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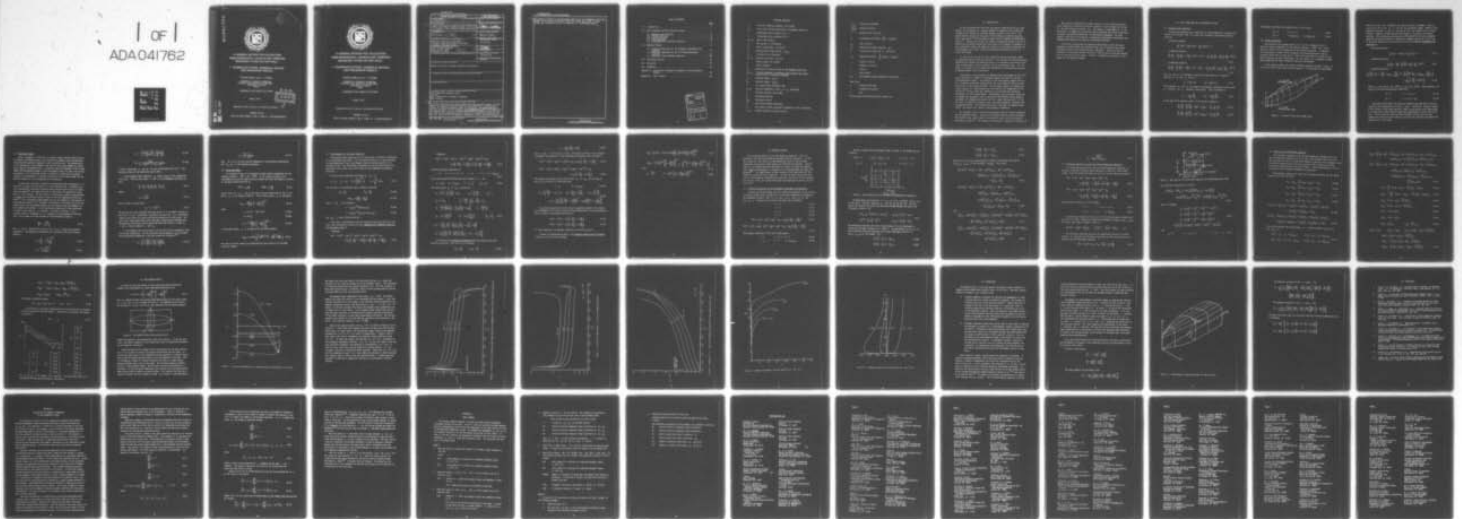
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A GENERAL METHOD FOR CALCULATING THREE-DIMENSIONAL LAMINAR AND --ETC(U)  
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**A GENERAL METHOD FOR CALCULATING  
THREE-DIMENSIONAL LAMINAR AND TURBULENT  
BOUNDARY LAYERS ON SHIP HULLS**

**I. COORDINATE SYSTEM, NUMERICAL METHOD  
AND PRELIMINARY RESULTS**

**TUNCER CEBECI and K. C. CHANG**

DEPARTMENT OF MECHANICAL ENGINEERING  
CALIFORNIA STATE UNIVERSITY AT LONG BEACH  
LONG BEACH, CALIFORNIA  
90840

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the numerical solution of the governing equation for the orthogonal curvilinear system; and obtaining preliminary results for simple ship forms. Further studies are in progress and will be reported in a forthcoming report.

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## PRINCIPAL NOTATION

A	Van Driest damping parameter, see (2.20b)
$cf_s$	local skin friction coefficient in streamwise direction
f	transformed vector potential for $\psi$
g	transformed vector potential for $\phi$
$h_1, h_2$	metric coefficients
$h_j$	net spacing in $\eta$ -direction
$h_z$	metric coefficient, see (2.15)
H	boundary-layer shape factor, $\delta^*/\theta_{11}$
$k_n$	net spacing in $x$ -direction
$K_1, K_2$	geodesic curvatures, see (2.5)
L	mixing length, see (2.20a)
p	static pressure
$q_e$	magnitude of velocity vector at the boundary-layer edge
$q_s, q_n$	velocity components in boundary-layer parallel and normal, respectively, to external streamline
$R_s$	Reynolds number, $u_e s_1/\nu$
$R_\theta$	Reynolds number, $q_e \theta_{11}/\nu$
s	arc length along coordinate line
u, v, w	velocity components in the $x_1, y, x_2$ directions
$u_\tau$	friction velocity, see (2.20c)
$u_\infty$	freestream velocity
$u_{ref}$	reference velocity
$x_1$	curvilinear surface coordinate
$x_2$	curvilinear surface coordinate orthogonal to the $x$ -coordinate
y, $x_3$	distance normal to the body surface

$x, y, z$	Cartesian coordinates
$-\rho \overline{u'v'}$ , $-\rho \overline{u'w'}$	Reynolds stresses
$\delta$	boundary-layer thickness
$\delta^*$	displacement thickness, $\int_0^{\infty} (1 - q_s/q_e) dy$
$\epsilon_m$	eddy viscosity
$\epsilon_m^+$	dimensionless eddy viscosity, $\epsilon_m/\nu$
$\eta$	similarity variable for $y$ , see (2.22)
$\theta_{11}$	momentum thickness, $\int_0^{\infty} q_s/q_e (1 - q_s/q_e) dy$
$\mu$	dynamic viscosity
$\nu$	kinematic viscosity
$\rho$	density
$\tau$	shear stress
$\phi, \psi$	two-component vector potential, see (2.23a)

#### Subscripts

$e$	boundary-layer edge
$s$	streamwise direction
$w$	wall

Primes denote differentiation with respect to  $\eta$

## 1.0 INTRODUCTION

The work reported here describes the progress made during the past year towards the development of a general method for computing three-dimensional incompressible laminar and turbulent boundary layers on ship hulls. The method employs an implicit two-point finite-difference method developed by Keller and Cebeci<sup>(1)</sup> and an algebraic eddy-viscosity formulation developed by Cebeci<sup>(2)</sup>. During the past year our efforts concentrated on the choice of an appropriate coordinate system; the calculation of the geometric parameters of this coordinate system; the numerical solution of the governing equations for the orthogonal curvilinear system; and obtaining preliminary results for simple ship forms.

In Section 2 we discuss the basic equations and the coordinate system. The governing equations which, in Section 2.1, are written for three-dimensional incompressible laminar and turbulent flows in an orthogonal curvilinear coordinate system require initial conditions on two intersecting lines. For this reason, in Section 2.2, we discuss the formulation of the governing equations to generate these initial conditions.

In principle, the calculation of boundary-layer development on ship hulls can be achieved in a number of coordinate systems applicable to arbitrary three-dimensional bodies. The presence of the free surface suggests the use of a streamline coordinate system because the flow boundaries and the coordinate lines in that system coincide. In our study, however, we abandon this system in favor of a body-oriented one (see Section 2.2) suggested by Miloh and Patel<sup>(3)</sup>, which avoids the complexity of calculating the streamline coordinates for each Froude number. Consequently, the coordinate system and its geometrical parameters such as the metric coefficients and the geodesic curvatures are computed once and for all for a specified ship hull. If the ship hull is described in analytic form, then these geometric parameters can be computed exactly. For arbitrary ship hulls, they must be computed numerically. We discuss this aspect of the problem in Appendix A.

For turbulent flows, the governing equations require closure assumptions for the Reynolds stresses. Here we use the eddy-viscosity concept and model the Reynolds shear stress terms by an algebraic eddy-viscosity formulation developed by Cebeci<sup>(2)</sup>. This phase of the problem is discussed in Section 2.4.

When physical coordinates are used, solutions of the governing boundary-layer equations are sensitive to the net spacings in the streamwise direction ( $x$ ) and the crosswise direction ( $z$ ), and require a large number of  $x$ - and  $z$ -stations. To remove this sensitivity and to generate the initial conditions, we express the governing equations in transformed coordinates. For this reason, in Section 2.5, we discuss a convenient transformation and express the boundary-layer equations in terms of transformed variables.

In Section 3 we describe the numerical method used to solve the basic equations for both laminar and turbulent flows, and in Section 4 we present preliminary results for a double-elliptic ship model. These sections are followed by a discussion section (Section 5) where we outline the work planned for the next phase of the studies. Finally, in Appendix B we present a user's manual for the computer program.

## 2.0 BASIC EQUATIONS AND THE COORDINATE SYSTEM

### 2.1 Boundary-Layer Equations

The governing boundary-layer equations for three-dimensional incompressible laminar and turbulent flows in a curvilinear orthogonal coordinate system are given by:

Continuity Equation

$$\frac{\partial}{\partial x_1} (h_2 u) + \frac{\partial}{\partial x_2} (h_1 w) + \frac{\partial}{\partial x_3} (h_1 h_2 v) = 0 \quad (2.1)$$

$x_1$ -Momentum Equation

$$\frac{u}{h_1} \frac{\partial u}{\partial x_1} + \frac{w}{h_2} \frac{\partial u}{\partial x_2} + v \frac{\partial u}{\partial x_3} + K_2 w^2 - K_1 u w = -\frac{1}{h_1} \frac{\partial}{\partial x_1} \left( \frac{p}{\rho} \right) + \frac{\partial}{\partial x_3} \left( \nu \frac{\partial u}{\partial x_3} - \overline{u'v'} \right) \quad (2.2)$$

$x_2$ -Momentum Equation

$$\frac{u}{h_1} \frac{\partial w}{\partial x_1} + \frac{w}{h_2} \frac{\partial w}{\partial x_2} + v \frac{\partial w}{\partial x_3} + K_1 u^2 - K_2 u w = -\frac{1}{h_2} \frac{\partial}{\partial x_2} \left( \frac{p}{\rho} \right) + \frac{\partial}{\partial x_3} \left( \nu \frac{\partial w}{\partial x_3} - \overline{u'w'} \right) \quad (2.3)$$

Here  $h_1$  and  $h_2$  are the metric coefficients and they are, in general, functions of  $x_1$  and  $x_2$ , that is,

$$h_1 = h_1(x_1, x_2) \quad h_2 = h_2(x_1, x_2) \quad (2.4)$$

The parameters  $K_1$  and  $K_2$  are known as the geodesic curvatures of the curves  $x_2 = \text{constant}$  and  $x_1 = \text{constant}$ , respectively. They are defined as

$$K_1 = -\frac{1}{h_1 h_2} \frac{\partial h_1}{\partial x_2} \quad K_2 = -\frac{1}{h_1 h_2} \frac{\partial h_2}{\partial x_1} \quad (2.5)$$

At the edge of the boundary layer, (2.2) and (2.3) reduce to

$$\frac{u_e}{h_1} \frac{\partial u_e}{\partial x_1} + \frac{w_e}{h_2} \frac{\partial u_e}{\partial x_2} + K_2 w_e^2 - K_1 u_e w_e = -\frac{1}{h_1} \frac{\partial}{\partial x_1} \left( \frac{p}{\rho} \right) \quad (2.6a)$$

$$\frac{u_e}{h_1} \frac{\partial w_e}{\partial x_1} + \frac{w_e}{h_2} \frac{\partial w_e}{\partial x_2} + K_1 u_e^2 - K_2 u_e w_e = -\frac{1}{h_2} \frac{\partial}{\partial x_2} \left( \frac{p}{\rho} \right) \quad (2.6b)$$

The boundary conditions for equations (2.1) to (2.3) are:

$$x_3 = 0 \quad u, v, w = 0 \quad (2.7a)$$

$$x_3 = \delta \quad u = u_e(x_1, x_2) \quad w = w_e(x_1, x_2) \quad (2.7b)$$

## 2.2 Initial Conditions

The solution of the system given by (2.1) to (2.3) subject to (2.7) requires closure assumptions for the Reynolds stresses,  $-\rho \overline{u'v'}$  and  $-\rho \overline{v'w'}$ . They also require initial conditions on two intersecting planes. In general, the construction of these initial conditions for three-dimensional flows on arbitrary bodies such as ship hulls is rather difficult due to a variety of bow shapes which may be quite complicated. For this reason, it is necessary to make some assumptions for them in order to start the calculations.

