

AD-A131 121 TECHNICAL FEASIBILITY STUDY OF NAVY PIER CONCEPTS  
CONCEPT 1 THE EXPEDITIONARY PIER(U) LIN (T Y)  
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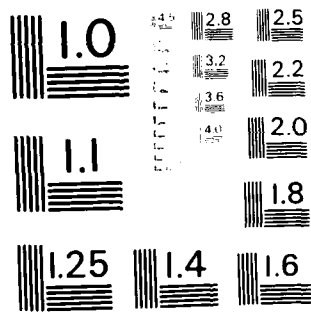
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UNCLASSIFIED NAVY PIER CONCEPTS-1/83 N00014-80-C-0869 F/G 13/2

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MURPHY RESOLUTION TEST TARGET  
NATIONAL BUREAU OF STANDARDS 1963

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NAVY PIER CONCEPTS  
REPORT No. 1/83

THE  
EXPEDITIONARY  
PIER

ADA 131121

SUBMITTED TO:

■ DEPARTMENT OF THE NAVY  
OFFICE OF NAVAL RESEARCH  
ARLINGTON, VIRGINIA

SUBMITTED BY:

■ T.Y. LIN INTERNATIONAL

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20 ABSTRACT (Continue on reverse side if necessary and identify by block number)  The expeditionary pier was developed to provide Navy and Marine Corps expeditionary forces operating overseas with the immediate use of full-service piers. These piers can accommodate from four to six ships and have provision for all the services expected from a permanent pier. Present design problems are limited to consideration of the hinge joint and anchoring system. The cost of expeditionary piers would be higher than traditional waterfront construction, but their ability to move with the Fleet is a great benefit.		

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## TECHNICAL FEASIBILITY STUDY OF PIER CONCEPTS

### 1. INTRODUCTION

The Navy pier concept study contract for the year 1982/83 called for the continuing development of three selected pier concepts to the point that their feasibility vis-a-vis the state-of-the-art technology may be assessed, and their deficiencies and problem areas identified. The three concepts selected for this study are:

- Concept 1 - **The Expeditionary Pier**
- Concept 2 - **The Floating Marina Pier**
- Concept 3 - **The Mobile Underwater Submarine Base**

Each concept will take up about one third of the contract year that began in September 1982. The results of the year's work will also be presented in a second paper to an engineering conference in or out of the USA.

This report addresses the technical feasibility of the first concept, i.e., the expeditionary pier, and is organized to cover the various study areas as follows:

- a. Investigation of the general validity of the design, and the parameters used in the development of the concept. Loading conditions, pier dimensions, structural sizes, flotation characteristics, etc. that were assumed for the original concept will be verified by more detailed analysis and designs during this investigation.
- b. Investigation of analytical and design feasibility: The purpose is to determine the adequacy of the state-of-the-art (SOA) technology in carrying out the analysis and the design of expeditionary piers.
- c. Investigation of constructional feasibility. Current construction methods that could be used in building the pier will be surveyed and assessed.
- d. Investigation and identification of problem areas and technological deficiencies. This will be assessed in relation to their influence and impact on the development of the pier.

### 2. THE EXPEDITIONARY PIER

To recap, the expeditionary pier was developed to provide the naval expeditionary force operating overseas the immediate use of a full-service pier. To be effective, it should be large enough to accommodate at least four, preferably six, destroyer class vessels. For these purposes, the pier must be self-sufficient, of sufficient size to berth four to six ships, relocatable and rapidly deployable. These features will be discussed below.

## **2.1 Self-Sufficiency**

Self-sufficiency means the pier is equipped to provide full berthing services to navy ships and, if necessary, to enable them to go "cold iron." In addition, it will also carry sufficient quantities of supplies to replenish the ships for long periods of time, perform minor repairs, and provide recreational and training facilities for the ship's crew.

## **2.2 Berthing Capacity**

The berthing capacity of the pier is determined by the size of the ships, the configuration and the size of the pier. Originally, the pier was designed for the future Navy ships, which were conceived to be small, fast and powerful vessels that are produced in sufficiently large numbers to achieve the effect of deterrence. Since the present destroyer class Navy ships are not much longer than the assumed future ship, the pier should provide the same number of berths for the present ships, if it is intended to be used in the near future. As shown, the berthing capacity of the pier for the various schemes is as follows:

Scheme A: Six big ships (destroyer class) and a number of smaller crafts along the central section of the spine pier.

Scheme B: Same as Scheme A

Scheme C: Four big ships plus smaller crafts

## **2.3 Relocatability**

The pier must not only be seaworthy, but also can be moved over long distances at reasonable speeds. Hence, the stream-lined, arrow-shaped configuration, and the retractability of the finger piers in Schemes A and B.

## **2.4 Rapid Deployability**

To be effective, the pier must be quickly operational upon arrival at site. In pursuit of this objective, the stiff-leg, single-point anchoring system has been developed for the use of the expeditionary pier.

## **3. DESIGN CONSIDERATIONS**

The following considerations have been assumed for the preliminary design of the pier, in addition to the four major criteria described above.

### **a. Materials**

Important criteria for the selection of structural materials must include lightness, strength and durability. Available materials that meet these criteria include lightweight concrete and steel. Either material is feasible and available. For the purpose of this investigation, lightweight concrete of 120 pcf density, and prestressing technique are envisaged.

**b. Environmental Loads**

The pier is designed for a wave height of 20 ft. in approximately sea state six, with wind velocity of 90 mph. The wavelength has been assumed to be equal to the length of the pier, for maximum sagging and hogging conditions.

**c. Fendering System**

Modern, fixed-position fendering systems could be used since both pier and ships move together with little variations. In the pier scheme, shown in this report, cell fenders of the buckling cylinder type have been used. These are mounted on the pier and located at such a level that they will always engage the berthing ships at the waterline region away from any hull protrusions.

**d. Self-propulsion**

Although the use of tugs is envisaged in moving the pier, a self-propulsion system on board the pier will be necessary to enable the pier to move under its own power in case of emergency. This system may well be needed to assist the anchoring system in keeping station during severe storm conditions.

**4. DESCRIPTION OF THE PIER**

The pier is essentially shaped like an arrow to reduce resistance against water during tow or operation. The difference between the three schemes presented herein, Schemes A, B and C, is in the inclusion of the finger piers in Scheme A and B, and the modification to the aft section of the pier in Schemes B and C. Scheme A, which is the original scheme, has two ships nesting against the back of the pier, i.e., perpendicular to the axis of the pier. In Scheme B, the aft section is modified to have the two ships berth alongside the pier in order to reduce current forces against the pier during operation. Scheme C is similar to Scheme B except it does not have the finger piers. The removal of the finger piers would bring the pier closer to the SOA, and make it more readily available for use in the near-term future.

As mentioned before the pier measures 940 ft in overall length. The hull for all three schemes is 51 ft deep, and when floating with normal live loads, takes a draft of approximately 34 ft., freeboard is therefore 17 ft. The hull is stiffened by structural walls that also serve as watertight bulkheads. Refer to Drawings 1 through 6.

**5. VALIDITY OF DESIGN**

The first task in this investigation is to confirm the validity of the preliminary design in general for the purpose of subsequent investigations. Specifically, the early design was checked to ensure that its structural system is adequate to withstand the assumed construction, towing and operation conditions, and that its

flotation and naval architectural characteristics are satisfactory. Considerable effort has also been made to resolve design problems posed by the two major innovations, i.e., the hinged joint for the finger piers and the stiff-legged swivelling suction anchor.

The results of this confirmatory investigation are contained in calculation sheets appended as Appendix A at the back of this report.

## **6. ANALYTICAL AND DESIGN FEASIBILITY**

The analysis and design of the expeditionary pier as represented in the drawings could be carried out with the state-of-the-art technology. This consists of design guidance provided by the various registration societies and technical and professional institutions. Similar prior design and constructions include the prestressed concrete pontoons that made up the Hood Canal Bridge in the State of Washington, the 460-ft. long prestressed concrete LPG barge that ARCO built for services in Indonesian waters, and the 700-ft. long prestressed concrete container pier now being built for Alaska's port of Valdez. This pier is 100 ft. wide and is made up of two 100 ft. by 350 ft. by 30 ft. deep pontoon units. The units are towed in from a fabrication yard 1,400 miles away, and joined and post-tensioned together upon arrival at the site. The pier is moored to eight hollow concrete gravity anchors each measuring 20 ft. square in plan and 13 ft. deep. The cost of this floating pier project was reported to be \$48 million.

The construction of our pier will be similar. It is different from the Valdez pier mainly in size.

Deficiencies in the state-of-the-art technology are present primarily in relation to the two innovations as mentioned before, i.e. the hinge joint for the finger piers and the stiff-legged swivelling single-point mooring involving a suction anchor. These are commented further as follows:

### **Hinge Joint Between Finger and Spine Piers**

The purpose of the hinge joint is to enable the finger pier to be retracted during tow. For easy connection, the joint is located at deck level and consists, as conceived, of a link mounted on the finger pier that is dropped into a pin mounted on the spine pier when the finger pier is maneuvered into position. The maximum tension or compression at each of the two connections has been worked out to be in the order of 5,280 kips. Although reversal of stress is unlikely, considerable stress variation that may range from 0 to the maximum of 7,800 kips is possible, and will require that fatigue conditions be considered in designing the joints.

A concept of the hinge connection is shown in Figure 9.

### **The Stiff-legged Swivelling Suction Anchor**

The stiff-legged single point swivelling anchoring system that is developed for the pier is considered most likely to succeed from among the various possibilities. The

challenge that is presented to the designer are represented in the requirement that, (1) the system must be capable of developing the unusually large holding force necessary to keep the pier/ship complex and on station and (2) the system can be quickly operational upon arrival at site.

The latter requirement also includes the expectancy of the pier to be able to be retrieved rapidly for quick departure.

These requirements will rule out anchoring systems that require long installation and retrieval times, or anchoring systems that would fix the orientation of the pier thus subjecting it to current forces from all directions. The pier must be made to turn and face the sea at all times in order to minimize the holding force on the anchor. This leaves only the single-point deadman type of anchors for consideration.

A deadman anchor normally consists of a large concrete box, measuring say 25 ft. by 25 ft. on plan, which is filled with ballast such as sand, gravel or lean concrete, after it is lowered to the sea bottom. It is unfortunately not very suitable for the purpose of our pier because of the impractically large size involved, and the difficulty of recovering the deadman upon departure of the pier from a location. The deadman may have to be cut loose and left behind.

The suction anchor will solve most of these problems because of its relatively smaller size and weight, its large holding power because it invokes the resistance provided by a relatively large body of soil mass, and the short installation and extrication times.

Uncertainties and problem areas are as follows:

1. Application limited only to soils that are penetrable by the suction anchor.
2. Limitation on anchor depth of water. The anchoring system as shown may be applicable to water depths of up to 80 ft.
3. The design of the swivelling joint at both ends of the stiff-leg.
4. The requirement of a swivelling connection to the anchor, for the suction system (or pressure system for the extraction of the anchor) that can rotate around the anchoring system together with the stiff-leg and the pier.
5. The inability of the hinge joint to take large and quick rotational movements.

## **7. MATERIAL FEASIBILITY**

No difficulty is foreseen in using lightweight concrete as the prime structural construction material for the pier. Lightweight concrete can be made in good and consistent quality under the present technology. It has also been proven as suitable material in marine applications. An oft-quoted example is the "lightweight concrete ship," SS Selma, which was built during World War I. After the war it was scuttled and sunk in tidal waters in Galveston Bay. When examined years later, both the

concrete and the reinforcement were found to be in excellent conditions even though some of the reinforcement was protected by a concrete cover of no more than half an inch.

The problem concerning material mainly boils down to one of quality control. Secondary problems do exist however. For example, if the pier is stationary in one location for a long period of time, it will have the problem of constantly removing marine growth outside the hull. The concrete could be coated, or additive could be included in the mix that will inhibit marine growth.

Deficiencies in present technology also include the development of fendering materials that can absorb much higher pressure than current practice, to be used in cushioning the impact between the finger and the spine piers. The pier will also need a repair system that can repair all except the most severe damage to the hull, quickly and effectively.

As an extension of the expeditionary pier concept, it will be useful to develop more energy-efficient equipment, possibly solar-powered, that will enhance and prolong the usefulness of the pier on duty overseas.

## **8. CONSTRUCTION FEASIBILITY**

Construction methods are already available to build concrete structures such as the expeditionary pier. Unless a large number of piers are constructed, it can be assumed that the pier, in view of its size, will be constructed in segmental units which are subsequently assembled and joined together to make up the whole pier. The size of the unit is generally limited to 300 or 400 ft. in both horizontal directions. There are several methods of constructing the segmental units. They could be constructed on a special-purpose barge, then launched into the water when completed to a certain stage. Alternatively, they could be built in a shipyard or dry-dock type of facility, on slipways, or in a floor basin, which is the method shown in Drawing Nos. 11 and 12. In this method, the construction site at sea front is excavated so that the base of the site is below the sea level by a depth that is sufficient for the partially completed lower portion of the pier section to be floated out. During construction the sea is kept out by a temporary dike. When the lower portion of the pier unit, which mainly consists of buoyancy chambers, is built, water is allowed into the flood basin to float the unit. The dike is then beached, and the unit towed to deeper sheltered waters where further construction is continued until completion.

Deficiencies in the state-of-the-art technology with respect to the construction of the pier also exist in the areas that had posed difficulties to design and analysis, i.e., the finger/spine pier connection and the single-point suction anchor. Specifically:

1. The hinge joint at both ends of the link that connects the finger pier to the spine pier will provide for rotational movement in the vertical and horizontal directions. However, the joint will not provide for torsional movement between the two piers although some of this movement could be

tolerated by introducing more looseness in the joint. A universal or ball joint will solve this movement problem, but it has yet to be developed for this application.

2. The problem of connecting service lines across the joint
3. The construction of a swivelling joint for the suction hose connected to the anchor, and the problem of controlling the lines to avoid them getting entangled with the other components of the anchoring system as it swivels about the anchor.
4. The joining of the pier units will pose construction problems because of their size. The joining method shown in Figure 7 is the SOA method today. It could conceivably be improved to provide for more tolerances or allow for greater construction variations, and shorter connection time.
5. Quality control. Although this is not a deficiency in the sense that it is beyond the state of the art, the control of the quality and the consistency of concrete in our case will assume greater importance in view of the magnitude of the construction. A special effort will have to be made to ensure proper and stringent quality control.

## 9. CONSIDERATION OF ALTERNATIVES

As can be expected, an innovative concept is open for a vast range of possible alternatives, not only of the design relevant to a specific purpose, but also of a great variety of predictable situations for which the pier may be applied. The design as presented for the pier is therefore far from being a finished product. In earlier discussion in this report, three configurations of the pier have been presented, each incorporating modifications that have to do with reducing water resistance, and with removing the finger piers that may not be technologically ready if the pier is going to be constructed in the near-term future.

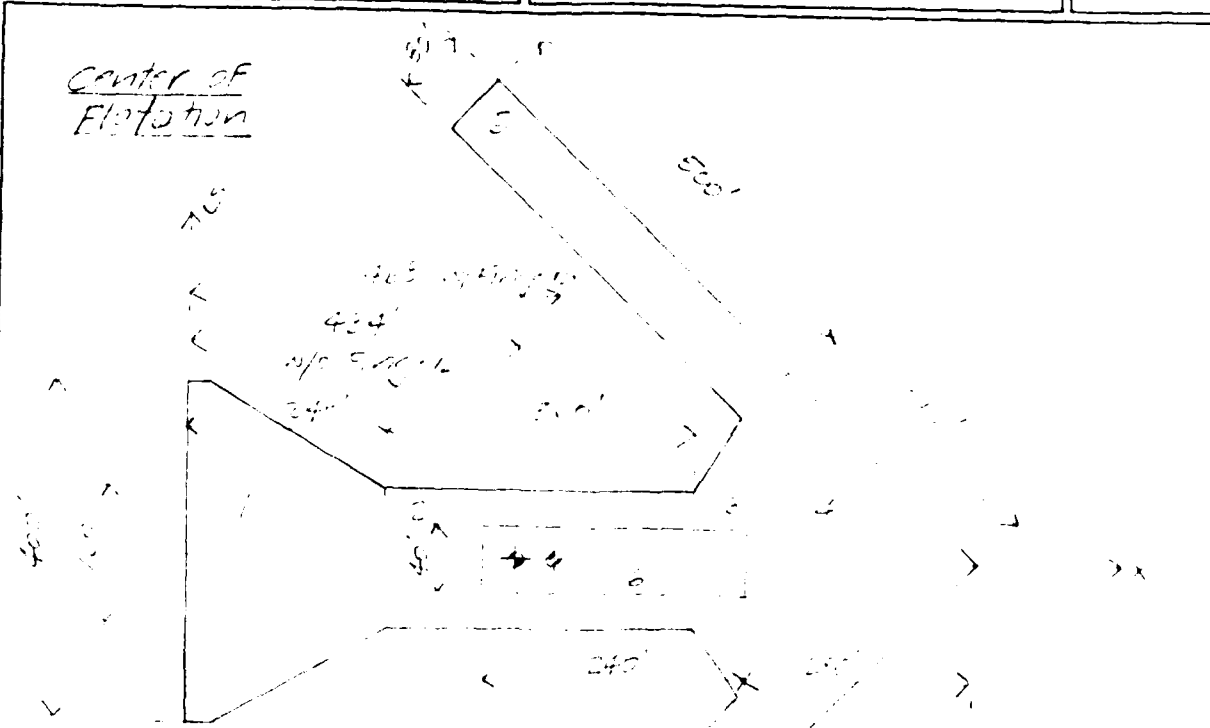
In view of the considerable size of the pier, it is conceivable that part of it, e.g., the finger pier, could be used as runways for military aircrafts, thus augmenting the role of the pier as a quasi-aircraft carrier. Figure 13 shows how the finger piers will look after their conversion into runways. By confining the runway to the forward part of the finger pier, it would still be possible to provide two berths along the lee side of the pier, thereby maintaining its berthing capacity to four destroyer class vessels in spite of the conversion. There is ample space below deck for the storage of planes and supplies to support the pier in this additional role.

## 10. CONCLUDING REMARKS

This investigation has shown the technological feasibility of an expeditionary pier that can be readily deployed to provide the full berthing facilities and services to Navy ships operating overseas. The cost of such a pier will be high. However, its availability will provide the Navy with a new option in its operation overseas, the value of which cannot be determined solely in terms of cost.

The development of the full pier, i.e., as shown in Scheme A or B, will depend on the successful development of two major innovations, i.e. the connection between the finger piers and the spine pier, and the stiff-legged swivelling suction anchoring system. These innovations and the problems they pose can be solved with additional efforts and time, part of which may run parallel to the further development of the pier itself. They should not in any case hamper the development of the pier, since at worst, these difficulties could be removed by supplementing them with systems that are already within the state of the art. For example, the incorporation of a dynamic positioning system to supplement whatever conventional anchoring system that may be used, instead of the innovative stiff-legged swivelling mooring as conceived. What it boils down to is whether there is a need for such a pier. If there is, there is sufficient reason to believe that the pier could be developed, designed and constructed based on the state-of-the-art technology in a matter of a few years.

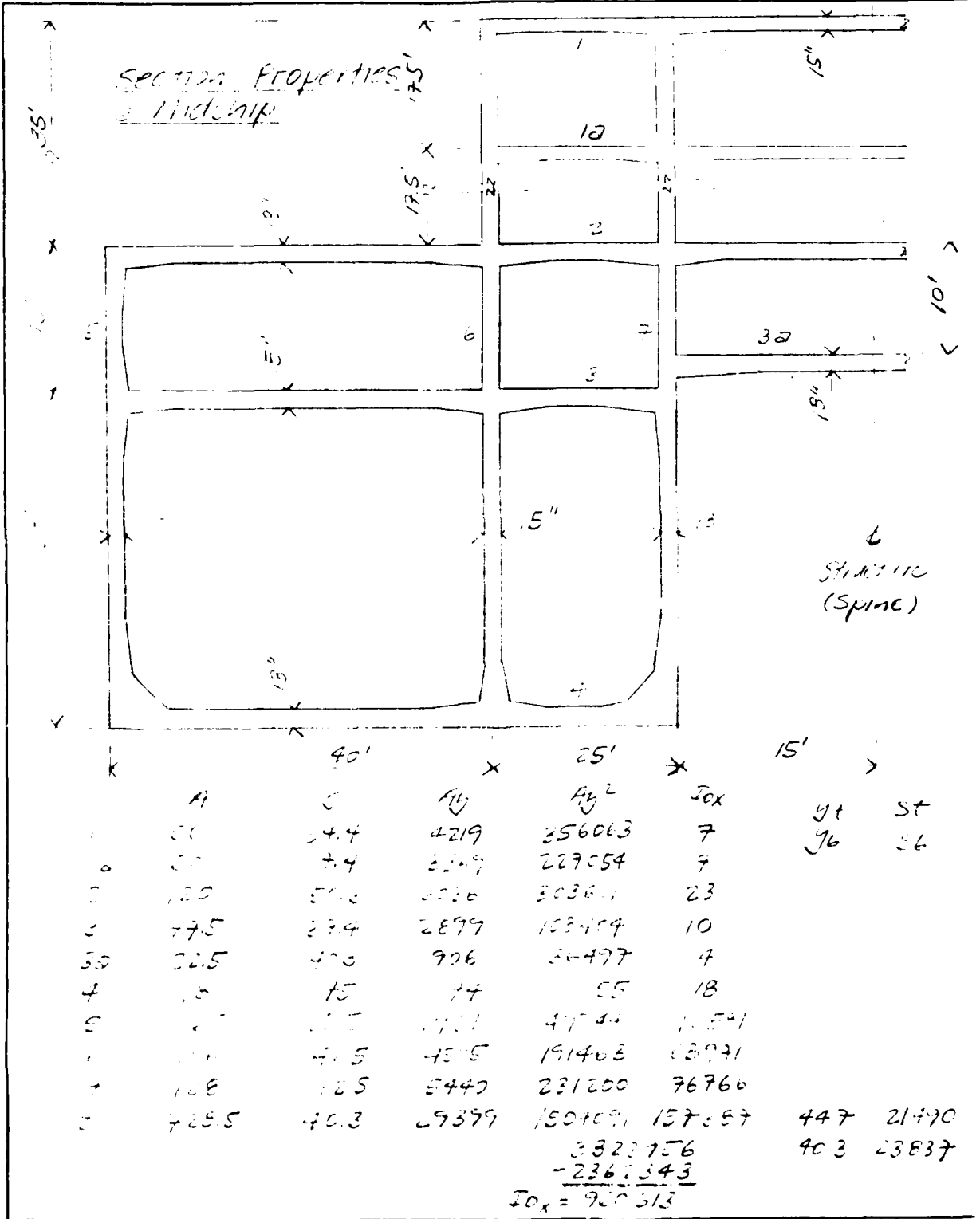




1/2 FLOTTING  
 $V = 1.31 \times 10^6$   
 $V = 1.31 \times 10^6$   
1/10 FLOTTING  
 $V = 2.04 \times 10^5$   
 $V = 400'$

	Area	Y	X	$Ax$	$Ay$	$I_x$
1	34400	100	200	3440000	3440000	2.22 x 10 <sup>10</sup>
2	57600	400	240	13824000	23040000	6.22 x 10 <sup>10</sup>
3	21000	200	200	4200000	4200000	4.0 x 10 <sup>9</sup>
4	20000	400	240	8000000	4800000	1.30 x 10 <sup>10</sup>
5	10000	100	200	1000000	2000000	4.07 x 10 <sup>9</sup>
6	10000	200	200	2000000	2000000	4.6 x 10 <sup>9</sup>
<b>Σ</b>	<b>2.84 x 10<sup>5</sup></b>			<b>1.31 x 10<sup>7</sup></b>	<b>7.8 x 10<sup>7</sup></b>	<b>1.90 x 10<sup>10</sup></b>

$I_{x1} = 7.99 \times 10^{10} - 2.84 \times 10^5 \times 200^2 = 1.95 \times 10^{10} \text{ F+2}$   
 $I_{x2} = 5.83 \times 10^{10} - 2.04 \times 10^5 \times 400^2 = 1.85 \times 10^{10} \text{ F+4}$



Calculating Average Seeding Moment

1) Max. A. outside bending moment  $M_{out}$

2) Min. A. outside bending moment  
31490 K-FT

3) Wave height  $H = 14.63$  Wave period  $T = 10.45$  Wave velocity  $V = 0.45 FC^2$   
4000 7500 FC = 7500 FT

$M_{out} = 14.63 \times 14.63 \times 31490 = 4.52 \times 10^7$  K-FT

3)  $M_{in}$  (K-FT)

3) Min. inside bending moment

$M_{in} = M_{out} - M_{sw}$

$M_{sw} =$  still water bending moment  
= 0

$M_{in} =$  maximum wave - induced bending moment  
=  $2.256 \times 10^7$

$\lambda = 0$  FOR  $C_u > 0.15$

$C_u = 1.17$   
 $C_u = 1.17 \times 10 + 1.57 \times 10^4 = 1.17 \times 10 + 1.57 \times 10^4 = 1.58 \times 10^4$

$\lambda = 0$   
 $\lambda = 16'$

$H_w =$  effective wave height of steady wave in FT.  
 $= [4.5L - 1.17 \times 10^4 + 1.57 \times 10^4] \times 10^{-2}$   
 $= 70.8$   $700 < L < 1000$  FT.

