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THESIS

CALCULATION OF THE TRANSITION MATRIX
FOR THE SCATTERING OF ACOUSTIC WAVES
FROM A THIN ELASTIC SPHERICAL SHELL
USING THE ATILA FINITE ELEMENT CODE

by

Arthur Lobo da Costa Ruiz

March, 1994

Thesis Advisors:

Clyde L. Scandrett
Steven R. Baker

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**CALCULATION OF THE TRANSITION MATRIX FOR THE
SCATTERING OF ACOUSTIC WAVES FROM A THIN ELASTIC
SPHERICAL SHELL USING THE ATILA FINITE ELEMENT CODE**

by

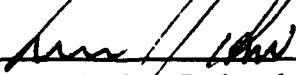
Arthur Lobo da Costa Ruiz
Lieutenant Commander, Brazilian Navy
BS, Brazilian Naval Academy, Rio de Janeiro, 1978
MS, University of Rio de Janeiro, 1992

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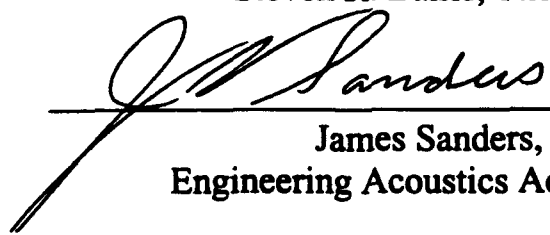
Author: _____


Arthur Lobo da Costa Ruiz

Approved By: _____


Clyde L. Scandrett, Thesis Co-Advisor


Steven R. Baker, Thesis Co-Advisor


James Sanders, Chairman
Engineering Acoustics Academic Committee

ABSTRACT

The transition matrix, relating the scattered and incident acoustic waves for a thin elastic spherical shell in a free-field environment, has been evaluated using the ATILA finite-element code. A three-dimensional finite-element model of a 0.5-m outer radius, 1-cm thick spherical steel shell surrounded by water was developed. The ATILA code was used to calculate the scattered pressure over the surface of the shell for incident waves represented as products of radial Hankel functions and spherical harmonics. The chosen driving frequency was 474 Hz, corresponding to a value of $ka=1$, where k is the wavenumber of sound in water and a is the radius of the shell. The ATILA results were compared with the results of analytical thin shell theory, and were found to agree for a model which divided the spherical shell surface into 72 approximately equal area triangular regions. Also computed for each component was the modal acoustical impedance of the shell. These results agreed within two percent for the zeroth order component and thirteen percent for the first order components.

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I. INTRODUCTION

The utilization of sonar systems in operations at sea depends on the ability to forecast their performance during the design and production phases of their development. For low frequency active sonar systems, arrays of large transducers are required to produce the necessary power output. Because of their size and power output, interactions among the elements of an array may need to be considered in modelling a given array.

Professors S. R. Baker of the Physics Department and C. L. Scandrett of the Mathematics Department of the Naval Postgraduate School are engaged in a joint research effort with the objective of developing the models necessary to predict the performance of arbitrary dense active sonar arrays.

The approach used in this research is based on the T-Matrix method. [Ref. 3].

The T-matrix method uses the superposition of spherical harmonics to represent the total radiated pressure from a transducer. The radiated pressure from one such transducer for each incoming harmonic will be

calculated in an effort to produce the transition matrix or T-matrix. The elements of the T-matrix may then be used to calculate scattered pressures resulting from the interaction between transducers.

The T-matrix method is useful in computing the scattered pressure resulting from an arbitrary incident pressure on a given scatterer. This thesis is concerned with the application of the T-matrix method on an array of thin-shelled elastic spheres.

Once the theoretical results from scattering of spherical harmonics incident on a sphere have been obtained, the calculations from the T-matrix method can be compared for each harmonic with the computation from the Finite Element Code ATILA for the purpose of validation.

Furthermore, the acoustic impedance calculated from the thin shell theory (Chapter II) can be compared with the acoustic impedance computed by ATILA's results.

The remainder of this thesis is divided into five chapters. Chapter II describes the theory involved in the T-Matrix method, the Pritchard Approximation, the finite element analysis of a structure excited by an impinging wave, and the theory of forced vibration of a spherical shell. Chapter III describes the spherical model used in the ATILA code. Chapter

IV presents and discusses the results. Chapter V presents the conclusions. Appendix A contains a copy of the FORTRAN program used to generate the spherical harmonic impinging wave. Appendix B presents a copy of the code used in ATILA. Appendix C presents a FORTRAN code [Ref. 17] to calculate elements of the Transition Matrix.

II. THEORY

A. "T-MATRIX" METHOD

The "T-Matrix" method is a procedure for computing the acoustic field due to multiple radiating and/or scattering bodies. An outline is given below of how this method may be applied to compute the acoustic pressure due to a pair of piezoelectric spherical shell transducers.

If a voltage is applied to a thin piezoelectric spherical shell in a free-field environment, a displacement of the surface of the shell can be obtained by the canonical equations [Ref. 1]. This movement generates an acoustic pressure field in the surrounding medium. This is indicated in Figure 1 , where a sphere excited by a voltage V produces a radial velocity u over the surface of the sphere. The perturbation of the sphere on the environment yields a radiated pressure $p^R(r,\theta,\phi)$.

The acoustic pressure at a field point z , located at spherical coordinates r , θ , and ϕ , equals $p^R(r,\theta,\phi)$, since the sphere is in a free-field environment. Numerical values of $p^R(r,\theta,\phi)$ can be found by application of a finite-element analysis code such as ATILA [Ref. 2].

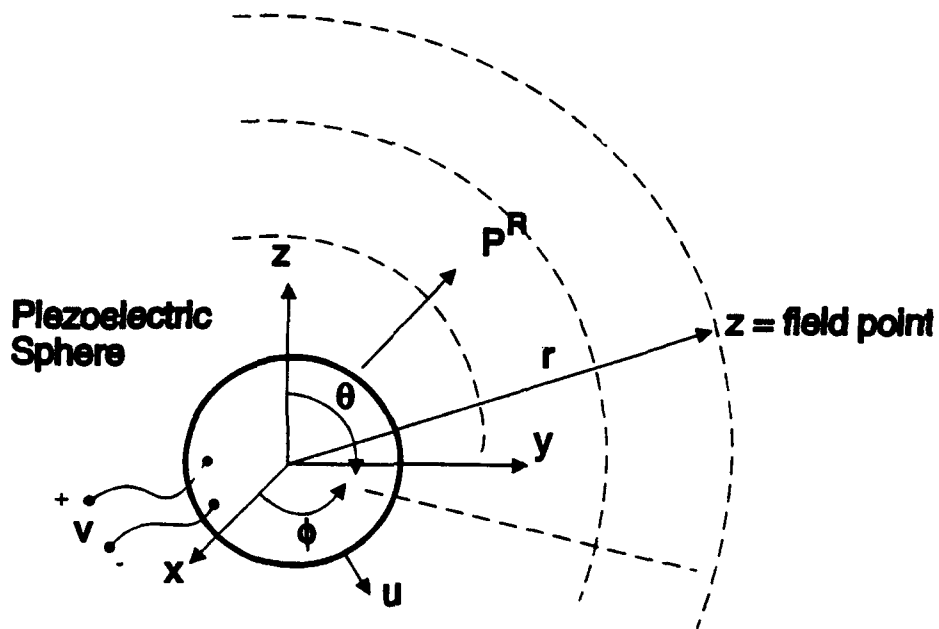


Figure 1. Single Radiation from a Piezoelectric Sphere.

Next, consider the case of an array of two thin piezoelectric spherical shell transducers that are close enough to be considered "in the near field" of one another (see Figure 2).

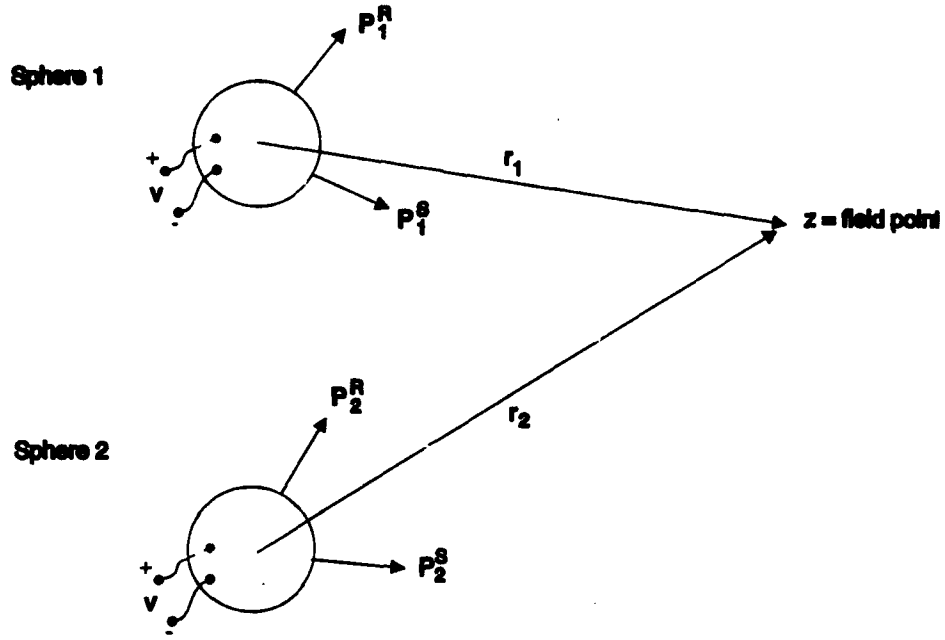


Figure 2. An Array of Two Thin Piezoelectric Spheres.

Outgoing waves at sphere 1 are composed of a radiated wave, p_1^R , resulting from the applied voltage V , and a scattered wave, p_1^S , due to the incident wave from sphere 2, p_1^I . The scattered pressure field can be mathematically represented by outgoing spherical Hankel functions and surface harmonics in the sphere 1 coordinate system as [Ref. 3]

$$p_1^S(r_1, \theta_1, \phi_1) = \sum_{n=0}^{\infty} \sum_{m=-n}^n P_{1nm}^S h_1^{(2)}(kr_1) P_n^m(\cos \theta_1) e^{im\phi_1} \quad (1)$$

where $h_n^{(2)}(kr_1) = j_n(kr_1) - iy_n(kr_1)$ is the Hankel function of the second kind (note that an $e^{i\omega t}$ harmonic time dependence has been assumed; for $e^{-j\omega t}$ harmonic time dependence $h_n^{(1)}(kr_1) = j_n(kr_1) + iy_n(kr_1)$ must be used instead

of $h^{(2)}$, P_{lnm}^S is the amplitude of the n , m th scattered wave component, k is the radial wave number, $P_n^m(\cos \theta_1)$ is the associated Legendre function, and

r_1 , θ_1 and ϕ_1 are the spherical coordinates of the field point z with origin at center of sphere 1. The radiated pressure field is similarly represented:

$$p_1^R(r_1, \theta_1, \phi_1) = \sum_{n=0}^{\infty} \sum_{m=-n}^n P_{lnm}^R h_n^{(2)}(kr_1) P_n^m(\cos \theta_1) e^{im\phi_1} \quad (2)$$

The incident pressure p_1^I is assumed to be of the form:

$$p_1^I(r_1, \theta_1, \phi_1) = \sum_{n=0}^{\infty} \sum_{m=-n}^n P_{lnm}^I j_n(kr_1) P_n^m(\cos \theta_1) e^{im\phi_1} \quad (3)$$

This form is valid provided r_1 is less than the distance between the origins of spheres 1 and 2.