In our study we choose the inviscid dividing streamline on which  $\partial p / \partial x_2 = 0$  to be one of the initial data line (see Figure 1). In the case of a rectilinear

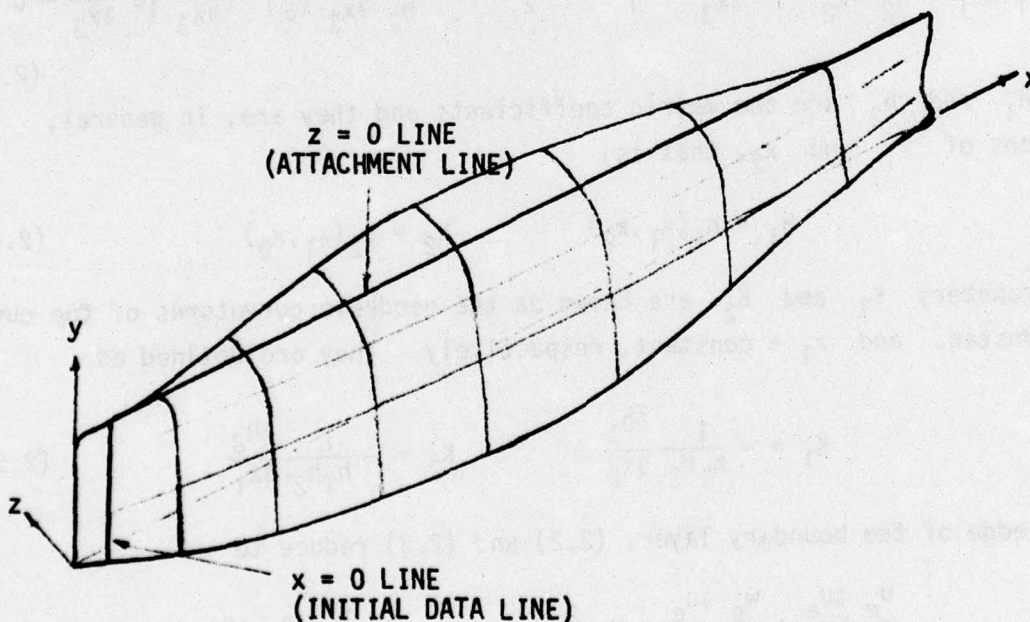


Figure 1. A sketch of the initial data lines

motion of a ship, this streamline runs along the plane of symmetry. We also select this line as the  $x_2 = 0$  line for the coordinate system. Because of the symmetry conditions  $w$  and  $K_1$  are zero on this line causing (2.3) to become singular. However, differentiation with respect to  $x_2$  yields a nonsingular equation. After performing the necessary differentiation for the  $x_2$ -momentum equation and taking advantage of appropriate symmetry conditions in (2.3) and in the other equations, we can write the so-called streamwise attachment-line equations as:

Continuity Equation:

$$\frac{\partial}{\partial x_1} (h_2 u) + h_1 h_2 w_z + \frac{\partial}{\partial x_3} (h_1 h_2 v) = 0 \quad (2.8)$$

$x_1$ -Momentum Equation:

$$\frac{u}{h_1} \frac{\partial u}{\partial x_1} + v \frac{\partial u}{\partial x_3} = \frac{u_e}{h_1} \frac{\partial u_e}{\partial x_1} + \frac{\partial}{\partial x_3} \left( v \frac{\partial u}{\partial x_3} - \overline{u'v'} \right) \quad (2.9)$$

$x_2$ -Momentum Equation

$$\begin{aligned} \frac{u}{h_1} \frac{\partial w_z}{\partial x_1} + w_z^2 + v \frac{\partial w_z}{\partial x_3} - K_2 u w_z + \frac{\partial K_1}{h_2 \partial x_2} u^2 = \frac{u_e}{h_1} \frac{\partial w_{ze}}{\partial x_1} + w_{ze}^2 - K_2 u_e w_{ze} + \frac{\partial K_1}{h_2 \partial x_2} u_e^2 \\ + \frac{\partial}{\partial x_3} \left[ v \frac{\partial w_z}{\partial x_3} - (\overline{v'w'})_z \right] \end{aligned} \quad (2.10)$$

where  $w_z = 1/h_2 \partial w / \partial x_2$  and  $(\overline{v'w'})_z = 1/h_2 \partial / \partial x_2 (\overline{v'w'})$ . These equations are subject to the following boundary conditions:

$$x_3 = 0 \quad u = v = w_z = 0 \quad (2.11a)$$

$$x_3 = \delta \quad u = u_e, w_z = w_{ze} \quad (2.11b)$$

The other initial data line should be selected near the bow of the ship along the line perpendicular to the  $x_2 = \text{const.}$  coordinate. However, because of the variety of bow shapes possible, some of which may be quite complicated, it is necessary to make further simplifying approximations to the flow on the line. In our study we assume the flow along the initial line to be two-dimensional without pressure gradient, and solve the governing two-dimensional equations for a flat plate.

### 2.3 Coordinate System

Since, in general, a ship form is a rather complex nondevelopable surface, a Cartesian coordinate system is not suitable for boundary-layer calculations. The streamline coordinate system is a very attractive one. The determination of the streamlines, the orthogonal lines and the associated geometrical parameters require considerable work. In addition, they are dependent on the Froude number and the Reynolds number if the displacement effect is taken into account. Consequently, this coordinate system, in addition to being hard to compute, becomes also uneconomical to use when the effects of the Froude number and the Reynolds number are to be systematically investigated.

In selecting a coordinate system for the boundary-layer calculations, it is desirable to choose one such that it is calculated only once. Although a general nonorthogonal coordinate system has been developed with success by Cebeci, Kaups and Ramsey<sup>(4)</sup>, the numerical work for evaluating the geometrical parameters is very involved. For this reason, in the present study, we have decided to abandon this coordinate system in favor of the one suggested by Miloh and Patel<sup>(3)</sup>. This coordinate system depends only on the geometry of the body and is calculated once and for all. Therefore, the effects of the draft, the trim, the Froude number, and the Reynolds number on ship boundary-layer development can be systematically studied. In this coordinate system, one of the coordinates is taken as the lines of  $x_1 = \text{constant}$ . The other coordinate  $x_2(x,z)$  constant, which is orthogonal to  $x_1 = \text{constant}$  lines everywhere on the ship hull, can be shown to be the solution of the differential equation

$$\frac{dz}{dx} = -\frac{f_x f_z}{1 + f_z^2} \quad (2.12)$$

Here  $y = f(x,z)$  defines the ship hull, and  $(x,y,z)$  denote the Cartesian coordinates. The geometrical parameters are calculated from the following formulas

$$h_1 = \left(1 + \frac{f_x^2}{1 + f_z^2}\right)^{1/2} \quad (2.13a)$$

$$h_2 = \frac{(1 + f_z^2)^{1/2}}{\partial x_2 / \partial z} \quad (2.13b)$$

$$K_1 = - \frac{f_x [f_{xz} (1 + f_z^2) - f_x f_z f_{zz}]}{(1 + f_z^2)^{3/2} (1 + f_x^2 + f_z^2)} \quad (2.13c)$$

$$K_2 = \frac{f_x f_{zz}}{(1 + f_z^2)^{3/2} (1 + f_z^2 + f_x^2)^{1/2}} \quad (2.13d)$$

It may be noted that  $K_1$  and  $K_2$  can also be obtained from (2.5). This provides a check on the expressions given above.

In the boundary-layer equations,  $h_2$  always occurs in the combination  $1/h_2 \partial/\partial x_2$ . Since we have chosen  $x$  and  $x_2$  as independent variables, we have

$$\frac{1}{h_2} \frac{\partial}{\partial x_2} = \frac{1}{h_2} \frac{\partial z}{\partial x_2} \frac{\partial}{\partial z} = \frac{1}{h_z} \frac{\partial}{\partial x_z} \quad (2.14)$$

where

$$h_z = h_2 \frac{\partial x_2}{\partial z} \quad (2.15)$$

From (2.13b) it follows that

$$h_z = (1 + f_z^2)^{1/2} \quad (2.16)$$

We see that once the curvatures have been found, it is no longer necessary to work with  $(x, x_2)$  coordinates; instead we can work with  $(x, z)$  coordinates. The use of this coordinate system offers the major computational advantage that the pressure distribution can be prescribed much more readily in terms of  $x$  and  $z$  than in terms of  $x$  and  $x_2$ .

In the boundary-layer calculations, we need the velocity components along the surface coordinates. For the coordinate system adopted, it can be shown that the velocity components in the two systems are related by<sup>(5)</sup>

$$u_e = \frac{(1 + f_z^2)U + f_x V - f_x f_z W}{[(1 + f_z^2)(1 + f_x^2 + f_z^2)]^{1/2}} \quad (2.17a)$$

$$w_e = \frac{Vf_z + W}{(1 + f_z^2)^{1/2}} \quad (2.17b)$$

Here  $(U, V, W)$  are the velocity components in the Cartesian coordinates, and  $(u_e, w_e)$  in the adopted coordinates.

#### 2.4 Turbulence Model

For turbulent flows, it is necessary to make closure assumptions for the Reynolds stresses,  $-\rho \overline{u'v'}$  and  $-\rho \overline{v'w'}$ . In our study, we satisfy this requirement by using the eddy-viscosity concept and relate the Reynolds stresses to the mean velocity profiles by

$$-\overline{u'v'} = \epsilon_m \frac{\partial u}{\partial y} \quad -\overline{v'w'} = \epsilon_m \frac{\partial w}{\partial y} \quad (2.18)$$

Here we have let  $x_3 = y$ . We use the eddy-viscosity formulation of ref. 2 and define  $\epsilon_m$  by two separate formulas. In the inner region,  $\epsilon_m$  is defined as

$$(\epsilon_m)_i = L^2 \left[ \left( \frac{\partial u}{\partial y} \right)^2 + \left( \frac{\partial w}{\partial y} \right)^2 \right]^{1/2} \quad (2.19)$$

where

$$L = 0.4 y [1 - \exp(-y/A)] \quad (2.20a)$$

$$A = 26 \nu u_\tau^{-1} \quad (2.20b)$$

$$u_\tau = \nu^{1/2} \left[ \left( \frac{\partial u}{\partial y} \right)_w^2 + \left( \frac{\partial w}{\partial y} \right)_w^2 \right]^{1/4} \quad (2.20c)$$

In the outer region,  $\epsilon_m$  is defined by the following formula:

$$(\epsilon_m)_o = 0.0168 \left| \int_0^\infty (\sqrt{u^2 + w^2} - \sqrt{u_e^2 + w_e^2}) dy \right| \quad (2.21)$$

The inner and outer regions are established by the continuity of the eddy-viscosity formula.

## 2.5 Transformation of the Basic Equations

The boundary-layer equations can be solved either in physical coordinates or in transformed coordinates. Each coordinate system has its own advantage. In three-dimensional flows, the computer time and storage required is an important factor. The transformed coordinates are then favored because the coordinates allow larger steps to be taken in the streamwise and spanwise directions.

We define the transformed coordinates by ( $x_3 = y$ )

$$x_1 = x_1, \quad x_2 = x_2, \quad d\eta = \left(\frac{u_e}{vs_1}\right)^{1/2} dy \quad s_1 = \int_0^{x_1} h_1 dx_1 \quad (2.22)$$

and introduce a two-component vector potential such that

$$uh_2 = \frac{\partial \psi}{\partial y} \quad wh_1 = \frac{\partial \phi}{\partial y} \quad (2.23a)$$

$$vh_1 h_2 = -\left(\frac{\partial \psi}{\partial x_1} + \frac{\partial \phi}{\partial x_2}\right) \quad (2.23b)$$

where  $\psi$  and  $\phi$  are defined as

$$\psi = (vu_e s_1)^{1/2} h_2 f(x_1, x_2, \eta) \quad (2.24a)$$

$$\phi = (vu_e s_1)^{1/2} u_{ref}/u_e h_1 g(x_1, x_2, \eta) \quad (2.24b)$$

and  $u_{ref}$  is some reference velocity.

