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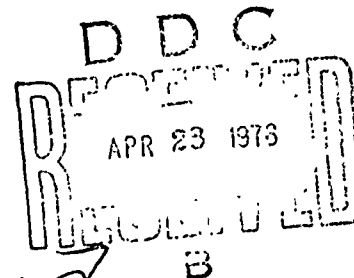
PREDICTION OF VERTICAL WATER-ENTRY FORCES ON OGIVES FROM CONE DATA

BY
J.L. Baldwin

15 AUGUST 1975

NAVAL SURFACE WEAPONS CENTER
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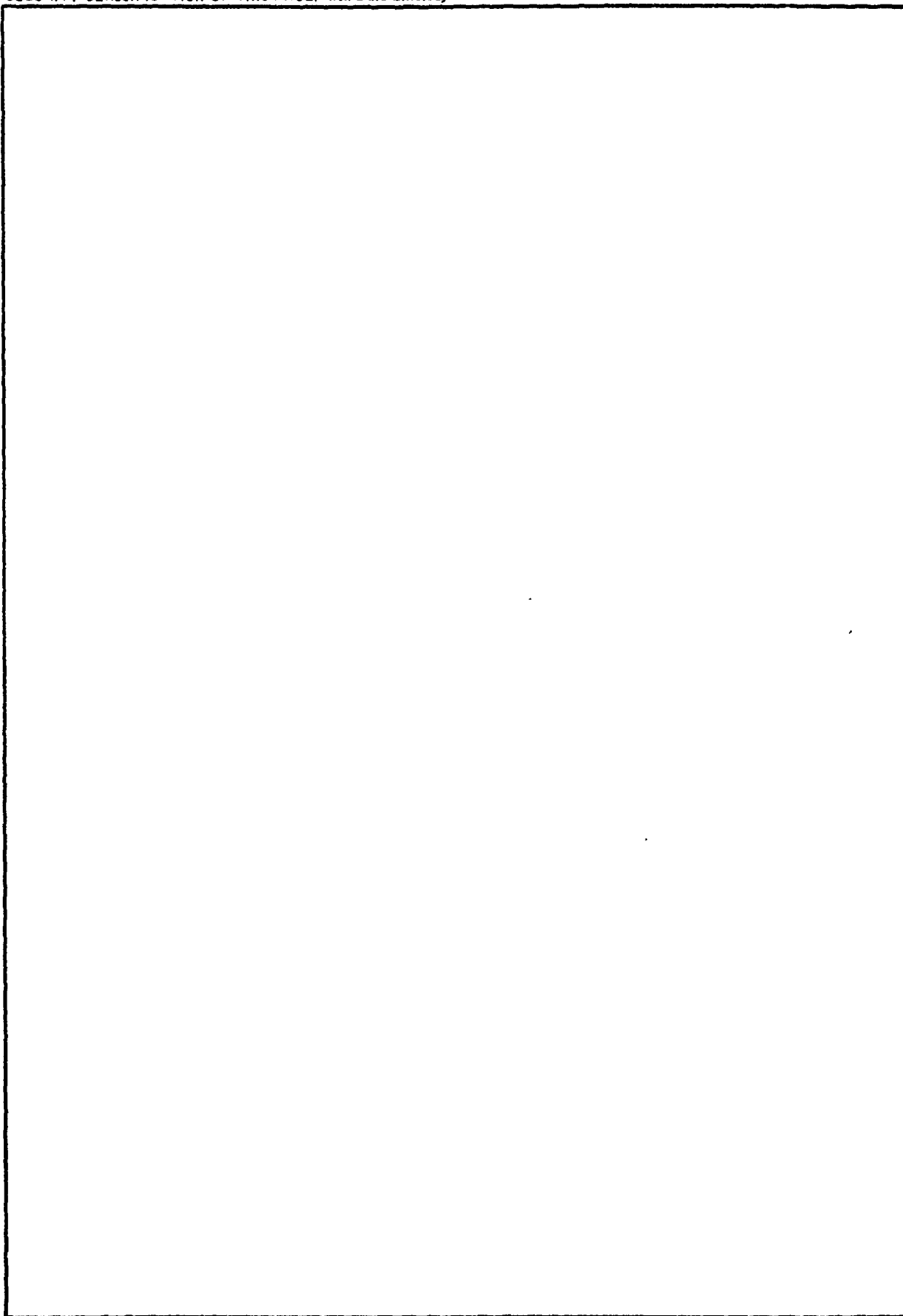
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PREDICTION OF VERTICAL WATER-ENTRY FORCES ON OGIVES FROM
CONE DATA

This report is a result of the continuing effort of the Naval Surface Weapons Center in the understanding of water-entry phenomena. The research reported herein was supported entirely by NAVSEA Code 03512 under task ORD-035B-501/URL2-301-003. The author would like to acknowledge Dr. Thomas Peirce of NAVSEA for his advice and interest in this program.

K. R. Enkenhus

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LIST OF SYMBOLS

A	Base area (πR^2)
B	Buoyancy force
C	A constant
C_d	Total drag coefficient
C_{ds}	Steady-state pressure drag coefficient
C_{dt}	Transient drag coefficient
C_f	Friction drag coefficient
D	Missile diameter
g	Gravitational constant
h	Penetration ratio $\left(\frac{\text{Sat maximum } C_d}{\text{cone length}} \right)$
K	Total mass constant
k	Added mass constant
L	Length
M	Missile mass
m	Added mass
R	Base radius
r	Local radius
S	Distance traveled after water contact
U	Instantaneous speed
U_o	Water contact speed
α	Total cone angle
β	Swept angle
ρ	Fluid mass density

INTRODUCTION

The two-dimensional case of water impact originally received the majority of attention due to its application to the landing of flying boats and seaplanes. Analytical prediction of the forces resulting from water entry began with a study of the landing impact of seaplane floats by von Karman in 1929 (Ref. (1)). In his paper, von Karman assumed that all the impact force was due to an increase in added mass, and that the amount of added mass was identical to one half of the added mass of a flat strip with width equal to the width of the float where intersected by the original water surface. The factor of one half was used because half of the assumed flow field was in air which contributed a negligible amount of mass to the total. In 1931, Wagner considered the effects of water-surface displacement during entry (Ref. (2)). Many subsequent investigators have also considered the problem of surface motion. Mayo in 1945 (Ref. (3)) and Pierson in 1951 (Ref. (4)) were still refining the prediction of seaplane impact. In addition to their specific contributions, they also include summaries of much of the previous work on the seaplane problem.

The prediction of the water-impact force experienced by axisymmetric bodies began as early as 1942 when Plesset applied von Karman's approach to the entry of torpedoes. In this method, the assumption was made that the added mass, "m" at any point during entry was the same as half that of a thin flat circular plate defined by the intersection of the nose and the original water surface. Half of the added mass for a disk is

$$m = 1.28 \rho \frac{\pi}{3} r^3 \quad (1)$$

where r is the radius and ρ is the water density. Differentiation with respect to the distance s yields:

$$\frac{dm}{ds} = (1.28) \rho \pi r^2 \frac{dr}{ds} \quad (2)$$

Also, the transient drag coefficient " C_{dt} " had been shown to be given by equation (3) (See Appendix A).

$$C_{dt} = \frac{2}{\rho A} \frac{dm}{ds} \quad (3)$$

Substituting equation (2) into equation (3) yields:

$$C_{dt} = \frac{2.56}{A} \pi r^2 \frac{dr}{ds} = 2.56 \left(\frac{r}{R} \right)^2 \frac{dr}{ds} \quad (4)$$

where R is the base radius and A is the base area.

For example, for a cone of total angle " α ":

$$r = S \tan \left(\frac{\alpha}{2} \right); C_{dt} = 2.56 \left(\tan \frac{\alpha}{2} \right)^3 \frac{S^2}{R^2} \text{ for } 0 \leq S \leq R \tan \frac{\alpha}{2}$$

$$C_{dt} = 0 \text{ for } S > R \tan \frac{\alpha}{2}$$

For a nose shape following a cubic relationship

$$r = CS^{1/3}; C_{dt} = \frac{.85C^3}{R^2} \text{ for } 0 \leq S \leq (R/C)^3$$

$$C_{dt} = 0 \text{ for } S > (R/C)^3$$

Then $C_{dt}(S)$ was easily calculated for any desired shape; however, as the theory was developed and experimental results accumulated, several substantial deficiencies became apparent. The predicted drag coefficients for 90 and 20 degree total angle cones are compared to experimental values (Ref. (5)) in Figure 1. Experimentally derived water-entry drag coefficients for cones are also given in Reference (6)).

The drag force predicted by this method becomes equal to zero when the base of the nose passes the undisturbed water surface and remains at zero for deeper penetration. In reality, the transient axial force continues below the water surface for about a nose length before the steady-state drag coefficient is obtained.

In most cases, the predicted drag coefficient is much larger than that obtained experimentally. This over-prediction seems to be caused by the assumption that the nose shape does not affect the added mass constant. Note that this method predicts that the final added mass and hence, the change in velocity during water entry is a function of only missile diameter.

Added mass constants for the various cones were derived theoretically by Shiftman and Spencer (Ref. (7)) and experimentally (Ref. (5)). In both cases, the mass constant was equal to zero at zero cone angle and increased monotonically with increasing cone angle.

The predicted value of rise time is normally longer than experimental value. This error results from the fact that the water surface usually does not remain flat during entry as assumed.

It is the purpose of this report to present a simple systematic method of applying the experimental results for cones given in reference (5) to the prediction of the total drag coefficient experienced during vertical water entry by ogives and other non-flat shapes.

BACKGROUND

The basic equations for axial forces encountered during water entry are included in Appendix A, which was copied from reference (5). The identification of ogives in this report follows the method used in reference (8), each ogive shape being uniquely defined by a cone angle and a swept angle. The ogive geometry section of reference (8) is also included in Appendix A.

PREDICTION METHOD

The proposed technique for estimating the total drag coefficient of axisymmetric bodies during vertical water entry consists of predicting a friction drag coefficient, a steady-state pressure drag coefficient and a transient drag coefficient, and then summing the three functions. Or

$$C_d(S) = C_f(S) + C_{ds}(S) + C_{dt}(S) \quad (5)$$

where each coefficient is for constant velocity entry and constant velocity cone entry data is used to calculate C_{ds} and C_{dt} . A computer program written in BASIC that performs these calculations is presented in Appendix B.

During the wetting process the water-entry force acts upon an area that extends above the original water surface as shown in Figure 2. The upper edge of this area is defined as the effective water surface and its distance from the tip of the missile is defined as the effective distance (S_{eff}). When the shape is a cone, self similar flow exists which implies that the distance from the tip to the original water surface divided by S_{eff} produces a constant. This ratio is approximated by the penetration ratio (h).

$$\frac{S \text{ at maximum } C_d}{\text{Cone Length}} = h \approx \frac{S}{S_{eff}} \quad (6)$$

The values of penetration ratio for total cone angles between ten degrees and 140 degrees was given in reference (5).

For more general shapes the value of S_{eff} is estimated by adding the incremented effect of each body angle as it passes the original water surface as shown by

$$S_{eff} = \int_0^S \frac{ds}{h} \quad (7)$$

where $h = f(\alpha)$ and $\alpha = g(S)$. Wetting is assumed completed when the effective distance equals the length of the nose.

The friction drag coefficient is estimated by: integrating a constant friction coefficient of .003 over the body to the effective water surface; taking the axial component and correcting the reference area to the base area of the body. This simple form was chosen so that the total drag coefficient would not be a function of entry speed and size. The addition of a varying friction coefficient could be made to the computer program subroutine beginning at line 1200 for increased accuracy for those cases that have significant friction drag.

