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DROPLET-TURBULENCE INTERACTIONS OVER A WIDE SPECTRAL RANGE

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I ABSTRACT

In the last year, the unsteady, three-dimensional, incompressible, viscous flow interactions between a single vortex tube advected by a uniform free stream and a spherical particle held fixed in space was investigated numerically for a range of particle Reynolds numbers between 20 and 100. Useful correlations of lift coefficient, moment coefficient, and drag coefficient with velocity fluctuation, Reynolds number, offset distance, and initial vortex size have been obtained and reported. A new mechanism based upon droplet lift has been suggested for the dispersion of sprays. Since the beginning of this year, the interactions between *a pair of vortex tubes* and a rigid sphere have been studied in order to generalize the findings from the previous investigation. Similar correlations for the force and moment coefficients have been found and are being reported. These correlations will be useful in predicting droplet trajectories. The investigation for the heat and mass transfer of a droplet interacting with vortex tubes is also under way. This should lead to useful correlations to predict droplet heating and vaporization in a flow with vortical fluctuations.

II OBJECTIVES

The objectives of this research program are to investigate the interactions of vaporizing droplets with a turbulent field of the type encountered in gas turbine combustors. It is intended to develop predictive capability through the use of correlations. There is special interest in the important and challenging high-frequency end of the energy spectrum where turbulent length scales are comparable to droplet size. The full Navier-Stokes equations were numerically solved and a simple mathematical description for the turbu-

lent velocity fluctuation was employed. In the mathematical description, turbulent-like fluctuations were simulated in a controlled way by introducing cylindrical vortices which have a length scale of the order of that of the droplet and a strength corresponding to a turbulent velocity fluctuation. From the calculations, instantaneous lift, drag, and torque coefficients, Nusselt number, and Sherwood number are determined. Time-averaged values of these fluctuating quantities are also determined. Such quantities should be useful in modelling droplet dispersion and modifications of heating and vaporization rates due to turbulence.

III SUMMARY OF RESEARCH

In the past year, the unsteady, three-dimensional, incompressible, viscous flow interactions between a single vortex tube advected by a uniform free stream and a spherical particle held fixed in space was investigated numerically for a range of particle Reynolds numbers between 20 and 100. The paper describing this investigation was accepted in *Journal of Fluid Mechanics* for publication and is in press. The final version of the manuscript is appended to this report. This investigation was also presented at the 46th annual meeting of the Division of Fluid Dynamics, American Physical Society, Albuquerque, New Mexico, November, 1993. The abstract is appended to this report. Since the beginning of this year, the interactions between *a pair of vortex tubes* and a rigid sphere have been studied in order to generalize the findings from the previous investigation. The paper describing this study has been written as a AIAA preprint for the 1995 AIAA Aerospace Sciences Meeting in Reno, Nevada. The main results are summarized in this report. The preprint is also appended to this report. The investigation for the heat and mass transfer of a droplet interacting with vortex tubes is also under way, and the

progress is summarized in this report.

For the sake of clarity, we hereby present our results under two separate sections; first, we will present results primarily pertaining to the fluid dynamics of the interactions. Next, under a separate section, we will discuss our progress on simulating the temperature field en route to computing the vaporization and species transport.

III.A. INVESTIGATING THE FLUID DYNAMICS OF INTERACTIONS

The unsteady, three-dimensional, incompressible, viscous flow interactions between *a pair of vortex tubes* advected by a uniform free stream and a rigid sphere held fixed in space have been investigated numerically in order to generalize the findings from the previous investigation (Kim, Elghobashi & Sirignano (JFM 1995)) concerned with a rigid sphere interacting a single vortex tube. A summary of the findings and their applications is provided as follow.

(i) The effects of the size and the offset distance of the pair of vortex tubes on the flow field were examined for $20 \leq Re \leq 100$. The lift and moment coefficients are found to be linearly proportional to the maximum fluctuation velocity (v_{max}) induced by the pair of vortex tubes of given size (σ) and offset distance of the vortex tube. The rms lift coefficient depends on v_{max} but is independent of σ when $\sigma \geq 2$. For very small values of σ , the lift coefficient depends linearly upon the circulation of the vortex tube. Furthermore, the equation for the lift coefficient of the sphere interacting with a single vortex tube is applicable to the case of a pair of vortex tubes when the separation distance between their centers is less than $2\sqrt{\sigma}$ vortex tube diameters. Similarly, the single tube results apply for the moment coefficient when the separation is less than $\sqrt{\sigma}$ vortex tube diameters. These separation distance limits are invariant within a range of Reynolds

number $20 \leq Re \leq 100$. The expression of the rms lift coefficient for a single vortex tube is written here for later use.

$$C_{L,rms} = 8.1 v_{max} Re^{-0.45}, \quad 2 \leq \sigma \leq 4, \quad (1)$$

where v_{max} is the maximum fluctuation velocity normalized by the free stream velocity.

(ii) The results in (i) can be applied to turbulent flows in order to obtain the rms lift force on a particle in dilute conditions. A turbulent flow possesses a wide spectrum of eddy sizes. The large eddies contain most of the turbulent kinetic energy and produce high velocity fluctuations, and so they are responsible for much of the dispersion of particles. However, the eddies that are orders of magnitude larger than the droplets or particles will move the droplets in a circulatory fashion by drag forces. This results in the global effect of droplets moving from regions of high concentration to regions of low concentration. The smaller eddies will move the droplets by a lift force whose direction is related to the direction of rotation. The particle size, at the two interesting extremes, may be small or comparable to the Kolmogorov length scale. When the size of particle is small compared to the Kolmogorov length scale, the rms lift coefficient of the particle is obtained by equation (1). In the other case, when the size of particle is comparable to the Kolmogorov length scale, the rms lift coefficient of the particle can still be obtained approximately by equation (1). The time during which the particle is influenced by the eddy is of the order of the shorter time of the eddy residence time in the particle vicinity or the eddy lifetime.

The deflection of the particle path will depend on the magnitude of the rms lift coefficient and the ratio, ρ_r , of the particle density to that of the carrier fluid ($A = \frac{3}{8} C_L / \rho_r$, where A is the dimensionless acceleration of the particle due to the lift force). This result provides a simple method to estimate the deflection of the particle trajectory in the dilute particle-laden turbulent flow. For example, for initial particle Reynolds number 20 and

the eddy size $2 \leq \sigma \leq 4$, we can integrate A twice to show that the angle of the droplet deflection is given by $\tan \theta = 5 v_{max} / \rho r$. This shows that the droplet deflection is important in the turbulent flow of low density ratio between the droplet and the carrier fluid such as in a combustor with high pressure. In near critical conditions with $v_{max} = 0.1$ or greater, the tangent of the deflection angle becomes of order of unity. At low pressure, the deflection angle is very modest. Equation (1) and the nondimensionalized Newton's second law show that the deflection decreases slowly as Reynolds number increases.

Note that the trajectory deflection for individual droplets results in dispersion for a spray. Furthermore, the driving mechanisms for the dispersion under analysis here extend beyond the well-known mechanism where large eddies (through the drag force on the droplets) "sweep" more of the droplets from regions of high number density to regions of low number density than in the opposite direction. Here, a new mechanism is identified and added; it relies on lift generated by interactions with smaller eddies. The new mechanism promises to be competitive if large velocity gradients appear due to the smaller vortical structures and if the velocity fluctuations from larger eddies are not too much larger than the velocity fluctuations from the smaller eddies. (This implies that the Reynolds number based upon the integral length scale and its associated velocity is not too large.)

(iii) The magnitude of the rms moment coefficient of the particle is one order of magnitude less than that of the rms lift coefficient when $Re \geq 20$. Furthermore, when the initial size of the vortex core is considerably larger than the sphere size ($\sigma \geq 4$), the effect of the shear flow (induced by the passage of the vortex tube) across the sphere diminishes and the torque on the particle decreases. Thus, the torque on the particle might be negligible in many applications.

(iv) When the top and bottom vortex tubes have positive and negative circulations, respectively, the induced velocity due to the vortex tubes adds its magnitude to the base flow along the stagnation streamline. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be higher than those of the axisymmetric flow past a sphere. As a consequence, the drag is increased. On the other hand, when the top and bottom vortex tubes have negative and positive circulations, respectively, the induced velocity due to the vortex tubes subtracts its magnitude from the base flow along the stagnation streamline. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be lower than those of the axisymmetric flow past a sphere. As a consequence, the drag is decreased.

III.B. INVESTIGATING THE TEMPERATURE FIELD AND VAPORIZATION

The foregoing fluid dynamical study is being extended to an investigation of the effects of advecting vortical tubes on the corresponding temperature field, vaporization, and species transport near liquid droplets.

The problem is inherently three-dimensional and non-linear; in particular, the moderate range of the Reynolds number considered and also the advection of the vortical structures in the domain further complicate the non-linear effects. Previous computational observations have thus indicated some difficulties in achieving numerical stability. To avoid such potential obstacles, the following systematic approach is considered:

(i) Starting with the existing Navier-Stokes solver, we include modules for solving the partial differential equations of the temperature field for both the gas phase and the liquid droplet interior. Thus, information on the temperature field inside and outside the droplet becomes available and the thermal field is mapped. Initially, the vortices are not included

in this step so that we expect axisymmetric solutions in this developmental part of the program. Dependence of the Nusselt number on the Reynolds number will be explored. The vortices will then be included in the domain and the variation of Nusselt number in a constant property field due to the advecting vortex tubes will be investigated.

We have had progress in computing the scalar quantities and we are currently near the end of this step. More precisely, we have computed the temperature field in both the gas and liquid field for a constant property domain. Figures (1) and (2) show the axisymmetric temperature field for both the gas and liquid phase, respectively, for an upstream gas temperature of 1000 K , an octane fuel droplet of 300 K initial temperature, $Re_{gas} = 100$, and Prandtl numbers of $Pr_{gas} = .7$ and $Pr_{liq} = 8.5$. (In both figures, the upstream flow is from right to left.)

The illustrative figures are those of the domain after 25 residence time units. In this constant property-simulation, the liquid droplet temperature observed through a planar cross-section shows that the hot upstream gas flow transfers heat to the liquid which is convected internally. Liquid recirculation results in the heated fluid moving aft to fore along the axis of symmetry. The figure shows the early heating of the droplet interior as the hot recirculating fluid moves towards the axis. This behavior is in quantitative agreement with previous axisymmetric predictions. Note that the scalar computation is performed here with a fully three-dimensional code. The axisymmetric solution only serves as a developmental benchmark.

We are about to include vortical interactions in this heat transfer computation. This will yield information and correlations about the modification of Nusselt number due to turbulent fluctuations.

(ii) Upon successful completion of the previous step, we will extend the modules to

simulate the liquid droplet vaporization and species transport in the gas phase. Likewise, dependence of Sherwood number on Reynolds number and its variation due to the advecting vortices will be computed.

To further assure numerical stability through a step-by-step monitoring of the numerical solution, we initially assume a constant property (density, viscosity, thermal conductivity, and heat capacity) domain in both the liquid and gas phase. (In principle, this is a simplified simulation and may be regarded as representing a cold liquid droplet in a hot liquid stream or in a hot incompressible gas stream.) The simulation will be extended to a variable property domain.

Finally, when investigating the effect of the advecting vortical tubes on the aforementioned scalar quantities, we will pursue the same perspective followed in investigating their effect on the fluid dynamics properties, described earlier in the report. Namely, useful correlations for explaining the impact of turbulence on droplet heating and vaporization will be obtained.

IV PUBLICATIONS

The following papers have resulted from the research performed under this research program.

1. Kim, I., Elghobashi, S. & Sirignano, W. A., "Three-dimensional flow interactions between a cylindrical vortex tube and a sphere," presented at the 46th Annual Meeting of the Division of Fluid Dynamics, American Physical Society, Albuquerque, New Mexico, November 21-23, 1993.
2. Kim, I., Elghobashi, S. & Sirignano, W. A., "Unsteady flow interactions between an

advected cylindrical vortex tube and a spherical particle," *J. Fluid Mech.* in press, 1995.

3. Kim, I., Elghobashi, S. & Sirignano, W. A., "Unsteady flow interactions between a pair of advected cylindrical vortex tubes and a rigid sphere," AIAA Preprint 95-0105, the 33rd Aerospace Sciences Meeting and Exhibit, Reno, Nevada, January 9-12, 1995.

V PROFESSIONAL PERSONNEL

W. A. Sirignano, Professor, Principal investigator.

S. E. Elghobashi, Professor.

I. Kim, Research associate, Ph. D. January 1990 – present.

M. Masoudi, Ph.D. student, research assistant.

VI INTERACTIONS

Our papers have been presented at conferences of the American Institute of Aeronautics and Astronautics and of the American Physical Society. Also, our results have been presented at the AFOSR Contractors Meeting. All of these presentations have led to informal interactions with government laboratory researchers and industrial representatives. Discussions about the practical implications of this research have been held with Dr. Melvin Rocquemore at Wright Patterson Air force Aeronautical Laboratory and with Dr. Hukam Mongia, formerly of Allison Gas Turbine Division and currently of General Electric. In recent discussions, it was agreed that we should examine a collaboration to apply the research results of this study soon after the final announcement on the DOD Focused Initiative is made.

Visibility for the research has also come from research seminar presentations at other universities and from major invited papers such as the ASME Freeman Scholar paper that appeared in the Journal of Fluids Engineering in late 1993.

