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Winston K Pendleton

WINSTON K. PENDLETON
Lt Colonel, USAF
Chief Scientist

Gordon L Hermann

GORDON L. HERMANN
Lt Colonel, USAF
Deputy Commander

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ABSTRACT

An axisymmetric numerical model of a liquid spray is presented. The model consists of coupled sets of partial differential equations for the gas phase and ordinary differential equations for the liquid phase. A numerical method based on the MAC method is presented and calculations show good agreement with experiment. The effect of a ceiling on the entrainment performance of a spray is found to be negligible. The examination of the calculated pressure field shows that pressure does not play an important role in the flow and may be ignored. This is in agreement with experiment and contributes additional validation to existing models which ignore this effect.

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LIST OF SYMBOLS

C_D	drag coefficient of a sphere
C_N	nozzle design parameter
d_0	inside diameter of nozzle (mm)
D	droplet diameter (mm)
D_t	diameter of spray envelope (m)
f_p	fraction of mass flow through the nozzle assigned to given trajectory
f_r	radial component of drag force acting on droplet (N)
f_z	axial component of drag force acting on droplet (N)
F_r	radial volume force term of momentum source (N/m^3)
Fr_{av}	average radial force acting on a particle in source cell (N)
F_z	axial volume force term of momentum source (N/m^3)
g	acceleration of gravity (m/s^2)
H_ℓ	spray height (m)
m	droplet mass (kg)
N	unit normal to boundary
N_p	number of particles in source cell at given instant in time
P	pressure (N/m^2)
ΔP_N	water delivery pressure (N/m^2)
Q_a	volume flow of air entrained into spray (m^3/min)
Q_w	volume flow of liquid through spray nozzle (ℓ/min)
r	variable of integration inside spray envelope
r_0	radial coordinate of particle trajectory
R	radial coordinate
Re	Reynolds number
RE	Reynolds number of nondimensionlization

t	time	(sec)	
u_r	radial velocity of particle	(m/s)	
u_z	axial velocity of particle	(m/s)	
U_0	particle injection velocity	(m/s)	
V_r	radial component of velocity	(m/s)	
V_z	axial component of velocity	(m/s)	
Z	axial coordinate	(m)	
Z_0	axial coordinate of particle trajectory	(m)	
ρ	density of gas	(kg/m ³)	
ρ_L	density of liquid	(kg/m ³)	
θ	initial angle of ejection of particle	(degrees)	
θ_{max}	maximum initial angle of particle ejection	(degrees)	
η	efficiency of spray		
δs	arc length of trajectory in source cell		
δt	time for particle to pass through source cell		
ν	kinematic viscosity of gas	(m ² /s)	
Δ	indicates incremental quantity		

1. INTRODUCTION

The application of liquid sprays is made in many different fields. Examples of a few are fire suppression, combustion, ventilation, and the dispersion of heavy, possibly toxic or flammable, gases. Interest at VKI in the applicability of water sprays for use as water curtains for the dispersion of heavy gases has mandated a need for better prediction methods for the analysis/design of water sprays.

The configuration for the spray to be analysed in this study is shown in figure 1. Liquid particles are ejected through a nozzle into an environment which is at rest in the absence of the spray. The aerodynamic drag acting on the liquid particles results in a loss in momentum of the particles. Since momentum must be conserved, the momentum loss is translated into a momentum gain by the fluid, causing the fluid to be set in motion. Air is thereby entrained into the spray (in much the same way as a jet) resulting in a two phase plume.

Modeling of this phenomenon has been done in the past. A one dimensional model (Refs. 3,5) has been developed at VKI and has shown good results though its applicability for the inclusion of boundary effects is limited. Two dimensional models, existing in the literature (Refs. 1,4,7), are much more able to handle the effects of boundaries. In this light, it is found desirable to have such a two dimensional model at VKI to complement the range of applicability of the existing one dimensional model. Thus, this project concerns the implementation of an axisymmetric numerical model into a computer code. The basic approach is similar to that followed in references 1 and 4.

In this report the model used is outlined, the numerical method for the solution of the model equations is described, and, finally, the results presented and discussed. The implemented computer code is included as an appendix to this report.

2. THE SPRAY MODEL

2.1 Introduction

The model of the spray consists of two distinct sets of equations, one set governing the gas phase and another governing the liquid phase. Linkage between these two sets of equations accounts for the following two physical phenomena : the first concerns the momentum transfer between liquid phase and gas phase, while the second involves the modification of the particle trajectories by the motion of the gas phase.

The gas phase may be modelled as a continuum using an Eulerian approach while the liquid phase, using a Lagrangian approach is modelled by considering a finite set of particles of varying size/initial trajectory.

2.2 The gas phase model

2.2.1 Gas phase equations

Since the gas phase occupies the most significant portion of the flow domain, it may be treated as a continuum. Making the standard assumptions of incompressibility and Newtonian fluid the following form of the Navier Stokes equations may be used. Here, due to the axisymmetric nature of the problem, the equations are written in cylindrical coordinates :

Continuity

$$\frac{1}{R} \frac{\partial}{\partial R} (RV_r) + \frac{\partial V_z}{\partial z} = 0 \quad (2.1)$$

Momentum

r-component

$$V_r \frac{\partial V_r}{\partial R} + V_z \frac{\partial V_r}{\partial z} = - \frac{\partial P}{\partial R} + \nu \left[\frac{\partial}{\partial R} \left\{ \frac{1}{R} \frac{\partial}{\partial R} (R V_r) \right\} + \frac{\partial^2 V_r}{\partial z^2} \right] + F_r \quad (2.2)$$

z-component

$$V_r \frac{\partial V_z}{\partial R} + V_z \frac{\partial V_z}{\partial z} = - \frac{\partial P}{\partial z} + \nu \left[\frac{1}{R} \frac{\partial}{\partial R} \left\{ R \frac{\partial V_z}{\partial R} \right\} + \frac{\partial^2 V_z}{\partial z^2} \right] + F_z \quad (2.3)$$

where V_r , V_z , P are the radial and axial component of velocity, and the pressure respectively; R and z are the radial and axial coordinate and ρ and ν represent the fluid density and kinematic viscosity. The terms F_r and F_z are the volume force terms which account for the contribution of momentum per unit volume from the liquid phase.

2.2.2 Boundary conditions for the gas phase

Considered here are three types of boundaries; a single axis of symmetry boundary, free boundaries and wall boundaries. These are shown in figure 1.

Velocity boundary conditions

Axis of symmetry

At the axis of symmetry the radial component of velocity must be equal to zero because of mass conservation and the symmetry assumption. Symmetry also requires that there be no shear stress in the axial direction, requiring the derivative of the axial velocity with respect to the radial coordinate to be zero.

$$V_r = 0 \quad (2.4)$$

$$\frac{\partial V_z}{\partial R} = 0 \quad (2.5)$$

Wall boundary

Along the wall boundaries the usual conditions for an impermeable, non-slip wall are used :

$$V_r = V_z = 0 \quad (2.6)$$

Free boundary

In the flow domain considered the free boundary is taken to be far from the spray domain. With this assumption the following approximate conditions may be applied. The velocity tangential to the boundary is small, and may be assumed equal to zero. Applying continuity, the boundary condition for the component of velocity normal to the boundary may be obtained.

Vertical free boundary

$$V_z = 0 \quad (2.7)$$

$$\frac{\partial (RV_r)}{\partial R} = 0 \quad (2.8)$$

Horizontal free boundary

$$V_r = 0 \quad (2.9)$$

$$\frac{\partial V_z}{\partial R} = 0 \quad (2.10)$$

Pressure boundary conditions

The pressure boundary conditions may be derived from the momentum equations 2.2 and 2.3. Since most gases have small viscosity it is permissible here to ignore the viscous terms of the momentum equations in deriving the pressure boundary conditions.

The resulting derivation shows that for the wall and axis of symmetry boundaries, the normal derivative of the pressure is equal to zero.

The resulting boundary condition on the free boundary is a bit more complicated using this approach. However, making use of experimental observation which has shown that variation of the pressure field is small throughout the flow domain, the pressure at the free boundary is assumed to be constant.

Wall or axis of symmetry

$$\frac{\partial P}{\partial N} = 0 \quad (2.11)$$

Free boundary

$$P = P_{ref} \quad (2.12)$$

2.3 The particle phase model

2.3.1 Particle equations

The liquid phase is modelled as a distribution of droplets of distinct sizes and trajectories. Using a Lagrangian approach individual particles are followed from injection until hitting the floor. By considering the number of particles of similar sizes/trajectories, the magnitude of droplet-gas

momentum exchange throughout the flow field may be determined. This approach assumes that the particles do not interact with each other, i.e., no collisions or particle break up. Making the additional assumptions that the particles are spherical and non-evaporating, the following equations of motion may be written.

$$m \frac{du_r}{dt} = - f_r \quad (2.13)$$

$$m \frac{du_z}{dt} = - f_z \pm mg \quad (2.14)$$

$$\frac{dr_0(t)}{dt} = u_r \quad (2.15)$$

$$\frac{dz_0(t)}{dt} = u_z \quad (2.16)$$

where u_r , u_z are the particle radial and axial velocities and r_0 and z_0 are the radial and axial position of the particles. "m" is the mass of the particle calculated as follows :

$$m = \rho_L \frac{\pi}{6} D^3 \quad (2.17)$$

where ρ_L is the density of the liquid and D is the particle diameter. f_r and f_z are the radial and axial component of drag force, related to F_r and F_z in equations 2.2 and 2.3 and are calculated as follows :

$$f_r = C_D \operatorname{Re} \frac{\pi D^3 \rho}{8} (u_r - V_r) \quad (2.18)$$

$$f_z = C_D \operatorname{Re} \frac{\pi D^3 \rho}{8} (u_z - V_z) \quad (2.19)$$

Re is the Reynolds number defined as

$$Re = \frac{\sqrt{(u_z - V_z)^2 + (u_r - V_r)^2} D}{\nu} \quad (2.20)$$

The drag coefficient C_D is calculated using a standard form fit for the drag coefficient of a sphere :

$$C_D = \frac{24}{Re} + \frac{6.}{1 + \sqrt{Re}} + .4 \quad (2.21)$$

2.3.2 Initial conditions for the particle equations

The initial conditions are derived from the properties of the spray nozzle. By definition the coordinate system is fixed at the nozzle so the initial conditions for equations 2.15 and 2.16 are simply :

$$r_0(0) = z_0(0) = 0 \quad (2.22)$$

The initial conditions for the force equations are derived by considering the particle ejection velocity from the nozzle and the initial angle of ejection (see figure 2) :

$$u_r(0) = U_0 \sin \theta$$

$$u_z(0) = U_0 \cos \theta \quad (2.23)$$

U_0 is calculated from the volume flow of the nozzle, Q_w :

$$U_0 = \frac{Q_w}{\frac{\pi}{4} d_0^2} \quad (2.24)$$

where d_0 is the inside diameter of the nozzle. The initial angle of ejection of the particle must be between 0 and θ_{\max} . θ_{\max} and d_0 are available from the manufacturers data.

2.3.3 Particle size

Particle size is calculated by using the following expression

$$D = C_N \frac{d_0^{2/3}}{\Delta P_N^{1/3}} \quad (2.25)$$

where C_N is a nozzle design parameter and ΔP_N is the water delivery pressure. These values may be determined from experiment or from manufacturers data.

2.4 Nondimensionalization of the model equations

To allow the resulting program to be used easily with any system of units, the model equations are nondimensionalized.

Velocities are nondimensionalized by the particle ejection velocity

$$\hat{V}_r = \frac{V_r}{U_0}, \quad \hat{V}_z = \frac{V_z}{U_0}, \quad \hat{u}_r = \frac{u_r}{U_0}, \quad \hat{u}_z = \frac{u_z}{U_0} \quad (2.26)$$

Pressure is non dimensionalized by the gas density, and the particle ejection velocity squared

$$\hat{P} = \frac{P}{\rho U_0^2} \quad (2.27)$$

Lengths are nondimensionalized by the spray height, H_ℓ , (see figure 1)

$$\hat{R} = \frac{R}{H_\ell}, \quad \hat{z} = \frac{z}{H_\ell}, \quad \hat{r}_0 = \frac{r_0}{H_\ell}, \quad \hat{z}_0 = \frac{z_0}{H_\ell} \quad (2.28)$$

Time is nondimensionalized by the spray height divided by the ejection velocity

$$\hat{t} = \frac{tU_0}{H_\ell} \quad (2.29)$$

The momentum source terms are nondimensionalized by the ejection velocity squared divided by the spray height

$$\hat{F}_r = F_r \frac{H_\ell}{U_0^2}, \quad \hat{F}_z = F_z \frac{H_\ell}{U_0^2} \quad (2.30)$$

Thus in the momentum equations (2.2) and (2.3), the viscosity is replaced by the reciprocal of the "Reynolds number of nondimensionalization"

$$\frac{1}{RE} = \frac{\nu}{U_0 H_\ell} \quad (2.31)$$

The nondimensionalization of the other model equations is straightforward. From this point on, the model equations are assumed nondimensional, and the hat ($\hat{}$) is neglected.

3. THE NUMERICAL METHOD

3.1 Introduction

It is difficult indeed to conceptualize a numerical method for the simultaneous solution of both sets of equations. A more natural method of solution involves iterative solution of these equations. Such an iterative solution is outlined by the flow chart in figure 3.

The specific numerical method to solve each set of equations differs due to their nature (ODE, PDE) and are considered in appropriate sections.

3.2 Numerical solution of the particle equations

The solution of the set of ordinary differential equations (ODE) governing the particle phase is made by a simultaneous fourth order Runge-Kutta technique which is well documented in the literature (Ref. 9).

3.3 Numerical solution of the gas equations

The numerical scheme for the solution of the gas phase equations is based on the MAC method (Refs. 8,10,11).

The gas phase equations are recast into the following form

$$\begin{aligned} \frac{\partial V_r}{\partial t} + \frac{1}{R} \frac{\partial}{\partial R} (V_r^2 R) + \frac{\partial (V_r V_z)}{\partial z} = - \frac{\partial P}{\partial R} + \\ + \frac{1}{RE} \left[\frac{\partial^2 V_r}{\partial R^2} + \frac{\partial}{\partial R} \left(\frac{V_r}{R} \right) + \frac{\partial^2 V_r}{\partial z^2} \right] + F_r \end{aligned} \quad (3.1)$$

$$\frac{\partial V_z}{\partial t} + \frac{1}{R} \frac{\partial}{\partial R} (V_r V_z R) + \frac{\partial V_z^2}{\partial z} = - \frac{\partial P}{\partial z} +$$

$$+ \frac{1}{RE} \left[\frac{1}{R} \frac{\partial V_z}{\partial R} + \frac{\partial^2 V_z}{\partial R^2} + \frac{\partial^2 V_z}{\partial z^2} \right] + F_z \quad (3.2)$$

$$\frac{1}{R} \frac{\partial P}{\partial R} + \frac{\partial^2 P}{\partial R^2} + \frac{\partial^2 P}{\partial z^2} = - \frac{\partial D}{\partial t} - \frac{\partial^2 (V_r)^2}{\partial R^2} - \frac{\partial^2 (V_z)^2}{\partial z^2}$$

$$- \frac{2}{R} \frac{\partial}{\partial R} (V_r)^2 - \frac{2}{R} \frac{\partial}{\partial z} (V_z V_r) - 2 \frac{\partial^2 (V_r V_z)}{\partial R \partial z} +$$

$$+ \frac{1}{RE} \left[\frac{1}{R} \frac{\partial D}{\partial R} + \frac{\partial^2 D}{\partial R^2} + \frac{\partial^2 D}{\partial z^2} \right] + \frac{1}{R} \frac{\partial (RF_r)}{\partial R} + \frac{\partial F_z}{\partial z} \quad (3.3)$$

These equations are written in unsteady form as the MAC solution procedure is an iterative one in which the steady state solution is the desired solution. To be noted here is the exchange of the continuity equation for a Poisson equation for the pressure. Contained in this Poisson equation is the variable D which is a dilation term representing the amount of continuity existing.

$$D = \frac{1}{R} \frac{\partial (V_r R)}{\partial R} + \frac{\partial V_z}{\partial z} \quad (3.4)$$

This term is used to reduce the nonlinear instabilities existing in the numerical solution of the Navier-Stokes equations (Ref. 11). Continuity is solved implicitly using this term.

3.3.1 Discretization of the MAC method equations

An appropriate discretization of the MAC method equation on a staggered grid (see Fig. 4) is given below :

$$\begin{aligned}
 \tilde{v}_{r_{i+1/2,j}} = & v_{r_{i+1/2,j}} + \Delta t \left\{ - \frac{1}{R_{i+1/2} \Delta R} \left[v_{r_{i+1,j}}^2 R_{i+1} - v_{r_{i,j}}^2 R_i \right] \right. \\
 & - \frac{1}{\Delta z} \left[v_r v_z_{i+1/2,j+1/2} - v_r v_z_{i+1/2,j-1/2} \right] \\
 & - \frac{1}{\Delta R} \left[p_{i+1,j} - p_{i,j} \right] + \frac{1}{RE} \left[\frac{1}{2\Delta R} \left[\left(\frac{v_r}{R} \right)_{i+3/2,j} \right. \right. \\
 & \left. \left. - \left(\frac{v_r}{R} \right)_{i-1/2,j} \right] + \frac{v_{r_{i+3/2,j}} - 2v_{r_{i+1/2,j}} + v_{r_{i-1/2,j}}}{\Delta R^2} \right. \\
 & \left. \left. + \frac{v_{r_{i+1/2,j+1}} - 2v_{r_{i+1/2,j}} + v_{r_{i+1/2,j-1}}}{\Delta z^2} \right] + F_r \right\}
 \end{aligned}
 \tag{3.5}$$

$$\begin{aligned}
 \tilde{V}_{z_{i,j+1/2}} &= V_{z_{i,j+1/2}} + \Delta t \left\{ \frac{-1}{R_i \Delta R} \left[V_r V_{z_{i+1/2,j+1/2}} R_{i+1/2} \right. \right. \\
 &\quad \left. \left. - V_r V_{z_{i-1/2,j+1/2}} R_{i-1/2} \right] - \frac{1}{\Delta z} \left[V_{z_{i,j+1}}^2 - V_{z_{i,j}}^2 \right] \right. \\
 &\quad \left. - \frac{1}{\Delta z} \left[P_{i,j+1} - P_{i,j} \right] + \right. \\
 &\quad \left. + \frac{1}{RE} \left[\frac{1}{R_i} \left[\frac{V_{z_{i+1,j+1/2}} - V_{z_{i-1,j+1/2}}}{2\Delta R} \right] \right. \right. \\
 &\quad \left. \left. + \frac{V_{z_{i+1,j+1/2}} - 2V_{z_{i,j+1/2}} + V_{z_{i-1,j+1/2}}}{\Delta R^2} \right. \right. \\
 &\quad \left. \left. + \frac{V_{z_{i,j+1/2}} - 2V_{z_{i,j+1/2}} + V_{z_{i,j-1/2}}}{\Delta z^2} \right] + F_z \right\} \quad (3.6)
 \end{aligned}$$

$$\begin{aligned}
 \frac{1}{R_i} &\frac{\tilde{P}_{i+1,j} - \tilde{P}_{i-1,j}}{2\Delta R} + \frac{\tilde{P}_{i+1,j} - 2\tilde{P}_{i,j} + \tilde{P}_{i-1,j}}{\Delta R^2} + \frac{\tilde{P}_{i,j+1} - 2\tilde{P}_{i,j} + \tilde{P}_{i,j-1}}{\Delta z^2} = \\
 &= - \frac{\tilde{V}_r^2}{\Delta R^2} \frac{V_{r_{i+1,j}} - 2V_{r_{i,j}} + V_{r_{i-1,j}}}{\Delta R^2} - \frac{\tilde{V}_z^2}{\Delta z^2} \frac{V_{z_{i,j+1}} - 2V_{z_{i,j}} + V_{z_{i,j-1}}}{\Delta z^2}
 \end{aligned}$$

$$\begin{aligned}
 & - \frac{2}{R_i} \frac{V_r^2_{i+1,j} - V_r^2_{i-i,j}}{2\Delta R} - \frac{2}{R_i} \frac{V_z V_r_{i,j+1} - V_z V_r_{i,j-1}}{2\Delta z} \\
 & - 2 \frac{V_r V_z_{i+1/2,j+1/2} + V_r V_z_{i-1/2,j-1/2} - V_r V_z_{i+1/2,j-1/2} - V_r V_z_{i-1/2,j+1/2}}{\Delta R \Delta z} \\
 & + \frac{1}{RE} \left[\frac{1}{R_i} \frac{D_{i+1,j} - D_{i-1,j}}{2\Delta R} + \frac{D_{i+1,j} - 2D_{i,j} + D_{i-1,j}}{\Delta R^2} + \frac{D_{i,j+1} - 2D_{i,j} + D_{i,j-1}}{\Delta z^2} \right] \\
 & + \frac{D_{i,j}}{\Delta t} + \frac{1}{R_i} \left[\frac{R_{i+1/2} F_{r_{i+1/2,j}} - R_{i-1/2} F_{r_{i-1/2,j}}}{\Delta R} \right] + \frac{F_{z_{i,j+1/2}} - F_{z_{i,j-1/2}}}{\Delta z}
 \end{aligned}
 \tag{3.7}$$

Here the tilde (~) indicates the updated quantities (N+1 iteration).

As can be seen, the MAC method is a two level scheme involving explicit solution of equations 3.5 and 3.6 for the velocities of the N+1 iteration and an iterative solution of equations 3.7 for the updated pressure field.

Note the discretization of the $\frac{\partial D}{\partial t}$ term. D^{N+1} is set equal to zero in an attempt to force continuity to exist at the N+1 time step. At steady state the time derivative term disappears and continuity will be satisfied, thereby solving the original problem.

This method was applied to the solution of laminar flow in a pipe as a test case. Solutions were obtained for low Reynolds numbers ($Re < 100$) but instabilities were observed for higher Reynolds numbers.

Because the solution of the gas equations involves a low gas viscosity ($\nu = 1.5 \times 10^{-5} \text{ m}^2/\text{s}$) and therefore high Reynolds numbers, the numerical method has to be modified.