By definition, the coefficients of the scattered pressure P_{lnm}^S are related to the coefficients of the incident pressure P_{lnm}^I by

$$\{P_1^S\} = [T] \{P_1^I\} \quad (4)$$

Where $\{\}$ denotes a column vector of coefficients and $[T]$ is the so-called "transition matrix" or T-Matrix, a property of the transducer.

Similar expressions to the above apply to the corresponding fields on sphere 2, i.e.,

$$p_2^S(r_2, \theta_2, \phi_2) = \sum_{n=0}^{\infty} \sum_{m=-n}^n P_{2nm}^S h_n^{(1)}(kr_2) P_n^m(\cos \theta_2) e^{im\phi_2} \quad (5)$$

$$\{P_2^S\} = [T] \{P_2^I\} \quad (6)$$

Note that it is assumed here for the sake of simplicity that the two transducers are identical, so that their T-matrices are also; in the general case this need not be so.

To proceed further it is necessary to be able to transform from one coordinate system into another. This is done using an "addition theorem", by which a spherical wave relative to one origin is expressed as a series of spherical waves relative to another [Ref. 3 and references therein]. Thus, for example, the incident pressure on sphere 1 can be expressed as the sum of the radiated and scattered pressures from sphere 2, transformed into the sphere 1 coordinate system using the addition theorem. In matrix form, this can be written

$$\{P_1^I\} = [G_{21}]\{P_2^R + P_2^S\}, \quad (7)$$

where $[G_{21}]$ represents the matrix of coefficients that transforms outgoing spherical harmonics at sphere 2 into standing wave spherical harmonics at sphere 1. Similarly, the incident pressure on sphere 2 is related to the radiated and scattered pressure on sphere 1 by

$$\{P_2^I\} = [G_{12}]\{P_1^R + P_1^S\}. \quad (8)$$

Using Equations (4) and (6) in Equations (7) and (8), we have a system of two equations in two unknowns:

$$\{P_1^S\} = [T][G_{21}]\{P_2^R + P_2^S\}, \quad (9)$$

$$\{P_2^S\} = [T][G_{12}]\{P_1^R + P_1^S\}. \quad (10)$$

Equations (9) and (10) may be written

$$\begin{bmatrix} I & -TG_{21} \\ -TG_{12} & I \end{bmatrix} \begin{Bmatrix} P_1^S \\ P_2^S \end{Bmatrix} = \begin{bmatrix} 0 & TG_{21} \\ TG_{12} & 0 \end{bmatrix} \begin{Bmatrix} P_1^R \\ P_2^R \end{Bmatrix} \quad (11)$$

where $[I]$ is the identity matrix.

If $\{P_1^R\}$, $\{P_2^R\}$, and $[T]$ can be found, such as by using the finite-element code ATILA, then, since $[G_{12}]$ and $[G_{21}]$ are known from the positions and orientations of the transducers through application of the addition theorem, the scattered pressures $\{P_1^S\}$ and $\{P_2^S\}$ can be determined. Then the resulting pressure at the field point z , $p(z)$, is given by

$$p(z) = p_1^R(z_1) + p_1^S(z_1) + p_2^R(z_2) + p_2^S(z_2) \quad , \quad (12)$$

with the forms for the radiated and scattered pressure fields given previously, where z_i represents the coordinates of the field point z relative to the origin of the i th sphere.

The above development may be generalized to an array composed of an arbitrary number N of identical elements. For the i th element,

$$\{P_i^S\} = [T]\{P_i^I\}. \quad (13)$$

Here the scattered pressure from the i th transducer is represented. The components of the radiated pressure, P_{imn}^R , may be found using such as ATILA finite-element code. Then the acoustic pressure at a field point z is the sum of the radiated and scattered pressures of each transducer.

$$p(z) = \sum_{i=1}^N [p_i^R(z_i) + p_i^S(z_i)] \quad (14)$$

B. THE "PRITCHARD APPROXIMATION"

The analytical calculation of mutual-radiation impedance, Z_{ij} , is made through the solution of the wave equation for the sound pressure produced by one transducer and by integrating that pressure over the radiating surface of another transducer [Ref. 4].

In Pritchard's original paper [Ref. 5], the mutual-radiation impedance was approximated for the case of flat circular pistons on an infinite rigid baffle. For small pistons and large separation, the Pritchard equation can be reduced to

$$Z_{12} = \frac{1}{2}(ka)^2 \left(\frac{\sin kd}{kd} + i \frac{\cos kd}{kd} \right) \rho c A. \quad (15)$$

Here an $e^{-i\omega t}$ harmonic time dependence is assumed. This approximation neglects secondary scattered pressures.

It can be noted that Z_{12} has a real and an imaginary part,

$$Z_{12} = R_{12} + jX_{12}. \quad (16)$$

Here, Z_{12} is the mutual-radiation impedance between transducers 1 and 2, d is the distance between the centers of two transducers (see Figure 3), a is the piston radius, k is the acoustic wave number, ρc is characteristic impedance of the environment, and A is the area of the pistons.

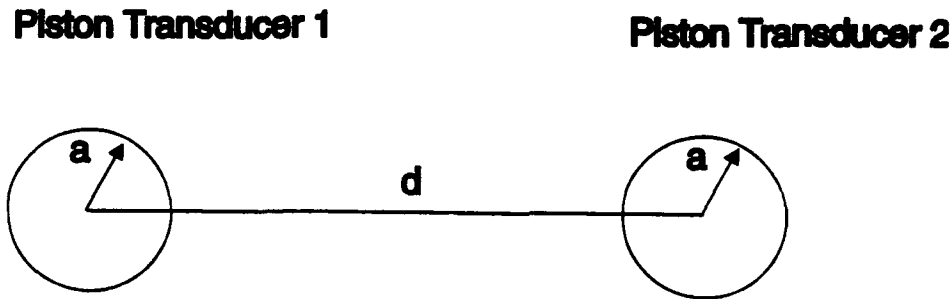


Figure 3. The Circular Pistons derived by Pritchard.

The self radiation resistance R_{ii} , for small ka , can be approximated by

$$R_{ii} = \frac{1}{2}(ka)^2 \rho c A. \quad (17)$$

The self radiation resistance is the real part of the transducer self-radiation impedance, written

$$\text{Re}(Z_{\text{self}}) = \frac{1}{2}(ka)^2 \rho c A. \quad (18)$$

Therefore, for small ka , the mutual-radiation impedance can be expressed as

$$Z_{12} = \text{Re}(Z_{\text{self}}) \left(\frac{\sin kd}{kd} + i \frac{\cos kd}{kd} \right) = \text{Re}(Z_{\text{self}}) \frac{e^{jkd}}{kd}. \quad (19)$$

The so-called "Pritchard Approximation" refers to the application of Equation (19) to find the mutual radiation impedance of an array of identical transducers, not restricted to baffled pistons, i.e., Equation (19) has been applied to volumetric arrays. To calculate the mutual-radiation impedance, the wave number, the distance between transducers, and the self-radiation resistance of the transducer must be known in order to apply Pritchard's approximation.

C. THE ATILA CODE AND THE HARMONIC ANALYSIS

1. The Code

ATILA is a finite element code developed at Institut Supérieur d'Electronique du Nord (ISEN) in France, and is intended for finite element analysis of underwater transducers. It uses the variational formulation of the governing mathematical equations of fluid-structure interactions [Refs. 6,7,8,9,10].

ATILA can solve a variety of problems, including: Static Analysis, Modal Analysis, and Harmonic Analysis.

Static Analysis yields the displacement field, the pressure field, and the electrical potential of an elastic or piezoelectric structure. Modal Analysis computes the eigenfrequencies and normal modes of an elastic, piezoelectric or magnetostrictive structure. Harmonic Analysis corresponds to a forced vibration problem where the excitation of the structure comes from an incident wave or from the voltage applied across the electrical terminals at a prescribed frequency. The analysis can be for a radiation or a scattering problem, for the given frequency in which the displacement field, electrical potential, electrical impedance of the structure, pressure field,

reflection and transmission coefficients, and transmitting voltage response are found.

For scattering problems, an incident wave is defined by the user. A default function is provided with ATILA which creates a plane wave traveling in the negative x axis direction. One can excite the structure with an arbitrary incident wave by adding a proper function at the end of the main FORTRAN program. Appendix A contains the FORTRAN program used to generate the incident spherical harmonic waves on the structure of the transducer.

ATILA calculates either the resulting "total pressure" or only the "scattered pressure." The total pressure, for the code, is the sum of the pressure generated by the electrical potential and the pressure of the scattered wave, plus the pressure of the impinging wave. In the case of scattered pressure, the pressure of the incident wave is not included .

The ATILA library has 46 different types of elements, used to model elastic, composite, piezoelectric, magnetostrictive, and magnetic materials as well as fluids, solid-fluid interfaces, and radiation dampers. Most of the elements use the same polynomial (quadratic) interpolation for both geometry and field variation (isoparametric elements).

2. Harmonic Analysis of an Elastic or Piezoelectric Structure

Excitation by an Impinging Wave

ATILA transforms equations of the Harmonic Analysis of a Radiating Piezoelectric Transducer into a matrix form. The motion equation is used for piezoelectric and elastic structures, Poisson's equation is used for piezoelectric material, and Helmholtz's equation is used for fluids [Ref. 2,11].

On the interfaces between the structure and the fluid, kinematics and dynamic continuity conditions hold under the assumption of no fluid cavitation at the fluid-structure interface. A radiation condition is applied to the external fluid boundary. Furthermore, to represent the condition of an impinging wave, the pressure and the flux fields are separated into incident and scattered parts.

In ATILA, the governing equations are written in matrix form as

$$\begin{bmatrix} [K_{uu}] - \omega^2[M] & [K_{u\phi}] & -[L] \\ [K_{u\phi}]^T & [K_{\phi\phi}] & [0] \\ -[L]^T & [0]^T & \frac{[H]}{A} - \frac{[M]}{B} \end{bmatrix} \begin{bmatrix} U \\ \Phi \\ P_{es} \end{bmatrix} = \begin{bmatrix} F - [L]P_i \\ -Q \\ \frac{1}{A}[G]P_{es} + \frac{\psi_i}{\rho\omega^2} + \left(\frac{[H]}{A} - \frac{[M]}{B}\right)P_i \end{bmatrix} \quad (20)$$

where:

U	:	vector of the nodal values of the components of the displacement field
Φ	:	vector of the nodal values of the electrical potential
P_i	:	vector of the nodal values of the incident pressure field
P_s	:	vector of the nodal values of the scattered pressure field
F	:	vector of the nodal values of the applied forces
Q	:	vector of the nodal values of the electrical charges
Ψ_i	:	vector of the nodal values of the integrated normal derivative of the incident pressure on the surface boundary S (the externally applied pressure field is proportional to the externally applied flux)
K_{uu}	:	stiffness matrix
$[K_{u\phi}]$	:	piezoelectric matrix
$[K_{\phi\phi}]$	:	dielectric matrix
$[M]$	:	consistent mass matrix
$[M_f]$	:	consistent fluid (pseudo-) mass matrix
$[H]$	:	fluid (pseudo-) stiffness matrix
$[L]$	:	coupling matrix at the fluid structure interface (connectivity matrix)
$[G]$	:	complex linear operator that is frequency dependent
$[0]$	:	zero matrix
ω	:	angular frequency
ρ	:	fluid density
c	:	fluid sound speed
A	:	constant of the material
B	:	constant of the material
$[]^T$	:	the superscript T means the matrices transpose.

The incident flux field can be expressed with the nodal values of the incident pressure normal derivative,

$$\psi_i = [D] \frac{\partial p_i}{\partial n}. \quad (21)$$

Here, $[D]$ is a matrix that the code dispenses for the damping elements.