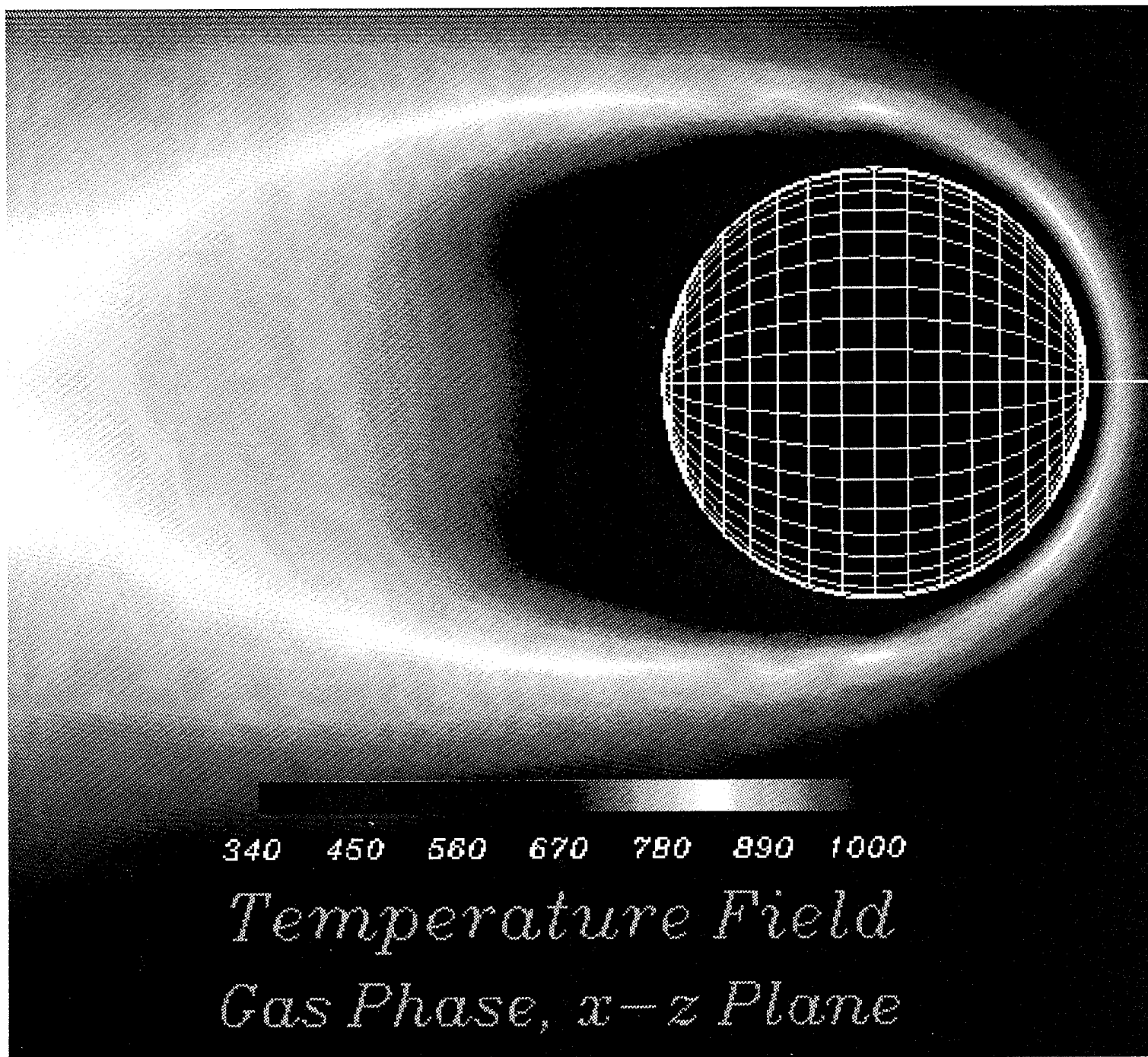


Figure 1. The gas phase temperature field in the absence of advecting vortices.

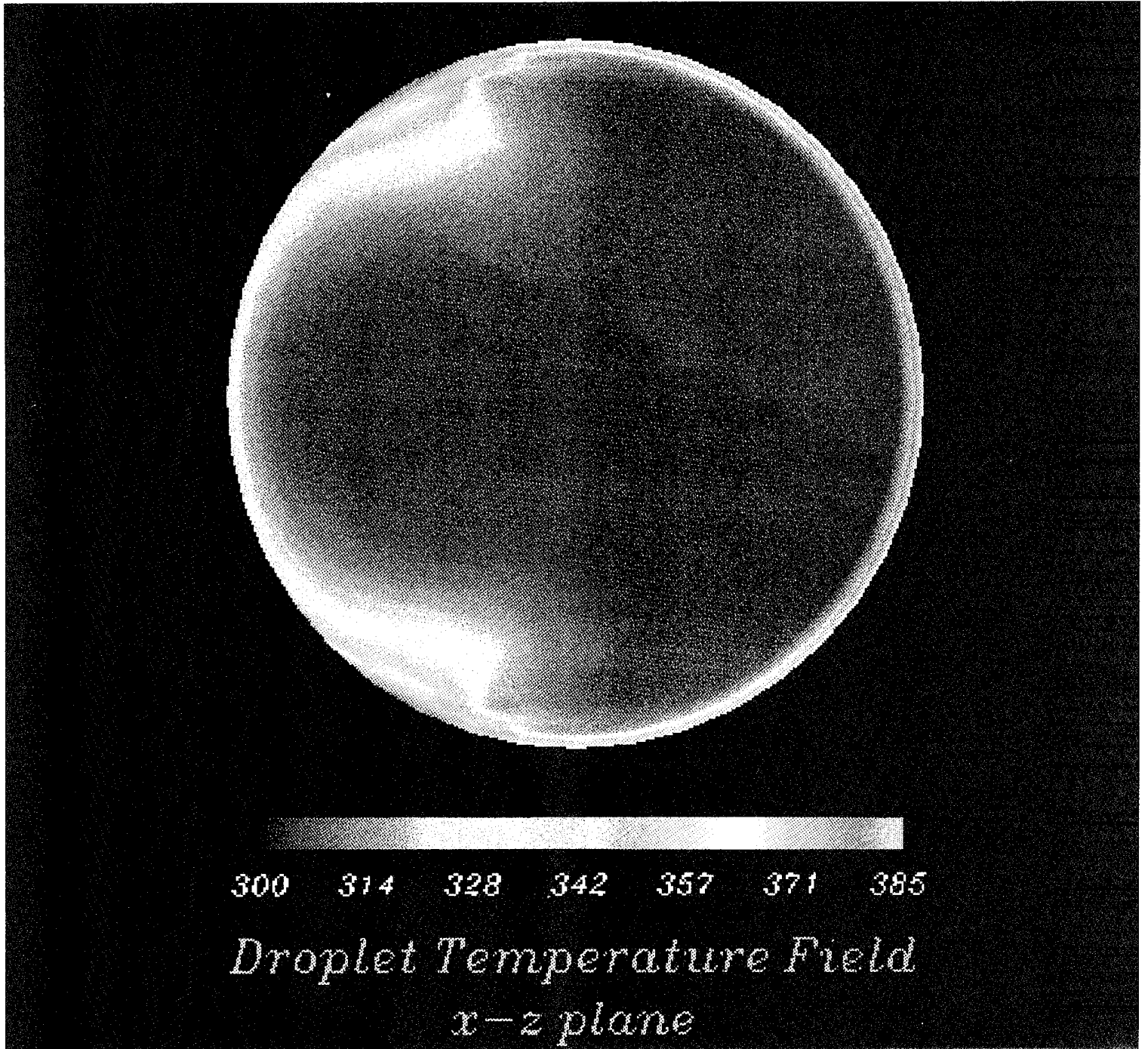


Figure 2. The temperature field inside the droplet in the absence of advecting vortices.

To appear in
J. Fluid Mech.

UNSTEADY FLOW INTERACTIONS BETWEEN AN ADVECTED CYLINDRICAL VORTEX TUBE AND A SPHERICAL PARTICLE

by

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Abstract

The unsteady, three-dimensional, incompressible, viscous flow interactions between a vortical (initially cylindrical) structure advected by a uniform free stream and a spherical particle held fixed in space is investigated numerically for a range of particle Reynolds numbers $20 \leq Re \leq 100$. The counter-clockwise rotating vortex tube is initially located ten sphere radii upstream from the sphere center. The finite-difference computations yield the flow properties and the temporal distributions of lift, drag, and moment coefficients of the sphere. Initially, the lift force is positive due to the upwash on the sphere, then becomes negative due to the downwash as the vortex tube passes the sphere. Varying the size of the vortex core (σ) shows that the rms lift coefficient is *linearly* proportional to the *circulation* of the vortex tube at small values of σ . At large values of σ , the rms lift coefficient is linearly proportional to the maximum fluctuation velocity (v_{max}) induced by the vortex tube but *independent* of σ . For intermediate values of σ , the rms lift coefficient depends on both σ and v_{max} (or equivalently both σ and the circulation). We observe some interesting flow phenomena in the near wake as a function of time due to the passage of the vortex tube.

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1 Introduction

This paper is concerned with the unsteady, three-dimensional, incompressible, viscous flow interactions between a vortical structure (initially cylindrical) advected by a uniform free stream and a spherical solid particle which is held fixed in space. This flow is equivalent to that of a spherical particle moving along a straight line and traversing the vortical structure at constant velocity. The particle Reynolds number based on the freestream velocity and particle diameter is in the range $20 \leq Re \leq 100$. We obtain the unsteady velocity and pressure distributions via the numerical solution of the time-dependent three-dimensional Navier-Stokes equations within a spherical domain surrounding the sphere and the moving vortex tube.

The motivation for studying this flow is the need to understand how the forces (drag, lift, and torque) imparted on a particle are influenced by fluctuations in the velocity and pressure of the carrier flow as is the case in particle-laden turbulent flows. Knowledge of the time-dependence of these forces and the unsteady flow field is essential for the accurate calculation of the particle trajectory and the heat and mass transfer rate of the particle (or droplet) which in turn is a prerequisite for predicting particle dispersion and vaporization rate in turbulent flows. The exact relations between these forces and the turbulent fluctuations cannot be obtained analytically due to the nonlinearity of the equations governing the motion of the particle and fluid. While the cylindrical vortex is far too simple to represent real turbulence, some important elementary understanding can result from this study.

Numerical simulation of the dispersion of particles in a turbulent flow requires the solution of the equation of particle motion. This equation which is classically known as the BBO (Basset-Boussinesq-Oseen) equation and has been re-derived recently by Maxey & Riley (1983) is restricted to low Reynolds number $Re \ll 1$, where $Re = d' |\mathbf{u}' - \mathbf{v}'|/\nu'$; \mathbf{v}' and \mathbf{u}' are the velocities of the particle and its surrounding fluid respectively, d' is the

particle diameter, and ν' is the fluid kinematic viscosity. Furthermore, the drag force in that equation consists of two terms, namely, the quasi-steady Stokes drag and the unsteady memory term (Basset). The former is purely viscous, whereas the latter depends on both the viscosity and particle acceleration relative to the fluid. The superposition of these two terms is a result of the linearization of the Navier-Stokes equations by Basset (1888). A more serious restriction (than $Re \ll 1$) in the equation of particle motion is that the velocity gradients in the carrier flow in the neighborhood of the particle should be very small. This requires that the shear Reynolds number $= (a'^2/\nu')(U'_o/L') \ll 1$, where a' is the particle radius, and (U'_o/L') is a reference gradient of the undisturbed velocity field. Therefore, the interesting case in which the eddy (or vortex) size is comparable to that of the particle cannot be properly treated by the standard equation of particle motion. This situation, in addition to being relevant to the fundamental understanding of fluid dynamics, is of practical interest as well. For example, in a typical gas turbine combustor where the Reynolds number is of the order of 10^5 and the integral length scale is of the order of $0.1m$, the smallest (Kolmogorov) length scale, η , is about $100\mu m$, which is comparable to the size of a typical fuel droplet. Fluid motion at the Kolmogorov length scale experiences the largest strain rates and scalar gradients in the flow. The largest scalar gradients control the important phenomena of heat and mass transfer and chemical reaction. Motion at the largest length scales ($\gg \eta$) contains most of the turbulence energy and governs the dispersion of particles (or droplets) but not the small-scale phenomena mentioned above.

Almost all application-oriented studies of dilute particle suspension calculate the drag on the particle using the standard drag curve. This drag curve has been obtained (experimentally and numerically) for a particle fixed in space subjected to a steady flow. In the case of unsteady flow, this drag relationship is an approximation that can be valid only if the time-scale of the particle motion is much larger than that of the flow. Note that empirical relationships for unsteady drag have been proposed (e.g., Houghton(1963),

Odar(1966), Schöneborn(1975), and Ingebo(1956)). The expression of Ingebo, derived experimentally, is valid only for the limited conditions of the experiment. All other expressions concern mainly purely harmonic flows. Additional interactions between the particle and the flow are the well known Saffman's lift due to uniform shear (1965, 1968) and the lift due to particle rotation (Rubinow & Keller(1961)). Saffman's lift force expression is valid only for $Re \ll R_{shear}^{1/2}$ and $R_{shear} \ll 1$, where $R_{shear} = (du'/dy')d'^2/(4\nu')$. Under these conditions, the lift due to particle rotation is negligible (Saffman(1965)). Recently, McLaughlin (1991) removed the restriction $Re \ll R_{shear}^{1/2}$ and provided a new form for the lift force.

Three-dimensional flow interactions between a vortical structure and a bluff body (a rigid sphere in the simplest form) at finite Reynolds number have not been investigated yet. Our present approach, outlined at the beginning of this section, is a first step toward better understanding of the physics of interaction between a particle and the carrier turbulent flow. For example, we examine the details of the temporal behavior of the flow structure around the sphere due to the passage of the vortex tube. Furthermore, we study the effects of varying the ratio of vortex tube size to particle size, Reynolds number, and offset distance between the particle and the vortex tube on the temporal distributions of the forces imparted on the particle (drag, lift, and torque) and the flow structure in the neighborhood of the particle.

The next section provides a mathematical description of the flow considered, the governing equations and the numerical solution procedure. Section 3 discusses the results including the numerical accuracy issues and the effects of varying the parameters listed above. Section 4 provides a summary of the work.

