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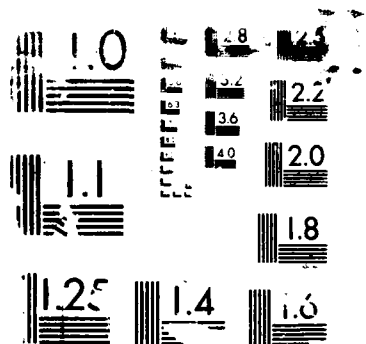
FREE BOUNDARY PROBLEMS ARISING IN THE CONTROL OF A
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FREE BOUNDARY PROBLEMS ARISING IN THE CONTROL OF A FLEXIBLE ROBOT ARM¹²

Thomas I. Seidman

Department of Mathematics and Statistics

University of Maryland Baltimore County

Catonsville, MD 21228 USA

(BITNET: seidman@umbc, arpanet: seidman@umbc3)

ABSTRACT: Modeling of a flexible robot arm mounted at a prismatic joint leads to a moving boundary problem. In a control-theoretic context with joint motion as control, these become *free boundary problems*. Some problems are discussed for transverse vibration (the beam equation) but the principal result is an *exact controllability theorem* for longitudinal vibration (the wave equation).

Key words: free boundary, robot arm, prismatic joint, beam equation, feedback, wave equation, exact controllability

1. INTRODUCTION

We consider the modeling of a robot arm consisting of a flexible beam with a *prismatic joint* — that is, a mounting of the arm in which one end is clamped but permits controlled in-out motion so the extension of the arm (i.e., the relevant portion of the beam) has variable length $\ell = \ell(t)$. In this case 'flexible' means only that the dynamics of the beam must be modeled by a partial differential equation: treatment as a rigid body is an inadequate approximation in that vibration (i.e., time-dependent deviation from the nominal rigid-body position) is a significant aspect of the problem. The space-time domain on which we consider the dynamics is thus

$$(1.1) \quad \mathcal{Q} = \mathcal{Q}_T : \{(t, s) : 0 < t < T; 0 < s < \ell(t)\}$$

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²This material was presented at the international colloquium: *Free Boundary Problems: Theory and Applications* held at Irsee (Bavaria, Germany) June 11-20, 1987. The present text will appear in the proceedings of that conference, edited by K.-H. Hoffmann and J. Sprekels.

where s denotes length (in *material coordinates*, measured from the tip of the arm). This is certainly a *moving boundary problem*. From a control-theoretic point of view, we note that the significant question is precisely the determination of the control $\ell(\cdot)$ so we will have a *free boundary problem*.

In the next section we analyze purely longitudinal vibration of such a beam. This is not, of course, the problem of greatest practical concern but we will be able to treat it with some degree of completeness — showing *exact controllability*, subject only to some rather mild conditions. The more significant problem of treating transverse vibration in this context requires more extensive analysis than can be presented here. A brief discussion of such problems³ is presented in Section 3. For our present purposes we take the beam to be (nominally) horizontal along the x -axis of the 'laboratory coordinate system' with the joint rigidly fixed⁴ at the origin. Included in our model is a load of mass m supported at the free end of the arm. We assume that the system is (except for the boundary condition at the joint) conservative — dissipative mechanisms such as friction and, e.g., air resistance can be neglected.

2. EXACT CONTROLLABILITY

Here we restrict our consideration to the purely *longitudinal* vibrations of the arm. Suppose we let $x(t, s)$ be the actual position (in the laboratory coordinate system) of the material point s at time t . The formulation of the dynamics takes a much simpler form if we take the unknown to be $v := [s + x]$ so, e.g., $v = \ell$ at the joint. For a hyperelastic stress-strain law⁵ one can easily show, using Hamilton's Principle of Stationary Action and a bit of manipulation, that v must satisfy the one-dimensional wave equation:

$$(2.1) \quad \rho v_{tt} = \psi(\cdot, v_s)_s \quad \text{on } Q_T$$

with the boundary conditions:

$$(2.2) \quad v = \ell \quad \text{at the joint: } s = \ell = \ell(t),$$

$$(2.3) \quad m v_{tt} = \psi(v_s) \quad \text{at the tip: } s = 0.$$

³Work on such problems has been initiated by P.K.C. Wang and, indeed, [2] should be acknowledged as the principal stimulus to the present work. Many interesting and important problems remain open.

