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## SCATSPHERE2 — A Computation for the Plane-Wave Scattering from a Submerged, Elastic, Spherical, Evacuated or Fluid-Filled, Thin Shell

by  
Raymond S. Cheng  
Francis M. Henderson

DTRC-90/029 SCATSPHERE2 — A Computation for the Plane-Wave Scattering from a Submerged, Elastic, Spherical, Evacuated or Fluid-Filled, Thin Shell

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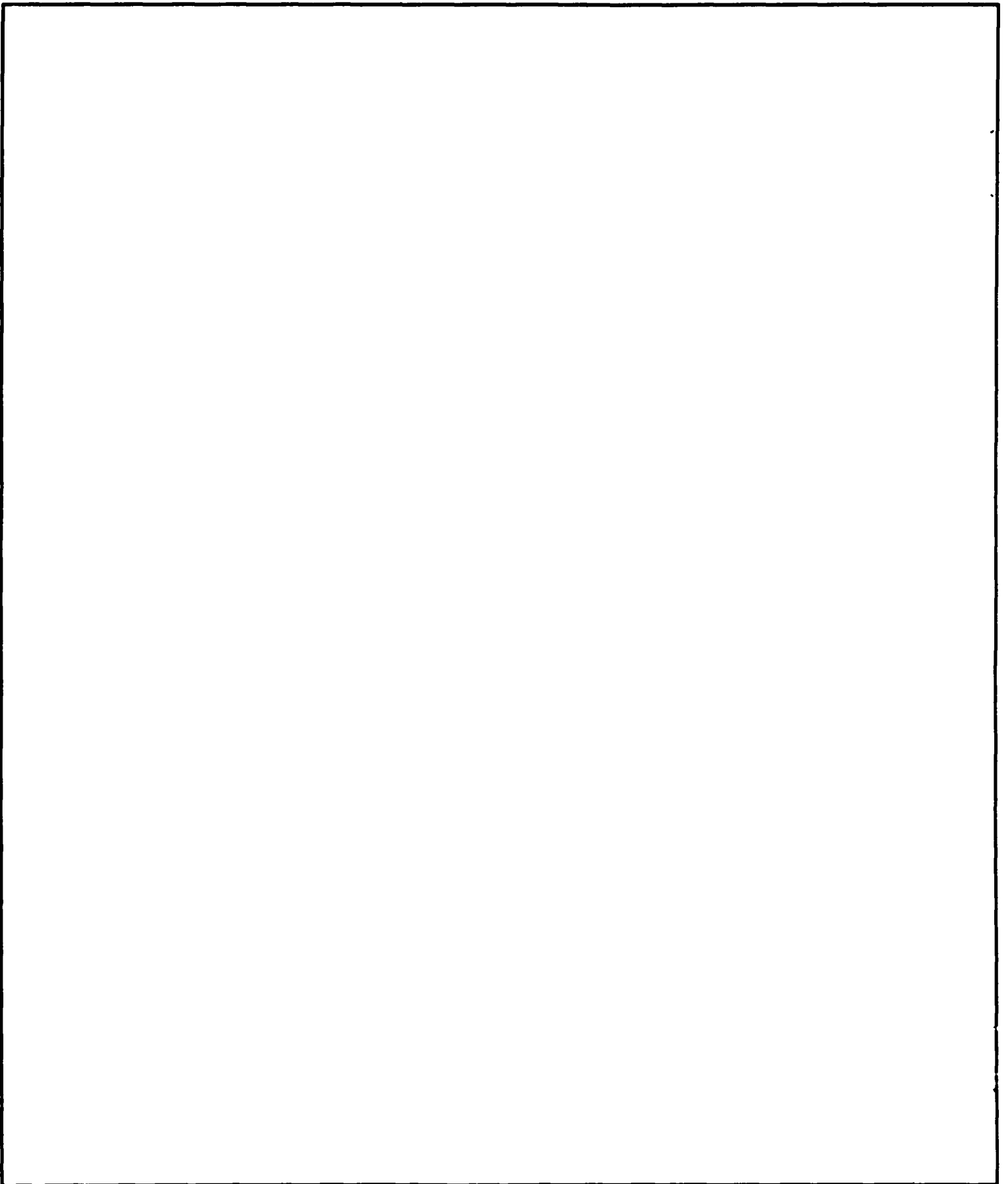
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## ABSTRACT

Program SCATSPHERE2, a modification of program SCATSPHERE by F.M. Henderson, computes the series solution for the time-harmonic plane-wave scattering by a submerged, elastic, spherical, evacuated or fluid-filled thin shell. For program SCATSPHERE2, this report presents the basic scattering theory, the user's instructions, and three examples involving underwater plane-wave scattering by (1) an evacuated steel spherical thin shell, (2) a fluid-filled steel spherical thin shell, and (3) a fluid-filled aluminum spherical thin shell.