2 Problem statement and formulation

2.1 The flow description

We consider the time-dependent, three-dimensional, incompressible, viscous flow interactions between an initially cylindrical vortex tube and a spherical solid particle. The vortex tube is moving with the laminar free stream, and a sphere is suddenly placed and held fixed in space as shown in figure 1. The initial offset distance, d_{off} , denotes the shortest distance, normalized by the sphere radius, between the initial vortical axis and the y - z plane, which is parallel to the free stream. All the variables are nondimensionalized using the sphere radius a'_o as the characteristic length and U'_∞ as the characteristic velocity, where the superscript $'$ denotes dimensional quantity. The cylindrical vortex tube, whose diameter is of the order of the sphere diameter, is initially located ten radii upstream from the center of the sphere. The effects of the vortex tube on the sphere are negligible at this initial distance because the magnitude of the initial velocity field induced by the vortex tube is less than 2 percent of the free stream velocity. Far upstream, the flow is uniform with constant velocity $U'_\infty \mathbf{k}$ parallel to the y - z plane. We have one symmetry plane, the x - z plane, as seen in figure 1.

Two coordinate systems are used in our formulation: the Cartesian coordinates (x,y,z) and the nonorthogonal generalized coordinates (ξ, η, ζ) . The origin of the former coincides with the sphere center. ξ is the radial, η is the angular, and ζ is the azimuthal coordinates. The nonorthogonal generalized coordinate system can be easily adapted to three-dimensional arbitrary geometries. In the present study, a spherical domain is used, and the grid reduces to an orthogonal, spherical grid. The grids are denser near the surface of the spherical particle, and the grid density in the radial direction is controlled by the stretching function developed by Vinokur (1983). Due to symmetry, the physical domain is reduced to a half spherical space. The domain of the flow is bounded by $1 \leq \xi \leq N_1$,

$1 \leq \eta \leq N_2$, $1 \leq \zeta \leq N_3$, where $\xi = 1$ and N_1 correspond, respectively, to the sphere surface and the farfield boundary surrounding the sphere; $\eta = 1$ and N_2 denote, respectively, the positive z -axis and the negative z -axis; $\zeta = 1$ and N_3 refer, respectively, to the x - z plane in the positive x -direction and the x - z plane in the negative x -direction. Uniform spacing ($\delta\xi = \delta\eta = \delta\zeta = 1$) is used, for convenience, for the generalized coordinates.

2.2 The vortex tube features

The initial vortex tube has a small core region with a radius σ (normalized by the sphere radius). This core is defined such that the initial velocity induced by the vortex tube approaches zero as the distance from the center of the vortex tube goes to zero, and at distances much greater than σ , the induced velocity approaches that of a point vortex (figure 2). We use the vortex tube construction of Spalart (1982), which has the following stream function:

$$\psi_v(x, z, t = 0) = -\frac{\Gamma_v}{2\pi} \ln[(x - x_o)^2 + (z - z_o)^2 + \sigma^2], \quad (1)$$

where Γ_v is the nondimensional circulation around the vortex tube at radius σ and at the initial time. Γ_v is positive when the vortex tube rotates counterclockwise, and x_o and z_o denote the location of the center of the vortex tube. The circulation around a circular path far away from the center of the vortex is given by $\Gamma_t = 2\Gamma_v$. The tangential velocity distribution of the vortex tube compared with a point vortex is shown in figure 2 for $\Gamma_v = 2.5$ and $\sigma = 1.0$. As shown in figure 2, the cylindrical vortex tube can be viewed as an evolution from the point vortex due to the cylindrical viscous diffusion. The initial pressure field due to the vortex tube is obtained by solving the radial component of the Navier-Stokes equations which balances the centrifugal acceleration and the pressure gradient for circular streamlines, and has the following form

$$p_v(x, z, t = 0) = -\frac{\Gamma_v^2}{2\pi^2} \frac{1}{(x - x_o)^2 + (z - z_o)^2 + \sigma^2}, \quad (2)$$

where p_v is nondimensional pressure defined by $p_v = (p'_v - p'_\infty)/\rho'U'^2$. The pressure due to the vortex tube attains its lowest value, $p_{v,min} = -\Gamma_v^2/(2\pi^2\sigma^2)$, at the center of the vortex tube and approaches zero at far distances from the center of the vortex. Equation (2) is used to prescribe only the initial pressure field generated by the vortex tube.

In order to gain insight about the properties of the vortex tube, we examine the flow field generated in the absence of the particle. We compute the induced velocity and vorticity field as a function of radius and time due to the vortex tube moving with the free stream ($U'_\infty \mathbf{k}$). The origin of the moving coordinate system is the center of the vortex tube. We solve the following linear diffusion equation which is the tangential component of the Navier-Stokes equations balancing the unsteady and diffusion terms for the tangential momentum (Batchelor (1967)).

$$\frac{\partial u_\theta}{\partial t} = \frac{2}{Re} \left(\frac{\partial^2 u_\theta}{\partial R^2} + \frac{1}{R} \frac{\partial u_\theta}{\partial R} - \frac{u_\theta}{R^2} \right) \quad (3)$$

where R is the radial distance from the center of the vortex tube, u_θ is the tangential velocity around the vortex tube normalized by the free stream velocity, and Re is the Reynolds number based on the reference length scale a'_o and the freestream velocity. Figures 3(a) and 3(b) show respectively the velocity and vorticity fields as a function of radial distance and time for $Re = 100$, $\Gamma_v = 2.5$, and $\sigma = 1.0$. The size of the vortex core becomes larger as time increases due to viscous diffusion, whereas the magnitudes of the tangential velocity and the vorticity inside the vortex core decrease. Note that this classical linearized analysis is not employed in the present study; rather a fully nonlinear computational analysis is performed.

2.3 Governing equations and boundary conditions

The continuity and momentum equations and the initial and boundary conditions are nondimensionalized using the sphere radius a'_o as the characteristic length and U'_∞ as the characteristic velocity.

$$\nabla \cdot V = 0 \quad (4)$$

$$\frac{\partial V}{\partial t} + \nabla \cdot VV = -\nabla p + \frac{2}{Re} \nabla^2 V \quad (5)$$

The governing equations (4) and (5) are cast in terms of the generalized coordinates (ξ, η, ζ) to treat a three-dimensional body of arbitrary shape. The numerical integration is performed using a cubic computational mesh with equal spacing ($\delta\xi = \delta\eta = \delta\zeta = 1$).

The velocities on the sphere surface are zero due to the no-slip condition, and the pressure on the sphere is obtained from the momentum equation. The boundary conditions are

$$\frac{\partial p}{\partial n} = \frac{2}{Re} \frac{\partial^2 V_n}{\partial n^2}, \quad u = v = w = 0 \quad \text{at } \xi = 1, \quad (6)$$

$$p = 0, \quad u = v = 0, \quad w = 1 \quad \text{at } \xi = N_1 \text{ and } N_{2mid} \leq \eta \leq N_2 \text{ (upstream)}, \quad (7)$$

$$p = 0, \quad \frac{\partial u}{\partial \xi} = \frac{\partial v}{\partial \xi} = \frac{\partial w}{\partial \xi} = 0 \quad \text{at } \xi = N_1 \text{ and } 1 \leq \eta < N_{2mid} \text{ (downstream)}, \quad (8)$$

$$\frac{\partial p}{\partial \zeta} = \frac{\partial u}{\partial \zeta} = \frac{\partial w}{\partial \zeta} = 0, \quad v = 0 \quad \text{at } \zeta = 1 \text{ and } N_3, \quad (9)$$

where u , v , and w are the velocities in the x , y , and z direction, respectively, V_n is the velocity in the direction normal to the sphere surface, and p is the pressure. n denotes the direction normal to the sphere surface, $\partial/\partial n = \sqrt{\xi_x^2 + \xi_y^2 + \xi_z^2} \partial/\partial \xi$, and $\eta = N_{2mid}$ denotes the mid-plane between $\eta = 1$ and N_2 . Equation (9) corresponds to the symmetry conditions and zero v velocity in the x - z symmetry plane. Conditions guaranteeing continuity in the η direction for p , u , v , and w on the axes $\eta = 1$ and $\eta = N_2$ are also imposed.

In order to start the numerical solution of equations (4) and (5), we provide initial velocity and pressure fields by superposing the flow fields due to the uniform stream and

the vortex tube in addition to the no-slip condition on the sphere surface:

$$p_o = p_v, \quad u_o = -\frac{\partial\psi_v}{\partial z}, \quad v_o = 0, \quad w_o = 1 + \frac{\partial\psi_v}{\partial x} \quad \text{except at } \xi = 1 \quad (10)$$

$$p_o = p_v, \quad u_o = v_o = w_o = 0 \quad \text{at } \xi = 1, \quad (11)$$

where ψ_v and p_v are given by equations (1) and (2), respectively.

The only nondimensional groupings appearing in the equations and initial and boundary constraints are the sphere Reynolds number, vortex tube radius, offset distance, and vortex circulation (or vortex Reynolds number).

The drag, lift, and moment coefficients are evaluated in dimensional form as follows.

$$F'_D = \int_S -p' \mathbf{n} \cdot \mathbf{k} \, dS' + \int_S \mathbf{n} \cdot \boldsymbol{\tau}' \cdot \mathbf{k} \, dS' \quad (12)$$

$$F'_L = \int_S -p' \mathbf{n} \cdot \mathbf{i} \, dS' + \int_S \mathbf{n} \cdot \boldsymbol{\tau}' \cdot \mathbf{i} \, dS' \quad (13)$$

$$M' = \int_S \mathbf{r}' \times \boldsymbol{\tau}' \, dS', \quad (14)$$

where S' denotes the surface of the sphere, \mathbf{n} is the outward unit normal vector at the surface, \mathbf{r}' is the position vector from the center of the sphere, and $\boldsymbol{\tau}'$ is the viscous stress tensor. The lift force is assumed positive when it is directed toward the positive x-axis. Due to symmetry, only the y-component of the moment is non-zero and is assumed positive in counter-clockwise direction.

The nondimensional coefficients of drag, lift, and moment are defined respectively as

$$C_D = \frac{F'_D}{\frac{1}{2} \rho' U_\infty'^2 \pi a_o'^2} \quad (15)$$

$$C_L = \frac{F'_L}{\frac{1}{2} \rho' U_\infty'^2 \pi a_o'^2} \quad (16)$$

$$C_M = \frac{M' \cdot j}{\frac{1}{2} \rho' U_\infty'^2 \pi a_o'^3} \quad (17)$$

Note that in this analysis, the sphere does not accelerate or rotate due to the aerodynamic forces and torque.

2.4 Numerical solution

We have developed a three-dimensional, implicit, finite-difference algorithm to solve simultaneously the set of the discretized partial differential equations. The method is based on an Alternating-Direction-Predictor-Corrector (ADPC) scheme to solve the time-dependent Navier-Stokes equations. ADPC is a slight variation of Alternating-Direction-Implicit (ADI) method. It is first-order accurate in time but is effective and implemented easily when embedded in a large iteration scheme (Patnaik 1986). The control volume formulation is used to develop the finite-difference equations from the governing equations with respect to the generalized coordinates (ξ, η, ζ) . One of the advantages of the control volume formulation is that mass and momentum are conserved over a single control volume, and hence the whole domain regardless of the grid fineness. An important part of solving the Navier-Stokes equations in primitive variables is the calculation of the pressure field. In the present work, a pressure correction equation is employed to satisfy indirectly the continuity equation (Anderson *et al.* 1984). The pressure correction equation is of the Poisson type and is solved by the Successive-Over-Relaxation (SOR) method.

The overall solution procedure is based on a cyclic series of guess-and-correct operations. The velocity components are first calculated from the momentum equations using the ADPC method, where the pressure field at the previous time step is employed. This estimate improves as the overall iteration continues. The pressure correction is calculated from the pressure correction equation using the SOR method, and new estimates for pressure and velocities are obtained. This process continues until the solution converges at

each time step.

3 Results and discussion

In subsection (3.1), we test the accuracy of the full three-dimensional solution procedure by predicting the axisymmetric flow over a single sphere and by examining the effects of grid resolution on the maximum lift coefficient of the sphere due to the interaction between a vortex tube and a sphere. In subsections (3.2), (3.3), (3.4), and (3.5), we discuss the three-dimensional interactions between a vortex tube and a sphere, the effects of the offset distance, the size of the vortex tube, and Reynolds number, respectively.

3.1 Numerical accuracy

Here we examine the flow generated by an impulsively started solid sphere in a quiescent fluid at two Reynolds numbers: 20 and 100. The time-dependent solution converges asymptotically to a steady-state which is in good agreement with the available experimental data and correlations as shown in tables 1 and 2. Table 1 lists the drag coefficients as a function of the computational grid density at Reynolds numbers 20 and 100 respectively, and compares them with the correlations of Clift *et al.* (1978). Table 2 shows the pressures at the front and rear stagnation points and the separation angle measured from the front stagnation point as a function of grid density at Reynolds number 20 and 100, in comparison with the data of Taneda (1956) and also with the correlations of Clift *et al.* (1978). Although the solution in these test cases are axisymmetric, none of the three velocity components in our formulation becomes identically zero. Therefore, the three-dimensional solution scheme is fully exercised here. The calculations were performed for three different grids, $(N_1 \times N_2 \times N_3) = (21 \times 21 \times 21)$, $(31 \times 31 \times 31)$, and $(41 \times 41 \times 41)$, in a computational domain with an outer boundary located at 21 sphere radii from the

sphere center.

We tested the solution procedure by varying the far-field boundary condition and by changing the location of the outer boundary. In the first test, the far-field outflow boundary condition was changed from $\partial\phi/\partial r = 0$ to $\partial\phi/\partial z = 0$ ($\phi = u, v, \text{ or } w$). There was almost no difference in the drag coefficient and the near wake size (the separation angle and length of the recirculation eddy) at Reynolds numbers 20 and 100. Our calculation shows that separation does not occur at Reynolds number 20. In the second test, the location of the outer boundary in downstream was changed from 21 to 41 sphere radii. There was virtually no change in the drag coefficient and the near wake size at both Reynolds numbers.

