILHS(N,D)=RILHS(K)
          CALL GLSQ(RILHS,CMSRI,ISP,IMAX,4,BUG,EFS,EPS)
          WRITE(6,122)(CMSRI(J),J=1,4)
          FORMAT('  RUDER STRENGTHS CN RI =',0F10.5)
          GOTO 121 WAKE TRAILER STRENGTHS
        DO 66 N=1,NIS
          UT(N)=0.0
          VIF(N)=0.0
          J=1
          DO 66 MCR=1,2
            DO 66 NSP=1,NSM
              VTA(N)=GTA(N)+IVSK1(N,NBSC,NSP,MCR)*CMSRI(J)
              VTB(N)=GTB(N)+IVSS1(N,NBSC,NSP,MCR)*CMSRI(J)
              J=J+1
            VELOCITY UN CL DUE TO FIN WAKE
          DO 64 N=1,NIS
            NFIN
            IF(YIRS1(N).GT.NHS)GO TO 64
            CONTINUE
            NDUF=N
            NIN=N-1
            DO 65 N=NDUF,NIS
              Y=YIAX1(N)
              YSD=Y*Y
              DO 65 M=1,INSTA
                CPI=XIN(M)-XB
                CPI=CPI /SQRT(YSD+(XIN(M)-XB)**2)
              VIF(N,M)=(CPI-1.0)*CPIINV/Y*GTI(N)
            VELOCITY UN CL DUE TO RUDDER WAKE
          DO 67 N=1,NIS
            NFIN
            IF(YIKR1(N).GT.NHS) GO TO 68
            CONTINUE

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PGM30574
PGM30575
PGM30576

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60  NUUTR=N
    NINR=N-1
    UU 09 N=NUUTR,NIS
    Y=YIRKI(N)
    Y0=Y#Y
    UU 09 M=1,NSIA
    CPI=XIN(M)-XBR
    CPI=CPI /SQR(YSG+(XIN(M)-XBR)**2)
    WIK(N,M)=(CPI-1.0)*CPIINV/Y*GTR(N)
    C    TOTAL VELOCITY AT STATION M UN CL
    UU 70 M=1,NSIA
    W(M)=WFS
    UU 71 N=NDUI,NIS
    W(M)=W(N)+WIF(N,M)
    UU 70 N=NUUTR,NIS
    W(M)=W(M)+WIK(N,M)
    WATL(0,193)(M),M=1,NSIA)
    C    FORMAT(' VELOCITY CN CL DUE TO FIN,RUDDLR,FREL SIR',/, (1ZF7.3))
    C    S1 SLOPE AT INPUT POINTS
    UU 72 M=1,NSIA
    SPIN(M)=SFILE(XIN(M),XIN,SINV,NSIA)
    WATL(0,194)(SPIN(N),M=1,NSIA)
    C    FORMAT(' SLOPE AT INPUT POINTS',/, (1ZF7.2))
    C    FORCES + MOMENTS DUE TO CIRC AROUND HULL DUE TO FIN
    NSDUM=NSIA-1
    WISH=NDUI-1
    GAMMA=0.0
    UU 01 N=1,NTSH
    GAMMA=GAMMA+GTF(N)
    S1Y=0.0
    S1N=0.0
    UU 02 M=1,NSDUM
    IF(XIN(M).GE.XTEB(1)) GO TO 02
    S1Y=S1Y+KHU*(M)*GAMMA*DELX
    S1N=S1N+KHU*(M)*GAMMA*DELX*XIN(M)
    C    CONTINUE