## ADMINISTRATIVE INFORMATION

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## INTRODUCTION

A few years ago, the NASHUA structural acoustics procedure<sup>1-5</sup> was created to model the radiation/scattering problem for submerged elastic obstacles by using the boundary element method for the exterior fluid and the finite element method for the structure. The NASHUA procedure could also model interior fluids using structural finite elements by using analogies between the equations for dynamic elasticity and acoustics.<sup>6-7</sup> To verify the NASHUA procedure, three benchmark problems, where the analytical series solutions<sup>8-11</sup> are known, were chosen: (1) time-harmonic uniformly-driven radiation by a submerged elastic spherical thin shell; (2) time-harmonic sector-driven radiation by a submerged elastic spherical thin shell; and (3) time-harmonic plane-wave scattering by a submerged elastic spherical thin shell. Programs RADSPHERE<sup>12</sup> and SCATSPHERE were developed by F.M. Henderson to compute the series solutions for the second and third benchmark problems.

Recently, the NASHUA procedure was enhanced<sup>13</sup> to model structural acoustic problems for submerged fluid-filled structures by using the boundary element method for both the exterior and interior fluids, and the finite element method for the structure. For interior fluids, using the boundary element method instead of the finite element method reduces the matrix computation significantly for fine meshes. To verify the new procedure, the time-harmonic plane-wave scattering by a submerged elastic fluid-filled spherical thin shell was chosen as a benchmark problem since the analytical series solution can be derived and computed. Here, a thin shell assumption is used to simplify the analytical series solution. Program SCATSPHERE2, a modification of program SCATSPHERE, was developed to compute the series solution for scattering by a submerged elastic spherical thin shell that is either evacuated or fluid-filled.

In the next section, the time-independent scattering theory is introduced. A brief review of the plane-wave scattering by a submerged evacuated spherical thin shell<sup>11</sup> is presented. Then the series solution for the plane-wave scattering by a submerged fluid-filled spherical thin shell is derived. Next, the user's instructions are outlined for program SCATSPHERE2. This section includes discussions of the program's inputs and outputs, as well as its graphics capabilities. Finally, the three examples presented are underwater plane-wave scattering by (1) an evacuated steel spherical thin shell; (2) a fluid-filled steel spherical thin shell; and (3) a fluid-filled aluminum spherical thin shell. The first two examples verify the updated NASHUA/NASTRAN procedure while the last example is compared with (and shown to agree with) both the thick shell theory<sup>14</sup> and experimental results.<sup>15-16</sup>

## UNDERWATER SCATTERING THEORY

The three-dimensional wave in the fluid is modeled by the homogeneous Helmholtz equation,

$$\Delta p(\mathbf{x}) + k^2 p(\mathbf{x}) = 0, \quad \mathbf{x} \text{ in } R^3, \quad (1)$$

where  $\Delta$  is the Laplacian operator,  $p$  is the pressure, and  $k$  is the wavenumber. Here, the time-dependent term  $\exp(-i\omega t)$  has been suppressed. Using separation of variables in spherical coordinates,<sup>17-18</sup> we

substitute  $p(r, \theta, \phi) = R(r)\Theta(\theta)\Phi(\phi)$ , where  $r$  is the radius,  $\theta$  is the angle from the  $z$ -axis, and  $\phi$  is the angle in the  $x$ - $y$  plane, into Eq. 1 to obtain the spherical Bessel equation,

$$\frac{d}{dr} \left( r^2 \frac{dR}{dr} \right) + (k^2 r^2 - (n^2 + n)) R = 0, \quad (2)$$

the associated Legendre equation,

$$\frac{\sin(\theta)}{\Theta(\theta)} \frac{d}{d\theta} \left[ \sin(\theta) \frac{d\Theta}{d\theta} \right] + (n^2 + n) \sin^2(\theta) = m^2, \quad (3)$$

and the one-dimensional wave equation,

$$\frac{-1}{\Phi(\phi)} \frac{d^2\Phi}{d\phi^2} = m^2. \quad (4)$$

The ranges of  $\phi$  and  $\theta$  restrict the general solution to those with integer eigenvalues,  $m$  and  $n$ . The general solution of Eq. 2 is expressed in terms of  $j_n(kr)$ , the spherical Bessel function of the first kind, and  $h_n(kr)$ , the spherical Hankel function of the first kind, both of order  $n$ . Note that  $j_n(kr)$  is finite at the origin and unbounded in the infinite space, while  $h_n(kr)$  is unbounded at the origin and satisfies the Sommerfeld radiation condition in the infinite space for the assumed  $\exp(-i\omega t)$  time dependence. The general solution of Eq. 3 is expressed in terms of  $P_n^m(\cos\theta)$ , the associated Legendre polynomials of order  $m$  and  $n$ . The general solution of Eq. 4 is the trivial one-dimensional wave solution. Note that  $m$  is the number of great nodal circles intersecting the  $z$ -axis, and  $n$  is the number of nodal circles coaxial with the  $z$ -axis. The plane-wave scattering by spherical obstacles is axisymmetric, and therefore, without loss of generality,  $d\Phi/d\phi=0$ ,  $m=0$ , and  $\Theta$  solves the Legendre equation.