The steady-state pressure drag coefficient is estimated to be the same as a cone with the same base diameter as the missile, undisturbed water surface diameter, and the same effective water surface body angle as the missile as illustrated in Figure 3. Addition of the friction drag coefficient to the steady-state pressure drag coefficient produces the steady-state drag coefficient which is computed up to the first maximum larger than .02. At all deeper depths the maximum value is used. When no maximum occurs on the body, the last value computed during wetting is defined as the maximum, and applied as above. In effect cavitation is predicted to begin at the depth which maximizes the steady-state drag coefficient.

The transient drag coefficient is also estimated from an equivalent cone. This cone has the same base diameter as the missile. The equivalent cone angle is the larger of the two body angles taken at the undisturbed water surface and at the effective water surface (see Figure 3). When wetting is completed the last value of transient drag coefficient and equivalent cone angle are retained in the program. At deeper depths the transient drag coefficient decays at the rate given in reference (5) for the equivalent cone angle. When the value of C_{dt} becomes smaller than two percent of C_{ds} , all calculations are terminated.

The calculating procedure may seem to be arbitrary and to violate accepted theory; however, it is presented as an engineering tool and during its development this method resulted in the best overall fit to available experimental data.

COMPARISON WITH EXPERIMENTS

The predicted values of the drag coefficient vs. depth function were computed for all axisymmetric shapes for which water-entry data were found. In addition, the steady-state drag coefficient was computed for all shapes for which the experimental value was found. Comparison of the calculated values with the experimental data showed that the predicted values would satisfy the accuracy requirements of most engineering applications.

The results of reference (5) were used to develop the numerical curve fits that form the core of the computing program. In order to determine the accuracy of the fitting operation, the maximum total drag coefficient, the penetration ratio and the final steady-state drag coefficient were computed for each cone angle reported in reference (5) and also for several blunter cones. The results of these calculations are compared to the experimental results in Table 1.

In the actual calculation procedure, the instantaneous drag coefficients are used; hence, it is of the utmost importance to have an accurate fit with regard to penetration ratio. An inspection of the results shows that the penetration ratio is calculated to within one percent of the experimental data for total cone angles between 20 and 140 degrees. Penetration ratios larger than unity were not allowed in the fitting process, which accounts for the large errors for the 10- and 15-degree cones. These large errors in penetration ratio for the ten and 15-degree cones caused the total maximum drag coefficient and the steady-state drag coefficient to also be lower than expected. In general, the values of calculated maximum total drag coefficients were larger than the experimental values. As indicated in reference (5), this is to be expected due to "roll off" that occurs at the peaks.

Upon discarding the values for maximum total drag coefficient for 30-degree cones, the remaining calculations and experimental values agree to within $\pm$ ten percent.

Steady-state drag coefficient values were found for ogives and paraboloids of revolution. The comparison of the experimental values at zero cavitation number and the calculated values are given in Table 2 for ogives and Table 3 for paraboloids.

TABLE 1
Comparison of Input and Curve Fit Results for Cones

Total Cone Angle	Cdmax Total Exp.	Penetration Ratio				Cds Assumed Fit	Cds Assumed Fit	Fit Assumed
		Fit	Exp.	Fit	Exp.			
10	.0574	.0556	1.037	.967	.932	.0595	.0538	.904
15	.0662	.0689	1.057	.952	.901	.0684	.0642	.939
20	.0873	.095	.943	.937	.994	.0866	.0867	1.001
30	.139	.168	.906	.913	1.008	.141	.149	1.056
45	.293	.314	.886	.883	.997	.257	.258	1.004
60	.466	.511	.857	.853	.995	.365	.358	.981
90	1.178	1.245	.802	.806	1.005	.497	.501	1.008
120	2.842	2.981	.763	.758	.993	.596	.597	1.002
140	5.547	5.322	.715	.719	1.006	.660	.664	1.006
160	--	10.3		.671			.740	
170	--	16.5		.640			.776	
175	--	23.3		.624			.788	
179	--	35.5		.610			.795	

TABLE 2
Comparison of Steady-State Drag Coefficient
for Tangent Ogives

Dia./ R (of cavity)	α/β	Exp. C_d (max)	Ref. No.	Calculated C_d (max)	Cal. Exp.	Cal. Exp. ($\sigma=.015$)
1/2	90/90	.29	9	.338	1.166	1.166
1	60/60	.195+.209 σ	10			
		.1515 +.595 σ	11			
		.22	12			
	Average	.189		.212	1.122	1.098
1.5	48.2/48.2	.14	13	.161	1.15	1.15
2	41.4/41.4	.114+.416 σ	14	.131	1.149	1.11
3	33.55/33.55	.0716+.625 σ	11			
		.13	12			
	Average	.1008		.0981	.973	.934
3.5	31/31	.09	13	.0875	.972	972
5	25.8/25.8	.056+.54 σ	11	.0672	1.200	1.048
7	21.8/21.8	.040+.55 σ	11	.0525	1.312	1.089

The calculated values of steady-state drag coefficients for ogives are about 12 percent higher than the average experimental values. It is interesting to note that a cavitation number change of $\sigma = .015$ would reduce this difference by approximately half. Any small error in cavitation number would not significantly reduce the differences for the blunter shapes.

The scant experimental data available pertaining to the steady-state drag coefficients of paraboloids of revolution are compared to calculated values in Table 3.

TABLE 3
Comparison of Steady-State Drag Coefficient
for Paraboloids of Revolution

L/D	Experimental	Ref.	Calculated		Cal./ Exp. Form
	$C_{ds} \text{ (max)}$ (Form)		$C_{ds} \text{ (max)}$ (Total)	$C_{ds} \text{ (max)}$ (Form)	
1	.125	15	.138	.130	1.040
3.33	.025	15	.0415	.0150	.600

Agreement is much better for the blunt paraboloid than for any of the blunt ogives. The discrepancy for the fine paraboloid may be due to the method used in reference (15) to adjust the experimental data to zero cavitation number. However, when the friction drag coefficient is added to the form drag coefficient to obtain the steady-state drag coefficient, the difference is less than 20 percent.

The principal purpose of this report is to propose a method for computing the total drag coefficient for pointed axisymmetric shapes during vertical water entry. In order to evaluate the accuracy of this technique, the total drag coefficient versus depth of penetration function was computed for all relevant shapes for which water-entry data was available. The results for cones have been previously discussed. The calculated values for ogives and cusps are graphically compared with the experimental data in Figures 4-7 and in Appendix C. The impact data for other shapes such as paraboloids were not found. However, current studies on such shapes should yield such data in the near future.

Numerical comparisons were easily possible with the four shapes reported in reference (8). The same instrumentation and type of models were used in this study as were used in the study of cones, which gave the basic empirical data for this method. Hence, if the method has any usefulness, one would expect good agreement between the calculated functions and the experimented results. The comparison of the values of maximum total drag coefficients and depth at which the drag coefficients become maximum are shown in Table 4.

It is seen that for cusps the calculated value for the maximum drag coefficient is about 12 percent higher than in the experimental results. This is somewhat greater than the error in reproducing the cone data. Perhaps the cusps have a larger percent roll-off than cones. The ogives are both computed about four percent low, which is interesting because the steady-state drag coefficients for these shapes were computed higher than the experimental values. In each of the four cases, the computed depth at which the maximum drag coefficient occurs agreed with the experimental data to within four percent. Figures 4, 5, 6 and 7 show the graphical comparison between the computed drag coefficients and the experimental results for these four shapes.

A comparison of computed maximum drag coefficients with experimental results for various tangent ogives is shown on Figure 8. The ogives reported in reference (8) were all secant ogives and were corrected to tangent ogives by the ratio of reference areas. As with the numerical comparison shown in Table 4, the computed values of the maximum total drag coefficients seem to be slightly lower than most of the experimental results. The sudden increase in computed drag coefficient values is caused by friction. This effect is accentuated because at a total cone angle of about 12 degrees and finer, the steady-state drag coefficient does not have a maximum value on the body. Hence, the friction drag is applied to the entire surface of the forebody. This result implies that the forebody becomes fully wetted for cone angles of 12 degrees and less. There has been no experimental verification of this prediction of wetting; therefore, the predicted drag values for very fine ogives should be used with care.

The graph of calculated maximum total drag coefficients for ogives and cusps with a 60-degree cone angle is shown on Figure 9. Also shown are the experimental values for four such shapes. As before, the calculated values for the cusp and cone are too large, while the calculated values for the ogives are probably too small.

TABLE 4

Comparison of Maximum Drag Coefficient
for Ogives and Cusps

Ogive	Cdmax Exp.	Cdmax Pred.	Pred. Exp.	Depth of Cdmax Exp.	Depth of Cdmax Pred.	Pred./exp.
60/43	.3027	.2806	.927	1.091	1.056	.968
88.4/321	.6854	.6516	.951	.978	.980	1.002
Cusp						
59.94/-43.4	1.481	1.658	1.120	2.286	2.268	.992
88.84/-31	2.366	2.670	1.128	1.295	1.281	.989

SUMMARY AND CONCLUSIONS

An empirical method of predicting the drag coefficient for vertical water entry of pointed shapes was developed. This method applied the experimental results previously obtained for the vertical water entry of cones of various angles to the geometric shape of the forebody of interest. Predicted drag coefficient functions agreed to within ± 25 percent of the available experimental results. The computer program, written in BASIC, was used to calculate the predicted drag coefficient function; it is given in Appendix B. For those who wish to write their own program, the method of prediction consists of the following steps:

1. Determine an expression for the instantaneous transitory drag coefficient for all cone angles.
2. Determine an expression for the instantaneous forebody pressure drag coefficient for all cone angles.
3. Determine an expression for penetration ratio for all cone angles.
4. Define the zone of action to lie between the original water surface and the effective water surface.
5. Divide the forebody into computing stations beginning at the tip.
6. For each station, compute the effective surface position by adding the effect of penetration ratio for all stations to the tip.
7. Calculate the friction drag from the tip to the effective surface associated with each station, and convert to coefficient form.
8. Calculate the pressure drag coefficient based upon the angle of the forebody at each effective surface position.
9. Add the friction drag coefficient and the forebody pressure drag coefficient to obtain the steady-state drag coefficient.
10. Use the value of the first maximum of the steady-state drag coefficient for all stations at greater depths.
11. At each station, calculate the transitory drag coefficient, using the maximum body angle in the zone of action.
12. Add the steady-state drag to the transitory drag coefficient to obtain the total drag coefficient at each station.
13. At stations for which the forebody base has been wetted, reduce the transitory drag coefficient proportionally to the cone of the last calculation angle.