Using these transformations and the relations given by (2.6) and (2.12), after some rearranging, we can write the x-momentum and z-momentum equations for the general case as:

x-Momentum

$$\begin{aligned} (bf'')' + m_1 f f'' - m_2 (f')^2 - m_5 f' g' + m_6 f'' g - m_8 (g')^2 + m_{11} \\ = m_{10} \left( f' \frac{\partial f'}{\partial x_1} - f'' \frac{\partial f}{\partial x_1} \right) + m_7 \left( g' \frac{\partial f'}{\partial x_2} - f'' \frac{\partial g}{\partial x_2} \right) \end{aligned} \quad (2.25)$$

z-Momentum

$$\begin{aligned} (bq'')' + m_1 fg'' - m_4 f'g' - m_3 (g')^2 + m_6 gg'' - m_9 (f')^2 + m_{12} \\ = m_{10} \left( f' \frac{\partial g'}{\partial x_1} - g'' \frac{\partial f}{\partial x_1} \right) + m_7 \left( g' \frac{\partial g'}{\partial x_2} - g'' \frac{\partial g}{\partial x_2} \right) \end{aligned} \quad (2.26)$$

and their boundary conditions as

$$\eta = 0, \quad f = f' = g = g' = 0, \quad \eta = \eta_\infty, \quad f' = 1, \quad g' = w_e/u_{ref} \quad (2.27)$$

Here primes denote differentiation with respect to  $\eta$ , and

$$f' = u/u_e, \quad g' = w/u_{ref}, \quad b = 1 + \epsilon_m^+, \quad \epsilon_m^+ = \epsilon_m/\nu \quad (2.28)$$

The coefficients  $m_1$  to  $m_{12}$  are given by

$$\begin{aligned} m_1 &= \frac{1}{2} \left( 1 + \frac{s_1}{u_e} \frac{\partial u_e}{\partial s_1} \right) - s_1 K_2, & m_2 &= \frac{s_1}{u_e} \frac{\partial u_e}{\partial s_1}, & m_3 &= 0 \\ m_4 &= -s_1 K_2, & m_5 &= \frac{u_{ref}}{u_e} \frac{s_1}{u_e h_2} \frac{\partial u_e}{\partial x_2} - s_1 K_1 \\ m_6 &= \frac{u_{ref}}{u_e} \left[ \frac{1}{2} \left( \frac{\partial s_1}{h_2 \partial x_2} - \frac{s_1}{u_e} \frac{\partial u_e}{h_2 \partial x_2} \right) - s_1 K_2 \right], & m_7 &= \frac{u_{ref}}{u_e} \frac{s_1}{h_2} \\ m_8 &= s_1 K_2 \left( \frac{u_{ref}}{u_e} \right)^2, & m_9 &= s_1 K_1 \left( \frac{u_e}{u_{ref}} \right), & m_{10} &= \frac{s_1}{h_1} \end{aligned} \quad (2.29)$$

$$m_{11} = s_1 \left[ \frac{1}{u_e} \frac{\partial u_e}{\partial s_1} + \frac{w_e}{u_e} \left( \frac{1}{u_e} \frac{\partial u_e}{h_2 \partial x_2} + K_2 \frac{w_e}{u_e} - K_1 \right) \right]$$

$$m_{12} = \frac{u_e}{u_{ref}} \left[ \frac{s_1}{u_e} \frac{\partial w_e}{\partial s_1} + \left( \frac{w_e}{u_e} \right) \frac{s_1}{u_e} \frac{\partial w_e}{h_2 \partial x_2} + s_1 K_1 - s_1 K_2 \frac{w_e}{u_e} \right]$$

To transform the streamwise attachment-line flow equations and their boundary conditions, (2.8) to (2.11), we define:

$$uh_2 = \frac{\partial \psi}{\partial y}, \quad h_1 w_z = \frac{\partial \phi}{\partial y} \quad (2.30a)$$

$$v = -\frac{1}{h_1 h_2} \left( \frac{\partial \psi}{\partial x_1} + h_2 \phi \right) \quad (2.30b)$$

with  $\psi$  and  $\phi$  still given by (2.24). With these variables, the streamwise attachment-line equations in the transformed coordinates can be written as

$$(bf'')' + m_1 f f'' - m_2 (f')^2 + m_6 f'' g + m_{11} = m_{10} \left( f' \frac{\partial f'}{\partial x_1} - f'' \frac{\partial f}{\partial x_1} \right) \quad (2.31)$$

$$(bg'')' + m_1 f g'' - m_4 f' g' - m_3 (g')^2 + m_6 g g'' - m_9 (f')^2 + m_{12} \\ = m_{10} \left( f' \frac{\partial g'}{\partial x_1} - g'' \frac{\partial f}{\partial x_1} \right) \quad (2.32)$$

The boundary conditions and the coefficients  $m_1$  to  $m_{12}$  are the same as those in (2.27) and in (2.29) except now

$$\eta = \eta_\infty \quad g' = w_{ze}/u_{ref} \quad (2.33a)$$

$$m_3 = \frac{s_1}{h_2} \frac{u_{ref}}{u_e}, \quad m_6 = m_3, \quad m_9 = \frac{u_e}{u_{ref}} s_1 \frac{\partial K_1}{h_2 \partial x_2}, \quad m_{11} = \frac{s_1}{u_e} \frac{\partial u_e}{\partial s_1} \quad (2.33b)$$

$$m_{12} = \frac{u_e}{u_{ref}} \left[ \frac{s_1}{u_e} \frac{\partial w_{ze}}{\partial s_1} + s_1 \left( \frac{w_{ze}}{u_e} \right)^2 - s_1 K_2 \left( \frac{w_{ze}}{u_e} \right) + s_1 \frac{\partial K_1}{h_2 \partial x_2} \right]$$

The governing equations for the initial conditions normal to the streamwise attachment-line flow are given by the two-dimensional flat-plate equations, which in terms of transformed variables are

$$(bf'')' + \frac{1}{2} f f'' = m_7 \left( g' \frac{\partial f'}{\partial x_2} - f'' \frac{\partial g}{\partial x_2} \right) \quad (2.34)$$

$$(bg'')' + \frac{1}{2} f g'' = m_7 \left( g' \frac{\partial g'}{\partial x_2} - g'' \frac{\partial g}{\partial x_2} \right) \quad (2.35)$$

For these equations, the boundary conditions are given by (2.27).

In terms of transformed variables, the algebraic eddy-viscosity formulas as given by (2.13) to (2.15) become

$$(\epsilon_m^+)_i = \left\{ 0.4 \eta [1 - \exp(-y^+/26)] \right\}^2 \left\{ R_s \left[ (f'')^2 + \left( \frac{u_{ref}}{u_e} g'' \right)^2 \right] \right\}^{1/2} \quad (2.36)$$

$$(\epsilon_m^+)_o = 0.0168 R_s^{1/2} \left\{ \left[ 1 + \left( \frac{w_e}{u_e} \right)^2 \right]^{1/2} \eta_\infty - \int_0^{\eta_e} \left[ f'^2 + \left( \frac{u_{ref}}{u_e} g' \right)^2 \right]^{1/2} d\eta \right\} \quad (2.37)$$

Here

$$R_s = \frac{u_e s_1}{\nu} \quad y^+ = \eta R_s^{1/4} \left[ f_w''^2 + \left( \frac{u_{ref}}{u_e} g_w'' \right)^2 \right]^{1/4} \quad (2.38)$$

### 3.0 NUMERICAL METHOD

We use the Box method to solve the governing equations. This is a two-point finite-difference method developed by Keller and Cebeci. So far this method has been applied to two-dimensional flows as well as three-dimensional flows and has been found to be efficient and accurate. Descriptions of this method have been presented in a series of papers and reports, and a detailed presentation is contained in a forthcoming book by Cebeci and Bradshaw<sup>(6)</sup>. For completeness we shall present a brief description of this method in application to our equations. At first we shall discuss the solution of the streamwise attachment-line equations, (2.31) and (2.32), and then the solution of the full three-dimensional flow equations as given by (2.25) and (2.26).

#### 3.1 Difference Equations for the Streamwise Attachment-Line Equations

According to the Box method, we first reduce the equations (2.31), (2.32), (2.27) and (2.33a) into a system of six first-order equations by introducing new dependent variables  $u(x_1, x_2, \eta)$ ,  $v(x_1, x_2, \eta)$ ,  $w(x_1, x_2, \eta)$  and  $t(x_1, x_2, \eta)$ . Equations (2.31) and (2.32) then can be written as

$$f' = u \quad (3.1a)$$

$$u' = v \quad (3.1b)$$

$$g' = w \quad (3.1c)$$

$$w' = t \quad (3.1d)$$

$$(bv)' + m_1fv - m_2u^2 + m_6gv + m_{11} = m_{10} \left( u \frac{\partial u}{\partial x_1} - v \frac{\partial f}{\partial x_1} \right) \quad (3.1e)$$

$$(bt)' + m_1ft - m_4uw - m_3w^2 + m_6gt - m_9u^2 + m_{12} = m_{10} \left( u \frac{\partial w}{\partial x_1} - t \frac{\partial f}{\partial x_1} \right) \quad (3.1f)$$

The boundary conditions (2.27) and (2.33a) become

$$\eta = 0 \quad f = g = u = w = 0 \quad (3.2a)$$

$$\eta = \eta_\infty \quad u = 1 \quad w = w_{ze}/u_{ref} \quad (3.2b)$$

We next consider the net rectangle shown in Figure 2 and denote the net points by

$$(x_1)_0 = 0 \quad (x_1)_n = (x_1)_{n-1} + k_n \quad n = 1, 2, \dots, N$$

$$\eta_0 = 0 \quad \eta_j = \eta_{j-1} + h_j \quad j = 1, 2, \dots, J$$

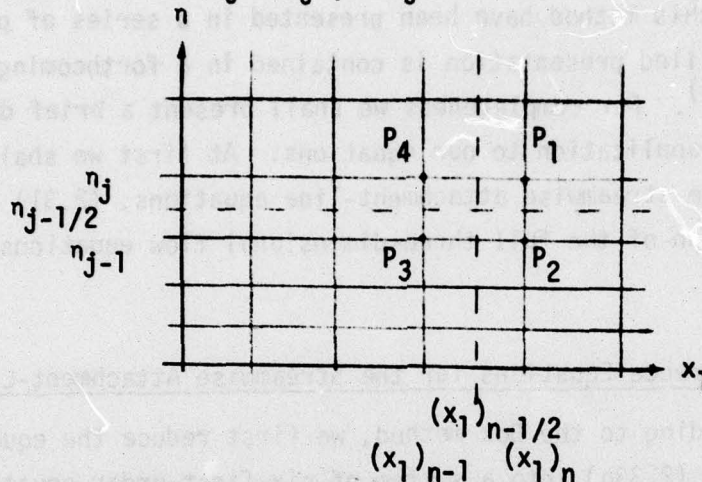


Figure 2. Net rectangle for the streamwise-attachment-line equations

We approximate the quantities  $(f, u, v, g, w, t)$  at points  $[(x_1)_n, \eta_j]$  of the net by functions denoted by  $(f_j^n, u_j^n, v_j^n, g_j^n, w_j^n, t_j^n)$ . We also employ the notation for points and quantities midway between net points and for any net function  $s_j^n$ :

$$\begin{aligned} (x_1)_{n-1/2} &= \frac{1}{2} [(x_1)_n + (x_1)_{n-1}] & \eta_{j-1/2} &= \frac{1}{2} (\eta_j + \eta_{j-1}) \\ s_j^{n-1/2} &= \frac{1}{2} (s_j^n + s_j^{n-1}) & s_{j-1/2}^n &= \frac{1}{2} (s_j^n + s_{j-1}^n) \end{aligned} \quad (3.3)$$

The difference equations which are to approximate (3.1) are formulated by considering one mesh rectangle as in Figure 2. We approximate (3.1a, b, c, d) using centered difference quotients and average them about the midpoint  $[(x_1)_n, \eta_{j-1/2}]$  of the segment  $P_1P_2$ .

$$h_j^{-1} (f_j^n - f_{j-1}^n) = u_{j-1/2}^n \quad (3.4a)$$

$$h_j^{-1} (u_j^n - u_{j-1}^n) = v_{j-1/2}^n \quad (3.4b)$$

$$h_j^{-1}(g_j^n - g_{j-1}^n) = w_{j-1/2}^n \quad (3.4c)$$

$$h_j^{-1}(w_j^n - w_{j-1}^n) = t_{j-1/2}^n \quad (3.4d)$$

Similarly, (3.1e,f) are approximated by centering them about the midpoint  $(x_1)_{n-1/2}, \eta_{j-1/2}$  of the rectangle  $P_1P_2P_3P_4$ . This gives:

$$\begin{aligned} h_j^{-1}[(bv)_j^n - (bv)_{j-1}^n] + (m_1^n + \alpha_n)(fv)_{j-1/2}^n - (m_2^n + \alpha_n)(u^2)_{j-1/2}^n \\ + m_6^n(gv)_{j-1/2}^n + \alpha_n[v_{j-1/2}^{n-1}f_{j-1/2}^n - f_{j-1/2}^{n-1}v_{j-1/2}^n] \\ = R_{j-1/2}^{n-1} - m_{11}^n \end{aligned} \quad (3.4e)$$

$$\begin{aligned} h_j^{-1}[(bt)_j^n - (bt)_{j-1}^n] + (m_1^n + \alpha_n)(ft)_{j-1/2}^n - (m_4^n + \alpha_n)(uw)_{j-1/2}^n \\ - m_3^n(w^2)_{j-1/2}^n + m_6^n(gt)_{j-1/2}^n - m_9^n(u^2)_{j-1/2}^n \\ + \alpha_n[w_{j-1/2}^{n-1}u_{j-1/2}^n - u_{j-1/2}^{n-1}w_{j-1/2}^n + t_{j-1/2}^{n-1}f_{j-1/2}^n \\ - f_{j-1/2}^{n-1}t_{j-1/2}^n] = T_{j-1/2}^{n-1} - m_{12}^n \end{aligned} \quad (3.4f)$$