$M_{in} = 5.8 \times 10^7$  K-FT  
 $= 5.8 \times 10^7$  K-FT



ABS Wave Barring Moment and Shear Distribution

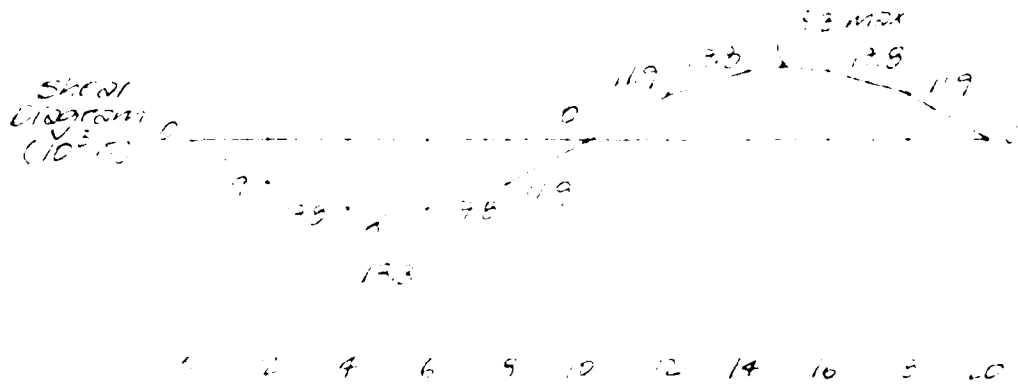
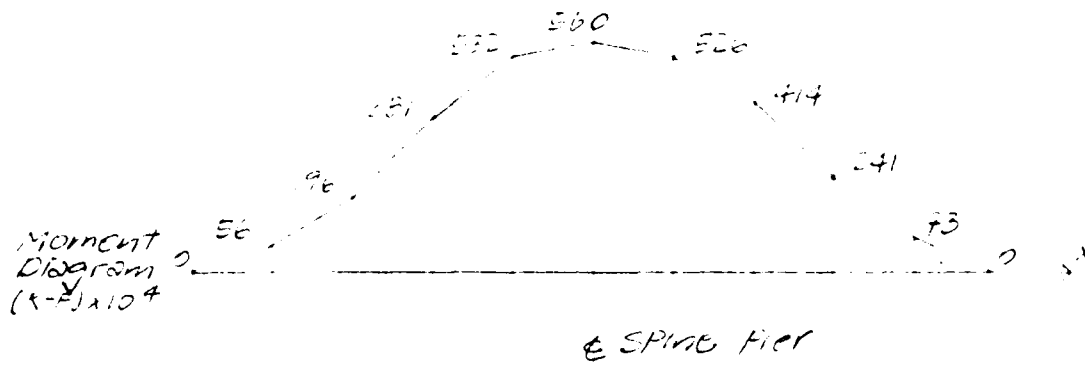
Max. =  $56 \times 10^6$  K-F      L = 990' , STATIONS @ 94'

STATION	LOADING MOMENT (USE HEIGHT = 20')	
0	0	= 0
2	$2.10 \times 56 \times 10^6$	= 560000
4	$2.35 \times 56 \times 10^6$	= 1320000
6	$2.65 \times 56 \times 10^6$	= 2308000
8	$2.95 \times 56 \times 10^6$	= 3320000
10	$3.2 \times 56 \times 10^6$	= 5600000
12	$3.94 \times 56 \times 10^6$	= 5264000
14	$3.74 \times 56 \times 10^6$	= 4144000
16	$3.43 \times 56 \times 10^6$	= 2908000
18	$3.2 \times 56 \times 10^6$	= 728000
20	0	= 0

SHEAR FORCES

0	0	= 0
2	$560000 / 94'$	= 11915
4	$(2320000 - 560000 - 11915 \times 94) / 94 + 11915$	= 17972
6	$2320000 - 11915 \times 188' = 17972$	= 18295
8		= 17972
10		= 11915
12		= 0
14		= 11915
16		= 17972
18		= 18295
20		= 7972
22		= 11915
24		= 0

UNSAFE Moment and Shear Envelopes



Spine Pier Loading: Supporting Columns

Primary Stresses  $M_{max} = 5600000 \text{ Lb-F}$

$$f_{top} = 5600000 / 21490 = 261 \text{ KSF} = 1310 \text{ PSI } (\pm)$$

$$f_{bot} = 5600000 / 5429 = 103 \text{ KSF} = 1021 \text{ PSI } (\pm)$$

Secondary Stresses of Bottom Slab: (+950')

a) At column due to buckling of 3 @ 20' longitudinal slab span 30'

Load Load:

$$\text{Concrete } 2.110 \text{ } = 0.21$$

$$\text{Arch + Misc. } = 0.10$$

$$\text{Dead load } = 0.064 + 0.0010$$

$$\text{Resultant } = 2.274$$

max. compress (supp)

Slab Section Properties:

At Support  $= 23''$  (assumed to mid-third of pier)

$$I = 0.754$$

$$S = 0.6 + 0.2$$

b) Moment =  $2.274 \times 10' = 22.74 \text{ K-F}$

$l_n > 2.0 \rightarrow$  one-way slab

$$M_{max} = 1/4 \times l_n^2$$

$$= 2.274 \times 10^2 / 4 = 56.85 \text{ K-F}$$

$$\text{Stresses } = 56.85 / 0.053 = 1072 \text{ PSI } (\pm)$$

Secondary Stresses of Top Slab: (+950')

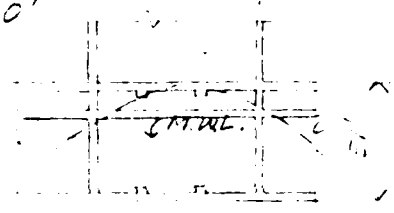
a) Loading:

$$\text{Load Load: Concrete } 3/10 \times 12 = 0.19$$

$$\text{Arch + Misc. } = 0.10$$

$$\text{Live load: } 0.40 \text{ KSF} = 0.40$$

$$\text{Total load } = 0.684$$



Cont Spine Pier Stresses @ Top & Base:

a) Moment @ support  $M = WLn^2/11$   $Ln = 19'$   
 $L_1/Ln > 2.0$   $L_1 = 40'$   
 $C = 0.68 \times 2.0^2/14 = 1.9$

Sub Section  $Ln = 13'$  anchored @ support  
 $I = 1113 \text{ FT}^4$   
 $S = 138 \text{ FT}^3$

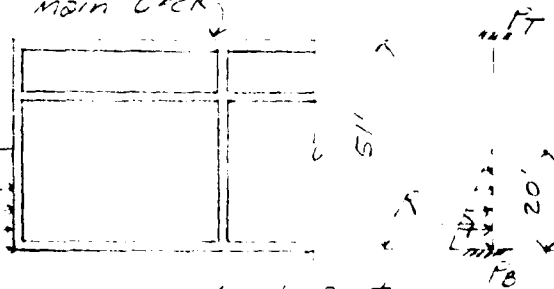
b) Stresses  $F = 1.9 \times 9 = 17.1 \text{ KSF}$   
 $17.1 \times 8 = 136.8 = 40 \text{ KSF}$   
 $= 320 \text{ PSI } (\pm)$

Additional Compression Stresses due to Water Head:

a) Lead water

1) Lead deck only, for maximum division  
 Draft = 19.5'

Resultant Compression  
 $F_{max} = 0.064 \times 100^2/2 = 324 \text{ K}$   
 $F = 324 \times 19.5/51 = 123 \text{ K}$   
 $F_2 = 123 - 112 = 11 \text{ K}$



Spine longitudinal sect.

Consider bottom only as top loading is taken by main deck which is not the critical one.

Bottom  $F/A = 1100/24 \times 12 = 47 \text{ PSI}$

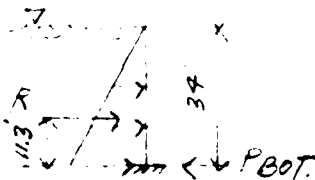
Top

b) Live load + Lead deck  
 Draft = 39'

Resultant =  $0.064 \times 100^2/2 = 324 \text{ K}$

Fact =  $324 \times 39.7/51 = 250 \text{ K}$

$F_{TOP} = 250 - 240 = 10 \text{ K}$



Cont' Still Water Head Stresses:

Stresses due to live + dead loads drift:

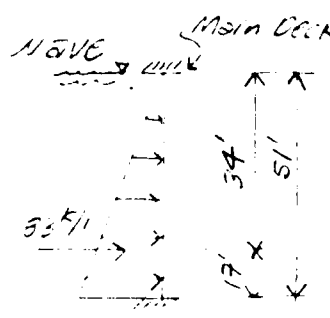
At support =  $T_{BOT} = 29000 / 24 \times 12 = 101 \text{ ksi}$   
 $T_{TOP} = 4000 / 18 \times 12 = 27 \text{ ksi}$

2) Wave Induced Water Head Stresses:

Assume avg. wave height of 20'

Total Water Head: Dead + Live loads + wave  
 $= 51' \text{ max}$   
 Resultant =  $0.64 \times 51^2 / 2$   
 $= 93 \text{ k/ft}$

Top =  $93 \times 17 / 51 = 39 \text{ k/ft}$   
 Bot. =  $93 \times 34 / 51 = 55 \text{ k/ft}$   
 Stresses:  $T_{BOT} = 55000 / 24 \times 12 = 144 \text{ ksi}$



Combination of Stresses:

	Max. Tension (ksi)		Max. Compression (ksi)	
	Longitudinal	Transverse	Longitudinal	Transverse
Moment	1910		1631	1631
Secondary	±20	±20	521	-
ε	2130	320	2152	1631
Water Head			+144	+144
Total Max.	2130	320	2296 ✓	1775 ✓

Longitudinal Prestressing:

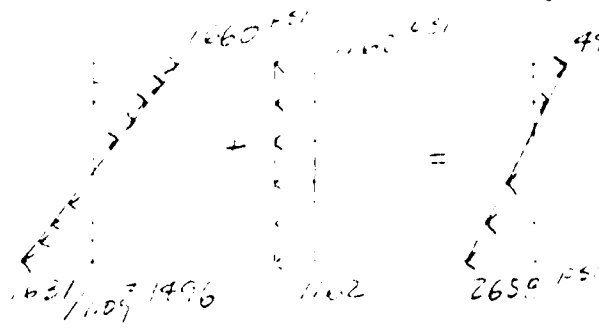
Partial - Allow 40% of prestress necessary to offset concrete stresses due to general curling. The balance being taken by tension in the slab.

TRANSVERSE AREA = ALLOWED PRESTRESS  
 IN THE SLAB TO BE 14% OF TOTAL CROSS-AREA  
 $14\% \times 1456 = 2038 \text{ SF} \rightarrow$  INCREASE  
 SF = 95% CONC. AREA  
 PRESTRESS =  $2793 \times \frac{1162}{1000} = 3223 \text{ PSI}$   
 CONCRETE STRESS = 1650 PSI  
 PRESTRESSING = 498 PSI (mild steel)

Final Stresses For Slab Prestressing (compression)

STRESS (PSI)	LONGITUDINAL	TRANSVERSE
Concrete curling	1650	2793
Slab Prestressing	3223	144
<b>Total</b>	<b>4873</b>	<b>2937</b>

$f_c \text{ REQUIRED} = 4873 - 144 = 4729 \text{ PSI}$   
 $1650 \text{ PSI} = 2500 \text{ PSI}$  (light weight concrete)



Total  
Prestressing Force:  
 $1162 \times \frac{144}{1000} \times 1456$   
 $= 243,630 \text{ K FORCE}$

Wave curling + Prestressing = Final Primary Stresses

Weight of Structure (Wing Fleet)

Wing Fleet:

Wing Fleet:  
 $0.120 \times 145 \times 1200 + 145 \times 160$   
 $= 20784 \text{ k}$   
 Top flange weight =  $0.045 \times \frac{1}{2} \times 10 \times 1200 = 1602$   
 Rib weight =  $0.100 \times 100 \times 3 \times 1200 = 17000$   
 Transverse stiffeners =  $0.120 \times 15 \times (200 + 700) \times 4$   
 $= 7750$   
Total = 91446 k

Wing Fleet:  $11000 = 34400 + 7500 = 75800 \text{ k}$

Wing beam =  $0.120 \times 1.953 \times 500 = 1791 \text{ k}$   
 Transverse stiffeners =  $0.120 \times \frac{1}{2} \times (200 + 700) \times 4 \times 1200$   
 $= 17910 \text{ k}$

Wing ribs =  $0.100 \times 100 \times 3 \times 1200 + 200 + 200$   
 $+ 260 \times 47 = 9165 \text{ k}$

Wing stiffeners =  $0.100 \times \frac{1}{2} \times (200 + 700) \times 4 \times 7$   
 $= 1190 \text{ k}$

Wing stiffeners =  $0.120 \times 75800 \times 4 \times \frac{1}{2} \times 1200$   
 $= 16658 \text{ k}$

Wing stiffeners =  $10 \times 0.05 \times 1200 \times 200 + 100 + 100 + 1000$   
 $+ 10 \times 40 + 420 \times 15 \times 10 = 9175 \text{ k}$

Total wing fleet = 145812 k

Total wing =  $16658 \times 75800 = 1074 \text{ k}$

Wing stiffeners =  $0.100 \times 75 \times 10 \times 3 = 22740 \text{ k}$

$\Sigma$  wing stiffeners =  $16658 \text{ k}$

Total wing fleet = 91446 + 16658 = 258336 k

Wing Fleet:

Wing beam =  $0.05 \times 1200 \times 10 = 7200$   
 Wing ribs =  $0.600 \times 20 \times 600 \times 4 = 12960$   
 Wing stiffeners =  $0.010 \times 15 \times 4 \times 1200 \times 7 = 1670$   
 Assumed weight of H2O  
Total wing fleet = 258336 k ←  
Total wing = 490664 k ←

PROJEC	12-3-1975
ITEM	EXPERIMENTAL
DESIGN	ROLL PERIOD
DATE	1-25-76

SHEET	A-13
OF	
REVISION	

Roll Periods

WIND

$$2.0 \times 10^6 \text{ lbs} \times \frac{1.5336 \text{ K}}{204,000 \text{ ft}} = 19.5$$

$$\begin{aligned} \text{Total WIND} &= 19.5' + 232,000 \text{ ft} \\ &= 19.5' + 17.8' \\ &= 37.0' \quad \leftarrow d \end{aligned}$$

VERTICAL CENTER OF GRAVITY H UNDER FULL LOAD

$$\begin{aligned} H &= \frac{(27,440 \times 40) + (20,000 \times 20.5) + (100,000 \times 28) + (20,000 \times 50) + (90,000 \times 75)}{490,664} \\ H &= \frac{19,274,932}{490,664} \\ &= 40.4' \quad \text{FROM BASE WVC} \end{aligned}$$

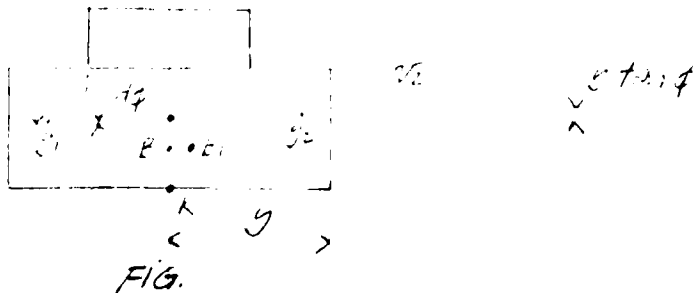
CENTER OF GRAVITY

$$KB = 40.4 / 2 = 20.2' \quad \text{FROM BASE WVC}$$

BASE RIGIDITY

$$k = \frac{250 \text{ } \dots}{100} = \frac{490,664 \text{ } \dots}{204,000 \times 37.6' \times 24 / 1.5}$$

Diagram of Structure Height



$$\bar{EM} = \frac{251}{20.16} = \frac{20 \times 9.8 \times 2}{\Delta \times 20.16}$$

$\Delta$  Volume of Displacement = 54

$\rho$  = Volume of 20% water

Rolling angle  $\phi =$

$$\phi = \frac{2.02 \times 10^4 \times 10^3}{2.02 \times 10^4 \times 10^3} = 0.17'$$

$$\Delta = \frac{2.02 \times 10^4 \times 10^3}{2.02 \times 10^4 \times 10^3} = 2.02 \times 10^3 \text{ (CF)}$$

$$\bar{EM} = \frac{251}{20.16} = 12.45'$$

$$\bar{EM} = \bar{EM} + \bar{EM}$$

$$\bar{EM} = 20.2'$$

$$\bar{EM} = \bar{EM} - \bar{EM}$$

$$\bar{EM} = 40.4'$$

$$\text{Rolling period } T_R = \frac{2\pi \sqrt{\bar{EM}}}{\sqrt{GM}}$$

$$\bar{EM} = 175'$$

$C = 1.5 = \text{constant for position}$

$\phi$	$\Delta$	$\Delta$ (CF)	$\bar{EM}$ (ft)	$\bar{EM}$ (ft)	$GM$ (ft)	$T_R$ (sec)
10°	207	$2.07 \times 10^3$	$1.85 \times 10^3$	105	125	9.9
20°	207	$2.07 \times 10^3$	$4.99 \times 10^3$	105	125	10.0
30°	207	$2.07 \times 10^3$		115		

Rolling moment  $M_R = W \times GM \times \sin^2 \phi$

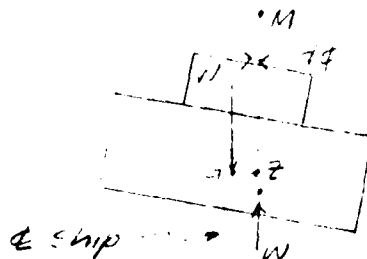
$$M_R = 490664 \times 1.48'$$

$$= 726183 \text{ ft}^2$$

$W = \text{weight of ship}$

$\bar{EM} = \text{height of ship}$

$$= GM \sin^2 \phi = .48'$$



**TYLON**  
INTERNATIONAL  
STRUCTURAL ENGINEERING  
315 Bay St., San Francisco, Ca 94133

PROJECT	INDY RACE
ITEM	EXIST TRACK RISE
DESIGN	- TCA PERIOD
DATE	8/30/10

SHEET	A-15
OF	
REVISION	

$$\overline{GM} = \frac{I_{xx} - \overline{KB}}{\nabla} = \frac{4222145 - 20.2 \times 10^6}{767000}$$

$$\phi = 10^\circ$$

$$I_{xx} = \frac{1}{12} \pi r^4 = \frac{1}{12} \pi (27')^4 = 4222145 \text{ ft}^4$$

$$\nabla = 214000 \times 3.6 \times 10^3 = 767000 \text{ ft}^3$$

$$\overline{KB} = \frac{1}{3} (27') = 9'$$

$$\overline{KB} = \frac{4222145 \times 0.27'}{767000} = 345'$$

$$\overline{KM} = 20.2' + 345' = 365'$$

$$\overline{GM} = 365' - 40.4' = 325'$$

$$\text{Pitch} = \frac{155 \times 120'}{325} = 56.3 \text{ SEC}$$

$$\phi = 10^\circ \rightarrow \text{Pitch} = 27 \text{ SEC}$$

Finger Flets

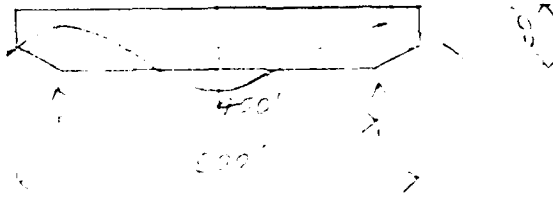
Use a Finger as a representation of structure as  
 interaction forces between main structure and finger  
 are less at primary than secondary level forces.  
 Support conditions at various points at the interface  
 of the structure in the design of connections.

1.0 1.0

1.0 State 0  
 Max. Wave Height = 20'  
 Max. Wave Length = 400'

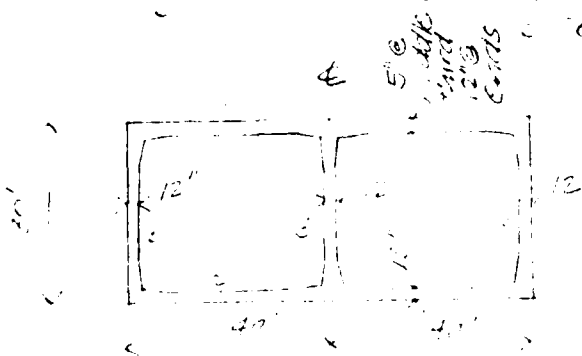
Section of  
 Fletting

& Finger



Section Properties of Finger

Height of Finger = 20' - 30' - 30'  
 Length of Finger = 15' - 20'



A	A <sub>1</sub>	A <sub>2</sub>	I <sub>1</sub>	I <sub>2</sub>	I <sub>3</sub>
96	29.4	2820	82908	10	10
70	21.03	60	33	15	15
90	15	1330	20005	2750	2750
282	15	4030	109927		
			63450		15
			46477		

$S_x = S_y = 46477 \text{ in}^3$   
 $= 5098 \text{ ft}^3$

ENVIRONMENTAL LOADS

NAVA 100 WALL - Primary Stresses

FRIDGE FRIS SECTION IS BUILT IN TWO STAGES. BUILT ONE  
 24' HIGH SECTION AND THEN TO 40' HIGH. FLOW DOWN  
 CONSTRUCTION IS NOT SEPARATE FROM TYPICAL WALL  
 OF FIRST FLOOR

MAXIMUM LOAD IS APPLIED AND OCCUR WHILE FRIS IS IN SERVICE.  
 THAT IS IN THE OPEN DESIGN FRIDGE FRIS PERIOD  
 HIGH, 40' LONG NAVA. (SEE STATE 6) A REDUCTION IN  
 FORCES CAUSED BY SALT WATER ENVIRONMENT.