The constants A and B are

$$A = \rho^2 c^2 \omega^2 \quad (22)$$

and

$$B = \rho^2 c^2. \quad (23)$$

Note that ATILA assumes a harmonic time dependance of $e^{j\omega t}$. The incident pressure is provided by the user through a FORTRAN program (see Appendix A), and for each input frequency the code outputs the complex displacement, the complex pressure, rotational and electric potential fields at each node, and the complex electrical impedance and admittance. For a more detailed discussion of ATILA's operation, the reader is advised to consult the manual [Ref. 2].

D. FORCED VIBRATION OF A SPHERICAL SHELL.

1. Theory

The mathematical equations governing the true vibrations and deformation of thin elastic shells were first derived by Love in 1888 [Ref. 12]. It assumes the following postulates:

- ♦ The shell is thin, compared to the smallest radius of curve of shell
- ♦ Deflections of the shell are small, relative to the shell's thickness
- ♦ There is no transverse normal stress acting on planes parallel to the middle surface of shell
- ♦ There are no changes to the normals of the reference surface after and no changes in length during deformation

The equations are

$$u_{\theta\theta} + u_{\theta}\cot\theta + (1+\nu)w_{\theta} - u\cot^2\theta - \nu u - \frac{1}{2}(3-\nu)\frac{\cos\theta}{\sin^2\theta}v_{\phi} + \frac{1}{2}(1+\nu)\frac{1}{\sin\theta}v_{\theta\phi} + \frac{1}{2}(1-\nu)\frac{1}{\sin^2\theta}u_{\phi\phi} = \frac{a^2}{C_p^2}u_{tt}, \quad (25)$$

$$\frac{1}{2}(1-\nu)[v_{\theta\theta} + v_{\theta}\cot\theta] + \frac{1}{2}(1-\nu)(2 - \csc^2\theta)v + (1+\nu)\frac{1}{\sin\theta}w_{\theta} + \frac{1}{2}(3-\nu)\frac{\cos\theta}{\sin^2\theta}u_{\phi} + \frac{1}{\sin^2\theta}v_{\phi\phi} + \frac{1}{2}(1+\nu)\frac{1}{\sin\theta}u_{\phi\theta} = \frac{a^2}{C_p^2}v_{tt}, \quad (26)$$

and

$$-(1 + \nu)[u_\theta + u \cot \theta + \frac{1}{\sin \theta} v_\phi + 2w] + \frac{(1-\nu^2)}{Eh} a^2 \sigma_a = \frac{a^2}{C_p^2} w_{tt}, \quad (27)$$

where the subscripts θ , ϕ , and t indicate partial differentiation and

- ν : Poisson's ratio.
- h : shell thickness.
- C_p : shell's "plate" velocity.
- E : Young's modulus.
- σ_a : outward normal stress applied to shell.
- a : spherical shell radius.
- θ, ϕ, ρ : spherical coordinates (see Figure 1).
- u, v, w : components of displacement (see Figure 4).

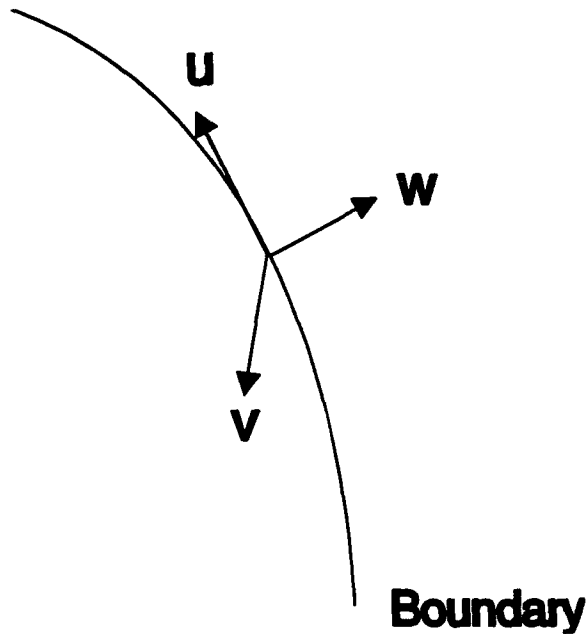


Figure 4. Components of Displacement.

To calculate the shell's plate velocity, C_p ,

$$C_p = \sqrt{\frac{E}{\rho_s(1-\nu^2)}} \text{ m/s} \quad (27)$$

is used [Ref.1].

Here, ρ_s is the density of shell's material.

By a series of manipulations detailed in Love's work, u and v are eliminated from the three equations, resulting in

$$w_{\theta\theta} + \cot \theta w_\theta + \left[\frac{(\Omega^2 + 1 - \nu)(\Omega^2 - 2(1 + \nu))}{\Omega^2 - (1 - \nu^2)} - \frac{m^2}{\sin^2 \theta} \right] w =$$

$$\frac{-a^2}{Eh[\Omega^2 - (1 - \nu^2)]} \left\{ \sigma_{a\theta\theta} + \sigma_{a\theta\theta} \cot \theta + (\Omega^2 + 1 - \nu - \frac{m^2}{\sin^2 \theta}) \sigma_a \right\}, \quad (28)$$

where the frequency dependent term, Ω , is

$$\Omega = \frac{\omega a}{C_p}, \quad (29)$$

and all ϕ dependence is of the form $e^{im\phi}$. A harmonic time dependence of the form $e^{i\omega t}$ has been assumed.

Here,

$$\omega = 2\pi f, \quad (30)$$

and f is the excitation frequency.

If the new independent variable η is introduced:

$$\eta = \cos \theta. \quad (31)$$

The derivations with respect to θ can be replaced by

$$\frac{\partial}{\partial \theta} = -\sqrt{1-\eta^2} \frac{\partial}{\partial \eta}, \quad (32)$$

and

$$\frac{\partial^2}{\partial \theta^2} = -\eta \frac{\partial}{\partial \eta} + (1-\eta^2) \frac{\partial^2}{\partial \eta^2}. \quad (33)$$

Writing Equation (28) in terms of (32) and (33):

$$(1-\eta^2)w_{\eta\eta} - \eta w_{\eta} + \frac{\eta}{\sqrt{1-\eta^2}}(-\sqrt{1-\eta^2} w_{\eta})$$

$$+ \left[\frac{(\Omega^2+1-v)(\Omega^2-2(1+v))}{\Omega^2-(1-v^2)} - \frac{m^2}{1-\eta^2} \right] w =$$

$$\frac{-a^2(1-v^2)}{Eh[\Omega^2-(1-v^2)]} \times$$

$$\left[-(1-\eta^2)\sigma_{a\eta\eta} - \eta\sigma_{a\eta} + \frac{\eta}{\sqrt{1-\eta^2}}(-\sqrt{1-\eta^2}\sigma_{a\eta}) + \left\{ (\Omega^2+1-v) - \frac{m^2}{1-\eta^2} \right\} \sigma_a, \right.$$

(34)

and it can be seen that w and σ_a are being operated on by Legendre's differential equation.

Representing the applied stress σ_a , and the normal component, w , in terms of the Legendre polynomial yields

$$w = \sum_{n,m} w_{nm} P_n^m(\eta) e^{im\phi} \quad (35)$$

and

$$\sigma_a = \sum_{n,m} F_{nm} P_n^m(\eta) e^{im\phi}, \quad (36)$$

where $P_n^m(\eta)$ are the associated Legendre eigenfunctions weighted by the excitation stress amplitude F_{nm} and the normal displacement amplitude w_m .

Legendre's equation can be written

$$(1 - \eta^2) \frac{\partial^2}{\partial \eta^2} P_n^m(\eta) - 2\eta \frac{\partial}{\partial \eta} P_n^m(\eta) - \frac{m^2}{1 - \eta^2} P_n^m(\eta) = -\lambda_n P_n^m(\eta), \quad (37)$$

where

$$\lambda_n = n(n + 1). \quad (38)$$

and by substitution of (36) and (37) into (40) using (39), one is left with the algebraic equation related F_{nm} to w_{nm} :

$$\left\{ \frac{(\Omega^2 + 1 - \nu) [\Omega^2 - 2(1 + \nu)]}{\Omega^2 - (1 - \nu^2)} - \lambda_n \right\} w_{nm} = \frac{-a^2}{hC_p^2 \rho_s [\Omega^2 - (1 - \nu^2)]} [\Omega^2 + 1 - \nu - \lambda_n] F_{nm} \quad (39)$$

In analogy with mechanical impedance [Ref. 13], the modal mechanical impedance, Z_n , can be defined as:

$$Z_n = \frac{F_{nm}}{i\omega w_{nm}}, \quad (40)$$

where the time dependence is given by $e^{-i\omega t}$.

It is then possible to write Equation (40) with modal mechanical impedance, as

$$Z_n = i \frac{hC_p \rho_s}{a\Omega} \left\{ \frac{[\Omega^2 - 2(1+\nu)](\Omega^2 + 1 - \nu - \lambda_n) - \lambda_n(1+\nu)^2}{\Omega^2 + 1 - \nu - \lambda_n} \right\}. \quad (41)$$

Now, for the scattering problem, the pressure directed radially inward will be the sum of the incidental spherical wave and the outgoing spherical wave.

$$F_{nm} = -\left\{ j_n(ka)I_{nm} + h_n^{(2)}(ka)R_{nm} \right\}, \quad (42)$$

where the term $j_n(ka)I_{nm}$ refers the incident spherical wave, p^i , and the term $h_n^{(2)}(ka)R_{nm}$ is the scattered spherical wave, p^s . The constants I_{nm} and R_{nm} are amplitudes of incident and scattered waves respectively.

$$p^i = \sum_{nm} I_{nm} j_n(kr) P_n^m(\eta) e^{im\phi}, \quad (43)$$

$$p^s = \sum_{nm} R_{nm} h_n^{(2)}(kr) P_n^m(\eta) e^{im\phi}, \quad (44)$$

$$w = \sum_{nm} w_{nm} P_n^m(\eta) e^{im\phi}. \quad (45)$$

Equations (43), (44), and (45) express the pressures and radial displacements in terms of spherical harmonics.

Applying Equation (42) in Equation (40) yields

$$I_{nm} j_n(ka) + R_{nm} h_n^{(2)}(ka) = -i\omega Z_n w_{nm}. \quad (46)$$

Furthermore, applying Euler's Equation [Ref. 13] at the surface of sphere,

$$\frac{\partial p}{\partial r} \Big|_{r=a} = -\rho_f \frac{\partial^2 w}{\partial t^2} \Big|_{r=a}. \quad (47)$$

where, ρ_f is the density of the fluid.

The pressure at surface, p , is the sum of the incident pressure, p^i , and the scattered pressure, p^s , as in

$$p = p^i + p^s. \quad (48)$$

Now, applying Equations (43), (44), (45) and (46) on (47),

$$I_{nm} j'_n(ka) + R_{nm} h_n^{(2)'}(ka) = \omega \rho_f C_f w_{nm}. \quad (49)$$

where $j'_n(ka)$ and $h_n^{(2)'}(ka)$ are the first derivatives of the spherical Bessel and Hankel functions.

From Equation (49),

$$w_{nm} = \frac{1}{\omega \rho_f C_f} \{ I_{nm} j'_n(ka) + R_{nm} h_n^{(2)'}(ka) \}. \quad (50)$$

Now, Equation (50) can be utilized in Equation (46), yielding

$$\rho_f C_f [I_{nm} j_n(ka) + R_{nm} h_n^{(2)}(ka)] = -iZ_n [I_{nm} j'_n(ka) + R_{nm} h_n^{(2)'}(ka)]. \quad (51)$$

Finally, we have an equation for calculation of the constant term R_{nm} ,

$$R_{nm} = - \left\{ \frac{iZ_n j'_n(ka) + \rho_f C_f j_n(ka)}{iZ_n h_n^{(2)'}(ka) + \rho_f C_f h_n^{(2)}(ka)} \right\} I_{nm} \quad (52)$$

The values of R_{nm} for each n and m will be the diagonal elements of the T-matrix of the thin spherical shell; each I_{nm} equals one.