3.3.2 Modification to the MAC method

To remove the instabilities upwind differencing was employed. This is not so straightforward for the staggered grid used and involves some additional approximations. The resulting discretization of the momentum equations is made below. Here only the convective terms are affected and therefore are the only ones considered.

$$\frac{\partial V_r}{\partial t} + V_r \frac{\partial V_r}{\partial r} + V_z \frac{\partial V_r}{\partial z}$$

$$\cong \frac{\tilde{V}_{r_{i+1/2,j}} - V_{r_{i+1/2,j}}}{\Delta t}$$

$$+ V_{r_{i+1/2,j}} \left\{ \begin{array}{l} \left[\frac{V_{r_{i+1/2,j}} - V_{r_{i-1/2,j}}}{\Delta r}, V_{r_{i+1/2,j}} > 0 \right] \\ \left[\frac{V_{r_{i+3/2,j}} - V_{r_{i+1/2,j}}}{\Delta r}, V_{r_{i+1/2,j}} < 0 \right] \end{array} \right.$$

$$V_{Z A U E} \left\{ \begin{array}{l} \frac{V_{r_{i+1/2,j+1}} - V_{r_{i+1/2,j}}}{\Delta z}, \quad V_{Z A U E} > 0 \\ \frac{V_{r_{i+1/2,j}} - V_{r_{i+1/2,j-1}}}{\Delta z}, \quad V_{Z A U E} < 0 \end{array} \right\}$$

$$V_{Z A U E} = \frac{1}{4} \left\{ V_{z_{i,j+1/2}} + V_{z_{i+1,j+1/2}} + V_{z_{i,j-1/2}} + V_{z_{i+1,j-1/2}} \right\} \quad (3.8)$$

$$\frac{\partial V_z}{\partial t} + V_r \frac{\partial V_z}{\partial r} + V_z \frac{\partial V_z}{\partial z}$$

$$\cong \frac{\tilde{V}_{z_{i,j+1/2}} - V_{z_{i,j+1/2}}}{\Delta t} +$$

$$+ V_{R A U E} \left\{ \begin{array}{l} \frac{-V_{z_{i+1,j+1/2}} - V_{z_{i,j+1/2}}}{\Delta R}, \quad V_{R A U E} > 0 \\ \frac{V_{z_{i,j+1/2}} - V_{z_{i-1,j+1/2}}}{\Delta R}, \quad V_{R A U E} < 0 \end{array} \right\}$$

$$+ V_{z_{i,j+1/2}} \left\{ \begin{array}{l} \frac{V_{z_{i,j+3/2}} - V_{z_{i,j+1/2}}}{\Delta z}, \quad V_{z_{i,j+1/2}} > 0 \\ \frac{V_{z_{i,j+1/2}} - V_{z_{i,j-1/2}}}{\Delta z}, \quad V_{z_{i,j+1/2}} < 0 \end{array} \right\} \quad (3.9)$$

The pressure equation remains the same.

This modification allowed a stable solution to be obtained, though the scheme did not allow the lower wall to be sensed, and no recirculation occurred (see Fig. 5). There another modification was employed. This involved changing the unsteady term in the momentum equations along the lower and axis of symmetry boundaries.

$$\frac{\partial V_z}{\partial t} \cong \frac{\tilde{V}_{z_{i,j+1/2}} - \frac{1}{2} (V_{z_{i,j+3/2}} + V_{z_{i,j-1/2}})}{\Delta t} \quad (3.10)$$

on lower boundary,

$$\frac{\partial V_r}{\partial t} \cong \frac{\tilde{V}_{r_{i+1/2,j}} - \frac{1}{2} (V_{r_{i+3/2,j}} + V_{r_{i-1/2,j}})}{\Delta t} \quad (3.11)$$

on axis of symmetry boundary.

This finally yielded plausible results.

3.3.3 Solution procedure for the MAC method

The solution procedure for the MAC method, as mentioned previously, involves an explicit solution of equations (3.5) and (3.6) with the modifications for the velocity field at the N+1 iteration and an iterative solution of (3.7) for the pressure field at the N+1 iteration. This iterative solution was made using a point by point SOR method. The overrelaxation factor used was $\omega = 1.5$, though no optimization has been done on this parameter.

3.4 Calculation of the source terms
in the Navier-Stokes equations

For the calculation of the source term, the particle trajectories are superimposed over the mesh used for the solution of the gas equations, (see figure 6). To calculate the source terms the arc length of each particle trajectory in a given cell is determined. Because the velocity along this trajectory is known, the time that a particle spends in this cell can be determined :

$$\delta t = \int \frac{ds}{\sqrt{V_r^2 + V_z^2}} \cong \frac{\delta s}{\sqrt{V_r^2 + V_z^2}} \quad (3.12)$$

for small δs and gradually varying V_r and V_z .

From this time the average number of particles in the cell is determined by considering the fraction of the mass flow through the nozzle assigned to this particular trajectory, f_p . The average force along the arc length is determined and then the contribution to the momentum source term along this trajectory in this particular cell is found as follows.

Number of particles on given trajectory in control volume :

$$N_p = \frac{Q_w \cdot f_p \cdot \delta t}{\frac{\pi}{6} D^3} \quad (3.13)$$

$$F_r = \frac{F_{rav} \times N_p}{\text{Volume of cell}} \quad (3.14)$$

similarly for F_z .

It should be noted that F_r and F_z are not calculated in the same cell because of the staggered grid (see figure 4).

4. RESULTS

Results were obtained for the consideration of the effect of increase of mass flow through the nozzle and the effect of a ceiling on entrainment properties of the spray, and finally, to examine the pressure field. The results were obtained by using the properties of a Lechler SZ1 spray nozzle as input to the program (see Table 1).

4.1 Comparison with experiment

First to validate the program a comparison of the entrainment properties of the calculated spray was made with the experimental results of an unconfined spray. These results are shown in figure 7. Here the entrainment efficiency is plotted versus the inverse of the non dimensional spray envelope diameter squared. The entrainment efficiency is defined as the volume flow of air entrained into the spray divided by the volume flow of water through the nozzle. The amount of air entrained into the spray is defined as follows :

$$Q_a = 2\pi \int_0^{D_t/2} r V_z(r) dr \quad (4.1)$$

Results were obtained by considering a spray nozzle mounted on a ceiling two meters above the floor, for all three mass flows considered. Although this is not an unconfined spray, previous results have shown little effect of the presence of the ceiling (Ref. 6). The axial positions where these results were obtained ranged from 20 percent of the spray height from the nozzle to 55 percent of this distance. It was impossible to calculate these values any closer to the nozzle because of the lack of resolution with 21×21 pressure nodes. At axial positions greater than this the floor seems to have a significant effect on the entrainment, so no comparison is made in this area.

The results displayed in figure 7 show good agreement with the McQuaid correlation (solid line) and the $\pm 20\%$ scatter of experimental data about this line. This gives confidence that the bulk of the flow field is well predicted.

4.2 Effect of increasing mass flow through nozzle

The effect of increasing mass flow is displayed in figures 8, 9 and 10. Here the velocity vectors are non dimensionalized by the ejection velocity of the particle. For the determination of the entrained air flow equation 4.1 is rearranged

$$Q_a = 2\pi U_0 \int_0^{D_t/2} r \frac{V_z(r)}{U_0} dr \propto 2\pi Q_w \int_0^{D_t/2} r \frac{V_z(r)}{U_0} dr \quad (4.2)$$

As can be seen from the results, the flow field non dimensionalized by the ejection velocity does not change significantly. Thus the integral in 4.2 is of the same order of magnitude for all three mass flows considered. It seems that the air entrainment in the spray is increased strongly by increasing the mass flow through the nozzle. This is a physically observable result.

Finally the efficiency of the spray is examined.

$$\eta = \frac{Q_a}{Q_w} = 2\pi \int_0^{D_t/2} r \frac{V_z(r)}{U_0} dr \quad (4.3)$$

Close examination of figures 8, 9 and 10 shows a very slight increase in the efficiency of the spray with increased mass flow, reflected by a slight increase in the above integral. This result is found to be true for some cases, though the inverse can also be true. This is because higher mass flow results in small droplets which exchange momentum with air more rapidly resulting in higher entrainment, but because of the

small droplet size the spray envelope contracts quickly slowing entrainment. Thus, for small spray lengths an increase in efficiency is observed relative to larger droplet sizes, but this is reversed as the spray length is increased.

4.3 Effect of ceiling on entrainment

Results were obtained for the case where the ceiling is removed. These results are shown in figures 11 and 12. Though not shown a free boundary was placed at a height of two meters above the spray nozzle. Comparison with the previous results of corresponding mass flow with ceiling included shows little change in the entrainment properties with or without a ceiling. This result is supported by experimental results (Ref. 3).

4.4 Examination of the pressure field

Along with the velocity field each calculation also gives the corresponding pressure field. The isobars of a typical pressure field are shown in figure 13. This figure validates the previous assumption in the derivation of the boundary condition that at the free boundary the pressure field varies little and the pressure can be set to a reference value.

The area of steepest pressure gradients is near the floor inside the spray envelope. However, the pressure gradient in this region is small compared to the momentum source term. The other area of high pressure gradient seems to be at the nozzle. In this region, again the pressure gradient is small compared to momentum source term inside the spray envelope, though outside the spray envelope the pressure gradient may be significant. Better resolution is necessary to draw a firm conclusion.

It can be concluded that for the bulk of the flow field ignoring the pressure gradient may be a good approximation. This is a particularly important result since the pressure calculations are the most time consuming and require the most number of statements in program. In the future the program may be modified to include only a closed form expression for a pressure-like variable which forces continuity to be satisfied. This should result in large savings in computer time and allow better resolution in the mesh.

5. CONCLUSIONS

It has been shown that the axisymmetric spray model implemented gives good results for the entrainment of air inside the spray envelope for the portion at the spray envelope which contains most of the entrained air. The bulk of the surrounding flow field is thus considered to be more or less correctly predicted.

The physically observable result that increasing the outflow from the nozzle increases entrainment of air is also found.

The effect of the ceiling on entrainment is seen to be negligible.

Finally, the examination of the pressure field indicates that neglecting the pressure gradient is a good approximation for the bulk of the flow field. This result leads to the recommendation that work continues on the model to incorporate a closed form expression for a pressure-like variable to force continuity to hold, while eliminating the need for the Poisson equation for the pressure. This will allow better results since the mesh may be made smaller due to the saving in computer time.

Application of this model to a spray in the upward facing mode is straightforward.

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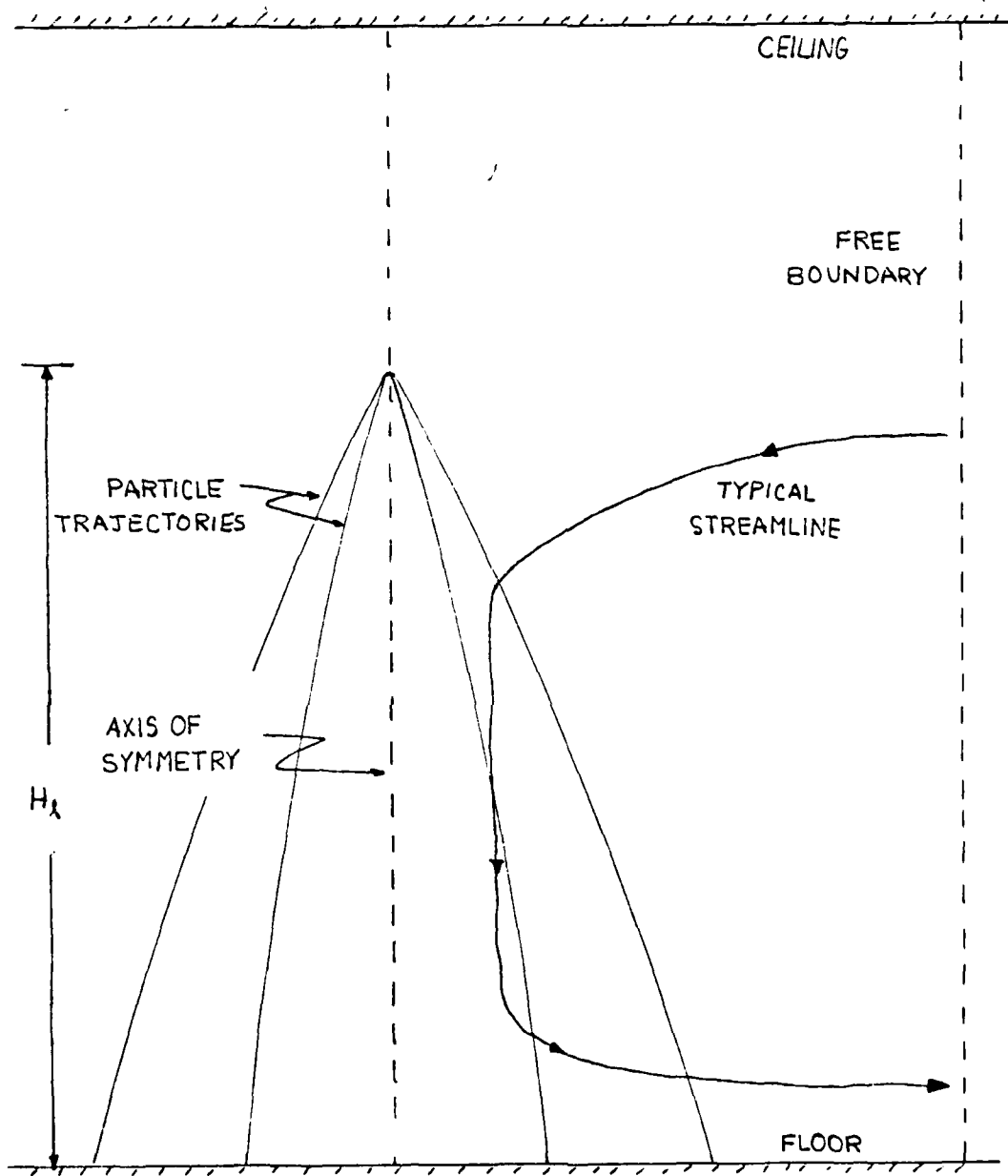