Here

$$\begin{aligned} R_{j-1/2}^{n-1} = \alpha_n[(fv)_{j-1/2}^{n-1} - (u^2)_{j-1/2}^{n-1}] - [h_j^{-1}\{(bv)_j^{n-1} - (bv)_{j-1}^{n-1}\} + m_1^{n-1}(fv)_{j-1/2}^{n-1} \\ - m_2^{n-1}(u^2)_{j-1/2}^{n-1} + m_6^{n-1}(gv)_{j-1/2}^{n-1} + m_{11}^{n-1}] \end{aligned} \quad (3.5a)$$

$$\begin{aligned} T_{j-1/2}^{n-1} = \alpha_n[(ft)_{j-1/2}^{n-1} - (uw)_{j-1/2}^{n-1}] - [h_j^{-1}\{(bt)_j^{n-1} - (bt)_{j-1}^{n-1}\} + m_1^{n-1}(ft)_{j-1/2}^{n-1} \\ - m_4^{n-1}(uw)_{j-1/2}^{n-1} - m_3^{n-1}(w^2)_{j-1/2}^{n-1} + m_6^{n-1}(gt)_{j-1/2}^{n-1} \\ - m_9^{n-1}(u^2)_{j-1/2}^{n-1} + m_{12}^{n-1}] \end{aligned} \quad (3.5b)$$

$$\alpha_n \equiv \frac{m_{10}^{n-1/2}}{(x_1)_n - (x_1)_{n-1}} \quad (3.5c)$$

### 3.2 Difference Equations for the Full Three-Dimensional Equations

The difference equations for the full three-dimensional equations, as given by (2.25) and (2.26), are again expressed in terms of a first-order system. With the definitions given by (3.1a) to (3.1d), they are written as

$$\begin{aligned} (bv)' + m_1 fv - m_2 u^2 - m_5 uw + m_6 gv - m_8 w^2 + m_{11} \\ = m_{10} \left( u \frac{\partial u}{\partial x_1} - v \frac{\partial f}{\partial x_1} \right) + m_7 \left( w \frac{\partial u}{\partial x_2} - v \frac{\partial g}{\partial x_2} \right) \end{aligned} \quad (3.6a)$$

$$\begin{aligned} (bt)' + m_1 ft - m_4 uw - m_3 w^2 + m_6 gt - m_9 u^2 + m_{12} \\ = m_{10} \left( u \frac{\partial w}{\partial x_1} - t \frac{\partial f}{\partial x_1} \right) + m_7 \left( w \frac{\partial w}{\partial x_2} - t \frac{\partial g}{\partial x_2} \right) \end{aligned} \quad (3.6b)$$

Their boundary conditions, (2.27), become:

$$\eta = 0 \quad f = g = u = w = 0 \quad (3.7a)$$

$$\eta = \eta_\infty \quad u = 1, \quad w = w_e / u_{\text{ref}} \quad (3.7b)$$

The difference equations for (3.1a) to (3.1d) are the same as those given by (3.4a) to (3.4d): they are written for the midpoint  $[(x_1)_n, (x_2)_i, \eta_{j-1/2}]$  of the net cube shown in Figure 3; that is,

$$h_j^{-1} (f_j^{n,i} - f_{j-1}^{n,i}) = u_{j-1/2}^{n,i}, \quad h_j^{-1} (u_j^{n,i} - u_{j-1}^{n,i}) = v_{j-1/2}^{n,i}, \quad \text{etc.}$$

The difference equations which are to approximate (3.6a,b) are rather lengthy. To illustrate the difference equations for these two equations, we consider the following model equation

$$(bv)' + m_1 fv + m_{11} = m_{10} u \frac{\partial u}{\partial x_1} + m_7 w \frac{\partial u}{\partial x_2} \quad (3.8)$$

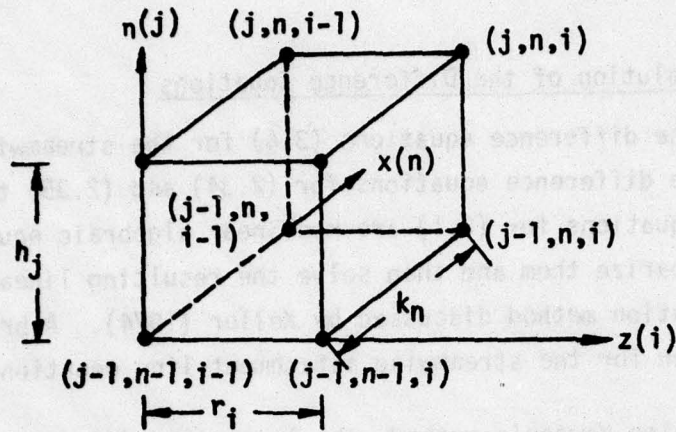


Figure 3. Net cube for the difference equations for three-dimensional flows.

The difference equations for (3.8) are:

$$\begin{aligned}
 & h_j^{-1} [(\bar{bv})_j - (\bar{bv})_{j-1}] + (m_1)_{j-1/2}^{n-1/2} (\bar{fv})_{j-1/2} + (m_{11})_{j-1/2}^{n-1/2} \\
 & = (m_{10})_{i-1/2}^{n-1/2} \bar{u}_{j-1/2} \frac{\bar{u}_n - \bar{u}_{n-1}}{k_n} + (m_7)_{j-1/2}^{n-1/2} \bar{w}_{j-1/2} \frac{\bar{u}_i - \bar{u}_{i-1}}{r_i}
 \end{aligned} \tag{3.9}$$

Here, for example,

$$\begin{aligned}
 \bar{v}_j &= \frac{1}{4} (v_j^{n,i} + v_j^{n,i-1} + v_j^{n-1,i-1} + v_j^{n-1,i}) \\
 \bar{u}_n &= \frac{1}{4} (u_j^{n,i} + u_j^{n,i-1} + u_{j-1}^{n,i} + u_{j-1}^{n,i-1}) \\
 \bar{u}_i &= \frac{1}{4} (u_j^{n,i} + u_j^{n-1,i} + u_{j-1}^{n,i} + u_{j-1}^{n-1,i})
 \end{aligned} \tag{3.10}$$

$$(m_1)_{i-1/2}^{n-1/2} = \frac{1}{4} [(m_1)_i^n + (m_1)_{i-1}^n + (m_1)_i^{n-1} + (m_1)_{i-1}^{n-1}]$$

and

$$z_0 = 0 \quad z_i = z_{i-1} + r_i \quad i = 1, 2, \dots, I \tag{3.11}$$

### 3.3 Solution of the Difference Equations

The difference equations (3.4) for the streamwise attachment-line flow and the difference equations for (2.34) and (2.35) together with the difference equations for (3.1) are nonlinear algebraic equations. We use Newton's method to linearize them and then solve the resulting linear system by the block-elimination method discussed by Keller (1974). A brief description of it will be given for the streamwise attachment-line equations.

Using Newton's method, the linearized difference equations for the system given by (3.4) and (3.2) are:

$$\delta f_j - \delta f_{j-1} - \frac{h_j}{2} (\delta u_j + \delta u_{j-1}) = (r_1)_j \quad (3.12a)$$

$$\delta u_j - \delta u_{j-1} - \frac{h_j}{2} (\delta v_j + \delta v_{j-1}) = (r_2)_j \quad (3.12b)$$

$$\delta g_j - \delta g_{j-1} - \frac{h_j}{2} (\delta w_j + \delta w_{j-1}) = (r_3)_j \quad (3.12c)$$

$$\delta w_j - \delta w_{j-1} - \frac{h_j}{2} (\delta t_j + \delta t_{j-1}) = (r_4)_j \quad (3.12d)$$

$$\begin{aligned} (\zeta_1)_j \delta v_j + (\zeta_2)_j \delta v_{j-1} + (\zeta_3)_j \delta f_j + (\zeta_4)_j \delta f_{j-1} + (\zeta_5)_j \delta u_j + (\zeta_6)_j \delta u_{j-1} \\ + (\zeta_7)_j \delta g_j + (\zeta_8)_j \delta g_{j-1} = (r_5)_j \end{aligned} \quad (3.12e)$$

$$\begin{aligned} (\beta_1)_j \delta t_j + (\beta_2)_j \delta t_{j-1} + (\beta_3)_j \delta f_j + (\beta_4)_j \delta f_{j-1} + (\beta_5)_j \delta w_j + (\beta_6)_j \delta w_{j-1} \\ + (\beta_7)_j \delta u_j + (\beta_8)_j \delta u_{j-1} + (\beta_9)_j \delta g_j + (\beta_{10})_j \delta g_{j-1} = (r_6)_j \end{aligned} \quad (3.12f)$$

Here we have dropped the superscripts  $n, i$  and have defined  $(r_k)_j, (\zeta_k)_j$  and  $(\beta_k)_j$  by

$$\begin{aligned} (r_1)_j &= f_{j-1} - f_j + h_j u_{j-1/2}, & (r_2)_j &= u_{j-1} - u_j + h_j v_{j-1/2}, \\ (r_3)_j &= g_{j-1} - g_j + h_j w_{j-1/2}, & (r_4)_j &= w_{j-1} - w_j + h_j t_{j-1/2} \end{aligned} \quad (3.13)$$

$$(r_5)_j = R_{j-1/2}^{n-1} - m_{11}^n - [(bv)_{j-1/2}' + (m_1 + \alpha_n)(fv)_{j-1/2} - (m_2 + \alpha_n)(u^2)_{j-1/2} + m_6(qv)_{j-1/2} + \alpha_n (v_{j-1/2}^{n-1} f_{j-1/2} - f_{j-1/2}^{n-1} v_{j-1/2})]$$

$$(r_6)_j = T_{j-1/2}^{n-1} - m_{12} - [(bt)_{j-1/2}' + (m_1 + \alpha_n)(ft)_{j-1/2} - (m_4 + \alpha_n)(uw)_{j-1/2} - m_3(w^2)_{j-1/2} + m_6(gt)_{j-1/2} - m_9(u^2)_{j-1/2} + \alpha_n (w_{j-1/2}^{n-1} u_{j-1/2} - u_{j-1/2}^{n-1} w_{j-1/2} + t_{j-1/2}^{n-1} f_{j-1/2} - f_{j-1/2}^{n-1} t_{j-1/2})]$$
(3.13)

$$(\zeta_1)_j = \frac{b_j}{h_j} + \frac{1}{2} (m_1 + \alpha_n) f_j - \frac{\alpha_n}{2} f_{j-1/2}^{n-1} + \frac{1}{2} m_6 g_j$$
(3.14a)

$$(\zeta_2)_j = -\frac{b_{j-1}}{h_j} + \frac{1}{2} (m_1 + \alpha_n) f_{j-1} + \frac{\alpha_n}{2} f_{j-1/2}^{n-1} + \frac{1}{2} m_6 g_{j-1}$$
(3.14b)

$$(\zeta_3)_j = \frac{1}{2} (m_1 + \alpha_n) v_j + \frac{\alpha_n}{2} v_{j-1/2}^{n-1}$$
(3.14c)

$$(\zeta_4)_j = \frac{1}{2} (m_1 + \alpha_n) v_{j-1} + \frac{\alpha_n}{2} v_{j-1/2}^{n-1}$$
(3.14d)

$$(\zeta_5)_j = -(m_2 + \alpha_n) u_j$$
(3.14e)

$$(\zeta_6)_j = -(m_2 + \alpha_n) u_{j-1}$$
(3.14f)

$$(\zeta_7)_k = \frac{1}{2} m_6 v_j, \quad (\zeta_8)_j = \frac{m_6}{2} v_{j-1}$$
(3.14g)

$$(\beta_1)_j = (\zeta_1)_j, \quad (\beta_2)_j = (\zeta_2)_j, \quad (\beta_3)_j = \frac{1}{2} (m_1 + \alpha_n) t_j + \frac{\alpha_n}{2} t_{j-1/2}^{n-1}$$

$$(\beta_4)_j = \frac{1}{2} (m_1 + \alpha_n) t_{j-1} + \frac{\alpha_n}{2} t_{j-1/2}^{n-1},$$

$$(\beta_5)_j = -\frac{1}{2} (m_4 + \alpha_n) u_j - m_3 w_j - \frac{\alpha_n}{2} u_{j-1/2}^{n-1}$$

$$(\beta_6)_j = -\frac{1}{2} (m_4 + \alpha_n) u_{j-1} - m_3 w_{j-1} - \frac{\alpha_n}{2} u_{j-1/2}^{n-1}$$
(3.15)

$$\begin{aligned}
 (\beta_7)_j &= -\frac{1}{2} (m_4 + \alpha_n) w_j - m_9 u_j + \frac{\alpha_n}{2} w_{j-1/2}^{n-1} \\
 (\beta_8)_j &= -\frac{1}{2} (m_4 + \alpha_n) w_{j-1} - m_9 u_{j-1} + \frac{\alpha_n}{2} w_{j-1/2}^{n-1} \\
 (\beta_9)_j &= \frac{1}{2} m_6 t_j, \quad (\beta_{10})_j = \frac{m_6}{2} t_{j-1}
 \end{aligned} \tag{3.15}$$