⁴In practice one would expect that the joint may, itself, be mounted on a moving support (e.g., on another arm) and/or that there is coupling with a controlled rotation around a vertical axis. One might also wish to include the possibility that the mounting is *nominally* rigid yet subject to vibration giving 'noise' in the boundary condition there; compare [1].

⁵We make the usual assumption that the *elastic potential* $\Psi = \Psi(s, r)$ is convex in r with minimum at $\Psi(s, 0) = 0$. We set $\psi(s, r) = \Psi_s(s, r)$ and let $\rho(s)$ be the mass density.

To this we adjoin specification of initial conditions:

$$(2.4) \quad \ell(0) = \ell^0 \text{ and } v = v^0, v_t = w^0 \text{ on } [0, \ell^0] \text{ at } t = 0,$$

subject to (2.2): $v^0(\ell^0) = \ell^0$ as a *compatibility condition*.

Suppose, in addition to (2.4), one were also to specify a target state for the terminal time $t = T$:

$$(2.5) \quad \ell(T) = \ell^T \text{ and } v = v^T, v_t = w^T \text{ on } [0, \ell^T] \text{ at } t = T,$$

again subject to (2.2). Our object is to prove *exact controllability*:

There should be a time $T > 0$ and an admissible control function $\ell(\cdot)$ on $[0, T]$ such that (2.1), (2.4), (2.2), (2.3) give (2.5)

under suitable (mild) hypotheses. By "admissible" we mean that $\ell(0) = \ell^0, \ell(T) = \ell^T$, and⁶

$$(2.6) \quad |\ell(t_1) - \ell(t_2)| \leq \kappa |t_1 - t_2| \quad \text{with } \kappa < 1$$

which will ensure that the free boundary curve: $\Gamma := \{s = \ell(t)\}$ is 'time-like' so a characteristic $[s \pm t = \text{const.}]$ can intersect Γ only once (transversely).

For simplicity of exposition we present in detail only the linear case⁷ for which (2.1) reduces to

$$(2.7) \quad v_{tt} = v_{ss} \quad \text{on } Q_T.$$

In this case the *method of characteristics* gives the simple decomposition

$$(2.8) \quad v(t, s) = f(s + t) + g(s - t)$$

into left- and right-moving waves. Note that differentiating (2.8) gives the identities:

$$(2.9) \quad f'(s + t) = [v_s(t, s) + v_t(t, s)]/2,$$

$$g'(s - t) = [v_s(t, s) - v_t(t, s)]/2.$$

The representation (2.8) gives the general (weak) solution of (2.7) and, in terms of f, g , the boundary conditions (2.2), (2.3) now become:

$$(2.10) \quad f(\ell(t) + t) + g(\ell(t) - t) = \ell(t),$$

$$(2.11) \quad m[f''(s) + g''(-s)] = f'(s) + g'(-s).$$

⁶Physically, this corresponds to a requirement that the controlled motion at the joint must always be slower than the wave propagation speed along the beam. For smooth $\ell(\cdot)$ it is equivalent to $|\dot{\ell}(t)| \leq \kappa < 1$.

⁷I.e., $\Psi(s, r) = \frac{1}{2}\beta(s)r^2$ so $\psi = \beta r$. We further simplify by assuming the beam is homogeneous so, in suitable units, $\rho = \beta = 1$.

Note that we can differentiate (2.10) with respect to t to obtain an ODE for ℓ :

$$(2.12) \quad \dot{\ell} = \frac{f'(\ell+t) - g'(\ell-t)}{1 - [f'(\ell+t) - g'(\ell-t)]}.$$

From the inequality: $|u-v| + |u+v| \leq 2 \max\{|u|, |v|\}$ it would follow that (2.12) is well-behaved — indeed, giving (2.6) — if we were always to have

$$(2.13) \quad |f'|, |g'| \leq \kappa/2$$

where defined.