2. Thinness Criteria

The theory of thin elastic shells is based upon the postulate that shells are thin. No exact definition of thinness is available, however.

From Junger & Feit [Ref. 14] we have a definition of "thick" plates, expressed as

$$h > \frac{\lambda_s}{20}, \quad (53)$$

where λ_s is the shear wavelength. An ad hoc treatment to account for thick shells that takes into account bending stresses introduces the parameter β ,

which is a function of the thickness, h , and the middle surface of the shell, a_m , as in

$$\beta^2 = \frac{h^2}{12a_m^2}. \quad (54)$$

A rule of thumb suggested by Kraus [Ref. 15], is that the thickness should be less than one tenth of the radius of curvature of the reference surface, i.e., $\beta^2 < 1/1200$.

On the other hand, when Kraus examines the dynamic analysis of shells [Ref. 15], he affirms that the theory based upon Love's postulate gives a reliable result in the range

$$0 < \Omega < \Omega_s, \quad (55)$$

where Ω_s is the dimensionless frequency of the first thickness shear mode:

$$\Omega_s = \frac{\omega_s a_m}{C_T}, \quad (56)$$

where

$$\omega_s = \frac{\pi C_T}{h}, \quad (57)$$

and

$$C_T = \left(\frac{E}{2(1+\nu)\rho_s} \right)^{\frac{1}{2}}. \quad (58)$$

Applying Equation (27) and (29) in Equation (55) yields

$$0 < \sqrt{\frac{\rho_s(1-v^2)}{E}} \omega a_m < \frac{1}{10} \sqrt{\frac{\rho_s(1-v^2)}{E}} \omega_s a_m. \quad (59)$$

Now, after simplifying Equation (59) and substituting Equation (57),

$$0 < \omega < \frac{2\pi C_T}{20h}, \quad (60)$$

which can be rewritten as

$$h < 0.05\lambda_s, \quad (61)$$

and is exactly the same definition offered by Junger & Feit [Ref. 14] listed in Equation (53).

At this point, the thinness criterion for applying the theory of thin elastic shell can be considered to be the expression in Equation (61). This criterion is obeyed in all subsequent calculations.

III. SPHERICAL THREE-DIMENSIONAL MODEL

A. INTRODUCTION

In order to apply the T-matrix method to modeling a dense sonar array, the radiated and scattered pressure from an individual transducer must be calculated. Using finite element analysis, the pressure scattered from a transducer due to an incident spherical harmonic wave can be computed very accurately. To employ the Finite Element Code ATILA, a three-dimensional model of an elastic spherical shell is used.

B. THE MODEL

A 0.5 m outer radius, one centimeter-thick spherical steel shell in water is modeled. An ATILA model is realized using 3D elastic elements, 3D fluid elements, 2D solid-fluid interface elements, and 2D radiation surface elements. The thickness was chosen in an attempt to comply with aspect ratio constraints [Ref. 2] for elements used by ATILA, and to be within the theoretical conditions of thinness (Chapter II).

In order to calculate the limit of thickness to which we can apply the theory of thin elastic shells (Chapter II), the following parameters of the model (for steel) are applied in Equation (58)

$$E = 0.125 \times 10^{12} \text{ (Young's modulus)}$$

$$\rho_s = 7500 \text{ Kg/m}^3, \text{ and}$$

$$\nu = 0.33 \text{ (Poisson's ratio).}$$

The value of C_T is then 3282.83 m/sec^2 , which for a chosen operating frequency of 474 Hz (corresponding to $ka=1$) results in an upper limit for the shell thickness of 0.346 m , and therefore a thickness of one centimeter is verifiably inside the limit.

The complete ATILA code for the elastic shell is presented in Appendix B. Note that, to write an ATILA code for a piezoelectric sphere, the same model can be used as for the elastic sphere. In this instance, the ATILA code can handle the piezoelectric sphere by changing the elements from elastic to piezoelectric type.

The model is composed of 2746 nodes and 288 elements. The mesh is shown in Figures 5 and 6.

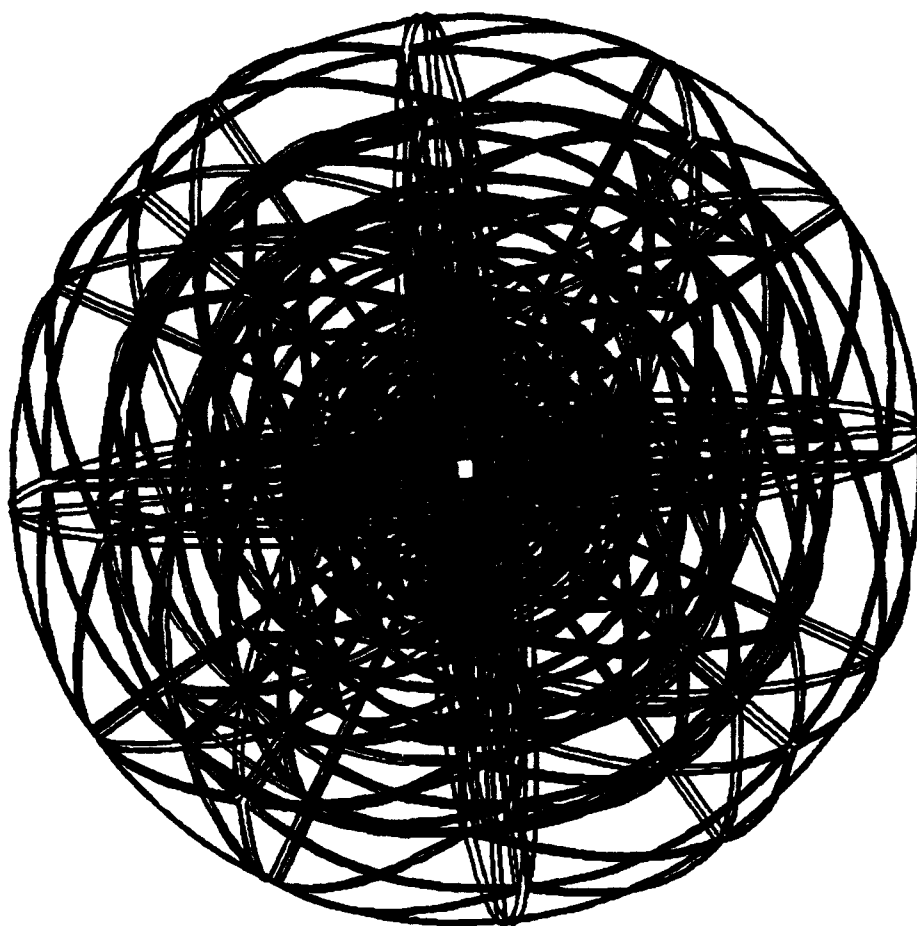


Figure 5. The Model, as Seen from the Side.

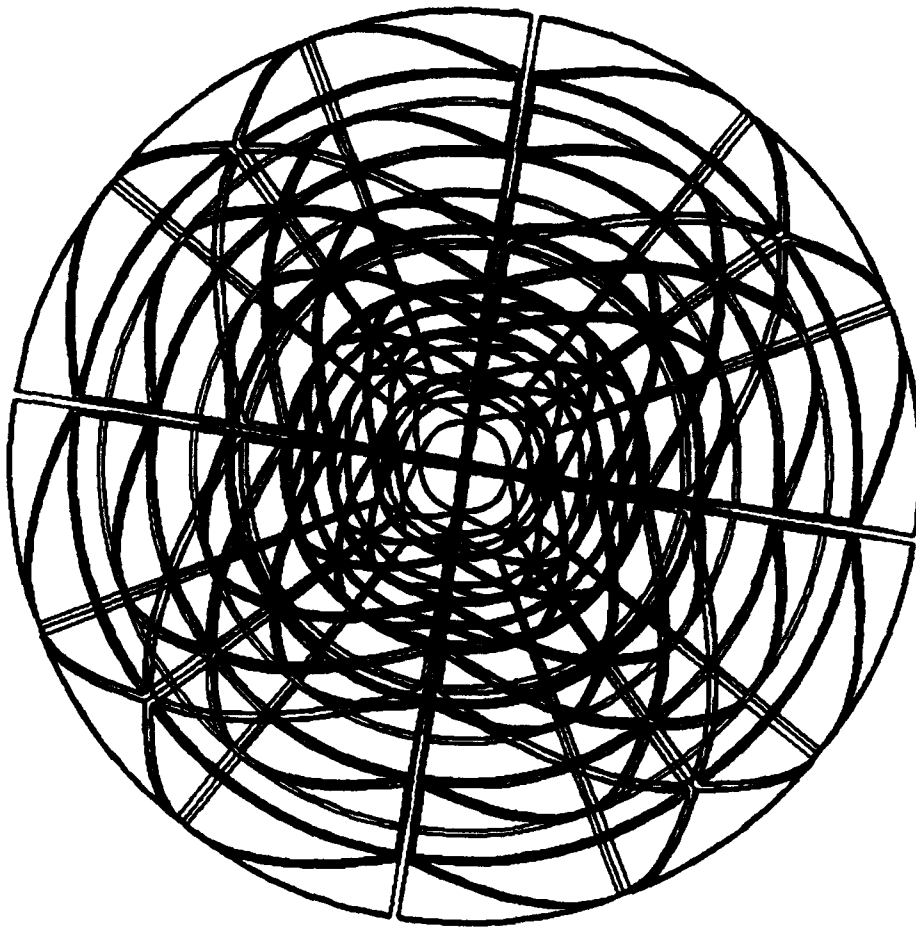


Figure 6. The Model, as Seen from Above.

The material properties of the shell are listed on the first page of Appendix B. The format is described in the ATILA user's manual [Ref. 2].

The elements used in the model followed the pattern from the manual [Ref. 2], and are presented in Table 1.

TABLE I. ATILA ELEMENTS		
Region	Element	Geometry
Elastic shell	SHEL06C	6-node triangular
Interface solid - fluid	TRIA12I	12-node triangular
Fluid	PRIS15F	15-node triangular prism
Radiation Surface	TRIA06R	6-node triangular

IV. RESULTS

A. THE T-MATRIX ELEMENTS AND THE ATILA CODE.

The definition of the T-matrix adopted in this work relates outgoing spherical waves (i.e. Hankel functions) to incident standing waves (i.e., spherical Bessel functions). However, using ATILA, calculation of the coefficients of the scattered pressure for each harmonic, $\{P^s\}$, requires that the incident wave [Appendix A] be an incoming traveling wave rather than a standing wave.

To use the results of the ATILA code in calculations, Equation (46) and its respective derivative, Equation (49) is adapted to

$$\overline{I}_{nm} h_n^{(1)}(ka) + \overline{R}_{nm} h_n^{(2)}(ka) = -i\omega Z_n \overline{W}_{nm}, \quad (62)$$

and its first derivative,

$$\overline{I}_{nm} h_n^{(1)'}(ka) + \overline{R}_{nm} h_n^{(2)'}(ka) = \omega \rho_f C_f \overline{W}_{nm}. \quad (63)$$

The coefficients $\overline{R}_{nm}$ are the amplitudes of the scattered wave components and the coefficients $\overline{W}_{nm}$ are the amplitudes of the components

of the radial displacement coefficients for an incoming wave with coefficients $\bar{I}_{nm}$.

Equation (52) (Chapter II) is used to calculate the R_{nm} coefficients from an incident standing wave, from where the T-Matrix elements are obtained. For incident incoming traveling waves, the R_{nm} elements can be calculated as follows.