TOTAL BENDING MOMENT = ABS RULES SECTION 6.3.2)

$$M_T = M_{DL} + M_{LL}$$

DL = DEAD WEIGHT BENDING MOMENT =

- DUE TO THE WALL IS STRUCTURE IS APPROXIMATELY  
 UNIFORM THICKNESS AND MASS AND UNDER AT  
 ALL POINTS DURING THE SERVICE CONDITION.  
 THIS MEANS THAT THE WALL IS BEING STIFF  
 WITH THE MAXIMUM DEAD WEIGHT FORCES.

$$M_{DL} = (24 \times 10 \times 10)$$

$$\begin{aligned} &= 24 \times 10 \times 10 \\ &= 2400 \text{ K-F} \\ &= 20.51 \end{aligned}$$

$$M_{LL} = 0.45 \times 6500 \times 24 \times 10 = 6400 \text{ K-F}$$

MAXIMUM ALLOWABLE BENDING MOMENT =

$$M_{all} = (F_c) =$$

$$\begin{aligned} &= (0.45 \times 6500) \times 24 \times 10 \text{ (IMPRESSION STRESS) (KSI)} \\ &= 6500 \text{ PSI} \\ &= 0.45 \times 6500 = 2925 \text{ KSI} \\ &= 211 \text{ K-F} \end{aligned}$$

SECT. MAX. ALLOWABLE BENDING MOMENT:

$S_x = 3098 \text{ F}^4$   
 $M_{all} = 211 \times 3098 = 653678 \text{ K-F}$

Determine the Max. Depth of Water Height by Projection:

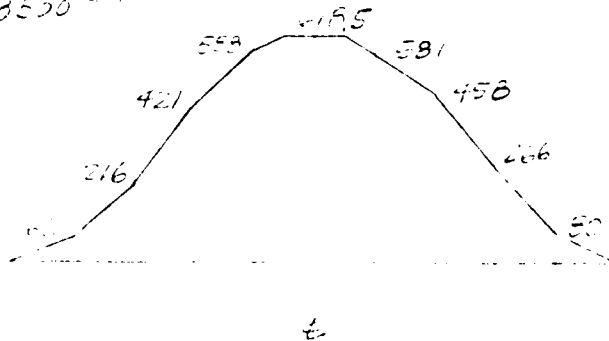
$\frac{M_{all} \cdot H}{\text{Max } M_E} = \frac{M_C}{M_{in}} \rightarrow H_{max} = \frac{M_E \cdot M_C}{M_{in}}$

$H_{max} = \frac{653678 \times 20^2}{634000} = 41.1' > 20'$

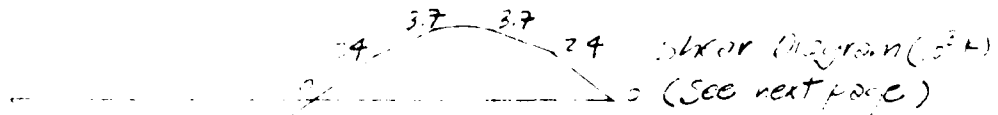
Max. Water Height is above L.C. limit of Sec. STATC 6  
For  $H = 20'$   $\rightarrow M_{max} = 20/20.5 \times 634000 = 618537 \text{ K-F}$

ABS. ALLOW. SECTION MOMENT DISTRIBUTION = Table 6.1

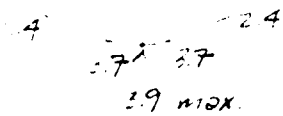
Cap. = 618500 K-F



Moment Diagram  
(K-F) x 10<sup>3</sup>  
7X



Shear Diagram (K-F)  
(See next page)



37 MAX.

WIND EXERCISES, MOMENTS AND SHEARS (450 KIPS)

POSITION 3.000' MOMENT (WIND HEIGHT = 20')

POSITION	WIND HEIGHT	MOMENT (KIP-FT)
0	0	= 0
2	216.500	= 61950
4	433.000	= 216475
6	649.500	= 420590
8	866.000	= 587075
10	1082.500	= 619500
12	1299.000	= 587070
14	1515.500	= 457690
16	1732.000	= 265955
18	1948.500	= 80405
20	2165.000	= 0

(WIND FORCE (K))

0	0	= 0
2	$(61950/20) \times 2$	= 2474
4	$((216475 - 61950) / 2) \times 2 + 2474$	= 2711
6	$2474 \times 500' + 2711$	= 3436
8		= 3711
10		= 2474
12		= 0
14		= 2474
16		= 3711
18		= 3436
20		= 3711
22		= 2474
24		= 0

Finger Hoopling Sagging

Primary Stresses = Max  $618500 \text{ Lbf}$

$$f_t = 618500 / 3078 = 199.6 \text{ } \frac{\text{Lbf}}{\text{in}^2} = 1380 \text{ } \frac{\text{psi}}{\text{in}^2}$$

$$f_c = 618500 / 3078 = 199.6 \text{ } \frac{\text{Lbf}}{\text{in}^2} = 1380 \text{ } \frac{\text{psi}}{\text{in}^2}$$

Secondary Stresses @ Bottom S/S

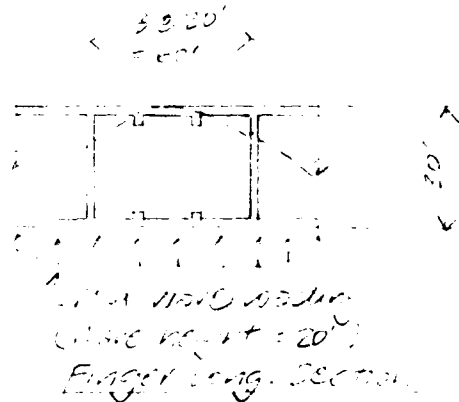
a) 4' bottom due to buoyancy =  
negative side of  $1' = 200'$   
D.L.:

$$\text{Concrete} = 15 \times 12 \times 15 = 0.18 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$

$$\text{Misc + Misc} = 0.05$$

$$\text{Max. buoyancy} = 1064 \times 100' = 1.9$$

$$\text{Resistant} = 1.69 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$



Side Section Properties

$$\text{At support: } I = 0.28 \text{ } \frac{\text{ft}^4}{\text{in}^4}$$

$$\text{Height } h = 15', S = 0.39 \text{ } \frac{\text{ft}^3}{\text{in}^3}$$

b) Moment:  $L_1 = 10', L_2 = 40'$

$$h_{\text{net}} = 20'$$

$$\text{Moment} = \frac{1}{11} L_1^2 L_2^2$$

$$= 1.69 \times 10^2 \times 1600 = 55 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$

c) Stresses:

$$C = 55 \times 1.33 = 107 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$

$$= 756 \text{ } \frac{\text{psi}}{\text{in}^2}$$

increase allowable by 33% as loading is temporary

Secondary Stresses @ Top S/S

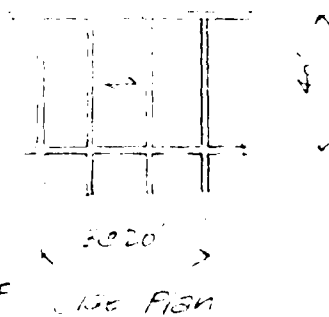
d) Loading:

$$\text{D.L. Concrete} = 15 \times 12 \times 12 = 0.15 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$

$$\text{Misc + Misc} = 0.10$$

$$\text{Live Load} = 1 \text{ Allowable so that} = 1.00 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$

$$\text{Freeboard same as Exped. H) } 0.55 \text{ } \frac{\text{Lbf}}{\text{in}^2}$$



Load Finger Pier Stresses @ Top Slab :

a) Moment:  $M = wL^2/11$   
 $= .55 \times 11^2 / 11 = 1.1 \text{ K-F}$

b) Section Mod.  $I = 116 \text{ FT}^4$   
 $t = 15 \quad S = 0.06 \text{ FT}^3$

c) Stresses:  $f = M/S = 18,126 = 29 \text{ KSF}$   
 $= 491 \text{ PSI} \quad (\pm)$

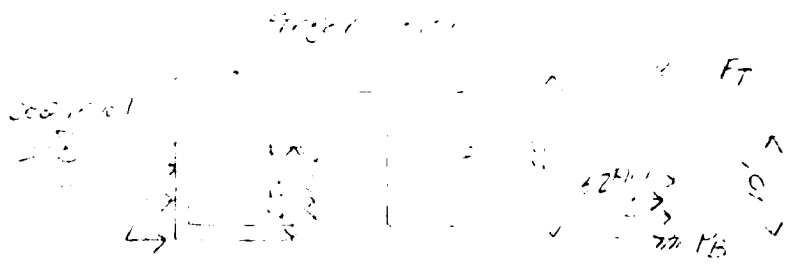
Combination of Stresses:

	Max Tension (PSI)		Max Compression (PSI)	
	Longitudinal	Transverse	Longitudinal	Transverse
Wind	1386	1386	1086	1386
Seawind	291	—	356	—
D	1867	1386	2142	1386
Water Head		-16	+106	+106
Total Max.	1967	1370	2248 ✓	1492 ✓

\* 2' x 2' total expansion due to water head!

1) Slab Water:

5' Load  $\rightarrow$   $L = 10'$   
 Resultant:  $10 \times 10 \times \frac{1}{2} = 50 \text{ K}$



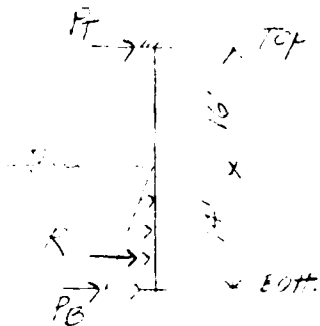
$F_T = 50 \times \frac{5}{10} = 25 \text{ K}$   
 $F_B = 25 \text{ K}$

NOTE: These reactions are high, since exterior slab also spans horizontally

TOP:  $f = \frac{50 \times 10 \times 10}{150} = 19 \text{ KSI}$  more  
 BOTTOM:  $f = \frac{250 \times 10 \times 10}{150} = 16 \text{ KSI}$

Cont' Water Head Stresses (Still Water)

2) Live Load + Dead Load,  $h_{draft} = 14'$   
(Under 250 PSF Live Load)



$$\text{Resultant} = 2004 \times 14 \times 1/2 = 6.341$$

AT  $1/3 = 47'$  FROM BOTTOM

Reactions:

$$R_A = 0.33 \times 47 \times 30 = 10.41$$

$$R_B = 5.341$$

STRESSES DUE TO DRAFT OF DEAD + LIVE:

$$f_{TOP} = 1500 \times 14 \times 12 = 55 \text{ PSI (TENSILE)}$$

$$f_{BOT} = 5300 \times 14 \times 12 = 21 \text{ PSI}$$

3) WAVE HEIGHT

ASSUME WAVE HEIGHT OF 30'

$$\text{Resultant} = 2004 \times 30 \times 1/2 = 29.41$$

$$R_A = 0.33 \times 30 = 10.5$$

$$R_B = 29 \times 0.3 = 19.41$$

Stresses:

$$f_{TOP} = 1500 \times 14 \times 12 = 55 \text{ PSI}$$

$$f_{BOT} = 19000 \times 14 \times 12 = 106 \text{ PSI}$$

ANALYTICAL PROVISIONS

1. Assume 15% CF increase in yield strength  
due to welding. This is based on the  
fact that the yield strength of the  
material is not a constant function of  
temperature.

Yield strength of material = 100,000 PSI  
Take 15% increase in yield strength =  $100,000 \times 1.15 = 115,000$   
Increase =  $115,000 - 100,000 = 15,000$  PSI → 15% increase

Yield strength =  $\frac{1386}{1.07} = 1295$  PSI

Residual Tension:  $1295 - 990 = 305$  (in steel)

2. Assume 10% increase in yield strength due to compression

Yield strength of material = 100,000 PSI

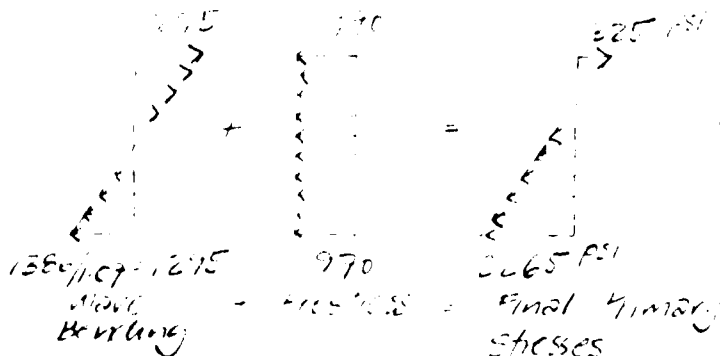
Yield strength of material =  $100,000 + 10\% = 110,000$  PSI

Yield strength of material =  $110,000 - 106,000 = 4,000$  PSI

Yield strength of material =  $\frac{750 + 106}{3.27} = 2438$  PSI

Yield strength of material =  $2438 - 225 = 2213$  PSI

Yield strength of material =  $2213 + 10\% = 2434$  PSI



Total Postensioning:

$140 \text{ KSF} \times 282 \text{ SF} = 39,480 \text{ K}$

Finger Floor WEIGHTS:

Dead Loads:

$$A = 282 \times 1/2 + 263 \times 2/3 = 270^{SF}$$

$$W_{slab} = 150 \left[ \frac{270}{12} \times 500 + 11,385 + 77,110 \right] = 1,441 \text{ K}$$

$$W_{beam} = 120 \times 1.5 \times 2 \times 12 = 540 \text{ K}$$

$$W_{part} = 300 \text{ K}$$

$$W_{flooring} = 50 \times 2 \times 4000 = 4000 \text{ K}$$

$$W_{partitions} = 2.100 \times 40000 = 84000 \text{ K}$$

$$\text{Total dead load} = 24639 \text{ K}$$

It is noted that in order to keep finger floor height at finger floor level as expeditiously as possible, the floor should be placed above the floor level. It is suggested that the floor be placed in order to bring main floor to same level.

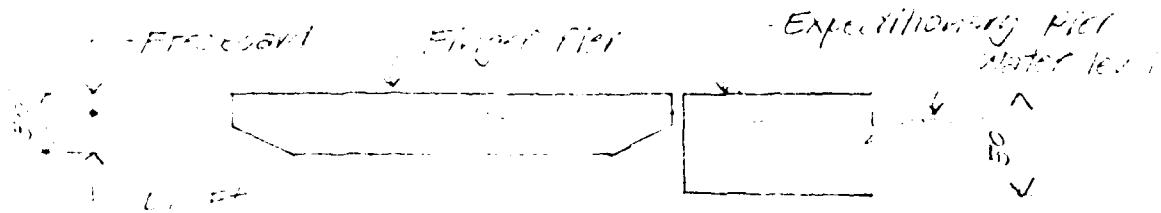
See also drawings for storage tanks, piping, etc. tanks, weapons, etc.

Height (ft)	Weight (K)	Total (K)	E (K)	Dist (ft)	From (ft)
0	0	24639	24639	9.6	20.6
10	3000	"	28639	11.0	19.0
20	6000	"	34639	12.6	17.4
30	9000	"	40639	14.2	15.8
40	12000	"	48639	15.7	14.3
50	15000	"	48639	17.3	12.7
60	24000	"	48639	19.8	11.2

LIFT and freedom

Expeditionary Pier maximum live load draft: 17.8'  
 DUE TO LIVE LOAD AVERAGE = 520 PSF  
 Total live load Area = 378400 sq'

LIVE LOAD RANGE (PSF)	TOTAL LIVE LOAD (K)	TOTAL AREA (SQ FT)	DRAFT (FT)	FREEDOM (FT)
0	255600	255600	19.6	30.4
100	273440	"	20.5	29.5
200	291280	"	25.4	24.6
300	309120	"	28.3	21.7
400	326960	"	31.2	18.8
500	344800	"	34.1	15.9



LIFT and freedom vs. LIFT FREEDOM COMPARISON =

Expeditionary Pier Draft (ft)	Freedom Pier Draft (ft)	Expeditionary Pier Freedom (ft)	Freedom Pier Freedom (ft)
17.8	30.4	9.9	Exp.
20.5	29.5	12.0	Exp.
25.4	24.6	19.2	Exp.
31.2	18.8	11.2	Exp.
34.1	15.9	18.8	Exp.

Joint Freeboard Comparison:

Summary:

- In order to maintain same freeboard between Finger and Expeditionary Pier either one of the pier must be bolstered. Assuming that standard wave the equivalent most common wave load in the E.P. is 400 PSF, then the additional uniform load on the finger pier must be around 330 PSF in order to maintain same freeboard of 19.5 ft.
- The minimum average superimposed load in the Expeditionary Pier should be around 330 PSF so that the maximum freeboard allowed of 20.5 ft. is not exceeded.
- Since the freeboards of the two structures will most likely be different due to the action of waves, wind and current it is necessary to joint the piers with either a hinge, free of rotation or chain like type of connection.
- To keep freeboard differences to a minimum there should be a system on the pier that monitors freeboards and adjusts other pier in order to put the decks of both structures.

Roll Period:

DRAFT:

$$\text{Lead load draft} = \frac{24639^k}{3.064 \times 50 \times 500} = 7.6'$$

Live load draft:

$$\begin{aligned} 3 \times 150 \text{ ft} &= \frac{6000}{3.064 \times 50 \times 500} = 2.37' \\ \Sigma(L \times C) &= 11.9' \end{aligned}$$

Vertical center of Gravity + under lead + live loads

$$H = \frac{24639^k \times 15' + 6000^k \times 30'}{30639^k} = 17.9' \text{ (From bottom)}$$

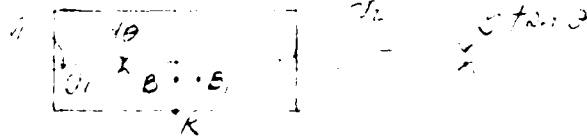
Center of buoyancy

$$\bar{KB} = 17.9' / 2 = 9' \text{ From base line.}$$

Block coefficient at Draft 1.9

$$C_b = \frac{35 \Delta}{L \times B \times d} = \frac{30639}{40000 \times 1.9 \times 2.29 / 35} = 1.0$$

Transverse Metacentric Height



$$\overline{GM} = \frac{B \bar{M}}{\sin \phi} = \frac{J_c \times g \times \rho_w}{\rho \times \tan \phi}$$

Use a small angle  $\phi$

$$J_c = \frac{1}{12} b^3 \times d \times \rho_w$$

$$J_c = 50'$$

$$V = \bar{L} \times \bar{B} \times d \times C_b = 40000 \times 1.9 \times 2.29 = 176 \times 10^3 \text{ F}^3$$

$$\overline{J} \rho_w = \frac{1}{12} b^3 d \rho_w = 55'$$

Cont' Finger Periods:

$$\overline{KM} = \overline{KB} + \overline{BM}$$

$$\overline{KB} = 9' \text{ FROM BASE}$$

$$\overline{GM} = \overline{KM} - \overline{KG}$$

$$\overline{KG} = H = 17.9' \text{ FROM BASE}$$

$$\text{FOR } \phi = 10^\circ$$

$$V = \frac{1}{2} 40^2 \tan 10^\circ \times 500'$$

$$= 70531 \text{ F}^3$$

$$\overline{BM} = \frac{70531 \times 53}{476 \times 10^2 \times \tan 10^\circ} = 44.5'$$

$$\overline{KM} = 9 + 44.5 = 53.5'$$

$$\overline{GM} = 53.5 - 17.9 = 35.6'$$

$$\Delta \text{CILL PERIOD } T_\phi = \frac{C \overline{B}}{\overline{GM}}$$

$$C = 0.52 \text{ (Pontoon)}$$

$$\overline{B} = 80'$$

$$T_\phi = \frac{0.52 \times 80}{\sqrt{35.6}} = 6.7 \text{ SEC } \leftarrow$$

PITCH PERIOD:

$$\overline{BM}_L = \frac{V \times \overline{GL}^2}{V \tan 10^\circ \phi}$$

$$\text{FOR } \phi = 10^\circ$$

$$V = \frac{1}{2} \left( \frac{500}{2} \right)^2 \tan 10^\circ \times 80 = 4408 \times 10^3 \text{ F}^3$$

$$V = 40000 \times 119 \times 10 = 476 \times 10^6 \text{ F}^3$$

$$\overline{GL} = \frac{1}{3} (500/2) = 333'$$

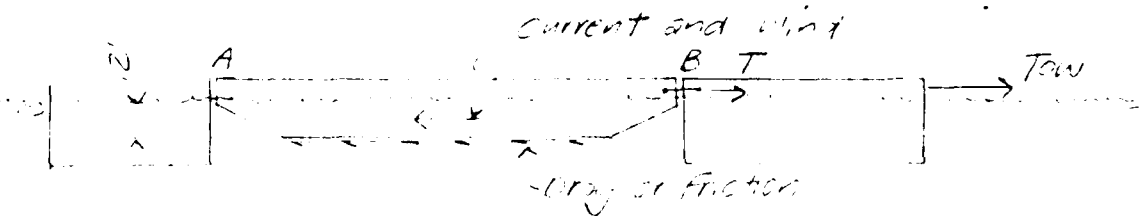
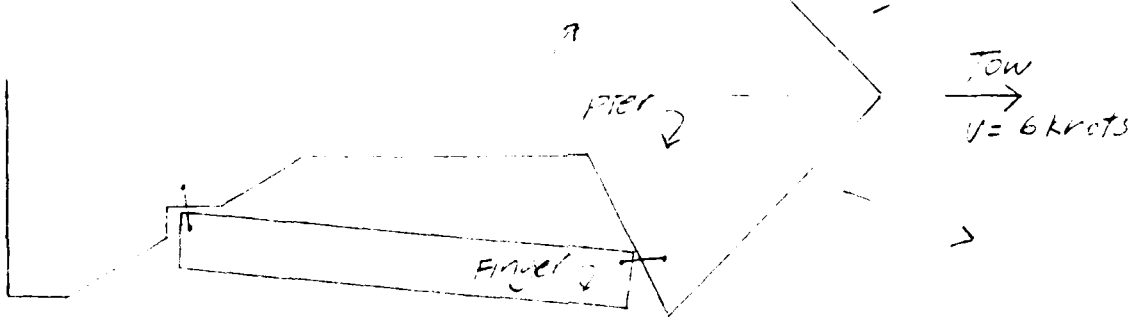
$$\overline{BM}_L = \frac{440800 \times 333}{476000 \times \tan 10^\circ} = 1749'$$

$$\overline{KM} = 9 + 1749 = 1758'$$

$$\overline{GM} = 1758 - 17.9 = 1740'$$

$$\text{PITCH} = \frac{0.52 \times 500'}{\sqrt{1740}} = 6.2 \text{ SEC } \leftarrow$$

Finger Connection loading during Towing



Forces caused by resistance of Finger to motion must be taken by connections @ A and B

RESISTANCE FORCES:  $R = SRF + R_f + R_{air}$

5) FRICTIONAL RESISTANCE  $R_f$  (FRICKE RESISTANCE)

$R_f = fSV$

$f$  = FRICTIONAL COEFFICIENT (FROM TABLE 4, SUP. III, DISTRICT (LOCAL ARCHITECTS, ETCETERA))  
 = 0.0008

$S$  = TOTAL WETTED AREA (SQ. FT.)