Let the notation "J&F" indicate a formula in *Sound, Structures, and Their Interaction* by Junger and Feit.<sup>11</sup> By an addition theorem,<sup>19(section 10.1.47)</sup> the free-space Green's function in spherical coordinates (J&F, section 6.10) is

$$G(\mathbf{x}, \mathbf{x}_0) = \frac{\exp(i k |\mathbf{x} - \mathbf{x}_0|)}{4 \pi |\mathbf{x} - \mathbf{x}_0|} \quad (5)$$

$$= \frac{-i k}{4 \pi} \sum_{n=0}^{\infty} \sum_{m=0}^n \epsilon_m \frac{(n-m)!}{(n+m)!} (2n+1) P_n^m(\cos\theta) P_n^m(\cos\theta_0) \cos(m(\phi - \phi_0)) h_n(kr) j_n(kr_0),$$

$$r \geq r_0, \quad \mathbf{x}, \mathbf{x}_0 \text{ in } R^3,$$

where the Neumann factor  $\epsilon_m$  is 1 for  $m=1$  and 2 for  $m > 1$ . By the same addition theorem, the incident plane-wave pressure (J&F, section 10.13) is

$$p_0(r, \theta) = \Phi_0 \sum_{n=0}^{\infty} (2n+1) i^n P_n(\cos\theta) j_n(kr), \quad (6)$$

where  $\Phi_0$  is the incident pressure magnitude.

#### EVACUATED SPHERICAL THIN SHELL

In this section, the series solution for the plane-wave scattering by a submerged evacuated spherical thin shell is reviewed. Assume that the shell is thin enough and the frequency is low enough that flexural stresses can be ignored as compared to membrane stresses. The plane-wave scattering by the submerged evacuated spherical thin shell requires finding (1) the shell impedance, (2) the fluid radiation impedance on the exterior surface using the Green's function for the exterior region, (3) the "rigid-body" scattered field, and (4) the radiated scattered field (due to shell vibration).

Using the equations for nontorsional axisymmetric motions (J&F, sections 7.102 and 7.103), the shell impedance for the  $n$ th axisymmetric mode (J&F, section 7.121) is

$$Z_n = -\frac{i \rho_s c_p}{\Omega} \frac{h}{a} \frac{[\Omega^2 - (\Omega_{n1})^2][\Omega^2 - (\Omega_{n2})^2]}{[\Omega^2 - (1 + \beta^2)(\nu + \lambda_n - 1)]}, \quad (7)$$

where  $\rho_s$  is the shell density,  $c_p = \sqrt{E/[\rho_s(1-\nu^2)]}$ ,  $E$  is the Young's modulus,  $h$  is the shell thickness,  $a$  is the shell mean radius,  $\Omega = \omega a / c_p$  is the dimensionless frequency,  $\beta = h/(a \sqrt{12})$ ,  $\nu$  is the Poisson's ratio, and

$\lambda_n = n(n+1)$ . The quantities  $\Omega_{n1}$  and  $\Omega_{n2}$  are the upper and lower shell resonance dimensionless frequencies, respectively. They are the solutions of the characteristic equation (J&F, section 7.114)

$$\begin{aligned} \Omega^4 - [1 + 3\nu + \lambda_n - \beta^2(1 - \nu - \lambda_n^2 - \nu\lambda_n)]\Omega^2 + (\lambda_n - 2)(1 - \nu^2) \\ + \beta^2[\lambda_n^3 - 4\lambda_n^2 + \lambda(5 - \nu^2) - 2(1 - \nu^2)] = 0. \end{aligned} \quad (8)$$

The exterior fluid impedance (the fluid radiation impedance at the exterior surface) is found by first finding the exterior region's Green's function which must satisfy the Sommerfeld radiation condition and the Neumann boundary condition. The former condition implies that the exterior region's Green's function is in terms of  $h_n(kr)$ . The latter condition and the Wronskian relation <sup>19</sup>(sections 8.14.11 and 3.14.14) implies that the Green's function for the exterior region specialized to the spherical surface (J&F, section 6.15) is

$$\begin{aligned} G(r, \theta, \phi | a, \theta_0, \phi_0) = \frac{1}{4\pi ka^2} \sum_{n=0}^{\infty} \sum_{m=0}^{\infty} \epsilon_m \frac{(n-m)!}{(n+m)!} (2n+1) \cos(m(\phi - \phi_0)) P_n^m(\cos\theta) P_n^m(\cos\theta_0) \frac{h_n(kr)}{h_n'(ka)}, \quad (9) \\ r \geq a. \end{aligned}$$

The Green's identity for the exterior region and the problem's axisymmetry imply that the fluid pressure (J&F, section 6.16 and 6.19) is

$$\begin{aligned} p(r, \theta, \phi) = \frac{\rho_f}{i\omega} \int_S G(r, \theta, \phi | a, \theta_0, \phi_0) w'(\theta_0, \phi_0) \sin(\theta_0) dS \\ = i \rho_f c_f \sum_{n=0}^{\infty} w_n' P_n(\cos\theta) \frac{h_n(kr)}{h_n'(ka)}, \quad r > a, \end{aligned} \quad (10)$$

where  $\rho_f$  is the fluid density,  $w'$  is the shell surface normal velocity, and  $c_f$  is the fluid sound speed. The exterior fluid impedance of axisymmetric mode  $n$  (J&F, section 6.29) is

$$z_n = \frac{p_n(a, \theta, \phi)}{w_n'(a, \theta, \phi)} = i \rho_f c_f \frac{h_n(ka)}{h_n'(ka)}. \quad (11)$$