From Equation (62) and (63),

$$\bar{R}_{nm} = - \left\{ \frac{iZ_n h_n^{(1)'}(ka) + \rho_f C_f h_n^{(1)}(ka)}{iZ_n h_n^{(2)'}(ka) + \rho_f C_f h_n^{(2)}(ka)} \right\} \bar{I}_{nm}. \quad (64)$$

We can relate R_{nm} and $\bar{R}_{nm}$. We, first, define :

$$X_{nm} = - \left\{ \frac{iZ_n (i y_n'(ka)) + \rho_f C_f (i y_n(ka))}{iZ_n h_n^{(2)'}(ka) + \rho_f C_f h_n^{(2)}(ka)} \right\} I_{nm}, \quad (65)$$

where $y_n(ka)$ is the spherical Bessel function of the second kind.

Applying Equation (65) on Equations (52) and (64), respectively :

$$R_{nm} - X_{nm} = -1 \quad (66)$$

and

$$\bar{R}_{nm} - X_{nm} = R_{nm} \quad (67)$$

Solving Equations (66) and (67) in terms of R_{nm} :

$$R_{nm} = \frac{1}{2} \{ \bar{R}_{nm} - 1 \} \quad (68)$$

Equation (68) is used to calculate the R_{nm} coefficients, and hence the T-Matrix elements for incident standing waves, from the $\overline{R_{nm}}$ coefficients calculated when ATILA code uses incoming incident waves to compute the scattered pressure.

The goal of this research is to compare the T-Matrix elements computed by scattered field pressure from the ATILA code with those determined by theoretical means (Appendix C).

The T-Matrix for an elastic spherical shell is a diagonal matrix where the non-zero elements equal the R_{nm} coefficients, for $I_{nm} = \overline{I_{nm}} = 1$. In this research, calculations of $\overline{R_{nm}}$ were performed up to the harmonic of second order, and the T-Matrix has the following format:

$$\begin{pmatrix} R_{00} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & R_{1-1} & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & R_{10} & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & R_{11} & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & R_{2-2} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & R_{2-1} & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & R_{20} & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & R_{21} & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & R_{22} \end{pmatrix}$$

In the T-matrix, for instance, the element $R_{2,-1}$ corresponds to the spherical harmonic with n equal to two and m equal to negative one. The

T-Matrix elements were theoretically computed using thin shell theory (Appendix C) and the results are presented in Table II.

TABLE II. THE ANALYTIC DIAGONAL ENTRIES OF THE T-MATRIX				
ELEMENT	REAL PART	IMAG PART	MAGNITUDE	PHASE (DEGREES)
$T_{11} = R_{00}$	-1.3465E-02	1.1525E-01	1.1604E-01	9.6663E+01
$T_{22} = R_{1,-1}$	-6.0255E-03	7.7390E-02	7.7624E-02	9.4452E+01
$T_{33} = R_{1,0}$	-6.0255E-03	7.7390E-02	7.7624E-02	9.4452E+01
$T_{44} = R_{1,1}$	-6.0255E-03	7.7390E-02	7.7624E-02	9.4452E+01
$T_{55} = R_{2,-2}$	-6.1773E-04	-2.4847E-02	2.4854E-02	-9.1424E+01
$T_{66} = R_{2,-1}$	-6.1773E-04	-2.4847E-02	2.4854E-02	-9.1424E+01
$T_{77} = R_{2,0}$	-6.1773E-04	-2.4847E-02	2.4854E-02	-9.1424E+01
$T_{88} = R_{2,1}$	-6.1773E-04	-2.4847E-02	2.4854E-02	-9.1424E+01
$T_{99} = R_{2,2}$	-6.1773E-04	-2.4847E-02	2.4854E-02	-9.1424E+01

Incident waves of the form

$$P_{inc}(r, \theta, \phi) = h_n^{(1)}(kr) P_n^m(\cos \theta) e^{im\phi} e^{i\omega t}$$

[Appendix A] for particular values of n and m were applied to the finite-element model spherical shell using the ATILA code and the resulting scattered pressures were calculated. For each incident wave of a particular spherical harmonic component (one n and m), the resulting reflection coefficients for all the spherical harmonic components (all n and m) were calculated from nodal values of the scattered pressure on the shell surface by a least-squares fitting procedure employing singular value decomposition

[Appendix C]. The results for the diagonal elements of the T-matrix are given in Table III. The last two columns of Table III give the error in the ATILA results for the diagonal components compared to the theoretical results of Table II.

TABLE III. THE ATILA DIAGONAL T-MATRIX ELEMENTS						
ELEMENT	REAL PART	IMAG PART	MAGNITUDE	PHASE (DEGREES)	MAG. ERROR (PERCENT)	PHASE ERROR (DEGREES)
T_{11}	-1.2624E-02	1.1429E-01	1.1498E-01	9.6303E+01	-0.91	-0.36
T_{22}	-6.8902E-03	8.2376E-02	8.2664E-02	9.4781E+01	6.49	0.33
T_{33}	-5.7422E-03	8.3113E-02	8.3311E-02	9.3952E+01	7.33	-0.5
T_{44}	-6.8668E-03	8.2363E-02	8.2649E-02	9.4766E+01	6.47	0.31
T_{55}	-2.8487E-03	-1.5203E-02	1.5468E-02	-1.0061E+02	-37.8	-9.19
T_{66}	-2.7899E-03	-1.5159E-02	1.5413E-02	-1.0043E+02	-38	-9.01
T_{77}	-2.4889E-03	-1.5287E-02	1.5489E-02	-9.9247E+01	-37.7	-7.82
T_{88}	-2.7847E-03	-1.5164E-02	1.5418E-02	-1.0041E+02	-38	-8.99
T_{99}	-2.8227E-03	-1.5182E-02	1.5443E-02	-1.0053E+02	-37.9	-9.11

The T-matrix for a spherically symmetric scattered should be diagonal, that is, there should be only one spherical harmonic component of the scattered wave for a single-component incident wave, and it should be the same component as the incident wave. Nonzero off-diagonal components computed for the T-matrix of the spherical elastic shell can be termed

"leakage" (in some abstract sense). The existence of leakage indicates a weakness in the model.

Nonzero off-diagonal components of the T-matrix which were greater than 10^{-14} of the diagonal element were found for a few of the ATILA results. These are given in Table IV below.

TABLE IV. SIGNIFICANT OFF-DIAGONAL T-MATRIX ELEMENTS						
element	real	imag	mag	phase	mag T_{ij} rel to T_{ii}	phase T_{ij} rel to T_{ii}
T_{71}	-7.6679E-04	5.3764E-04	9.3650E-04	1.4496E+02	8.14E-03	48.7
T_{62}	2.2172E-05	7.2191E-06	2.3318E-05	1.8035E+01	2.82E-04	-76.7
T_{84}	-8.9863E-06	-3.8094E-07	8.9944E-06	-1.7757E+02	1.09E-04	-272.3
T_{95}	1.6194E-05	1.4535E-05	2.1761E-05	4.1911E+01	1.41E-03	142.5
T_{28}	-1.1214E-05	4.1657E-08	1.1214E-05	1.7979E+02	7.28E-04	280.2
T_{17}	5.5328E-03	-4.3539E-03	7.0405E-03	-3.8200E+01	4.55E-01	61
T_{48}	3.5575E-05	3.6843E-05	5.1215E-05	4.6003E+01	3.32E-03	146.4
T_{59}	9.8906E-03	7.9364E-03	1.2681E-02	3.8744E+01	8.21E-01	139.3

It is seen that there is significant leakage from the 2,0 to the 0,0 component T_{17} , and from the 2,2 to the 2,-2 component T_{59} , for the finite-element model employed. This could indicate that either the mesh is too coarse, particularly that of the shell, or that the radius of the radiation damping elements which terminate the fluid mesh is too small. These possibilities remain to be investigated.

B. THE ACOUSTIC IMPEDANCE

To further validate the three-dimensional model used in the ATILA code, the acoustic impedance computed analytically and with ATILA output were also compared. The theoretical acoustic impedance, for each harmonic, is given by Equation (42).

On the other hand, output from ATILA can be used to determine the impedance from Equation (41), where F_{nm} will be the total field pressure. Total pressures are output from ATILA when the pressure command in ATILA code (Appendix B) is "PRESSURE TOTAL" [Reference 2]. The terms W_{nm} are also computed from the ATILA output. Averaged values of F_{nm} and W_{nm} were used to compute an approximation to the acoustic impedance of each harmonic. The results are presented in Table VI.

TABLE V. RESULTS OF ACOUSTIC IMPEDANCE		
Theoretical Values	From ATILA	Error and Number of Points
$Z_0=0.00-i\ 0.8396E+07$	$Z_0=0.5804E+03-i\ 0.82305E+07$	2% and 146
$Z_1=0.00+i\ 0.6945E+06$	$Z_1=0.26659E+04+i\ 0.60223E+06$	13% and 138

V. CONCLUSIONS

The transition matrix, or "T-matrix", relating the scattered and incident acoustic waves for a thin elastic spherical shell in a free-field environment has been evaluated using the ATILA finite-element code. A three-dimensional finite-element model of a 0.5-m outer radius, 1-cm thick spherical steel shell surrounded by water was developed [Appendix A]. In this model the spherical shell surface is divided into seventy-two approximately equal area triangular regions. The ATILA code was used to calculate the scattered pressure over the surface of the shell for incident waves represented as products of radial Hankel functions and spherical harmonics. This was done for all components through the second order in the Hankel function. The chosen driving frequency was 474 Hz, corresponding to a value of $ka=1$, where k is the wavenumber of sound in water and a is the radius of the shell. The ATILA results were compared with the results of analytical thin shell theory. For the diagonal elements of the T-matrix, the two results were found to agree within one, seven, and thirty-eight percent, respectively, for the zeroth, first, and second order

components. The ATILA results also produce a few nonzero off-diagonal T-matrix elements which do not appear in the theoretical results, and which should not be present according to symmetry considerations. The reason for the appearance of nonzero off-diagonal elements in the ATILA results is not known at this time and warrants further investigation. Also computed for each harmonic component was the modal acoustical impedance of the shell. These results agreed with the results of thin shell theory within two percent for the zeroth order component and thirteen percent for the first order components.

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APPENDIX A

FUNCTION INCPRE(X,Y,Z,K)

* PROGRAM BY ARTHUR LOBO DA COSTA RUIZ 12/23/93

* FUNCTION:

* COMPUTES THE INCIDENT SPHERICAL HARMONIC PRESSURE AT THE POINT
* (X,Y,Z), FOR THE WAVENUMBER K

* VARIABLES INPUT:

* X,Y,Z: CARTESIAN COORDINATES OF THE POINT
* K: WAVENUMBER

* VARIABLES OUTPUT:

* RADIUS :R IN METERS
* PHI IN DEGREE (HORIZONTAL ANGLE)
* THETA IN RADIANS (AZIMUTAL ANGLE)
* INCPRE : PRESSURE AT POINT IN PASCAL

DOUBLE PRECISION K,X,Y,Z,R,PHI,THETA,KR
REAL*8 PMN(-2:2,0:2),LOUT
INTEGER N,M,NMAX
COMPLEX*16 INCPRE,H(0:2),HOUT

* N AND M ARE ORDERS FOR HANKEL AND LEGENDRE FUNCTIONS

N=0

M=0

* HOUT AND LOUT ARE HANKEL AND LEGENDRE OUTPUTS
* REF TO N AND M

NMAX=2

* TRANSFORM CARTESIAN COORDINATES (X,Y,Z)

* INTO SPHERICAL COORDINATES

* R(RADIUS), PHI(AZIMUTAL ANGLE) AND THETA(POLAR ANGLE)

R=DSQRT(X*X+Y*Y+Z*Z)

PHI=DATAN2D(Y,X)

IF((X.EQ.0).AND.(Y.EQ.0)) PHI=0.0D0

THETA=DACOS(Z/R)