Figure 1 Spray and flow domain in downward facing mode

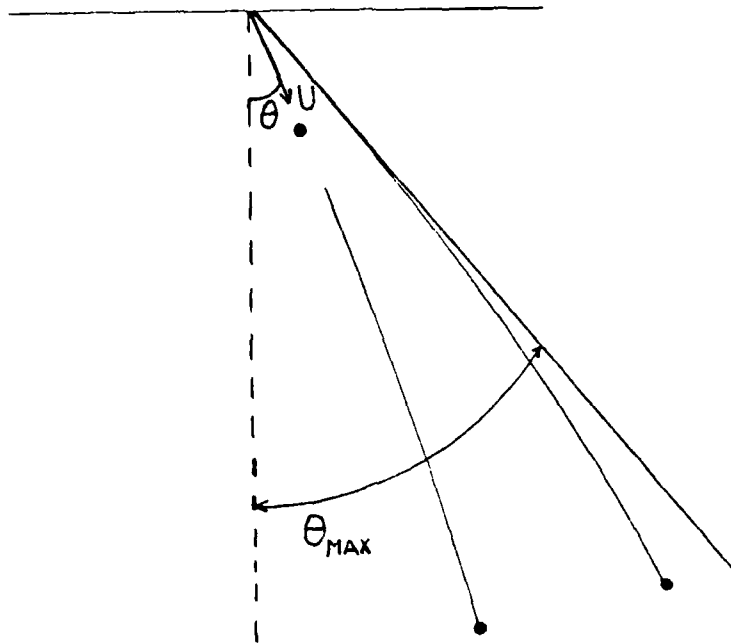


Figure 2 Nozzle Configuration

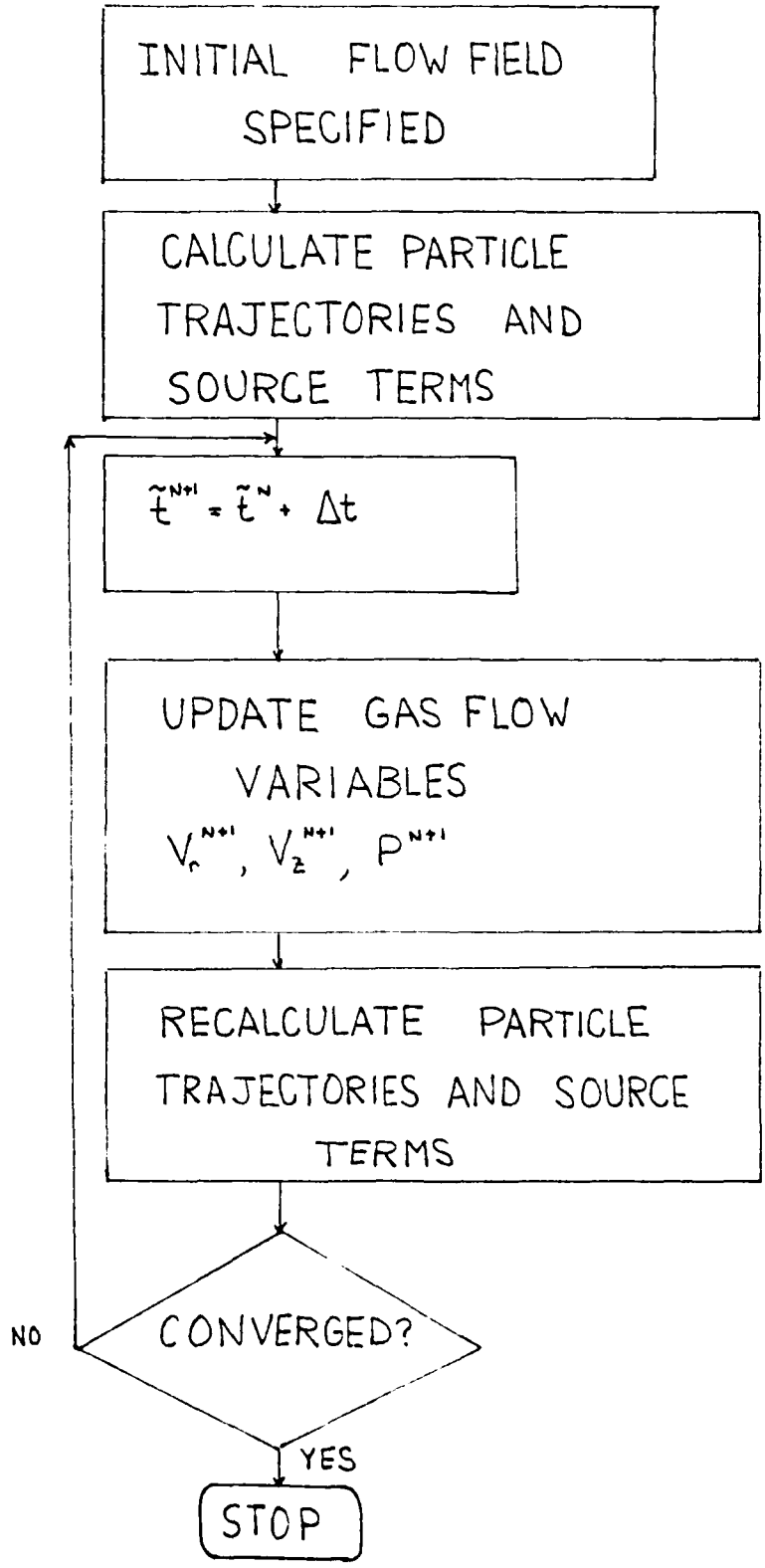


Figure 3 Scheme for numerical solution

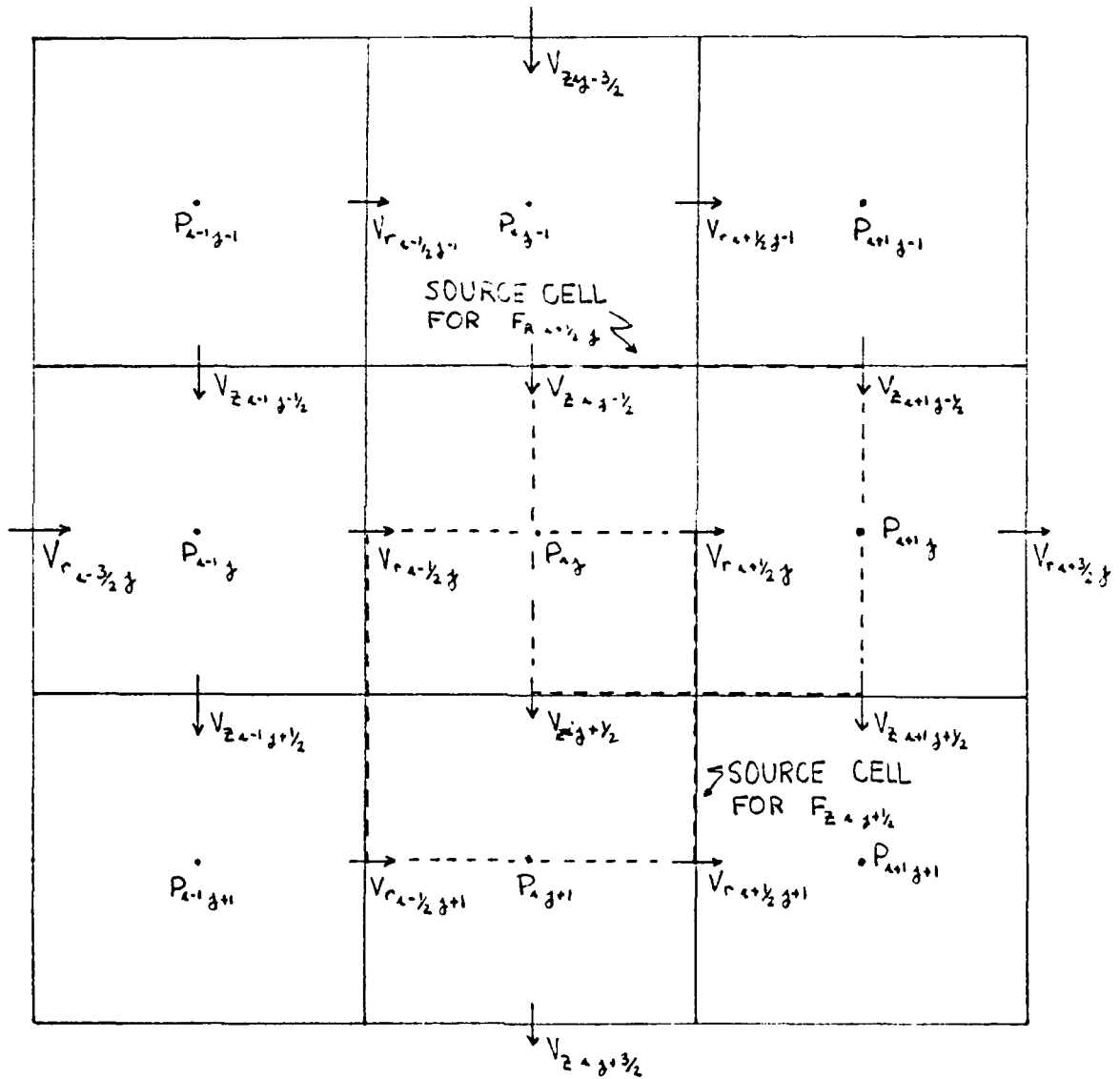


Figure 4 Staggered MAC grid with source cells

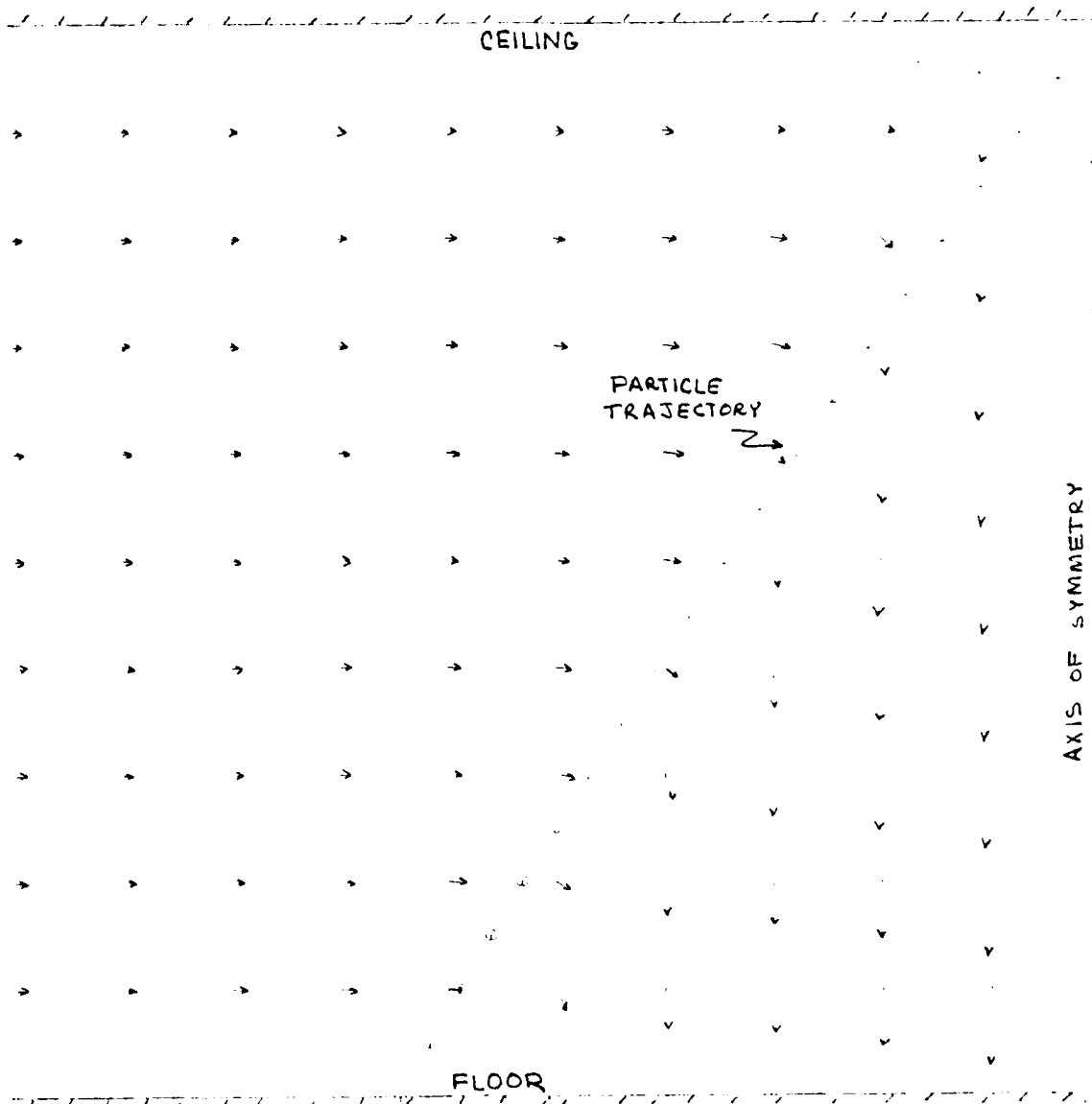


Figure 5 Non-recirculating spray

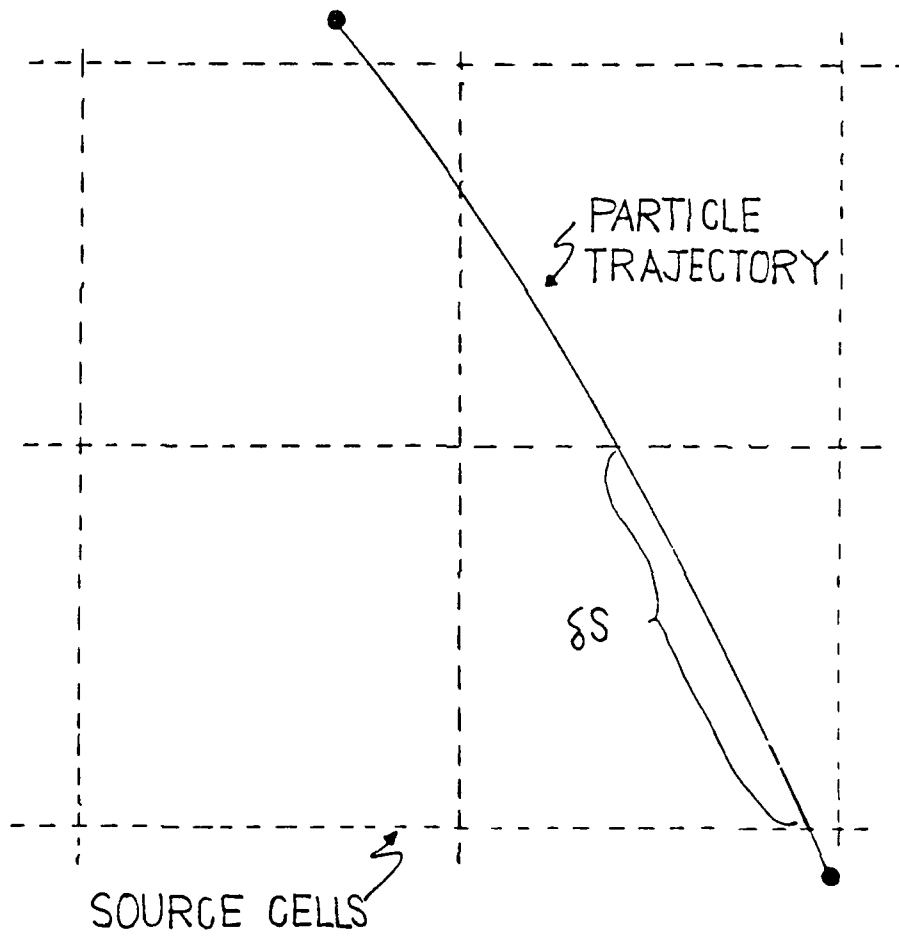


Figure n Particle trajectory superimposed on source cells

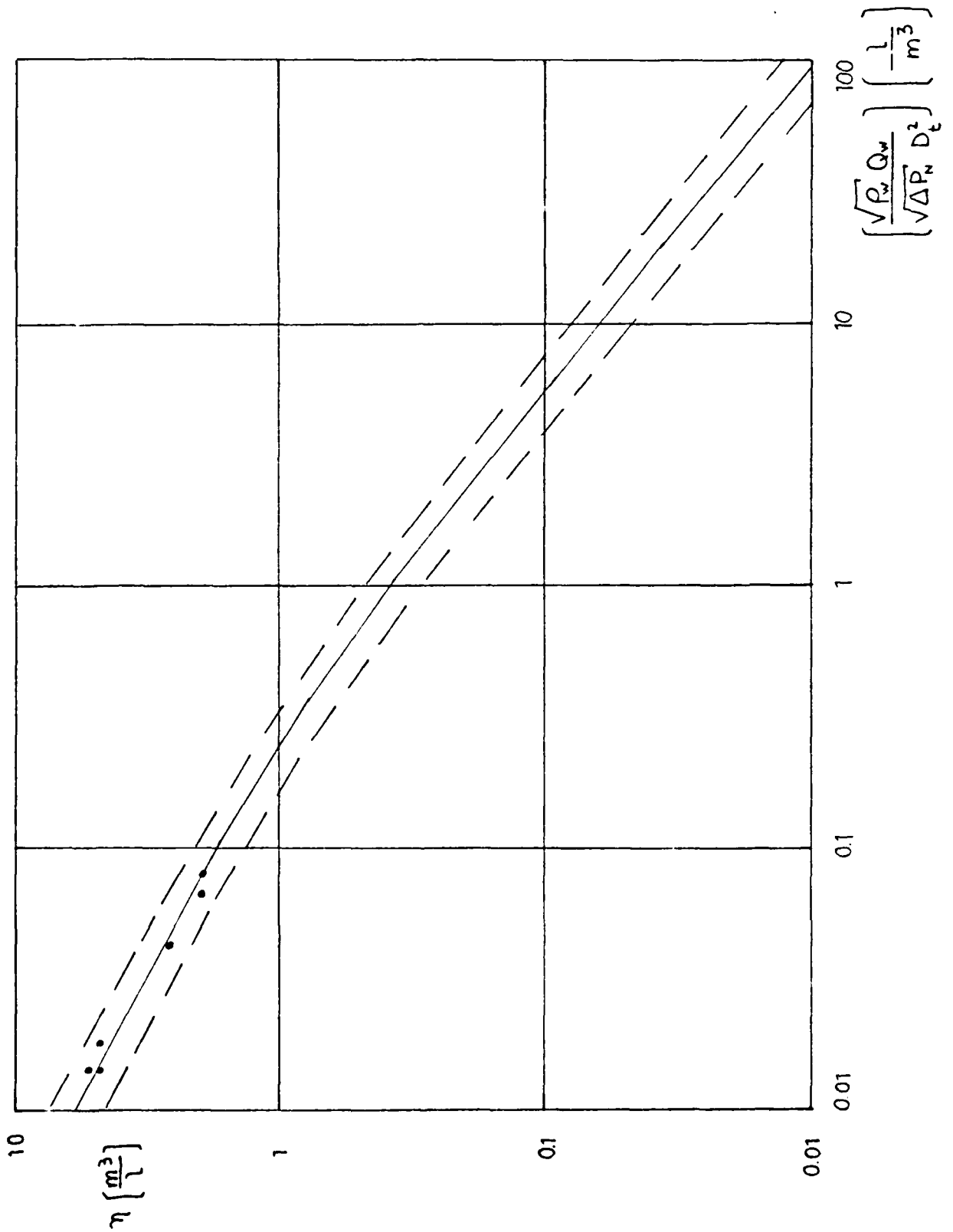
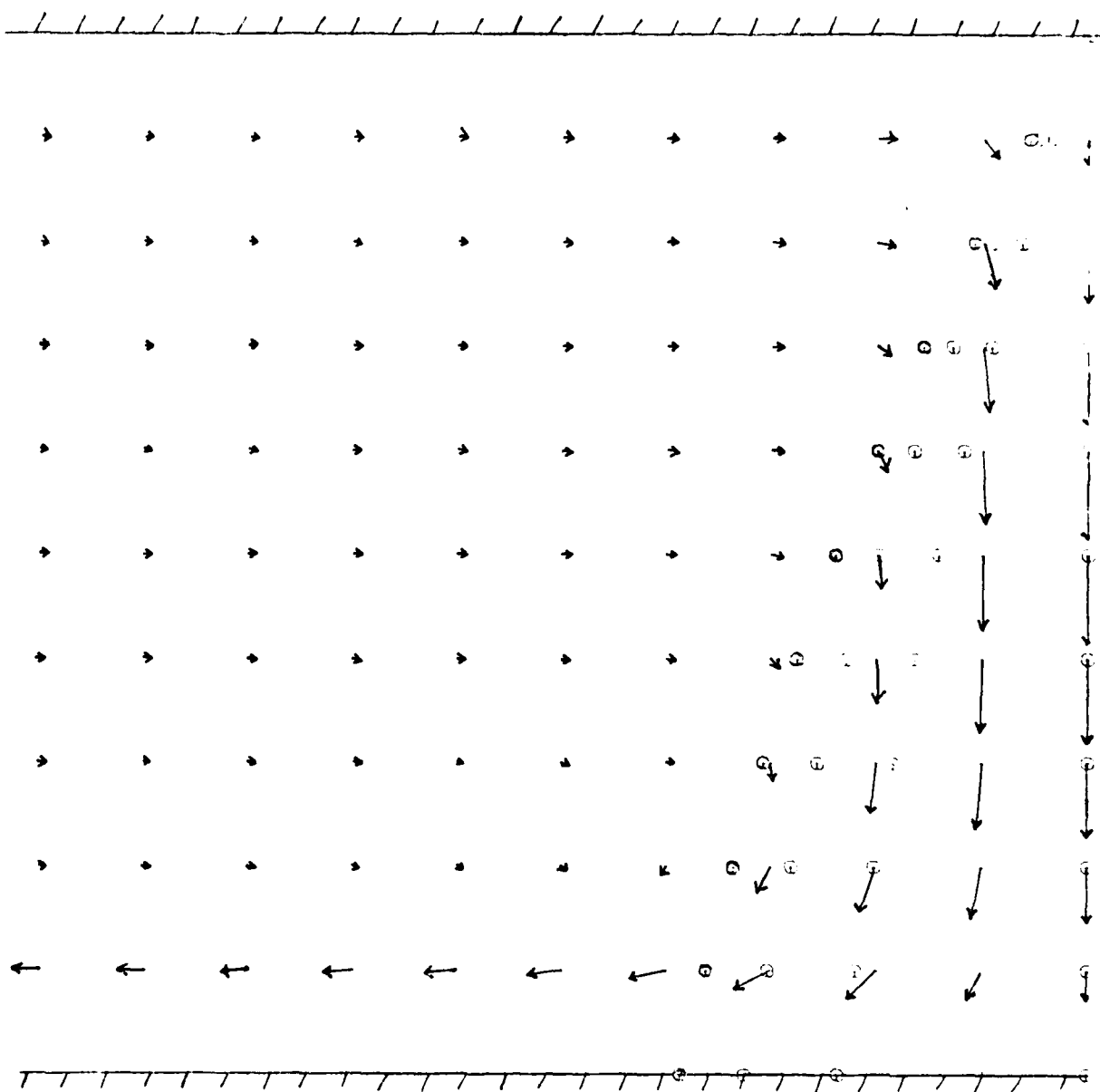


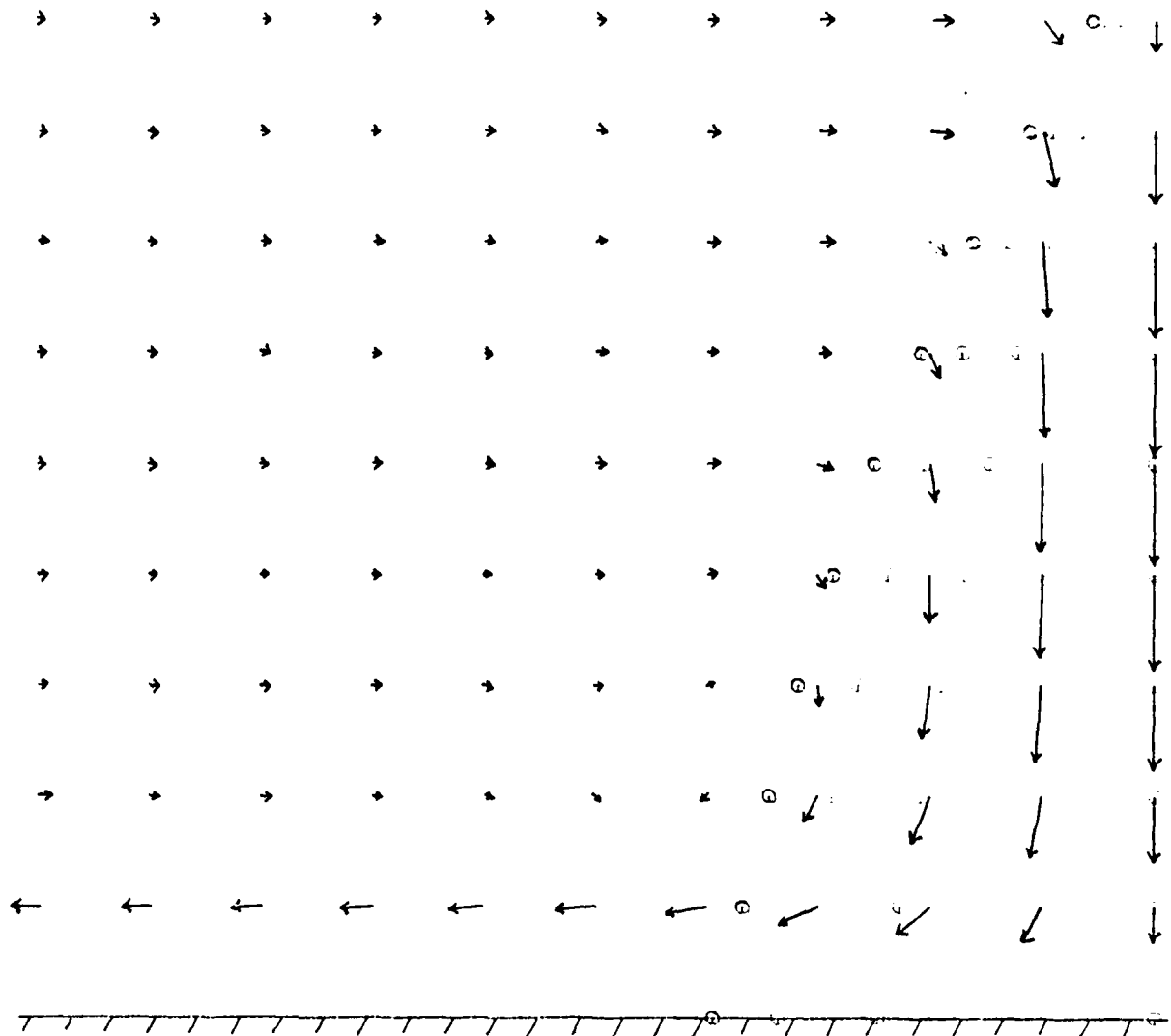
Figure 7 Comparison of air entrainment with experimental results



← CORRESPOND TO A CAR VELOCITY TO INJECTION VELOCITY RATIO OF 10 PERCENT
 ○ ○ ○ ○ REPRESENT PARTICLE TRAJECTORIES

LEHLER 21 NOZZLE
 NOZZLE DIAMETER 1.14 MM
 HALF ANGLE OF SPRAY 30 DEGREE
 LIQUID VOLUME FLOW 4.4 LIT PER MIN
 DROPLET SIZE 1.00 MM
 SPRAY HEIGHT 0.0 M

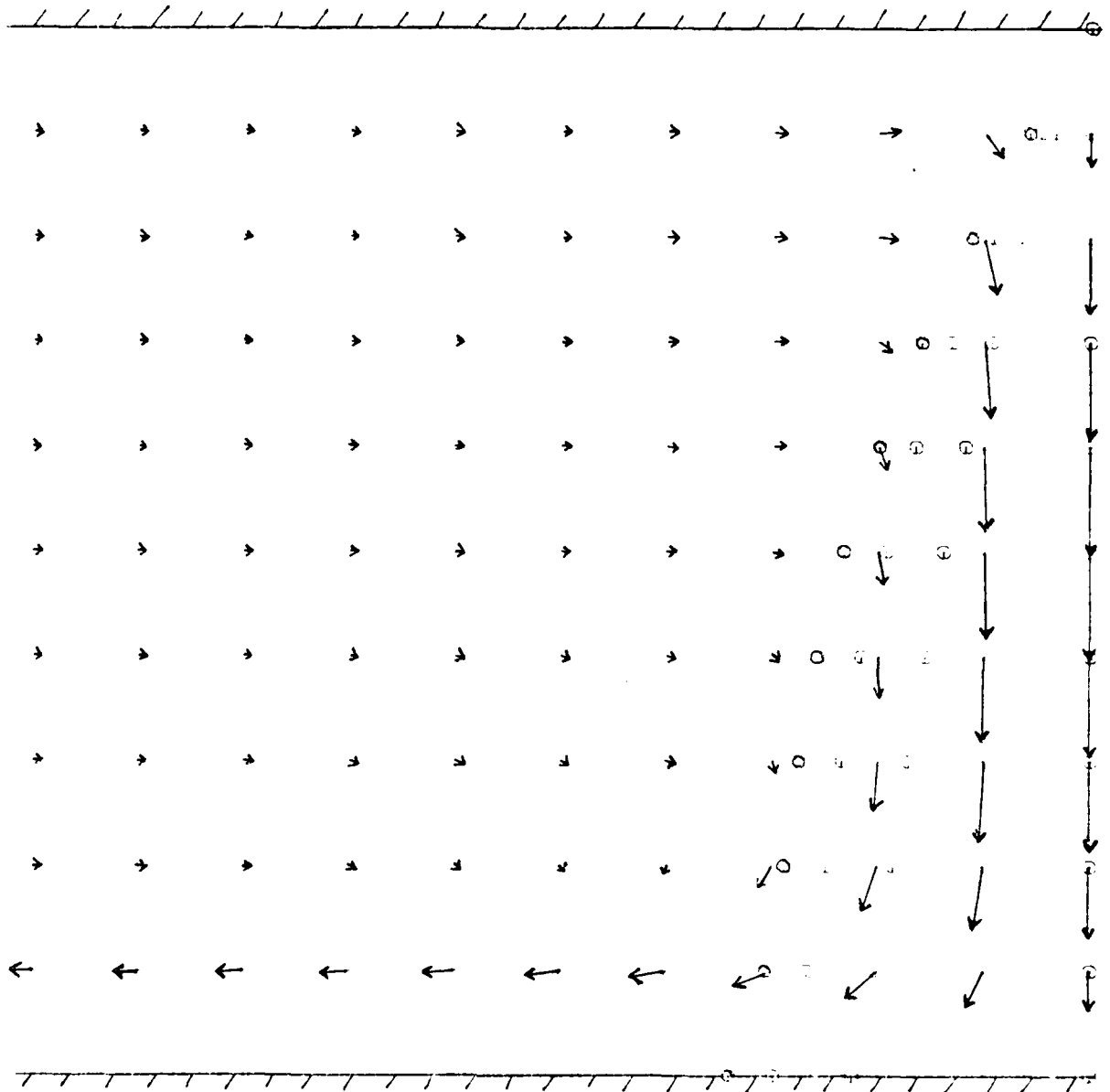
FIGURE 8 FLOW FIELD AND PARTICLE TRAJECTORIES