=  $27.5 \times 12 + 2 \times 12 \times 12$

=  $50200 \text{ FT}^2$

$V$  = VELOCITY OF SHIP  
 = 6 KNOTS

$\Delta$  = DENSITY OF WATER (under 120' lead)  
 = 1.2 (under 120' lead)  
 $V$  = VOLUME OF DISPLACEMENT (FT<sup>3</sup>)  
 =  $12 \times 500 \times 85 = 480000 \text{ FT}^3$

$R_f = 0.0008 \times 50200 \times 480000$   
 = 19276800 LBS

Finger Resistance to Motion - cont'

b) Wave Making + Eddy Resistance =  $R$

From Fig. 15 Naval Architects' Handbook  
 $V = 6 \text{ knots} = 500'$   
 Interpolate to find  $R_f = 3k$

c) Air Resistance:  $V_{air} = 40 \text{ mph}$

$$F_{air} = C_{yw} \times \frac{\rho}{2} \times V_{air}^2 \times A_s$$

wind @  $90^\circ$  to longitudinal axis

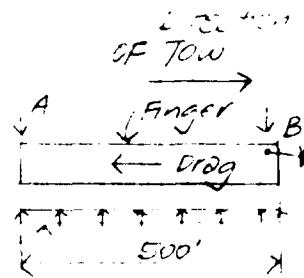
$$V_{air} = 40 \text{ mph} \times 5280 \text{ / mile} / 3600 \text{ / hour} = 59 \text{ ft/sec}$$

$$C_{yw} = 1.0$$

$$\rho = 0.00237 \text{ #-sec}^2 / \text{ft}^4 \text{ @ } 66^\circ \text{F}$$

$$R_{air} = 1.0 \times \frac{1}{2} \times 0.00237 \times 59^2 \times 500 \times 12'$$

$$= 37k$$



d) Current Resistance:  $V_c = 4 \text{ knots @ } 90^\circ$  to longitudinal axis  
 Assume that the current acts @  $40 \text{ mph}$  and acting towing

$$R_c \approx 1.1 A_s V_c^2$$

$$k = 15$$

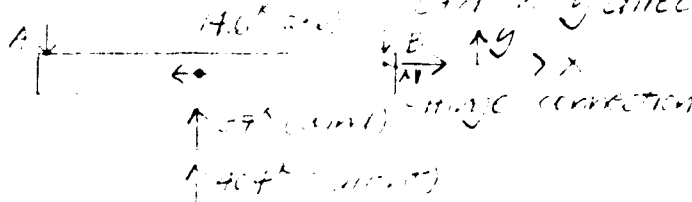
$$R_c = 15 \times 6000 \times 4^2 = 144k$$

$$A_s = 500 \times 12' = 6000 \text{ #}$$

$$V = 17 \text{ ft/s}$$

Final Design

Load Combination II = due in x direction  
 compared to load case I



$$\sum F_y = 0 \Rightarrow F_y = (146 + 146) / 2 = 146k$$

$$\sum F_x = 0 \Rightarrow B_x = 221k$$

$$\text{Tension @ B} = (221^2 + 146^2)^{1/2} = 262k \ll 5280k$$

Appendix - Environmental and Conditions:

Exped. Pier Loading

General Wind Loading

1150g

Exp. Pier  $V_{10} V_{AS}$

Wind  $13 \text{ mph}^2 \rightarrow \text{Ship } 13 \text{ mph} = 10$

Exp. Pier  $\# = \frac{10^2}{10^2} = 1.0$

Exp. Pier  $\# = 1.0$  (Exp. Pier or other materials)

Exp. Pier  $V = 90 \text{ mph} = 100 \text{ mph}$

Exp. Pier  $\# = \text{Exposed Area}$

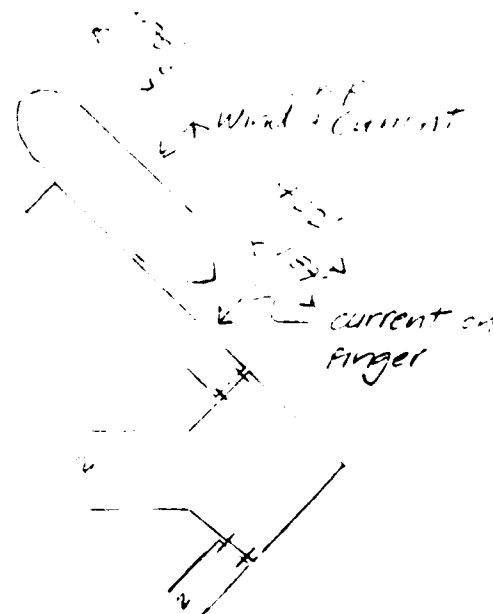
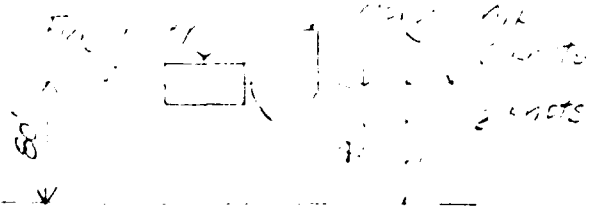
Exp. Pier Wind Loading

$L = 470'$   $H = 90'$   $A_g = 8800 \text{ ft}^2$  (Exp. Pier)

$F_{wind} = \frac{1}{2} \rho V^2 C_d A_g (1.0)$   
 $= 28'$

Exp. Pier Wind Loading

Exp. Pier Wind Loading



Exp. Pier Wind Loading

$L = 470'$   
 $C_d = 1.0$

Design for Transverse current loading:

Determine current forces by:

1. Approximate Method only

For future Navy Pier 470' x 80' x 35'

APPROXIMATE METHOD:

$$F_C = K \cdot V^2 \cdot L$$

$K = 0.5$  for curved surfaces ←

$$F_C = 0.5 \cdot (59)^2 \cdot (59)$$

$$L = 370 \cdot 2' = 740 \text{ ft}$$

$$F_C = 573 \text{ k}$$

$V = \text{avg. current velocity} = 59 \text{ Ft/s}$

Using a current of 1000

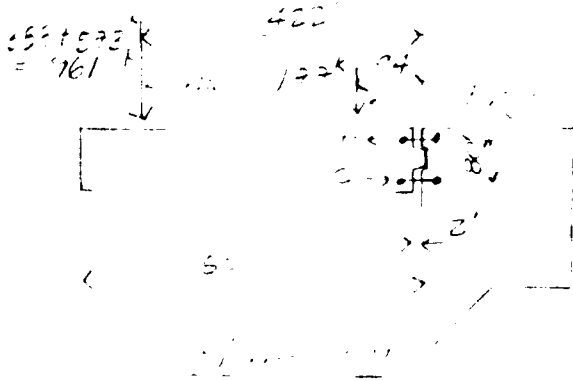
$$F_C = 0.5 \cdot (1000)^2 \cdot (1.5 \cdot (2244) \cdot 59)$$

$$= 1.17 \text{ k}$$

$$L = 15, A_c = 12 \cdot 187 = 2244 \text{ ft}^2$$

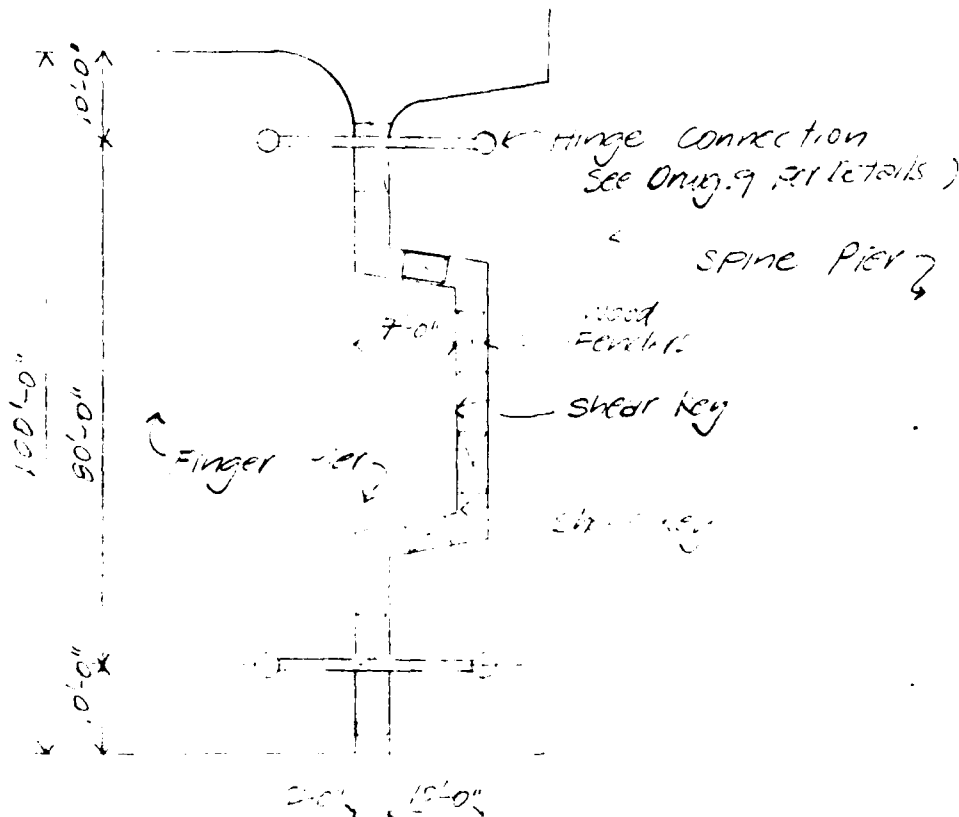
$$L = 59 \text{ ft/s}$$

Most loading will be wind + current:

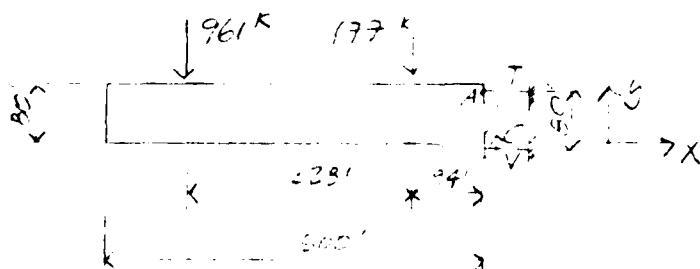


DESIGN ASSUMPTIONS:

1. IF hinged connection
- Take tension and compression couple in link out between finger, SP-C piers.
- No energy is dissipated into water (no drag) All load goes to connections



Load combination I : wind + current @ service conditions governs  
Finger Body Diagram



$$\sum F_y = 0$$

$$V = 961 + 177 = 1138 \text{ k (INTO SHEAR KEY)}$$

$$\sum M_A = 0$$

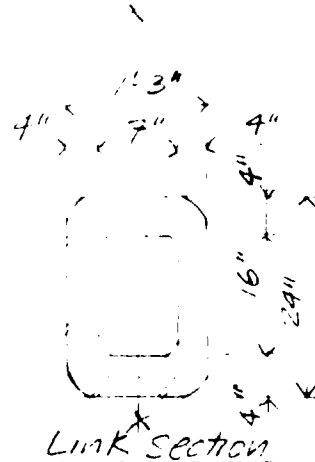
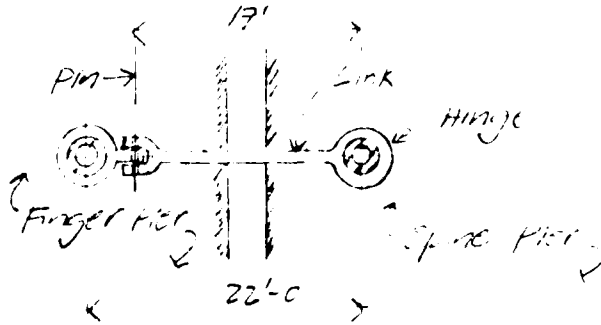
$$C = T = \frac{(961 \times 228 + 177 \times 94)}{80} = 5277 \text{ k}$$

Length of Connection

LINK DESIGN

Compression load: 5280k ← - governs

Cont' Link Design



Hinge Connection Part

Use  $F_y = 50$  ksi steel

Rectangular Box Section

$$A_{req} = \frac{5280}{50} = 105.6 \text{ in}^2$$

Try  $A = 248 \text{ in}^2$  (see section above)

$$I_{xx} = \frac{24 \times 16^3}{12} - \frac{16 \times 12^3}{12} = 6292 \text{ in}^4$$

$$r_g = \sqrt{\frac{6292}{248}} = 5 \text{ in}$$

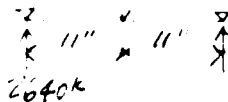
$$KL/r_g = 10 \times 17.12 / 5 = 40 < 300 \text{ satisfy Euler req.}$$

$F_c$  allowable = 26 ksi in compression

$$A_{req} = \frac{5280}{26 \text{ ksi}} = 203 \text{ in}^2 < 248 \text{ in}^2 \text{ OK}$$

Pin Design

$$P = 5280 \text{ k}$$



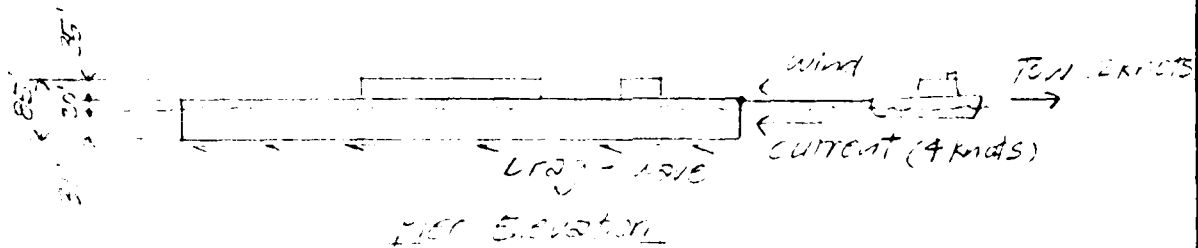
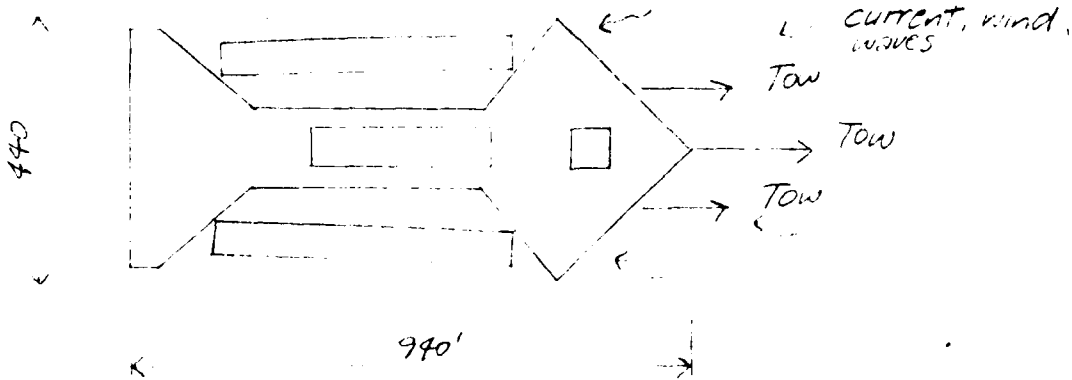
$$\text{REACTION} = 5280 / 2 = 2640 \text{ k}$$

$F_y = 50$  ksi (pin)

$$A_{req} = \frac{2640}{50 \times 0.4} = 132 \text{ in}^2$$

Use 16"  $\phi$  pin,  $A_{pin} = 200 \text{ in}^2$

Total Resistance to Movement =



Environmental Loads

current = 4 knots

wind = 45 mph

velocity = 6 knots

wave making + Eddy

LRA = 1516

total resistance =  $L + W + V + I + K_c$

1) Friction Resistance - MAX. drag will occur when pier is moving at speed of 6 knots without any current or when a 4 knot current plus a towing speed of 4 knots occur.

use LRA at 1 knot speed and 4 knots current see current resistance

$$\text{Reynold's Number} = \frac{VD}{\mu} = \frac{771328 \times 440 \times 992}{8.0 \times 10^{-4}} = 3.9 \times 10^8 > 10^4$$

Cont Resistance to Movement

b) Extrapolate from Fig 35, Naval Architecture by Baxter

$L = 990'$

$V = 6 \text{ knots}$

$R_{REF} \approx 10 \text{ tons} = 20^k$

$R_{TOTAL} = R_{REF} + R_{EFF}$   
 $= 20 + 2 \times 3$   
 $= 26^k$

←  $\Sigma R$

c)  $R_W = C_{yw} \cdot \frac{1}{2} \cdot \rho \cdot V^2 \cdot A_s$

$V_{air} = 40 \text{ mph} = 59 \text{ F/S}$

$C_{yw} = 1.0$

$\rho = 0.00237 \text{ lb/ft}^3 @ 65^\circ\text{F}$

$A_s = 440' \times 31' + 120' \times 35'$   
 $= 17840 \text{ ft}^2$

$R_{W, E.P.} = 1.0 \times \frac{1}{2} \times 0.00237 \times 59^2 \times 17840 = 74^k$

$R_{W, TOTAL} = R_{W, E.P.} + R_{W, FF}$   
 $= 74 + 0$   
 $= 74^k$

←  $\Sigma R_W$

d) CURRENT RESISTANCE = USE 6 knots (relative velocity)

$R_{CEP} = R_{CEP} = C_D A_p \cdot \frac{1}{2} \rho V^2$   
 $= 1.2 \times 21080 \times 199 \times \frac{7.7^2}{2}$   
 $= 1490^k$

$A_p = 440' \times 31' + 240' \times 31'$   
 $= 21080 \text{ ft}^2$   
 $V = 7.7 \text{ F/S}$   
 $C_D = 1.2 \text{ (rectangle)}$

$R_{C, TOTAL} = R_{CEP} + R_{CFP}$   
 $= 1490^k$

←  $\Sigma R_C$

TOTAL RESISTANCE =  $26^k + 74^k + 1490^k$   
 $= 1590^k$

←  $\Sigma R_T$

EFFECTIVE HORSE-POWER required

$E.H.P. = \frac{R_T \cdot V \cdot 1.48}{33000}$

$V = 6 \text{ knots}$   
 $\downarrow$  factor

$= \frac{159000 \times 6 \times 1.48}{33000} = 29293 \times 1.2 = 35152 \text{ Emp.}$

USE 3-1200 H.P. TAG COSTS ← # TAG COSTS req'd.  
OR 2-18000 H.P. TAG COST

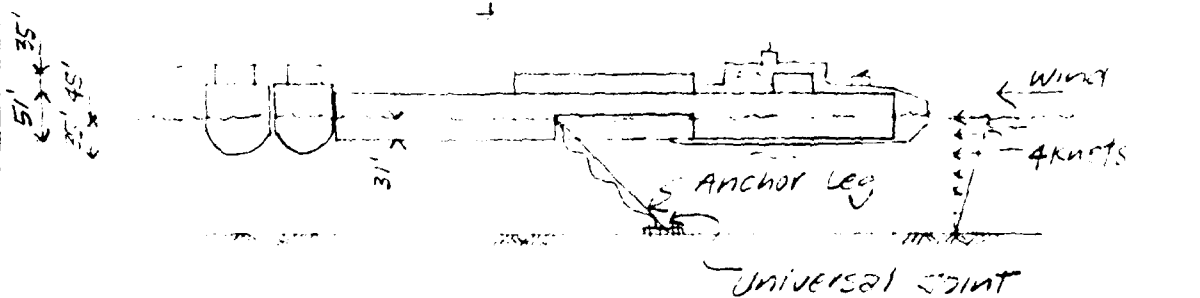
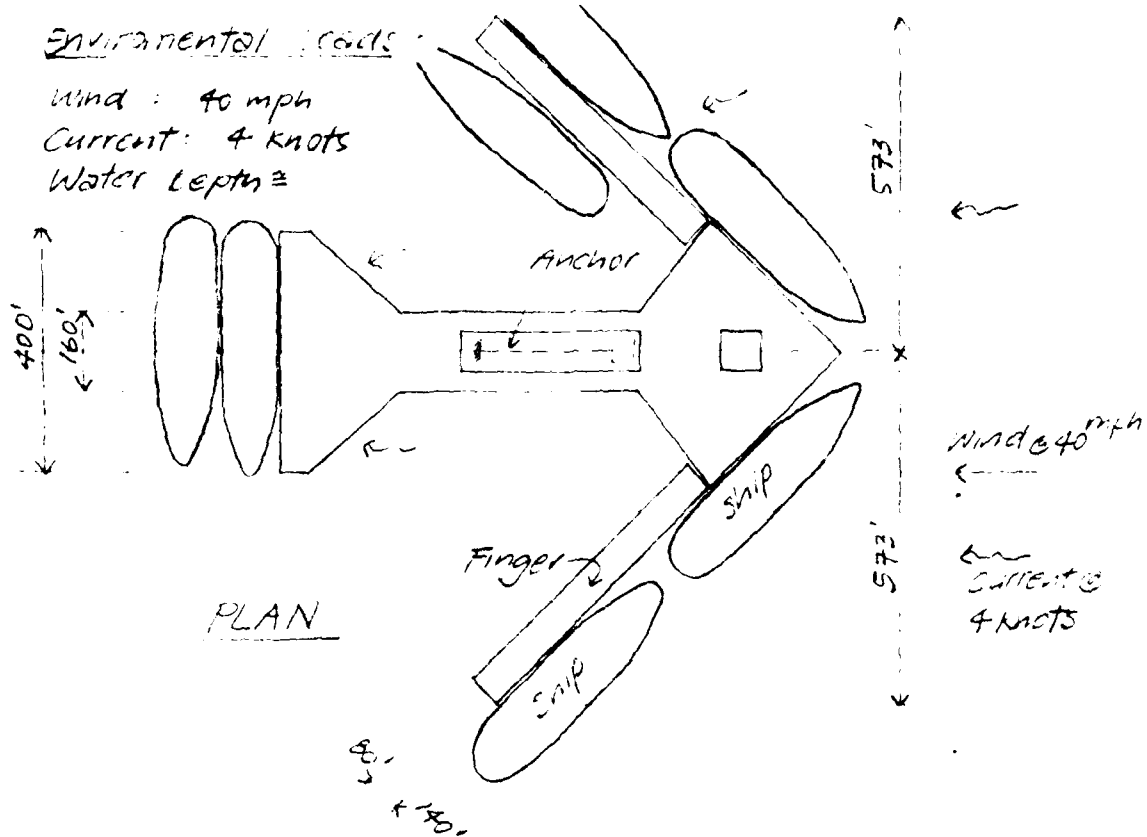
**TYLON**  
INTERNATIONAL  
STRUCTURAL ENGINEERING  
315 Bay St. San Francisco, Ca 94133

PROJECT: Navy Piers  
ITEM: Expeditionary Pier  
DESIGN: Anchor Leg  
DATE: 11/92 KM