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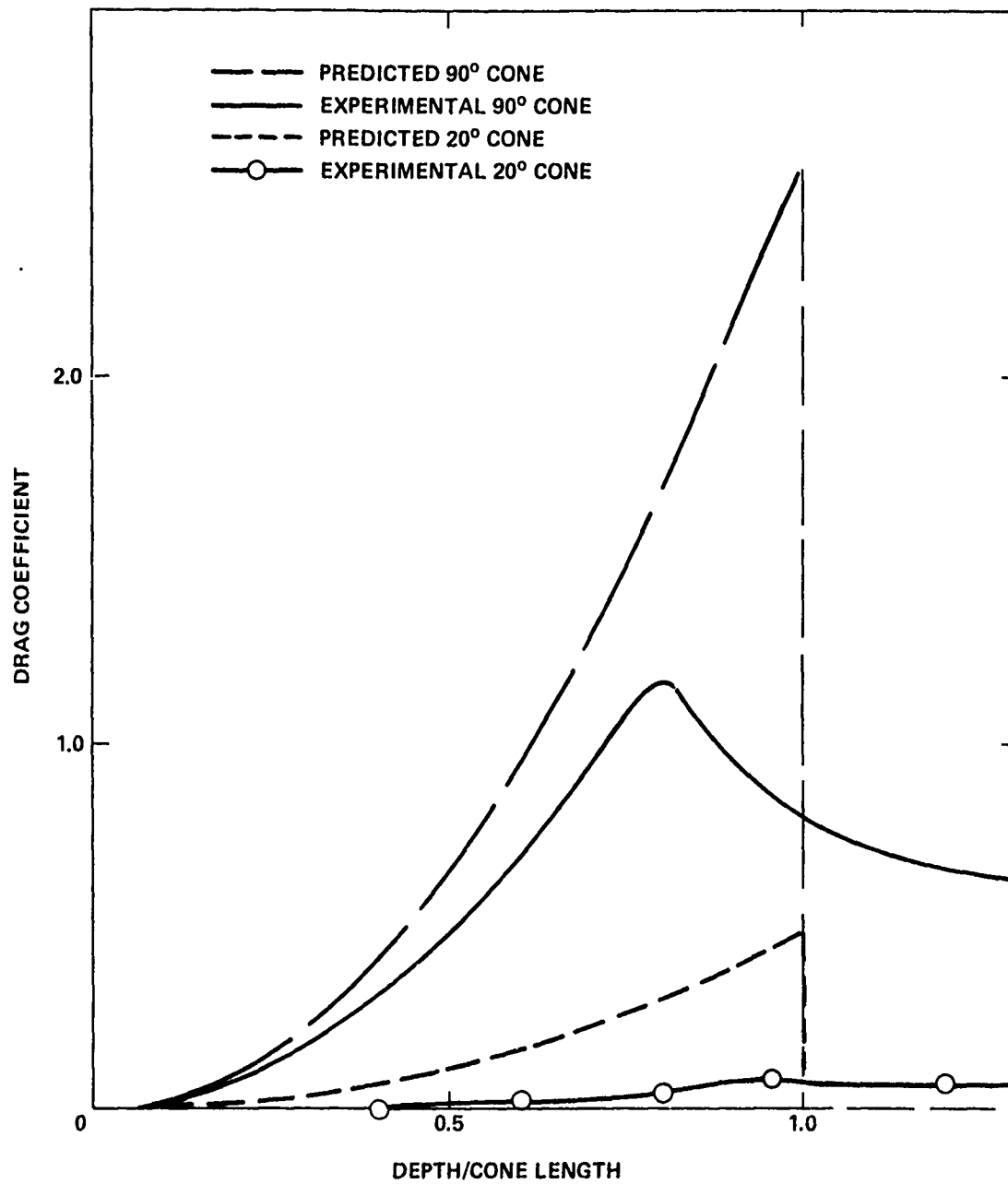