← CORRESPONDS TO A GAS VELOCITY v_0
 INFINITE VELOCITY RATIO OF 20 PERCENT
 ○ ○ ○ ○ REPRESENT PARTICLE TRAJECTORIES

NEHLER ZWEIFEL
 NOZZLE DIAMETER 4.40 MM
 HALF ANGLE OF SPRAY 30°
 LIGHT VOLUME FLOW 10.3 LT PER MIN
 DROPLET SIZE 0.01 MM
 SPRAY HEIGHT 1.00 M

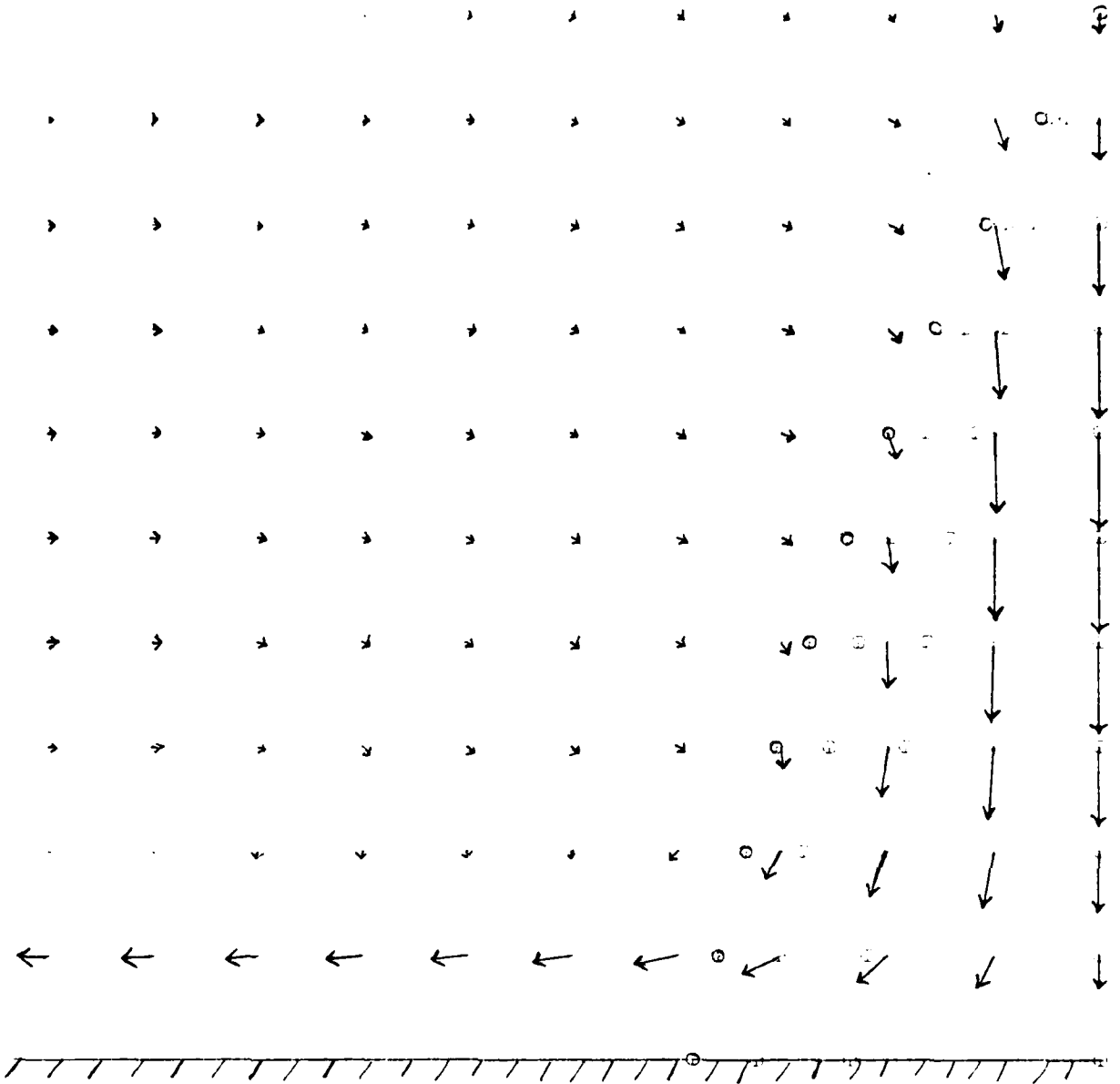
FIGURE 9. LOW SPEED FLOW FIELD OF TRACER PARTICLES



← CORRESPONDS TO A GAS VELOCITY TO
INJECTION VELOCITY RATIO OF 10 PERCENT
○○○○ REFERENCE PARTICLE TRAJECTORY

LEHLEN 21 M 12.5
NOZZLE DIAMETER 1.40 MM
HALF ANGLE OF SPRAY 30.0 DEGREE
LIQUID VOLUME FLOW 2.40 LT PER MIN
NOZZLE VELOCITY 140 M
SPRAY WEIGHT 0.004

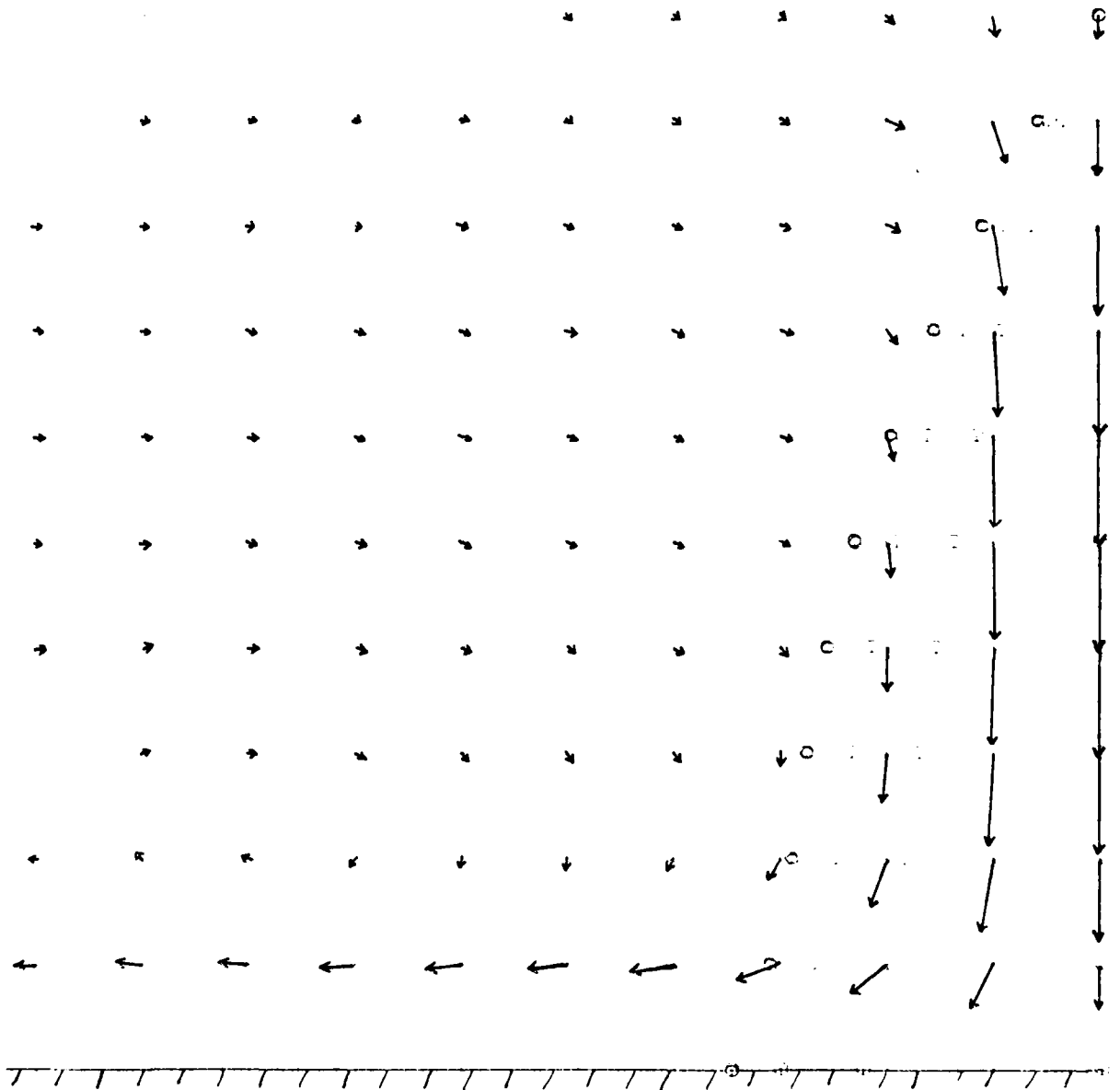
FIGURE 10 FLOW FIELD AND PARTICLE TRAJECTORY



← CORRESPONDS TO A GA. VELOCITY OF
 INJECTION VELOCITY RATIO OF 0.2. FREQUENT
 ○ ○ ○ ○ REPRESENTS PARTICLE TRAJECTORY

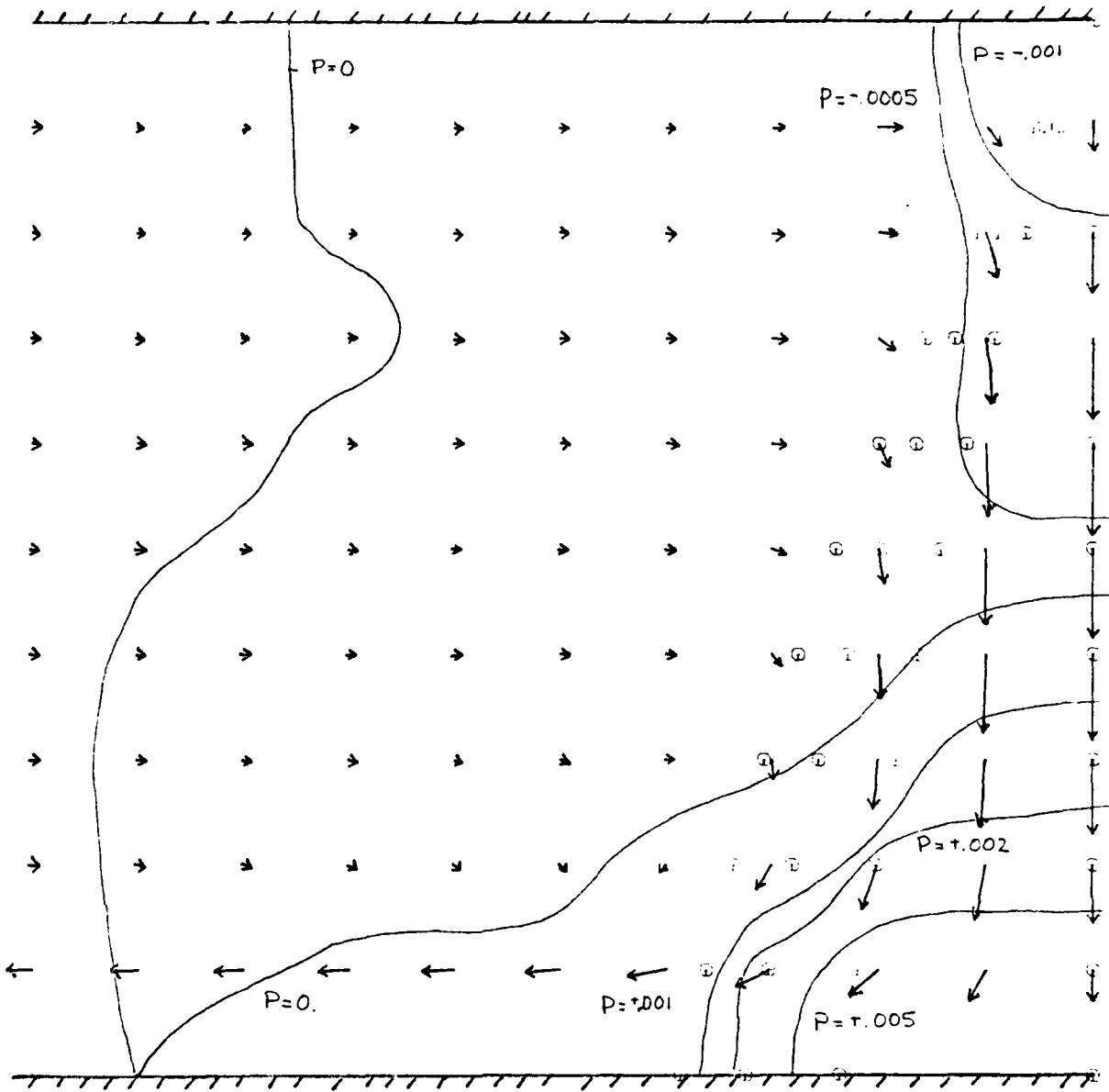
LECHLER 121 NOZZLE
 NOZZLE DIAMETER 4.4 MM
 HALF ANGLE OF SPRAY 30 DEGREE
 LIQUID VOLUME FLOW 1.8 L PER MIN
 DROPLET SIZE 1.07 MM
 SPRAY HEIGHT 1.0 M

FIGURE 11. FLOW FIELD IN THE NEAR FIELD OF THE SPRAY



← CORRESPONDS TO A GAS VELOCITY U
 INJECTION VELOCITY RATIO OF 0.5 PER CM
 ○ ○ ○ ○ REPRESENTS PARTICLE TRAJECTORIES

CHANNEL 20 INCHES
 PARTICLE DIAMETER 4.1 MIC
 WALL ANGLE 20 DEGREE
 CHANNEL WALL FLOW 0.15 PER MIN
 PARTICLE SIZE 0.40 MIC
 WALL HEIGHT 1.5 IN



← CORRESPOND TO A GA VELOCITY TO INJECTION VELOCITY RATIO OF 10 PER CENT
 ○ ○ ○ REPRESENT PARTICLE TRAJECTORIES

$P = \text{PRESSURE} / \rho U_0^2$

LEINER 21 N. 1111
 NOTICE IN AIR
 HALF ANGLE OF SPRAY 30 DEGREE
 LIQUID COLUMN FLOW AT 1000 PSI
 DROPLET SIZE 100 MICRONS
 SPRAY HEIGHT 100 CM

d_o mm	Q_w lt/min	ΔP_N kPa	D mm	θ_{max}
4.4	6.6	34.3	1.05	30°
	13.3	176.5	.61	30°
	21.1	617.8	.40	30°

Table 1. Properties of Lechler SZL Nozzle

APPENDIX - LISTING OF SPRAY COMPUTER PROGRAM

MAIN Program	p. 40
Subroutine COEFF	p. 49
Subroutine PSOURC	p. 55
Subroutine SOR	p. 60
Subroutine GASVEL	p. 65
Subroutine PARTCL	p. 68
Input FILE	p. 74
Output FILE	p. 75

MAIN PROGRAM

```

100 DIMENSION VK(50,50),VZ(50,50),X(50),Y(50)
200 DIMENSION VEG(50,50),VZB(50,50)
300 DIMENSION PRESR(50,50),GUESS(50,50)
400 DIMENSION ABC(50),BOC(50),CBC(50),DBC(50)
500 DIMENSION CR(50,50),CS(50,50),CSA(50,50),CSE(50,50)
600 DIMENSION CU(50,50),CS(50,50),CV(50,50),CR(50,50)
700 DIMENSION F(50,50),F(50,50),FZ(50,50)
800 DIMENSION FR(50,50),FZ(50,50)
900 DIMENSION DF(10),THEIAG(10),FFACQ(10)
1100 COMMON /C/,CWA,CSE,CSW,CS,CF,CR,CV,F
1200 COMMON /SUBBOUNDARY/ ANCF,AKC,CFC,FAC,ACROSS,BKROSS,CCROSS,DCROSS
1300 COMMON /BOUNDARY/ ILEFT,RIGHT,ITOP,BOTTOM
1400 COMMON /AIR V/ VZ,VI
1500 COMMON /COORD/ X,Y
1600 COMMON /DEL/ DELA,DELY,DELF
1700 COMMON /REYNOLDS/ RE
1800 COMMON /SPRAY D/ DZ,DI,DR,DR,GR,GRU
1900 COMMON /SOURCE/ FR,FZ
2000 READ N,IF
2100 *****
2200 *****
2300 *****
2400 *****
2500 *****
2600 *****
2700 *****
2800 *****
2900 *****
3000 *****
3100 *****
3200 *****
3300 *****
3400 *****
3500 *****
3600 *****
3700 *****
3800 *****
3900 *****
4000 *****
4100 *****
4200 *****

```

```

C *****
C THIS PROGRAM SOLVES ITERATIVELY BOTH THE WAVIER STONES
C EQUATIONS WITH FURTHER SOURCE TERMS INCLUDED, USING THE
C FAC METHOD, AND THE EQUATIONS OF MOTION OF A PARTICLE
C USING A FOURTH ORDER RUNGE-KUTTA METHOD TO DETERMINE THE
C SOURCE TERMS
C
C THIS MAIN PROGRAM SETS UP THE INPUT FOR THE FIVE SUBROUTINES
C COEFF, POUNDG, SPR, GRVGL, AND FAC10, AND THEN ITERATES
C FOR THE SOLUTION.
C
C COEFF CALCULATES THE COEFFICIENTS OF THE LEFT HAND SIDE OF
C THE FULSBOE EQUATION FOR THE PRESSURE. THIS CALCULATION
C IS ONLY PERFORMED ONCE SINCE THESE COEFFICIENTS ARE ONLY
C FUNCTION OF SPACIAL POSITION.
C
C SOURCE CALCULATES THE SOURCE TERM (RIGHT HAND SIDE) OF THE
C PRESSURE EQUATION. THE SOURCE TERM IS DEPENDENT ON THE
C LOCAL VELOCITY AND MUST BE RECALCULATED AT EACH ITERATION.
C
C SOL CALCULATES THE SOLUTION OF THE DISCRETIZED FULSBOE

```



```

8300 READ(20,220)DURPAR
8400 FORMAT(//1A,12)
8500 WRITE(30,226)DURPAR
8600 FORMAT(1A,10)NUMBER OF PARTICLE SIZES/TRAJECTORIES= ',12)
8700 READ(20,221)
8800 FORMAT(//)
8900 ENTER DROPLET DIAMETER/NOZZLE DIAMETER, INITIAL HALF ANGLE OF
9000 SPRAY, AND FRACTION OF VOLUME FLOW FOR EACH PARTICULAR
9100 PROPLET SIZE/TRAJECTORY
9200 IOTFRQ=0
9300 I=222 I=1, DURPAR
9400 READ(20,223)DP(1),THEFA(1),FRACG(1)
9500 WRITE(30,212)DP(1),THEFA(1),FRACG(1)
9600 FORMAT(1A,10)DROPLET DIAMETER/NOZZLE DIAMETER= ',F10.5,2X,
9700 I HALF ANGLE OF SPRAY= ',F10.5,2X, VOLUME FLOW FRACTION= ',
9800 2F10.5)
9900 IOTFRQ=IOTFRQ+FRACG(1)
10000 CONTINUE
10100 FORMAT(1A,3)F10.5,2X))
10200 IF(ABS(1.-IOTFRQ).LT.0.02)GO TO 225
10300 WRITE(30,224)
10400 FORMAT(1A,15)NUMBER OF NOZZLE VOLUME FLOW FRACTIONS IS NOT
10500 EQUAL TO ONE)
10600 STOP
10700 CONTINUE
10800 ENTER INVERSE FLOODE NUMBER SQUARED, G, AND REYNOLDS NUMBER
10900 OF BUBBLE STABILIZATION
11000 READ(20,213)G,NC
11100 FORMAT(//1A,2)F10.5,2X))
11200 WRITE(30,214)G,NC
11300 FORMAT(1A,10)INVERSE FLOODE NUMBER SQUARED= ',F10.5,2X,
11400 REYNOLDS NUMBER OF BUBBLE STABILIZATION= ',E12.5)
11500 ENTER RELSITY OF GAS/LIQUID
11600 READ(20,215)RG
11700 FORMAT(//1A,2)F10.5,2X))
11800 WRITE(30,216)RG
11900 FORMAT(1A,10)DENSITY OF GAS/LIQUID OF LIQUID= ',F10.5)
12000 SET THE TIME STEP-(STEADY STATE CASES THE CONVERGED SOLUTION)
12100 THE SIZE OF THE TIME STEP IS SUBJECT TO STABILITY RESTRICTIONS
12200 READ(20,217)DELT
12300 FORMAT(//1A,10)DELT
12400 WRITE(30,218)DELT
12500 FORMAT(1A,10)TIME STEP = ',F16.7)
12600 SET THE NUMBER OF ITERATIONS FOR OTHER LOOP OF MAC METHOD
12700 READ(20,219)NITER
12800 FORMAT(//1A,10)NITER
12900 WRITE(30,216)NITER
13000 FORMAT(1A,10)NUMBER OF ITERATIONS= ',18)