The "rigid-body" scattered pressure field (J&F, section 10.16) is found by using the Neumann boundary condition and the incident plane-wave pressure. Using the asymptotic expansion for the Bessel function, the far-field "rigid-body" scattered pressure field (J&F, section 10.17) is

$$p_{sr}(r, \theta, \phi) = -\frac{i e^{ikr} \Phi_0}{kr} \sum_{n=0}^{\infty} (2n+1) P_n(\cos\theta) \frac{j_n'(ka)}{h_n'(ka)}, \quad r \gg a. \quad (12)$$

From (J&F, section 11.25) to (J&F, section 11.28), the shell velocity response to the incident and "rigid-body" scattered pressure is

$$\begin{aligned} w'(r, \theta, \phi) &= -\sum_n \frac{p_n P_n(\cos\theta)}{(Z_n + z_n)} \\ &= -\frac{\Phi_0}{k^2 a^2} \sum_{n=0}^{\infty} \frac{i^{n+1} (2n+1) P_n(\cos\theta)}{(Z_n + z_n) h_n'(ka)}, \end{aligned} \quad (13)$$

where  $p_n = (Z_n + z_n) w_n'$  is used. From Eqs. 10 and 13 and the asymptotic expansion for the Bessel function, the far-field radiated pressure due to the shell motion (J&F, section 11.29) is

$$p_r(r, \theta, \phi) = -\frac{i \rho_f c_f e^{ikr} \Phi_0}{kr} \sum_{n=0}^{\infty} \frac{(2n+1) P_n(\cos\theta)}{(Z_n + z_n) [k a h_n'(ka)]^2}, \quad r \gg a. \quad (14)$$

The total far-field scattered pressure (J&F, section 11.30), which is the sum of the "rigid-body" scattered pressure and the radiated pressure, is

$$p_{se}(r, \theta, \phi) = -\frac{i e^{ikr} \Phi_0}{kr} \sum_{n=0}^{\infty} \frac{(2n+1) P_n(\cos\theta)}{h_n'(ka)} \left[ j_n'(ka) - \frac{\rho_f c_f}{(Z_n + z_n) (ka)^2 h_n'(ka)} \right], \quad r \gg a. \quad (15)$$

For more in-depth theoretical presentation, see *Sound, Structures, and Their Interaction*, by Junger and Feit.<sup>11</sup>

## FLUID-FILLED SPHERICAL THIN SHELL

In this section, the analytical plane-wave scattering solution for the submerged fluid-filled spherical thin shell is derived by extending the theory from the previous section. The extended solution requires finding (1) the interior fluid impedance using the Green's function for the interior region, and (2) the radiated pressure field due to shell vibration.

The Green's function for the interior region is expressed in terms of  $j_n(kr)$  because it must be finite inside the sphere. Using the Neumann boundary condition to find the unknown coefficient, the Green's function for the interior region is

$$G(r, \theta, \phi | a, \theta_0, \phi_0) = \frac{1}{4\pi k a^2} \sum_{n=0}^{\infty} \sum_{m=0}^{\infty} \epsilon_m \frac{(n-m)!}{(n+m)!} (2n+1) \cos(m(\phi - \phi_0)) P_n^m(\cos\theta) P_n^m(\cos\theta_0) \frac{-j_n(kr)}{j_n'(ka)}, \quad (16)$$

$$r \leq a.$$

The Green's identity for the interior region and the problem's axisymmetry imply

$$p(r, \theta, \phi) = \frac{\rho_g}{i\omega} \int_S G(r, \theta, \phi | a, \theta_0, \phi_0) w'(\theta_0, \phi_0) \sin(\theta_0) dS \quad (17)$$

$$= i \rho_g c_g \sum_{n=0}^{\infty} w_n' P_n(\cos\theta) \frac{-j_n(kr)}{j_n'(ka)}, \quad r \leq a,$$

where  $\rho_g$  is the interior fluid density, and  $c_g$  is the interior fluid sound speed. Thus, the interior fluid impedance of mode  $n$  is

$$\zeta_n = \frac{p_n}{w_n'} = i \rho_g c_g \frac{-j_n(ka)}{j_n'(ka)}. \quad (18)$$

The only difference between computing the evacuated and fluid-filled cases is that the interior fluid impedance  $\zeta_n$  needs to be added to the shell modal impedance  $Z_n$ . Thus, the far-field radiated pressure is

$$p_r(r, \theta, \phi) = -\frac{i \rho_f c_f e^{ikr} \Phi_0}{k r} \sum_{n=0}^{\infty} \frac{(2n+1) P_n(\cos \theta)}{(Z_n + z_n + \zeta_n) [k a h_n'(ka)]^2}, \quad r \gg a. \quad (19)$$

The total far-field scattered pressure (which is the sum of the "rigid-body" scattered pressure and the radiated pressure) is

$$p_{se}(r, \theta, \phi) = -\frac{i e^{ikr} \Phi_0}{k r} \sum_{n=0}^{\infty} \frac{(2n+1) P_n(\cos \theta)}{h_n'(ka)} \left[ j_n'(ka) - \frac{\rho_f c_f}{(Z_n + z_n + \zeta_n) (k a)^2 h_n'(ka)} \right], \quad r \gg a. \quad (20)$$

### PROGRAM IMPLEMENTATION

Program SCATSPHERE2 computes the spherical Bessel function values after using IMSL library calls for the cylindrical Bessel function values.<sup>19</sup>(section 10.1.1) The Legendre polynomials are computed using a recursion relation.<sup>20</sup> The infinite series is approximated by a finite number of terms. More terms are added until either the relative error criterion is met or the maximum number of terms allowed is used.