FIG. 1 EXPANDING DISK PREDICTIONS FOR 90 AND 20 DEGREE CONES
COMPARED TO EXPERIMENTAL RESULTS

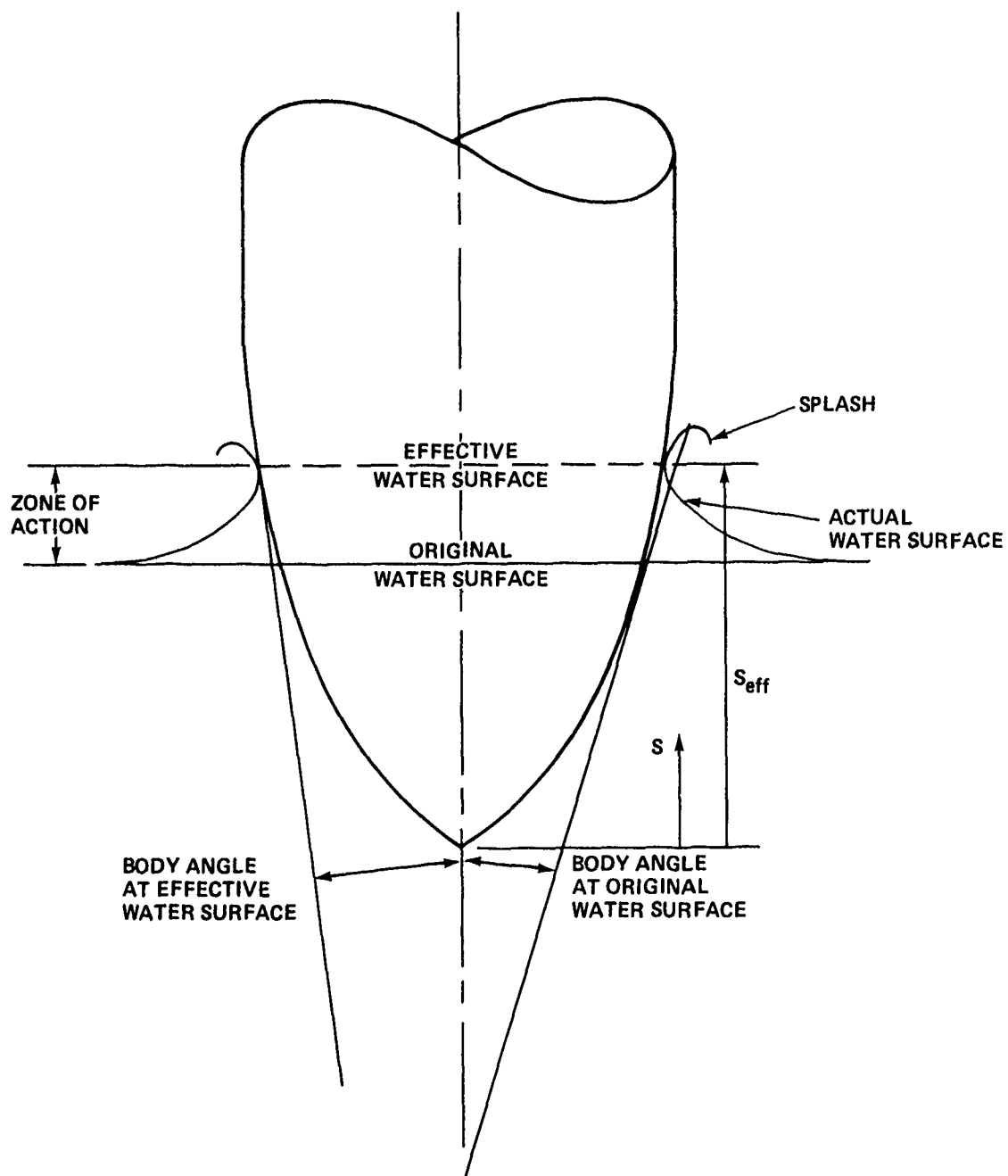


FIG. 2 WATER ENTRY NOMENCLATURE

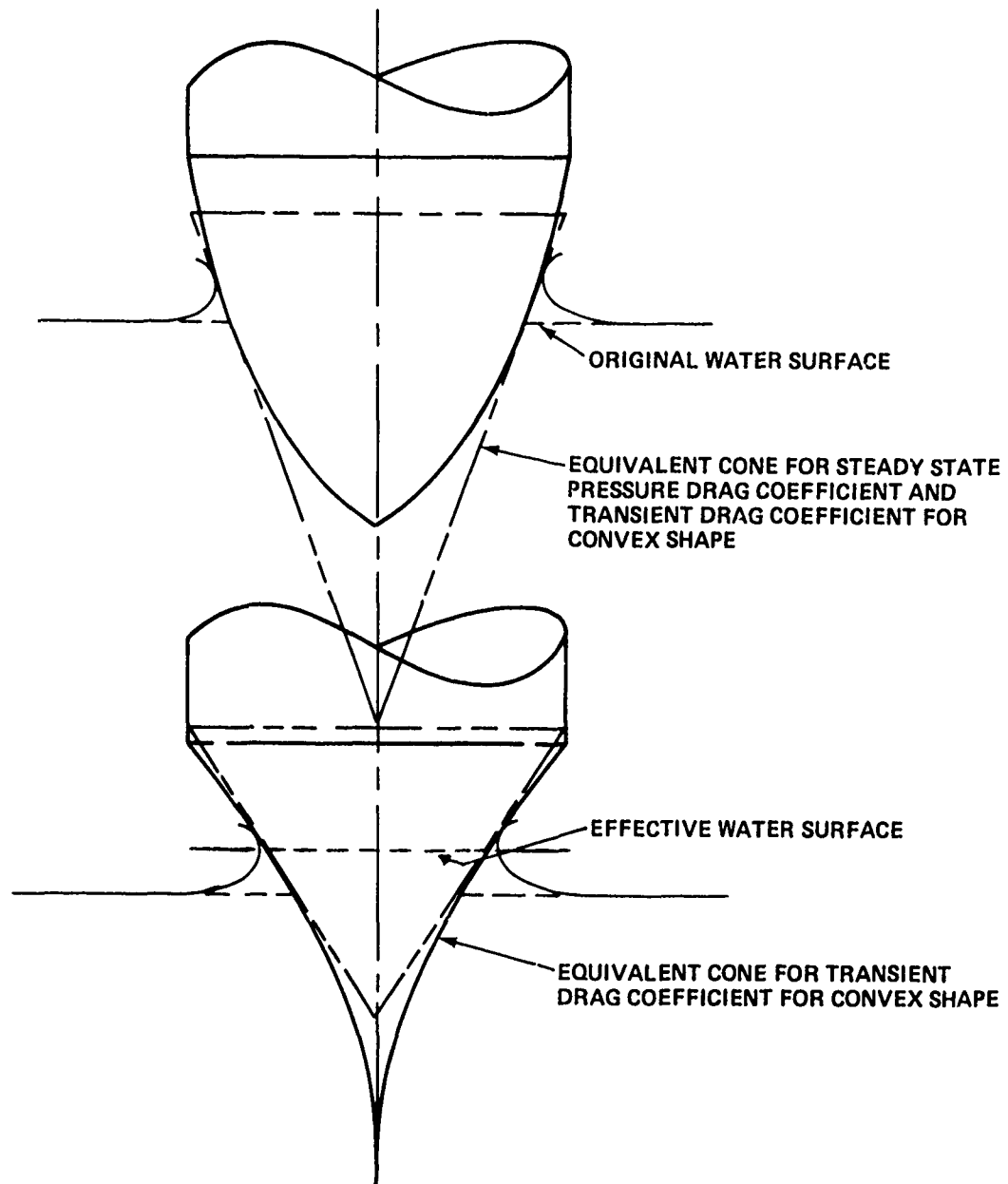


FIG. 3 EQUIVALENT CONES

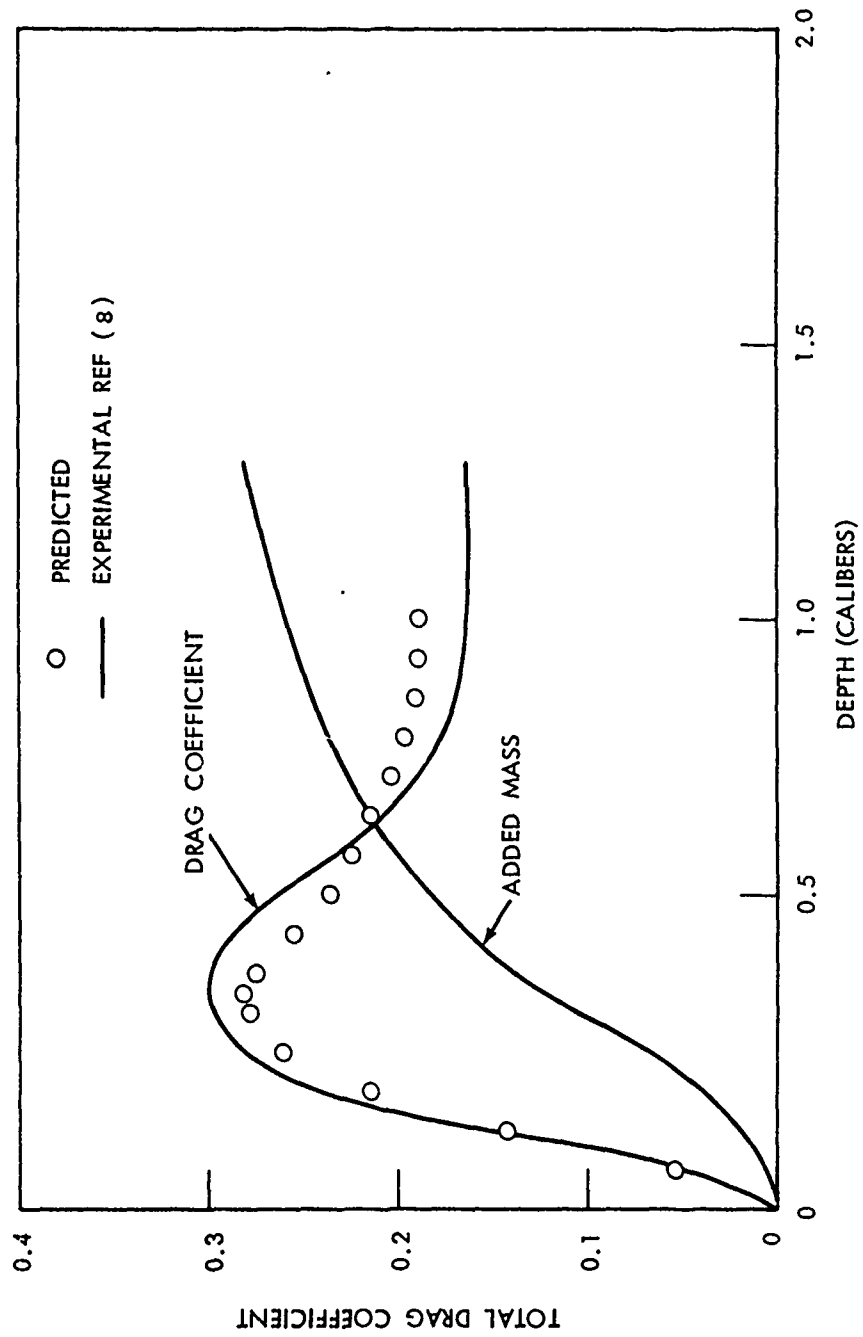


FIG. 4 TOTAL DRAG COEFFICIENT VS DEPTH FOR 60/43 OGIVES

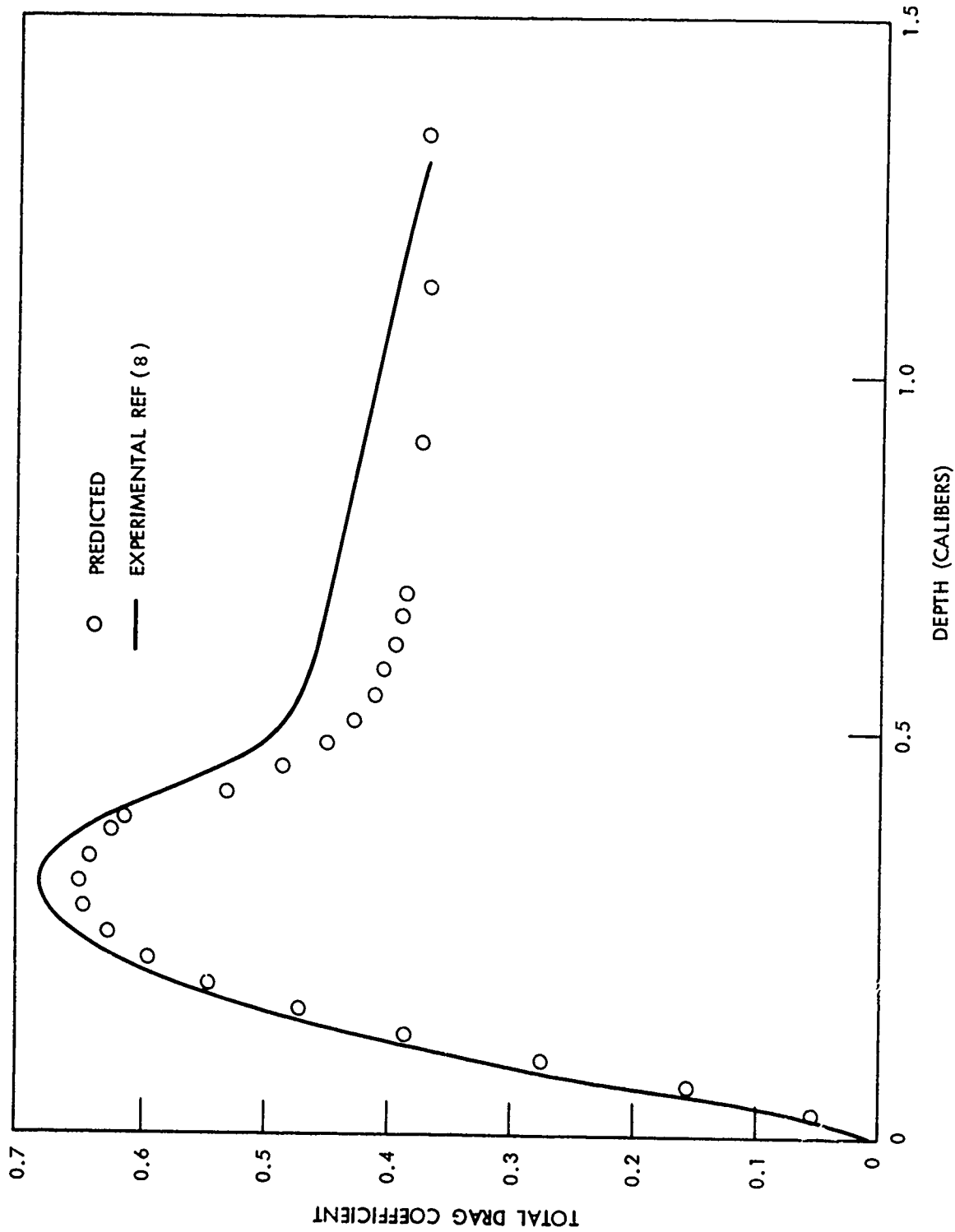


FIG. 5 TOTAL DRAG COEFFICIENT VS DEPTH FOR 88.4/32 OGIVES

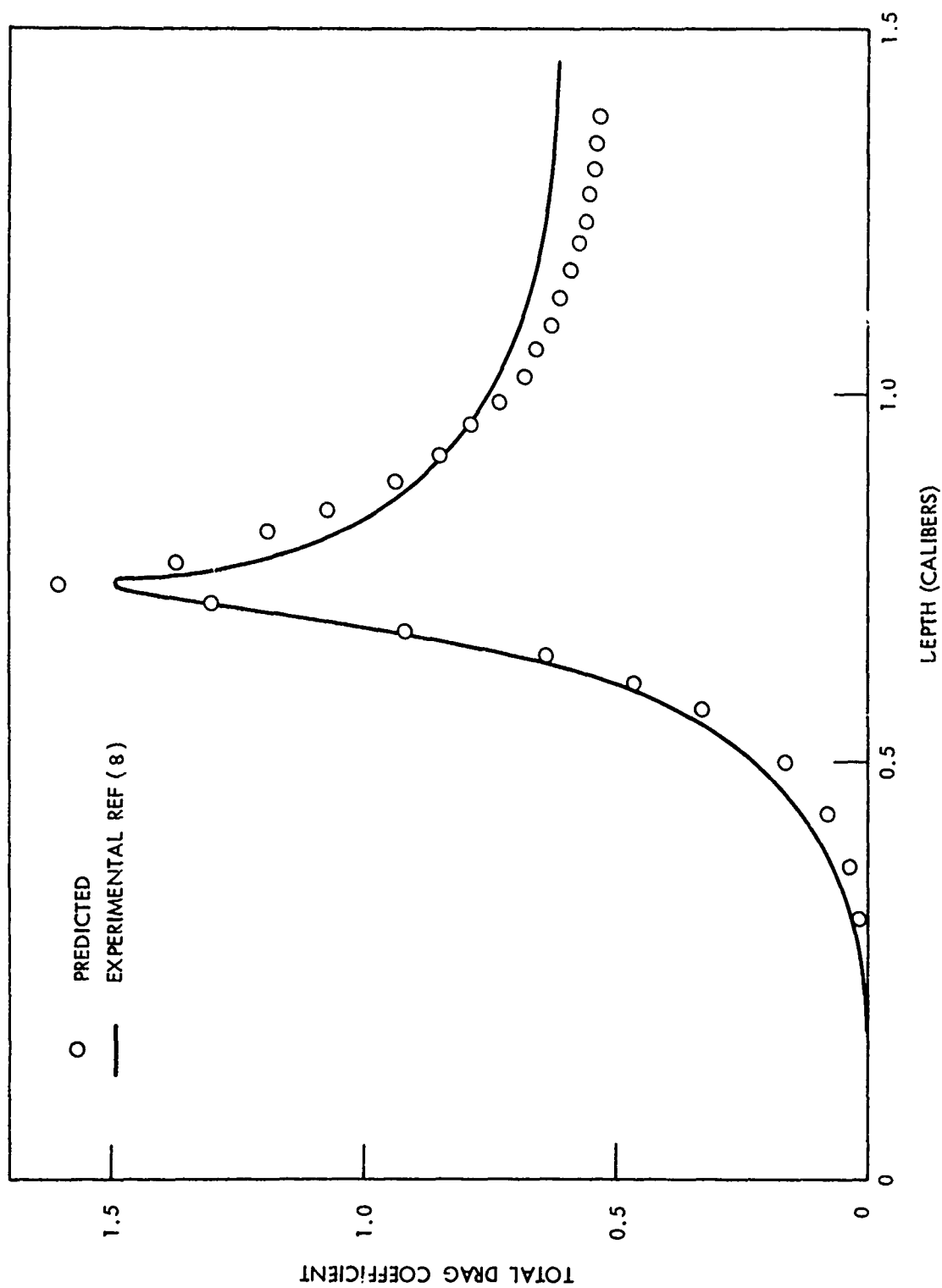


FIG. 6 TOTAL DRAG COEFFICIENT VS DEPTH FOR 59.94/-43.4 CUSPS

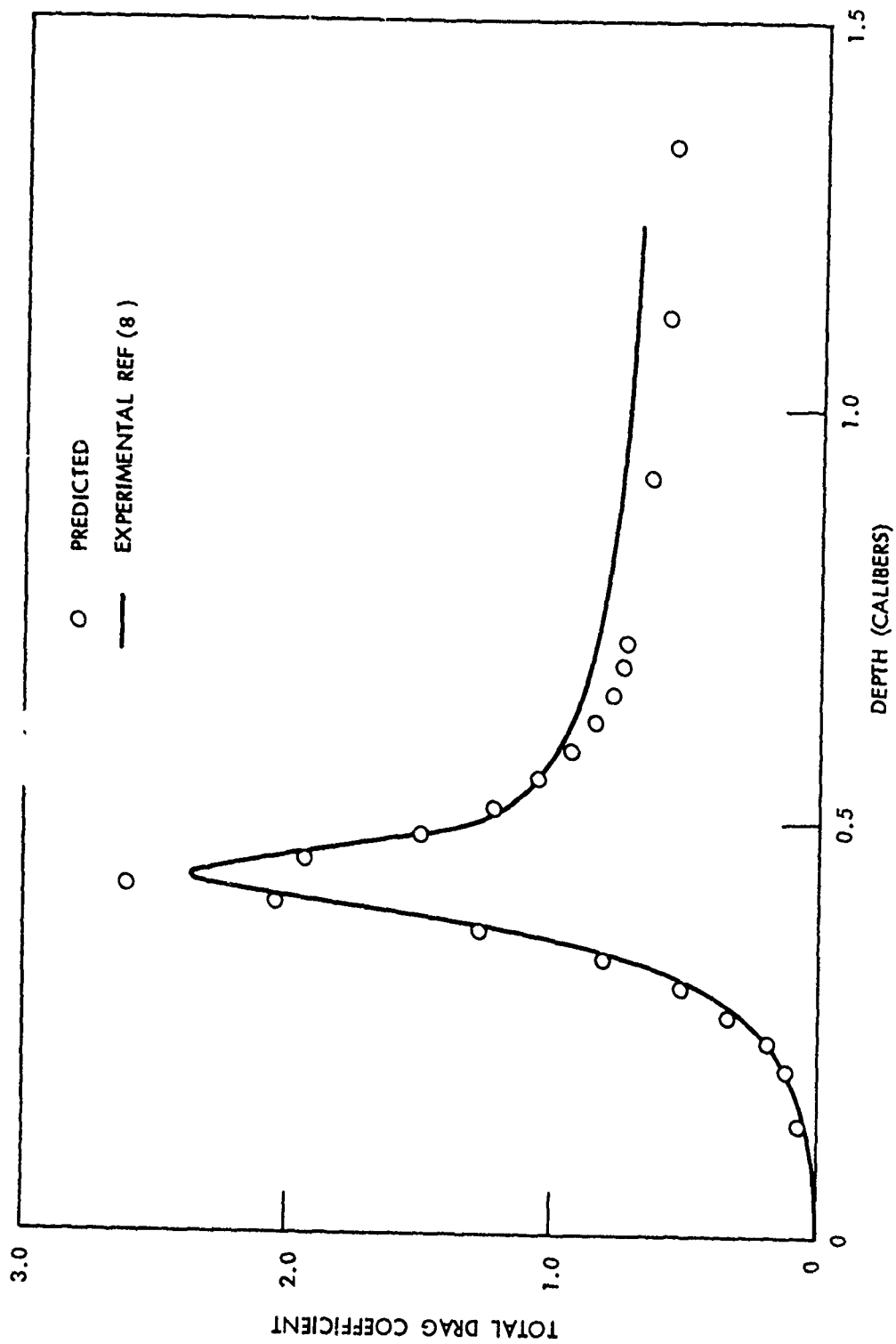


FIG. 7 TOTAL DRAG COEFFICIENT VS DEPTH FOR 88.94/-31

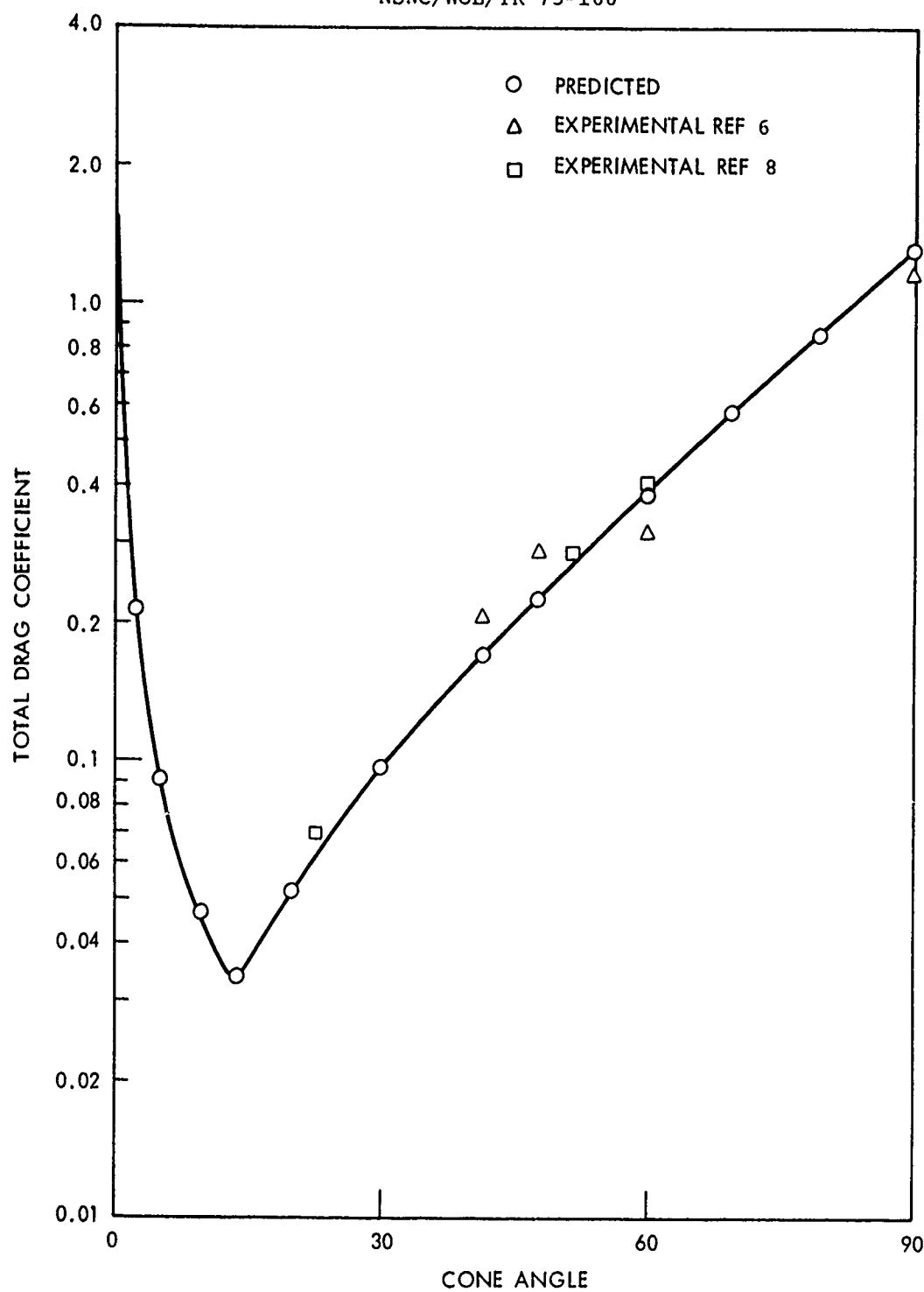
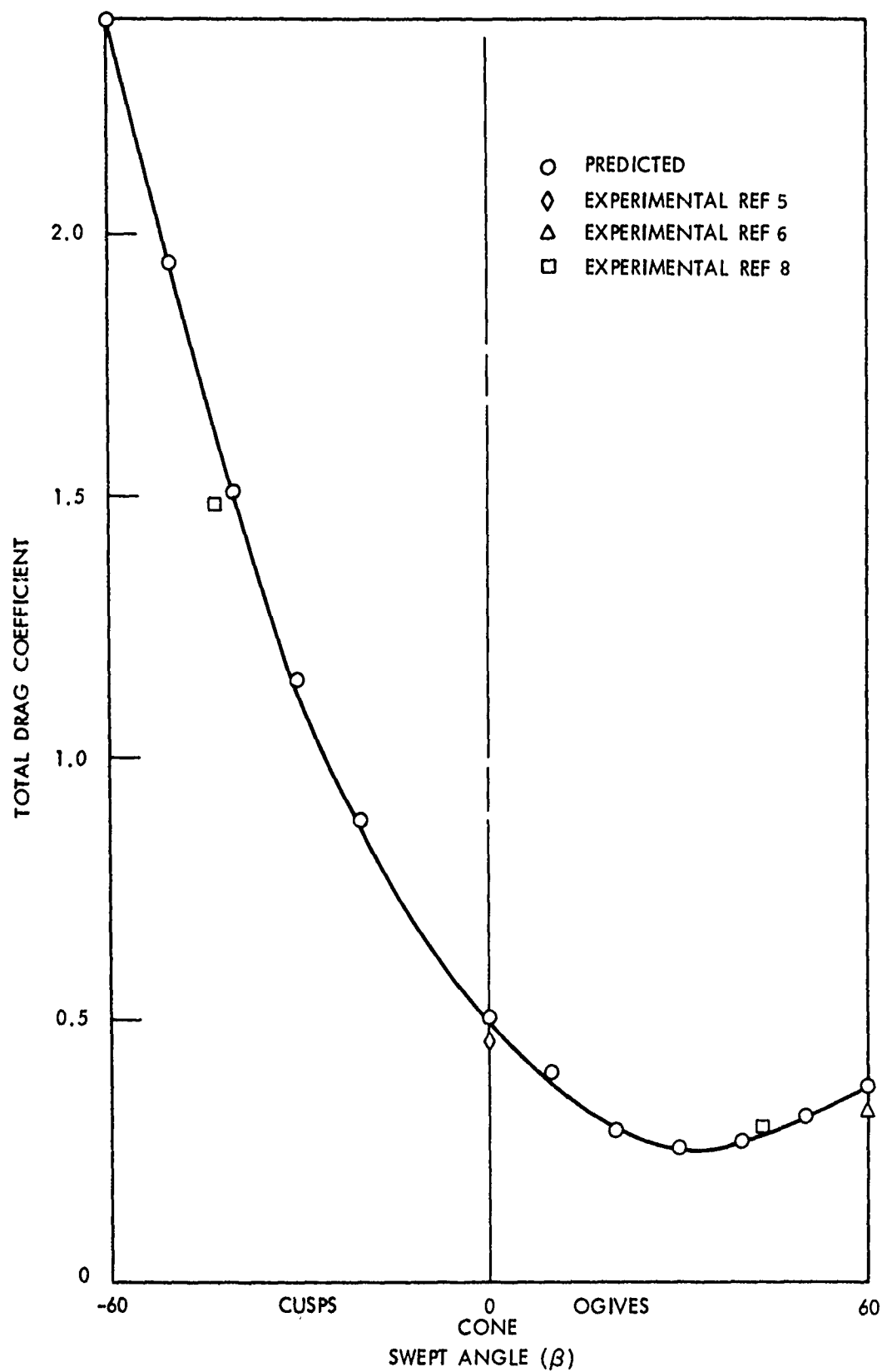


FIG. 8 TOTAL DRAG COEFFICIENT VS CONE ANGLE FOR TANGENT OGIVES

FIG. 9 TOTAL DRAG COEFFICIENT VS SWEPT ANGLE FOR $60/\beta$ OGIVES

APPENDIX A

BASIC EQUATION

The equation relating the momentum just before first contact with the momentum at some later time is given as Equation (1), and the force equation is given as Equation (2)