```

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13100
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16800
16900
17000

C SET SWITCH FOR INITIAL SOLUTION (0=START SOLVE FROM AIR AT REST)
C (1=INITIAL SOLUTION PROVIDED IN FIGURE 19)
C READ(20,219)INITSO
C 219 FORMAT(//IX,11)
C *****
C SET UP GRID
C *****
C SET NUMBER OF PRESSURE NODES IN RADIAL DIRECTION
C SET NUMBER OF PRESSURE NODES IN AXIAL DIRECTION
C 209 FORMAT(//IX,2(I2,2X))
C 210 FORMAT(IX,10X=1,12,2X,10X=1,12)
C CALCULATE PRESSURE NODE LOCATIONS
C DELX=RL/PL0AT(CX-1)
C DELY=ZL/PL0AT(CY-1)
C DO 5 I=1,IX
C X(I)=PL0AT(I-1)*DELX
C CONTINUE
C DO 6 J=1,IY
C Y(J)=PL0AT(J-1)*DELY
C CONTINUE
C *****
C SET OR SUBRODITE INPUTS FOR SOLUTION OF POISSON EQUATION
C *****
C SPECIFY TYPE OF BOUNDARY CONDITION ON EACH BOUNDARY
C 0=CIRCULAR BC
C 1=RECTANG BC
C I=X BOUNDARY OR X=NRADIUS BOUNDARY
C I=1 BC
C I=J BOUNDARY OR X=0 BOUNDARY
C I=LEFT=1
C J=Y BOUNDARY OR Y=ALLEG BOUNDARY
C IUP=1

```



```

21500      DO 19 I=1,IX+1
21600      DO 19 J=1,IY
21700      VR(I,J)=0.
21800      CONTINUE
21900      C 19 CALCULATE BOUNDARY VALUES OF VELOCITY
22000      DO 7 I=1,IX
22100      VZ(I,IX+1)=0.
22200      VZ(I,1)=VZ(I,IY+1)
22300      CONTINUE
22400      DO 8 I=1,IX+1
22500      VR(I,1)=0.
22600      VR(I,IY)=0.
22700      CONTINUE
22800      GO TO 52
22900      C 10 INITIAL SOLUTIONS READ FROM FILE 19 IF I-PSOL=1
23000      91 READ(19,93)
23100      93 FORMAT(//////)
23200      DO 491 I=1,NUMPAK
23300      491 READ(19,497)
23400      497 FORMAT(IY)
23500      DO 45 J=1,IY
23600      DO 46 I=1,IX
23700      1X3.Y3.VR(I,J)
23800      46 CONTINUE
23900      READ(19,305)X1,I1,VR(IX+1,J)
24000      45 CONTINUE
24100      DO 47 I=1,IX
24200      READ(19,306)X1,I1,VZ(I,IY+1)
24300      CONTINUE
24400      92 CONTINUE
24500      C*****
24600      C
24700      C ITERATE FOR THE SOLUTION OF THE PRESSURE AND VELOCITY FIELDS
24800      C *****
24900      C
25000      C CALCULATE COEFFICIENTS OF LEFT HAND SIDE OF PDE
25100      C SAME FOR ALL TIME
25200      C
25300      C CALL COEFF(IX,IY,PT50F)
25400      C
25500      C SET SOURCE TERMS EQUAL TO ZERO
25600      C DO 82 I=1,IX
25700      C
25800      C

```

```
25900 DO 82 J=1,NY
26000 FK(I,J)=0.
26100 FZ(I,J)=0.
26200 CONTINUE
26300 C CALCULATE INITIAL SOURCE TERMS
26400 DO 81 I=1,NXPAP
26500 WRITE(28,307)
26600 CALL PARTICL(R,Z,I,0,0Z,DF(I),TOL,FAO(I),FRACO(I))
26700 DO 84 J=1,NY*2
26800 WRITE(28,308)P(J),Z(O),T(O),UF(J),0Z(J)
26900 CONTINUE
27000 C CONTINUE
27100 DO 138 J=1,NY
27200 WRITE(28,351)R(I),FK(2,I),FK(3,I),FK(4,I),FK(5,I)
27300 WRITE(28,351)FZ(1,I),FZ(2,I),FZ(3,I),FZ(4,I),FZ(5,I)
27400 FCONT(1),5(12,5,2X))
27500 CONTINUE
27600 DO 52 IITER=1,6011
27700 C
27800 C UPDATE SOURCE TERM FOR PRESSURE EQUATION USING NEW
27900 C VELOCITY
28000 C CALL PSOURC(OA,AY,PRESSE)
28100 C UPDATE PARTICLE FIELD AND PARTICLE SOURCE TERMS
28200 C EVERY TENth ITERATION
28300 B=L+1
28400 IF(L-I,10)GO TO 234
28500 B=0
28600 C SET SOURCE TERMS EQUAL TO ZERO
28700 DO 83 I=1,NA
28800 DO 83 J=1,NY
28900 FK(I,J)=0.
29000 FZ(I,J)=0.
29100 C CONTINUE
29200 C RECALCULATE SOURCE TERMS
29300 DO 85 I=1,NXPAP
29400 CALL PARTICL(R,Z,I,0F,0Z,DF(I),TOL,FAO(I),FRACO(I))
29500 CONTINUE
29600 C
29700 C
```

```

29800 C ITERATE FOR NEW PRESSURE FIELD USING SUB ROUTINE
29900 C CALL SUB(PRESSR,ITER,MOSEGA,IX,MY,ITERMAX)
30000 C
30100 C OBTAIN NEW VELOCITY FIELD
30200 C CALL GASVEL(IX,MY,VZU,VNU,PRESSR)
30300 C
30400 C STORE NEW VELOCITY FIELD AND CALCULATE CAUCHY RESIDUAL
30500 C VDR=0.
30600 C SUBRUK=0.
30700 C DO 29 J=1,IX+1
30800 C DO 29 J=1,IX
30900 C VDR=VDR+(VNU(I,J)-VNU(I,J))**2
31000 C SUBRUK=SUBRUK+VDR(I,J)**2
31100 C VU(I,J)=VNU(I,J)
31200 C CONTINUE
31300 C DO 28 J=1,IX+1
31400 C DO 28 J=1,IX
31500 C VDR=VDR+(VZU(I,J)-VZU(I,J))**2
31600 C SUBRUK=SUBRUK+VZU(I,J)**2
31700 C VZ(I,J)=VZU(I,J)
31800 C CONTINUE
31900 C VDR=SQRT(VDR/SUBRUK)
32000 C IF SOLUTION STARTS TO BLOW UP STOP PROGRAM
32100 C IF(SUBRUK/FLOAT(IX*IX).GT.100.)STOP
32200 C RAD=PI/2+1
32300 C RHE=PI/2+1
32400 C IF(L.EQ.0)WRITE(6,53)ITER,VDR,VU(I,J),VZ(I,J),VNU(I,J),
32500 C PRESSR(IX,MY),VU(2,MY),VZ(2,MY),PRESSR(2,MY)
32600 C 53 FORMAT(IX,I4,ZX,E12.6,6(2X,F10.5))
32700 C DO 51 J=1,IX+1
32800 C DO 51 J=1,IX
32900 C WRITE(30,202)X(I),Y(J),PRESSR(I,J),VZ(I,J),VNU(I,J)
33000 C CONTINUE
33100 C CONTINUE
33200 C *****
33300 C OBTAIN SOLUTION WITH PRESSURE AND VELOCITY FIELDS
33400 C AND PARTICLE TRAJECTORIES
33500 C *****
33600 C *****
33700 C *****
33800 C *****
33900 C *****
34000 C *****
34100 C *****
34200 C *****
34300 C *****
34400 C *****
34500 C *****
34600 C *****

```

```

34700
34800
34900
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37600
37700
37800
37900
38000
38100
38200

303 FORMAT(IX,2X,'POSITION',4X,'POSITION',4X,'POSITION',4X,'POSITION')
1 POSITION',4X,'POSITION',15X,'POSITION',4X,'POSITION')
DO 35 J=1,NX
DO 36 I=1,NX
XIH=X(I)-DELX/2.
YIH=Y(J)-DELY/2.
WRITE(30,304)X(I),Y(J),PRESSR(I,J),X(I),YIH,VZ(I,J)
1,XIH,Y(J),VR(I,J)
36 CONTINUE
304 FORMAT(IX,5(F10.5,2X))
XIH=X(I)+DELX/2.
WRITE(30,305)X(I),Y(J),VR(I,J)
35 CONTINUE
305 FORMAT(IX,72X,3(F10.5,2X))
YIH=Y(J)+DELY/2.
DO 37 I=1,NX
WRITE(30,306)X(I),YIH,VZ(I,J)
37 CONTINUE
306 FORMAT(IX,36X,3(F10.5,2X))
WRITE(30,307)
307 FORMAT(IX,11X,'MATERIAL',2X,'AXIAL DIS',2X,' LINE '
1X,'MATERIAL',2X,'AXIAL VEL')
CALL PARTICLE TRACK FOR DP= ',F10.5,2X,'THETA= ',
1F10.5,2X,'VOLUME FLOW FRACTION= ',F10.5)
WRITE(30,307)
1X,'RADIUS ',2X,'AXIAL DIS',2X,' LINE '
1X,'MATERIAL',2X,'AXIAL VEL')
CALL PARTICLE TRACK FOR DP= ',F10.5,2X,'THETA= ',
1F10.5,2X,'VOLUME FLOW FRACTION= ',F10.5)
WRITE(30,308)R(I),Z(I),I(I),DE(I),RZ(I)
38 CONTINUE
310 CONTINUE
308 FORMAT(IX,5(F10.5,2X))
STOP
END

```

COEFF SUBROUTINE

```

100 SUBROUTINE COEFF(COX,COY,GUESS)
200 DIMENSION CW(50,50),Cw(50,50),CSE(50,50),CSW(50,50)
300 DIMENSION C(50,50),CP(50,50),F(50,50)
400 DIMENSION XC(50),Y(50),GROSS(50,50)
500 DIMENSION AXC(50),AYC(50),CBC(50),CBC(50)
600 DIMENSION FODD(50,50)
700 COMMON CUE,CW,CSE,CSW,CH,CS,CE,CV,CP,F
800 COMMON /BOUNDARY1/AXC,AYC,CBC,CBC,ACROSS,PCROSS,CROSS,DCROSS
900 COMMON /BOUNDARY2/ ILEFT,IRIGHT,ITOP,BOTTOM
1000 COMMON /BOUNDARY3/ FODD
1100 COMMON /DELTA/ DELTA,DEBY,DELT
1200 COMMON /COORD/ X,Y
1300 *****
1400 *****
1500 THIS SUBROUTINE CALCULATES THE TOP LEFT HAND SIDE OF THE
1600 DISCRETIZED PDE FOR THE POISSON TYPE EQUATION FOR
1700 THE PRESSURE
1800 *****
1900 *****
2000 *****
2100 *****
2200 *****
2300 *****
2400 *****
2500 *****
2600 *****
2700 *****
2800 *****
2900 *****
3000 *****
3100 *****
3200 *****
3300 *****
3400 *****
3500 *****
3600 *****
3700 *****
3800 *****
3900 *****
4000 *****
4100 *****

```

***** INPUT TO THIS SUBPROGRAM IS AS FOLLOWS:

"COX" AND "COY" ARE THE NUMBER OF PRESSURE NODES IN THE X AND Y DIRECTIONS

X=RADIAL DIRECTION, Y=AXIAL DIRECTION

"GUESS" IS THE GUESSED SOLUTION VECTOR FOR USE WITH ADAPTIVE COEFFICIENTS (NOT USED HERE)

***** INPUT FROM COMMON BLOCK

THE "BOUNDARY" COMMON BLOCK CONTAINS THE INFORMATION FOR THE FIRST AND LAST NODES AT THE FOUR BOUNDARIES. THE FIRST FOUR VECTORS CONTAIN THE COEFFICIENTS OF THE ORIGINAL DERIVATIVES. THE LAST FOUR VECTORS DEAL WITH THE CROSS DERIVATIVES AT THE FOUR CORNERS. SEE WRITE-UP FOR MORE INFORMATION

THE "BOUNDARY2" COMMON BLOCK CONTAINS THE SWITCHES FOR THE TYPE OF BOUNDARY CONDITION AT THE FOUR BOUNDARIES.

=1 FOR PERMANENT BOUNDARY CONDITIONS
=0 FOR DIRICHLET BOUNDARY CONDITIONS

```

4200
4300
4400
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4600
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4900
5000
5100
5200
5300
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6000
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6400
6500
6600
6700
6800
6900
7000
7100
7200
7300
7400
7500
7600
7700
7800
7900
8000
8100
8200

```

THE "BOUNDARY3" CORNER BLOCK CONTAINS THE COEFFICIENTS TO
 THE RIGHT HAND SIDE OF THE PDE DUE TO PERMANENT BOUNDARY
 CONDITIONS

 THE "DEL" CORNER BLOCK CONTAINS THE RADIAL AND AXIAL
 INCREMENTAL LENGTHS "DELX" AND "DELY", AND THE
 INCREMENTAL TIME "DELT"

 THE "CORNER" CORNER BLOCK CONTAINS THE RADIAL AND AXIAL
 COORDINATES X AND Y OF THE PRESSURE NODES

 ***** THE CORNER BLOCK *****

 THE CORNER BLOCK MUST BE DEFINED AT EACH POINT IN THE
 DISCRETIZED PDE AS A FUNCTION OF THE MOLECULE'S POSITION IN THE DOMAIN. MOST
 OF THESE COEFFICIENTS ARE CALCULATED IN THIS SUBROUTINE AND
 ARE DEFINED BELOW:
 CN-COEFF OF NORTH POINT
 CW-COEFF OF WEST POINT
 CSW-COEFF OF SOUTHWEST POINT
 CS-COEFF OF SOUTH POINT
 CSE-COEFF OF SOUTHEAST POINT
 CE-COEFF OF EAST POINT
 C-COEFF OF CENTRAL POINT
 F -RIGHT HAND SIDE OF DISCRETIZED PDE (CALCULATED IN
 SUBROUTINE PSODEC)

 ***** CALCULATE COEFFICIENTS OF UPPER POINTS *****
 DO 50 I=2,NX-1
 DO 50 J=2,NY-1
 C(I,J)=0.
 C(I,J)=-C(I,J)
 C(I,J)=-C(I,J)

```

8300      CS*(1,J)=C*(1,J)
8400      CCI=1./DELX**2
8500      CA=1./DELX**2
8600      CCF=0.
8700      CEX=(1./X(1))/(2.*DELX)
8800      CCI(1,J)=CCI+CCI
8900      CS(1,J)=CC1-CBY
9000      CE(1,J)=CA+CBX
9100      CW(1,J)=CA-CBX
9200      CCI(1,J)=-2.*CA-2.*CC1
9300      FLOW(1,J)=0.
9400      CCF(1,J)=0.
9500
9600      C*****
9700      C**MODIFY COEFFICIENTS OF INTERIOR POINTS NEXT TO BOUNDARY
9800      C**FOR "Flooded Conditions"
9900      C*****
10000     C*****
10100     C**MODIFY COEFFICIENTS AT LEFT BOTTOM CORNER FOR WEIRAGE
10200     C**CONDITIONS ON BUFFER AND/OR LEFT SIDES
10300     C**
10400     C**
10500     C**
10600     C**
10700     C**
10800     C**
10900     C**
11000     C**
11100     C**
11200     C**
11300     C**
11400     C**
11500     C**
11600     C**
11700     C**
11800     C**
11900     C**
12000     C**
12100     C**
12200     C**
12300     C**
12400     C**
12500     C**
12600     C**
12700     C**
12800     C**

```

```

CS*(1,J)=C*(1,J)
CCI=1./DELX**2
CA=1./DELX**2
CCF=0.
CEX=(1./X(1))/(2.*DELX)
CCI(1,J)=CCI+CCI
CS(1,J)=CC1-CBY
CE(1,J)=CA+CBX
CW(1,J)=CA-CBX
CCI(1,J)=-2.*CA-2.*CC1
FLOW(1,J)=0.
CCF(1,J)=0.

```

MODIFY COEFFICIENTS OF INTERIOR POINTS NEXT TO BOUNDARY
FOR "Flooded Conditions"

MODIFY COEFFICIENTS AT LEFT BOTTOM CORNER FOR WEIRAGE
CONDITIONS ON BUFFER AND/OR LEFT SIDES

```

CP(2,2)=CP(2,2)+CW(2,2)+FLOWAT(LEFT)+CS(2,2)
1*FLOWAT(LEFT)+CSE(2,2)+FLOWAT(LEFT)+1*BOUIM)
C(2,2)=C(2,2)+CW(2,2)+FLOWAT(LEFT)
CE(2,2)=CE(2,2)+CSE(2,2)+FLOWAT(LEFT)
FLOW(2,2)=-((-CW(2,2)+DELTA*BOC(2)-C*W(2,2)*DELX*BOC(3))
1*FLOWAT(LEFT)+(-CS(2,2)+DELTA*BOC(2)-CSE(2,2)*DELX*BOC(3))
2*FLOWAT(LEFT)+(C*W(2,2)+(-DELTA*BOC(2)-DELX*BOC(2)+DELX*DELX*
3*CCF(2,2))*FLOWAT(LEFT)+1*BOUIM)
C*(2,2)=C*(2,2)+FLOWAT(1-LEFT)/2)
C*(2,2)=C*(2,2)+FLOWAT(1-LEFT)
C*(2,2)=C*(2,2)+FLOWAT(1-LEFT)
CSE(2,2)=CSE(2,2)+FLOWAT(1-BOUIM)
CSE(2,2)=CSE(2,2)+FLOWAT(1-BOUIM)

```

MODIFY COEFFICIENTS NEXT TO LEFT BOUNDARY FOR WEIRAGE
CONDITIONS ON LEFT SIDE

```

DO 10 J=3,NY-2
C*(2,J)=C*(2,J)+CW(2,J)*FLOWAT(LEFT)
CE(2,J)=CE(2,J)+C*(2,J)*FLOWAT(LEFT)
CS(2,J)=CS(2,J)+C*(2,J)*FLOWAT(LEFT)
FLOW(2,J)=-(-C*(2,J)+DELTA*BOC(J+1)-C*(2,J)*DELX*BOC(J)
1-C*(2,J)*DELX*BOC(J-1))*FLOWAT(LEFT)
C*(2,J)=C*(2,J)+FLOWAT(1-LEFT)

```

```

14900 C Cw(2,J)=Cw(2,J)*FLOAT(1-ILEFT)
15000 CSw(2,J)=Csw(2,J)*FLOAT(1-ILEFT)
15100 CONTINUE
15200
15300 C MODIFY COEFFICIENTS AT LEFT TOP CORNER FOR MESH AND
15400 CONDITIONS ON TOP AND/OR LEFT SIDES
15500 CS(2,MY-1)=CS(2,MY-1)+Csw(2,MY-1)*FLOAT(ILEFT)
15600 C1(2,MY-1)=C1(2,MY-1)+Cw(2,MY-1)*FLOAT(ILEFT)+
15700 1*Cw(2,MY-1)*FLOAT(ITOP)+Csw(2,MY-1)*FLOAT(ILEFT)*ITOP
15800 C2(2,MY-1)=C2(2,MY-1)+Cw(2,MY-1)*FLOAT(ITOP)
15900 ICROSS=ILEFT*ITOP
16000 FOCUSD(2,MY-1)=-((C1(2,MY-1)+DELA*BC(MY-1)-CSw(2,MY-1)
16100 1*DELA*BC(MY-2))*FLOAT(ILEFT))+(-C1(2,MY-1)*DELY*ABC(2)-
16200 2*C1(2,MY-1)*DELY*ABC(3))*FLOAT(ITOP)+Cw(2,MY-1)*(-DELA
16300 3*DECC(MY-1)-DELY*ABC(2)-DELA*DELY*BCROSS))*FLOAT(ICROSS)
16400 CSw(2,MY-1)=CSw(2,MY-1)*FLOAT(1-ILEFT)
16500 Cw(2,MY-1)=Cw(2,MY-1)*FLOAT(1-ILEFT)
16600 Csw(2,MY-1)=Csw(2,MY-1)*FLOAT(1-ILEFT)
16700 C1(2,MY-1)=C1(2,MY-1)*FLOAT(1-ILEFT)
16800 C2(2,MY-1)=C2(2,MY-1)*FLOAT(1-ITOP)
16900 C3(2,MY-1)=C3(2,MY-1)*FLOAT(1-ITOP)
17000 CONTINUE
17100
17200 C MODIFY COEFFICIENTS NEXT TO TOP BOUNDARY FOR MESH AND
17300 CONDITIONS ON TOP SIDE
17400 DO 20 I=3,NA-2
17500 C1(1,MY-1)=C1(1,MY-1)+Cw(1,MY-1)*FLOAT(ITOP)
17600 C2(1,MY-1)=C2(1,MY-1)+C1(1,MY-1)*FLOAT(ITOP)
17700 C3(1,MY-1)=C3(1,MY-1)+C1(1,MY-1)*FLOAT(ITOP)
17800 FOCUSD(1,MY-1)=-((C1(1,MY-1)+Cw(1,MY-1)*DELY*ABC(1-1)-
17900 1*C1(1,MY-1)*DELY*ABC(1-1)-C1(1,MY-1)*DELA*ABC(1+1))*FLOAT(ITOP)
18000 Cw(1,MY-1)=Cw(1,MY-1)*FLOAT(1-ITOP)
18100 C1(1,MY-1)=C1(1,MY-1)*FLOAT(1-ITOP)
18200 C2(1,MY-1)=C2(1,MY-1)*FLOAT(1-ITOP)
18300 CONTINUE
18400
18500 C MODIFY COEFFICIENTS AT RIGHT TOP CORNER FOR MESH AND
18600 CONDITIONS ON TOP AND/OR RIGHT SIDES
18700 C1=NA-1
18800 C2=NY-1
18900 C3(NA,MY)=C3(NA,MY)+Csw(NA-1,MY-1)*FLOAT(IRIGHT)
19000

```