KR=K*R

* NMAX IS THE MAXIMUM NUMBER OF HARMONICS

```

*
* CALL HANKEL(KR,NMAX,H)
*
* SUBROUTINE HANKEL RETURNS SPHERICAL HANKEL
*   FUNCTIONS JN AND I YN
*
* CALL LEGNDR(THETA,NMAX,PMN)
* LOUT=PMN(M,N)
*
* SUBROUTINE LEGNDR RETURNS ASSOCIATE LEGENDRE FUNTION
*
* INCPRE=H(N)*LOUT*DCMPLX(DCOSD(M*PHI),DSIND(M*PHI))
* IF ((R.LE.0.501).AND.(R.GT.0.499)) THEN
*   PRINT *, "X,Y,Z =",X,Y,Z
*   PRINT *, "PHI,THETA =",PHI,THETA
*   PRINT *, "K,R,KR =",K,R,KR
*   PRINT *, "REAL HANKEL =",HOUT
*   PRINT *, "LEGENDRE =",LOUT
*   PRINT *, INCPRE
*   PRINT *, "*****"
* ELSE
*   CONTINUE
* ENDIF
* RETURN
* END
C *****
C
C SUBROUTINE HANKEL(X,NMAX,H)
C IMPLICIT REAL*8 (A-H,O-Z)
C COMPLEX*16 H(0:NMAX)
C
C GIVEN THE VARIABLE X, AND THE MAXIMUM ORDER NMAX,
C THIS ROUTINE GENERATES THE SPHERICAL HANKEL FUNCTION HN
C FOR ALL N FROM 0 TO NMAX (INCLUSIVE)
C INPUT:
C X = DOUBLE PREC. VARIABLE (RADIUS)
C NMAX = INTEGER MAXIMUM ORDER OF BESSEL FUNCTIONS DESIRED
C OUTPUT:
C H(N) = ARRAY OF SPHERICAL HANKEL FUNCTIONS HN(X), WHERE
C HN = JN + I YN C C THIS ROUTINE IS BASED ON THE RECURSION FORMULAE
C FROM ABRAMOWITZ & STEGUN: 10.1.10 & 10.1.15, PP.438-9
C THE F' S ARE THE COEFFICIENTS OF ORDER N & -(N+1),
C THE FO' S ARE OLD F' S, FOR RECURSION
C
C IF ( X .LE. 0.0D0 ) THEN
C   H(0) = DCMPLX(1.0D0,-1.0D35)
C   DO 2 N = 1, NMAX
C     H(N) = CMPLX(0.0D0,-1.0D35)
C 2 CONTINUE
C RETURN
C END IF

```

```

SX = DSIN(X)
CX = DCOS(X)
XINV = 1.0D0/X
M1N = -1.0D0
FN = XINV
FMN = 0.0D0
FNO = FMN
FMNO = FN
DO 4 N = 0, NMAX
H(N) = CMPLX( FN*SX + M1N*FMN*CX, -FN*CX + M1N*FMN*SX )
T1 = (2*N+1)*XINV
T2 = T1*FN - FNO
FNO = FN
FN = T2
T2 = -T1*FMN - FMNO
FMNO = FMN
FMN = T2
M1N = -M1N
4 CONTINUE
RETURN
END

```

```

C
C *****
C
C
C SUBROUTINE LEGNDR(THETA,NMAX,PMN)
C IMPLICIT REAL*8 (A-H,O-Z)
C REAL*8 PMN(-NMAX:NMAX,0:NMAX)
C
C GIVEN THE VARIABLE THETA, AND THE MAXIMUM ORDER NMAX,
C THIS ROUTINE GENERATES THE ASSOC. LEGENDRE FUNCTIONS PMN
C OF THE ARGUMENT COS(THETA) (THETA MUST BE BETWEEN 0 & PI)
C FOR ALL N FROM 0 TO NMAX (INCLUSIVE)
C AND FOR ALL M FROM -N TO N (SOME OTHERS SET TO ZERO)
C INPUT:
C THETA = VARIABLE (POLAR ANGLE), MUST BE BETWEEN 0 & PI (INCL.)
C NMAX = INTEGER MAXIMUM ORDER OF LEGENDRE FUNCTIONS DESIRED
C OUTPUT:
C PMN = DOUBLE PREC. ARRAY, CONTAINS ASSOC. LEGENDRE FNS
C
C THIS ROUTINE IS BASED ON THE RECURSION FORMULAE
C FROM ABRAMOWITZ & STEGUN
C
C X = DCOS(THETA)
C SINTHT = DSIN(THETA)
C IF ( SINTHT .GT. 0. ) THEN
C   SININV = 1.0D0/SINTHT
C ELSE
C   SININV = 0.0D0
C END IF
C SET VALUES FOR N = 0, 1 (NMAX MUST BE AT LEAST 1)
C PMN(0,0) = 1.0D0

```

```

PMN(1,0) = 0.0D0
PMN(-1,0) = 0.0D0
PMN(0,1) = X
PMN(1,1) = -SINTHT
PMN(-1,1) = SINTHT*0.5D0
C IN LOOP, TNP1 = 2*N+1, TNP2FC = (2*N+2)!, M1N = (-1)**(N+1)
  TNP1 = 1.0D0
  TNP2FC = 2.0D0
  M1N = -1.0D0
  DO 4 N = 1, NMAX-1
    TNP1 = TNP1 + 2.0D0
    TNP2FC = TNP2FC * TNP1 * (TNP1+1)
    M1N = -M1N
    DO 3 M = -N, N
      PMN(M,N+1) = (TNP1*X*PMN(M,N) - (N+M)*PMN(M,N-1))/(N-M+1)
    3 CONTINUE
    PMN(N+1,N) = 0.0D0
    PMN(-N-1,N) = 0.0D0
    PMN(N+1,N+1) = (X*PMN(N,N+1) - TNP1*PMN(N,N)) * SININV
    PMN(-N-1,N+1) = M1N*PMN(N+1,N+1)/TNP2FC
  4 CONTINUE
C DO 120 N=0,NMAX
C DO 120 M=-N,N
C120 WRITE(6,130) N,M,PMN(M,N)
C130 FORMAT(1X,' N= ',I4,1X,' M= ',I4,1X,' PMN= ',F13.6)
  RETURN
END

```

APPENDIX B

```
*****
* ATILA MODEL ESHELL4: 3-D ELASTIC SPHERICAL SHELL, 1 CM THICK
* AUTHOR: BAKER    SHELL RADIUS 0.5 M, MADE OF NEW CURVED SHELL ELEMENTS
* DATE: MAR 1994   FLUID RADIUS 2.5 M
*                  EACH OCTANT DIVIDED INTO 9 REGIONS
*                  2 FLUID LAYERS, EACH 0.25 M THICK,
*                  3 FLUID LAYERS, EACH 0.5 M THICK
*****
```

```
PRINTING = 2
RADIATION DIPOLAR
ANALYSIS HARMONIC
MATERIAL
25CD4SH * PROPERTIES FOR CURVED COMPOSITE SHELL ELEMENTS
0.215E+12 0.330E+00 0.750E+04 0.000E+00 0.000E+00 0.000E+00 &
0.215E+12 0.330E+00 0.750E+04 0.000E+00 1.0      0.000E+00
LCPDDC = 6
NLOAD = 1
FREQUENCY 474.
GEOMETRY
```

```
1
2.5 * RADIUS OF RADIATION BOUNDARY = 2.5 M
2
1 0 0.01 * THICKNESS OF SHELL = 1 CM
```

```
PRESSURE SCATTERED
SKYLINE REAL
PRECISION DOUBLE
NEWAXES SPHERICAL
0 0 0 0 0 0
```

```
NODES
* 0001 * 0.490 000 000.0
* 0002 * 0.490 015 000.0
* 0003 * 0.490 015 090.0
* 0004 * 0.490 015 180.0
* 0005 * 0.490 015 270.0
* 0006 * 0.490 030 000.0
* 0007 * 0.490 030 045.0
* 0008 * 0.490 030 090.0
* 0009 * 0.490 030 135.0
* 0010 * 0.490 030 180.0
* 0011 * 0.490 030 225.0
* 0012 * 0.490 030 270.0
* 0013 * 0.490 030 315.0
* 0014 * 0.490 045 000.0
* 0015 * 0.490 045 030.0
```

* 0016 *	0.490	045	060.0
* 0017 *	0.490	045	090.0
* 0018 *	0.490	045	120.0
* 0019 *	0.490	045	150.0
* 0020 *	0.490	045	180.0
* 0021 *	0.490	045	210.0
* 0022 *	0.490	045	240.0
* 0023 *	0.490	045	270.0
* 0024 *	0.490	045	300.0
* 0025 *	0.490	045	330.0
* 0026 *	0.490	060	000.0
* 0027 *	0.490	060	022.5
* 0028 *	0.490	060	045.0
* 0029 *	0.490	060	067.5
* 0030 *	0.490	060	090.0
* 0031 *	0.490	060	112.5
* 0032 *	0.490	060	135.0
* 0033 *	0.490	060	157.5
* 0034 *	0.490	060	180.0
* 0035 *	0.490	060	202.5
* 0036 *	0.490	060	225.0
* 0037 *	0.490	060	247.5
* 0038 *	0.490	060	270.0
* 0039 *	0.490	060	292.5
* 0040 *	0.490	060	315.0
* 0041 *	0.490	060	337.5
* 0042 *	0.490	075	000.0
* 0043 *	0.490	075	018.0
* 0044 *	0.490	075	036.0
* 0045 *	0.490	075	054.0
* 0046 *	0.490	075	072.0
* 0047 *	0.490	075	090.0
* 0048 *	0.490	075	108.0
* 0049 *	0.490	075	126.0
* 0050 *	0.490	075	144.0
* 0051 *	0.490	075	162.0
* 0052 *	0.490	075	180.0
* 0053 *	0.490	075	198.0
* 0054 *	0.490	075	216.0
* 0055 *	0.490	075	234.0
* 0056 *	0.490	075	252.0
* 0057 *	0.490	075	270.0
* 0058 *	0.490	075	288.0
* 0059 *	0.490	075	306.0
* 0060 *	0.490	075	324.0
* 0061 *	0.490	075	342.0
* 0062 *	0.490	090	000.0
* 0063 *	0.490	090	015.0
* 0064 *	0.490	090	030.0
* 0065 *	0.490	090	045.0
* 0066 *	0.490	090	060.0

* 0067 *	0.490	090	075.0
* 0068 *	0.490	090	090.0
* 0069 *	0.490	090	105.0
* 0070 *	0.490	090	120.0
* 0071 *	0.490	090	135.0
* 0072 *	0.490	090	150.0
* 0073 *	0.490	090	165.0
* 0074 *	0.490	090	180.0
* 0075 *	0.490	090	195.0
* 0076 *	0.490	090	210.0
* 0077 *	0.490	090	225.0
* 0078 *	0.490	090	240.0
* 0079 *	0.490	090	255.0
* 0080 *	0.490	090	270.0
* 0081 *	0.490	090	285.0
* 0082 *	0.490	090	300.0
* 0083 *	0.490	090	315.0
* 0084 *	0.490	090	330.0
* 0085 *	0.490	090	345.0
* 0086 *	0.490	105	000.0
* 0087 *	0.490	105	018.0
* 0088 *	0.490	105	036.0
* 0089 *	0.490	105	054.0
* 0090 *	0.490	105	072.0
* 0091 *	0.490	105	090.0
* 0092 *	0.490	105	108.0
* 0093 *	0.490	105	126.0
* 0094 *	0.490	105	144.0
* 0095 *	0.490	105	162.0
* 0096 *	0.490	105	180.0
* 0097 *	0.490	105	198.0
* 0098 *	0.490	105	216.0
* 0099 *	0.490	105	234.0
* 0100 *	0.490	105	252.0
* 0101 *	0.490	105	270.0
* 0102 *	0.490	105	288.0
* 0103 *	0.490	105	306.0
* 0104 *	0.490	105	324.0
* 0105 *	0.490	105	342.0
* 0106 *	0.490	120	000.0
* 0107 *	0.490	120	022.5
* 0108 *	0.490	120	045.0
* 0109 *	0.490	120	067.5
* 0110 *	0.490	120	090.0
* 0111 *	0.490	120	112.5
* 0112 *	0.490	120	135.0
* 0113 *	0.490	120	157.5
* 0114 *	0.490	120	180.0
* 0115 *	0.490	120	202.5
* 0116 *	0.490	120	225.0
* 0117 *	0.490	120	247.5