$$U_0 M - U(M+m) = \int_0^t B \, dt - Mgt + \frac{\rho}{2} \int_0^t C_{ds} AU^2 dt \quad (1)$$

$$- \frac{dU}{dt} (M+m) - U \frac{dm}{dt} = B - Mg + \frac{\rho}{2} C_{ds} AU^2 \quad (2)$$

Substituting $U^2 \frac{dm}{ds}$ for the second term of Equation (2) and rearranging terms gives:

$$- \frac{dU}{dt} (M+m) - B + Mg = \frac{dm}{ds} U^2 + \frac{\rho}{2} C_{ds} AU^2 \quad (3)$$

If the total drag coefficient is defined as

$$C_d = \frac{2}{\rho A} \frac{dm}{ds} + C_{ds} \quad (4)$$

then Equation (3) may be rewritten as

$$- \frac{dU}{dt} (M+m) - B + Mg = \frac{1}{2} \rho C_d AU^2 \quad (5)$$

The value of added mass (m) and $\frac{dm}{ds}$ can be calculated using Equation (3) and the total drag coefficient C_d was computed from Equation (4). However, a simpler procedure is to determine m from (1) and C_d from (5). In order to ensure an accurate evaluation of the integrals in Equation (1), the added mass (m) was also computed by integrating $\frac{dm}{ds}$ as given in Equation (4) using the computed value of C_d .