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 P0H00047
 P0H00048

1177

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S1Y=51Y+0.5*KRHU*W(NSTA)*GAMMA*DELX
S1W=S1W+0.5*KRHU*W(NSTA)*GAMMA*DELX*XIN(NSTA)
GET FORCES + MOMENTS DUE TO SOURCES ON CL
SOURZ=0.5*DELX*W(1)*KRU*CU*SPRM(1)
SOURM=-SOURZ*XIN(1)
SOURZ=SOURZ+0.5*DELX*W(NSTA)*KRU*CU*SPRM(NSTA)*2
SOURM=SOURM+0.5*DELX*W(NSTA)*KRU*CU*SPRM(NSTA)*XIN(NSTA)*2
LENGTH=XIN(1)-XIN(NSTA)
DO 73 N=2,NSJUM
  IF(XIN(N).GE.0.0) FAC=1.0
  IF(XIN(N).LT.0.0) FAC=1.0-2.0*(XIN(N)/LENGTH)
  SOURZ=SOURZ+DELX*W(M)*KRU*CU*SPRM(M)*FAC
  SOURM=SOURM-DELX*W(M)*KRU*CU*SPRM(M)*XIN(M)*FAC
  MOMENT DUE TO BUOYETS
  DUB1=-DELX*W(1)*KRU*CU*SINV(1)
  DUB2=DUB1+DELX*W(NSTA)*KRU*CU*SINV(NSTA)*.5
  DO 74 M=2,NSJUM
    IF(XIN(M).GE.0.0) FAC=1.0
    IF(XIN(M).LT.0.0) FAC=1.0+2.0*(XIN(M)/LENGTH)
    DUB1=DUB1-2.0*DELX*W(M)*KRU*CU*SINV(M)*FAC
    FORCE AND MOMENT UN FIN
  S1Z=0.0
  S1M=0.0
  DO 77 N=1,NBS
    S1Z=S1Z+KHL*JU*DBT1*(GS(N,1)*(CMSSI(1)+CMSSI(3))+GS(N,2)*(CMSSI(2)
    +CMSSI(4)))
    ALL=XLEB(1)-SLEF*(YB51(N)-RF)
    IF(XLE.0)T.XLEB(1) XLE=XLEB(1)
    XCP1=ALL-.25*CHUR(N)
    XCP2=XLE-.50*CHUR(N)
    S1M=S1M+RHC*UU*DBT1*(CMSSI(1)*XCP1*GS(N,1)+CMSSI(3)*XCP2*GS(N,1)+C
    MSSI(2)*XCP1*GS(N,2)+CMSSI(4)*XCP2*GS(N,2))
    FORCE AND MOMENT UN RUDDER
  X1Z=0.0
  X1M=0.0
  DO 50 N=1,NBS
  
```

```

      KLZ=KLZ+RHU*JU*DBTRI*(GR(N,1))*(CMSR1(1)+CMSR1(3))+GR(N,2)*(CMSR1(2
1)+CMSR1(4)))
      ALLE=XLEBK(1)-SLEEK*(YBRI(N)-RFR)
      IF(XLE=0, XLEB(1)) XLE=XLEB(1)
      XCP1=XLL-0.25*CHOKK1(N)
      XCP2=XLE-0.5J*CHOKK1(N)
      K1=K1M+KHC*UD*DETR1*(CMSR1(1))*XCP1*GR(N,1)+CMSR1(3)*XCP2*GR(N,1)+
      CMSR1(4)*XCP1*GR(N,2)+CMSR1(4)*XCP2*GR(N,2)
      TOTAL FORCE AND MOMENT
      KLZ=-KLZ
      SZ=-SZ
      FZDK=KLZ+SZ+SUURZ
      ZUIN=K1M+SI+SUURM+DUUM
      WRITE(6,106) NSTA,DU,WFS,RHO
      FORMAT(1,10,14,' FREE STREAM FORWARD SPEED ',F8.3,
100 1. SWAY VELOCITY ',F8.3,' WATER DENSITY ',F8.4)
      WRITE(6,206) (XIN(J),J=1,NSTA)
      FORMAT(1, XIN ', (2F10.5))
      WRITE(6,201) (KIN(J),J=1,NSTA)
      FORMAT(1, KIN ', (6F10.5))
      WRITE(6,202) ALPHA,ALR,DELTA(1)
      FORMAT(1, WAKE AND RUDDER ANGLES ', 3F20.5)
      WRITE(6,203) KSL(1),KRUD(1)
      FORMAT(1, SAIL AND RUDDER TIP RADII ',2F20.5)
      WRITE(6,107) KLZ,SZ,SUURZ,FZDK
      FORMAT(1, RUDDER SIDE FORCE ',F10.3,' FIN SIDE FORCE ',F10.3,' SUUR
107 ICE SIDE FORCE ',F10.3,' TOTAL SIDE FORCE ',F10.3)
      WRITE(6,108) K1,SI,SUURM,DUUM,ZDKM
      FORMAT(1, K1 MOMENT',F11.4,' FIN MOMENT',F11.4,' SOURCE MOMENT',
100 1F11.4,' DOUBLET MOMENT',F11.4,' TOTAL MOMENT',F11.4)
      WRITE(6,105)SIY,SIN
      FORMAT(1, HEAVE FORCE ON HULL',F11.4,' PITCH MOMENT',F11.4)
      STOP
      END

```

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P0M30049
P0M30050
P0M30051
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P0M30082

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C
FUNCTION SFIL(X,XI,YI,NC)
THIS WORKS ONLY FOR DECREASING VALUES OF X WITH INDEX
DIMENSION XI(2),YI(2)
DO 1 X,XI,X2,X3,Y1,Y2,Y3=Y1*(2.*X-(X2+X3))/((X1-X2)*(X1-X3))+Y2*(
12.*X-(X1+X3))/((X2-X1)*(X2-X3))+Y3*(2.*X-(X1+X2))/((X3-X1)*(X3-X2)
2)
IF(X.GT.XI(3)) GO TO 1
IF(X.LT.XI(NU-2)) GO TO 4
DO 10 3
1 Y=0YDX(A,XI(3),XI(2),XI(1),YI(3),YI(2),YI(1))
GO TO 99
2 Y=0YDX(A,XI(NU),XI(NU-1),XI(NU-2),YI(NU),YI(NU-1),YI(NU-2))
GO TO 99
3 DO 4 I=3,NC
I=1
IF(X.LE.XI(I).AND.X.GT.XI(I+1)) GO TO 3
CONTINUE
5 Y=0YDX(A,XI(I+1),XI(I),XI(I-1),YI(I+1),YI(I),YI(I-1))
99 SFIL=Y
RETURN
END