The boundary conditions become

$$\delta f_0 = \delta g_0 = \delta u_0 = \delta w_0 = 0, \quad \delta u_j = \delta w_j = 0 \tag{3.16}$$

The solution of the linear system given by (3.12) and (3.16) is obtained by using the block elimination method. According to this method, the system is written as

$$\mathbf{A} \begin{matrix} \delta \\ \tilde{x} \end{matrix} = \begin{matrix} r \\ \tilde{r} \end{matrix} \tag{3.17}$$

Here

$$\mathbf{A} = \begin{bmatrix} A_0 & C_0 & & & & \\ B_1 & A_1 & C_1 & & & \\ & B_j & A_j & C_j & & \\ & & B_{j-1} & A_{j-1} & C_{j-1} & \\ & & & B_j & A_j & \end{bmatrix} \quad \begin{matrix} \delta_0 \\ \delta_1 \\ \vdots \\ \delta_j \\ \vdots \\ \delta_j \end{matrix}$$

$$\begin{matrix} \tilde{x} = \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_j \\ \vdots \\ x_j \end{bmatrix} & \tilde{r}_j = \begin{bmatrix} (r_1)_j \\ (r_2)_j \\ (r_3)_j \\ (r_4)_j \\ (r_5)_j \\ (r_6)_j \end{bmatrix} & \tilde{\delta}_j = \begin{bmatrix} \delta f_j \\ \delta u_j \\ \delta v_j \\ \delta g_j \\ \delta w_j \\ \delta t_j \end{bmatrix} \end{matrix}$$

The  $A_j, B_j, C_j$  in  $\mathbf{A}$  denote  $6 \times 6$  matrices. The solution of (3.17) is obtained by the procedure described in reference 6.

#### 4.0 PRELIMINARY RESULTS

In order to check the method, we have considered the calculation of boundary-layer development on a body represented analytically by

$$y = f(x,z) = B \left[ 1 - \left( \frac{x}{L} \right)^2 \right]^{1/2} \left[ 1 - \left( \frac{z}{H} \right)^2 \right]^{1/2} \quad (4.1)$$

This is a double-elliptic ship having round edges except for the sharp corners at  $x = \pm L$  and  $z = \pm H$  as shown in figure 4. The potential-flow solutions for the body of  $L:H:B = 1:0.1:0.125$  were obtained from the Douglas-Neumann

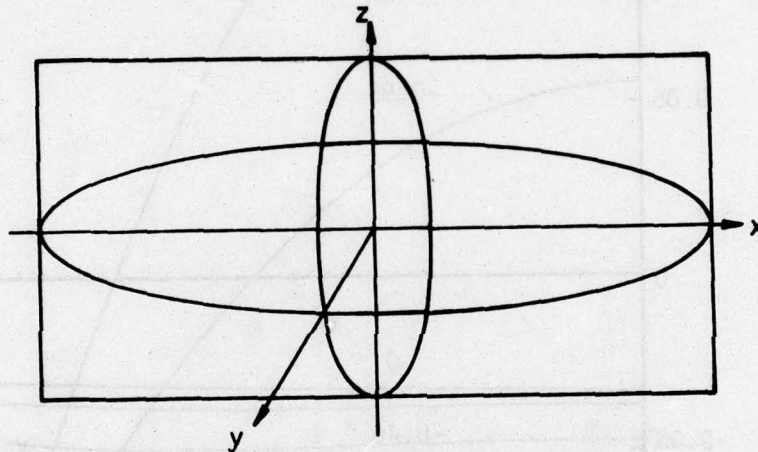


Figure 4. The double-elliptic ship given by (4.1).

computer program for three-dimensional flows (see figure 5). To get the solutions, 120 control elements on the surface were used, 12 along the x-direction and 10 along the z-direction.

Before we describe our boundary-layer calculations for the ship given by (4.1), it is useful to discuss the pressure distribution for this body. As can be seen from figure 5, the streamwise pressure gradient is initially favorable in the bow region and then adverse up to the midpoint of the body. This is followed by a region of favorable pressure and then by a sharp adverse pressure gradient very close to the stern. The crosswise pressure gradient varies in a more complex manner. Near the bow, the pressure decreases as  $z$  increases. As the flow moves downstream, the location of the minimum pressure moves up and finally reaches the water surface. This implies that the cross-flow will reverse its sign a number of times. As a result, in some regions we

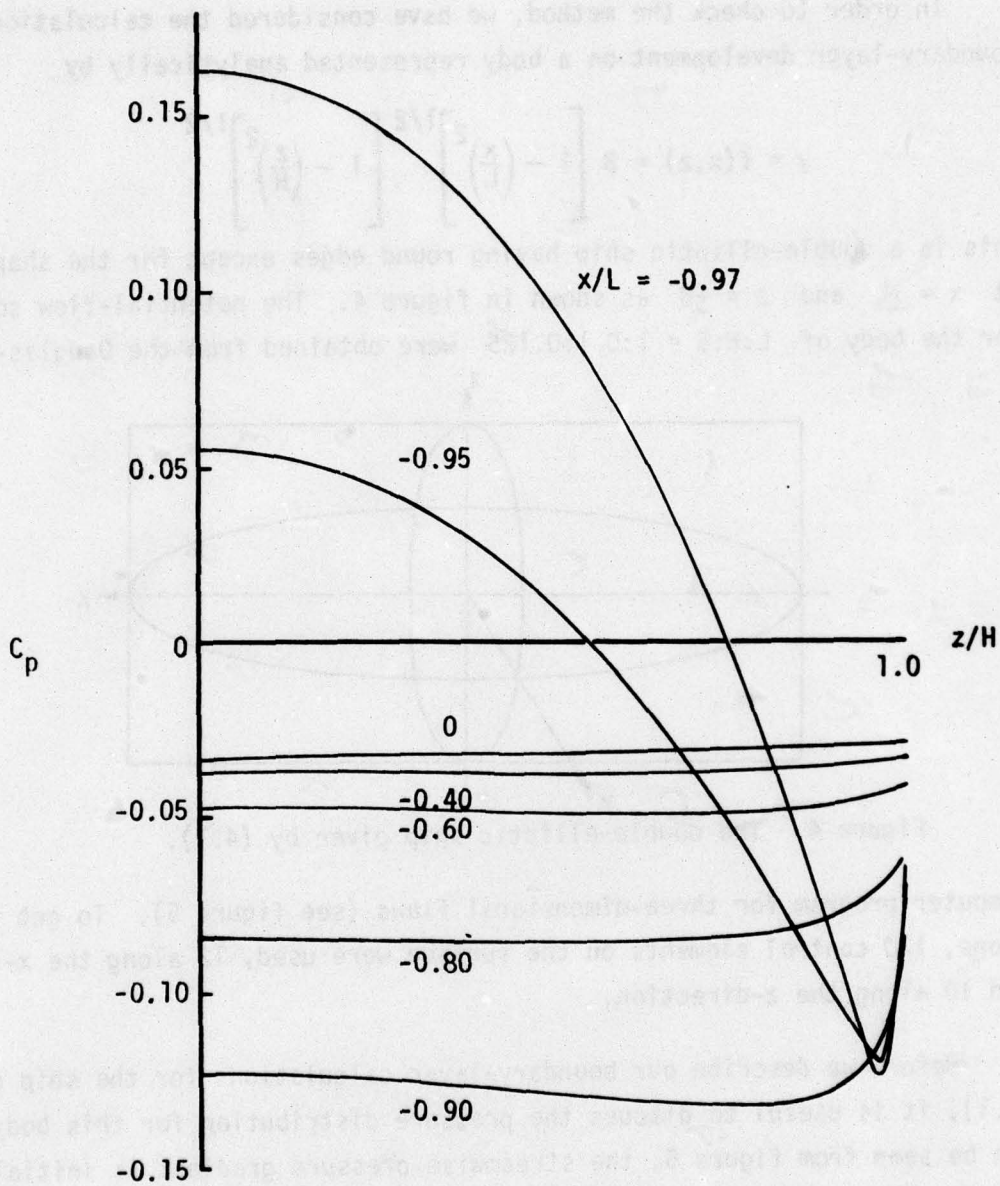


Figure 5. Pressure distribution for a double-elliptic ship form of 1:0.1:0.125.

may expect the cross-over type of the crossflow profiles (i.e., where the direction of the crossflow changes across the boundary layer). This conclusion is drawn from considering the pressure gradients only. The real situation is more complicated because, in addition, there are the upstream effects and the curvature effects on the flow characteristics.

The boundary-layer computation starts from  $x_1/L = -0.97$ . The region between this point and the bow is not considered for two reasons: first, the pressure distribution in this region is not available from the potential-flow computer program and second, the sharp corner at  $x_1/L = -1.0$  and  $z/H = 1.0$  implies an infinite curvature and, hence, the boundary-layer assumptions are no longer valid in that region. In our boundary-layer calculations, we have used 32 points along the  $x_1$ -direction and 12 points along the  $x_2$ -direction. In the normal direction, we have taken approximately 40 points. The flow was assumed to be laminar at first and then was specified to be turbulent at the third ( $x_1/L$ )-station, namely at  $x_1/L = -0.925$ .

Some of the computed results for  $R_L = 10^7$  are shown in figures 6 to 10. Figures 6, 7, and 8 show the streamwise variation of the local skin-friction coefficient, the shape factor, and the Reynolds number based on the momentum thickness for  $z/H = 0, 0.1, 0.2,$  and  $0.3,$  respectively. Figures 9 and 10 show some of the typical streamwise and crosswise velocity profiles along  $z/H = 0.3$ . In these two figures, the profiles for  $z/H = 0.0$  correspond to the streamwise attachment solutions ( $z/H = 0$ ). We see from figure 3 that the flow along the attachment line separates at approximately  $x/L = 0.95$ , or near the stern, where the pressure gradient is sharply adverse. As  $z/H$  increases, the point of separation moves upstream. Along the line,  $z/H = 0.3$ , the flow separates at about  $x/L = 0.80$ , where the pressure is almost constant. This unusual behavior results from the crossflow effect. A better marching procedure is required to remedy the situation as is discussed in the next section.