We examined the effects of grid resolution on the lift coefficients of the sphere due to the flow interaction between a cylindrical vortex tube flowing with the free stream and a sphere fixed in space at Reynolds numbers 20 and 100. The lift coefficients are obtained for the offset distance $d_{off} = 0$, vortex core radius $\sigma = 1$, and maximum fluctuation velocity $v_{max} = \Gamma_v/(2\pi\sigma) = 0.4$. Table 3 shows the maximum negative lift coefficient (C_{Lmax2}) of the sphere as a function of the computational grid density at Reynolds numbers 20 and 100. The calculations were performed for three different grids, $(N_1 \times N_2 \times N_3) = (21 \times 21 \times 21)$, $(31 \times 31 \times 31)$, and $(41 \times 41 \times 41)$ for $Re = 20$, and four different grids, $(N_1 \times N_2 \times N_3) = (21 \times 21 \times 21)$, $(31 \times 31 \times 31)$, $(41 \times 41 \times 41)$, and $(51 \times 51 \times 51)$ for $Re = 100$, in a computational domain with an outer boundary located at 21 sphere radii from the sphere center. The result of the $31 \times 31 \times 31$ grid differs by 0.42% from that of the $41 \times 41 \times 41$ grid for $Re = 20$, and the result of the $41 \times 41 \times 41$ grid differs by 2.5% from that of the $51 \times 51 \times 51$ grid for $Re = 100$. Figure 4 provides additional results on the effect of grid resolution on convergence and shows the distributions of the pressure and shear stress coefficients (normalized by the dynamic pressure) around the sphere in the x-z plane of symmetry in the positive x-direction for the same parameters as used above with $Re = 100$. The pressure and shear stress distributions were obtained

at $t = 12$ about which the lift coefficient reaches its maximum in negative value. The pressure coefficient at $(x,z) = (0, -1)$ of the $41 \times 41 \times 41$ grid differs by 0.93% from that of the $51 \times 51 \times 51$ grid. The same calculations were performed by changing the location of the outer boundary in downstream from 21 to 41 sphere radii. There was virtually no change in the lift, moment, and drag coefficients.

In order to examine the far-field boundary effects, we repeated the simulation as above for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = 0.4$ but with a box-type computational domain with symmetry boundary conditions on its sides. The lift, moment, and drag coefficients of the box-type computational domain at $t = 12$ differ by 0.12%, 0.17%, and 0.13% respectively from those of the spherical computational domain used in the present paper. The spherical computational domain gives a little finer resolution than does the box-type computational domain with the same number of grid points, and so smoother contour lines for the vorticity and stream lines in the x-z symmetry plane. We also solved the same problem as above by employing a complete computational domain without the symmetry plane and periodic boundary condition in ζ direction. In that case, the lift coefficients differ by 0.16% from those in table 3 where the symmetry condition was employed. The $41 \times 41 \times 41$ grid is used in the following calculations.

The run for the interaction between a vortex tube and a sphere at Reynolds number 100 with the $41 \times 41 \times 41$ grid required 2.62 mega words, a dimensionless time step of $\Delta t = 0.0025$, and a total time of 2.95 cpu hours on Cray Y-MP8/864 for the final time of $t_f = 24$. Each time step takes about 1.11 cpu seconds. Another test was performed to examine the effects of the time step. The same calculation as above was repeated for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = 0.4$ but with the time step reduced by half. The lift, moment, and drag coefficients differ by 0.25%, 0.19%, and 0.003% respectively from those obtained with the time step used in the presented results.

3.2 Interactions of a vortex tube and a sphere

We consider the interactions of a vortical structure advected by the free stream and a sphere suddenly placed in the flow and held fixed in space at Reynolds number 100. The vortical structure is initially a cylindrical vortex tube rotating counter-clockwise in figure 1 with a nondimensional radius of unity and an offset distance of zero, and located at 10 sphere-radii upstream from the center of the sphere.

3.2.1 Flow structure

In order to illustrate better the fluid motion, we consider the flow field in the x-z plane of symmetry, which is defined as the *principal plane*, where the strongest interactions occur between the vortical structure and the sphere.

Pseudo-streamlines are employed in the following illustrations. The pseudo-streamlines are obtained from the pseudo-stream function which is defined by assuming that the velocity field in the principal plane does not change in the perpendicular direction to the principal plane and by using the two-dimensional stream function definition. The sphere surface in the principal plane is used as a reference streamline ($\psi_{ps} = 0$). We note that a real stream function ψ cannot be defined and calculated from the velocity in the principal plane due to the existence of a divergence associated with the third component of velocity. Nevertheless, for descriptive purposes only, it is convenient to use the two-dimensional stream function definition to present approximations to the streamline pattern.

Figures 5(a)-(l) display the pseudo-streamlines (left column) and the contour lines of y-component vorticity (right column) in the principal plane at $t = 0, 1, 6, 9, 10, 11, 12, 13, 15, 18, 21,$ and 30 for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = \Gamma_v / (2\pi\sigma) = 0.4$. The contour values of the pseudo-streamlines are $0, \pm 0.02, \pm 0.1, \pm 0.3$. The contour values of the vorticity are $\pm 0.4, \pm 0.5, \pm 0.8, \pm 1.4, \pm 2$, with the highest magnitude at the sphere surface. The solid and dotted lines in the figures represent positive and nega-

tive values. Figures 6(a)-(j) show the pressure coefficient, $2(p - p_\infty)/\rho U_\infty^2$, and shear stress coefficient, $2\tau_{r\theta}/\rho U_\infty^2$, around the sphere in the principal plane, respectively, at $t = 1, 6, 9, 10, 11, 12, 13, 15, 18$, and 21. Note that figure 6(a) is for $t = 1$ which corresponds to figure 5(b).

At $t = 0$, figure 5(a) shows that spherical vortex sheet is generated around the sphere due to the no-slip condition at the sphere surface. The subsequent figures show that the vortex sheet around the sphere is advected downstream as well as diffused outwards from the sphere. The vorticity on the edge of the vortex core is 0.4 at $t = 0$ for $\Gamma_v = 2.51$ and $\sigma = 1$ which correspond to $v_{max} = \Gamma_v/(2\pi\sigma) = 0.4$. The vortex tube is initially cylindrical and thus should appear as a circle in the principal plane. But, the vortex tube in figure 5(a) is not an exact circle because the grid resolution is relatively coarse at the initial location of the vortex tube which is far upstream from the sphere and the linear interpolation is used to draw the contour lines. However, we calculate analytically the exact velocity and pressure fields induced by the vortex tube by using equations (1) and (2), and prescribe them as initial conditions. Therefore, the magnitudes of the initial velocity components at a given location (x,z) are fixed no matter what grid distribution is used. Thus, the circulation around a large circle enclosing the vortex tube remains the same as that of the vortex tube. The velocity and pressure fields as a function of time are almost not affected by the initial vortex tube shape obtained by the linear interpolation. The line connecting the front and rear stagnation points in the standard axisymmetric flow over a single sphere, which is the $x = 0$ line in the principal plane, will be used as a reference line. We refer to the region above the line as ‘upper’ region and that below the line as ‘lower’ region.

For $0 < t \leq 9$, the vortex tube is upstream of the sphere as shown in figures 5(b)-(d). The vortex tube rotating counter-clockwise produces downwash upstream of itself and upwash downstream. Therefore, the front stagnation point on the sphere is shifted below the plane $x = 0$ due to the upwash, and thus, the fluid particles in the upper left region

move faster than do those in the lower left region of the sphere. As a consequence, lower pressure and higher shear stress act in the upper left region compared to the lower left region as shown in figures 6(a)-(c), and this causes a positive lift force on the sphere. Note that in figures 6(a)-(j) the clockwise direction is considered positive for the shear stress in the upper region of the sphere, and counterclockwise direction is considered positive for the shear stress in the lower region. On the other hand, the shift of the front stagnation point below the plane $x = 0$ causes the fluid particles to continue to accelerate after $\theta = 90^\circ$ at the bottom of the sphere but to begin to decelerate before $\theta = 90^\circ$ at the top of the sphere where θ is measured from the negative z-axis, as shown in the pressure distribution around the sphere in the principal plane in figures 6(a)-(c). Thus, the fluid particles move faster in the bottom and lower right regions than in the top and upper right regions of the sphere. As a consequence, lower pressure and higher shear stress act in the bottom and lower right regions compared to the top and upper right regions as shown in figures 6(a)-(c), and this causes the fluid particles turning around the upper eddy to be pushed into the lower region of the near wake as shown in figures 5(c) and 5(d). Figures 5(c) and 5(d) also show that the upper eddy is formed by the fluid separating on the upper portion of the sphere as in the case of axisymmetric flow past a sphere without the presence of the vortex tube. On the other hand, the lower eddy is not formed by the fluid separating on the lower portion of the sphere, but rather by the fluid turning around the upper eddy and being entrained by the lower flow. This lower eddy is detached from the sphere. A portion of the fluid moving around the top of the sphere passes between the detached lower eddy and the sphere. A similar flow pattern was found by Kim, Elghobashi & Sirignano (1993) in their study of three-dimensional flow over two spheres placed side by side.

For $9 < t \leq 10$, the figures 5(d) and 5(e) show that the vortex tube contacts the boundary layer of the sphere.

For $10 < t \leq 13$, the figures 5(f)-(h) show that the vortex tube goes around the bottom

of the sphere. The vortex tube is now downstream of the front stagnation point in the axisymmetric flow past a sphere and produces downwash on the sphere. Therefore, the front stagnation point on the sphere is shifted above the plane $x = 0$, and thus the fluid particles in the lower left region move faster than do those in the upper left region of the sphere. As a consequence, lower pressure and higher shear stress act in the lower left region compared to the upper left region as shown in figures 6(e)-(g), and this causes the negative lift force on the sphere. On the other hand, the shift of the front stagnation point above the plane $x = 0$ causes the fluid particles to move faster in the boundary layer of the top and upper right regions compared to that of the bottom and lower right regions of the sphere as shown in figure 7, which shows the tangential velocity profiles, $u_\theta(r)$, at $\theta = 90^\circ$ on the top and the bottom of the sphere in the principal plane at $t = 12$, and this causes the higher shear stress in the top and upper right regions compared to the bottom and lower right regions as shown in the figures 6(f) and 6(g). However, during this time period, the pressure distributions at the top and bottom of the sphere in the principal plane have different features from those of the shear stress due to the following reason. The counter-clockwise vortex tube in the uniform stream produces a flow field in which the fluid velocity is less than that of the uniform stream above the vortex tube and higher than that of the uniform stream below the vortex tube with respect to fixed coordinate system in space. Due to this shear flow, the fluid velocity on the edge of (and outside) the boundary layer at the bottom of the sphere is larger than that at the top of the sphere as shown in figure 7, and thus the pressure at the bottom of the sphere is lower than that at the top of the sphere. This pressure difference causes the fluid particles turning around the top of the sphere to be pushed into the lower region of the wake forming an S-shaped path (figures 5(g) and 5(h)). The combined effect of the upward shift of the front stagnation point due to downwash of the vortex tube and the velocity difference between the top and the bottom of the sphere due to the shear flow induced by the vortex tube results in a higher magnitude of the maximum negative force than that of the maximum positive

force, as will be shown in detail in section of 3.2.2. The upper separation eddy becomes smaller during this time period, because the pressure difference between the upper and lower wake just downstream of the sphere becomes larger when the vortex tube passes the plane $z = 1$ (tangent to the rear stagnation point in axisymmetric flow), and more fluid particles are pushed into the lower wake just behind the sphere. At $t = 12$ and 13 , no separation eddies appear in the wake as shown in figures 5(g) and 5(h), and the flow does not separate in upper region of the sphere as shown in figures 5(g), 5(h), 6(f), and 6(g). We note that the separation point in the principal plane is the point at which the shear stress vanishes.

The reason for the passage of the counterclockwise-rotating vortex tube around the bottom of the sphere rather than around the top is as follows. First, note that the well-known two-dimensional, inviscid case of a vortex interacting with cylinder has a counterclockwise (clockwise) rotating vortex rotating clockwise (counterclockwise) around the cylinder. In our case, the opposite behavior suggests that viscosity is important in this phenomenon. Note further that the vorticity levels associated with the viscous boundary layer on the sphere are greater than those associated with the tube. When the counterclockwise-rotating vortex tube comes close to the sphere boundary layer, it augments the magnitude of the edge velocity in the lower boundary layer and reduces the edge velocity in the upper boundary layer. The result is a higher strength vorticity in the lower boundary layer than in the upper boundary layer (see the vorticity contours in figures 5(d)-5(f)). (The magnitude of the highest vorticity in the lower boundary layer is 15% higher than that in the upper boundary layer at $t = 9$.) Consequently, the vorticity in the lower boundary layer induces a velocity in the downward direction at the location of the vortex tube with higher magnitude than that induced by the vorticity in the upper boundary layer. This downward induced velocity advects the vortex tube below the sphere.

For $13 < t \leq 19$, the vortex tube is downstream of the sphere as shown in figures 5(i)

and 5(j) and produces downwash on the sphere. Therefore, the negative lift force acts on the sphere due to the shift of the front stagnation point above the plane $x = 0$ in a similar manner as for $10 < t \leq 13$, but the negative lift force is reduced as the vortex tube moves further downstream. The shift of the front stagnation point above the plane $x = 0$ causes the fluid particles to move faster in the top and upper right regions than in the bottom and lower right regions of the sphere, and this causes the lower pressure and higher shear stress in the top and upper right regions compared to those in the bottom and lower right regions as shown in the figures 6(h) and 6(i). However, because the vortex tube is still intersecting the near wake and thus produces strong downwash in the near wake, the fluid particles turning around the top of the sphere are pushed into the lower region of the near wake. This allows no room for the lower eddy to grow. On the other hand, the upper eddy grows as the vortex tube moves downstream because the fluid particles turning around the top of the sphere experience less force pushing them into the lower region of the near wake.