```

P0M30001
P0M30002
P0M30003
P0M30004
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P0M30011
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P0M30020
P0M30021

```

FUNCTION FILLI(X,XI,YI,NC)
  FITS ONLY FOR DECREASING VALUES OF X
  DIMENSION XI(2),YI(2)
  A(X1,X2,X3,X,Y1,Y2,Y3)=Y1*(X-X2)*(X-X3)/((X1-X2)*(X1-X3))+Y2*(X-X1
  1)*(X-X3)/((X2-X1)*(X2-X3))+Y3*(X-X1)*(X-X2)/((X3-X1)*(X3-X2))
  IF(X-XI(1))2,3,1
  Y=YI(1)
  GO TO 99
  1 Y=A(XI(3),XI(2),XI(1),X,YI(3),YI(2),YI(1))
  GO TO 99
  2 IF(X-XI(2))5,6,1
  3 Y=YI(2)
  GO TO 99
  4 DO 7 I=5,NC
  M=1
  IF(X-XI(1))7,9,8
  Y=YI(1)
  GO TO 99
  5 CONTINUE
  6 Y=A(XI(M-2),XI(M-1),XI(M),X,YI(M-2),YI(M-1),YI(M))
  FILLI=Y
  RETURN
END

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PGM30001
 PGM30002
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```

SUBROUTINE GLSQ(A,X,IL,N,M,ALPHA,E1,E2)
GLSQ SUBROUTINE *****
DIMENSION A( 32, 21),X(1),IL(1)
MM=M+1
LL=1
DO 00 J=1,MM
IL(J)=0
I=1
DO 5 K=1,MM
II=I+1
DO 4 J=II,N
IF (ABS(A(J,K))-E1)4,4,6
I1=SQRT((A(J,K))**2+(A(I,K))**2)
S=A(J,K)/I1
C=A(I,K)/I1
DO 5 L=K,MM
I2=C*A(I,L)+S*A(J,L)
A(J,L)=-S*A(I,L)+C*A(J,L)
A(I,L)=I2
LL=LL+1
CONTINUE
IF (ABS(A(I,K))-E2)3,3,8
IL(K)=I
I=I+1
GO TO INDE
X(MM)=-1.0
II=M
DO 55 I=1,M
X(I)=0.0
DO 50 J=1,M
IF (IL(II))50,30,31
S=0.0
LL=II+1
I=IL(II)
DO 32 K=LL,MM
S=S+A(I,K)*X(K)

```

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0454 PGM30001
0453 PGM30002
0455 PGM30003
0456 PGM30004
0457 PGM30005
0458 PGM30006
0459 PGM30007
0460 PGM30008
0461 PGM30009
0462 PGM30010
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0487 PGM30035
0488 PGM30036
PAGE 22