```

17100 CF(CX1,XY1)=CF(CX1,XY1)+CF(CX1,XY1)*FLOAF(I,RIGHT)
17200 1+CF(CX1,XY1)*FLOAF(I,TOP)+CF(CX1,XY1)*FLOAF(I,TOP)*F(I,RIGHT)
17300
17400 CW(CX1,XY1)=CW(CX1,XY1)+CW(CX1,XY1)*FLOAF(I,TOP)
17500 ICROSS=IRIGHT*I,TOP
17600 F00J0D(CX1,XY1)=-((-CF(CX1,XY1)*DELA*DBC(CY-1)
17700 1-CSE(CX1,XY1)+DELY*DBC(CX-2)))*FLOAF(I,RIGHT)+(-CW(CX1,XY1)*
17800 2*DELY*ABC(CY-1)-CW(CX1,XY1)*DELY*ABC(CX-2))*FLOAF(I,TOP)+
17900 3(CW(CX1,XY1)+(-DELA*DBC(CY1)-DELY*ABC(CX1)))*FLOAF(I,RIGHT)
18000 4*CF(CROSS)))*FLOAF(I,CROSS)
18100 CW(CX1,XY1)=CW(CX1,XY1)+FLOAF(I,I,TOP)
18200 CF(CX1,XY1)=CF(CX1,XY1)*FLOAF(I,I,TOP)
18300 CW(CX1,XY1)=CW(CX1,XY1)*FLOAF(I,I,TOP)/2)
18400 CF(CX1,XY1)=CF(CX1,XY1)*FLOAF(I,I,RIGHT)
18500 CSE(CX1,XY1)=CSE(CX1,XY1)*FLOAF(I,I,RIGHT)
18600
18700
18800
18900
19000
19100
19200
19300
19400
19500
19600
19700
19800
19900
20000
20100
20200
20300
20400

```

C MODIFY COEFFICIENTS NEXT TO RIGHT BOUNDARY FOR FLUWIDE
 C CONDITIONS ON RIGHT SIDE
 C

DO 30 J=3,MY-2
 CM(CX-1,J)=CM(CX-1,J)+CM(CX-1,J)*FLOAF(I,RIGHT)
 CP(CX-1,J)=CP(CX-1,J)+CF(CX-1,J)*FLOAF(I,RIGHT)
 CS(CX-1,J)=CS(CX-1,J)+CSE(CX-1,J)*FLOAF(I,RIGHT)
 F00J0D(CX-1,J)=-(-CF(CX-1,J)+DELY*ABC(CY+1)-
 2*FLOAF(I,RIGHT)+DELA*DBC(CJ)-CSE(CX-1,J)*FLOAF(I,I,RIGHT)))*
 FLOAF(I,RIGHT)
 CW(CX-1,J)=CW(CX-1,J)+FLOAF(I,I,RIGHT)
 CF(CX-1,J)=CF(CX-1,J)*FLOAF(I,I,RIGHT)
 CSE(CX-1,J)=CSE(CX-1,J)*FLOAF(I,I,RIGHT)
 C

30
 C MODIFY COEFFICIENTS AT RIGHT BOUNDY CORNER FOR FLUWIDE
 C CONDITIONS ON BUTTLE AND/OR FLIGHT SLIPS
 C

```

20500 C0(CX-1,2)=C0(CX-1,2)+C0(CX-1,2)*FLOAF(RIGHT)
20600 C1(CX-1,2)=C1(CX-1,2)+C1(CX-1,2)*FLOAF(LEFT)+C0(CX-1,2)*
20700 FLOAF(LEFT)+C1(CX-1,2)*FLOAF(LEFT)+FLOAF(LEFT)
20800 C2(CX-1,2)=C2(CX-1,2)+C2(CX-1,2)*FLOAF(LEFT)
20900 FLOAF(LEFT)
21000 1-DELTA*DBC(3)*FLOAF(RIGHT)+(-C0(CX-1,2)+FLOAF(LEFT)-C0(CX-1,2))*
21100 2-C0(CX-1,2)*DELTA*CBC(CX-2)*FLOAF(LEFT)+C0(CX-1,2)*
21200 3(-DELTA*DBC(2)-DELTA*CBC(CX-1)-DELTA*DELTA*DBC(2))*FLOAF(
21300 FLOAF(LEFT))
21400 C0(CX-1,2)=C0(CX-1,2)+FLOAF(LEFT)
21500 C1(CX-1,2)=C1(CX-1,2)+FLOAF(LEFT)
21600 C2(CX-1,2)=C2(CX-1,2)+FLOAF(LEFT)+FLOAF(LEFT)/2)
21700 C3(CX-1,2)=C3(CX-1,2)+FLOAF(LEFT)
21800 C4(CX-1,2)=C4(CX-1,2)+FLOAF(LEFT)
21900
22000
22100
22200
22300
22400
22500
22600
22700
22800
22900
23000
23100
23200
23300
23400
23500

```

CC

COEFFICIENTS NEXT TO BOTTOM BOUNDARY FOR REWARD

CONDITIONS ON BOTTOM SIDE

```

00 40 I=3,JA=2
C0(1,2)=C0(1,2)+C0(1,2)*FLOAF(LEFT)
C1(1,2)=C1(1,2)+C1(1,2)*FLOAF(LEFT)
C2(1,2)=C2(1,2)+C2(1,2)*FLOAF(LEFT)
FLOAF(LEFT)
1C0(1,2)=C0(1,2)+C0(1,2)*DELTA*CBC(1-1)-C0(1,2)*DELTA*CBC(1)-
C0(1,2)+C0(1,2)*FLOAF(LEFT)
C1(1,2)=C1(1,2)+FLOAF(LEFT)
C2(1,2)=C2(1,2)+FLOAF(LEFT)
C3(1,2)=C3(1,2)+FLOAF(LEFT)
C4(1,2)=C4(1,2)+FLOAF(LEFT)

```

40

COEFFICIENTS ON BOTTOM SIDE

PSOURC SUBROUTINE

```

100  SUBROUTINE PSOURC(CX, CY, GORSS)
200  DIMENSION CWF(50,50), CCF(50,50), CSE(50,50), CSA(50,50)
300  DIMENSION CW(50,50), CH(50,50), CS(50,50)
400  DIMENSION CE(50,50), CP(50,50), F(50,50), GUESS(50,50)
500  DIMENSION VR(50,50), VZ(50,50), X(50), Y(50), FROD(50,50)
600  DIMENSION ABC(50), BEC(50), CFC(50), DMC(50)
700  DIMENSION FR(50,50), FZ(50,50)
800  COMMON /CWF/ CWF, CCF, CSE, CSW, CU, CS, CF, CW, CCF, F
900  COMMON /DEP/ DELX, DELY, DELT
1000 COMMON /AIR V/ VZ, VR
1100 COMMON /ROTORAF Y3/ FROD
1200 COMMON /CORNU/ X, Y
1300 COMMON /WEYRDS/ FE
1400 COMMON /SOURCE/ FR, FZ
1500 C*****
1600 C THIS SUBROUTINE CALCULATES THE RIGHT HAND SIDE OF THE
1700 C DISCRETIZED PDE FOR THE PULSSUB TYP EQUATION. FOP

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1800 C THE PRESSURE.
1900 C
2000 C ***HELPFUL TO THIS SUBPROGRAM IS AS FOLLOWS:
2100 C
2200 C "IX" AND "IY" ARE THE NUMBER OF PRESSURE NODES IN THE
2300 C X AND Y DIRECTIONS
2400 C
2500 C X=RADIAL DIRECTION, Y=AXIAL DIRECTION
2600 C
2700 C "GUESS" IS THE GUESSED SOLUTION VECTOR FOR USE WITH
2800 C NONLINEAR COEFFICIENTS
2900 C
3000 C THE "DEL" COMMON BLOCK CONTAINS THE RADIAL AND AXIAL
3100 C INCREMENTAL DELTAS AND "DELA" AND "DELT", AND THE
3200 C INCREMENTAL TIME "DELT"
3300 C
3400 C THE "AIR V" COMMON BLOCK CONTAINS THE RADIAL AND AXIAL
3500 C COORDINATES X AND Y OF THE PRESSURE NODES
3600 C
3700 C THE "BOUNDARY3" COMMON BLOCK CONTAINS THE CONTRIBUTIONS
3800 C TO THE RIGHT HAND SIDE OF THE PLE DUE TO "BOUNDARY
3900 C BOUNDARY CONDITIONS
4000 C
4100 C THE "REYNOLDS" COMMON BLOCK CONTAINS THE REYNOLDS NUMBER
4200 C OF THERMIONIZATION
4300 C
4400 C THE "SOURCE" COMMON BLOCK CONTAINS THE SOURCE TERMS OF THE
4500 C MOMENTUM EQUATIONS
4600 C
4700 C ***ALSO REQUIRED AS INPUT FOR THIS SUBPROGRAM ARE THE SET OF
4800 C FUNCTION SUBPROGRAMS WHICH CALCULATE THE SQUARE OF THE
4900 C VELOCITIES, THE CROSS MULTIPLICATION OF THE VELOCITIES,
5000 C AND THE COLLISION TERM. THESE ARE GIVEN AFTER THIS
5100 C THIS PROGRAM.
5200 C
5300 C ***OUTPUT THRU THE COMMON BLOCK
5400 C
5500 C THE COMMON BLOCK OUT LINED CONTAINS THE COEFFICIENTS OF THE
5600 C DISCRETIZED PDE AT EACH POINT IN THE COMPUTATIONAL MOLECULE
5700 C AS A FUNCTION OF THE MOLECULE'S POSITION IN THE DOMAIN. ONE
5800 C OF THESE COEFFICIENTS IS CALCULATED IN THIS SUBROUTINE.

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10760 10  CONTINUE
10770 C  MODIFY RIGHT HAND SIDE AT LEFT FOR CORNER FOR REMAIN
10780 C  CONDITIONS ON TOP AND/OR LEFT SIDES
10790 C
10800 C  F(CX,XY-1)=F(CX,XY-1)+FBOUND(2,XY-1)
10810 C
10820 C  MODIFY RIGHT HAND SIDE NEXT TO TOP BOUNDARY FOR REMAIN
10830 C  CONDITIONS ON TOP SIDE
10840 C
10850 C  DO 20 I=3,MAXZ
10860 C  F(CI,XY-1)=F(CI,XY-1)+FBOUND(1,XY-1)
10870 C  CONTINUE
10880 C
10890 C  MODIFY RIGHT HAND SIDE AT RIGHT FOR CORNER FOR REMAIN
10900 C  CONDITIONS ON TOP AND/OR RIGHT SIDES
10910 C
10920 C  XAI=IX-1
10930 C  XBI=XI-1
10940 C  F(CX,XYI)=F(CX,XYI)+FBOUND(CX,XYI)
10950 C
10960 C  MODIFY RIGHT HAND SIDE NEXT TO RIGHT BOUNDARY FOR REMAIN
10970 C  CONDITIONS ON RIGHT SIDE
10980 C
10990 C  DO 30 J=3,XY-2
11000 C  F(CX-1,J)=F(CX-1,J)+FBOUND(CX-1,J)
11010 C  CONTINUE
11020 C
11030 C  MODIFY RIGHT HAND SIDE AT RIGHT BOUNDARY FOR REMAIN
11040 C  CONDITIONS ON BOTTOM AND/OR RIGHT SIDES
11050 C
11060 C  F(CX-1,2)=F(CX-1,2)+FBOUND(CX-1,2)
11070 C
11080 C  MODIFY RIGHT HAND SIDE NEXT TO BOTTOM BOUNDARY FOR REMAIN
11090 C  CONDITIONS ON BOTTOM SIDE
11100 C
11110 C  DO 40 I=3,MAXZ
11120 C  F(CI,2)=F(CI,2)+FBOUND(CI,2)
11130 C  CONTINUE
11140 C  RETURN
11150 C  END
11160 C
11170 C
11180 C
11190 C
11200 C
11210 C
11220 C
11230 C
11240 C
11250 C
11260 C
11270 C
11280 C
11290 C
11300 C
11310 C
11320 C
11330 C
11340 C
11350 C
11360 C
11370 C
11380 C
11390 C
11400 C
11410 C
11420 C
11430 C
11440 C
11450 C
11460 C
11470 C
11480 C
11490 C

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15000 C*****
15100 C FUNCTION SUBPROGRAM FOR SUBROUTINES PSUBMC AND GASVEL.
15200 C
15300 C
15400 C COMPUTES THE SQUARE OF THE RADIAL VELOCITY
15500 C FUNCTION VRSQR(I,J)
15600 DIMENSION VP(50,50),VZ(50,50)
15700 COMMON /AIR VZ,VP
15800 VRSQR=.25*(VP(I+1,J)+VP(I,J))**2
15900 RETURN
16000 END
16100 C
16200 C COMPUTES THE SQUARE OF THE AXIAL VELOCITY
16300 C FUNCTION VZSR(I,J)
16400 DIMENSION VZ(50,50),VR(50,50)
16500 COMMON /AIR VZ,VR
16600 VZSR=.25*(VZ(I,J+1)+VZ(I,J))**2
16700 RETURN
16800 END
16900 C
17000 C COMPUTES THE QUANTITY VR*VZ AT PRESSURE NODE I,J
17100 C FUNCTION VZVR(I,J)
17200 DIMENSION VZ(50,50),VR(50,50)
17300 COMMON /AIR VZ,VR
17400 VZVR=.25*(VR(I+1,J)+VR(I,J))*(VZ(I,J+1)+VZ(I,J))
17500 RETURN
17600 END
17700 C
17800 C COMPUTES THE QUANTITY VR*VZ AT THE CORNER OF A RECTANGLE
17900 C FORMED BY FOUR PRESSURE NODES (I-1,J-1),I,J,I,J+1) IN THE
18000 C PLANE.
18100 C FUNCTION VZVZ(I,J)
18200 DIMENSION VZ(50,50),VR(50,50)
18300 COMMON /AIR VZ,VR
18400 VZVZ=.25*(VR(I,J)+VR(I-1,J))*(VZ(I,J)+VZ(I-1,J))
18500 RETURN
18600 END
18700 C
18800 C COMPUTES THE DILATION TERM
18900 C FUNCTION D(I,J,K)
19000 DIMENSION VR(50,50),VZ(50,50)
19100 COMMON /AIR VZ,VR
19200 CELT=(VR(I+1,J)-VR(I,J))/DELX
19300 IF (K.EQ.0)GO TO 1
19400 CELT=(VR(I+1,J)*(K+DELX/2.)-VR(I,J)*(K-DELX/2.))
19500 /((K+DELX)
19600 )/(K+DELX)
19700 END
19800 C
19900 C
20000 C

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SOR SUBROUTINE

```

100 SUBROUTINE SOR(SUBM, ITP, AORGG, IA, AT, IIRMAX)
200 DIMENSION SOLN(50,50), GUESS(50,50)
300 DIMENSION CR(50,50), CM(50,50), CS*(50,50), CW*(50,50)
400 DIMENSION CH(50,50), CE(50,50), CC(50,50), CC*(50,50)
500 DIMENSION CP(50,50), FC(50,50)
600 DIMENSION ABC(50), ABC*(50), CBC(50), DEC(50)
700 DIMENSION CDE, CDE*, CS*, CE, CC*, CF, CW, CP, F
800 COMMON /BOUNDARY/ ABC, ABC*, CBC, CBC*, CC*, CC*, CF, CW, CP, F
900 COMMON /BOUNDARY2/ ILEFT, IRIGHT, ITOP, IBOULR
1000 COMMON /DELT/ DELTA, DELTA*, DELT
1100 C*****
1200 C THIS PROGRAM USES THE SUCCESSIVE-OVER-RELAXATION SCHEME TO
1300 C ITERATE FOR THE ELLIPTIC EQUATION SOLUTION
1400 C
1500 C ***** INPUT TO THIS SUBPROGRAM IS AS FOLLOWS:
1600 C
1700 C "SUBM" HOLDS THE GUESSED SOLUTION VECTOR INITIALLY
1800 C AFTER EACH ITERATION "SOLN" HOLDS THE UPDATED SOLUTION
1900 C AND "GUESS" HOLDS THE OLD SOLUTION.
2000 C
2100 C DETAILS ON THE GUESSED SOLUTION
2200 C THE GUESSED SOLUTION CAN BE CHOSEN AT BOUNDARY POINTS AS THE
2300 C USER WISHES IT. AT THE BOUNDARIES THE GUESSED SOLUTION MUST
2400 C CONTAIN THE VALUES OF THE SOLUTION AT THE BOUNDARIES WHERE THE
2500 C DIFFICULTY CONDITIONS EXIST, INCLUDING CORNER POINTS. ALONG THE
2600 C BOUNDARIES WHERE THE DROPCAP CONDITIONS EXIST THE USER MAY
2700 C SUPPLY GUESSED VALUES THROUGH THESE ARE USED IN THE
2800 C CALCULATION
2900 C
3000 C "AORGG" CONTAINS THE FIRST GUESSED VALUE OF THE RELAXATION
3100 C FACTOR, UPDATED AT SUCCESSIVE ITERATIONS BY USING THE RATIO
3200 C OF THE RESIDUALS AT SUCCESSIVE ITERATIONS.
3300 C
3400 C "IA" AND "ITP" ARE THE NUMBER OF DISCRETIZED POINTS IN
3500 C THE X AND Y DIRECTIONS
3600 C
3700 C "IIRMAX" IS THE MAXIMUM NUMBER OF ITERATIONS TO BE ALLOWED.
3800 C IF THE SOLUTION DOES NOT CONVERGE BY IIRMAX ITERATIONS A
3900 C MESSAGE IS PRINTED OUT OF THE SCREEN
4000 C

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4100 C***THEORY THROUGH THE CORNER INDICES ARE AS FOLLOWS:
4200 C
4300 C THE CORNER BLOCK REF DATA CONTAINS THE COEFFICIENTS OF THE
4400 C DISCRETIZED FOR AT EACH POINT IN THE COMPUTATIONAL MESH FOR
4500 C FOR EACH COMPUTATIONAL SUBBLOCK. THIS INFORMATION IS
4600 C CALCULATED IN THE SUBROUTINE CORN.
4700 C
4800 C THE "BOUNDARY" CORNER BLOCK CONTAINS THE INFORMATION FOR THE
4900 C BEST AND BOUNDARY CONDITIONS AT THE FOUR BOUNDARIES. THE
5000 C FIRST FOUR VECTORS CONTAIN THE COEFFICIENTS OF THE NORMAL
5100 C DERIVATIVES. THE LAST FOUR ENTRIES DEAL WITH THE CROSS
5200 C DERIVATIVES AT THE FOUR CORNERS. SEE ARTICLE-UP FOR MORE
5300 C INFORMATION.
5400 C
5500 C THE "BOUNDARY2" CORNER BLOCK CONTAINS THE SWITCHES FOR THE
5600 C TYPE OF BOUNDARY CONDITIONS AT THE FOUR BOUNDARIES
5700 C =1 FOR MIDDLE BOUNDARY CONDITIONS
5800 C =2 FOR DIAPHRAGM BOUNDARY CONDITIONS
5900 C THE BOUNDARY IS DEFINED AS BOUNDARY WITH J=1
6000 C RIGHT BOUNDARY IS DEFINED AS BOUNDARY WITH I=1
6100 C LEFT BOUNDARY IS DEFINED AS BOUNDARY WITH I=1
6200 C
6300 C THE "DEL" CORNER BLOCK CONTAINS THE RADIAL AND AXIAL
6400 C INCREMENTAL LENGTHS "DELA" AND "DELT", AND THE
6500 C INCREMENTAL TIME "DELT"
6600 C
6700 C
6800 C
6900 C***OUTPUT OF THE SUBPROGRAM IS AS FOLLOWS:
7000 C
7100 C "SOL" IS THE SOLUTION VECTOR SOLVED FOR DURING THE ITERATIONS
7200 C IT REPRESENTS THE UPDATED SOLUTION VECTOR DURING THE ITERATIONS
7300 C AND THE FINAL SOLUTION AT CONVERGENCE
7400 C
7500 C "ITER" IS THE NUMBER OF ITERATIONS REQUIRED FOR CONVERGENCE
7600 C
7700 C***OUTPUT I/O DATA FILES
7800 C
7900 C THE VALUES OF THE RESIDUAL, THE RESIDUAL NORMALIZED BY THE
8000 C SOLUTION, AND THE RELAXATION FACTOR AT EACH ITERATION ARE
8100 C STORED IN FILE FOR031.DAT
8200 C
C*****

```

8300 C PUT GUESSED SOLUTION INTO SOLUTION VECTOR
8400 DO 5 I=1,IX
8500 DO 5 J=1,IY
8600 GUESS(I,J)=SUBM(I,J)
8700 C=V(I,I)
8800
8900 C START ITERATION FOR SOLUTION
9000 DO 100 IITER=1,IITERMAX
9100 C DETERMINE SUBM AT INFERIOR POINTS FOR CURRENT SWEEP
9200
9300 C IITERVE=(IITER+1)/2-IITER/2
9400 C FOR IITER ODD SWEEP DIRECTION IS FORWARD, IITER EVEN SWEEP
9500 C DIRECTION IS BACKWARD
9600 IITERVE=0 FOR IITER=EVEN, IITERVE=1 FOR IITER=ODD
9700 JFACT=0
9800 JFACT=0
9900 JFACT=0
10000 JFACT=0
10100 JFACT=0
10200 JFACT=0
10300 JFACT=0
10400 JFACT=0
10500 JFACT=0
10600 JFACT=0
10700 JFACT=0
10800 JFACT=0
10900 JFACT=0
11000 JFACT=0
11100 JFACT=0
11200 JFACT=0
11300 JFACT=0
11400 JFACT=0
11500 JFACT=0
11600 JFACT=0
11700 JFACT=0
11800 JFACT=0
11900 JFACT=0
12000 JFACT=0
12100 JFACT=0
12200 JFACT=0
12300 JFACT=0
12400 JFACT=0
12500 JFACT=0
12600 JFACT=0
12700 JFACT=0

```

CALCULATE RESIDUAL - QUADRATIC METHOD
 WORK UP RESIDUAL, MAXIMIZED BY WORK OF SOLUTION
 TO DETERMINE CONVERGENCE