For $t \geq 20$, the vortex tube is far downstream of the sphere as shown in figures 5(k) and 5(l) and produces weak downwash on the sphere, and thus the lift force on the sphere is almost zero as will be shown in section of 3.2.2. The weak downwash causes the front stagnation point on the sphere to be shifted slightly above the plane $x = 0$, and thus the fluid particles move slightly faster in the top and upper right regions than in the bottom and lower right regions of the sphere, and this causes the higher shear stress and lower pressure in the top and upper right regions as shown in the figure 6(j). Now, the downward force due to the vortex tube is very weak in the near wake because the vortex tube is far downstream. Therefore, the lower eddy grows, and due to the lower pressure in the upper region of the sphere, the fluid particles turning around the lower eddy are pushed up into the upper near wake as shown in figures 5(k) and 5(l).

We now examine a three-dimensional view of the vortex tube by considering the y-component of vorticity vector. Figures 8(a) and 8(b) show two views of a three-

dimensional contour surface of $\omega_y = 0.2$ at $t = 20$ for the flow depicted in figure 5. Figure 8(a) shows a side view looking normal to the principal plane, whereas figure 8(b) shows a view looking down with an acute angle toward the y-z plane. The ellipse in figure 8(b) is the boundary of the spherical computational domain viewed at an angle. It appears as a circle when viewed normal to the principal plane. The sphere is at the center of the domain in figures 8(a) and 8(b). Figure 8(b) shows that the portion of the vortex tube in the principal plane is retarded, due to its interaction with the sphere, compared with the rest of the vortex tube (in the y-z plane with an its axis parallel to the y-axis) outside the principal plane. By measuring the radial extent of the contour surface (of $\omega_y = 0.2$ at $t = 20$) in figures 8(a) and 8(b), we find that the maximum radius of the contour surface outside the principal plane is 1.58 which is very close to the value of 1.6 taken from figure 3(b).

3.2.2 Lift, moment, and drag coefficients and effect of tube circulation

Figure 9 shows the lift coefficients of the sphere as a function of time for $Re = 100$, $d_{off} = 0$, and $\sigma = 1$. The lift coefficients are computed for four different maximum fluctuation velocities ($v_{max} = \Gamma_v / (2\pi\sigma)$) due to the vortex tube, with magnitudes equal to 0.1, 0.2, 0.3, and 0.4 (normalized by free stream velocity). Due to the sudden placement of the sphere into the stream, it initially takes a small time ($0 < t < 0.6$) for the initial flow perturbations to vanish.

As discussed earlier, when the vortex tube approaches the sphere ($0 \leq t \leq 9.4$), it produces upwash resulting in a positive lift force on the sphere. The maximum positive lift coefficient $C_{L,max1}$ occurs at $t = 7.2$. On the other hand, when the vortex tube passes the sphere, it produces downwash and high fluid velocity near the bottom of the sphere resulting in a negative lift force. The magnitude of the negative lift is greater than the positive lift (figure 9). The maximum negative lift coefficient $C_{L,max2}$ occurs at $t = 11.8$

about when the center of the vortex tube passes the plane $z = 1$. The lift coefficient is *linearly* proportional to the maximum fluctuation velocity (or the circulation of the vortex tube for constant vortex core radius) until the vortex tube contacts the sphere boundary layer ($t \leq 9.4$). The maximum positive lift coefficient $C_{L,max1}$ is expressed by

$$C_{L,max1} = c v_{max} , \quad (18)$$

where the proportionality constant $c = 0.8$. For $t > 9.4$, the relation between the lift coefficient and v_{max} deviates slightly from linearity, but the maximum negative lift coefficient $C_{L,max2}$ is linearly proportional to v_{max} with $c = -1.66$. After the lift coefficient reaches its maximum negative value, it decays quickly towards zero because the vortex tube vorticity is diffused in the sphere wake. The time averaged lift coefficient (averaged over a time span between $t = 0.6$ and the maximum time 24) for all values of v_{max} is nearly zero ($O(10^{-2})$). As mentioned earlier, the behavior of $C_L(t)$ during the period $0 < t < 0.6$ is influenced by the initial flow perturbation, and thus its value during this initial period is excluded from the averaging process. The root mean square $C_{L,rms}$ of the lift coefficient as a function of time is also linearly proportional to v_{max} with $c = 0.65$ as will be shown in table 4 in section 3.4.

Figure 10 shows the temporal development of the moment coefficients for the sphere under the same conditions of figure 9. The moment coefficients are obtained for four different values of $v_{max} = 0.1, 0.2, 0.3, \text{ and } 0.4$.

As the vortex tube approaches the sphere, the downward shift of the front stagnation point (due to the upwash) causes higher shear stress in the upper left region compared to the lower left region generating a negative (clockwise) torque. At the same time, the downward shift causes higher shear stress in the bottom and lower right regions compared to the top and upper right regions as explained in section 3.2.1 generating a positive torque. The two torques compete with each other and result in a net weak torque in the interval $0 < t \leq 9$.

As the vortex tube passes the sphere ($9 < t \leq 14$), the upward shift of the front stagnation point (due to the downwash) causes higher shear stress in the lower left region compared to the upper left region generating a positive torque. At the same time, the upward shift causes higher shear stress in the top and upper right regions compared to the bottom and lower right regions as explained in section of 3.2.1 generating a negative torque. However, the effect of this negative torque is diminished by the shear flow induced by the vortex across the sphere which produces high shear stress at the bottom of the sphere. As a consequence, a net high positive torque acts on the sphere. The maximum positive moment coefficient $C_{M,max}$ occurs at $t = 11.4$. $C_{M,max}$ is linearly proportional to v_{max} with a proportionality constant $c = 0.14$.

When the vortex tube is relatively far downstream from the sphere ($t > 15$), the positive torque due to the shear stress in the lower left region competes with the negative torque due to the shear stress in the top and upper right regions. This results in a net weak negative torque which becomes smaller as the vortex tube moves farther downstream. We note that the torque depends only the distribution of the shear stresses ($\tau_{r\theta}$ and $\tau_{r\phi}$) and is relatively small compared to the lift force.

The time averaged moment coefficient (averaged over a time span between $t = 0.6$ and 24) for all values of v_{max} is nearly zero ($O(10^{-3})$), and the root mean square $C_{M,rms}$ of the moment coefficient is approximately linearly proportional to v_{max} with $c = 0.05$.

Figure 11 shows the drag coefficients of the sphere as a function of time for the same conditions of figure 9. The drag coefficients are computed for four different values of $v_{max} = 0.1, 0.2, 0.3$, and 0.4 .

As discussed earlier, the sudden placement of the sphere in the flow results in initially large values of shear stress and pressure on the sphere, and hence a large drag as shown in figure 11. Figure 5(e) shows that at about $t = 10$ the center of the vortex tube is located near the front stagnation point which is slightly below the point $(x, y, z) = (0, 0, -1)$. Due to the low pressure at the center of the vortex tube, the pressure coefficient at the front

stagnation point ($C_{po} = 0.818$) is lower than that of the axisymmetric flow past a sphere without the vortex tube ($C_{po,axi} = 1.107$) as shown in figure 6(d). Also, the maximum shear stresses in the upper and lower regions are lower than that of the axisymmetric flow without the vortex tube. This causes the drag on the sphere to be lower than that of the axisymmetric flow without the vortex tube. As the vortex tube moves around the bottom of the sphere, the front stagnation point is shifted above the plane $x = 0$ due to the downwash. Consequently, high pressure and high shear stress act in the upper and lower left regions, respectively, as explained earlier in section 3.2.1. This increases the drag during the period $10 < t \leq 13.4$. For $t > 13.4$, the drag approaches that of the axisymmetric flow as the vortex tube moves further downstream.

The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of v_{max} is nearly zero ($O(10^{-4})$). The unsteady drag coefficient of the axisymmetric flow past a sphere was computed for a sphere suddenly placed in the uniform stream without the vortex tube.

3.3 Effects of the offset distance

We examine the effects of the offset distance on the flow field by varying d_{off} while using the same flow conditions of the preceding section 3.2.

3.3.1 Offset distance $1 \leq d_{off} \leq 4$

The temporal behaviors of the lift and moment coefficients of the sphere for $d_{off} = 1$ are similar to those in the case of $d_{off} = 0$. The main features distinguishing the case of $d_{off} = 1$ from that of $d_{off} = 0$ is that in the former, the vortex tube splits into two parts when the vortex tube passes the sphere. The attraction of the vortex tube to the positive vorticity in the boundary layer at the bottom of the sphere causes some portion of the vortex tube moves around the bottom of the sphere, whereas the other portion of

it moves on the top of the sphere, as shown in figures 12(a)-(h). Figures 12(a)-(h) display the contour lines of y-component vorticity in the principal plane at $t = 9, 10, 11, 12, 13, 15, 18,$ and 21 for $Re = 100$, $d_{off} = 1$, $\sigma = 1$, and $v_{max} = 0.4$. The contour values of the vorticity are $\pm 0.4, \pm 0.5, \pm 0.8, \pm 1.4, \pm 2$. Due to its longer interaction with the sphere for $d_{off} = 1$ than for $d_{off} = 0$, the magnitudes of the lift and moment coefficients of the sphere are close to those in the case of $d_{off} = 0$ despite its positive offset distance initially. Equation (18) is approximately valid for $C_{L,max1}, C_{L,max2}, C_{L,rms}, C_{M,max}$, and $C_{M,rms}$ with the same proportionality constants as in the case of $d_{off} = 0$.

Figure 13(a) shows the drag coefficients of the sphere as a function of time for $Re = 100$, $d_{off} = 1$, and $\sigma = 1$. The drag coefficients are obtained with two different maximum fluctuation velocities due to the vortex tube, $v_{max} = 0.1$ and 0.2 . The temporal behavior of the drag coefficients is *different* from that of the case of $d_{off} = 0$. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of v_{max} is not nearly zero but increased linearly as v_{max} increases. The time averaged drag coefficient $C_{D,ave}$ is expressed by

$$C_{D,ave} = C_{D,axi} + \beta v_{max}, \quad (19)$$

where the constant $\beta = 0.2$, and $C_{D,axi}$ is the time averaged value of the drag coefficient in the case of axisymmetric flow ($v_{max} = 0$). The drag coefficients reach their maximum at about $t = 10$. The maximum drag coefficient $C_{D,max}$ is expressed also by equation (19) but with $\beta = 0.62$, and $C_{D,axi}$ here is the local value of the axisymmetric drag coefficient at the time of $C_{D,max}$. At about $t = 10$, the center of the vortex tube is located above the front stagnation point. Thus, the induced velocity due to the vortex tube adds its magnitude to the base flow along the stagnation streamline, and so the dynamic pressure ahead of the front stagnation point becomes higher than that of the axisymmetric flow past a sphere. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be higher than those of the axisymmetric flow past a

sphere. As a consequence, the drag is increased. When the offset distance is negative, the reverse phenomena would occur, and the drag would be decreased. This will be discussed later in section 3.3.2.

Figures 14(a) and 14(b) display the contour lines of y-component vorticity in the principal plane at $t = 9$ and 12 for $Re = 100$, $d_{off} = 2$, $\sigma = 1$, and $v_{max} = 0.4$. The contour values of vorticities are the same as those of previous sections. Figures 14(a) and 14(b) show that the vortex tube passes above the sphere. The behavior of the lift coefficients with time for $d_{off} = 2, 3$, and 4 is similar to that of the case of $d_{off} = 0$ and 1 . However, their magnitudes are smaller than those for $d_{off} = 0$ and 1 and decay with d_{off} exponentially as shown in figure 15, where the magnitude of the negative maximum lift coefficient and the maximum moment coefficient for $Re = 100$, $\sigma = 1$, and $v_{max} = 0.2$ are presented as a function of d_{off} . The positive maximum lift coefficient $C_{L,max1}$ for $d_{off} \geq 2$ is expressed by

$$C_{L,max1} = c_1 v_{max} \exp(c_2 |d_{off}|), \quad (20)$$

where $c_1 = 0.99$ and $c_2 = -0.3$. The negative maximum lift coefficient $C_{L,max2}$ is expressed by equation (20) with $c_1 = -2.64$ and $c_2 = -0.38$, and the rms lift coefficient $C_{L,rms}$ is expressed also by equation (20) with $c_1 = 0.88$ and $c_2 = -0.28$.

The behavior of the moment coefficients with time for $d_{off} = 2, 3$, and 4 is also similar to that of the case of $d_{off} = 0$ and 1 . However, their magnitudes are smaller than those for $d_{off} = 0$ and 1 and decay with a negative power of d_{off} as shown in figure 15. The maximum moment coefficient $C_{M,max}$ for $d_{off} \geq 2$ is expressed by

$$C_{M,max} = c_3 v_{max} |d_{off}|^m, \quad (21)$$

where $c_3 = 0.185$ and $m = -1.501$. The rms moment coefficient $C_{M,rms}$ is expressed by equation (21) with $c_3 = 0.056$ and $m = -1.185$.