* 0118 *	0.490	120	270.0
* 0119 *	0.490	120	292.5
* 0120 *	0.490	120	315.0
* 0121 *	0.490	120	337.5
* 0122 *	0.490	135	000.0
* 0123 *	0.490	135	030.0
* 0124 *	0.490	135	060.0
* 0125 *	0.490	135	090.0
* 0126 *	0.490	135	120.0
* 0127 *	0.490	135	150.0
* 0128 *	0.490	135	180.0
* 0129 *	0.490	135	210.0
* 0130 *	0.490	135	240.0
* 0131 *	0.490	135	270.0
* 0132 *	0.490	135	300.0
* 0133 *	0.490	135	330.0
* 0134 *	0.490	150	000.0
* 0135 *	0.490	150	045.0
* 0136 *	0.490	150	090.0
* 0137 *	0.490	150	135.0
* 0138 *	0.490	150	180.0
* 0139 *	0.490	150	225.0
* 0140 *	0.490	150	270.0
* 0141 *	0.490	150	315.0
* 0142 *	0.490	165	000.0
* 0143 *	0.490	165	090.0
* 0144 *	0.490	165	180.0
* 0145 *	0.490	165	270.0
* 0146 *	0.490	180	000.0
* 0147 *	0.000	000	000.0
* 0148 *	0.000	000	000.0
* 0149 *	0.000	000	000.0
* 0150 *	0.000	000	000.0
* 0151 *	0.000	000	000.0
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* 0168 *	0.000	000	000.0

* 0169 *	0.000	000	000.0
* 0170 *	0.000	000	000.0
* 0171 *	0.000	000	000.0
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* 0173 *	0.000	000	000.0
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* 0175 *	0.000	000	000.0
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* 0199 *	0.000	000	000.0
* 0200 *	0.000	000	000.0
* 0201 *	0.495	000	000.0
* 0202 *	0.495	015	000.0
* 0203 *	0.495	015	090.0
* 0204 *	0.495	015	180.0
* 0205 *	0.495	015	270.0
* 0206 *	0.495	030	000.0
* 0207 *	0.495	030	045.0
* 0208 *	0.495	030	090.0
* 0209 *	0.495	030	135.0
* 0210 *	0.495	030	180.0
* 0211 *	0.495	030	225.0
* 0212 *	0.495	030	270.0
* 0213 *	0.495	030	315.0
* 0214 *	0.495	045	000.0
* 0215 *	0.495	045	030.0
* 0216 *	0.495	045	060.0
* 0217 *	0.495	045	090.0
* 0218 *	0.495	045	120.0
* 0219 *	0.495	045	150.0

* 0220 *	0.495	045	180.0
* 0221 *	0.495	045	210.0
* 0222 *	0.495	045	240.0
* 0223 *	0.495	045	270.0
* 0224 *	0.495	045	300.0
* 0225 *	0.495	045	330.0
* 0226 *	0.495	060	000.0
* 0227 *	0.495	060	022.5
* 0228 *	0.495	060	045.0
* 0229 *	0.495	060	067.5
* 0230 *	0.495	060	090.0
* 0231 *	0.495	060	112.5
* 0232 *	0.495	060	135.0
* 0233 *	0.495	060	157.5
* 0234 *	0.495	060	180.0
* 0235 *	0.495	060	202.5
* 0236 *	0.495	060	225.0
* 0237 *	0.495	060	247.5
* 0238 *	0.495	060	270.0
* 0239 *	0.495	060	292.5
* 0240 *	0.495	060	315.0
* 0241 *	0.495	060	337.5
* 0242 *	0.495	075	000.0
* 0243 *	0.495	075	018.0
* 0244 *	0.495	075	036.0
* 0245 *	0.495	075	054.0
* 0246 *	0.495	075	072.0
* 0247 *	0.495	075	090.0
* 0248 *	0.495	075	108.0
* 0249 *	0.495	075	126.0
* 0250 *	0.495	075	144.0
* 0251 *	0.495	075	162.0
* 0252 *	0.495	075	180.0
* 0253 *	0.495	075	198.0
* 0254 *	0.495	075	216.0
* 0255 *	0.495	075	234.0
* 0256 *	0.495	075	252.0
* 0257 *	0.495	075	270.0
* 0258 *	0.495	075	288.0
* 0259 *	0.495	075	306.0
* 0260 *	0.495	075	324.0
* 0261 *	0.495	075	342.0
* 0262 *	0.495	090	000.0
* 0263 *	0.495	090	015.0
* 0264 *	0.495	090	030.0
* 0265 *	0.495	090	045.0
* 0266 *	0.495	090	060.0
* 0267 *	0.495	090	075.0
* 0268 *	0.495	090	090.0
* 0269 *	0.495	090	105.0
* 0270 *	0.495	090	120.0

* 0271 *	0.495	090	135.0
* 0272 *	0.495	090	150.0
* 0273 *	0.495	090	165.0
* 0274 *	0.495	090	180.0
* 0275 *	0.495	090	195.0
* 0276 *	0.495	090	210.0
* 0277 *	0.495	090	225.0
* 0278 *	0.495	090	240.0
* 0279 *	0.495	090	255.0
* 0280 *	0.495	090	270.0
* 0281 *	0.495	090	285.0
* 0282 *	0.495	090	300.0
* 0283 *	0.495	090	315.0
* 0284 *	0.495	090	330.0
* 0285 *	0.495	090	345.0
* 0286 *	0.495	105	000.0
* 0287 *	0.495	105	018.0
* 0288 *	0.495	105	036.0
* 0289 *	0.495	105	054.0
* 0290 *	0.495	105	072.0
* 0291 *	0.495	105	090.0
* 0292 *	0.495	105	108.0
* 0293 *	0.495	105	126.0
* 0294 *	0.495	105	144.0
* 0295 *	0.495	105	162.0
* 0296 *	0.495	105	180.0
* 0297 *	0.495	105	198.0
* 0298 *	0.495	105	216.0
* 0299 *	0.495	105	234.0
* 0300 *	0.495	105	252.0
* 0301 *	0.495	105	270.0
* 0302 *	0.495	105	288.0
* 0303 *	0.495	105	306.0
* 0304 *	0.495	105	324.0
* 0305 *	0.495	105	342.0
* 0306 *	0.495	120	000.0
* 0307 *	0.495	120	022.5
* 0308 *	0.495	120	045.0
* 0309 *	0.495	120	067.5
* 0310 *	0.495	120	090.0
* 0311 *	0.495	120	112.5
* 0312 *	0.495	120	135.0
* 0313 *	0.495	120	157.5
* 0314 *	0.495	120	180.0
* 0315 *	0.495	120	202.5
* 0316 *	0.495	120	225.0
* 0317 *	0.495	120	247.5
* 0318 *	0.495	120	270.0
* 0319 *	0.495	120	292.5
* 0320 *	0.495	120	315.0
* 0321 *	0.495	120	337.5

* 0322 *	0.495	135	000.0
* 0323 *	0.495	135	030.0
* 0324 *	0.495	135	060.0
* 0325 *	0.495	135	090.0
* 0326 *	0.495	135	120.0
* 0327 *	0.495	135	150.0
* 0328 *	0.495	135	180.0
* 0329 *	0.495	135	210.0
* 0330 *	0.495	135	240.0
* 0331 *	0.495	135	270.0
* 0332 *	0.495	135	300.0
* 0333 *	0.495	135	330.0
* 0334 *	0.495	150	000.0
* 0335 *	0.495	150	045.0
* 0336 *	0.495	150	090.0
* 0337 *	0.495	150	135.0
* 0338 *	0.495	150	180.0
* 0339 *	0.495	150	225.0
* 0340 *	0.495	150	270.0
* 0341 *	0.495	150	315.0
* 0342 *	0.495	165	000.0
* 0343 *	0.495	165	090.0
* 0344 *	0.495	165	180.0
* 0345 *	0.495	165	270.0
* 0346 *	0.495	180	000.0
* 0347 *	0.000	000	000.0
* 0348 *	0.000	000	000.0
* 0349 *	0.000	000	000.0
* 0350 *	0.000	000	000.0
* 0351 *	0.000	000	000.0
* 0352 *	0.000	000	000.0
* 0353 *	0.000	000	000.0
* 0354 *	0.000	000	000.0
* 0355 *	0.000	000	000.0
* 0356 *	0.000	000	000.0
* 0357 *	0.000	000	000.0
* 0358 *	0.000	000	000.0
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* 1891 *	1.500	105	090.0
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* 1895 *	1.500	105	162.0
* 1896 *	1.500	105	180.0
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* 1900 *	1.500	105	252.0
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* 1905 *	1.500	105	342.0
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* 1918 *	1.500	120	270.0
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* 1933 *	1.500	135	330.0
* 1934 *	1.500	150	000.0
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* 1940 *	1.500	150	270.0
* 1941 *	1.500	150	315.0
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* 2013 *	1.750	030	315.0
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* 2016 *	1.750	045	060.0
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* 2019 *	1.750	045	150.0
* 2020 *	1.750	045	180.0
* 2021 *	1.750	045	210.0
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* 2023 *	1.750	045	270.0
* 2024 *	1.750	045	300.0
* 2025 *	1.750	045	330.0
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END

APPENDIX C.

```

PROGRAM BNDMAT
CALL SVDSUB
STOP
END
C
C *****
C
C SUBROUTINE SVDSUB
C *****
C  PARAMETER (NHARM=2,MSPHER=3,
C 1  IBLOCK=(NHARM+1)**2,NPOINT=146)
C  IMPLICIT REAL*8 (A-H,O-Z)
C  COMPLEX*16 CZERO,CONE,CI,HRADB(0:NHARM),MATINF(NPOINT,IBLOCK),
C 1  WK(NPOINT),SIG(IBLOCK),EXTRA(IBLOCK),VM(IBLOCK,IBLOCK),
C 2  UM(NPOINT,NPOINT),UIINV(IBLOCK,NPOINT),PHIFAC,PDATA(NPOINT),
C 3  ACHK(IBLOCK),RES(NPOINT),MATCHK(NPOINT,IBLOCK),HRADR(0:NHARM),
C 4  CTEM1,ZZCHK
C  REAL*8 KK,KA,PMX(-NHARM:NHARM,0:NHARM),KR
C  COMMON/COM0/CZERO,CONE,CI,ORIGIN(0:MSPHER,3)
C *****
C  PI=4.0D0*DATAN(1.0D0)
C
C THIS ROUTINE INITIALIZES ARRAYS FOR THE MAIN ROUTINE
C SPECIFICALLY, IT:
C   SETS COMPLEX CONSTANTS FOR 0, 1, AND I
C   COMPUTES HANKEL & BESSEL FUNCTIONS (& DERIVATIVES) FOR RADIUS
C
C INPUT:
C  KK = WAVENUMBER
C  RADSPH = RADIUS OF SPHERE
C  CFLD=FLUID WAVE SPEED 1524 METERS/SEC
C  FLDDEN=FLUID DENSITY 1000 KG/METER**3
C  RHOC = RHO*C, RADIATION IMPEDANCE OF MEDIUM
C  X,Y,Z = COORDINATES FOR EACH SPHERE
C  NPOINT = NUMBER OF DATA POINTS USED IN THE CALCULATION
C
C SET COMPLEX CONSTANTS
C  CZERO = DCMPLX(0.0D0,0.0D0)
C  CONE = DCMPLX(1.0D0,0.0D0)
C  CI = DCMPLX(0.0D0,1.0D0)
C INPUT THE EXCITATION MODE MMI AND NNI
C  MMI=0
C  NNI=2
C  III=NNI*NNI+NNI+MMI+1