SHEET:  
A-37  
OF \_\_\_\_\_  
REVISION:

Environmental Loads

Wind: 40 mph  
Current: 4 knots  
Water Depth =



Expeditionary Pier (HEIGHT = 31' (400<sup>sq</sup> Uniform Live Load))  
Future Navy Ship = 470' x 140' x 30'  
HEIGHT = ASSUME 35'  
Assume 2 ships nested @ one time when current  
(4 knots) and wind (40 mph)

Wind Load :

$$F_w = C_{gw} \cdot C_e \cdot P_w V_w^2 A_s$$

$$C_{gw} = 1.0 \quad \text{wind @ } 90^\circ$$

$$F_w = 0.00237 \frac{\text{lb} \cdot \text{sec}^2}{\text{ft}^4} @ 68^\circ\text{F}$$

$$V_w = 40 \text{ mph} = 59 \text{ F/s}$$

$$A_s \text{ total} = A_s \text{ ship} + A_s \text{ pier}$$

$$= 2 \times 470' \times 45' \cos 45^\circ + 470' \times 45' + 17340' \text{ FROM PREVIOUS CALCS}$$

$$= 128722 \text{ ft}^2$$

$$F_w = 10 \times 1/2 \times 0.00237 \cdot 59^2 \times 128722 \text{ lb}$$

$$= 531 \text{ K}$$

Current Load :

APPROXIMATE Method :

$$F_c = K A_s V_c^2$$

$$= 1.0 \times 107321 \times 5.1^2$$

$$= 2820 \text{ K}$$

$$A_s \text{ total} = A_s \text{ ship} + A_s \text{ pier}$$

$$= 6 \times 470' \times 35' \times \cos 45^\circ$$

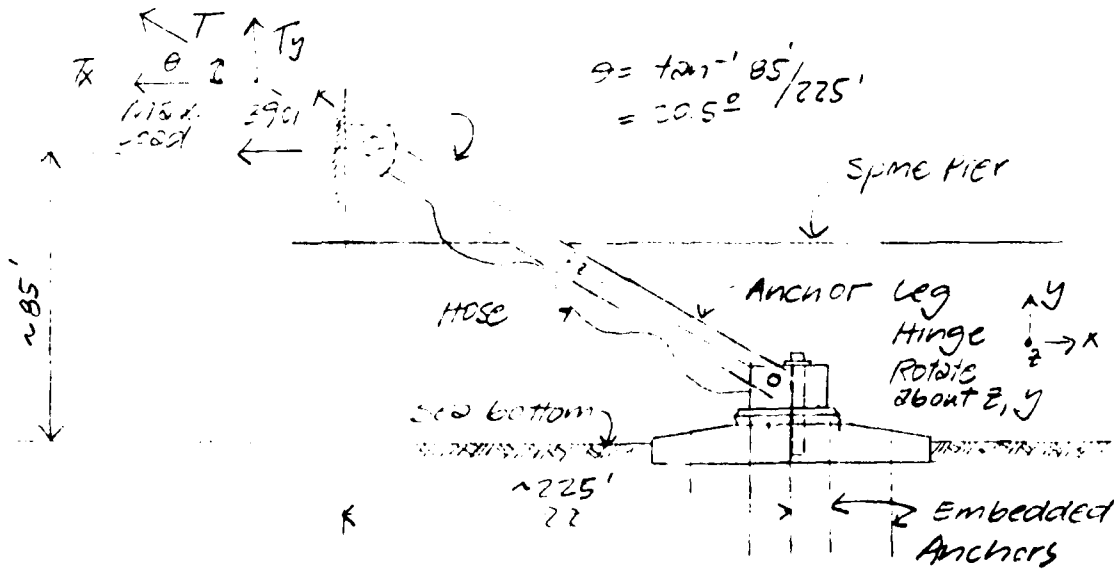
$$+ 470' \times 35' + 440' \times 31' + 240' \times 31'$$

$$= 107321 \text{ ft}^2$$

$$V_c = 4 \text{ knots} = 5.1 \text{ F/s}$$

$$\text{TOTAL LOAD} = 531 + 2820$$

$$= 3351 \text{ K}$$



Cont' Anchor leg loading:

$$T_y = T_x \tan \theta$$

$$= 3901 \times \tan 20.6^\circ$$

$$= 1474k$$

$$T = \sqrt{T_x^2 + T_y^2} = \sqrt{3901^2 + 1474^2}$$

$$= 4170^k$$

leg design  $F_y = 50 \text{ ksi}$

1) Tension:  $A_{req} = \frac{4170}{50 \times 0.6}$   
 $= 139 \text{ in}^2$   
 USE avg.  $A = 200 \text{ in}^2$

2) Bending:

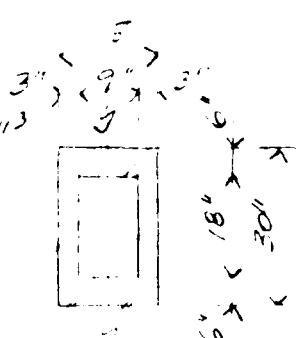
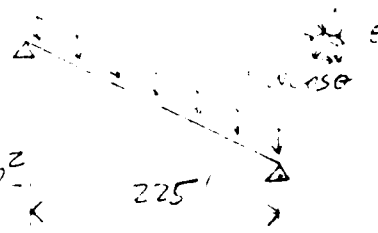
2-in weight:  $200 \times .49$   
 $= 98 \text{ k}$   
 $= 2.65 \text{ k/ft}$

$M_{max} = \frac{wL^2}{8} = \frac{2.65 \times \cos 20.6^\circ \times 240^2}{8}$   
 $= 4601 \text{ k-ft}$

$S_{x-x \text{ req}} = \frac{4601 \times 12}{1.5 \times 0.66}$   
 $= 1672 \text{ in}^3$

$S_{15 \times 30} = \frac{15 \times 30^3}{12} = 1950 \text{ in}^3 > 1672 \text{ in}^3$   
 OK.

$A_{prov} = 288 \text{ in}^2 > 139 \text{ in}^2 \text{ req.}$  ✓

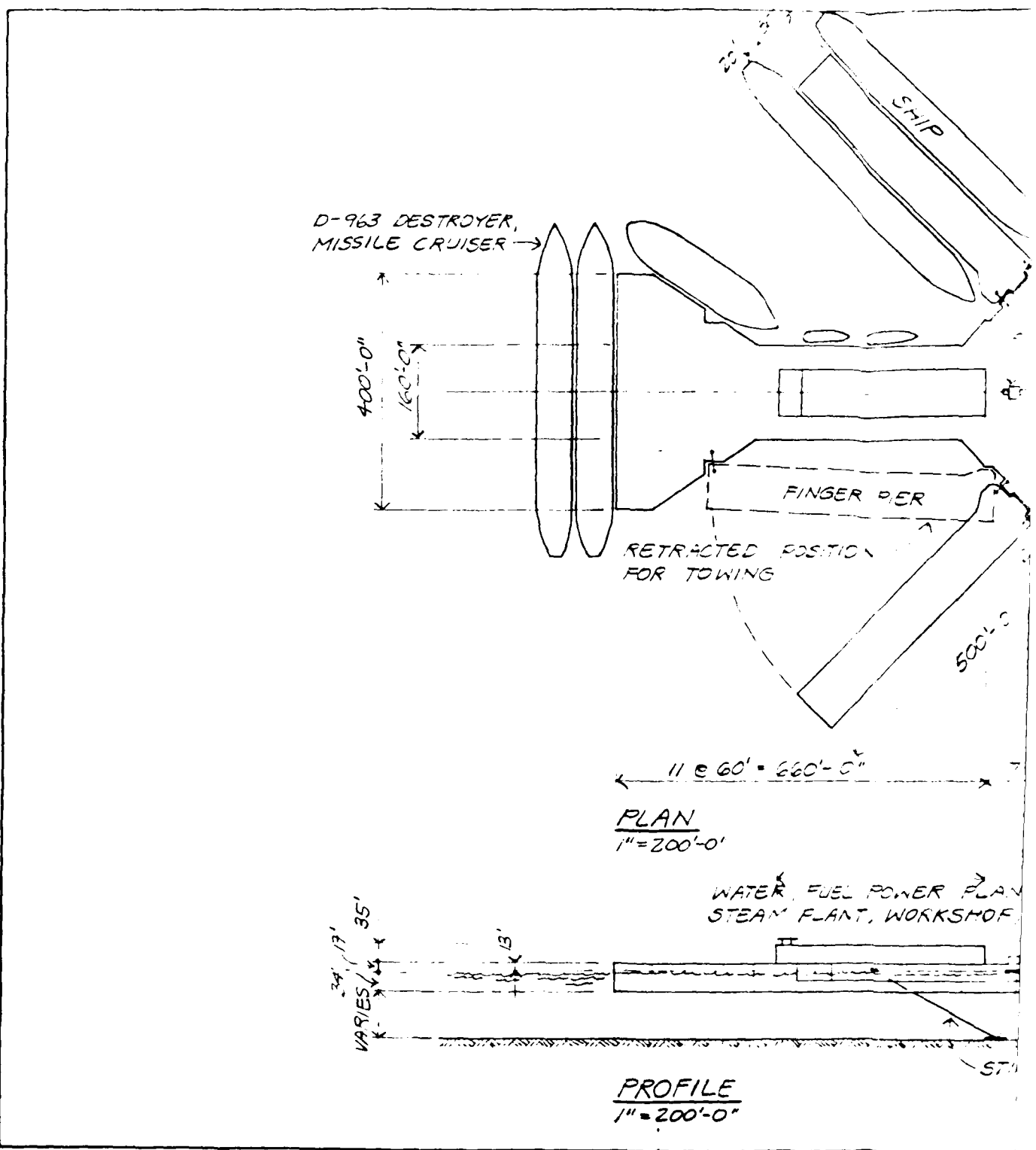


leg X-Section @ midspan

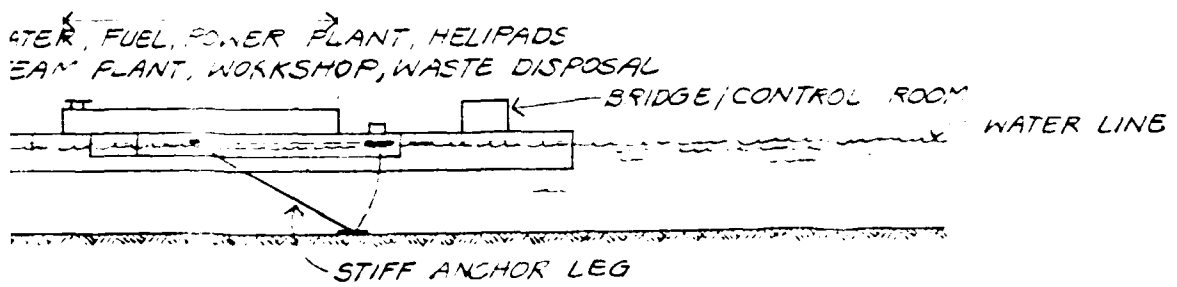
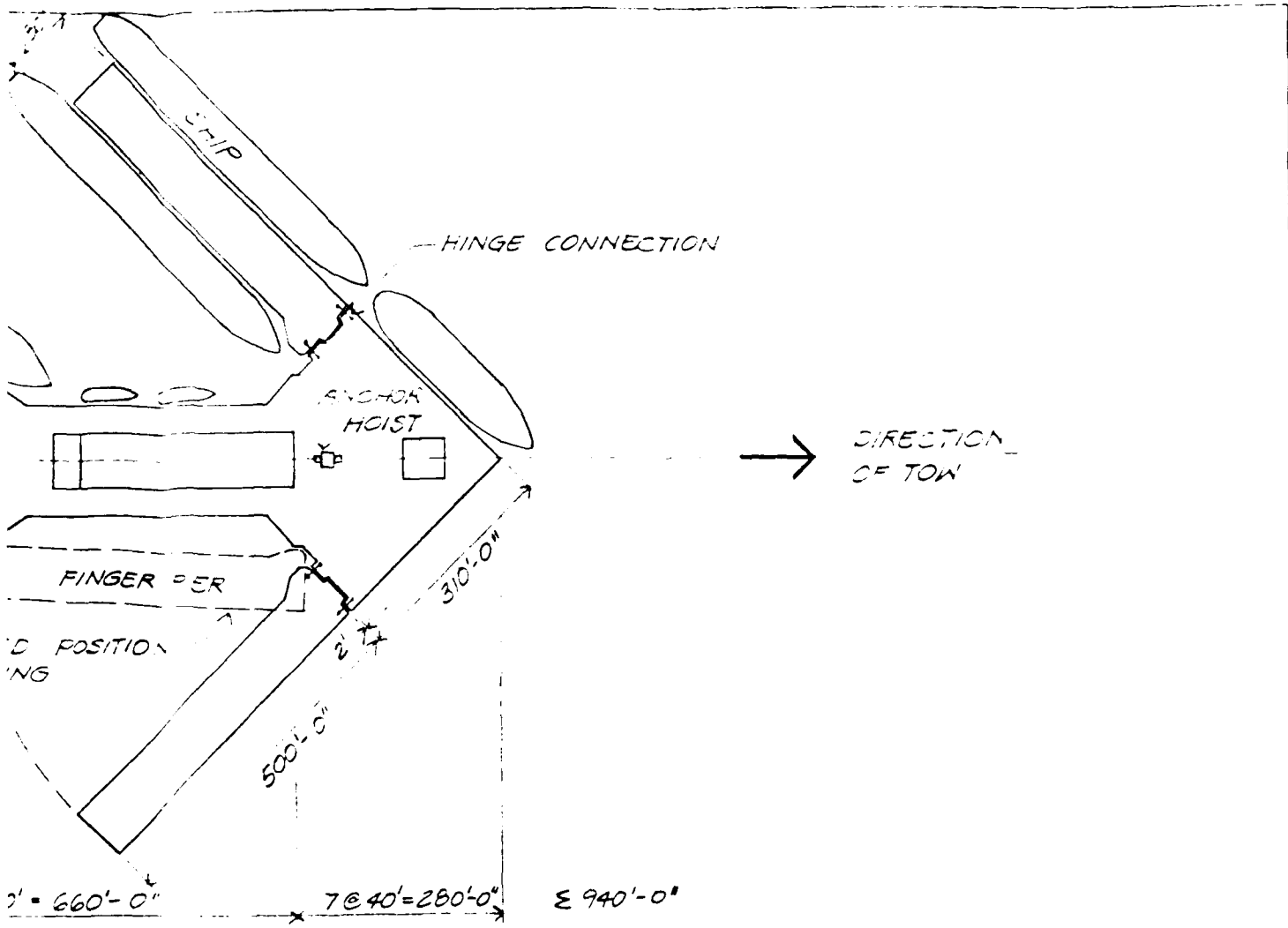
Pin Design:

shear  $V = 4601/2 = 2300k$   
 $A_{req} = \frac{2300}{0.4} = 115 \text{ in}^2$   
 USE 15"  $\phi$  PIN

PROJECT NO. \_\_\_\_\_ DESIGN \_\_\_\_\_ DRAFTING \_\_\_\_\_



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		100%	Dec 82



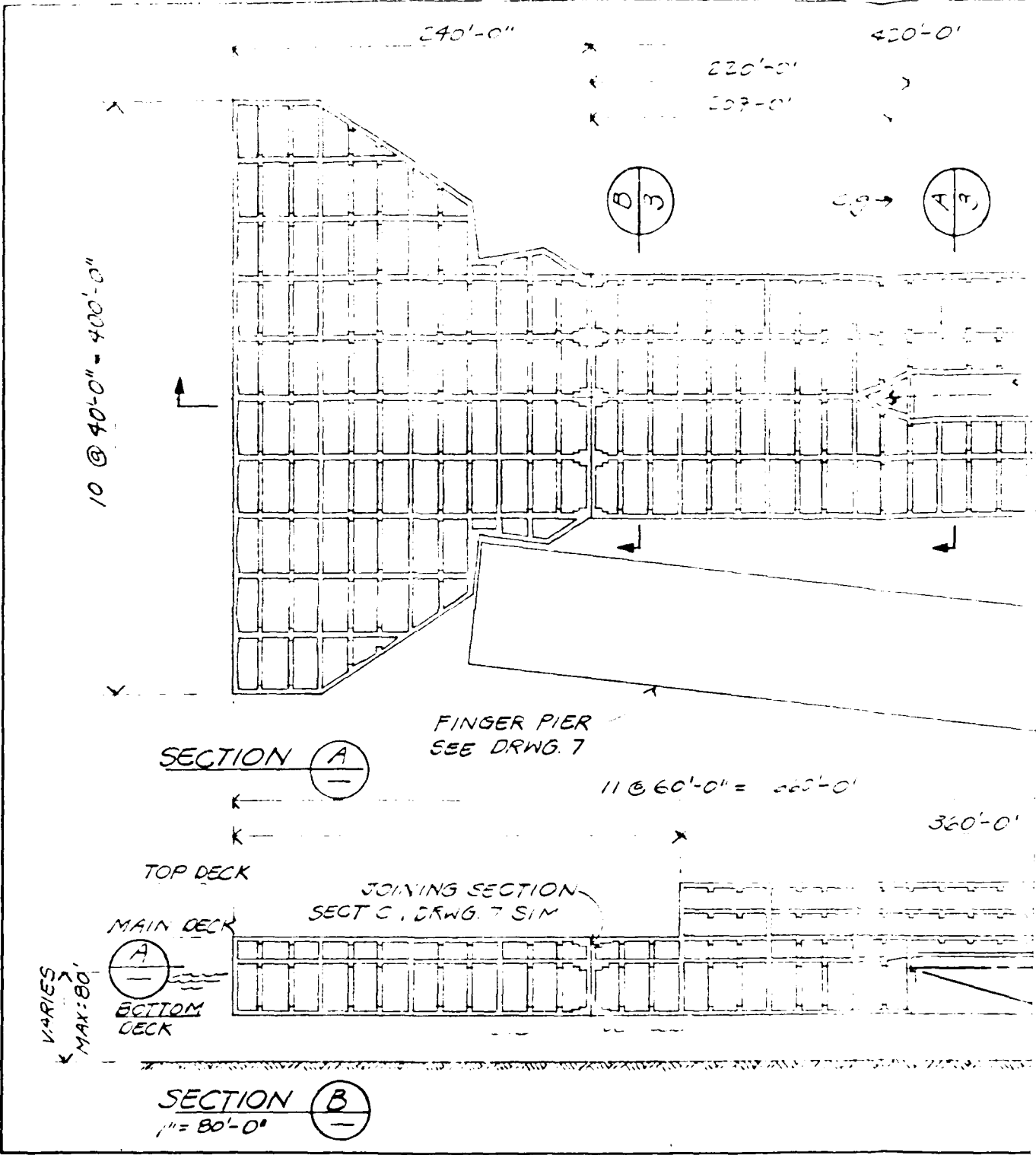
NO	REVISION	DATE

Issued For	Date	By	SHEET TITLE <b>SCHEME A</b>	SHEET NO <b>1</b>
100%	Dec, 82	RM		
PROJECT			<b>EXPEDITIONARY PIER</b>	

DRAFTING

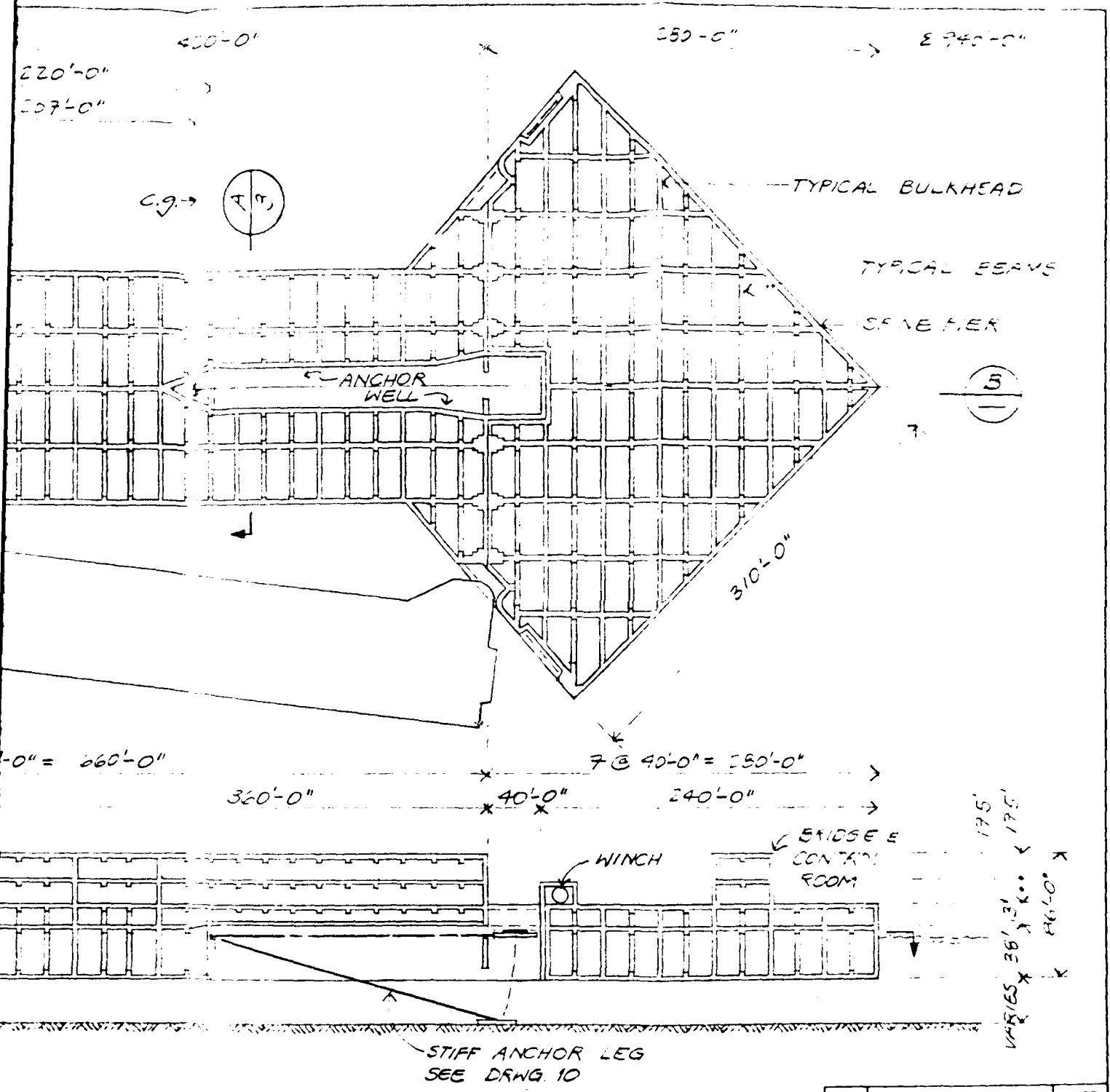
DESIGN

PROJECT NO



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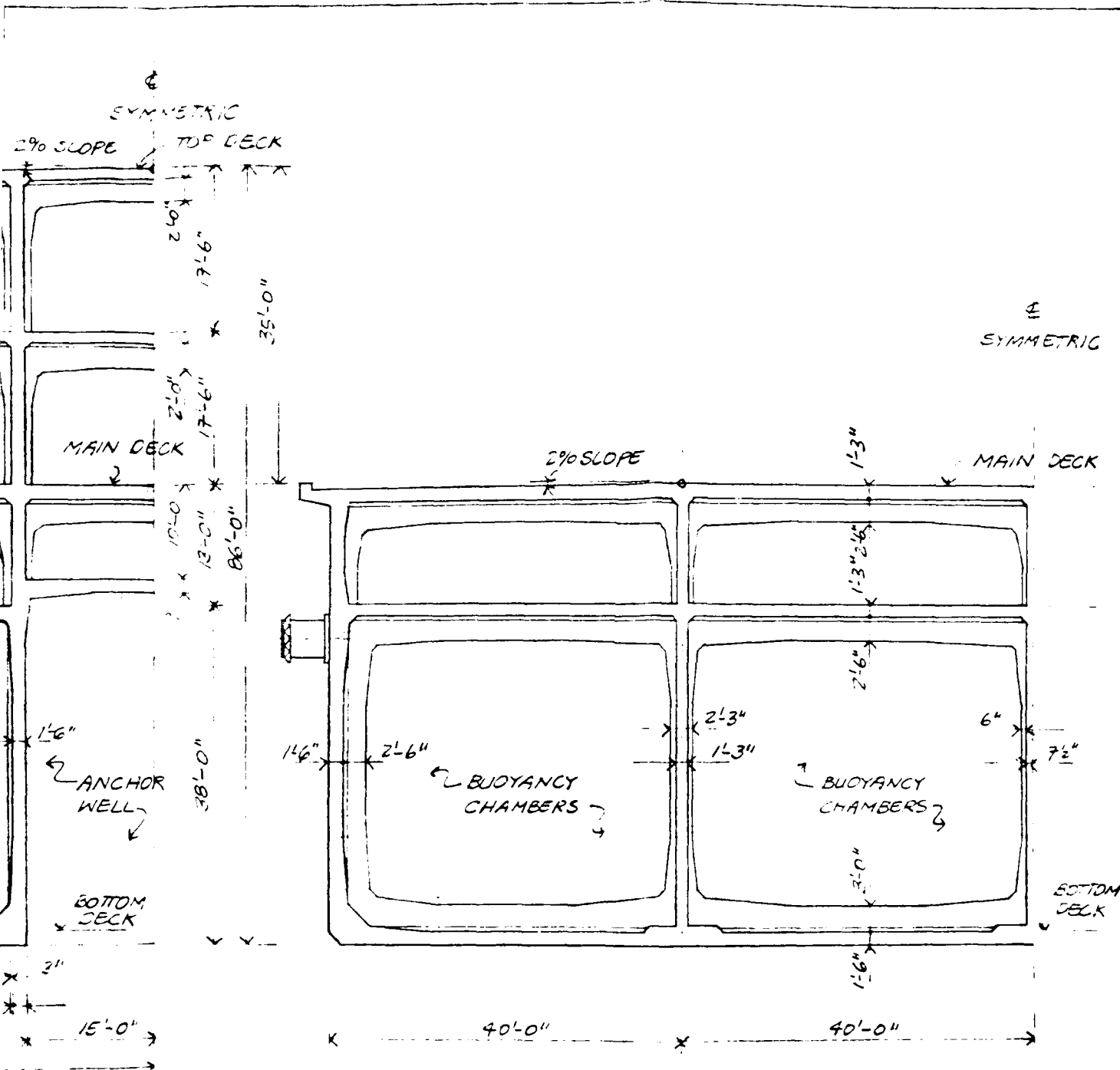
NO	REVISION	DATE