Figures 13 (b), (c), and (d) show the drag coefficients of the sphere as a function of time for $d_{off} = 2, 3$, and 4 , respectively, with $Re = 100$ and $\sigma = 1$. The drag coefficients

are obtained with two different maximum fluctuation velocities due to the vortex tube, $v_{max} = 0.1$ and 0.2 . The drag coefficients reach their maximum at about $t = 10$. The maximum drag coefficient $C_{D,max}$ for $d_{off} = 2$ is higher than that for $d_{off} = 1$ because the magnitude of the induced velocity added to the base flow along the stagnation streamline for $d_{off} = 2$ is higher than that for $d_{off} = 1$. We note that the radius of the vortex core is greater than unity at $t = 10$ due to the diffusion (and the maximum induced velocity occurs at the edge of the vortex core) as shown in figure 3(a) for $Re = 100$, $\sigma = 1$, and $v_{max} = 0.4$. $C_{D,max}$ becomes smaller as d_{off} is greater than 2. $C_{D,max}$'s for $d_{off} = 2, 3$, and 4 are expressed by equation 19 with $\beta = 0.9, 0.8$, and 0.68 , respectively. $C_{D,ave}$'s for $d_{off} = 2, 3$, and 4 are expressed also by equation (19) with $\beta = 0.31, 0.33$, and 0.32 , respectively. We note that the magnitude of the deviation of the drag coefficient from that of the axisymmetric flow decays slowly with d_{off} , in contrast with fast decay of the lift and moment coefficients with d_{off} .

3.3.2 Offset distance $-1 \geq d_{off} \geq -4$

Note that the sign reversal of the initial tube vorticity with the offset distance kept positive is a mirror image of the case where the sign of the offset distance is changed and the sign of the initial vorticity is kept constant. Therefore, we consider only change in sign of the offset distance and keep the counter-clockwise rotation.

The behavior of the lift coefficients with time for $-1 \geq d_{off} \geq -4$ is similar to that of the case of $d_{off} = 0$. However, their magnitudes are smaller than that for $d_{off} = 0$ with same v_{max} and decay exponentially with d_{off} as shown in figure 15. The positive maximum lift coefficient, the negative maximum lift coefficient, and the rms lift coefficient for $d_{off} \leq -1$ are expressed by equation (20) with $c_1 = 0.942$ and $c_2 = -0.295$, $c_1 = -1.95$ and $c_2 = -0.35$, and $c_1 = 0.74$ and $c_2 = -0.27$, respectively.

The behavior of the moment coefficients with time for $-1 \geq d_{off} \geq -4$ is similar

to that of the case of $d_{off} = 0$. However, their magnitudes are smaller than that for $d_{off} = -1$ with same v_{max} and decays with a negative power of $|d_{off}|$ as shown in figure 15. The maximum moment coefficient and the rms moment coefficient are expressed by equation (21) with $c_3 = 0.09$ and $m = -1.264$, and $c_3 = 0.0318$ and $m = -1.047$, respectively.

Figure 16(a) shows the drag coefficients of the sphere as a function of time for $Re = 100$, $d_{off} = -1$, and $\sigma = 1$. The drag coefficients are obtained with two different maximum fluctuation velocities due to the vortex tube, $v_{max} = 0.1$ and 0.2 . The behavior of the drag coefficients with time is *different* from that of the case of $d_{off} = 0$. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere at each maximum fluctuation velocity is not near zero but is increased linearly in negative value as the maximum fluctuation velocity becomes higher. The minimum drag coefficients occur at about $t = 10$. The minimum drag coefficient and the time averaged drag coefficient are expressed by equation (19) with $\beta = -0.78$ and -0.2 , respectively.

At about $t = 10$, the center of the vortex tube is located below the front stagnation point. Thus, the induced velocity due to the vortex tube subtracts its magnitude from the base flow along the stagnation streamline, and so the dynamic pressure ahead of the front stagnation point becomes lower than that of the axisymmetric flow past a sphere. This causes the pressure at the front stagnation point and the shear stresses in the upper and lower left regions to be lower than those of the axisymmetric flow past a sphere. As a consequence, the drag is decreased. From this result, we deduce that if the sphere were *free to move* rather than fixed, it would experience *lower drag* than that of a sphere subjected to an axisymmetric flow unless the initial offset distance is large positive. The lower drag will be caused by the upward motion of the sphere due to the upwash when the vortex tube approaches it, and thus the center of the vortex tube will be located below the front stagnation point of the sphere. This will cause lower dynamic pressure ahead of the front stagnation point.

Figures 16(b)-(d) show the drag coefficients of the sphere as a function of time for $d_{off} = -2, -3$, and -4 with $Re = 100$ and $\sigma = 1$. The minimum drag coefficients occur at about $t = 10$. The minimum drag coefficient $C_{D,min}$ for $d_{off} = -2$ is lower than that for $d_{off} = -1$, because the magnitude of the induced velocity subtracted from the base flow along the stagnation streamline is higher for $d_{off} = -2$ compared to $d_{off} = -1$. The magnitude of $C_{D,min}$ becomes smaller as d_{off} is less than -2 . The minimum drag coefficients for $d_{off} = -2, -3$, and -4 are expressed by equation 19 with $\beta = -0.9, -0.8$, and -0.68 , respectively. The time averaged drag coefficients for $d_{off} = -2, -3$, and -4 are expressed also by equation 19 with $\beta = -0.28, -0.3$, and -0.29 , respectively. We note that the magnitude of the deviation of the drag coefficient from that of the axisymmetric flow decays slowly with d_{off} , in contrast with fast decay of the lift and moment coefficients with d_{off} .

3.4 Effects of the size of the vortex tube

We examine the effects of the size of the vortex tube on the flow field by performing computations similar to those in section 3.2 for $Re = 100$, $d_{off} = 0$, and five different sizes of the vortex tube, $0.25 \leq \sigma \leq 4$ in addition to the base case $\sigma = 1$. Each simulation is performed with two different values of $v_{max} = 0.1$ and 0.3 .

Table 4 shows the maximum positive lift coefficient, the maximum negative lift coefficient, the rms lift coefficient, the maximum moment coefficient, and the rms moment coefficient as a function of v_{max} for six different initial radii of the vortex tube, $\sigma = 4, 3, 2, 1, 0.5$, and 0.25 . All the coefficients are linearly proportional to v_{max} at each σ . When $\sigma \geq 2$, $C_{L,max1}$ and $C_{L,rms}$ become *independent* of σ , but the magnitudes of $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ for $\sigma = 4$ are smaller than those for $\sigma = 2$ and 3 . When σ approaches zero, all the coefficients tend to be proportional to (σv_{max}) which is the *circulation* of the vortex tube divided by 2π ($\Gamma_v = 2\pi\sigma v_{max}$). For example, $C_{L,rms}$ is expressed by

$$\begin{aligned}
C_{L,rms} &= c_1 v_{max}, \quad 2 \leq \sigma \leq 4 \\
&= c_2 v_{max} \sigma^n, \quad 0.25 \leq \sigma < 2, \quad 0.75 \geq n \geq 0.5,
\end{aligned} \tag{22}$$

where the constant $c_1 = 1$ and $c_2 = 0.65$, and n depends on σ and should approach unity as σ reaches zero. For $C_{L,max1}$, $c_1 = 1.1$ and $c_2 = 0.8$. $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ for $\sigma \leq 3$ are also expressed by equation (22) with $c_1 = -2$ and $c_2 = -1.66$, $c_1 = 0.14$ and $c_2 = 0.14$, and $c_1 = 0.055$ and $c_2 = 0.05$, respectively. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of σ is nearly zero ($O(10^{-4})$).

We note that $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ for $\sigma = 4$ are, respectively, smaller than those for $\sigma = 2$ and 3, and the reason is explained as follows. When the initial size of the vortex core is considerably larger than the sphere size ($\sigma \geq 4$), the effect of the shear flow (induced by the passage of the vortex tube) across the sphere diminishes. We explained in section 3.2.1 that the magnitude of $C_{L,max2}$ depends on the combined effects of the downwash and the shear flow across the sphere due to the vortex tube. As a result, the magnitude of $C_{L,max2}$ decreases and approaches $C_{L,max1}$ as $\sigma \gg 1$. $C_{M,max}$ and $C_{M,rms}$ also decrease for the same reason. In addition, when the initial size of the vortex core is larger than the sphere size, the effect of the wake behind the sphere on the vortex tube diminishes. As a consequence, the magnitudes of the lift and moment coefficients decay slowly towards zero after they peak near the time of passage of the vortex tube center by the plane $z = 1$.

Summarizing the effects of the vortex size, the maximum positive lift coefficient and the rms lift coefficient depend only on the circulation Γ_v at small values of σ while they depend only on v_{max} (and not σ) at large values of σ . For mid-range values of σ , they depend on both σ and v_{max} (or equivalently both σ and Γ_v).

In section 3.3, we investigated the effect of the offset distance on the flow field for

$Re = 100$ and $\sigma = 1$. We now examine the effect of initial offset distance of the vortex tube on the lift and moment coefficients of the sphere as a function of the size of the tube at $Re = 100$. The values of $C_{L,max1}$, $C_{L,max2}$, and $C_{L,rms}$ of the sphere for initial offset distance of the vortex tube in the range of $-0.5\sqrt{\sigma} \leq d_{off} \leq \sqrt{\sigma}$ are within 5% difference from their values for $d_{off} = 0$. On the other hand, $C_{M,max}$ and $C_{M,rms}$ of the sphere for initial offset distance of the vortex tube in the same range vary by 13% from their values for $d_{off} = 0$.

3.5 Effects of Reynolds number

Computations similar to those in section 3.2 were performed for four different Reynolds numbers in the range of $20 \leq Re \leq 80$, $d_{off} = 0$, and $\sigma = 1$ with $v_{max} = 0.2$ in addition to the base case $Re = 100$. We also performed the same calculation with two different values of $v_{max} = 0.1$ and 0.3 and found that $C_{L,max1}$, $C_{L,max2}$, $C_{L,rms}$, $C_{M,max}$, and $C_{M,rms}$ are linearly proportional to v_{max} for each Reynolds number. Figures (17)-(20) show results for $v_{max} = 0.2$.

Figure 17 shows the total maximum positive lift coefficient and the coefficients due to pressure and viscous contributions as a function of Reynolds number for $\sigma = 1$ and $d_{off} = 0$ with $v_{max} = 0.2$. The coefficient due to pressure contribution is a little higher than that due to viscous contribution at Reynolds number 100. Both coefficients due to pressure and viscous contributions increase as Reynolds number decrease, but the viscous coefficient becomes greater. The total maximum positive lift coefficient increases with a negative power of Reynolds number as Reynolds number decreases as will be shown in figure 19 on a log-log scale and is expressed by

$$C_{L,max1} = A v_{max} Re^P, \quad (23)$$

where the constant $A = 3.5$ and $P = -0.32$.

Figure 18 shows the total maximum negative lift coefficient and the coefficients due

to pressure and viscous contributions as a function of Reynolds number for $\sigma = 1$ and $d_{off} = 0$ with $v_{max} = 0.2$. The magnitude of the coefficient due to pressure contribution is 2.38 times higher than that due to viscous contribution at Reynolds number 100. As mentioned in section 3.2, the sphere experiences the maximum negative lift coefficient whose magnitude is greater than the maximum positive lift coefficient when the vortex tube passes the sphere, because the vortex tube produces high fluid velocity gradient across the sphere as well as downwash on the sphere. (We note that the shear flow effect induced by the vortex tube would diminish when the size of the vortex tube becomes large.) Thus, the pressure contribution is much higher than the viscous contribution to the total maximum negative lift coefficient. The magnitude of the coefficient due to viscous contributions increases as Reynolds number decreases, on the other hand, that due to pressure contribution decreases as Reynolds number decreases. As a consequence, the magnitude of the total maximum negative lift coefficient is not sensitive to the change of the Reynolds number and slowly increases as Reynolds number decreases.

Figure 19 shows the rms lift coefficient and also the maximum positive lift coefficient as a function of Reynolds number on a log-log scale for $\sigma = 1$ and $d_{off} = 0$ with $v_{max} = 0.2$. The rms lift coefficient increases with a negative power of Reynolds number as Reynolds number decreases and is expressed by equation (23) with $A = 2.3$ and $P = -0.275$ for $\sigma = 1$. The effect of Reynolds number ($20 \leq Re \leq 80$) on the lift coefficient was investigated for the vortex size larger than $\sigma = 1$ ($2 \leq \sigma \leq 4$). The maximum positive lift coefficient and the rms lift coefficient are linearly proportional only to v_{max} and independent of σ when $\sigma \geq 2$ at fixed Reynolds number as in section 3.4 for $Re = 100$. The rms lift coefficient is expressed by equation (23) with $A = 8.1$ and $P = -0.45$ and written again here for later use.

$$C_{L,rms} = 8.1 v_{max} Re^{-0.45}, \quad 2 \leq \sigma \leq 4. \quad (24)$$

For the maximum positive lift coefficient, $A = 8.9$ and $P = -0.45$.

Figure 20 shows the maximum moment coefficient and the rms moment coefficient as a function of Reynolds number for $\sigma = 1$ and $d_{off} = 0$ with $v_{max} = 0.2$. The maximum moment coefficient and the rms moment coefficient are affected by only the viscous effect and increases with a negative power of Reynolds number as Reynolds number decreases. $C_{M,max}$ and $C_{M,rms}$ follow the form of equation (23) with $A = 1.95$ and $P = -0.56$ for the former, and $A = 1.05$ and $P = -0.665$ for the latter for $\sigma = 1$. The effect of Reynolds number ($20 \leq Re \leq 80$) on the moment coefficient was investigated for $2 \leq \sigma \leq 4$. The behavior of the moment coefficient at each Reynolds number is similar to that of the moment coefficient at $Re = 100$ which was explained in section 3.4. $C_{M,max}$ and $C_{M,rms}$ follow the form of equation (23) with $A = 5.5$ and $P = -0.83$ for the former, and $A = 3.1$ and $P = -0.88$ for the latter for $2 \leq \sigma \leq 3$.

The variation of the lift and moment coefficients for $-0.5\sqrt{\sigma} \leq d_{off} \leq \sqrt{\sigma}$ from those for $d_{off} = 0$, which is given in section 3.4 for $Re = 100$, decreases at fixed σ as Reynolds number decreases. For example, at $Re = 20$, the difference in the lift coefficient is 4% and that in the moment coefficient is 10%.