For the vertical entry of cones, the added mass should be given by

$$m = \rho k r^3 = \rho k (\tan \frac{\alpha}{2})^3 S^3 \quad (6)$$

Then

$$\frac{dm}{ds} = 3\rho k \left(\tan \frac{\alpha}{2}\right)^3 s^2 \quad (7)$$

Substituting (7) into (3) yields

$$-\frac{dU}{dt} (M+m) - B + Mg = (3\rho k \left(\tan \frac{\alpha}{2}\right)^3 s^2 + \frac{\rho}{2} C_{ds} A) U^2 \quad (8)$$

If the total mass constant (K) is defined as:

$$K = k + \frac{C_{ds}}{6} \left(\tan \frac{\alpha}{2}\right)^{-1} \pi \quad (9)$$

then (8) can be rewritten as

$$-\frac{dU}{dt} (M+m) - B + Mg = 3\rho K \left(\tan \frac{\alpha}{2}\right)^3 s^2 U^2 \quad (10)$$

This form of the equation was chosen to compare easily the experimental results with the theoretical results obtained in reference 3.

It should be noted that the total mass constant (K) is also a drag coefficient related to the area intersected by the original water surface. The relationship between the mass constant (K) and the total drag coefficient C_d is obtained by subtracting Equation (5) from Equation (10) and dividing out ρ and U^2 :

$$C_d = 6K \left(\tan \frac{\alpha}{2}\right)^3 s^2 (A)^{-1} \quad (11)$$

If the area used to relate C_d to drag force is the area intersected by the heaved water surface, then the instantaneous drag coefficient is given by

$$C_{d_i} = (6K \tan \frac{\alpha}{2}) (\pi)^{-1} \quad (12)$$

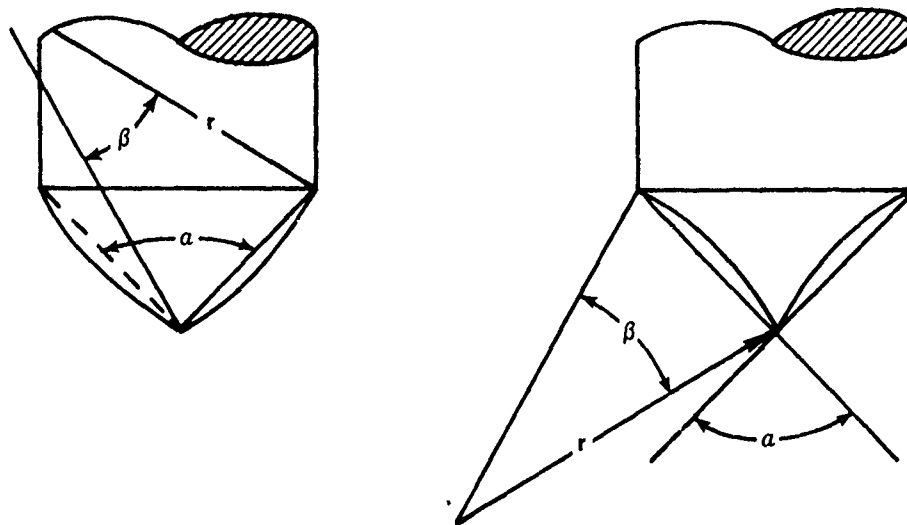
OGIVE GEOMETRY

There seems to be no standard method used in the literature to define ogive shapes. An ogive shape is a pointed body formed by rotating a circular arc. If the water-impact forces do indeed depend chiefly upon the geometry of the body, the shape will have to be defined more precisely. The method chosen for use in this report uses two angles and one reference length. One angle is the tip angle of a cone of the same length and base as the ogive and the other angle is the arc length (swept angle) used to generate the ogive section. The base diameter is used as the reference length. The tip angle, base angle, slopes, and radii are all easily computed for both normal ogives and inverse ogives (cusps), as shown in Figure A-1. Note that in Figure A-1 the inclusion of a minus sign with the swept angle β changes the ogive to the cusp. An inspection of the computer program shows a continuing similarity in the geometric equations. The only difference, if any, is a minus sign for the cusp shape. The swept radius r is one parameter that has the same equation for both shapes.

$$r = r/2 \sin (\alpha/2) \sin (\beta/2)$$

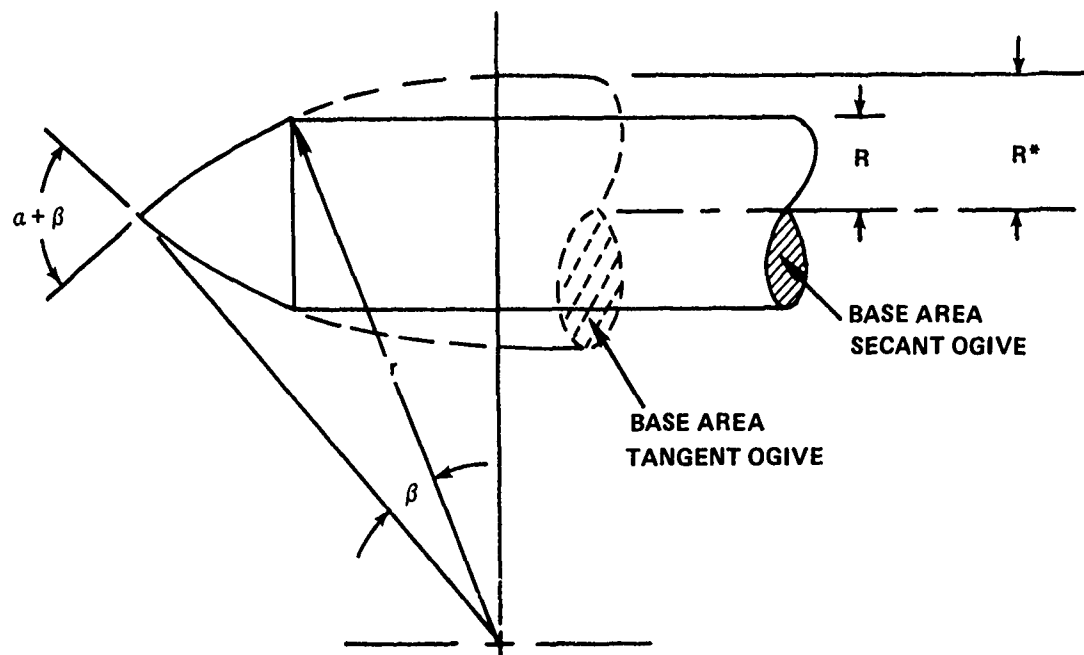
In the comparison of the results for different ogives, it is often possible to relate the forces to the base area of a tangent ogive. A method for computing the ratio between the base area of a secant ogive and corresponding tangent ogive is shown in Figure A-2. Also, the nomenclature must be such that the actual tip angle remains constant.

In this report it is assumed that the tangent ogive is a limit of this geometric family of shapes. This implies that the swept angle may not be larger than the cone angle. It is also defined that the limit case for cusps is when the swept angle is equal to the cone angle. For cone angles of 90 degrees or more, this definition merely states that the base angle cannot be negative.



OGIVE	PROPERTY	CUSP
a/β	"NAME"	a/β
$a + \beta$	TIP ANGLE	$a - \beta$
$a - \beta$	BASE ANGLE	$a + \beta$

FIG. A-1 OGIVE GEOMETRY



$$\text{BASE AREA RATIO} = (R/R^*)^2$$

$$(R/R^*)^2 = \left(\frac{2 \sin a/2 \sin \beta/2}{1 - \cos \frac{a + \beta}{2}} \right)^2$$

FIG. A-2 BASE AREA COMPARISON OF SECANT AND TANGENT OGIVES

APPENDIX B

This section consists of a listing and sample run of the computer program written in BASIC used to calculate the predicted values of the drag coefficient given in this report. By the selection of suitable constants, it is possible to obtain predicted drag coefficient versus depth functions for shapes defined by the polynomial

$$R = K1 \cdot D + K2 \cdot D^2 + K3 \cdot D^3 + K4 \cdot D^4 + K5 \cdot D^5 + K6 \cdot D^3$$

or for ogives or cusps. It is also possible to obtain the printout either at the point of the first maximum of the total drag coefficient or at uniformly spaced distance intervals. The format used in this program was chosen in an effort to allow ready understanding, easy modification, and maximum flexibility.

The major sections of the program are:

Lines 100-180 consist of an alphabetical list of constants needed to operate the program.

Lines 200-550 contain the commands that establish the order of calculations. Also the penetration ratio is calculated on line 250.

Line 554 contains the print command used with the condensed print format.

Lines 700-790 contain the calculation of the radius and angle for shapes defined by the polynomial.

Lines 900-990 contain the calculation of the radius and angle for ogives and cusps. The headings used with these shapes are also printed.

Lines 1200-1290 contain the calculations of the skin friction drag coefficient. A constant value of .003 is assumed for the friction drag coefficient.

Lines 1400-1490 contain the calculations of steady-state pressure drag coefficient.

Lines 1600-1690 contain the calculations of the transitory drag coefficient.

Lines 2000-2290 contain the calculations of transitory and total drag coefficients provided the base of the forebody shape is submerged.

Lines 8000-8100 contain the print statements used with the long print format.

In order to run a problem it is necessary to enter the following constants. Note: all distances must have the same units and determine the unit used in output.