### USER'S INSTRUCTIONS

SCATSPHERE2 input data consist of 11 record types for 25 parameters which are illustrated in Table 1. All variables are read in as unformatted (i.e., separated by commas). Records 7a and 7b are read in only if the field pressure calculations are done, i.e., NFLAG = 2 or 3. Record 7b consists of an array of radii whose dimension depends on NRADII. After NINT is recorded, the next record, Record 11, is read NINT times. The dimensionless frequency in Table 1 is defined as  $\Omega = k a c_f / c_p$ . The complex Young's modulus is  $E(1 + i\eta)$ .

The series is computed for as many terms as needed until the relative error criterion is satisfied or the maximum number of terms is used. Recommended values are NTERMS = 40 and ERRCRT =  $1.0 \times 10^{-6}$ . If erratic behavior occurs, then increase NTERMS and decrease ERRCRT. Overall, the computation is quite inexpensive even for small ERRCRT. The maximum value for NTERMS is 201, and the maximum number of

Table 1. SCATSPHERE2 input data.

Record No.	Parameter	Description
1	TITLE	Title Card (up to 75 characters)
2	$\Phi_0$	Incident Pressure Magnitude
3a	$a$	Shell Radius
3b	$h$	Shell Thickness
3c	$\rho_s$	Shell Density
3d	$\nu$	Shell Poisson's Ratio
3e	$E$	Shell Young's Modulus
3f	$\eta$	Shell Damping (or Loss) Factor
4a	$\rho_f$	Exterior Fluid Density
4b	$c_f$	Exterior Fluid Sound Speed
5a	$\rho_g$	Interior Fluid Density (= 0 for evacuated shell)
5b	$c_g$	Interior Fluid Sound Speed (= 0 for evacuated shell)
6a	IPRINT	Print Control Flag (0-5) 0 = Minimal Printing 1 = Normal Printing 2 = Above Normal Printing 3 = Low Debug Printing 4 = Debug Printing 5 = Excess Printing
6b	NFLAG	Calculation Type Flag (1-5) 1 = Surface Pressure Calculation Only 2 = Surface and Field Pressure Calculation 3 = Field Pressure Calculation Only 4 = Surface Pressure and Radial Velocity Calculation 5 = Surface Radial Velocity Calculation Only

Table 1 (Continued)

Record No.	Parameter	Description
7a	NRADII	Number of Observation Radii: If NFLAG = 2 or 3
7b	$r$	Radii Array: 1 to NRADII: If NFLAG = 2 or 3
8a	NTERMS	Maximum Number of Series Terms
8b	ERRCRT	Relative Error Criterion
9a	$\theta_1$	Starting Angle from z-Axis
9b	$\Delta\theta$	Delta Angle from z-Axis ( $> 0$ )
9c	$\theta_2$	Ending Angle from z-Axis ( $> \theta_1$ )
10	NINT	Number of Frequency Runs
11a	$\Omega_1$	Starting Dimensionless Frequency: 1 to NINT
11b	$\Delta\Omega$	Delta Dimensionless Frequency: 1 to NINT
11c	$\Omega_2$	Ending Dimensionless Frequency: 1 to NINT

samples, which is the number of  $ka$ 's times the number of angles, is 2001.

Program SCATSPHERE2 produces two output files, tapes 11 and 13. Tape 11 contains the input data and a table for the nondimensional pressure  $|p_r/p_{0a}|$ . Tape 13 contains an input X-Y PLOT file which is processed by the LOGOS graphics program.<sup>21</sup> Tape 12 is not currently used, but the user could restore this tape for input to F.M. Henderson's program PLOTTER, which creates polar plots and uses the software DISSPLA.<sup>22</sup> For more information on using program PLOTTER, see F.M. Henderson's report on RADSPHERE.<sup>12</sup>

The setup for a typical CRAY COS job is as follows:

```
JOB,JN=SCAT,T=60,US=BUDGET. RUN SCATSPHERE
ACCOUNT,AC=xxxxxxxxxx,US=yyyy,UPW=zzzz.
CFT77,ON=AXM.
ACCESS,DN=IMSL,OWN=PUBLIC.
SEGLDR,CMD='LIB=IMSL',GO.
```

```

DISPOSE, DN=TAPE11, ...
DISPOSE, DN=TAPE12, ...
DISPOSE, DN=TAPE13, ...
/EOF
PROGRAM SCATSPHERE2
.
.
.
END
/EOF
Forward Scattering for a Fluid-Filled Steel Spherical Shell
20.
5., 0.15, 7669., 0.3, 2.07E11, 0.01
1000., 1524.
1000., 1524.
1, 3
1, 100.
80, 1.E-8
0., 30., 0.
1
0.005, 0.005, 5.0

```

Note that the code is in FORTRAN-77 and IMSL library is used for the Bessel function calculations.

## NUMERICAL EXAMPLES

In this section, we present three examples: underwater plane-wave scattering by (1) an evacuated steel spherical thin shell, (2) a fluid-filled steel spherical thin shell, and (3) a fluid-filled aluminum spherical thin shell. The first two examples verify the updated NASHUA procedure, while the last example is compared to and agrees with the numerical implementation of the thick shell theory<sup>14</sup> and experimental results.<sup>15-16</sup> All the examples are for a thin shell; that is, the shell thickness is no more than 5% of the shell radius.