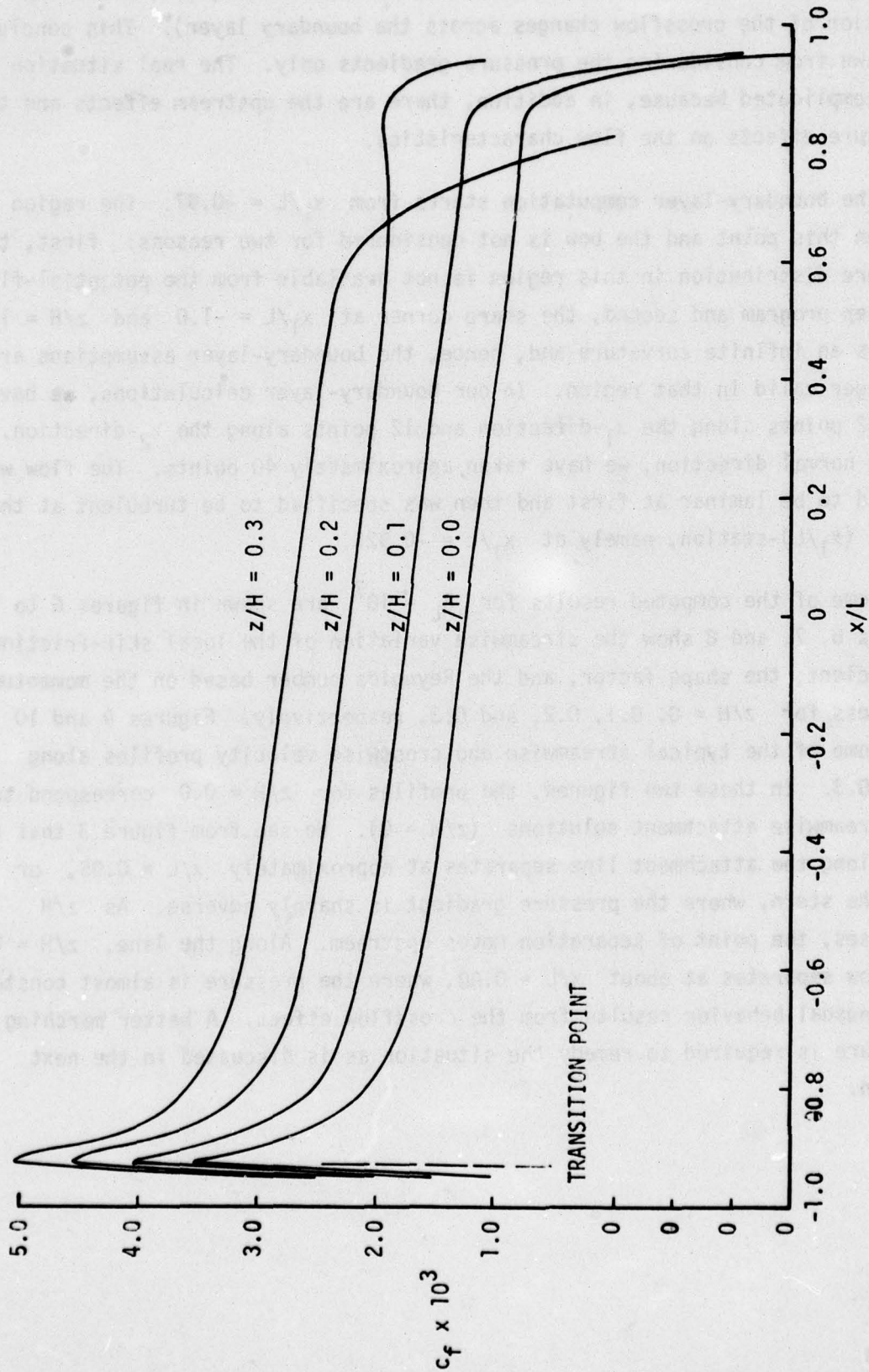


Figure 6. Variation of skin-friction coefficient along the streamwise direction for the double-elliptic ship.

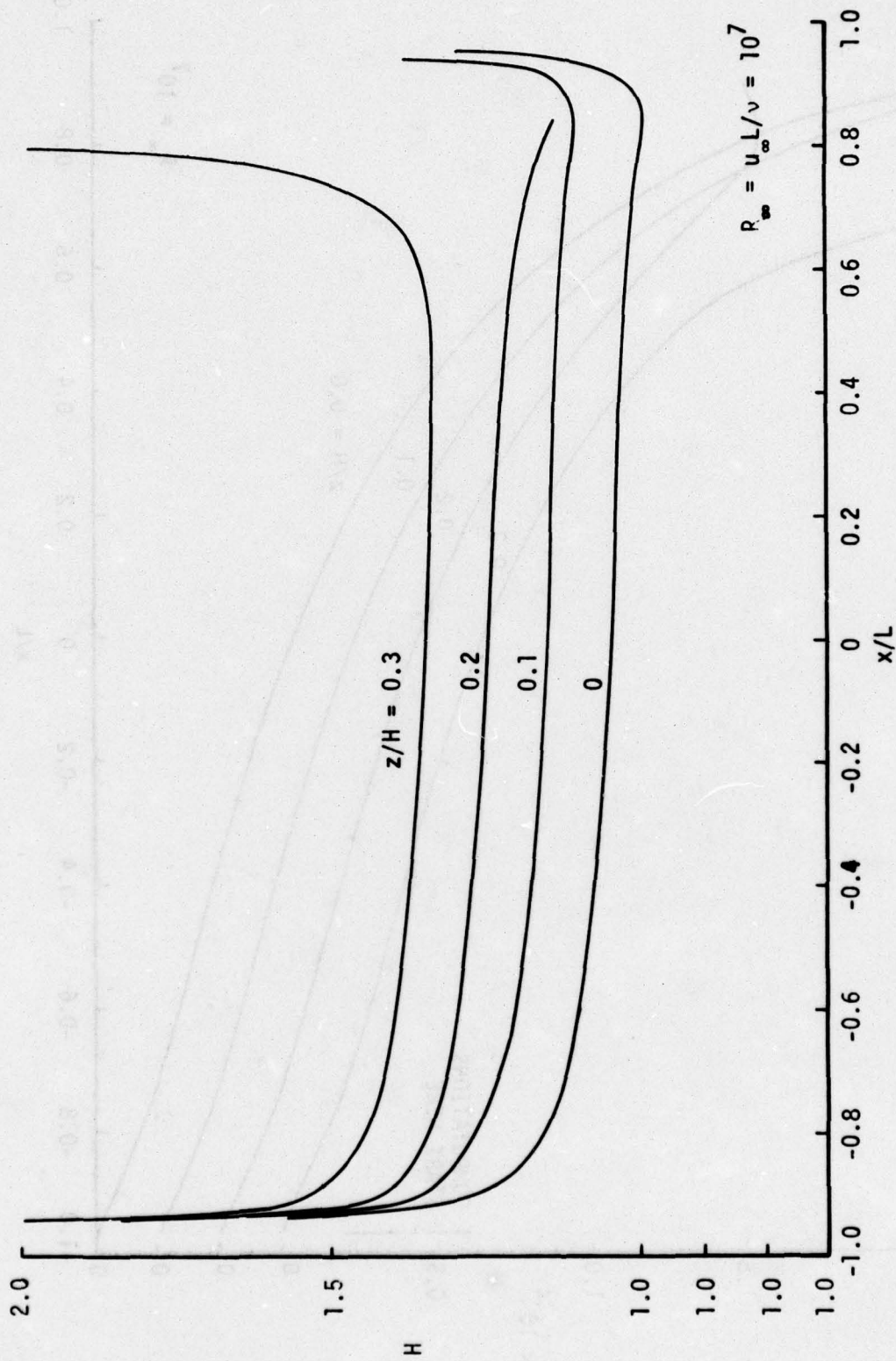


Figure 7. Variation of shape factor  $H$  on the streamwise direction.

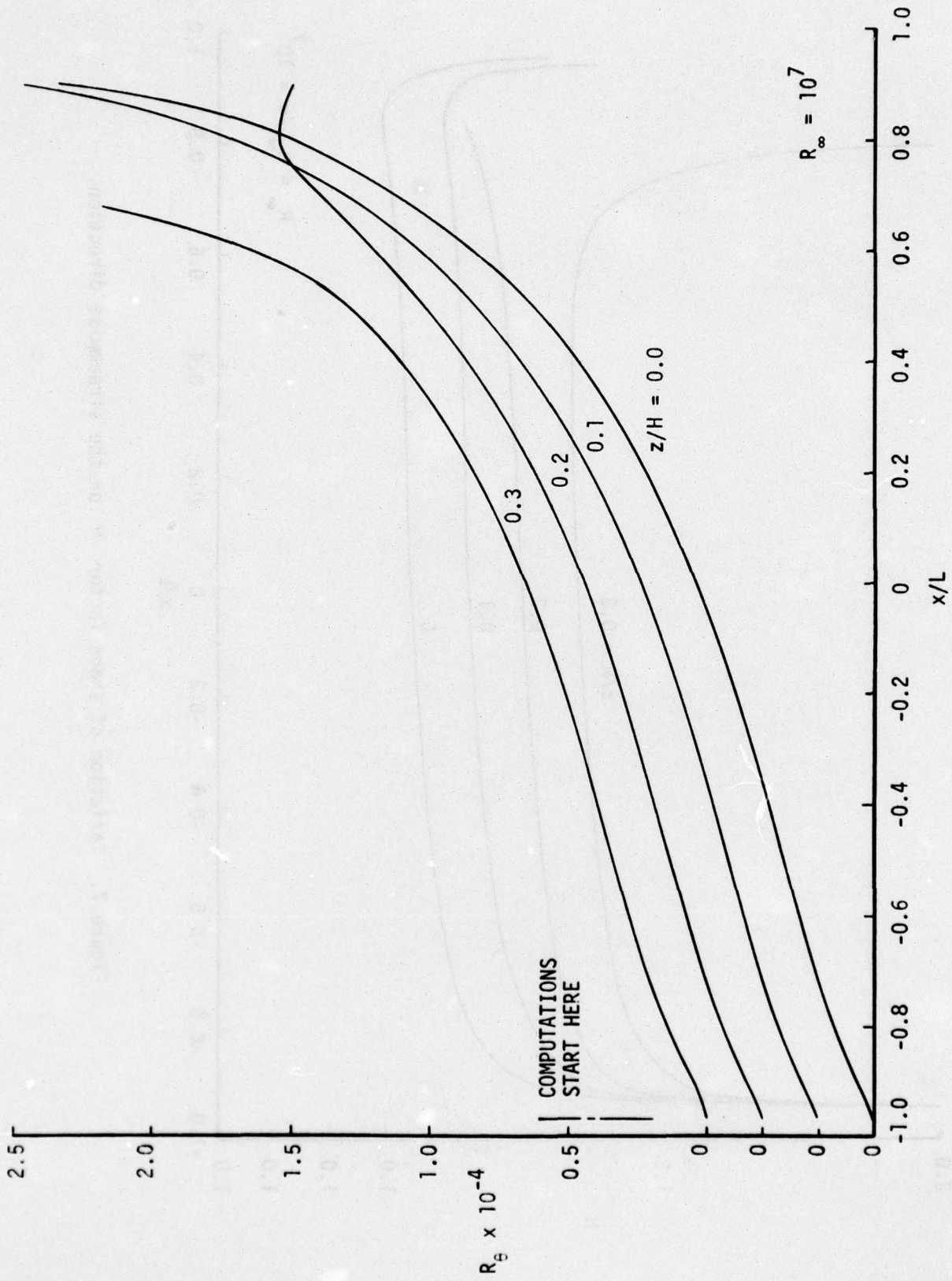


Figure 8. Variation of  $R_e$  along the streamwise direction for the double-elliptic ship.