1. Line 115 D1 = distance between computational step.
In long print format every tenth computational step is printed
2. Lines 135-151 K1-K6 = coefficients of polynomial
Lines 152 K1 = total cone angle in degrees of ogive or cusp
Lines 153 K8 = swept angle in degrees of ogive or cusp
3. Lines 154 P = -1 for cusp
P = 1 for ogive
P = 2 for polynomial
4. Line 157 P4 = 0 printout of first maximum only
P4 = 1 full printout
5. Line 160 R 1 = radius of base

The various constants are defined as:

- A. temporary name for an angle
- A. body angle at the original water surface
- A3 previous A4
- A4 body angle at the effective surface
- C C6 when $R4=R1$
- C1 incremental friction drag coefficient
- C2 friction drag coefficient
- C3 previous C4
- C4 steady-state pressure drag coefficient
- C5 previous C6
- C6 transitory drag coefficient
- C7 previous C8
- C8 total drag coefficient ($C2 + C4 + C6$)
- D1 distance increment
- D2 distance from tip to original water surface
- D3 $D1/H$
- D4 distance from tip to effective water surface
- H penetration ratio
- K1-K6 constants that determine the polynomially defined shapes
- K7 cone angle of ogive or cusp
- K8 swept angle of ogive or cusp

M2 distance from tip to base for ogive or cusp
 P shape control (input on line 154)
 P1 print control full printout only
 P3 heading control (heading printed once only)
 P4 print control (input on line 157)
 P5 control that stops for first maximum
 Q radius of curvature of ogive or cusp
 R1 base radius
 R2 body radius at original water surface
 R3 previous P4
 R4 body radius at effective water surface
 S temporary area
 S1 base area
 S2 cross section area at original water surface
 S4 cross section area at effective water surface
 S6 surface area increment
 Z dummy

NSWC/WOL/TR 75-100

LIST Fig. B-1 - Listing and Sample Run of Computer
Program Written in BASIC

75/07/10. 17.19.18.
PROGRAM OBJ464

```

101 LET A4=.0001
102 LET C2=0
105 LET C4=0
110 LET C8=0
115 LET D1=.01
120 LET D2=0
125 LET D4=0
130 LET D5=0
134 LET K=1
135 LET K1=0
140 LET K2=0
145 LET K3=0
149 LET K4=0
150 LET K5=.38
151 LET K6=0
152 LET K7=60
153 LET K8=43
154 LET P=1
155 LET P1=-1
156 LET P3=1
157 LET P4=2
158 LET P5=0
160 LET R1=1
165 LET R3=0
166 LET R4=0
170 LET S1=3.14159*R1*2
180 LET Z=0
200 LET P1=P1+1
209 IF P4=0 THEN 220
210 IF P1<9.5 THEN 220
215 GO SUB 8000
220 LET D2=D2+D1
230 LET D=D2
235 IF P<1.5 THEN 245
240 GO SUB 700
241 REM SUB 900 IS FOR OGIVES
243 GO TO 250
245 GO SUB 900
250 LET H=1-.6905E-2*A+8.745E-5*A*2-6.567E-7*A*3
260 LET A2=A
270 LET R2=R
280 LET D3=D1/H
290 LET D4=D4+D3
300 LET D=D4
310 LET R3=R4
320 LET A3=A4
325 IF P<1.5 THEN 335
330 GO SUB 700

```

CONT.

NSWC/WOL/TR 75-100

```

332  G0 T0 340
335  G0 SUB 900
340  LET A4=A
350  LET R4=R
360  IF R4>R1 THEN 2000
370  IF A4<.1 THEN 2000
380  G0 SUB 1200
385  G0 SUB 1400
390  IF A4>A2 THEN 420
400  LET A=A2
410  G0 T0 450
420  LET A=A4
450  G0 SUB 1600
460  IF P5>.5 THEN 470
461  LET C5=C2+C4
462  IF C3>C5 THEN 465
463  LET C3=C5
464  G0 T0 470
465  LET P5=1
470  LET C7=C8
480  LET C8=C3+C6
490  IF C8<.02 THEN 200
500  G0 T0 510
510  IF C3<C4 THEN 530
520  LET C4=C3
530  LET C8=C3+C6
535  IF P4=0 THEN 540
536  G0 T0 550
540  IF C7<C8 THEN 550
544  PRINT D2,C8,C3,A2,D4
546  G0 T0 9999
550  G0 T0 200
700  LET R=K1*D+K2*D2+K3*D3+K4*D4
705  LET R=R+K5*D1.5+K6*D1.3
710  LET A=K1+2*K2*D+3*K3*D2+4*K4*D3
715  LET A=A+.5*K5*D(-.5)+.3*K6*D(-.7)
720  LET A=57.3*ATN(A)
760  IF P3>1 THEN 790
762  PRINT "SHAPE GIVEN BY POLYNOMIAL"
763  PRINT "POLY CONST ="K1,K2,K3
764  PRINT K4,K5,K6
765  PRINT
766  PRINT "BODY RADIUS="R1
767  PRINT
768  PRINT "DIST","TOTAL CD","CDS","ANGLE","EFF DIST"
785  LET P3=2
790  RETURN
900  IF P3=1 THEN 930
905  LET U=.5*K7/57.3
910  LET V=.5*K8/57.3
920  LET Q=R1/(COS(U-V)-COS(U+V))
930  LET X=Q*SIN(U+P*V)-P*D

```

CONT.

```

940 LET Y=SQR(G+2-X+2)
950 LET A=ATN(X/Y)
960 LET R=P*Q*(COS(A)-COS(U+P*V))
970 LET A=57.3*A
971 IF P3>1 THEN 990
972 IF P>-.5 THEN 975
973 PRINT"THE SHAPE IS A CUSP"
974 GO TO 976
975 PRINT "THE SHAPE IS AN OGIVE"
976 PRINT
977 PRINT "BODY RADIUS="R1
978 PRINT "CONE ANGLE ="K7
979 PRINT "SWEPT ANGLE ="K8
980 PRINT "RAD OF CURV ="Q
981 PRINT
982 PRINT"DIST","TOTAL CD","CDS","ANGLE","EFF DIST"
988 LET P3=2
990 RETURN
1200 LET S6=3.14159*(R3+R4)*D3
1210 LET C1=.003*S6/S1
1220 LET C2=C2+C1
1290 RETURN
1400 LET C3=C4
1410 LET C4=3E-3-1.36E-3*A+1.076E-3*A+2-3.04E-5*A+3+3.415E-7*A+4
1420 LET C4=C4-1.362E-9*A+5
1430 LET S=3.14159*R2+2
1440 LET C4=C4*S/S1
1450 LET C4=C4/H+2
1490 LET C9=C4
1600 LET C6=1.76E-2+1.093E-2*A-8.663E-4*A+2+3.912E-5*A+3
1610 LET C6=C6-2.556E-7*A+4
1620 LET C6=C6*TAN((A-3)/57.3)
1630 LET C6=C6*S/S1
1690 RETURN
2000 PRINT
2005 LET D2=D2-D1
2010 GO SUB 8000
2015 IF P4<.5 THEN 9000
2020 LET A=A2
2030 IF P>.5 THEN 2050
2040 LET A=A4
2050 LET R=R1
2060 LET C=C6
2065 LET D=D2
2067 LET T9=A/57.3
2068 LET T8=TAN(T9)
2070 LET M2=D2
2080 LET T2=10*D1
2200 LET B4=.661*T2/M2
2210 LET A9=.3/(1+(T2/R1))+2+EXP(-9*B4)
2220 LET S6=C*(1/1.3)*A9
2230 LET C8=C3+S6
2240 LET D2=T2+D

```

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2250 PRINT D2,C8
2260 LET T2=T2+10*D1
2262 IF T2<.8*D THEN 2270
2265 LET T2=T2+50*D1
2270 IF S6<.02*C3 THEN 9000
2290 G0 T0 2200
2990 G0 T0 9000
8000 PRINT D2,C8,C3,A2,D4
8050 LET P1=0
8060 LET Z=Z+1
8065 IF Z<4.5 THEN 8100
8070 PRINT
8075 LET Z=0
8100 RETURN
9000 LET Z=Z
9999 END
READY.

BYE

464 LOG OFF. 17.26.52.
464 CP 1.518 SEC.

NSWC/WOL/TR 75-100

RUN

75/07/10. 17.16.26.
PROGRAM OBJ464

THE SHAPE IS AN OGIVE

BODY RADIUS= 1
CONE ANGLE = 60
SWEPT ANGLE = 43
RAD OF CURV = 2.72888

DIST	TOTAL CD	CDS	ANGLE	EFF DIST
.1	3.33095E-2	1.15109E-2	48.2419	.126227
.2	9.71428E-2	.038293	45.1794	.250941
.3	.161628	7.17684E-2	42.2736	.37427
.4	.214612	.106033	39.4961	.496305
.5	.252057	.136926	36.8255	.617108
.6	.273642	.161581	34.245	.736721
.7	.280685	.178187	31.7414	.855173
.8	.275171	.185841	29.3037	.972478
.9	.261225	.186357	26.9229	1.08864
1.	.247124	.186357	24.5914	1.20366
1.1	.234417	.186357	22.3025	1.31754
1.2	.223612	.186357	20.0506	1.43025
1.3	.21481	.186357	17.8305	1.54178
1.4	.207832	.186357	15.6378	1.6521
1.47	.203852	.186357	13.9005	1.73949
1.57	.198672			
1.67	.195151			
1.77	.192743			
1.87	.191084			
1.97	.189931			

CP 1.099 SECS.

RUN COMPLETE.

APPENDIX C

This section consists of graphs which compare the experimental and predicted values of total drag coefficient.

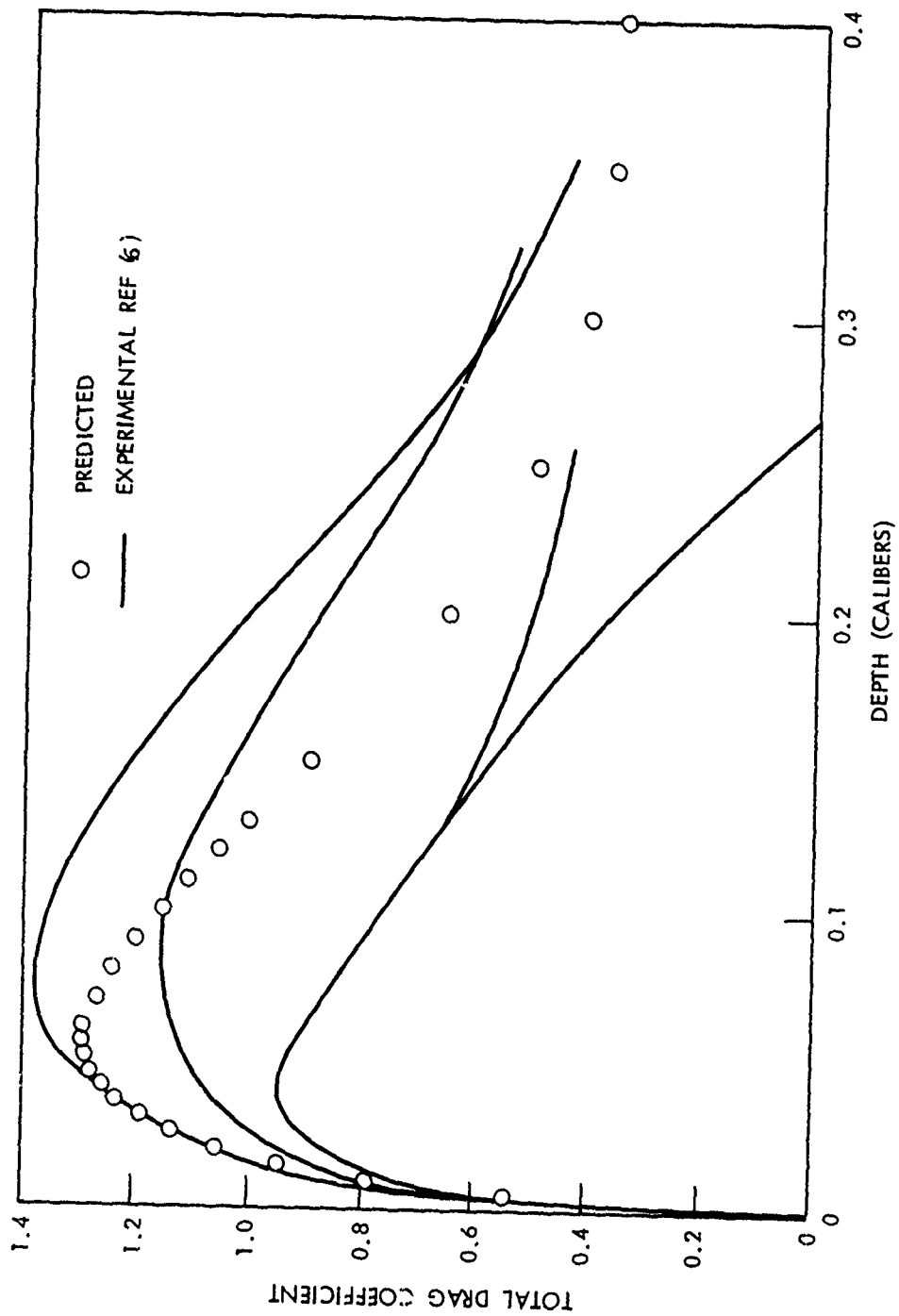


FIG. C-1 TOTAL DRAG COEFFICIENT VS DEPTH FOR 90/90 OGIVES (HEMISPHERES)