Issued For	Date	By
100%	JUL 82	RM

SHEET TITLE	SPINE LONGITUDINAL SECTIONS - SCHEME A
PROJECT	EXPEDITIONARY PIER

SHEET NO	2
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SECTION B  
 $\frac{3}{4}'' = 1'-0''$  2

NO.	REVISION	DATE
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Issued For	Date	By
.00%	SEP 53	A.M.

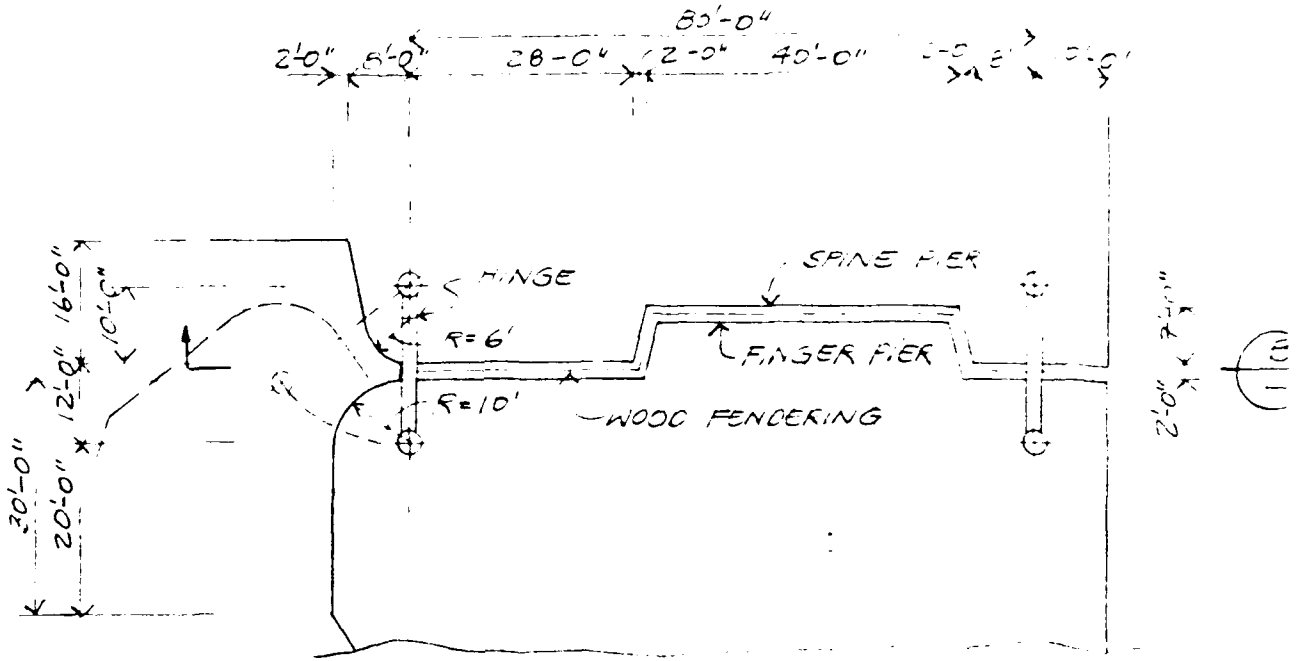
SHEET TITLE	SPINE PIER CROSS SECTION-SCHEME A
PROJECT	EXPEDITIONARY PIER

SHEET NO.	3
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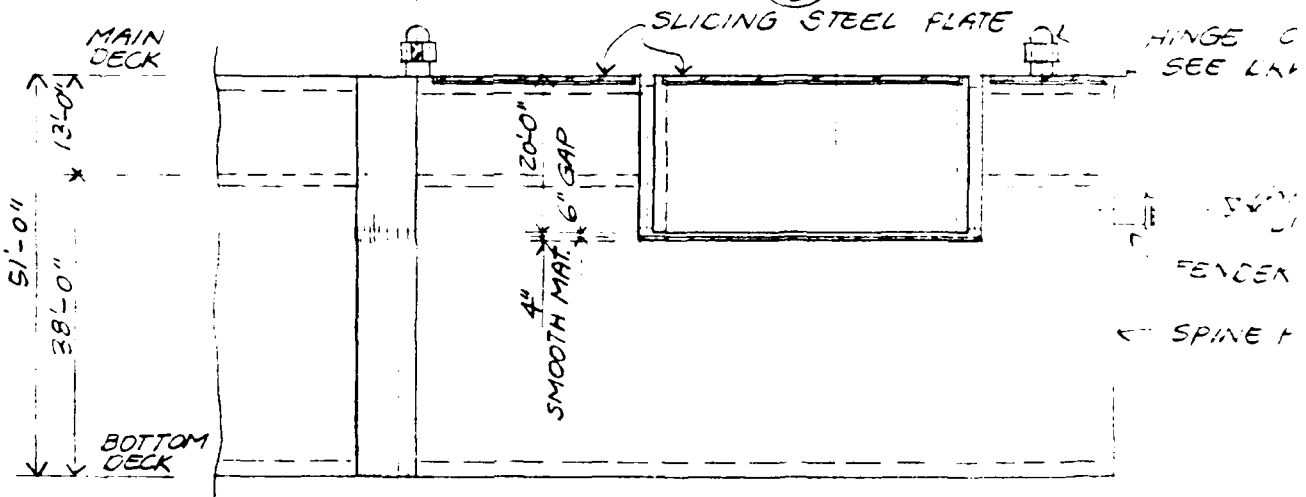
DRAFTING

DESIGN

PROJECT NO.



DETAIL (A) 2



SECTION (B) 1" = 20'-0"

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Issued For

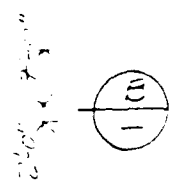
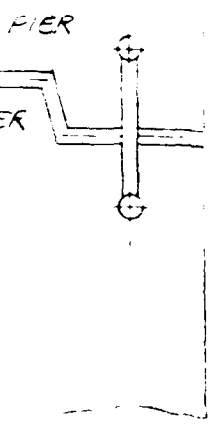
100%

Date

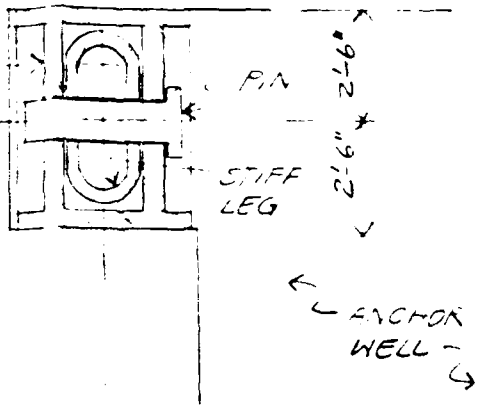
Jan. 6

2'-0" B' → 10' 6" →

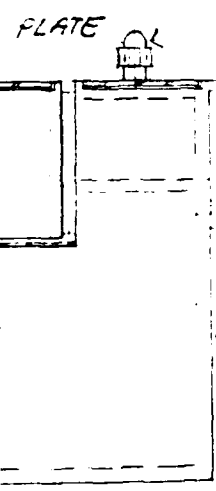
2'-6" →



BEARING AND TENSION PL.



SECTION C  
10



HINGE CONNECTION SEE DRAWG 9

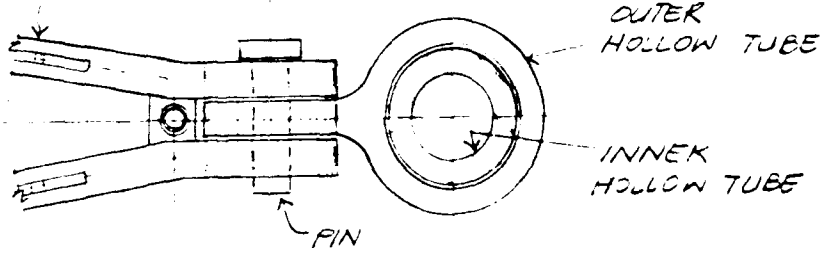
WATER LINE

FENCE

SPINE PIER

STIFF LEG ANCHOR

← CENTERLINE OF ANCHOR HINGE



← CENTERLINE OF HOSE OPENING

SECTION D  
4" = 1'-0"

NO	REVISION	DATE

Issued For	Date	By
100%	Jan. 83	KM

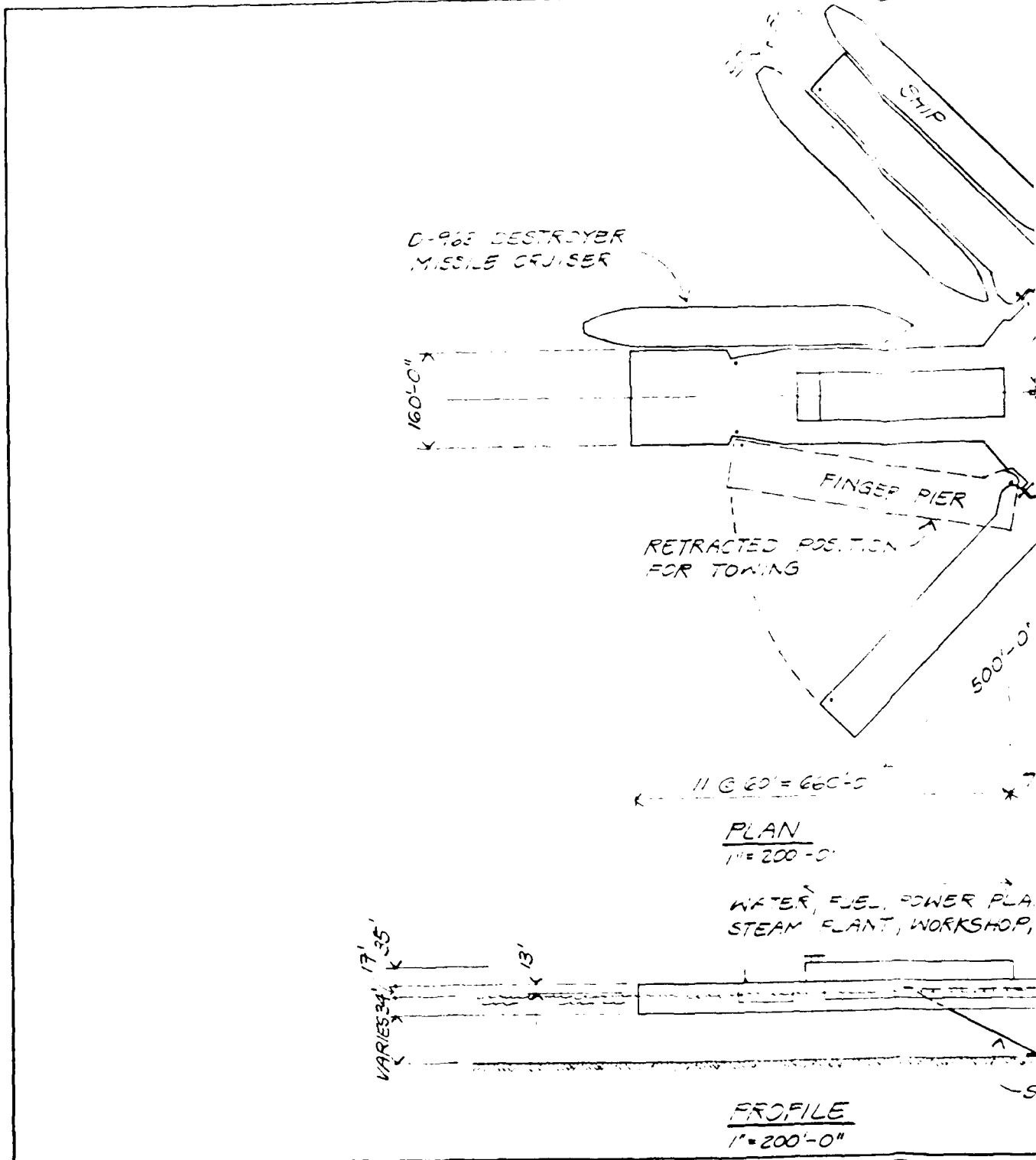
SHEET TITLE	SPINE PIER DETAILS
PROJECT	EXPEDITIONARY PIER

SHEET NO	4
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DRAFTING

DESIGN

PROJECT NO



**TYLIN**

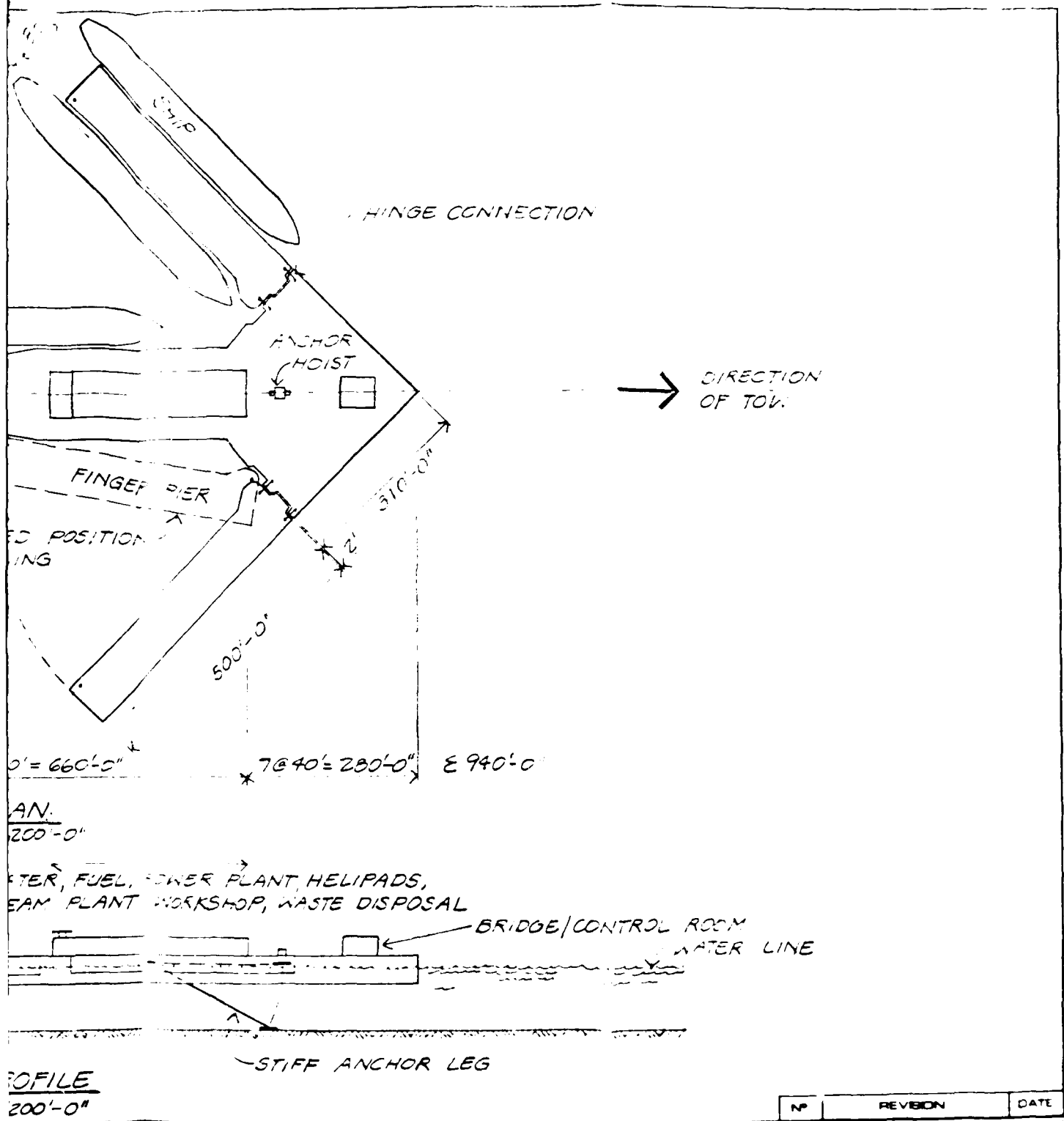
INTERNATIONAL  
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Issued For

50%

Date

Jan 1971



PLAN

200'-0"

WATER, FUEL, POWER PLANT, HELIPADS,  
TEAM PLANT WORKSHOP, WASTE DISPOSAL

BRIDGE/CONTROL ROOM  
WATER LINE

PROFILE

200'-0"

STIFF ANCHOR LEG

N°	REVISION	DATE
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Issued For	Date	By	SHEET TITLE	SHEET N°
100%P	Jan, 83	RM	SCHEME B	5
			PROJECT	
			EXPEDITIONARY P.E.R	

DRAFTING

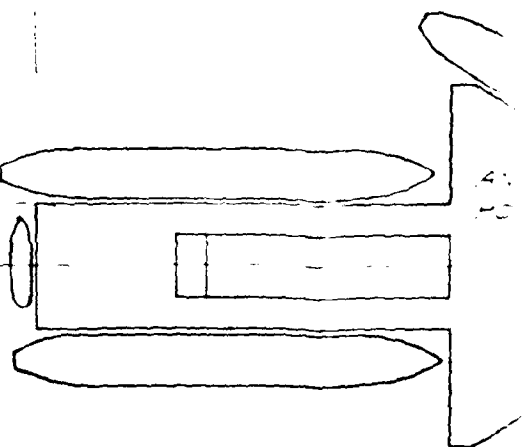
DESIGN

PROJECT NO

D-963 DESTROYER,  
MISSILE CRUISER

160'-0"

540'-0"



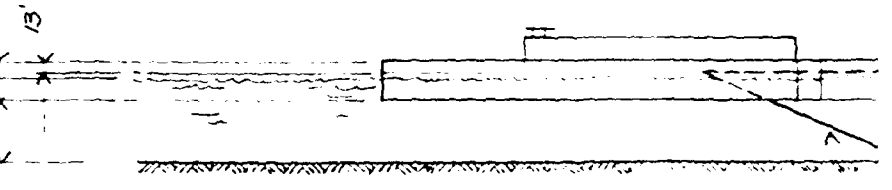
113.60 = 360'-0"

940'-0"

PLAN

WATER FUEL, POWER  
HELIPAD, STEAM PLANT  
WORKSHOP, WASTE DIS

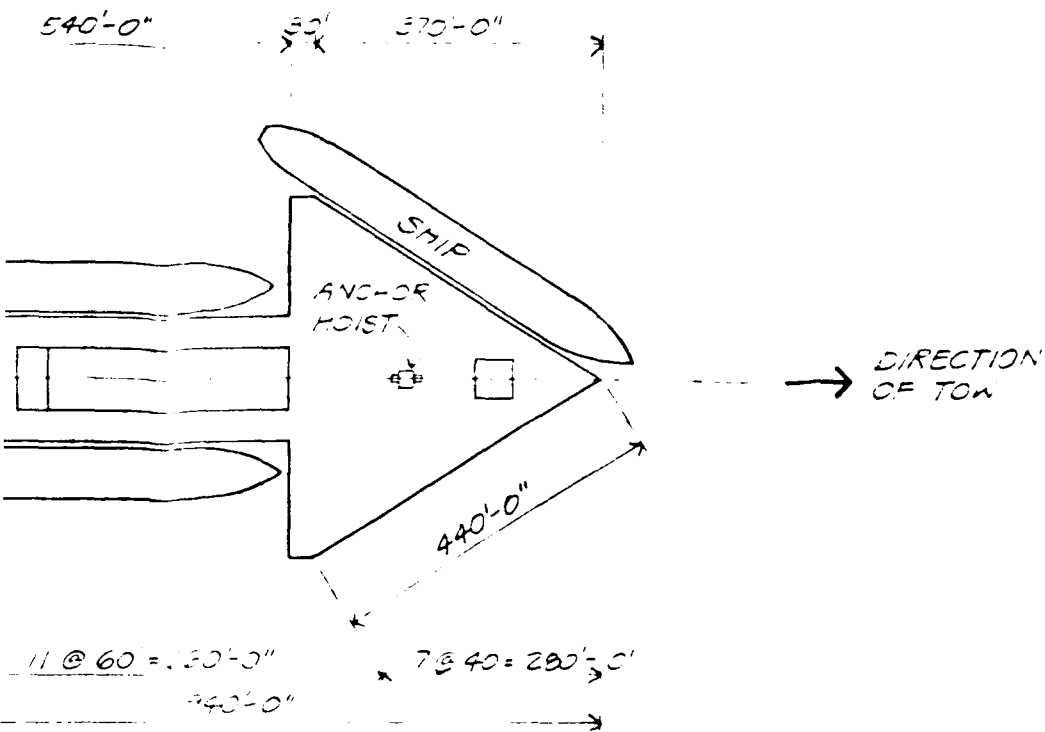
VARIES 34'  
17'  
13'