```

RESIDU=RESIDU
RESIDU=0
SUBM=0

```

```

12660 DO 20 J=2,NY-1
12960 DO 20 I=2,NX-1
13000 CCRKOS=CR(I,J)*SUBS(I+1,J)+CR(I,J)+CR(I,J)*SUBS(I-1,J)+
13100 1CR(I,J)+SUBS(I,J+1)+CR(I,J)+SUBS(I,J-1)
13200 CCURK=CR(I,J)*SUBS(I+1,J+1)+CR(I,J)+SUBS(I-1,J+1)
13300 1+CR(I,J)*SUBS(I+1,J-1)+CR(I,J)+SUBS(I-1,J-1)
13400 SUBRS=CCURK+CCURK+CR(I,J)*SUBS(I,J)+F(I,J)
13500 RESID=RESID+SUBRS**2
13600 SOLRES=SOLRES+SUBS(I,J)**2
13700 CONTINUE
13800 RESID=SQRT(RESID)
13900 RESOR=RESOR/SQRT(SUBRS)
14000 ITERATION FINISHED IF RESIDUAL IS SMALL ENOUGH
14100 IF (RESOR.LT.0.01)GO TO 2
14200 IF (ITER.LT.2)GO TO 100
C
C FOR ITERATIONS AFTER THE FIRST, RECALCULATE OMEGA BASED ON
C THE CURRENT RESIDUAL AND THE RESIDUAL AT THE PREVIOUS ITERATION
C
ALAMDA=RESOR/RESID
WRITE(6,301)ITER,OMEGA,RESOR,RESOR
WRITE(31,301)ITER,OMEGA,RESOR,RESOR
301 FORMAT(IX, ' ITERATION ', I4, IX, ' OMEGA=', F7.5, IX, ' RESIDUAL=',
C IF (ALAMDA.GT.1.1)GO TO 100
C OMEGA=2.7(1.+SQRT(1.-ALAMDA))
100 CONTINUE
15000 SPIKE(0,20)ITER
201 FORMAT(IX, ' SOLUTION NOT CONVERGED AFTER ', I4, ' ITERATIONS')
C
C DETERMINE SOLUTION AT BOUNDARY FOR DEUMAN; BOUNDARY CONDITIONS
C
DETERMINE SOLUTION AT BOUNDARY FOR DEUMAN; BC AT LEFT
2 IF (ITER.LT.0)GO TO 31
DO 41 J=2,NY-1
SUBS(1,J)=SUBS(2,J)-DELX*DBS(J)
41 CONTINUE
C
DETERMINE SOLUTION AT BOUNDARY FOR DEUMAN; BC AT RIGHT
31 IF (ITER.LT.0)GO TO 32
DO 42 J=2,NY-1
SUBS(NX,J)=SUBS(NX-1,J)-DELX*DBS(J)
42 CONTINUE
C
DETERMINE SOLUTION AT BOUNDARY FOR DEUMAN; BC AT TOP

```

```

17100
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17400
17500
17600
17700
17800
17900
18000
18100
18200
18300
18400
18500
18600
18700
18800
18900
19000
19100
19200
19300
19400
19500
19600
19700
19800
19900
20000

32 IF (I,GA,1,0,0) GO TO 33
   DO +3 I=2,IX-J
   SUB3(1,XY)=SUB3(1,XY-1)-DELTA*ABC(1)
33 CONTINUE
   DETERMINE SOLUTION AT BOUNDARY FOR NEUMANN BC AT BOTTOM
   IF (IBOTTOM.EQ.0) GO TO 34
   DO +4 I=2,IX-1
   SUB3(1,1)=SUB3(1,2)-DELTA*CBC(1)
34 CONTINUE
   DETERMINE SOLUTION AT THE LEFT CORNER FOR NEUMANN CONDITION
   AT FOR 2ND LEFT SIDES
   IF (IDLEFT*ITOP.EQ.0) GO TO 35
   SUB3(1,0Y)=SUB3(1,XY-1)+SUB3(2,XY)-SUB3(2,XY-1)-
   DELTA*DELTACROSS
35 DETERMINE SOLUTION AT BOTTOM LEFT CORNER FOR NEUMANN CONDITION
   AT BOTTOM AND LEFT SIDES
   IF (IDLEFT*IBOTTOM.EQ.0) GO TO 36
   SUB3(1,1)=SUB3(2,1)+SUB3(1,2)-SUB3(2,2)+DELTA*DELTACROSS
36 DETERMINE SOLUTION AT TOP RIGHT CORNER FOR NEUMANN CONDITION
   AT TOP AND RIGHT SIDES
   IF (IDRIGHT*ITOP.EQ.0) GO TO 37
   SUB3(GX,0Y)=SUB3(GX,XY-1)+SUB3(GX-1,XY)-SUB3(GX-1,XY-1)+
   DELTA*DELTACROSS
37 DETERMINE SOLUTION AT BOTTOM RIGHT CORNER FOR NEUMANN CONDITION
   AT BOTTOM AND RIGHT SIDES
   IF (IDRIGHT*IBOTTOM.EQ.0) GO TO 38
   SUB3(GX,1)=SUB3(GX,2)+SUB3(GX-1,1)-SUB3(GX-1,2)-
   DELTA*DELTACROSS
38 RETURN
   END

```


C**ALSO REQUIRED AS INPUT FOR THIS SUBROUTINE ARE THE SET
 OF EQUATION SUBPROGRAMS WHICH CALCULATE THE SQUARE OF THE
 VELOCITIES, AND THE CROSS MULTIPLICATION OF THE VELOCITIES.
 THESE ARE GIVEN AFTER THE SUBROUTINE PSUBKC

5100
 5200
 5300
 5400
 5500
 5600
 5700
 5800
 5900
 6000
 6100
 6200
 6300
 6400
 6500
 6600
 6700
 6800
 6900
 7000
 7100
 7200
 7300
 7400
 7500
 7600
 7700
 7800
 7900
 8000
 8100
 8200
 8300
 8400
 8500
 8600
 8700
 8800
 8900
 9000
 9100
 9200
 9300
 9400
 9500
 9600
 9700
 9800
 9900
 10000

```

C**0001P0T
C
C "VRJ" AND "VZJ" ARE THE UPDATED ARRAYS OF RADIAL AND
C AXIAL VELOCITY
C*****
C CALCULATE VELOCITIES AT INTERIOR POINTS
C CALCULATE THE AXIAL COMPONENT OF VELOCITY
  DO 10 I=2,NX-1
  DO 10 J=2,NY-1
    UPWARD DIFFERENCING
    VRAVJ=-.25*(VP(I,J-1)+VR(I,J)+VR(I+1,J)+VR(I+1,J-1))
    VRAVJ=(VZ(I,J)-VZ(I-1,J))/DELX
    IF (VRAVJ.GT.0.)DFKVR=(VZ(I+1,J)-VZ(I,J))/DELY
    DEKVR=(VZ(I,J)-VZ(I-1,J))/DELY
    IF (VZ(I,J).GT.0.)DEPVZ=(VZ(I,J+1)-VZ(I,J))/DELY
    VZAVJ=VZ(I,J)
    IF (VRAVJ.VZAVJ).VZAVJ=.5*(VZ(I,J+1)+VZ(I,J-1))
    VZLI+(C-VRAVJ)*DEKVR-VZ(I,J)*DEKVR
    2-(V(I,J)-V(I,J-1))/DELY+(1./RF)*(VZ(I+1,J)-
    3VZ(I-1,J))/X(I)*2.*DELX+(VZ(I+1,J)-2.*VZ(I,J)+
    4VZ(I-1,J))/DELY**2+(VZ(I,J+1)-2.*VZ(I,J)+VZ(I,J-1))
    5/DELY**2)+F2(I,J-1)
  10 CONTINUE
C
C 10 CALCULATE THE RADIAL COMPONENT OF VELOCITY
  DO 20 I=2,NX
  DO 20 J=2,NY-1
    UPWARD DIFFERENCING
    VZAVJ=-.25*(VZ(I,J)+VZ(I,J+1)+VZ(I-1,J)+VZ(I-1,J+1))
    DEKVR=(VR(I,J)-VR(I-1,J))/DELY
    IF (VR(I,J).LT.0.)DEKVR=(VR(I+1,J)-VR(I,J))/DELY
    DEKVR=(VR(I,J)-VR(I-1,J))/DELY
    IF (VZAVJ.LT.0.)DEPVZ=(VR(I,J+1)-VR(I,J))/DELY
    RHALF=(X(I+1)+X(I))/2.
    VRAVJ=VR(I,J)
    IF (VRAVJ.VZAVJ).VZAVJ=.5*(VR(I+1,J)+VR(I-1,J))
    VRAVJ+(C-VRAVJ)*DEKVR-VZ(I,J)*DEKVR
    2-(V(I,J)-V(I-1,J))/DELY+(1./RF)*(VZ(I+1,J)-
    3(VZ(I+1,J)/RHALF+VZ(I,J))-VR(I-1,J)/RHALF+VZ(I,J)+
    4(VZ(I+1,J)-2.*VR(I,J)+VR(I-1,J))/DELY**2+
    5(VZ(I,J+1)-2.*VR(I,J)+VR(I-1,J))/DELY**2)+F2(I-1,J)
  20 CONTINUE
  
```

```
10160
10200
10300
10400
10500
10600
10700
10800
10900
11000
11100
11200
11300
11400
11500
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11700
11800
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13400
13500
13600
13700
13800
13900
14000
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14500
14600

C      CALCULATE VELOCITIES AT BOUNDARIES
C**CALCULATE SOLUTION ON LEFT BOUNDARY
C      ZERO SHEAR STRESS ON AXIS OF SYMMETRY
C      DVZ/DZ=0.
      DO 30 J=2,NY
      VZ0(I,J)=(4.*VRM(2,J)-VZ0(3,J))/3.
C      CONTINUE
C      RADIAL DERIVATIVE OF VELOCITY IS ZERO
      DO 40 J=2,NY-1
      VR0(I,J)=-VRM(2,J)
C      CONTINUE
C**CALCULATE SOLUTION AT BOTTOM BOUNDARY
C      AXIAL VELOCITY =0. SOLID BOUNDARY
      DO 50 I=1,NX
      VZ0(I,NY+1)=-VZ0(I,NY)
C      CONTINUE
C      RADIAL VELOCITY =0. NO SLIP CONDITION
      DO 60 I=1,NX+1
      VR0(I,NY)=0.
C      CONTINUE
C**CALCULATE SOLUTION AT TOP BOUNDARY
C      AXIAL VELOCITY =0. SOLID BOUNDARY
      DO 90 I=1,NX
      VZ0(I,1)=-VZ0(I,2)
C      CONTINUE
C      RADIAL VELOCITY =0. NO SLIP CONDITION
      DO 100 I=1,NX+1
      VR0(I,1)=0.
C      CONTINUE
C**CALCULATE SOLUTION AT RIGHT BOUNDARY
C      MASS CONSERVATION
      DO 70 J=2,NY
      VR0(NX+1,J)=VR0(NX,J)*(X(NX-1)+VELX/2.)/(X(NX)+DELX/2.)
C      NO VORTICITY
      DO 80 J=2,NY-1
      VZ0(NX,J)=VZ0(NX-1,J)+(VR0(NX,J)-VR0(NX,J-1))*DELX/DELZ
C      NO AXIAL VELOCITY
      VZ0(NX,J)=0.
C      CONTINUE
      FLOW
      END
```

PARTCL SUBROUTINE

```

100 SUBROUTINE PARTCL(F,Z,lambda,UR,UZ,DT,THETA0,FRACO)
200 DIMENSION UR(50),UZ(50),I(50),R(50),Z(50)
300 DIMENSION FORCER(50),FORCEZ(50)
400 DIMENSION FR(50,50),FZ(50,50)
500 COMMON /DELZ,DELT,DELT
600 COMMON /SPRAY DIM/ ZL,n,nL,hp,G,PU
700 COMMON /REYNOLDS/ RE
800 COMMON /SOURCE/ FR,FC
900 REAL H,dt
1000 PI=3.14159
1100 *****
1200 THIS SUBROUTINE CALCULATES THE PARAMETERS OF THE LIQUID
1300 PHASE, NAMELY THE PARTICLE TRAJECTORIES, THE PARTICLE
1400 VELOCITY ALONG THE TRAJECTORY, AND THE EXCHANGE OF MOMENTUM
1500 TO THE GAS PHASE (THE SOURCE TERMS).
1600 *****
1700 THESE CALCULATIONS ARE MADE BY INTEGRATING THE EQUATIONS OF
1800 MOTION OF THE PARTICLE, AND DETERMINING THE NET DRAG FORCE
1900 ACTING ON THE PARTICLES IN A GIVEN VOLUME.
2000 A FOURTH ORDER INTEGRATION IS USED FOR THE INTEGRATION OF
2100 THE EQUATIONS OF MOTION.
2200 *****
2300 NOTE: THIS PROGRAM IS SET UP ONLY FOR THE CASE WHERE THE NOZZLE
2400 POSITION CORRESPONDS WITH A PRESSURE NODE LOCATION
2500 *****
2600 INPUT TO THIS PROGRAM IS AS FOLLOWS:
2700 *****
2800 "DP" - PARTICLE DIAMETER / NOZZLE DIAMETER
2900 "THETA0" - THE INITIAL HALF ANGLE OF THE SPRAY AT THE
3000 EJECTION POINT
3100 "FRACO" - IS THE FRACTION OF MASS FLOW OF EACH PARTICULAR
3200 TRAJECTORY OR DROPLET SIZE
3300 *****
3400 *****
3500 INPUT THRU COMMON BLOCK

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3900
3700 THE "SPRAY DIA" COMMON BLOCK CONTAINS THE PARAMETERS OF THE
3800 SPRAY
3900
4000 "ZL"-THE AXIAL LENGTH OF THE DOMAIN DIVIDED BY THE HEIGHT OF
4100 SPRAY
4200 "WH"-EQUALS ONE FOR A DOWNWARD FACING SPRAY
4300 "RL"-THE RADIAL LENGTH OF THE DOMAIN DIVIDED BY THE HEIGHT OF
4400 THE SPRAY
4500 "HP"-HEIGHT OF THE SPRAY FROM GROUND / NOZZLE DIAMETER
4600 "GP"-INVERSE PROBE NUMBER SQUARED
4700 "GG"-ACCELERATION OF GRAVITY*HEIGHT OF SPRAY/EJECTION VELOCITY**2
4800 "RG"-DENSITY OF GAS/DENSITY OF LIQUID
4900
5000 THE "DEL" COMMON BLOCK CONTAINS THE RADIAL AND AXIAL INCREMENTAL
5100 DISTANCE AND (THROUGH NOT USED IN THIS SUBROUTINE) THE INCREMENTAL
5200 TIME
5300
5400 THE "REYNOLDS" COMMON BLOCK CONTAINS THE REYNOLDS NUMBER OF
5500 MULTIDIMENSIONALIZATION - EJECTION VELOCITY*SPRAY HEIGHT/GAS
5600 VISCOSITY
5700
5800 **OUTPUT
5900
6000 "Z" AND "R" ARE THE ARRAYS CONTAINING THE AXIAL AND RADIAL
6100 COMPONENTS OF THE PARTICLE TRAJECTORIES
6200
6300 "UZ" AND "UR" ARE THE ARRAYS CONTAINING THE AXIAL AND RADIAL
6400 COMPONENTS OF THE SPRAY VELOCITY ALONG THE SPRAY TRAJECTORY
6500 **OUTPUT INTO COMMON BLOCK
6600
6700 THE "SOURCE" COMMON BLOCK CONTAINS THE TWO DIMENSIONAL
6800 ARRAY OF THE RADIAL AND AXIAL SOURCE TERMS *PR* AND *FZ*
6900
7000 *****
7100 DELZ=DEL Y/2.
7200 X0Y=FIX(ZL/DELZ)
7300 XZ=FIX(RL/DELX)+1
7400 YZ=FIX(ZL/DELY)+1
7500
7600 DETERMINATION OF PARTICLE TRAJECTORY AND VELOCITY
7700 ALONG THE TRAJECTORY
7800 *****
7900 INITIAL CONDITIONS FOR PARTICLE EQUATIONS
8000 GZ(1)=SIC(THETA0)
8100 GZ(1)=COS(THETA0)
8200 R(1)=0.

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8300 F(1)=0.
8400 Z(1)=0.
8500 CONST=.75*RO*HP**2/(DP**2*RE.)
8600 C04ST2=1.5/(CP**3*HP**2)
8700 DO 10 I=1,NUNIT
8800 C START FIRST STEP OF RUNGE-KUTTA INTEGRATION
8900 EK0=DELZ*(UR(1)/UZ(1))
9000 IF0=DELZ/UZ(1)
9100 C CALCULATE AIRFIELD VELOCITY
9200 CALL AIRVEL(UR(1),Z(1),VR,VZ)
9300 REYLD=RE*DP*SQRT((UR(1)-VR)**2+(UZ(1)-VZ)**2)/HP
9400 C2=-CD(REFYLD)*REYLD*CONST
9500 UR0=UR(1)
9600 UZ0=DELZ*(C2*(UR(1)-VR))/UZ(1)
9700 C START SECOND STEP OF RUNGE-KUTTA INTEGRATION
9800 UR1=UR(1)+URK0/2.
9900 UZ1=UZ(1)+UZK0/2.
10000 KR1=DELZ*(UR1/UZ1)
10100 TR1=DELZ/UZ1
10200 R1=UR(1)+KR1/2.
10300 Z1=Z(1)+DELZ/2.
10400 C CALCULATE AIRFIELD VELOCITY
10500 CALL AIRVEL(UR1,Z1,VR,VZ)
10600 REYLD=RE*DP*SQRT((UR1-VR)**2+(UZ1-VZ)**2)/HP
10700 C2=-CD(REFYLD)*REYLD*CONST
10800 UR1=DELZ*(C2*(UR1-VR))/UZ1
10900 UZ1=DELZ*(C2*(UZ1-VZ)+G)/UZ1
11000 C START THIRD STEP OF RUNGE-KUTTA INTEGRATION
11100 UR1=UR(1)+URK1/2.
11200 UZ1=UZ(1)+UZK1/2.
11300 KR2=DELZ*(UR1/UZ1)
11400 TR2=DELZ/UZ1
11500 R1=UR(1)+KR2/2.
11600 C CALCULATE AIR FIELD VELOCITY
11700 CALL AIRVEL(UR1,Z1,VR,VZ)
11800 REYLD=RE*DP*SQRT((UR1-VR)**2+(UZ1-VZ)**2)/HP
11900 C2=-CD(REFYLD)*REYLD*CONST
12000 UR2=DELZ*(C2*(UR1-VR))/UZ1
12100 UZ2=DELZ*(C2*(UZ1-VZ)+G)/UZ1
12200 C START FOURTH STEP OF RUNGE-KUTTA INTEGRATION
12300 UR1=UR(1)+URK2
12400 UZ1=UZ(1)+UZK2
12500 KR3=DELZ*(UR1/UZ1)
12600 TR3=DELZ/UZ1
12700 R1=UR(1)+KR3
12800 Z1=Z(1)+DELZ

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C
CALCULATE AIR FIFTH VELOCITY
CALL AIRVEL(R1,Z1,VK,VZ)
R1=RD+RE*DP+SQR(CR1-VK)**2+(UZ1-VZ)**2)/HP
C2=-CD(CR1*RD)*RE*PI*CONST
URK3=DFLZ*(C2*(CR1-VK))/UZ1
UZK3=DFLZ*(C2*(UZ1-VZ)+G)/UZ1
C
CALCULATE VALUE AT END OF INTEGRATION STEP
R(1+1)=R(1)+(1./b.)*(RK0+2.*RK1+2.*RK2+RK3)
UR(1+1)=UR(1)+(1./b.)*(URK0+2.*URK1+2.*URK2+URK3)
UZ(1+1)=UZ(1)+(1./b.)*(UZK0+2.*UZK1+2.*UZK2+UZK3)
CONTINUE
10
CALCULATION OF TRAJECTORY AND VELOCITIES ALONG TRAJECTORY
COMPLETE
C
CALCULATE DIMENSIONAL FORCE COEFFICIENTS ALONG THE
TRAJECTORY
C
CALL AIRVEL(R1,Z(1),VK,VZ)
VLOT=SQR(CR1-VK)**2+(UZ(1)-VZ)**2)
REYLD=VF*VR*VLOT/NU
FORCE(1)=-CD(REYLD)*VLOT*(UR(1)-VR)*HP*PI*DP**2/8.
FORCZ(1)=-CD(REYLD)*VLOT*(UZ(1)-VZ)*HP*PI*DP**2/8.
CALL AIRVEL(R(NUMY+1),Z(NUMY+1),VR,VZ)
VLOT=SQR(CR(NUMY+1)-VR)**2+(UZ(NUMY+1)-VZ)**2)
REYLD=RE*VR*VLOT/NU
FORCE(NUMY+1)=-CD(REYLD)*VLOT*(UR(NUMY+1)-VR)*HP*PI*DP**2/8
FORCZ(NUMY+1)=-CD(REYLD)*VLOT*(UZ(NUMY+1)-VZ)*HP*PI*DP**2/8
C1=PI*DL**3/(b.*RU)
DO 30 J=2,NUMY
FORCZ(J)=C1*(UZ(J)-UR(J+1))/(Z.*DFLZ)-G)
FORCE(J)=C1*(UR(J+1)-UR(J+1))/(Z.*DFLZ)
CONTINUE
30
C
CALCULATE SOURCE TERMS IN EACH CONTROL VOLUME
IFR1=1
JFR1=FIX((ZL-H)/DELZ+.5)+1
IFZ1=1
JFZ1=FIX((ZL-H)/DELZ)+1
DO 50 J=1,NUMY