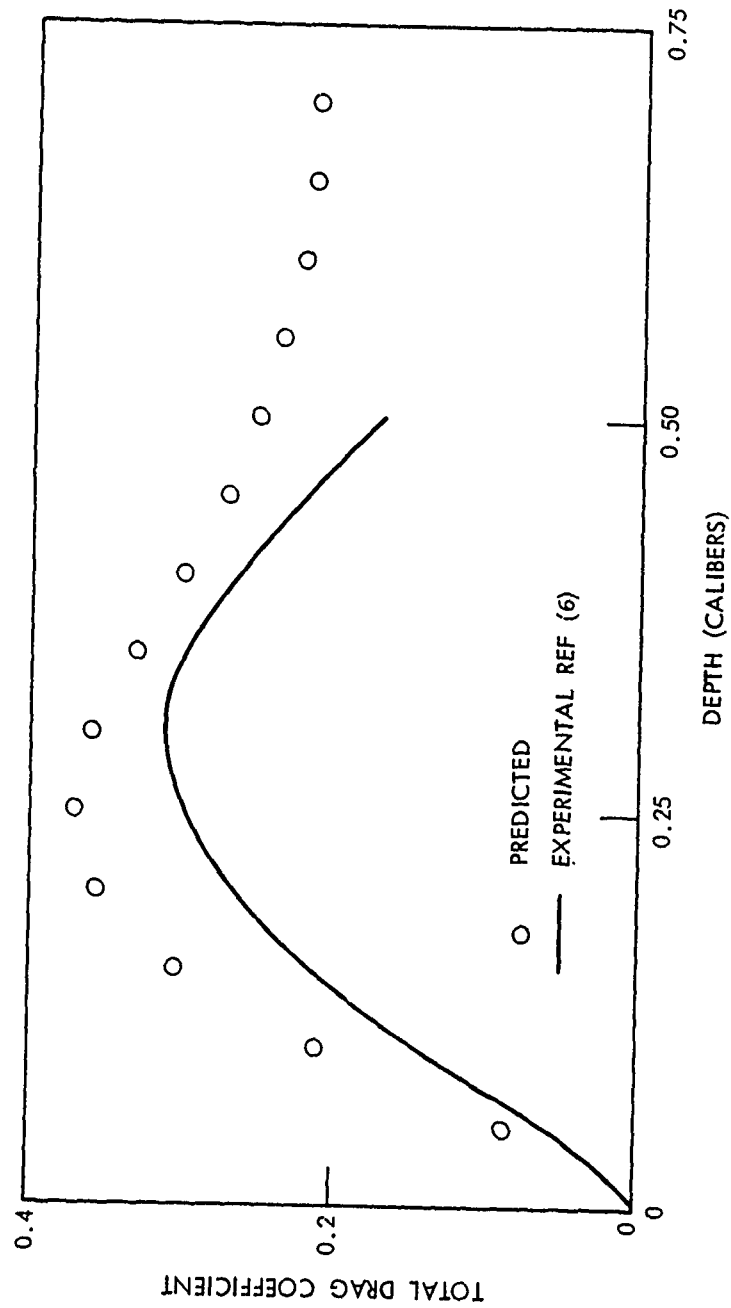


FIG. C-2 TOTAL DRAG COEFFICIENT VS DEPTH FOR 60/60 OGIVES

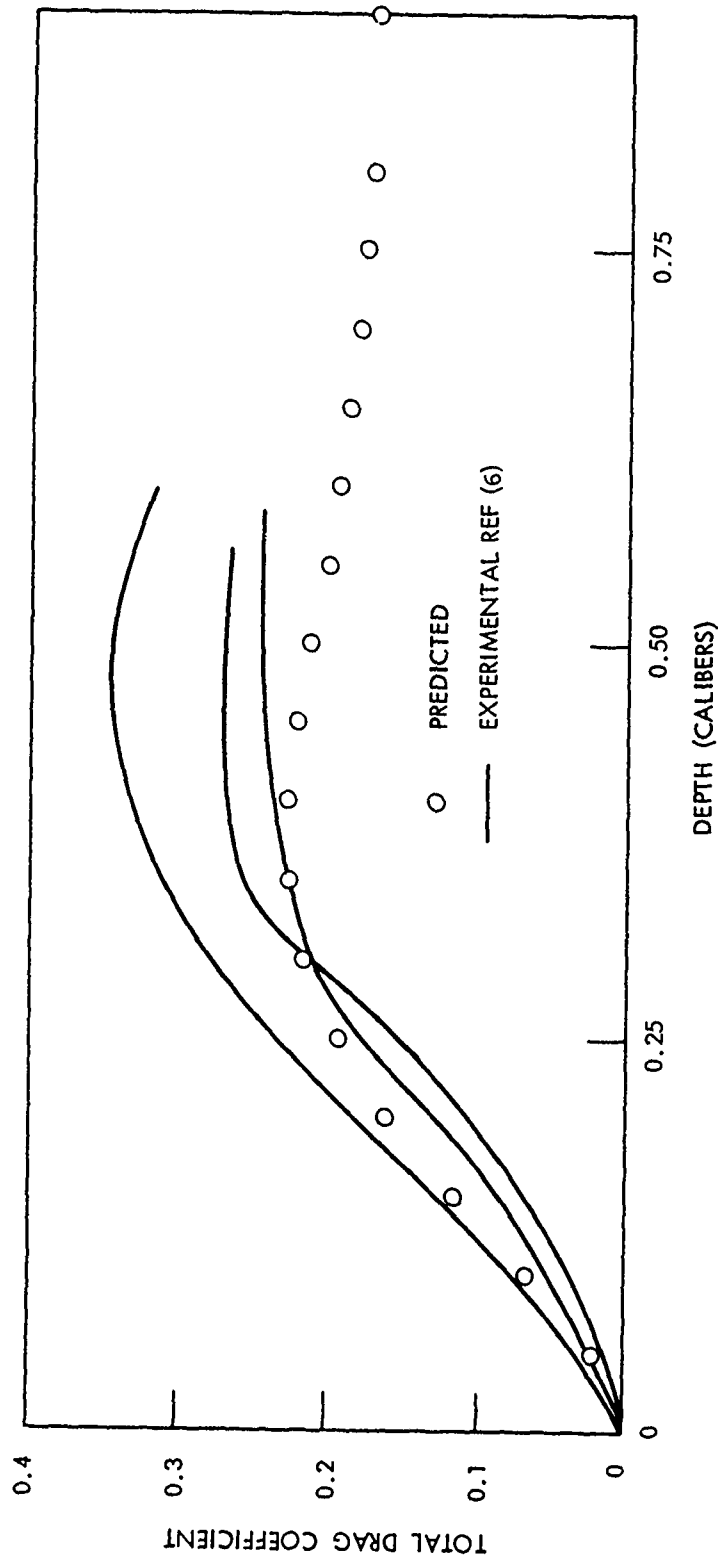


FIG. C-3 TOTAL DRAG COEFFICIENT VS DEPTH FOR 48.18/48.18 OGIVES

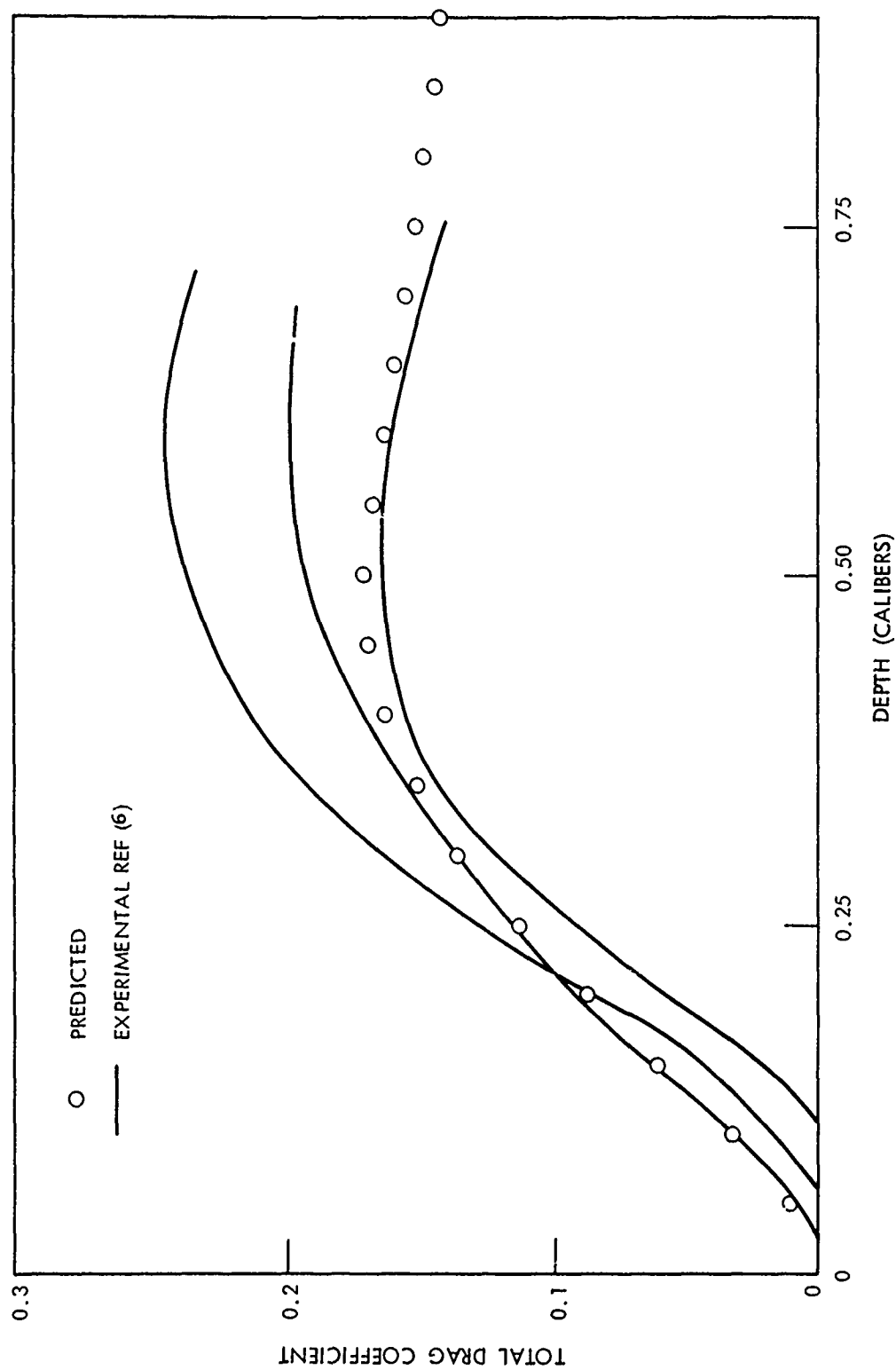


FIG. C-4 TOTAL DRAG COEFFICIENT VS DEPTH FOR 41.44/41.44 OGIVES

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