```

```

C INPUT WAVENUMBER, SPHERE RADIUS, AND MEDIUM' S DENSITY* SOUND SPEED
RADSPH=0.5D0
CFLD=1492.94D0
HERTZ=474.0D0
KK = 2*PI*HERTZ/CFLD
FLDDEN=1000.0D0
RHOC=CFLD*FLDDEN
KA = KK*RADSPH
WRITE(6,21) KK,RADSPH,RHOC
21 FORMAT(5X,' K= ',1E10.4,' A= ',1E10.4,' RHO*C = ',1E10.4)
CALL HANKEL(KA,NHARM,HRADB)
*****
C THE DATA FILE MUST RESIDE IN UNIT 12 TO DO THE FOLLOWING
DO 10 I=1,NPOINT
READ(12,15) X,Y,Z,PDATA(I)
15 FORMAT(E11.3,E12.3,E12.3,E13.3,E12.3)
IF(I.EQ.1) THEN
RBND=DSQRT(X**2+Y**2+Z**2)
KR=KK*RBND
PRINT *,KR
CALL HANKEL(KR,NHARM,HRADR)
ENDIF
THETX=DATAN2(DSQRT(X**2+Y**2),Z)
CALL LEGNDR(THETX,NHARM,PMX)
PHIX=DATAN2(Y,X)
C THE FOLLOWING IS TO CHECK OUTPUTS FROM ATILA N=1, M=-1
C
C ZZCHK=DCMPLX( 0.97307D0, 0.23051D0)*DCONJG(HRADR(0))
C ZZCHK=DCMPLX(0.98795D0, 0.15478D0)*DCONJG(HRADR(NNI))*
C 1 PMX(MMI,NNI)*DCMPLX(DCOS(MMI*PHIX),DSIN(MMI*PHIX))
ZZCHK=DCMPLX(0.99876D0,-0.049693D0)*DCONJG(HRADR(NNI))*
1 PMX(MMI,NNI)*DCMPLX(DCOS(MMI*PHIX),DSIN(MMI*PHIX))
C
C
C PDATA(I)=ZZCHK
PHASED=DATAN2(DIMAG(PDATA(I)),DREAL(PDATA(I)))
PHASEA=DATAN2(DIMAG(ZZCHK),DREAL(ZZCHK))
C
WRITE(6,16) X,Y,Z,PDATA(I),CDABS(PDATA(I)),
1 ZZCHK,CDABS(ZZCHK)
ZZCHK=ZZCHK-PDATA(I)
WRITE(6,17) ZZCHK,CDABS(ZZCHK)
17 FORMAT(13X,2E17.5,E17.5)
WRITE(6,16) PHASED,PHASEA,PHASEA-PHASED
16 FORMAT(7X,3E17.6)
16 FORMAT(2X,6E12.3,/,38X,3E12.3)
WRITE(25,125) PHIX,PHASED
125 FORMAT(2X,2F13.5)
DO 10 N=0,NHARM
DO 10 M=-N,N
PHIFAC=DCMPLX(DCOS(M*PHIX),DSIN(M*PHIX))

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JJ=N*N+N+M+1
MATINF(I,J)=DCONJG(HRADR(N))*PMX(M,N)*PHIFAC
MATCHK(I,J)=MATINF(I,J)
10 CONTINUE
CALL ZSVDC(MATINF,NPOINT,NPOINT,IBLOCK,SIG,EXTRA,UM,NPOINT,VM,
1 IBLOCK,WK,21,INFOX)
WRITE(6,30) (SIG(I),I=1,IBLOCK)
30 FORMAT(1X,2E20.5)
DO 40 I=1,IBLOCK
DO 40 J=1,NPOINT
UIINV(I,J)=CZERO
DO 40 K=1,IBLOCK
IF(CDABS(SIG(K)).LT.10E-8) GO TO 40
UIINV(I,J)=UIINV(I,J)+VM(I,K)*DCONJG(UM(J,K))/SIG(K)
40 CONTINUE
DO 50 I=1,IBLOCK
ACHK(I)=CZERO
DO 50 K=1,NPOINT
ACHK(I)=ACHK(I)+UIINV(I,K)*PDATA(K)
50 CONTINUE
WRITE(6,160)
160 FORMAT(1X,' THE FOLLOWING ARE THE UNMODIFIED T MATRIX ELEMENTS' ,
1 1X,' OR THE VALUES OF THE RADIATED PRESSURES' )
DO 55 N=0,NHARM
DO 55 M=-N,N
I=N*N+N+M+1
CTEM1=ACHK(I)
55 WRITE(6,60) I,CTEM1,CDABS(CTEM1)
60 FORMAT(1X,I3,1X,3E20.5)
WRITE(6,161)
161 FORMAT(1X,' THE FOLLOWING ARE THE TRUE T MATRIX ELEMENTS' )
C III IS THE EXCITATION MODE EG FOR M=0,N=0 III=1
DO 155 N=0,NHARM
DO 155 M=-N,N
I=N*N+N+M+1
IF((N.EQ.NNI).AND.(M.EQ.MMI)) THEN
CTEM1=0.5D0*(ACHK(I)-1.0D0)
ELSE
CTEM1=ACHK(I)
ENDIF
155 WRITE(6,60) I,CTEM1,CDABS(CTEM1)
C WANT TO GET A HANDLE ON THE RELATIVE ERROR INVOLVED IN THE SVD CALC
DO 70 J=1,NPOINT
RES(J)=DCMPLX(0.0D0,0.0D0)
DO 80 I=1,IBLOCK 80 RES(J)=RES(J)+MATCHK(J,I)*ACHK(I)
RES(J)=RES(J)-PDATA(J)
70 CONTINUE
DO 100 I=1,NPOINT
RESM=RESM+RES(I)*DCONJG(RES(I))
DATAM=DATAM+PDATA(I)*CONJG(PDATA(I))
100 CONTINUE

```

```

      RELERR=RESM/DATAM
      WRITE(6,110) RESM,DATAM,RELERR
110  FORMAT(1X,' MAGNITUDE OF RESIDUAL = ', E15.6/,1X,
1     ' MAGNITUDE OF DATA = ', E15.6/,1X,
2     ' RELATIVE ERROR IN THE CALCULATION = ', E15.6)
      RETURN
      END
C
C *****
C
      SUBROUTINE HANKEL(X,NMAX,H)
      IMPLICIT REAL*8 (A-H,O-Z)
      COMPLEX*16 H(0:NMAX)
C
C  GIVEN THE VARIABLE X, AND THE MAXIMUM ORDER NMAX,
C  THIS ROUTINE GENERATES THE SPHERICAL HANKEL FUNCTION HN
C  FOR ALL N FROM 0 TO NMAX (INCLUSIVE)
C  INPUT:
C    X = DOUBLE PREC. VARIABLE (RADIUS)
C    NMAX = INTEGER MAXIMUM ORDER OF BESSEL FUNCTIONS DESIRED
C  OUTPUT:
C    H(N) = ARRAY OF SPHERICAL HANKEL FUNCTIONS HN(X), WHERE
C          HN = JN + I YN
C
C  THIS ROUTINE IS BASED ON THE RECURSION FORMULAE
C  FROM ABRAMOWITZ & STEGUN: 10.1.10 & 10.1.15, PP.438-9
C  THE F' S ARE THE COEFFICIENTS OF ORDER N & -(N+1),
C  THE FO' S ARE OLD F' S, FOR RECURSION
C
      IF ( X .LE. 0.0D0 ) THEN
        H(0) = DCMLPX(1.0D0,-1.0D35)
        DO 2 N = 1, NMAX
          H(N) = CMPLX(0.0D0,-1.0D35)
2      CONTINUE
        RETURN
      END IF
      SX = DSIN(X)
      CX = DCOS(X)
      XINV = 1.0D0/X
      M1N = -1.0D0
      FN = XINV
      FMN = 0.0D0
      FNO = FMN
      FMNO = FN
      DO 4 N = 0, NMAX
        H(N) = CMPLX( FN*SX + M1N*FMN*CX, -FN*CX + M1N*FMN*SX )
        T1 = (2*N+1)*XINV
        T2 = T1*FN - FNO
        FNO = FN
        FN = T2
        T2 = -T1*FMN - FMNO

```

```

      FMNO = FMN
      FMN = T2
      M1N = -M1N
4 CONTINUE
      RETURN
      END
C
C .....
C
C SUBROUTINE LEGNDR(THETA,NMAX,PMN)
C IMPLICIT REAL*8 (A-H,O-Z)
C REAL*8 PMN(-NMAX:NMAX,0:NMAX)
C
C GIVEN THE VARIABLE THETA, AND THE MAXIMUM ORDER NMAX,
C THIS ROUTINE GENERATES THE ASSOC. LEGENDRE FUNCTIONS PMN
C OF THE ARGUMENT COS(THETA) (THETA MUST BE BETWEEN 0 & PI)
C FOR ALL N FROM 0 TO NMAX (INCLUSIVE)
C AND FOR ALL M FROM -N TO N (SOME OTHERS SET TO ZERO)
C INPUT:
C THETA = VARIABLE (POLAR ANGLE), MUST BE BETWEEN 0 & PI (INCL.)
C NMAX = INTEGER MAXIMUM ORDER OF LEGENDRE FUNCTIONS DESIRED
C OUTPUT:
C PMN = DOUBLE PREC. ARRAY, CONTAINS ASSOC. LEGENDRE FNS
C
C THIS ROUTINE IS BASED ON THE RECURSION FORMULAE
C FROM ABRAMOWITZ & STEGUN
C
      X = DCOS(THETA)
      SINHT = DSIN(THETA)
      IF ( SINHT .GT. 0. ) THEN
        SININV = 1.0D0/SINHT
      ELSE
        SININV = 0.0D0
      END IF
C SET VALUES FOR N = 0, 1 (NMAX MUST BE AT LEAST 1)
      PMN(0,0) = 1.0D0
      PMN(1,0) = 0.0D0
      PMN(-1,0) = 0.0D0
      PMN(0,1) = X
      PMN(1,1) = -SINHT
      PMN(-1,1) = SINHT*0.5D0
C IN LOOP, TNP1 = 2*N+1, TNP2FC = (2*N+2)!, M1N = (-1)**(N+1)
      TNP1 = 1.0D0
      TNP2FC = 2.0D0
      M1N = -1.0D0
      DO 4 N = 1, NMAX-1
        TNP1 = TNP1 + 2.0D0
        TNP2FC = TNP2FC * TNP1 * (TNP1+1)
        M1N = -M1N
        DO 3 M = -N, N
          PMN(M,N+1) = (TNP1*X*PMN(M,N) - (N+M)*PMN(M,N-1))/(N-M+1)

```

```

3  CONTINUE
   PMN(N+1,N) = 0.0D0
   PMN(-N-1,N) = 0.0D0
   PMN(N+1,N+1) = (X*PMN(N,N+1) - TNP1*PMN(N,N)) * SININV
   PMN(-N-1,N+1) = M1N*PMN(N+1,N+1)/TNP2FC
4  CONTINUE
C   DO 120 N=0,NMAX
C   DO 120 M=-N,N
C120 WRITE(6,130) N,M,PMN(M,N)
C130 FORMAT(1X,' N= ',I4,1X,' M= ',I4,1X,' PMN= ',F13.6)
      RETURN
      END

```

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