We investigate the lift, moment, and drag coefficients at Reynolds number 20 in order to find out Reynolds number effect in more detail. Figure 21 shows the lift coefficients of the sphere as a function of time for $Re = 20$, $d_{off} = 0$, and $\sigma = 1$. The lift coefficients are obtained with four different maximum fluctuation velocities due to the vortex tube, $v_{max} = 0.1, 0.2, 0.3$, and 0.4 . The maximum positive lift coefficient $C_{L,max1}$ occurs at $t = 6.6$, and the maximum negative lift coefficient $C_{L,max2}$ occurs at $t = 12.5$. The lift coefficient is linearly proportional to the maximum fluctuation velocity (or the circulation of the vortex tube) at each time over the whole time computed ($0 \leq t \leq 24$). This shows that the nonlinear effect at $Re = 20$ is much less than that at $Re = 100$. In contrast to figure 13 which shows the lift coefficient for $Re = 100$, figure 21 shows that the lift coefficient decays slowly to zero after it attains the maximum negative value. This indicates that for $Re = 100$ viscous diffusion in the wake is much stronger than that

in the upstream; on the other hand, viscous diffusion is rather uniformly important all around the sphere at $Re = 20$ compared to $Re = 100$. The behavior of the moment and drag coefficients with time is similar to that of the case of $Re = 100$. The time averaged lift and moment coefficients are nearly zero ($O(10^{-2})$ and $O(10^{-3})$, respectively), and the time averaged drag coefficient is close to that of the axisymmetric flow without the vortex tube (The difference between them is $O(10^{-2})$).

One of the reviewers noted that some of our results can be explained using dimensional analysis as follows. The lift force on the sphere can be expressed in a functional form as

$$F'_L(t') = f(U'_\infty, \rho', \mu', a'_o, \Gamma', \sigma', d'_{off}; t').$$

From dimensional analysis, it follows that

$$C_L(t) = \frac{F'_L}{\frac{1}{2}\rho'U'^2_\infty\pi a'^2_o} = f\left(\frac{\sigma'}{a'_o}, \frac{U'_\infty\sigma'}{\Gamma'}, \frac{\mu'}{\rho'U'_\infty a'_o}, \frac{d'_{off}}{a'_o}; \frac{t'U'_\infty}{a'_o}\right) = f(\phi_1, \phi_2, \phi_3, \phi_4; t),$$

where ϕ_i are the four parameters appearing on the left hand side of the last equal sign. If Reynolds number and the dimensionless offset distance are fixed, $C_L(t)$ will be a function of ϕ_1 , ϕ_2 , and t . Furthermore, when σ'/a'_o is small, we expect that a'_o is more important than σ' . Then C_L should be a function only of ϕ_1/ϕ_2 and t yielding:

$$C_L(t) = f\left(\frac{\Gamma'}{U'_\infty a'_o}; t\right) = f(\Gamma; t). \quad (25)$$

In the opposite limit, a'_o should be unimportant and

$$C_L(t) = f\left(\frac{\Gamma'}{U'_\infty \sigma'}; t\right) = f(v_{max}; t). \quad (26)$$

Equations (25) and (26) are consistent with our results in section 3.4.

Finally, we discuss the effect of the initial location of the vortex tube upstream from the center of the sphere. We have shown earlier that the maximum positive lift coefficient is expressed at given σ and Re as $C_{L,max1} = cv_{max}$ when the initial location of the vortex,

l , is 10 radii. For a different initial location, say l^* , the equation for the maximum positive lift coefficient should be modified as follows.

$$\begin{aligned}
C_{L,max1} &= c v_{max} \\
&= c \frac{\sigma^*}{\sigma} v_{max}^* \\
&= c \sqrt{1 - \frac{10.04(l^* - l)}{Re \sigma^2}} v_{max}^* \\
&= c^* v_{max}^*, \tag{27}
\end{aligned}$$

where we used $v_{max}/v_{max}^* = \sigma^*/\sigma$ and $\sigma^{*2} - \sigma^2 = 10.04(l^* - l)/Re$, which are obtained from the evolution of a point vortex in a viscous fluid (Batchelor (1967)). v_{max}^* denotes the maximum fluctuation velocity due to the vortex tube whose initial location is l^* radii upstream from the center of the sphere. Note that the proportionality constant c is now modified as c^* for the new initial location of the vortex l^* .

We examined the accuracy of equation (27) by performing computations for $l^* = 8$ and 12 with the same parameters as used in section 3.2.1 except the initial location of the vortex tube. The magnitude of $C_{L,max1}$ obtained from equation (27) differs by 0.2% from that of the full computations. The equation of the rms lift coefficient should be also modified as equation (27) for the new initial location of the vortex tube. In addition, the time span, t_{ave} , over which averaging the lift coefficient is performed should be modified according to $t_{ave}^* = t_{ave} + (l^* - l)$.

4 Conclusions

As a first step towards better understanding the physics of interaction between a particle and the turbulent carrier flow, we have investigated numerically the unsteady,

three-dimensional, incompressible, viscous flow interactions between a vortical (initially cylindrical) structure advected by a uniform free stream and a spherical particle suddenly placed and held fixed in space for a range of particle Reynolds numbers $20 \leq Re \leq 100$. The counter-clockwise rotating cylindrical vortex tube is initially located ten radii upstream from the center of the sphere.

A summary of our findings and their applications is provided as follow.

(i) One significant finding in our study is that the rms lift coefficient for a particle is *linearly* proportional to the upwash (or downwash induced by the vortex tube motion) on the particle normal to the direction of the free stream in our case (or the direction of the particle motion in the case of a free particle) and is independent of the size of the vortex tube when the size of the vortex is greater than that of the particle, $2 \leq \sigma \leq 4$. This result can be applied to turbulent flows containing small concentration of particles in order to obtain the rms lift force on a particle. A turbulent flow possesses a wide spectrum of eddy sizes. The large eddies contain most of the turbulent kinetic energy and produce high velocity fluctuations, and so they are responsible for the dispersion of particles. The particle size, at the extremes, may be comparable to either the integral length scale or to the Kolmogorov length scale. When the size of particle is comparable to the integral length scale, the rms lift coefficient of the particle is obtained by equation (24). Furthermore, our results tend to support the idea that equation (24) would be applicable to the case of an eddy much larger than the particle. Thus, when the size of particle is comparable to the Kolmogorov length scale, the rms lift coefficient of the particle can be calculated approximately by equation (24), where v_{max} is the maximum velocity fluctuation due to an eddy of size comparable to the integral length scale. The time during which the particle is influenced by the eddy is of the order of the eddy life time.

The deflection of the particle path will depend on the magnitude of the rms lift coefficient and the ratio, ρ_r , of the particle density to that of the carrier fluid ($C_L = \frac{8}{3}\rho_r A$, where A is the dimensionless acceleration of the particle due to the lift force). This result

provides a simple method to estimate the deflection of particle trajectory in the dilute particle-laden turbulent flow. Equation (24) and the nondimensionalized Newton's second law show that the deflection increases slowly as Reynolds number of the particle decreases.

(ii) The magnitude of the rms moment coefficient of the particle is one order of magnitude less than that of the rms lift coefficient when $Re \geq 20$. Furthermore, when the initial size of the vortex core is considerably larger than the sphere size ($\sigma \geq 4$), the effect of the shear flow (induced by the passage of the vortex tube) across the sphere diminishes and the torque on the particle decreases. Thus, the torque on the particle might be negligible in many applications.

(iii) When a vortex tube advected by a uniform free stream approaches a sphere, the sphere experiences *lower drag* than that of a sphere subjected to an axisymmetric flow if the sphere were *free to move* rather than fixed, unless the initial offset distance of the vortex tube is large positive, as explained in section 3.3.2. The lower drag is caused by the upward motion of the sphere due to the upwash of the approaching vortex tube, and thus the center of the vortex tube would be located below the front stagnation point of the sphere. This causes lower dynamic pressure ahead of the front stagnation point.

(iv) Some interesting unsteady phenomena in the near wake have been discovered. The shape of the near wake behind the spherical particle is controlled by the pressure difference between the top and bottom of the near wake as was indicated by Kim *et al.* (1993). The instantaneous flow patterns around a spherical particle in a turbulent flow would include some of those described in this paper. For example, our recent results (to be published), from a study of the interactions between a vortex pair advected by a uniform free stream and a sphere, show that the streamlines are similar to those described in the present paper.

We are also studying the heat and mass transfer for a droplet interacting with an array of vortices in high-temperature and high-pressure environment such as that in a gas turbine combustor.

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5 References

- Anderson, D. A., Tannehill, J. C. & Pletcher, R. H. 1984 *Computational Fluid Mechanics and Heat Transfer*. Hemisphere Publishing.
- Basset, A. B. 1888 *A Treatise on Hydrodynamics*, vol 2, p. 285. Dover.
- Batchelor, G. K. 1967 *An introduction to fluid dynamics*, p. 201. Cambridge University Press.
- Clift, R., Grace, J. R. & Weber, M. E. 1978 *Bubbles, Drops, and Particles*. Academic Press, New York.
- Houghton, G. 1963 The behavior of particles in a sinusoidal velocity field accelerating in a viscous fluid. *Proc. Roy. Soc.* **272**, 33-43.
- Ingebo, R. D. 1956 Drag coefficient and relative velocity in bubbly, droplet, or particulate flows. NASA TN 3762.
- Kim, I., Elghobashi S. & Sirignano, W. A. 1993 Three-dimensional flow over two spheres placed side by side. *J. Fluid Mech.* **246**, 465-488.
- Maxey, M. R. & Riley, J. J. 1983 Equation of motion for a small rigid sphere in a nonuniform flow. *Phys. Fluids* **26**, 883-889.
- McLaughlin, J. B. 1991 Inertial migration of a small sphere in linear shear flow. *J. Fluid Mech* **224**, 261-274.
- Odar, F. 1966 Verification of the proposed equation for calculation of the forces on a sphere accelerating in a viscous fluid. *J. Fluid Mech* **25**, 591-592.
- Patnaik, G. 1986 A numerical solution of droplet vaporization with convection. Ph.D. Dissertation, Carnegie-Mellon University.
- Rubinow, S. I. & Keller, J. B. 1961 The transverse force on a spinning sphere moving in

a viscous fluid. *J. Fluid Mech* **12**, 447-459.

Saffman, P. G. 1965 The lift on a small sphere in a slow shear flow. *J. Fluid Mech* **22**, 385-400.

Saffman, P. G. 1968 The lift on a small sphere in a slow shear flow - Corrigendum. *J. Fluid Mech* **31**, 624.

Schoneborn, P. R. 1975 The interaction between a single particle and an oscillating fluid. *Int. J. Multiphase Flow* **2**, 307-317.

Spalart, P. R. 1982 Numerical simulation of separated flows. Ph. D. Thesis, Stanford University.

Taneda, S. 1956 Experimental investigation of the wake behind a sphere at low Reynolds number. *J. Phys. Soc. Japan* **11** 1104-1108.

Vinokur, M 1983 On one-dimensional stretching functions for finite-difference calculations. *J. Comput. Phys.* **50**, 215-234.

$N_1 \times N_2 \times N_3$	C_{DP}	C_{DV}	C_D	C_D^*
$Re = 20$				
21 × 21 × 21	1.048	1.777	2.825	
31 × 31 × 31	1.039	1.740	2.779	
41 × 41 × 41	1.037	1.731	2.768	2.74
$Re = 100$				
21 × 21 × 21	0.560	0.590	1.150	
31 × 31 × 31	0.535	0.582	1.117	
41 × 41 × 41	0.526	0.581	1.107	1.09

Table 1. Drag coefficients as a function of grid density at $Re = 20$ and 100 , where * denotes the data from the correlation of Clift *et al.* (1978).

$N_1 \times N_2 \times N_3$	P_{of}	P_{or}	θ_s	θ_s^*
$Re = 20$				
21 × 21 × 21	0.668	-0.201	180	
31 × 31 × 31	0.707	-0.178	180	
41 × 41 × 41	0.721	-0.171	180	180
$Re = 100$				
21 × 21 × 21	0.554	-0.0938	124.09	
31 × 31 × 31	0.554	-0.0831	125.62	
41 × 41 × 41	0.554	-0.0792	126.16	126.5

Table 2. Pressure at the front and rear stagnation points and the separation angle measured from the front stagnation point as a function of grid density at $Re = 20$ and 100 , where * denotes the data from Taneda (1956) and the correlation of Clift *et al.* (1978).

$N_1 \times N_2 \times N_3$	$C_{L,max2,p}$	$C_{L,max2,v}$	$C_{L,max2}$
$Re = 20$			
$21 \times 21 \times 21$	-0.336	-0.359	-0.695
$31 \times 31 \times 31$	-0.358	-0.363	-0.722
$41 \times 41 \times 41$	-0.362	-0.363	-0.725
$Re = 100$			
$21 \times 21 \times 21$	-0.336	-0.159	-0.495
$31 \times 31 \times 31$	-0.433	-0.186	-0.619
$41 \times 41 \times 41$	-0.476	-0.192	-0.668
$51 \times 51 \times 51$	-0.494	-0.191	-0.685

Table 3. Maximum negative lift coefficients as a function of grid density at $Re = 20$ and 100 .

v_{max}	$C_{L,max1}$	$C_{L,max2}$	$C_{L,rms}$	$C_{M,max}$	$C_{M,rms}$
$\sigma = 4$					
0.3	0.337	-0.566	0.307	0.0361	0.0161
0.1	0.113	-0.190	0.102	0.0121	0.00546
$\sigma = 3$					
0.3	0.338	-0.604	0.299	0.0414	0.0169
0.1	0.114	-0.201	0.0997	0.0139	0.00577
$\sigma = 2$					
0.3	0.318	-0.611	0.271	0.0455	0.0169
0.1	0.107	-0.203	0.0907	0.0155	0.00593
$\sigma = 1$					
0.3	0.238	-0.500	0.195	0.0410	0.0138
0.1	0.0810	-0.165	0.0658	0.0145	0.00514
$\sigma = 0.5$					
0.3	0.147	-0.322	0.119	0.0290	0.00953
0.1	0.0499	-0.106	0.0402	0.0103	0.00355
$\sigma = 0.25$					
0.3	0.0815	-0.177	0.0660	0.0175	0.00587
0.1	0.0275	-0.0583	0.0221	0.00604	0.00210

Table 4. Maximum positive lift coefficient, maximum negative lift coefficient, root mean square of the lift coefficient, maximum moment coefficient, and root mean square of the moment coefficient as a function of v_{max} for six different radii of the vortex tube, $\sigma = 4, 3, 2, 1, 0.5,$ and 0.25 .