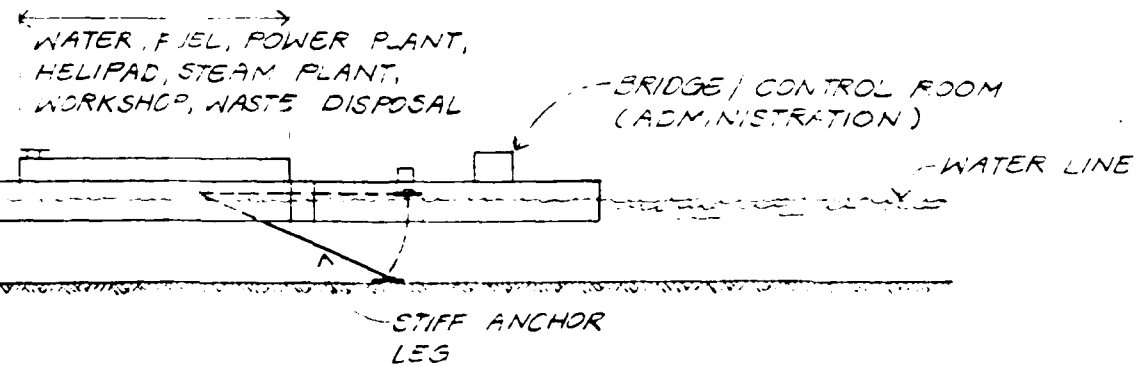
PROFILE

1"=200'-0"

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		100%	DEC 90



PLAN



PROFILE  
1" = 200'-0"

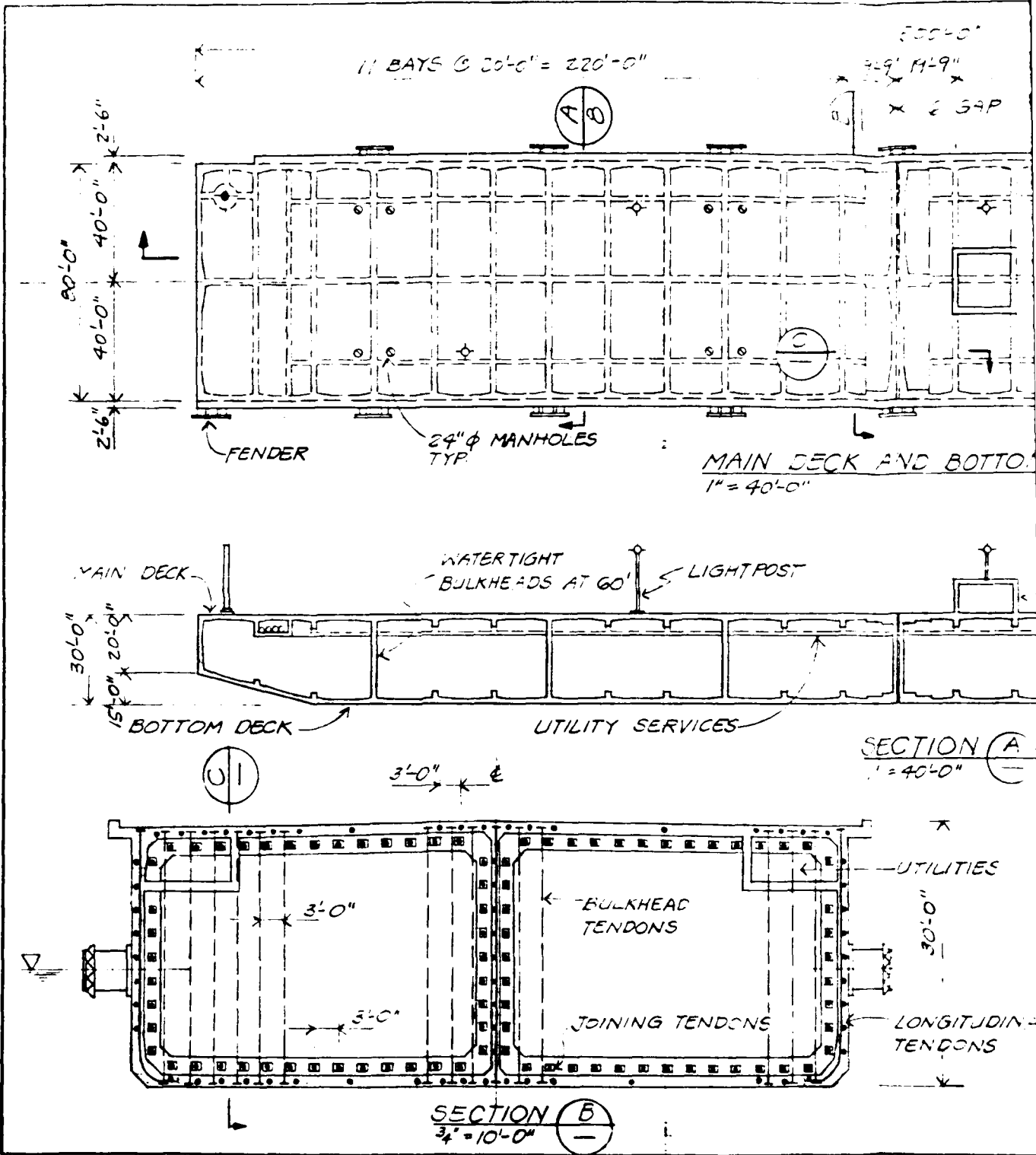
NO	REVISION	DATE
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Issued For	Date	By	SHEET TITLE	SHEET NO
100%	Dec, 82	KM	SCHEME C	6
			PROJECT:	
			EXPEDITIONARY PIER	

DRAFTING

DESIGN

PROJECT NO



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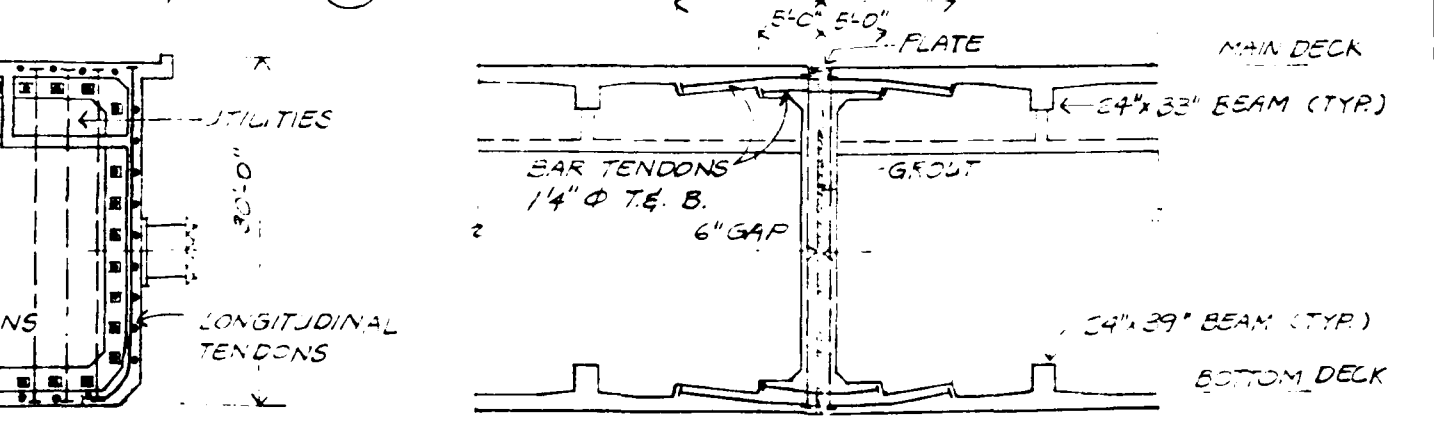
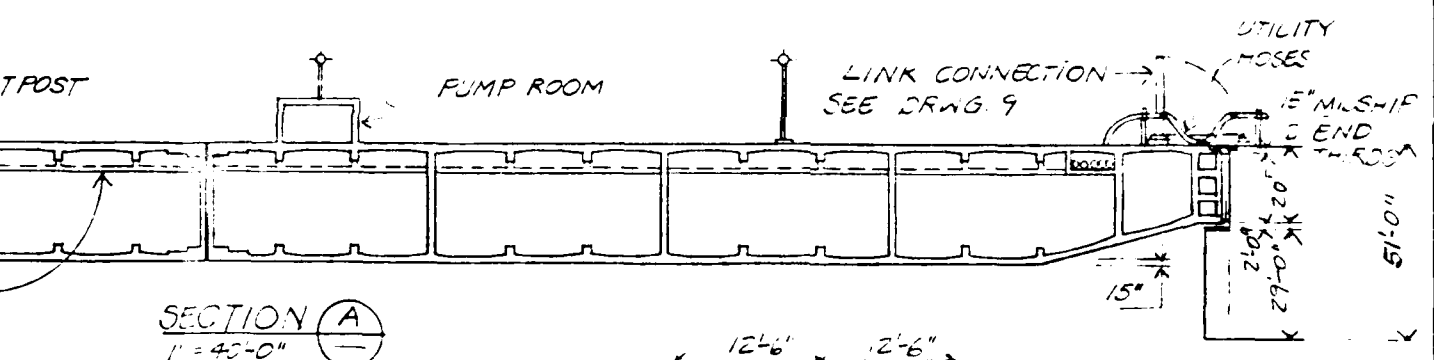
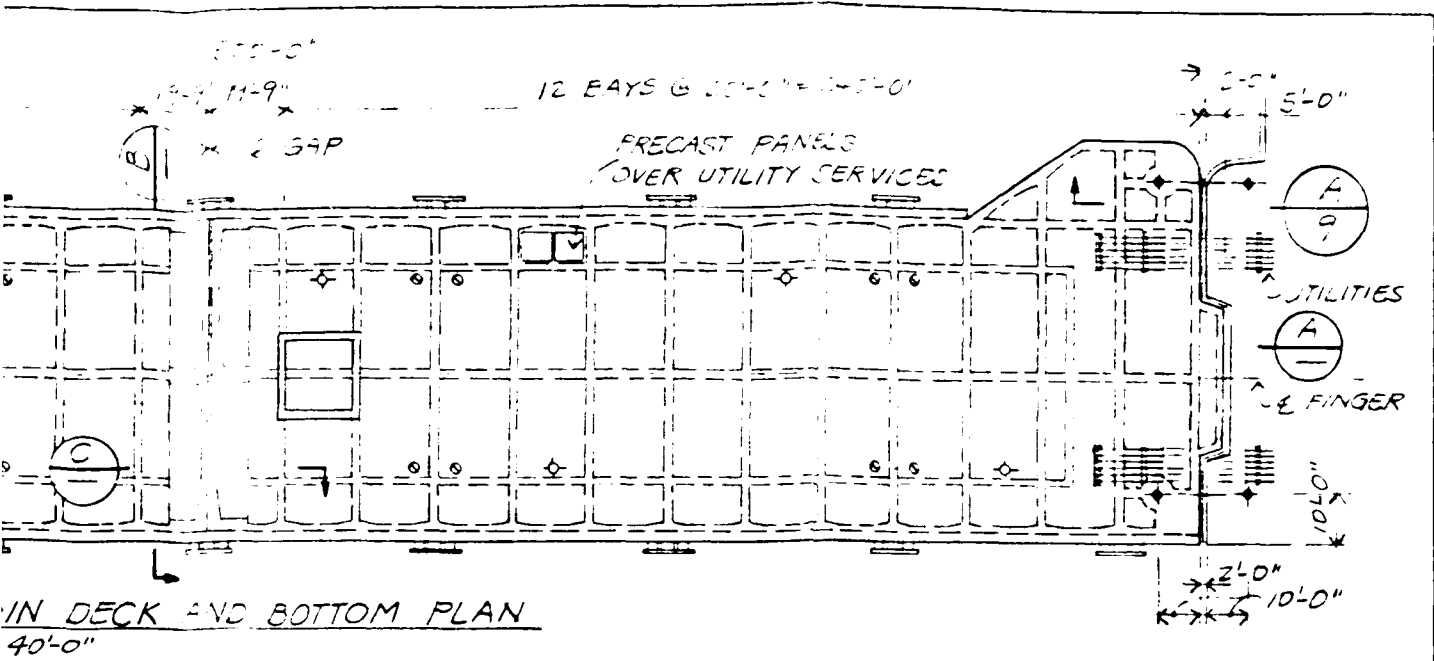
315 Bay St., San Francisco, Ca 94133 Tel (415) 992-1060

Issued For

100%

Date

Dec. 82



Issued For	Date	By	SHEET TITLE	SHEET NO
100%	Dec. 82	RM	FINGER PIER SECTIONS & DETAILS	7
			PROJECT	
			EXPEDITIONARY PIER	

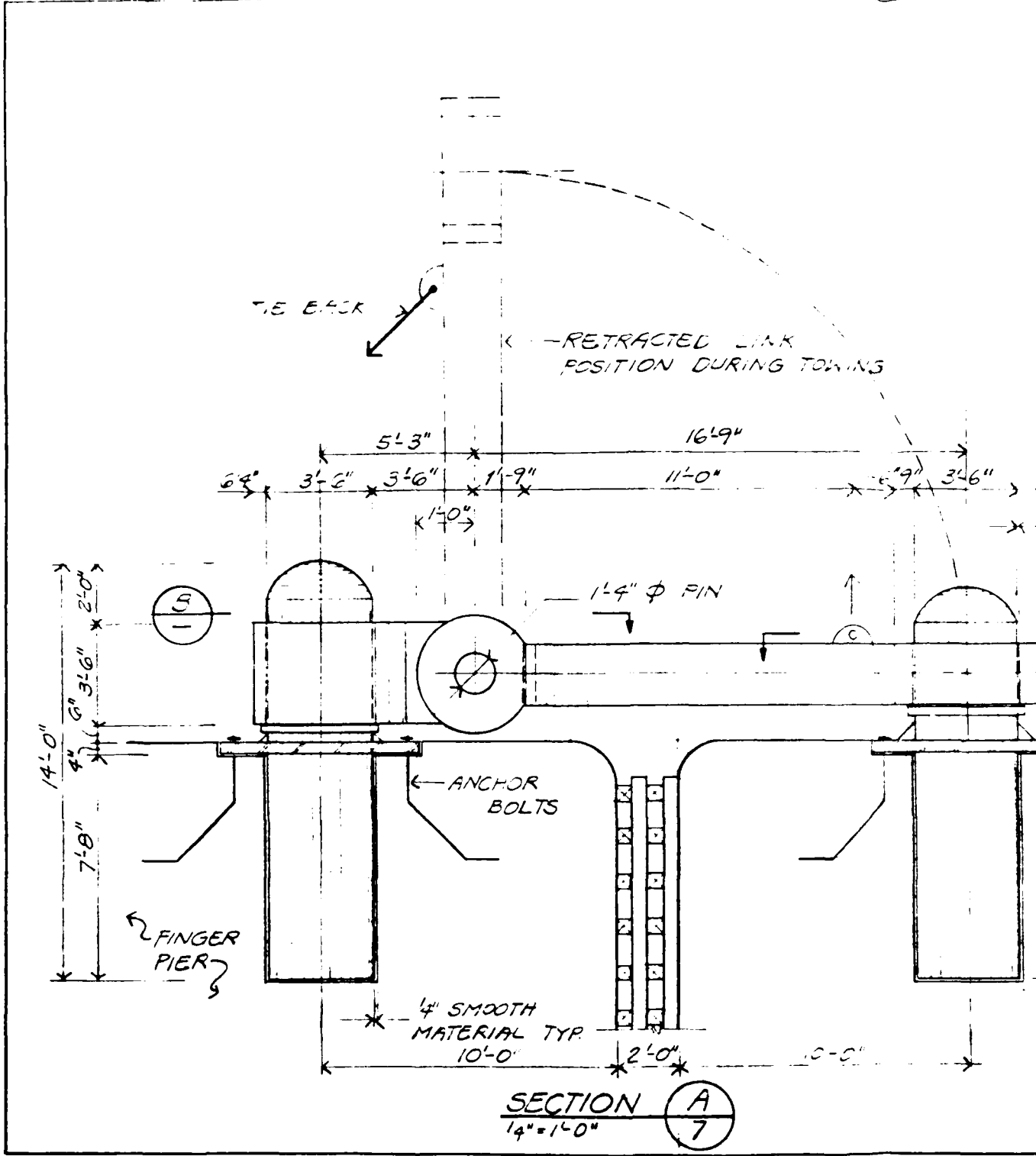




DRAFTING

DESIGN

PROJECT NO



**TYLON**

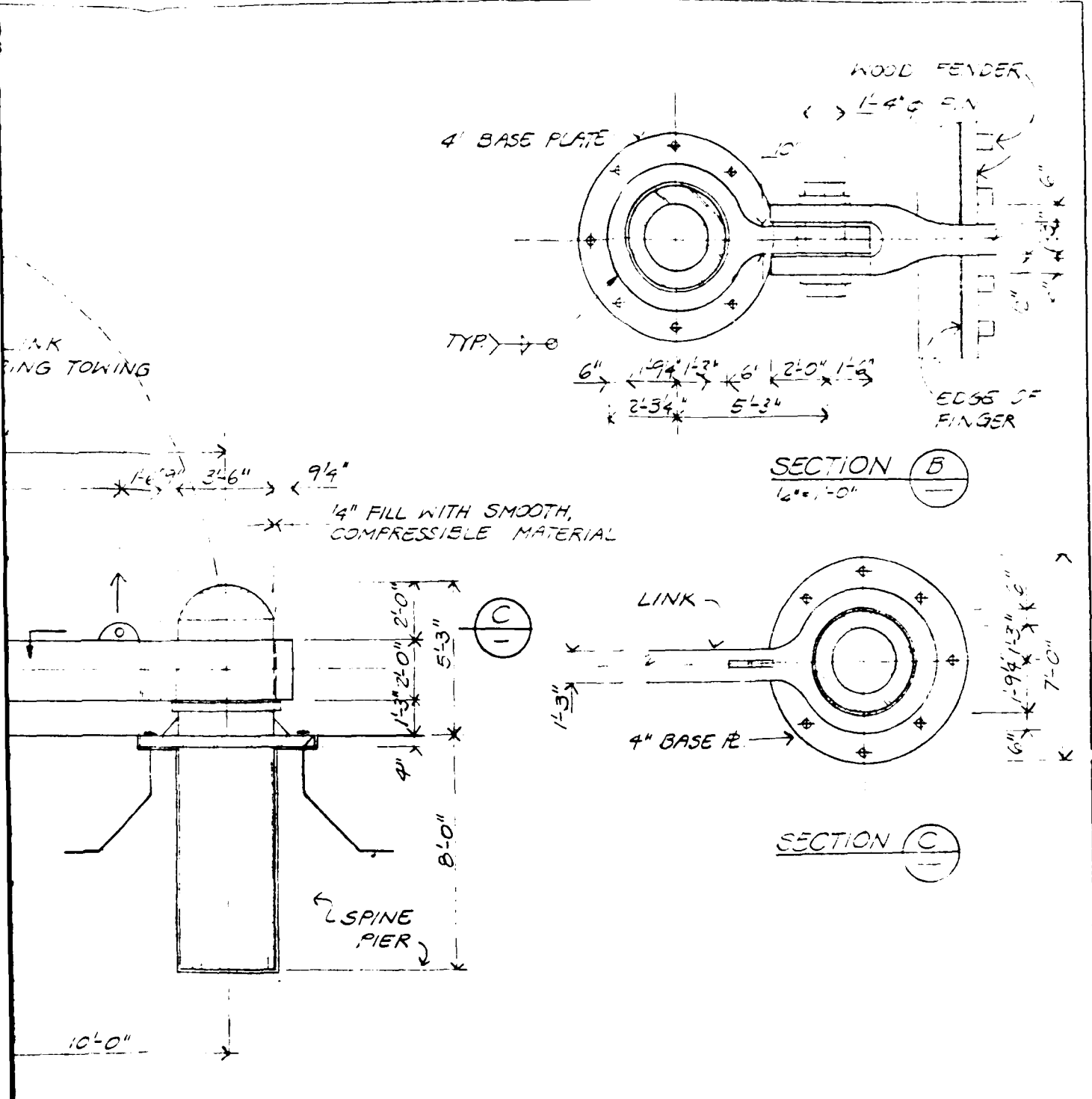
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315 Bay St., San Francisco, Ca 94133 Tel (415) 682-1050

Issued For

CO'S

Date

2/27/55



Revised For			Date	By	SHEET TITLE	SHEET NO.
100%			8/1/55	KM	HINGE CONNECTION FINGER/SPINE	1
PROJECT					EXPEDITIONARY PIER	