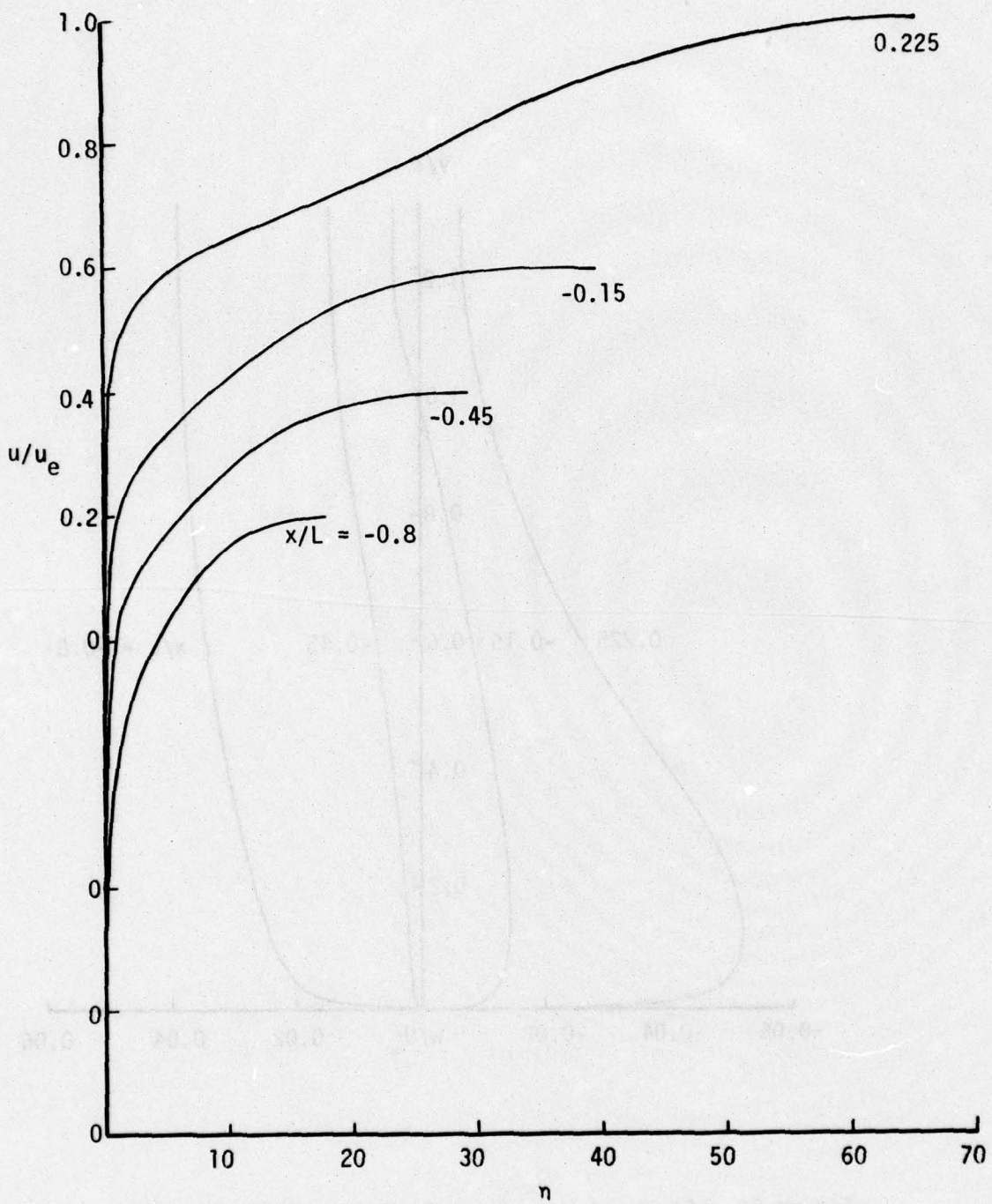


Figure 9. Computed streamwise velocity profiles at  $z/H = 0.3$ .

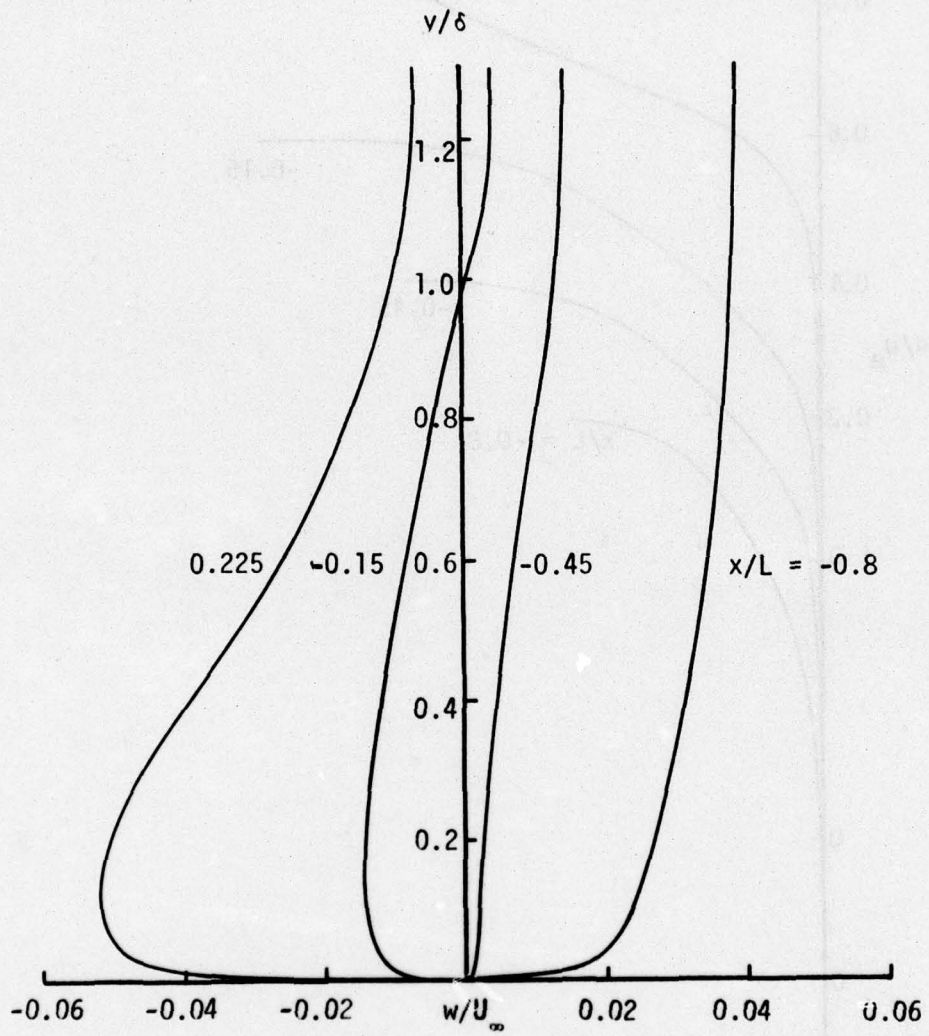


Figure 10. Computed spanwise velocity profiles at  $z/H = 0.30$ .

## 5.0 DISCUSSION

The present effort is one step towards developing a general method for computing three-dimensional boundary layers on ship hulls. Obviously further studies are needed in the following areas:

1. A better numerical procedure for handling the computation of flows in which there are regions of negative crossflow. Our present calculations here and in the related problems<sup>(7)</sup> demonstrate that in regions of the flow field where there is a significant layer of crossflow opposing the marching direction, oscillations can occur in the generated solution. These are caused by the stability condition on the difference equations which does not permit the correct domain of dependence of the differential equation to be utilized at the particular point being computed.
2. A better coordinate system than the one used in this study is required. As discussed in Section 2.3, a nonorthogonal system is appropriate for ship hulls. Although the governing equations are slightly more complicated than those for an orthogonal system, that complication does not cause any difficulties in their numerical solution. The big advantage of the nonorthogonal system is the control it allows over the coordinate net spacing. In orthogonal systems, a family of surfaces which intersects the hull surface has been selected, the orthogonals are consequently determined and, for some ship hulls, can lead to coordinate nets which are unacceptably sparse at some areas.

These necessary studies indicated above are presently in progress. A recently developed procedure, which looks particularly promising, uses the notation of domain of dependence and follows the characteristics of the locally plane flow. In this procedure, the direction of  $w$  at each  $n_j$ -grid point is examined and the difference equations formulated accordingly. In addition, the new procedure utilizes a modification of the Box scheme similar to the zig-zag differencing scheme proposed by Krause et al.<sup>(8)</sup> and successfully used by Cebeci<sup>(9)</sup> to compute the unsteady flow generated by an impulsively started circular cylinder. The finite-difference equations, with the

zig-zag differencing used to advance in time, have been solved from time  $t = 0$  until the time the separation point has moved from the rear of the cylinder to its final steady-state value of  $\theta = 106^\circ$ . The flow reversal has been accounted for satisfactorily and the calculations extended further in time than many previous studies.

An example of a nonorthogonal coordinate system is given by the intersection of meridian planes and parallel cuts. Consider the hull as given in the usual Cartesian coordinate system, that is,  $x$  along the ship axis,  $y$  and  $z$  in the crossplane (see Figure 11). We select  $x$  as one of the coordinates and the polar angle  $\phi$  in the  $yz$ -plane as the other coordinate. It is only necessary to define the hull as a family of points in the planes  $x = \text{constant}$ . Then the data is spline-fitted in each crossplane as  $y$  vs  $\phi$  and  $z$  vs  $\phi$ , and interpolated for constant values of  $\phi$ . Then another set of spline fits in planes  $\phi = \text{constant}$  for  $y$  vs  $x$  and  $z$  vs  $x$  completes the definition of the coordinate system. The lines formed by the intersection of the planes  $x = \text{constant}$  and  $\phi = \text{constant}$  with the hull surface constitute the nonorthogonal coordinate net on the surface, and the third boundary-layer coordinate is taken as the distance normal to the surface in accordance with first-order boundary-layer approximations.

Since the spline-fitting also yields derivatives, the metric coefficients and the geodesic curvatures of the coordinate lines can be calculated by using the formulas given below.

The metric coefficients:

$$h_1^2 = 1 + \left(\frac{\partial z}{\partial x}\right)_\phi^2 + \left(\frac{\partial y}{\partial x}\right)_\phi^2$$

$$h_2^2 = \left(\frac{\partial y}{\partial \phi}\right)_x^2 + \left(\frac{\partial z}{\partial \phi}\right)_x^2$$

The angle between the coordinate lines

$$\cos\theta = \frac{1}{h_1 h_2} \left[ \left(\frac{\partial y}{\partial \phi}\right)_x \left(\frac{\partial y}{\partial x}\right)_\phi + \left(\frac{\partial z}{\partial \phi}\right)_x \left(\frac{\partial z}{\partial x}\right)_\phi \right]$$

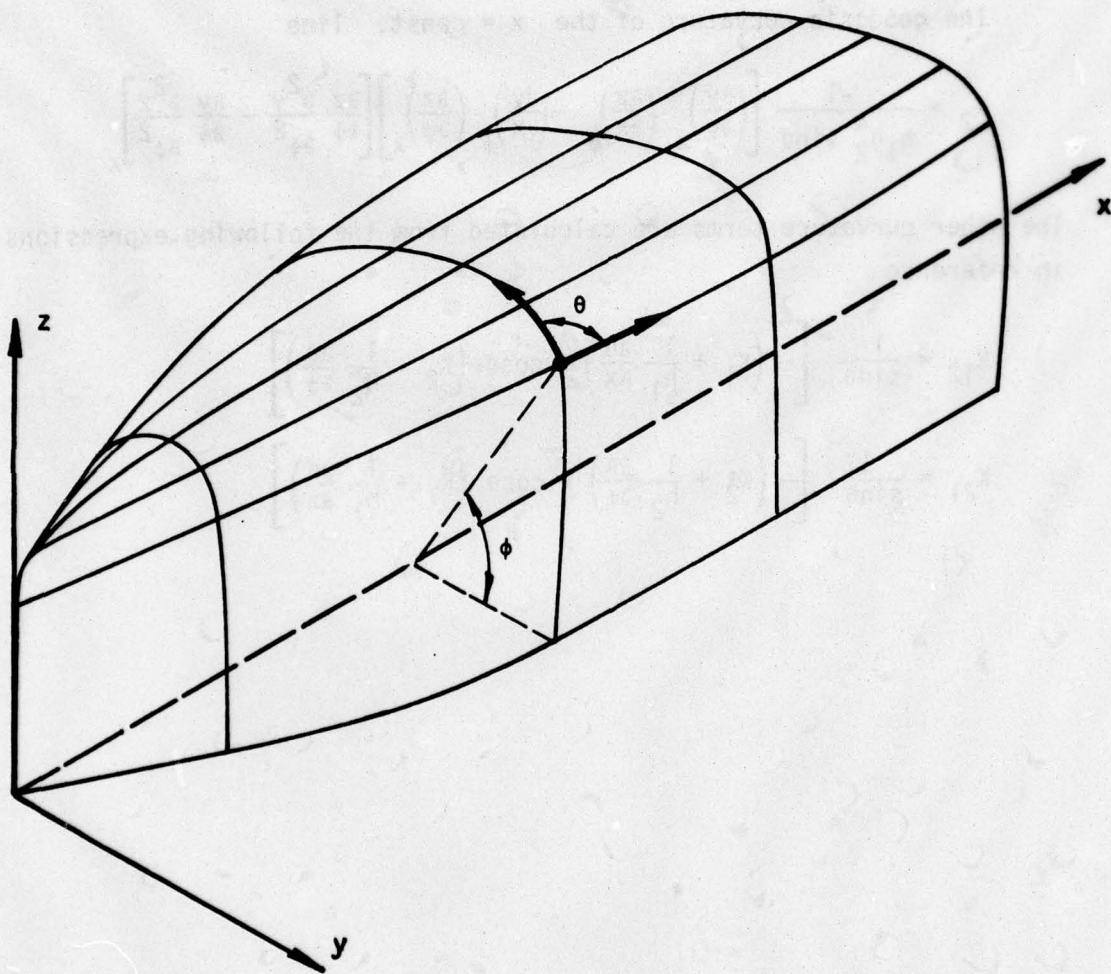


Figure 11. Nonorthogonal coordinate system for the ship hull.