The initial data v^0, w^0 in (2.4) determine f, g on $[0, \ell^0]$ using (2.9) with $t = 0$. In particular, we have g' determined on $D_0^g := \{(t, s) : 0 \leq s - t \leq \ell^0\}$. We then can use (2.11) as an ODE for g' to obtain:

$$(2.14) \quad g'(s) = \frac{1}{2}[v_s^0 + w^0](-s) + e^{s/m}w^0(0) + \frac{1}{m} \int_0^{-s} e^{(s+r)/m}[v_s^0 + w^0](r) dr$$

for $-\ell^0 \leq s \leq 0$, which defines g' from (2.4) on $D_1^g := \{(t, s) : -\ell^0 \leq s - t \leq 0\}$. Finally, we note that $v(\ell^0, 0)$ is determined by the data of (2.4).

Similarly, we note that the target data in (2.5) directly determine f' on $D_0^f := \{(t, s) : T \leq s + t \leq \ell^T + T\}$, using (2.9) with $t = T$, and give,⁸ now using (2.11) as an ODE for f' :

$$(2.15) \quad f'(T - \tau) = \frac{1}{2}[v_s^T + w^T](\tau) + e^{-\tau/m}w^T(0) + \frac{1}{m} \int_0^\tau e^{-\tau/m}[v_s^T + w^T](r) dr$$

for $0 \leq \tau \leq \ell^T$, which defines f' from (2.5) on $D_1^f := \{(t, s) : T - \ell^T \leq s + t \leq T\}$. From (2.5) we also, then, determine $v(T - \ell^T, 0)$.

We now choose Cauchy data on the segment: $\mathcal{G} := \{(t, 0) : \ell^0 \leq t \leq T - \ell^T\}$, taking $v(t, 0)$ linear in t , making v_t constant:

$$(2.16) \quad v_t(t, 0) = c := \frac{v(T - \ell^T, 0) - v(\ell^0, 0)}{T - [\ell^0 + \ell^T]}$$

on $\mathcal{G}$. This gives $v_s(t, 0) = m v_{tt} = 0$ by (2.3) with $\psi(r) = r$. Using (2.9), we now have

$$(2.17) \quad \begin{aligned} f' &= c/2 & \text{on } D_2^f &:= \{(t, s) : \ell^0 \leq s + t \leq T - \ell^T\}, \\ g' &= -c/2 & \text{on } D_2^g &:= \{(t, s) : \ell^T - T \leq s - t \leq -\ell^0\}. \end{aligned}$$

⁸To avoid any possibility of conflict we assume that T is chosen so $T > \ell^0 + \ell^T$, giving $-\ell^0 < \ell^T - T$ and $\ell^0 < T + \ell^T$; later we will strengthen this requirement to (2.18). Note that the computation, here, from (2.5) is independent (to within translation in t) of the particular choice of T .

Note that if we require that

$$(2.18) \quad T \geq [\ell^0 + \ell^T] + |v(T - \ell^T, 0) - v(\ell^0, 0)|/\kappa,$$

then (2.16), (2.17) give (2.13) on D_2^f, D_2^g . If we now impose the requirement that

$$(2.19) \quad |v_s^0|, |w^0| \leq \kappa/4 \text{ on } [0, \ell^0],$$

$$|v_s^T|, |w^T| \leq \kappa/4 \text{ on } [0, \ell^T],$$

then (2.13) holds on D_0^f, D_0^g directly from (2.9) and it easily follows from (2.14), (2.15), that we will also have (2.13) on D_1^f, D_1^g .

Theorem 1: Suppose one is given the data of (2.4), (2.5) subject to compatibility with (2.2) and the condition (2.19) with $\kappa < 1$. (Assume w^0, w^T are continuous and $v^0, v^T \in C^1$.) Then there exists $T > 0$ and an admissible control function $\ell(\cdot) \in C^1[0, T]$ such that the solution v of (2.7), (2.4), (2.2), (2.3) also satisfies (2.5).

PROOF: As above, solve (2.7), (2.4), (2.3) on the triangle $\Delta_0 = \{(t, s) : 0 \leq t, s; s + t \leq \ell^0\}$ and (2.7), (2.5), (2.3) on $\Delta_T = \{(t, s) : 0 \leq T - t, s; s - t \leq \ell^T - T\}$ — obtaining, in particular, $v(\ell^0, 0)$ and $v(T - \ell^T, 0)$. Then choose T subject to (2.18) and specify Cauchy data on $\mathcal{G}$ as above — i.e., (2.16) and $v_s = 0$. Thus, f is determined on $D^f = D_0^f \cup D_1^f \cup D_2^f$ and g on $D^g = D_0^g \cup D_1^g \cup D_2^g$ with (2.13) satisfied on $S = D^f \cap D^g$ as in the discussion above. Note that our assumptions give f', g' continuous on S .