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50 X(II)=-S/A(I,II)
51 II=II-1
51 IF(IL(MM))50,51,50
51 ALPHA=0.0
51 GO TO 52
51 I=IL(MM)
51 ALPHA=A(I,MM)
52 RETURN
END

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0489 PGM30037
0490 PGM30038
0491 PGM30039
0492 PGM30040
0493 PGM30041
0494 PGM30042
0495 PGM30043
0496 PGM30044
0497 PGM30045

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PGM3J001
 PGM3J002
 PGM3J003
 PGM3J004
 PGM3J005
 PGM3J006
 PGM3J007
 PGM3J008
 PGM3J009
 PGM3J010
 PGM3J011
 PGM3J012
 PGM3J013

SAMPLE DATA DECK FOR NUMERICAL PREDICTION OF FORCES AND MOMENTS

C	1	2	3	4	5	6	7	8	9	10
13	14.772	2.606	2.0	.166	.000	-.333				
.833	.666	.500	.333	-1.166						
-.500	-.833	-1.000	-1.166	.147	.143	.136				
.000	.577	.110	.133	.143						
.124	.100	.086	.059	.023						
.414	.115	.141	-.153	-.242						
.172	-1.051	-1.051	-1.150	-1.150	0.0					

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C-324 C-354 C-376 W-385 W-311 W-217
 N-12 CIRC HIDE 10102490200-179900-161940-081440-070720-050510-040730-040340-032660-025110-016720-00364
 FRAILERS FOR UNIT MESSAGES AT BASILE, RI AND TIVOLI, RI U-2642 W-9806 -0-0017 -0-2294
 VEL AT CP X = -1.07 V J-003
 VEL AT SAME POINT DUE TO MAILERS UN ELATE= 1.0741 AND WAKL= 0-5501 DUE TO UNIT MADE 1.1
 VELOCITY AT SAME POINT DUE TO HULL= 3.73504

1	1	0-57260380
1	2	-0-559501944
1	3	0-04307384
1	4	-0-624054607
2	1	0-60171576
2	2	-0-655978212
2	3	0-10459769
2	4	-0-701655447
3	1	0-33100610
3	2	-0-752165166
3	3	1-34476515
3	4	-0-558740543
4	1	5-9232322
4	2	-0-560572756
4	3	-1-39094405
4	4	-0-431747728
5	1	0-35770015
5	2	-0-12009202
5	3	0-0446380
5	4	-0-016263063
6	1	0-1114185
6	2	-0-077626127
6	3	3-7666522
6	4	-0-152452654
7	1	3-01725134
7	2	-0-0020540701
7	3	1-11485707
7	4	-0-261757914
8	1	5-47426508
8	2	0-164218364
8	3	-1-46513624
8	4	-0-421196526
9	1	5-92160420
9	2	1-37450551
9	3	2-07247304
9	4	1-28567334
10	1	3-72471714
10	2	1-2410821
10	3	2-47001171
10	4	0-003512752
11	1	2-53007412
11	2	1-23001873
11	3	1-04642202
11	4	0-234513312
12	1	2-052669308
12	2	1-223245366
12	3	-1-30686516
12	4	-0-524190080
13	1	2-03132740
13	2	3-47222241
13	3	2-45050672
13	4	3-25080430
14	1	2-36217008
14	2	3-22205508

NO. OF STATIONS IS FREE STREAM FORWARD SPEED 14.772 SWG3 VISCALITY 2.0006 WATER DENSITY 1.0000
 XIN 0.83300 0.00000 0.00000 0.33300 0.10000 0.00000 0.00000 0.00000 0.00000
 -0.50000 -0.00000 -0.00000 -0.00000 -0.00000 -0.00000 -0.00000 -0.00000 -0.00000
 RIN 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000
 0.12400 0.10000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000 0.00000
 WAKE AND RUDDER ANGLES 0.17400 0.17400 0.17400 0.17400 0.17400 0.17400 0.17400 0.17400 0.17400
 SAIL AND RUDDER TIP ANGLE 0.41400 0.41400 0.41400 0.41400 0.41400 0.41400 0.41400 0.41400 0.41400
 RUDDER SILL FORCE 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000
 R1 MOMENT 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000 1.00000
 HEAVE FORCE ON HULL -0.00000 PITCH MOMENT 0.00000

1.00000 TOTAL SIDE FORCE 1.00000 TOTAL MOMENT 1.00000
 0.00000 TOTAL MOMENT 0.00000 TOTAL MOMENT 0.00000