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17100 JFRZ=IFIX((ZL-RT+Z(J+1))/DELT+*.5)+1
17200 JFR2=IFIX(R(J+1)/DELA)+1
17300 FFAVE=(FUNCR(J)+FUNCR(J+1))/2.
17400 JFZ2=IFIX((ZL-RT+Z(J+1))/DELY)+1
17500 JFZ2=IFIX(R(J+1)/DELA+.5)+1
17600 FZAVE=(FUNCEZ(J)+FUNCEZ(J+1))/2.
17700 TDFL=SQRT((R(J+1)-R(J))*2+(Z(J+1)-Z(J))*2)
17800 ICPUSR=JFR2-JFR1+1
17900 ICRUSZ=JFZ2-JFZ1+1
18000 DO 60 I=1,ICPUSR
18100 DELTK=R(J+1)-R(J)
18200 DELTB=FLOAT(IFR1)*DELA-R(J)+DELA*FLOAT(I-1)
18300 DELTE=R(J+1)-FLOAT(IFR2-1)*DELA+DELA*FLOAT(ICRUSR-I)
18400 DELTB=DELA
18500 IF(DELTK.GT.DELTB)DELTB=DELTK
18600 IF(DFLTB.GT.DELTB)DELTB=DFLTB
18700 IF(DFLTR.GT.DELTB)DELTB=DFLTR
18800 SCALE=1.
18900 IF(ABS(R(J+1)-R(J)).GT.0.)SCALE=DELTB/(R(J+1)-R(J))
19000 DELTZ=SCALE*DELZ
19100 DT=DELTZ/((UZ(O)+UZ(J+1))/2.)
19200 FF(IFR1+1-1,JFR1)=FF(IFR1+1-1,JFR1)-FFAVE*DT*FRACO*CONSTZ/
19300 1((2.*DELA+FLOAT(IFR1+1-2)+DELA)*PI*DELA*DELZ)
19400 CONTINUE
19500 JFR1=JFR2
19600 DO 70 I=1,ICRUSZ
19700 JFLTR=R(J+1)-R(J)
19800 DELTB=(FLOAT(IFZ1)-.5)*DELA-R(J)+DELA*FLOAT(I-1)
19900 DELTE=R(J+1)-FLOAT(IFZ2-1)*DELA+DELA*FLOAT(ICRUSZ-I)
20000 DELTB=DELA
20100 IF(DELTK.GT.DELTB)DELTB=DELTK
20200 IF(DFLTB.GT.DELTB)DELTB=DFLTB
20300 IF(DFLTR.GT.DELTB)DELTB=DFLTR
20400 IF(DELTK.GT.DELTB)DELTB=DELTK
20500 SCALE=1.
20600 IF(ABS(R(J+1)-R(J)).GT.0.)SCALE=DELTB/(R(J+1)-R(J))
20700 DELTZ=SCALE*DELZ
20800 DT=DELTZ/((UZ(J)+UZ(J+1))/2.)
20900 F1=.5+FLOAT(IFZ1+1-3)
21000 F1=MAX(0.,F1)+DELA
21100 F2=(.5+FLOAT(IFZ1+1-2))*DELA
21200 FZ(IFZ1+1-1,JFZ1)=FZ(IFZ1+1-1,JFZ1)-FZAVE*DT*FRACO*CONSTZ/
21300 1((PI*(FZ2+2-KI**2)+DELTZ)
21400 CONTINUE

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21500 JFZ1=JFZ2
21600 IFZ1=IFZ2
21700 CUFFLAG=
21800 RETURN
21900 END
22000 C THIS FUNCTION SUBPROGRAM GIVES THE DRAG COEFFICIENT
22100 OF A SPHERE AS A FUNCTION OF REYNOLDS NUMBER
22200 OF FUNCTION CODE
22300 CD=(24./Re)*D./(1.+D*P1(Re))+.4
22400 RETURN
22500 END
22600 C THIS SUBROUTINE CALCULATES THE AXIAL AND RADIAL COMPONENTS
22700 OF AIR VELOCITY AT POINTS INTERIOR TO THE SAC CPLED BY
22800 INTERPOLATING THE DISCRETIZED MAC CELL VELOCITIES
22900 SUBROUTINE AIRVEL(R,Z,VKDFRZ,VZDFRZ)
23000 DIMENSION VR(50,50),VZ(50,50)
23100 COMMON /DEL/ DELX,DELY,DELT
23200 COMMON /AIR V/ VZ,VR
23300 COMMON /SPRAY DIF/ ZL,H,KL,HE,G,R0
23400 READ H,HE
23500 IVR=FIX((R/DELX+.5)+1)
23600 JVR=FIX((ZL-H+Z)/DELY)+1
23700 IVZ=FIX((R/DELX)+1)
23800 JVZ=FIX((ZL-H+Z)/DELY+.5)+1
23900 DX=(R/DELX+.5-FLDZT(IVR)+1.)*DELX
24000 DY=(ZL-H+Z)/DELY-FLDZT(JVR)+1.)*DELY
24100 VFORZ=VR(IVR,JVR)+VR(IVR+1,JVR)-VR(IVR,JVR)*DX/DELY
24200 I+(VR(IVR,JVR+1))-VR(IVR,JVR)*DY/DELY
24300 VX=(R/DELX-FLDZT(IVZ)+1.)*DELX
24400 VY=(ZL-H+Z)/DELY+.5-FLDZT(JVZ)+1.)*DELY
24500 VZDFRZ=VZ(IVZ,JVZ)+VZ(IVZ+1,JVZ)-VZ(IVZ,JVZ)*DX/DELY
24600 I+(VZ(IVZ,JVZ+1))-VZ(IVZ,JVZ)*DY/DELY
24700 RETURN
24800 END

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INPUT FILE

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100 INPUT FOR SPRAY PROGRAM
200 FUELS, DIENSIONS
300 ENTER RADIAL AND AXIAL DUFATH DIMENSIONS; RU AND ZL;
400 NUMBER OF STABILIZED BY THE SPRAY HEIGHT
500 1XXXXXXXXX22XXXXXXXXXXXXXXXXXX
600 1.
700 ENTER SPRAY HEIGHT/SPRAY LENGTH, H, AND SPRAY LENGTH/NOZZLE DIAMETER, HP
800 1XXXXXXXXX22XXXXXXXXXXXXXXXXXX
900 1.
1000 ENTER NUMBER OF PARTICLE SIZES OR TRAJECTORIES
1100 1XX
1200 4
1300 ENTER DROPLET DIAMETER/NOZZLE DIAMETER, DP, INITIAL ANGLE OF SPRAY,
1400 THEIA), AND VOLUME FLOW FRACTION OF EACH PARTICULAR TRAJECTORY, FRACO
1500 1XXXXXXXXX22XXXXXXXXXX22XXXXXXXXXXXXXXXXXX
1600 .09091 .5236 .1667
1700 .04091 .4195 .3333
1800 .04091 .3217 .3333
1900 .04091 .0
2000 ENTER INVERSE FROUDE NUMBER**2 ,G, AND REYNOLDS NOE OF LIQUID, RE
2100 1XXXXXXXXX22XXXXXXXXXXXXXXXXXX
2200 .03636 309/300.
2300 ENTER DENSITY OF LIQUID, RD
2400 1XXXXXXXXXX
2500 .0012024
2600 ENTER TIME STEP, DELT, SUBJECT TO STABILITY RESTRICTIONS
2700 1XXXXXXXXXX
2800 0.02
2900 ENTER NUMBER OF ITERATIONS OF MAC METHOD
3000 1XXXXXXXXXX
3100 1000
3200 ENTER SWITCH FOR INITIAL SOLUTION (0=NONE, 1=IN FILE 19)
3300 1X
3400 0
3500 ENTER NUMBER OF PRESSURE BUOFS IN RADIAL AND AXIAL DIRECTIONS
3600 6X
3700 1XX22XX
3800 11 11

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0.0000	0.7000	0.00152	0.00000	0.65000	0.16509	-0.05000	0.70000	-0.00435
0.1000	0.7000	0.00152	0.10000	0.65000	0.16126	0.05000	0.70000	0.00435
0.2000	0.7000	0.00047	0.20000	0.65000	0.06707	0.15000	0.70000	0.01356
0.3000	0.7000	0.00093	0.30000	0.65000	0.01460	0.25000	0.70000	0.02232
0.4000	0.7000	0.00041	0.40000	0.65000	0.00293	0.35000	0.70000	0.01153
0.5000	0.7000	-0.00001	0.50000	0.65000	0.00413	0.45000	0.70000	-0.01090
0.6000	0.7000	-0.00002	0.60000	0.65000	0.00391	0.55000	0.70000	-0.01159
0.7000	0.7000	-0.00002	0.70000	0.65000	0.00298	0.65000	0.70000	-0.01184
0.8000	0.7000	-0.00002	0.80000	0.65000	0.00184	0.75000	0.70000	-0.01179
0.9000	0.7000	-0.00001	0.90000	0.65000	0.00080	0.85000	0.70000	-0.01147
1.0000	0.7000	0.00000	1.00000	0.65000	0.00000	0.95000	0.70000	-0.01038
0.0000	0.8000	0.00436	0.00000	0.75000	0.15927	-0.05000	0.80000	-0.00959
0.1000	0.8000	0.00436	0.10000	0.75000	0.14302	0.05000	0.80000	0.00959
0.2000	0.8000	0.00217	0.20000	0.75000	0.09427	0.15000	0.80000	0.02909
0.3000	0.8000	0.00035	0.30000	0.75000	0.02988	0.25000	0.80000	0.03046
0.4000	0.8000	0.00003	0.40000	0.75000	0.00108	0.35000	0.80000	0.01097
0.5000	0.8000	-0.00003	0.50000	0.75000	0.00723	0.45000	0.80000	-0.00528
0.6000	0.8000	-0.00004	0.60000	0.75000	0.00639	0.55000	0.80000	-0.00838
0.7000	0.8000	-0.00004	0.70000	0.75000	0.00490	0.65000	0.80000	-0.00879
0.8000	0.8000	-0.00003	0.80000	0.75000	0.00327	0.75000	0.80000	-0.00892
0.9000	0.8000	-0.00003	0.90000	0.75000	0.00178	0.85000	0.80000	-0.00961
1.0000	0.8000	0.00000	1.00000	0.75000	0.00000	0.95000	0.80000	-0.01137
0.0000	0.9000	0.00865	0.00000	0.85000	0.11267	-0.05000	0.90000	-0.01024
0.1000	0.9000	0.00865	0.10000	0.85000	0.10401	0.05000	0.90000	0.01687
0.2000	0.9000	0.00551	0.20000	0.85000	0.07801	0.15000	0.90000	0.05109
0.3000	0.9000	0.00222	0.30000	0.85000	0.04244	0.25000	0.90000	0.07159
0.4000	0.9000	0.00059	0.40000	0.85000	0.01955	0.35000	0.90000	0.07511
0.5000	0.9000	0.00016	0.50000	0.85000	0.01168	0.45000	0.90000	0.07005
0.6000	0.9000	0.00004	0.60000	0.85000	0.00820	0.55000	0.90000	0.06440
0.7000	0.9000	0.00000	0.70000	0.85000	0.00629	0.65000	0.90000	0.05954
0.8000	0.9000	-0.00001	0.80000	0.85000	0.00511	0.75000	0.90000	0.05552
0.9000	0.9000	-0.00004	0.90000	0.85000	0.00468	0.85000	0.90000	0.05220
1.0000	0.9000	0.00000	1.00000	0.85000	0.00000	0.95000	0.90000	0.04986
0.0000	1.0000	0.00865	0.00000	0.95000	0.03787	-0.05000	1.00000	0.04493
0.1000	1.0000	0.00865	0.10000	0.95000	0.03500	0.05000	1.00000	0.00000
0.2000	1.0000	0.00551	0.20000	0.95000	0.02637	0.15000	1.00000	0.00000
0.3000	1.0000	0.00222	0.30000	0.95000	0.01440	0.25000	1.00000	0.00000
0.4000	1.0000	0.00059	0.40000	0.95000	0.00657	0.35000	1.00000	0.00000
0.5000	1.0000	0.00016	0.50000	0.95000	0.00391	0.45000	1.00000	0.00000
0.6000	1.0000	0.00004	0.60000	0.95000	0.00274	0.55000	1.00000	0.00000
0.7000	1.0000	0.00000	0.70000	0.95000	0.00210	0.65000	1.00000	0.00000
0.8000	1.0000	-0.00001	0.80000	0.95000	0.00170	0.75000	1.00000	0.00000
0.9000	1.0000	-0.00004	0.90000	0.95000	0.00150	0.85000	1.00000	0.00000
1.0000	1.0000	0.00000	1.00000	0.95000	0.00000	0.95000	1.00000	0.00000
0.0000	0.0000	0.00000	0.00000	1.05000	-0.03787	-0.05000	1.00000	0.00000
0.1000	0.0000	0.00000	0.10000	1.05000	-0.04500	0.05000	1.00000	0.00000
0.2000	0.0000	0.00000	0.20000	1.05000	-0.02637	0.15000	1.00000	0.00000
0.3000	0.0000	0.00000	0.30000	1.05000	-0.01440	0.25000	1.00000	0.00000
0.4000	0.0000	0.00000	0.40000	1.05000	-0.00657	0.35000	1.00000	0.00000
0.5000	0.0000	0.00000	0.50000	1.05000	-0.00391	0.45000	1.00000	0.00000
0.6000	0.0000	0.00000	0.60000	1.05000	-0.00274	0.55000	1.00000	0.00000
0.7000	0.0000	0.00000	0.70000	1.05000	-0.00210	0.65000	1.00000	0.00000
0.8000	0.0000	0.00000	0.80000	1.05000	-0.00170	0.75000	1.00000	0.00000
0.9000	0.0000	0.00000	0.90000	1.05000	-0.00150	0.85000	1.00000	0.00000
1.0000	0.0000	0.00000	1.00000	1.05000	0.00000	0.95000	1.00000	0.00000

PARTICLE TRAJECTORIES

PARTICLE TRAJECTORY FOR DE= 0.99091				PARTICLE TRAJECTORY FOR DE= 0.33330			
RADIUS	AXIAL DIST	TIME	THETA=	RADIUS	AXIAL DIST	TIME	THETA=
0.02000	0.00000	0.00000	0.50000	0.02000	0.00000	0.00000	0.50000
0.02882	0.05000	0.06294	0.41970	0.02882	0.05000	0.06294	0.41970
0.05735	0.10000	0.13758	0.29072	0.05735	0.10000	0.13758	0.29072
0.08535	0.15000	0.22471	0.23817	0.08535	0.15000	0.22471	0.23817
0.11177	0.20000	0.32512	0.19306	0.11177	0.20000	0.32512	0.19306
0.13643	0.25000	0.43981	0.15540	0.13643	0.25000	0.43981	0.15540
0.15916	0.30000	0.57101	0.12475	0.15916	0.30000	0.57101	0.12475
0.18021	0.35000	0.72201	0.09857	0.18021	0.35000	0.72201	0.09857
0.19961	0.40000	0.89701	0.07656	0.19961	0.40000	0.89701	0.07656
0.21736	0.45000	1.10095	0.05845	0.21736	0.45000	1.10095	0.05845
0.23307	0.50000	1.33558	0.04309	0.23307	0.50000	1.33558	0.04309
0.24661	0.55000	1.60183	0.03266	0.24661	0.55000	1.60183	0.03266
0.25798	0.60000	1.90082	0.02466	0.25798	0.60000	1.90082	0.02466
0.26730	0.65000	2.23161	0.01884	0.26730	0.65000	2.23161	0.01884
0.27500	0.70000	2.59267	0.01641	0.27500	0.70000	2.59267	0.01641
0.28158	0.75000	2.98208	0.01584	0.28158	0.75000	2.98208	0.01584
0.28613	0.80000	3.39359	0.01691	0.28613	0.80000	3.39359	0.01691
0.29611	0.85000	3.83341	0.02093	0.29611	0.85000	3.83341	0.02093
0.30690	0.90000	4.29096	0.02645	0.30690	0.90000	4.29096	0.02645
0.32292	0.95000	4.77512	0.03332	0.32292	0.95000	4.77512	0.03332
0.34092	1.00000	5.30573	0.04057	0.34092	1.00000	5.30573	0.04057
PARTICLE TRAJECTORY FOR DE= 0.99091				PARTICLE TRAJECTORY FOR DE= 0.33330			
RADIUS	AXIAL DIST	TIME	THETA=	RADIUS	AXIAL DIST	TIME	THETA=
0.00000	0.00000	0.00000	0.42639	0.00000	0.00000	0.00000	0.42639
0.02354	0.05000	0.06094	0.35072	0.02354	0.05000	0.06094	0.35072
0.04698	0.10000	0.13074	0.30519	0.04698	0.10000	0.13074	0.30519
0.06984	0.15000	0.21273	0.25495	0.06984	0.15000	0.21273	0.25495
0.09164	0.20000	0.30658	0.21191	0.09164	0.20000	0.30658	0.21191
0.11210	0.25000	0.41255	0.17579	0.11210	0.25000	0.41255	0.17579
0.13105	0.30000	0.53115	0.14472	0.13105	0.30000	0.53115	0.14472
0.14856	0.35000	0.66475	0.11933	0.14856	0.35000	0.66475	0.11933
0.16475	0.40000	0.81429	0.09804	0.16475	0.40000	0.81429	0.09804
0.17965	0.45000	0.98279	0.07981	0.17965	0.45000	0.98279	0.07981
0.19323	0.50000	1.17191	0.06439	0.19323	0.50000	1.17191	0.06439
0.20544	0.55000	1.38467	0.05141	0.20544	0.55000	1.38467	0.05141
0.21636	0.60000	1.62119	0.04124	0.21636	0.60000	1.62119	0.04124
0.22569	0.65000	1.87611	0.03334	0.22569	0.65000	1.87611	0.03334
0.23425	0.70000	2.15542	0.02746	0.23425	0.70000	2.15542	0.02746
0.24177	0.75000	2.45991	0.02387	0.24177	0.75000	2.45991	0.02387
0.24892	0.80000	2.78631	0.02260	0.24892	0.80000	2.78631	0.02260
0.25604	0.85000	3.09623	0.02614	0.25604	0.85000	3.09623	0.02614
0.26736	0.90000	3.45099	0.03340	0.26736	0.90000	3.45099	0.03340
0.28190	0.95000	3.84617	0.03829	0.28190	0.95000	3.84617	0.03829
0.29837	1.00000	4.29944	0.04326	0.29837	1.00000	4.29944	0.04326

PARTICLE TRAJECTORY FOR D1 = 0.09091 VOLUME FLOW FRACTION= 0.33330

RADIUS	AXIAL VELOCITY	TIME	AXIAL VELOCITY	TIME	AXIAL VELOCITY
0.00000	0.00000	0.00000	0.31618	0.00000	0.32170
0.01695	0.05000	0.05797	0.26975	0.31618	0.94670
0.03325	0.10000	0.12368	0.23048	0.26975	0.01063
0.04950	0.15000	0.20045	0.19448	0.23048	0.04793
0.06575	0.20000	0.28722	0.16400	0.19448	0.61017
0.07917	0.25000	0.38523	0.13657	0.16400	0.54076
0.09349	0.30000	0.49378	0.11059	0.13657	0.38648
0.10822	0.35000	0.61114	0.09435	0.11059	0.14250
0.11775	0.40000	0.73759	0.08417	0.09435	0.49597
0.12874	0.45000	0.87404	0.07153	0.08417	0.37356
0.13874	0.50000	1.03015	0.06068	0.07153	0.31433
0.14765	0.55000	1.19315	0.05168	0.06068	0.31640
0.15639	0.60000	1.36917	0.04414	0.05168	0.27444
0.16410	0.65000	1.55801	0.03858	0.04414	0.27162
0.17141	0.70000	1.75962	0.03414	0.03858	0.25591
0.17841	0.75000	1.97516	0.03111	0.03414	0.24017
0.18533	0.80000	2.20455	0.02978	0.03111	0.22478
0.19273	0.85000	2.44955	0.03072	0.02978	0.21169
0.20120	0.90000	2.71420	0.03391	0.03072	0.19632
0.21172	0.95000	3.00050	0.03765	0.03391	0.18079
0.22387	1.00000	3.31230	0.03367	0.03765	0.16098

PARTICLE TRAJECTORY FOR D2 = 0.09091 VOLUME FLOW FRACTION= 0.16670

RADIUS	AXIAL VELOCITY	TIME	AXIAL VELOCITY	TIME	AXIAL VELOCITY
0.00000	0.00000	0.00000	0.00000	0.00000	1.00000
0.00001	0.05000	0.05393	0.00055	0.00000	0.97139
0.00009	0.10000	0.11638	0.00170	0.00055	0.74710
0.00020	0.15000	0.16762	0.00142	0.00170	0.65855
0.00028	0.20000	0.20815	0.00083	0.00142	0.59051
0.00034	0.25000	0.25070	0.00053	0.00083	0.53835
0.00037	0.30000	0.45370	0.00065	0.00053	0.49752
0.00047	0.35000	0.55763	0.00000	0.00065	0.46433
0.00035	0.40000	0.66343	-0.00027	0.00000	0.43045
0.00032	0.45000	0.78065	-0.00029	-0.00027	0.41340
0.00027	0.50000	0.91071	-0.00037	-0.00029	0.39326
0.00023	0.55000	1.04085	-0.00039	-0.00037	0.37559
0.00010	0.60000	1.17692	-0.00048	-0.00039	0.35962
0.00010	0.65000	1.31696	-0.00051	-0.00048	0.34472
0.00001	0.70000	1.46715	-0.00060	-0.00051	0.33013
-0.00010	0.75000	1.62213	-0.00074	-0.00060	0.31517
-0.00025	0.80000	1.78143	-0.00103	-0.00074	0.29882
-0.00043	0.85000	1.95760	-0.00118	-0.00103	0.28012
-0.00070	0.90000	2.14341	-0.00164	-0.00118	0.25734
-0.00096	0.95000	2.33903	-0.00262	-0.00164	0.22881
-0.00091	1.00000	2.54584	-0.00134	-0.00262	0.19310

DATE
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