Starting with the initial condition: $\ell(0) = \ell^0$, solve⁹ (2.12) to obtain Γ . This solution curve satisfies (2.10) and (2.6). Note that the solution curve Γ cannot leave S before $t = T$ by, e.g., crossing the characteristic: $[s + t = \ell^T + T]$ — say, at a point (ℓ, t_*) — since this would give, by (2.10):

$$\begin{aligned} \ell^T + T - t_* &= \ell_* = f(\ell_*, t_*) + g(\ell_*, -t_*) \\ &= f(\ell^T + T) + g(\ell^T + T - 2t_*) \\ &= [\ell^T - g(\ell^T - T)] + g(\ell^T + T - 2t_*), \end{aligned}$$

contradicting the fact that $|g'| \leq \kappa/2 < 1/2$; similarly, Γ cannot leave S by crossing $[s - t = \ell^T - T]$ since $|f'| < 1/2$. It follows that $\ell(\cdot)$ is well-defined on $[0, T]$ with $\ell(T) = \ell^T$.

This specification of $T, \ell(\cdot)$ defines $\mathcal{Q}_T$. In view of (2.6), the geometry of Γ ensures that $\mathcal{Q}_T \subset \Delta_0 \cup \Delta_T \cup S$ so f, g are both defined on all of $\mathcal{Q}_T$. Now (2.8) gives a (weak) solution v of (2.7) and the construction ensures that we also have (2.4), (2.5), (2.2), (2.3) as desired. $\square$

⁹A solution exists since the right hand side of (2.12) is continuous in ℓ (as long as one remains in S). Our assumptions do not give a Lipschitz condition; thus uniqueness — not really needed for the Theorem — would have to be obtained by a different argument: noting that (2.13) makes the right hand side of (2.10) contractive in ℓ .

Remarks: The regularity required for the data — continuity of v_s^0, w^0, v_s^T, w^T — can be weakened by a limit argument. In this case one retains (2.6) although ℓ may no longer be continuous. There is also no really new difficulty in considering spatially variable coefficients: $\rho v_{tt} = (\beta v_s)_s$. The argument must be modified in detail¹⁰ but is essentially similar in spirit.

It is worth noting that the construction in the proof above can be viewed from a slightly different perspective. After solving on Δ_0 and Δ_T one fills in Cauchy data on a 'gap' $\mathcal{G}$ so as to consider an *initial value problem* with the roles of t and s reversed — taking, say, v^0, v^T (perhaps suitably extended to larger s) as *boundary data*. The choice of data on $\mathcal{G}$ and the bounds (2.19) are such as to ensure that $|v_s| \leq \kappa < 1$ which then makes the right hand side of (2.10) contractive in ℓ so (2.10) determines $\ell(t)$ uniquely for each t . (One must also, of course, verify that this gives $\ell(\cdot)$ as an 'admissible control function'.)

Stated this way, it is clear that the argument does not really depend on the linearity of (2.7) or on the possibility of a decomposition (2.8). Without providing details, we note only that one can treat (2.1) from this point of view under quite mild conditions on $\psi(\cdot)$. If we set $\phi := \psi^{-1}$, $w = v_t$ and $u = \psi(v_s)$ so $v_s = \phi(u)$ we can rewrite (2.1) in terms as a system:

$$(2.20) \quad u_s = \rho w_t \quad w_s = \phi(u)_t$$

suggesting the role-reversed problem. The crux of the argument, now, is to show that 'reasonable bounds' on the initial and target data can be imposed to ensure that one will have (for a suitable construction of $\mathcal{G}$ and of the data there, satisfying $m w_t = u$) the necessary bound: $|v_s| = |\phi(u)| \leq \kappa$ to be able to apply the Contractive Mapping Theorem to (2.10). $\square$

3. TRANSVERSE VIBRATION

In this section we present an extremely brief introduction to the class of problems which might be considered. As above, we consider the robot arm as a nominally horizontal beam with a load m at the tip and with no dissipation or body forces (e.g., ignoring friction, gravity, ...). Here, however, we consider *transverse vibrations*. For simplicity we consider only vertical vibration, so the principal unknown is the vertical deflection $u = u(t, s)$. Thus, the 'laboratory coordinates' of a material point s will be $[x, y, z] = [\ell(t) - s, 0, u(t, s)]$. Further, we ignore shear and work with a linearized