### EVACUATED STEEL SPHERICAL THIN SHELL

This example was used to verify the updated NASHUA procedure and to check program SCATSPHERE2. Figure 1 illustrates the time-harmonic plane-wave scattering problem for a submerged evacuated steel spherical thin shell. Table 2 presents the shell and fluid properties used. Note that the interior fluid properties  $\rho_g$  and  $c_g$  are set to zero.

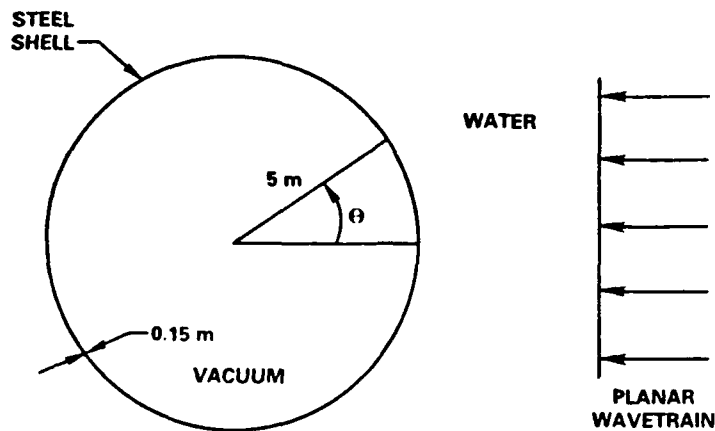


Fig. 1. Plane-wave scattering from a submerged elastic spherical shell.

Table 2. Spherical thin steel shell and fluid properties.

Description	Parameter	Measurement
Shell Radius	$a$	5.0 m
Shell Thickness	$h$	0.15 m
Shell Density	$\rho_s$	7669.0 kg/m <sup>3</sup>
Shell Poisson Ratio	$\nu$	0.3
Shell Young Modulus	$E$	2.07 x 10 <sup>11</sup> N/m <sup>2</sup>
Shell Loss Factor	$\eta$	0.0
Exterior Fluid Density	$\rho_f$	1000 kg/m <sup>3</sup>
Exterior Fluid Sound Speed	$c_f$	1524 m/s
Interior Fluid Density	$\rho_g$	0 kg/m <sup>3</sup>
Interior Fluid Sound Speed	$c_g$	0 m/s

For the NASHUA finite element model, 40 axisymmetric conical shell elements spanning the 180 degrees between the two spherical poles were used, making a total of 205 structural and 41 fluid degrees of freedom (DOF) for the exterior fluid. The nondimensional frequency increment was about  $ka = 0.02$  in the range  $0 < ka < 5$ .

For SCATSPHERE2: the maximum number of series terms NTERMS = 40, the relative error criterion ERRCRT =  $1.0 \times 10^{-4}$ , NFLAG = 3 for the field pressure calculation only, NINT = 1 for one frequency sweep, and NRADII = 1 for one radial observation per frequency point. The nondimensional frequency range  $0 < ka < 5$  was swept using a frequency increment of  $ka = 0.005$ .

Figures 2 and 3 show the forward and back-scattered pressure fields vs. the dimensionless frequency. These figures show that NASHUA and SCATSPHERE2 agree well.

#### FLUID-FILLED STEEL SPHERICAL THIN SHELL

This example was used to verify the updated NASHUA procedure. The properties are the same as in the previous example except that the spherical thin shell is filled with the same interior fluid as the exterior fluid.

For the NASHUA finite element model, 40 axisymmetric conical shell elements spanning the 180 degrees between the two spherical poles were used, making a total of 205 structure DOF and 41 fluid DOF for each fluid region. The nondimensional frequency increment was about  $ka = 0.05$  in the nondimensional frequency range  $0 < ka < 5$ .

For SCATSPHERE2, the maximum number of series terms NTERMS = 80, the relative error criterion ERRCRT =  $1.0 \times 10^{-8}$ , NFLAG = 3 for the field pressure calculation only, NINT = 1 for one frequency sweep, and NRADII = 1 for one radial observation per point. Note that NTERMS and ERRCRT are different from those used in the evacuated case. These changes were made to avoid convergence problems due to the Bessel function computations. The nondimensional frequency increment was  $ka = 0.005$  in the nondimensional frequency range  $0 < ka < 5$ .

Figures 4 and 5 plot the frequency response of the forward and back-scattered nondimensional pressure  $|p_r / p_{0a}|$ . Both figures show very good agreement between the two approaches.