NO.	REVISION	DATE

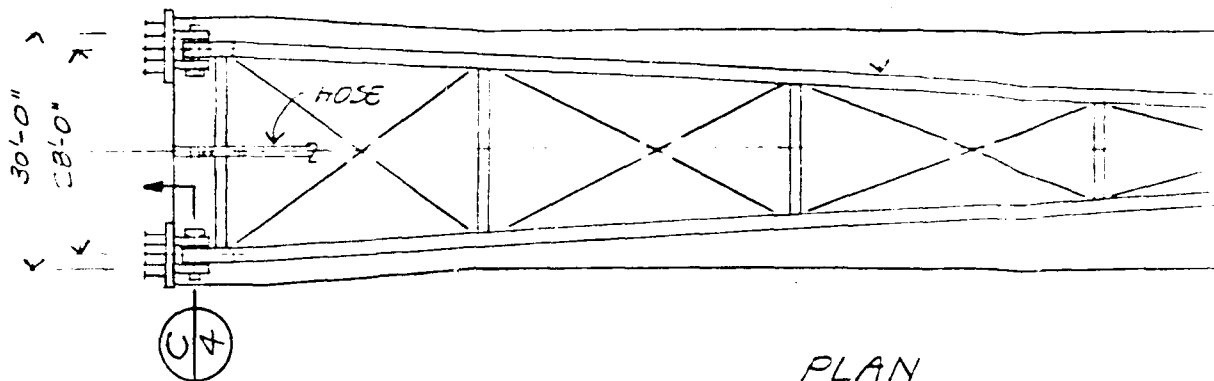
DRAFTING

DESIGN

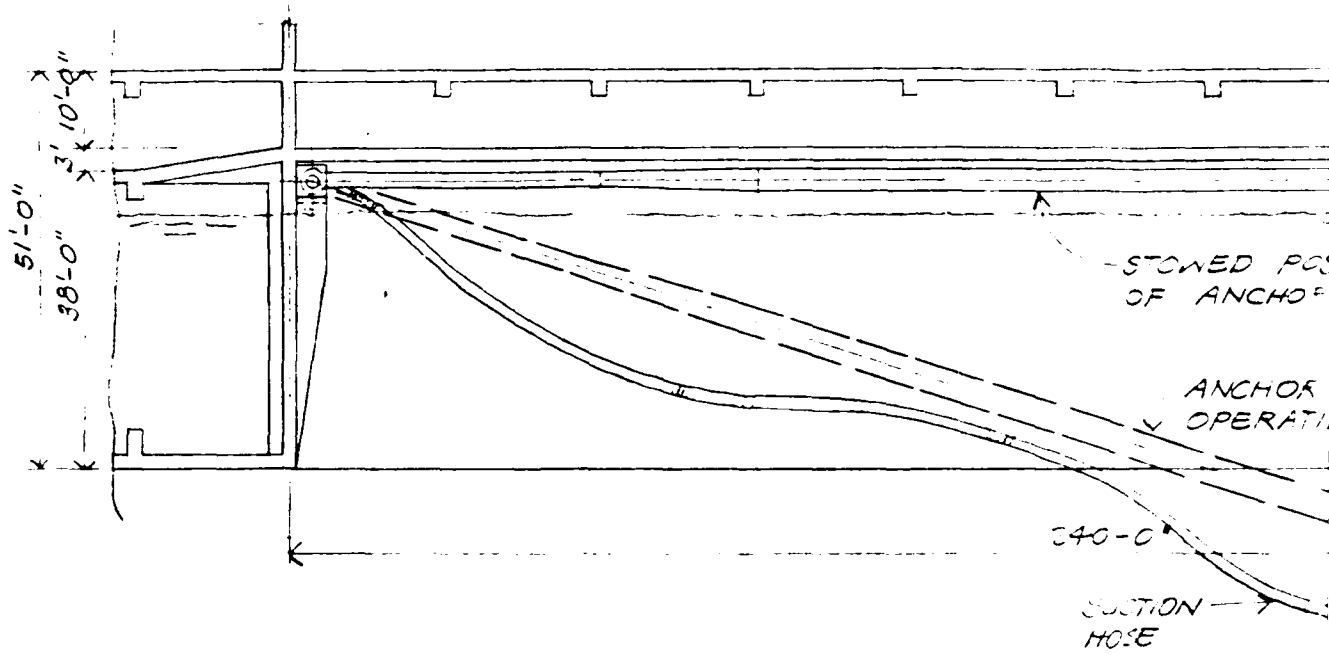
PROJECT NO

23 40-0' = 23

LEG



PLAN  
1" = 200'-0"



ELEVATION  
1" = 200'-0"



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100%	JAN. 5,

60 40'-0" = 240'-0"

LEG

40'-0"  
PER  
CENTERLINE

EDGE OF ANCHOR WELL

PLAN  
"=200'-0"

ANCHOR  
HOIST

MAIN  
DECK

LOWER DECK

WATER LEVEL

STOWED POSITION  
OF ANCHOR BOOM

ANCHOR BOOM IN  
OPERATING POSITION

DRAFT: 24'-0"  
VARIES

240'-0"

SUCTION  
HOSE

SEA BOTTOM

PROPELLANT  
EMBEDDED  
ANCHORS

D  
4

ELEVATION  
"=200'-0"

NO.	REVISION	DATE
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Issued For	Date	By
100%	Jan. 63	FM

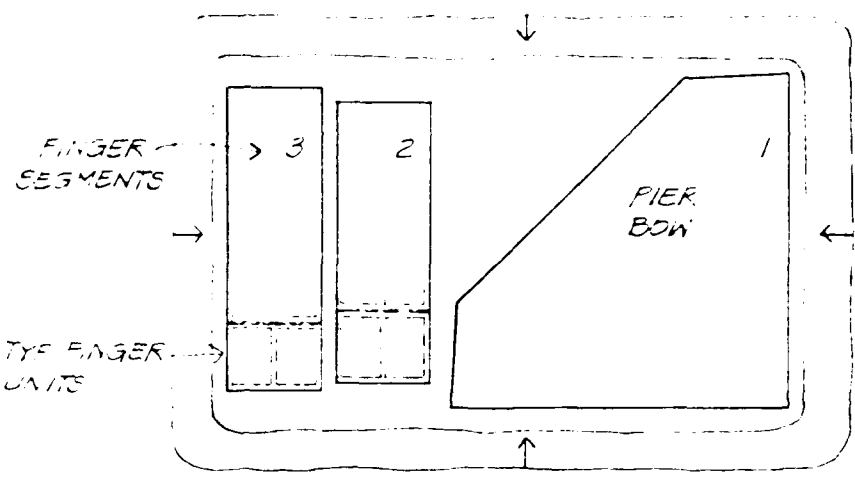
SHEET TITLE **STIFF LEG ANCHOR**  
PROJECT **EXPEDITIONARY PIER**

SHEET NO  
**10**

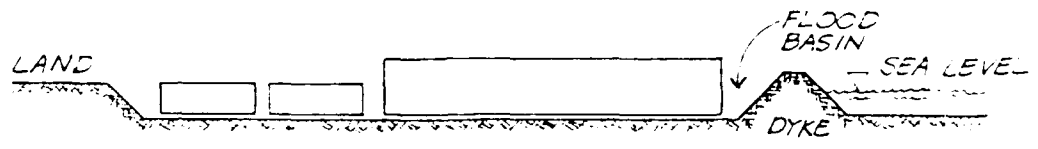
DRAFTING

DESIGN

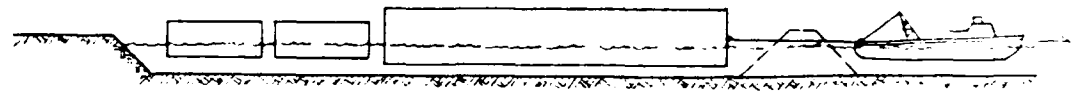
PROJECT NO



FLOOD BASIN PLAN  
 3/4" = 10'-0" TYP.

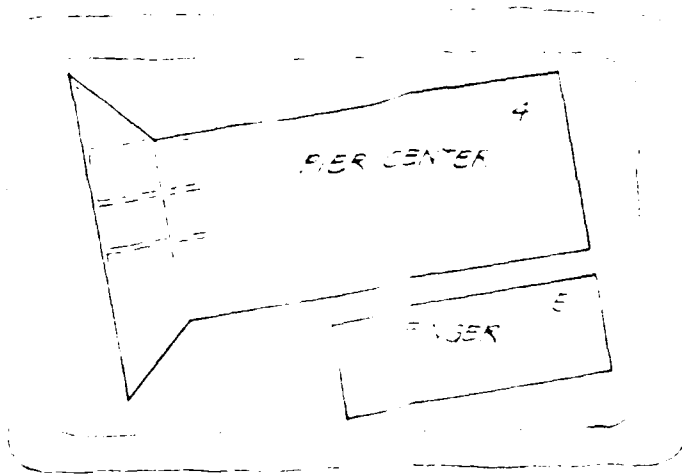


- ① CONSTRUCTION IN FLOOD BASIN
- a) BUILD ONE 520'x320' BASIN
  - b) JOINT PRECAST UNITS OF PIER BOW (1) AND FINGER (2,3) BY POSTENSIGNING
  - c) INSTALL UTILITY SYSTEM



- ② TOW TO ASSEMBLY SITE
- a) REMOVE DYKE TO FLOAT UNITS
  - b) TOW TO ASSEMBLY AND FINISHING SITE

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		100%	Dec, 82

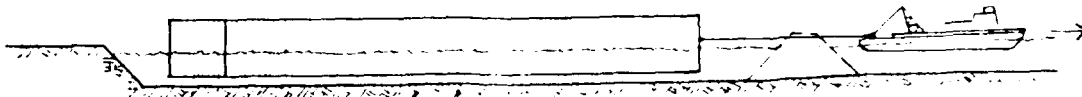


FLOOD BASIN PLAN



③ CONSTRUCTION IN FLOOD BASIN

- a) REBUILD DYKE
- b) PUMP WATER OUT
- c) JOINT PRECAST UNITS OF PIER CENTER (4) AND FINGER SEGMENT (5) BY POSTENSIONING
- d) INSTALL UTILITY SYSTEM



④ TOW TO ASSEMBLY SITE

- a) REPEAT STEP 2

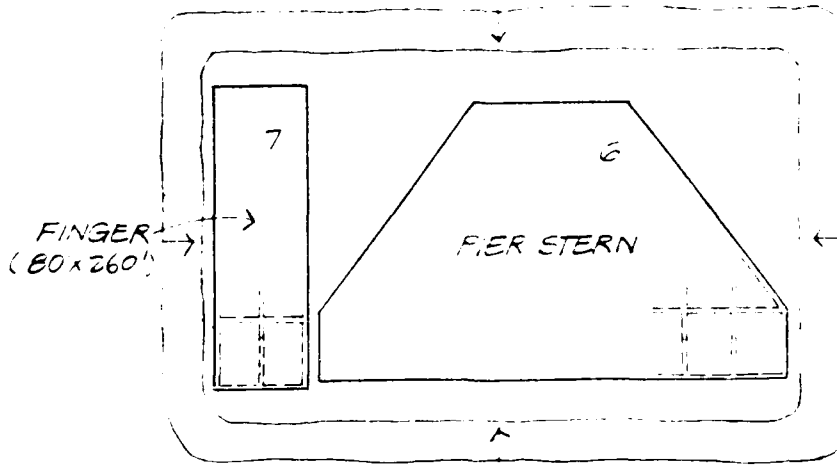
NO	REVISION	DATE

Issued For	Date	By	SHEET TITLE <b>FLOOD BASIN CONSTRUCTION METHOD</b>	SHEET NO <b>11</b>
%	Dec, 82	RM		
PROJECT			<b>EXPEDITIONARY PIER</b>	

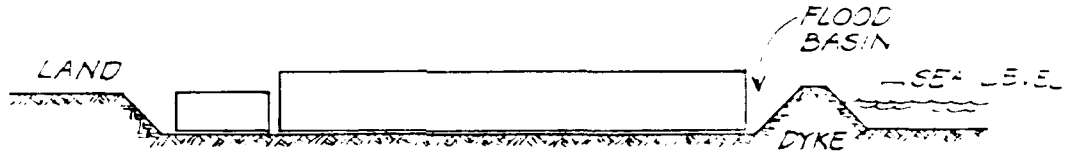
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PROJECT NO



FLOOD BASIN PLAN



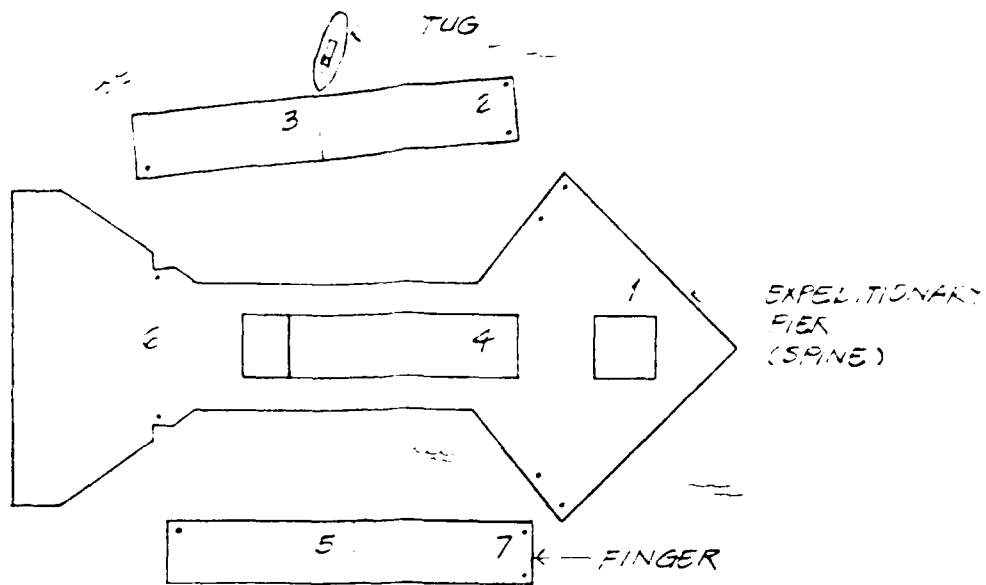
⑤ CONSTRUCTION IN FLOOD BASIN

- a) REPEAT STEP 3a, 3b
- b) JOINT PRECAST UNITS OF PIER STERN (6) AND FINGER (7) BY POSTENSIONING
- c) INSTALL UTILITY SYSTEM
- d) REPEAT STEP 2

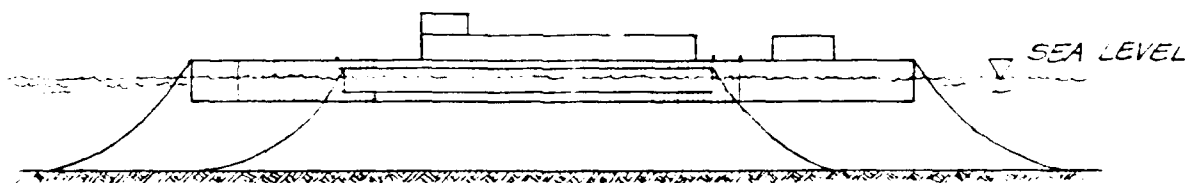


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ASSEMBLY SITE PLAN



6 ASSEMBLY PIER SEGMENTS

- a) JOIN EXPEDITIONARY PIER (1,4,6) AND FINGERS (2,3 FIRST THEN 5,7) BY POSTENSIONING
- b) TUG FINGERS INTO SPINE
- c) COMPLETE CONSTRUCTION
- d) HOOK UP UTILITY LINES
- e) READY FOR USE

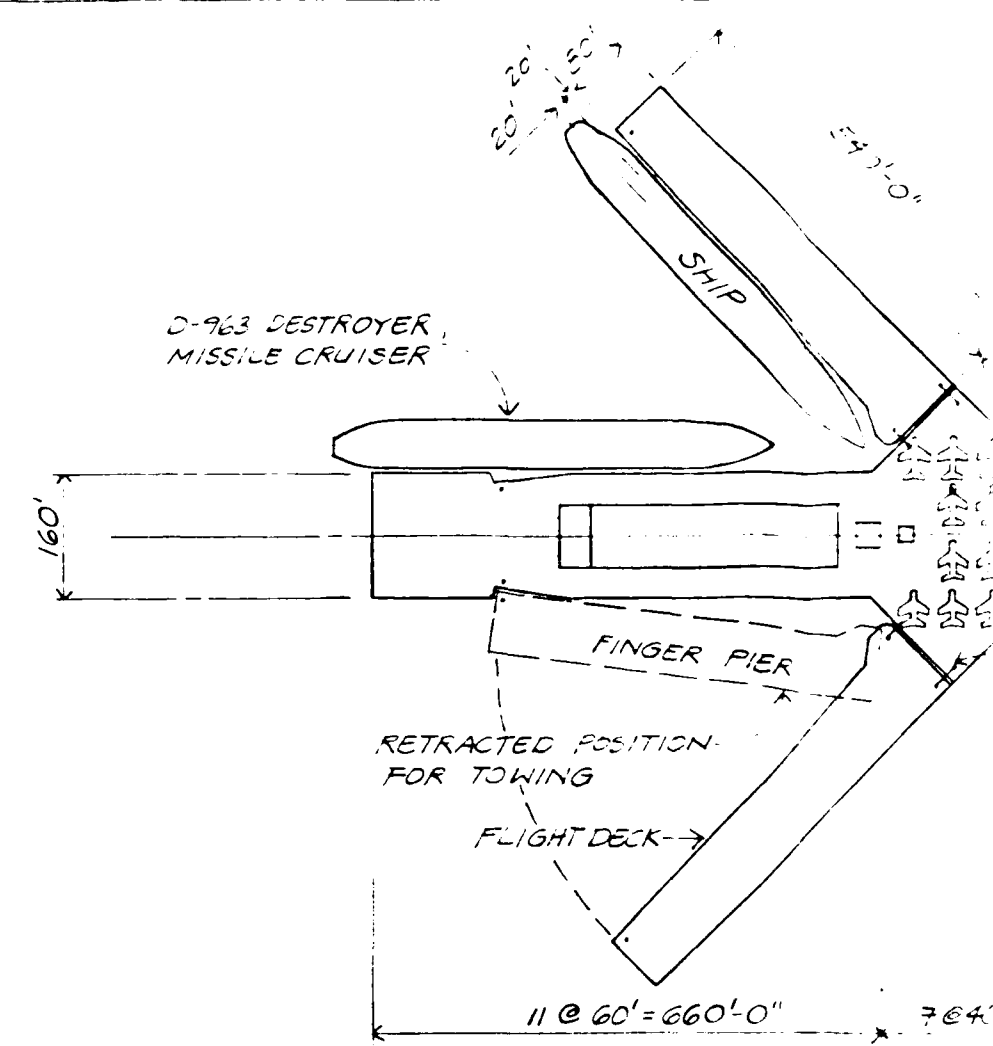
NO	REVISION	DATE
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Issued For	Date	By	SHEET TITLE	SHEET NO
100%	DEC 82	RM	FLOOD BASIN CONSTRUCTION METHOD EXPEDITIONARY PIER	12

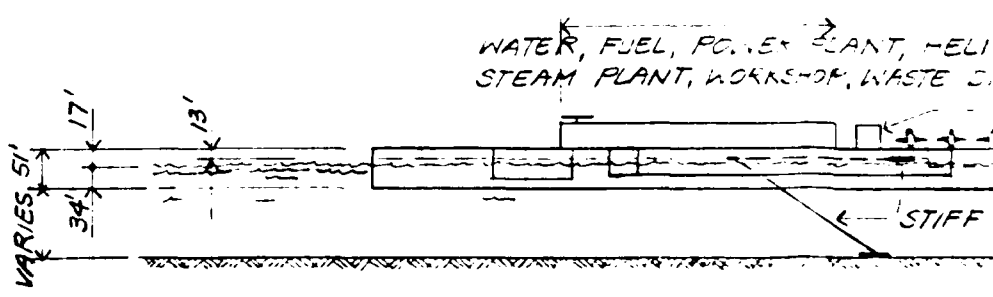
DRAFTING

DESIGN

PROJECT N9



PLAN

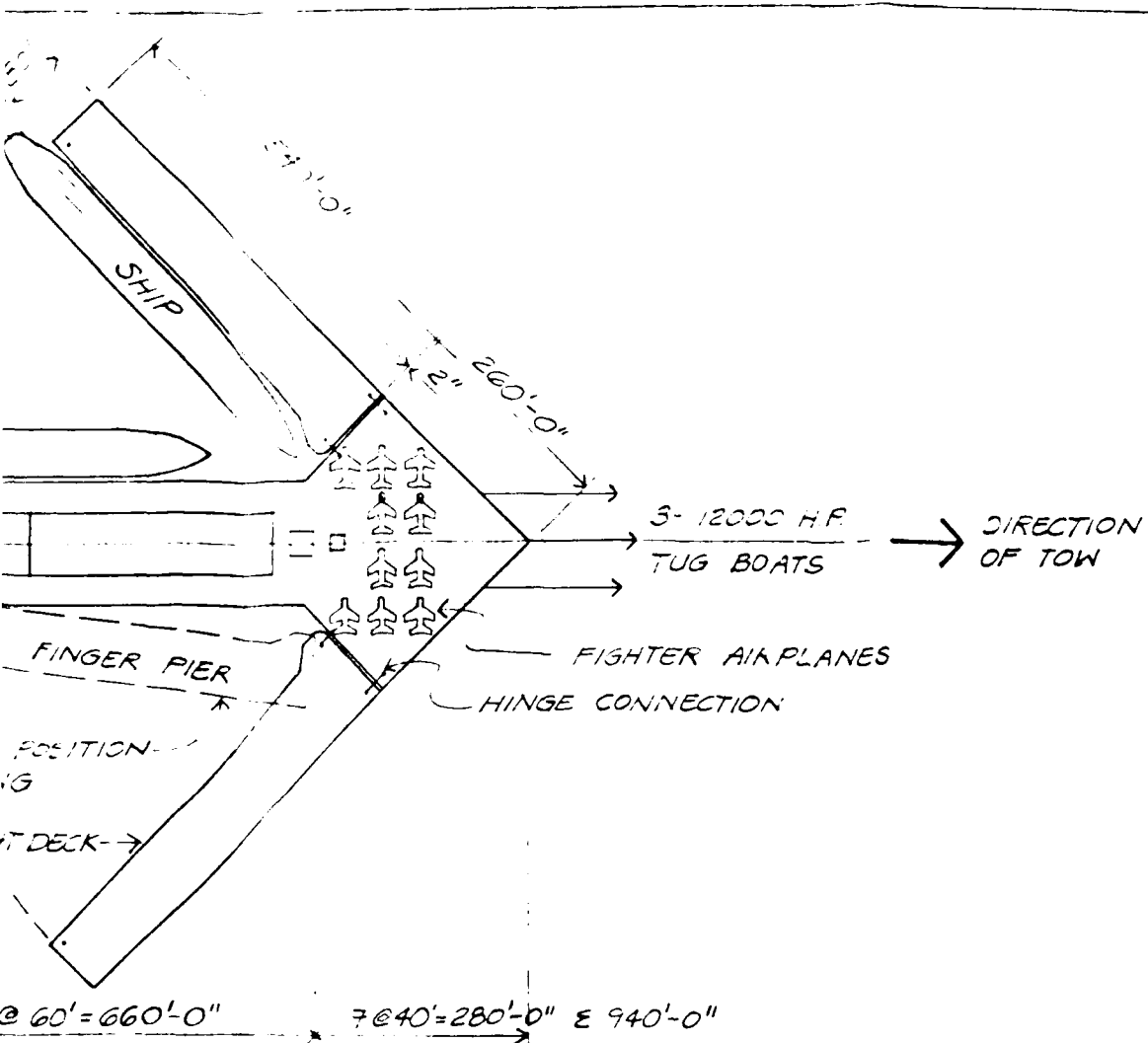


PROFILE  
1" = 200'

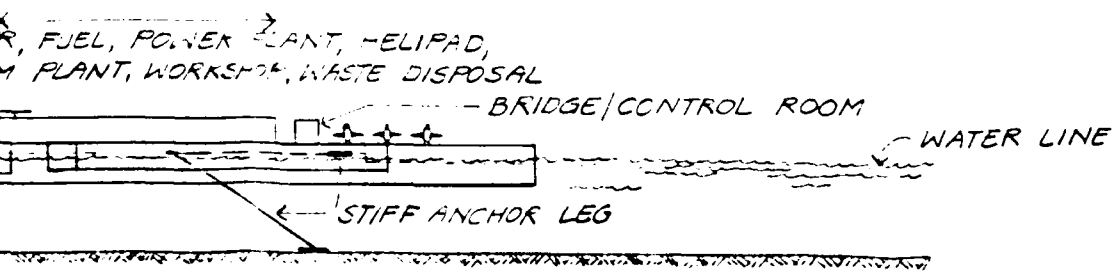


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100%	Dec, 8



PLAN



PROFILE

1" = 200'

NO	REVISION	DATE

Issued For	Date	By
100%	DEC, 82	KM

SHEET TITLE	SCHEME D
PROJECT	EXPEDITIONARY PIER

SHEET NO	13
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END

DATE  
FILMED

9 - 83

DTI