¹⁰E.g., the characteristics are no longer straight lines and the propagation along them may no longer be constant. We now must take κ to be a constant less than the *minimum* sound speed and impose a more stringent condition (2.19).

theory.¹¹

We begin with the 'clamped' boundary conditions at the prismatic joint:

$$(3.1) \quad u = u_s = 0 \quad \text{at the joint: } s = \ell(t).$$

Next, in this setting a geometric analysis shows that the stress deformation for a segment ds of the beam is proportional to $[\text{radius of curvature}]^{-2}$ so the strain energy (elastic potential) is $\approx \frac{1}{2} \beta u_{ss}^2 ds$ where β (sometimes written as EI) corresponds to stiffness. The corresponding kinetic energy is:

$$\frac{1}{2} \int_0^{\ell(t)} [\rho (\dot{\ell}^2 + u_t^2) + \alpha u_{st}^2] ds + \frac{1}{2} [m (\dot{\ell}^2 + u_t^2) + a u_{st}^2]_{s=0}$$

where α is the cross-sectional moment. By Hamilton's Principle, we obtain the Euler-Bernoulli beam equation¹²

$$(3.2) \quad \rho u_{tt} + (\alpha u_{stt})_s + (\beta u_{ss})_{ss} = 0 \quad \text{on } \mathcal{Q}_T$$

(where $\mathcal{Q}_T$ is as above in (1.1)) and the other boundary conditions:

$$(3.3) \quad m u_{tt} + \alpha u_{stt} = (\beta u_{ss})_s, \quad a u_{stt} + \beta u_{ss} = 0 \quad \text{at the tip: } s = 0.$$

The system, if the control function $\ell(\cdot)$ were specified, would consist of (3.2) with (3.1) and (3.3) together with initial conditions (2.4). We note here, however, that the type of problem having particular control-theoretic significance is that which arises when $\ell(\cdot)$ is determined in *feedback form*:

$$(3.4) \quad \ell(t) = \Lambda[u(t, \cdot)],$$

i.e., one couples (3.4) with the other conditions as an implicit characterization of the free boundary Γ .

In (3.4), we consider $\Lambda[\cdot]$ as some suitable functional — presumably a function $\Phi(\omega)$ for some (finite) set of implementable observations:¹³ $\omega = C[u]$. For example, one might take Λ to be of the form $\Lambda[u] = \Phi(C[u])$ with

$$(3.5) \quad C : u \mapsto \{u|_{s=0}, u_{tt}|_{s=s_*}\} =: \omega.$$

¹¹Use of a linearized stress-strain relation is a consequence of the fundamental beam assumption: 'infinitesimal' cross-section, giving infinitesimal deformations. The significant assumption is the simplifying but rather restrictive one that the arm always remains straight enough to ignore the so-called 'geometric nonlinearities' — essentially, that one can adequately approximate $[\text{radius of curvature}]^{-2}$ by u_{ss}^2 , linear velocity by $[\dot{\ell} + u_t k]$, angular velocity $[\arctan u_s]_t$ by $-u_{st}$, etc.

¹²Note that the equation is just the standard beam equation when we use s as the independent variable. This formulation is somewhat different from that of [2].

¹³Alternatively, we might consider $\Lambda = \Phi(z)$ where z is the solution of an auxiliary system of the form: $\dot{z} = F(z, \omega)$.

Such an *observation operator* C might be implementable by, say, visual observation of the tip position and placing an accelerometer on the beam at s_* , allowing for the (known) effect of the joint motion. This would give, as an example,

$$(3.6) \quad \ell(t) = \Phi(u(t, 0), u_{tt}(t, s_*))$$

for some given function Φ .

Once a specific form of the feedback functional $\Lambda[\cdot]$ has been prescribed, the problem is analysis¹⁴ of the full system: (3.2), (2.4), (3.1), (3.3), and, e.g., (3.6) to determine existence and uniqueness as well as such properties of control-theoretic interest as stability.

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¹⁴From the control-theoretic viewpoint the interesting analysis begins after this analysis has been done: to determine the feedback functionals with desirable properties and, perhaps, to make an optimal choice.

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