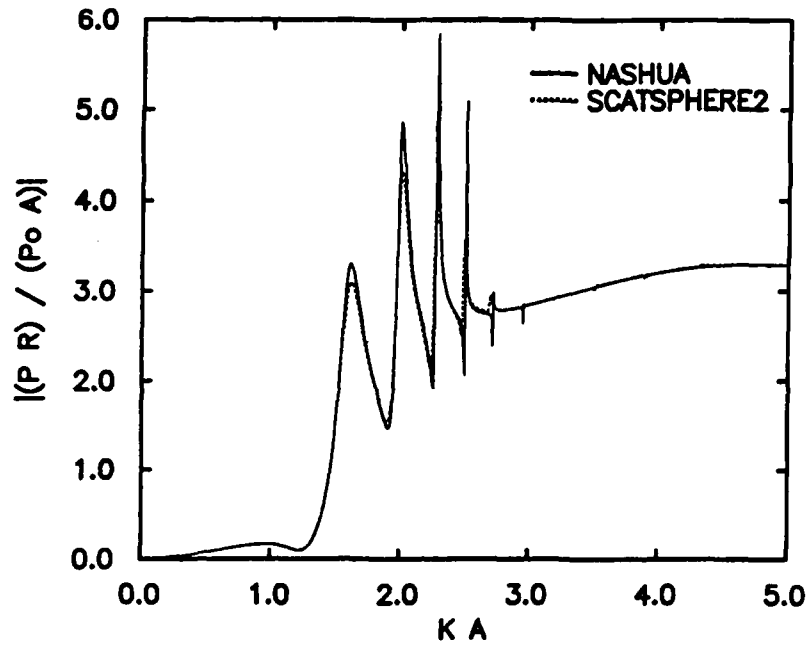


Fig. 2. Forward-scattered field by a submerged evacuated steel spherical shell.

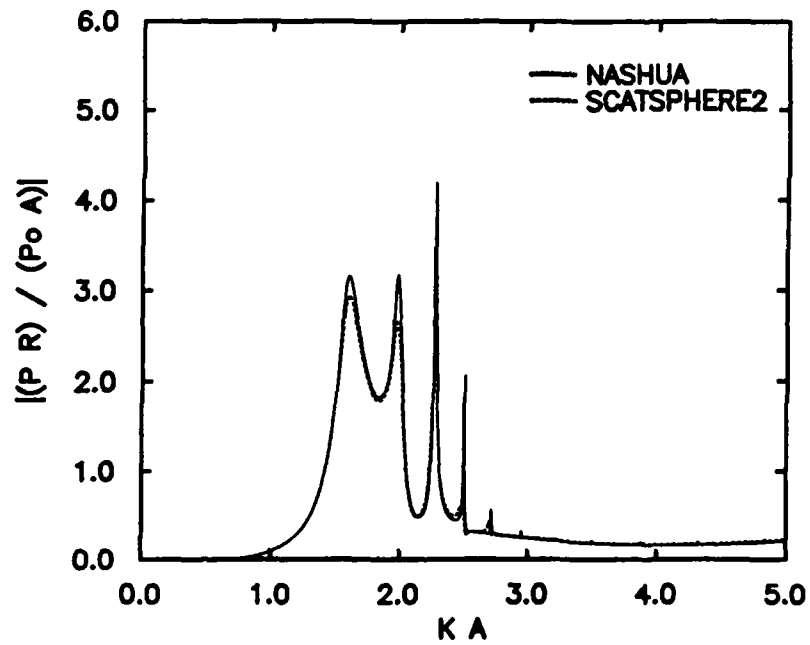


Fig. 3. Back-scattered field by a submerged evacuated steel spherical shell.

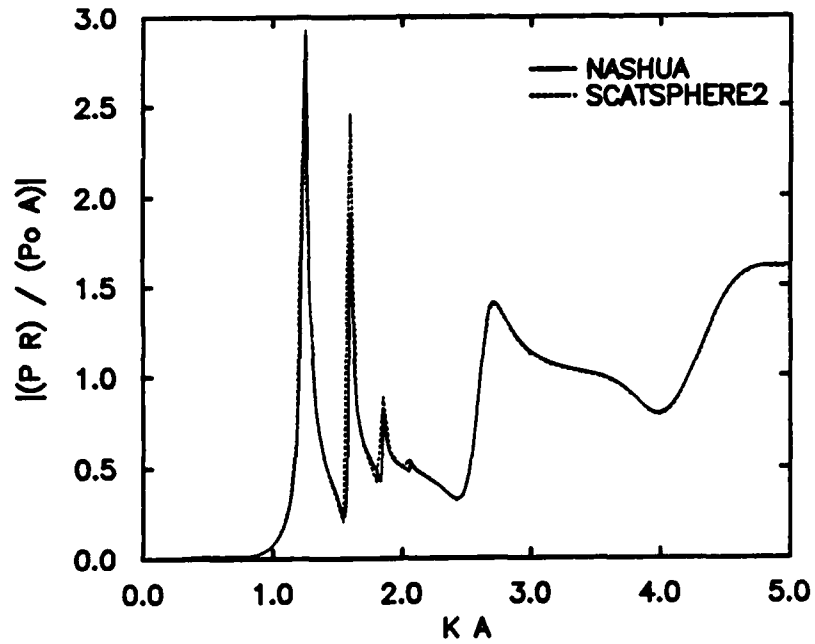


Fig. 4. Forward-scattered field by a submerged fluid-filled steel spherical shell.