The geodesic curvature of the  $\phi = \text{const.}$  line

$$K_1 = \frac{1}{h_1 h_2 \sin\theta} \left\{ \left[ \left( \frac{\partial y}{\partial \phi} \right)_x \left( \frac{\partial z}{\partial x} \right)_\phi - \left( \frac{\partial y}{\partial x} \right)_\phi \left( \frac{\partial z}{\partial \phi} \right)_x \right] \cdot \left( \frac{\partial z}{\partial x} \frac{\partial^2 y}{\partial x^2} - \frac{\partial y}{\partial x} \frac{\partial^2 z}{\partial x^2} \right)_\phi \right. \\ \left. + \left[ \left( \frac{\partial y}{\partial \phi} \right)_x \left( \frac{\partial^2 y}{\partial x^2} \right)_\phi + \left( \frac{\partial z}{\partial \phi} \right)_x \left( \frac{\partial^2 z}{\partial x^2} \right)_\phi \right] \right\}$$

The geodesic curvature of the  $x = \text{const.}$  line

$$K_2 = \frac{-1}{h_1 h_2^4 \sin\theta} \left[ \left( \frac{\partial y}{\partial \phi} \right)_x \left( \frac{\partial z}{\partial x} \right)_\phi - \left( \frac{\partial y}{\partial x} \right)_\phi \left( \frac{\partial z}{\partial \phi} \right)_x \right] \left[ \frac{\partial z}{\partial \phi} \frac{\partial^2 y}{\partial \phi^2} - \frac{\partial y}{\partial \phi} \frac{\partial^2 z}{\partial \phi^2} \right]_x$$

The other curvature terms are calculated from the following expressions given in reference

$$K_{12} = \frac{1}{\sin\theta} \left[ - \left( K_1 + \frac{1}{h_1} \frac{\partial \theta}{\partial x} \right) + \cos\theta \left( K_2 + \frac{1}{h_2} \frac{\partial \theta}{\partial \phi} \right) \right]$$

$$K_{21} = \frac{1}{\sin\theta} \left[ - \left( K_2 + \frac{1}{h_2} \frac{\partial \theta}{\partial \phi} \right) + \cos\theta \left( K_1 + \frac{1}{h_1} \frac{\partial \theta}{\partial x} \right) \right]$$

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APPENDIX A  
CALCULATION OF GEOMETRIC PARAMETERS  
OF THE COORDINATE SYSTEM

As discussed in Section 2.3, the geometrical parameters and external velocity components along the adopted coordinates require the quantities  $f_x$ ,  $f_z$ , etc., on a ship hull. A ship hull is usually described in the form of a family of curves. For example, it may be given as the curves generated by the intersection of the hull with planes ( $x = \text{constant}$ ) perpendicular to the free surface at various stations along the ship hull. The problem is then to find, with sufficient accuracy, the derivatives at an arbitrary point on the hull. When the ship hull is described analytically, then these quantities and consequently the geometric parameters of the coordinate system can be computed exactly. For arbitrary ship hulls, they must be computed numerically.

The exact solutions for the three-dimensional velocity potential is not available even on a double ship model, where the free surface is treated as a rigid plane. It is usually given experimentally or calculated numerically by the method of Hess and Smith. In this method the singularities (e.g., source, dipole, vortex, etc.) are distributed on the body surface, and the governing equation is transformed from Laplace's equation in the field into a Fredholm integral equation over the body surface. This integral equation expresses the condition of zero normal velocity on the surface. To solve the integral equation numerically the body surface is divided into a large number of small surface elements, which may be plane or curved. On each element the strength of a singularity is assumed to be either constant or variable in such a way that the overall distribution may be defined by a single unknown parameter for each element. The normal-velocity boundary condition is required to hold at one so-called control point on each element. This yields a number of linear equations equal to the number of unknown parameters defining the singularity distribution. This set of equations is solved by either direct elimination or Gauss-Seidel iteration.

The direct elimination is accomplished by the Douglas-developed SOLVIT routine, which is applicable to a large full matrix that will not fit into high-speed core storage. Recently this method has been shown to be much faster than standard IBM subroutines. Once the equations have been solved, the resulting singularity distribution is then used to calculate values of fluid velocity and pressure at the control points. As can be easily understood the

control points of the potential-flow computations will not coincide with the points where the boundary layer is to be computed. There is, therefore, a need to develop a numerical method to interpolate a function of two independent variables.

There are a number of well-known numerical techniques that can be used for interpolation and differentiation of a function of one, two, or more variables. Here we shall make use of the surface splines for interpolating and obtaining the first derivatives of a function of two variables and the Fourier series expansion for finding the second derivatives.

The surface spline is based on the small deflection equation on an infinite elastic plate that deforms in bending only. The procedure is to represent the deflection due to a point load. The deflection of the entire spline is then taken as the sum of all the point load distributions subject to the boundary condition that the surface should become flat a long distance from the applied loads. This results in a system of linear equations which is solved for the spline coefficients. The final system of equations is given below. (For details of derivation, see ref. 10.)

$$\sum_{i=1}^N F_i = 0 \quad (\text{A.1})$$

$$\sum_{i=1}^N x_i F_i = 0 \quad (\text{A.2})$$

$$\sum_{i=1}^N z_i F_i = 0 \quad (\text{A.3})$$

$$a_0 + a_1 x_j + a_2 z_j + \sum_{i=1}^N F_i r_{ij}^2 \ln r_{ij}^2 = w_j, \quad j = 1, N \quad (\text{A.4})$$

where

$$r_{ij}^2 = (x_j - x_i)^2 + (z_j - z_i)^2 \quad (\text{A.5})$$

If the function being interpolated has one or two planes of symmetry or antisymmetry, then use can be made of images to improve the accuracy of the fit or to reduce the number of simultaneous equations. If  $w$  is symmetrical about  $x_0$ , the system of equations becomes

$$\sum_{i=1}^N \bar{F}_i = 0 \quad (\text{A.6})$$

$$\sum_{i=1}^N z_i \bar{F}_i = 0 \quad (\text{A.7})$$

$$a_0 + a_2 z_j + \sum_{i=1}^N (r_{ij}^2 \ln r_{ij}^2 + \bar{r}_{ij}^2 \ln \bar{r}_{ij}^2) \bar{F}_i = w_j \quad j = 1, N \quad (\text{A.8})$$

where

$$\bar{r}_{ij}^2 = (x_i + x_j - 2x_0) + (z_j - z_i)^2 \quad (\text{A.9})$$

Similar results can be obtained for  $z$  symmetry and for both  $x$  and  $z$  symmetry. The system of equations (A.1) to (A.5) is solved using the above-described SOLVIT routine.

Equation (A.4) can be differentiated to give the first derivatives of  $w$ , or the slopes of the spline

$$\frac{\partial w_j}{\partial x} = a_1 + 2 \sum_{i=1}^N F_i (1 + \ln r_{ij}^2) (x_j - x_i) \quad (\text{A.10})$$

$$\frac{\partial w_j}{\partial z} = a_2 + 2 \sum_{i=1}^N F_i (1 + \ln r_{ij}^2) (z_j - z_i) \quad (\text{A.11})$$

Though (A.4) can be successively differentiated to give higher-order derivatives, for example

$$\frac{\partial^2 w_j}{\partial z^2} = 2 \sum_{i=1}^N F_i (1 + \ln r_{ij}^2) + 4 \sum_{i=1}^N F_i (z_j - z_i)^2 / r_{ij}^2 \quad (\text{A.12})$$

they are indeterminate as  $(x_j, z_j) \rightarrow (x_i, z_i)$ . For removing this problem, Harder and Desmarais<sup>(10)</sup> suggested replacing the term  $r^2 \ln r^2$  in (A.4) with  $r^2 \ln (r^2 + \epsilon)$ . Physically we apply distributed loads instead of point loads on the plate. The new generated spline surface passes through  $N$  points and has all derivatives everywhere. The difficulty of using this alternative is to determine the distribution of  $\epsilon$  over the spline surface and Harder and Desmarais<sup>(10)</sup> did not give the relation between  $\epsilon$  and other known quantities explicitly.

Once the values of  $w$  have been found for some constant  $x_i$  (or  $z_i$ ) using the surface spline, the derivatives can be obtained by using the cubic linear spline because  $w$  is now a function of one variable. The cubic spline, in which a piecewise, third-order polynomial is used to describe a function between two discrete points, has been widely used to interpolate and find derivatives of a function of one variable because it is easily to calculate and gives generally satisfactory results.

Once the values of  $w$  and its first derivatives  $\partial w / \partial x$  and  $\partial w / \partial z$  have been found for some constant  $x_i$  (or  $z_i$ ) using the surface spline, the second derivatives can be obtained by the Fourier series expansion method developed by James<sup>(11)</sup> at Douglas Aircraft Company. The method has the advantage that it also smoothes the data and the results are not sensitive to irregularity of the input data.

## APPENDIX B

### USER'S MANUAL

The present computer program is written such that various orthogonal coordinate systems can be used. In Case I, the user provides the body geometry and the corresponding inviscid velocity components on the body surface in terms of the Cartesian coordinate system. The geometrical parameters  $h_1$ ,  $h_2$ ,  $K_1$ ,  $K_2$  are computed internally for the coordinate system used in this report. In Case II, the user supplies the geometrical parameters and the inviscid velocity components  $u_e$  and  $w_e$  along the coordinate lines. The input data for each case is described below.

#### Case I

1. Read the values of ICASE, NXT and NZT in I5 Format, three variables on one card.

ICASE     1

NXT       Total number of x-stations for computing boundary layers,  
 $3 \leq \text{NXT} \leq 35$

NZT       Total number of z-stations for computing boundary layers,  
 $3 \leq \text{NZT} \leq 20$

2. Read the values of  $X(I)$ ,  $I=1,2,\dots,\text{NXT}$  in F10.0 Format, one set of eight per card.

$X(I)$      Values of  $x$  where the boundary layers are computed in meter or ft.

$X(I)$  denote the Cartesian coordinates on the body.

3. Read the values of  $Z(K)$ ,  $K=1,2,\dots,\text{NZT}$  in F10.0 Format, one set of eight per card.

$Z(K)$      Values of  $z$  where the boundary layers are computed in meter or ft.

$Z(K)$  are in the Cartesian coordinates on the body. In order to get better results, a sudden change in  $z$  interval (i.e.,  $\Delta z_K = z_K - z_{K-1}$ ) should be avoided.

4. Read the values of Y, UE, VE, and WE. The variables are punched in F10.0 format, one set of four per card in the following order:

{[Y(I,K),UE(I,K),VE(I,K),WE(I,K),K=1,NZT],I=1,NXT}

Y y-value at  $(x_i, z_k)$  on the body surface.  
UE Inviscid velocity component along x-direction at  $(x_i, z_k)$   
VE Inviscid velocity component along y-direction at  $(x_i, z_k)$   
WE Inviscid velocity component along z-direction at  $(x_i, z_k)$

Here x, y, and z are in Cartesian coordinates, Y is in meter or feet and UE, VE, and WE are in m/sec or ft/sec.

5. Read TITLE in 20A4 format. It is used for identifying the problem under study. It should not be greater than the length of 80 including blank.
6. Read the variables, NXT, NZT, NXTRAN, VISC, and UREF in one card. The first three variables are in I5 Format, the remaining variables in F10.0 Format.

NXT Total number of x-stations for computing boundary layers,  
 $3 \leq \text{NXT} \leq 35$

NZT Total number of z-stations for computing boundary layers,  
 $3 \leq \text{NZT} \leq 20$

NXTRAN Number of x-stations at which the flow changes from laminar to turbulent. If the flow is laminar, set the value arbitrarily greater than NXT.

VISC Kinematic viscosity of the fluid, in  $\text{m}^2/\text{sec}$  or  $\text{ft}^2/\text{sec}$ .

UREF A reference velocity, in m/sec or ft/sec.

### Case II

In Case II, the input data are almost the same as for Case I except for the following changes:

- a. ICASE set equal to 2.
- b. X(I) and Z(K) are given in the corresponding coordinate system instead of the Cartesian coordinate system.

c. Replace the data described in item 4 by

{[UE(I,K),WE(I,K),H1(I,K),H2(I,K),CK1(I,K),CK2(I,K),K=1,NZT],  
I=1,NXT)

The variables are punched in F10.0 format, six variables in one card

UE Inviscid velocity component along x-direction

WE Inviscid velocity component along z-direction

H1 Metric coefficient along x-direction,  $h_1$

H2 Metric coefficient along z-direction,  $h_2$

CK1 Geodesic curvature coefficient  $K_1$ , see (2.5)

CK2 Geodesic curvature coefficient  $K_2$ , see (2.5)

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