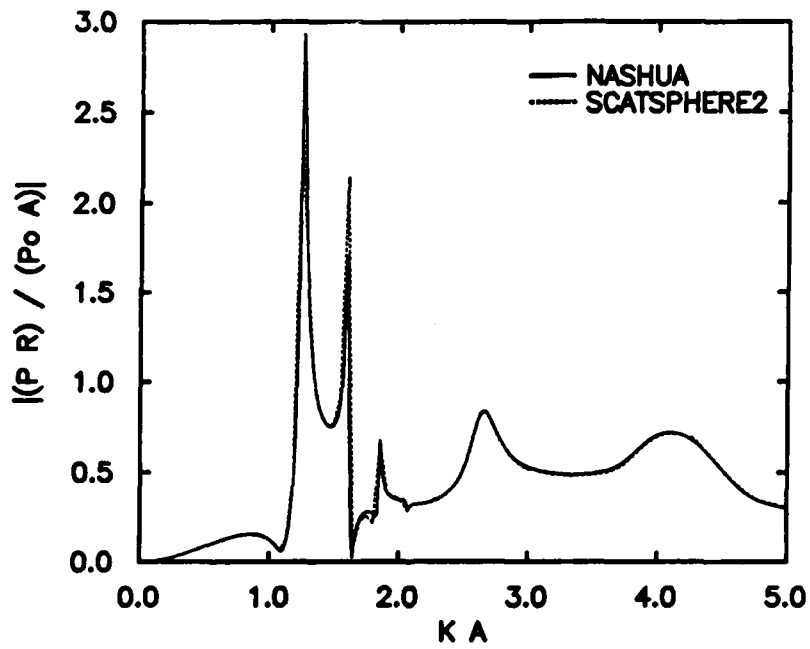


Fig. 5. Back-scattered field by a submerged fluid-filled steel spherical shell.

Table 3. Spherical thin aluminum shell and fluid properties.

Description	Parameter	Measurement
Shell Radius	$a$	4.875 m
Shell Thickness	$h$	0.25 m
Shell Density	$\rho_s$	2700.0 kg/m <sup>3</sup>
Shell Poisson Ratio	$\nu$	0.355
Shell Young Modulus	$E$	0.675 x 10 <sup>11</sup> N/m <sup>2</sup>
Shell Loss Factor	$\eta$	0.00
Exterior Fluid Density	$\rho_f$	1000 kg/m <sup>3</sup>
Exterior Fluid Sound Speed	$c_f$	1410 m/s
Interior Fluid Density	$\rho_g$	1000 kg/m <sup>3</sup>
Interior Fluid Sound Speed	$c_g$	1410 m/s

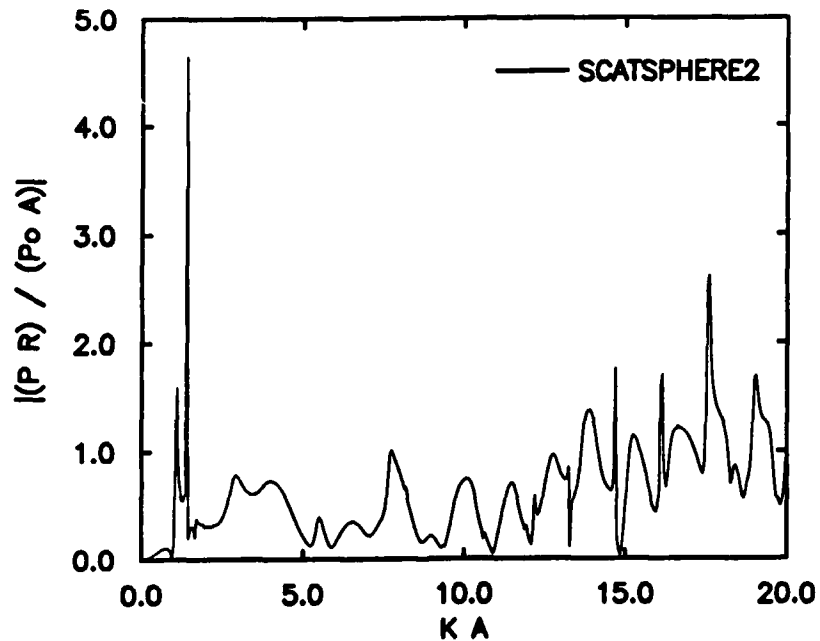


Fig. 6. Back-scattered field by a submerged fluid-filled aluminum spherical shell.

## FLUID-FILLED ALUMINUM SPHERICAL THIN SHELL

This example was chosen from earlier papers<sup>14-15</sup> as another example to verify program SCATSPHERE2. Table 3 presents the shell and fluid properties used. Figure 6 gives the back-scattered nondimensional absolute pressure for nondimensional frequency  $0 < ka < 20$ . Close comparison of Fig. 6 with Fig. 4b in Hickling's paper<sup>14</sup> and Fig. 1b in Diercks's and Hickling's paper,<sup>15</sup> shows good agreement between SCATSPHERE2 computations and results obtained by Diercks and Hickling.

### SUMMARY

Program SCATSPHERE2 solves the time-harmonic plane-wave scattering by a submerged elastic spherical thin shell that is evacuated or fluid-filled. This code was verified by the updated NASHUA procedure,<sup>24</sup> the numerical implementation of the thick shell theory,<sup>14</sup> and experimental results.<sup>15</sup> This program allows the user to examine the surface velocity at any angle, and the scattered pressure field at any radius and angle. Program SCATSPHERE2 also allows the user to create polar plots, tables, and X-Y plots. In the future, program SCATSPHERE2 could be modified for other axisymmetric problems such as including additional interior mediums (solids and/or fluids) or spherical-wave scattering.

### ACKNOWLEDGMENTS

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