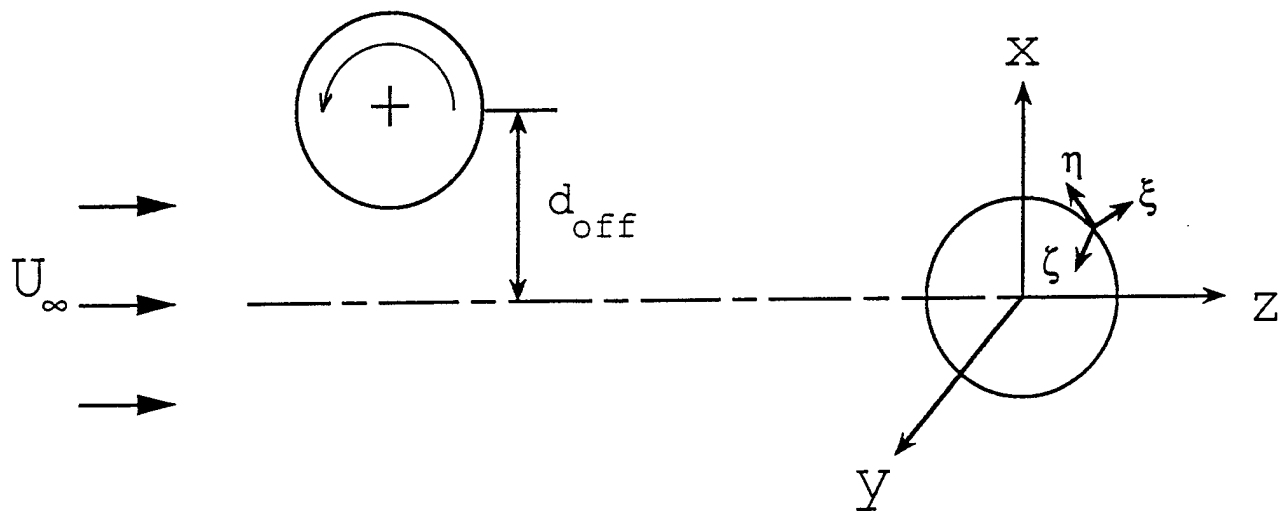
Figure Captions

- Figure 1. Flow geometry and coordinates
- Figure 2. Comparison of tangential velocities induced by a point vortex and a vortex tube for $\Gamma_v = 2.5$ and $\sigma = 1$.
- Figure 3. (a) Velocity and (b) vorticity fields due to a vortex tube as a function of radial distance and time for $Re = 100$, $\Gamma_v = 2.5$, and $\sigma = 1$.
- Figure 4. Pressure and shear stress distributions around the sphere in the x-z plane of symmetry as a function of grid resolution at $t = 12$ for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = 0.4$.
- Figure 5. Pseudo-streamlines (left column) and contour lines of y-component vorticity (right column) in the principal plane at (a) $t = 0$, (b) 1, (c) 6, (d) 9, (e) 10, (f) 11, (g) 12, (h) 13, (i) 15, (j) 18, (k) 21, and (l) 30 for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = 0.4$.
- Figure 6. Pressure and shear stress distributions around the sphere in the principal plane at (a) $t = 1$, (b) 6, (c) 9, (d) 10, (e) 11, (f) 12, (g) 13, (h) 15, (i) 18, and (j) 21 for $Re = 100$, $d_{off} = 0$, $\sigma = 1$, and $v_{max} = 0.4$.
—, upper; ---, lower.
- Figure 7. Tangential velocity profile, $u_\theta(r)$, at $\theta = 90^\circ$ on the top and the bottom of the sphere in the principal plane at $t = 12$.
- Figure 8. Two views of a three-dimensional contour surface of $\omega_y = 0.2$ at $t = 20$ for the flow depicted in figure 5; (a) a side view looking normal to the principal plane (b) a view looking down with an acute angle toward the y-z plane.
- Figure 9. Lift coefficients of the sphere as a function of time and v_{max} for $Re = 100$, $d_{off} = 0$, and $\sigma = 1$.

- Figure 10. Moment coefficients of the sphere under the same conditions of figure 9.
- Figure 11. Drag coefficients of the sphere under the same conditions of figure 9.
- Figure 12. Contour lines of y-component vorticity in the principal plane at (a) $t = 9$, (b) 10, (c) 11, (d) 12, (e) 13, (f) 15, (g) 18, and (h) 21 for $Re = 100$, $d_{off} = 1$, $\sigma = 1$, and $v_{max} = 0.4$.
- Figure 13. Drag coefficients of the sphere as a function of time and v_{max} for (a) $d_{off} = 1$, (b) 2, (c) 3, and (d) 4 with $Re = 100$ and $\sigma = 1$.
- Figure 14. Contour lines of y-component vorticity in the principal plane at (a) $t = 9$ and (b) 12 for $Re = 100$, $d_{off} = 2$, $\sigma = 1$, and $v_{max} = 0.4$.
- Figure 15. Magnitude of $C_{L,max2}$ and $C_{M,max}$ as a function of d_{off} for $Re = 100$, $\sigma = 1$, and $v_{max} = 0.2$.
- Figure 16. Drag coefficients of the sphere as a function of time and v_{max} for (a) $d_{off} = -1$, (b) -2 , (c) -3 , and (d) -4 with $Re = 100$ and $\sigma = 1$.
- Figure 17. Total maximum positive lift coefficient and the coefficients due to pressure and viscous contributions as a function of Reynolds number for $d_{off} = 0$ and $\sigma = 1$ with $v_{max} = 0.2$.
- Figure 18. Total maximum negative lift coefficient and the coefficients due to pressure and viscous contributions as a function of Reynolds number under the same conditions of figure 17.
- Figure 19. Root mean square of the lift coefficient and maximum positive lift coefficient as a function of Reynolds number under the same conditions of figure 17.
- Figure 20. Maximum moment coefficient and root mean square of the moment coefficient as a function of Reynolds number under the same conditions of figure 17.

Figure 21. Lift coefficients of the sphere as a function of time and v_{max}
for $Re = 20$, $d_{off} = 0$, and $\sigma = 1$.

Fig. 1



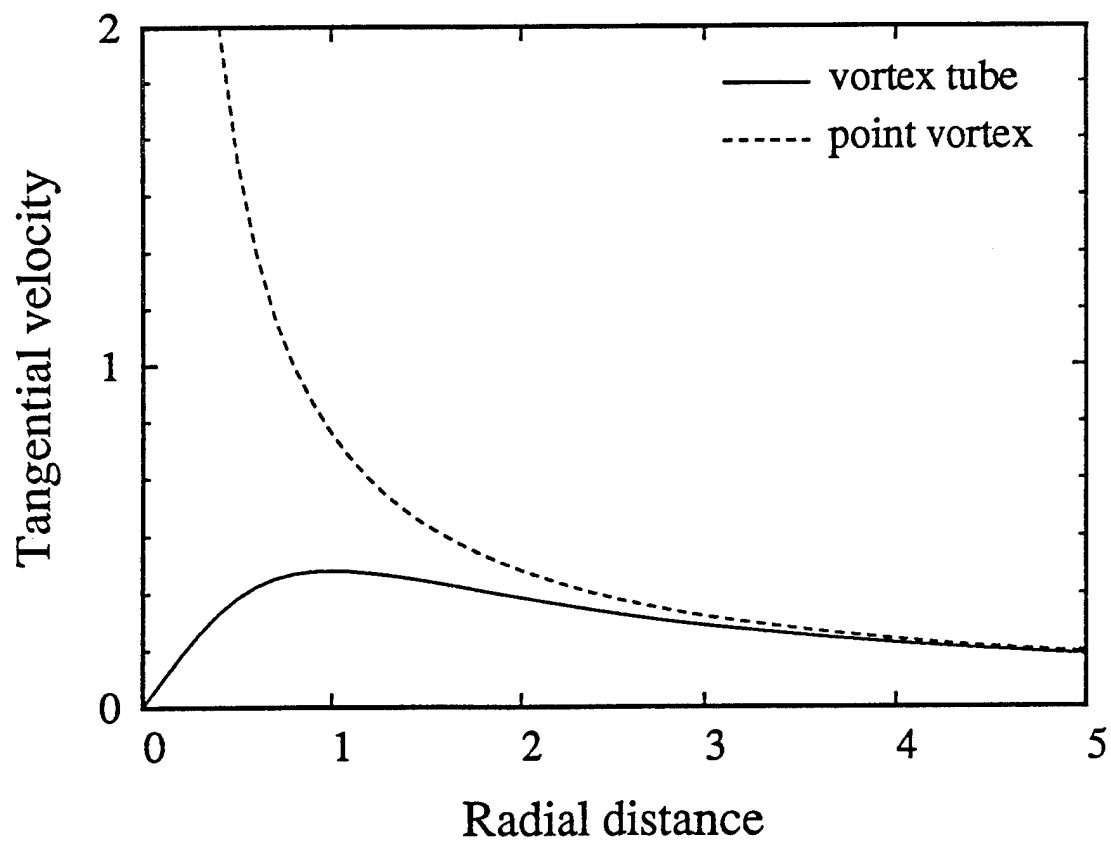


Fig. 3(a)

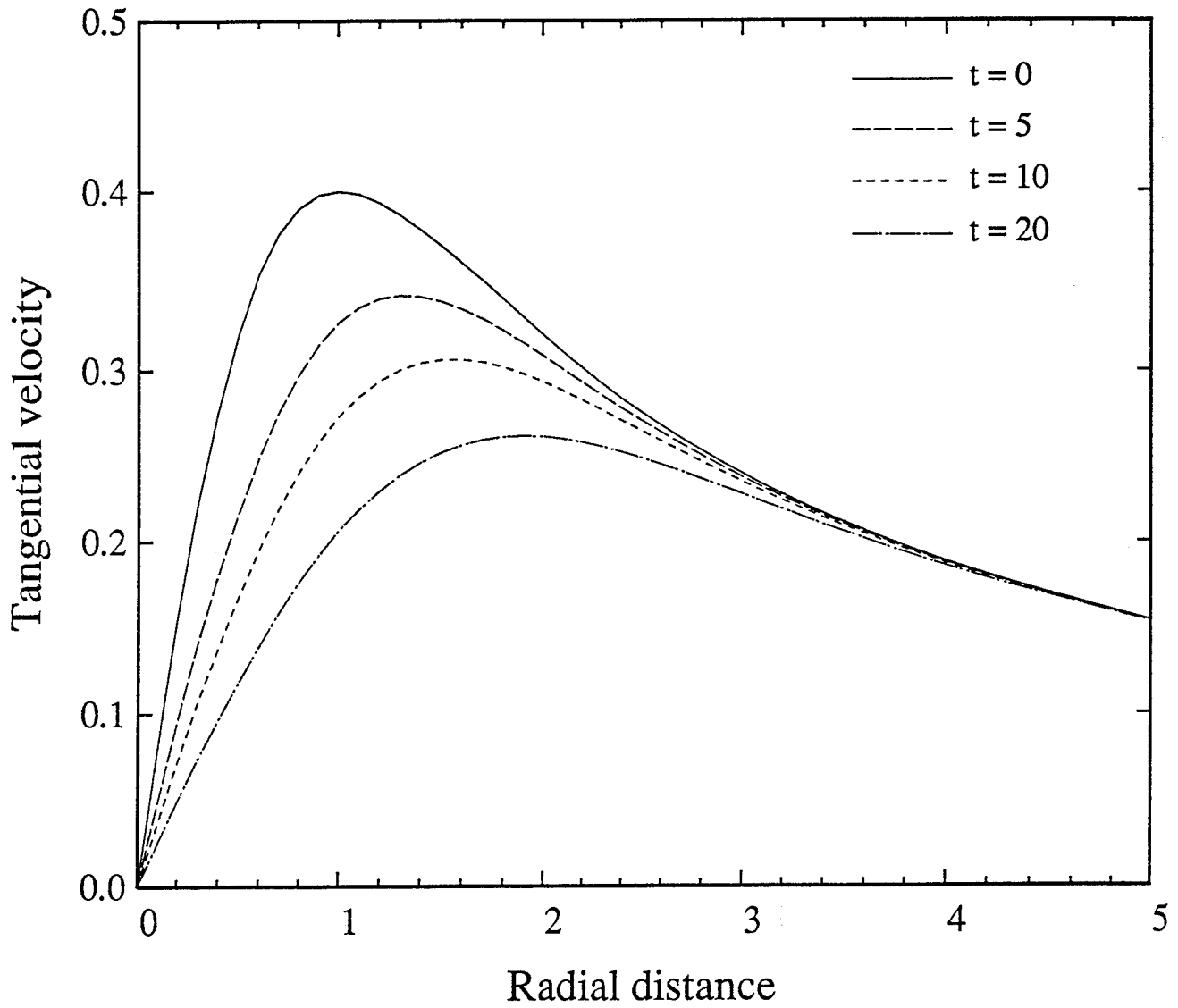


Fig. 3(b)

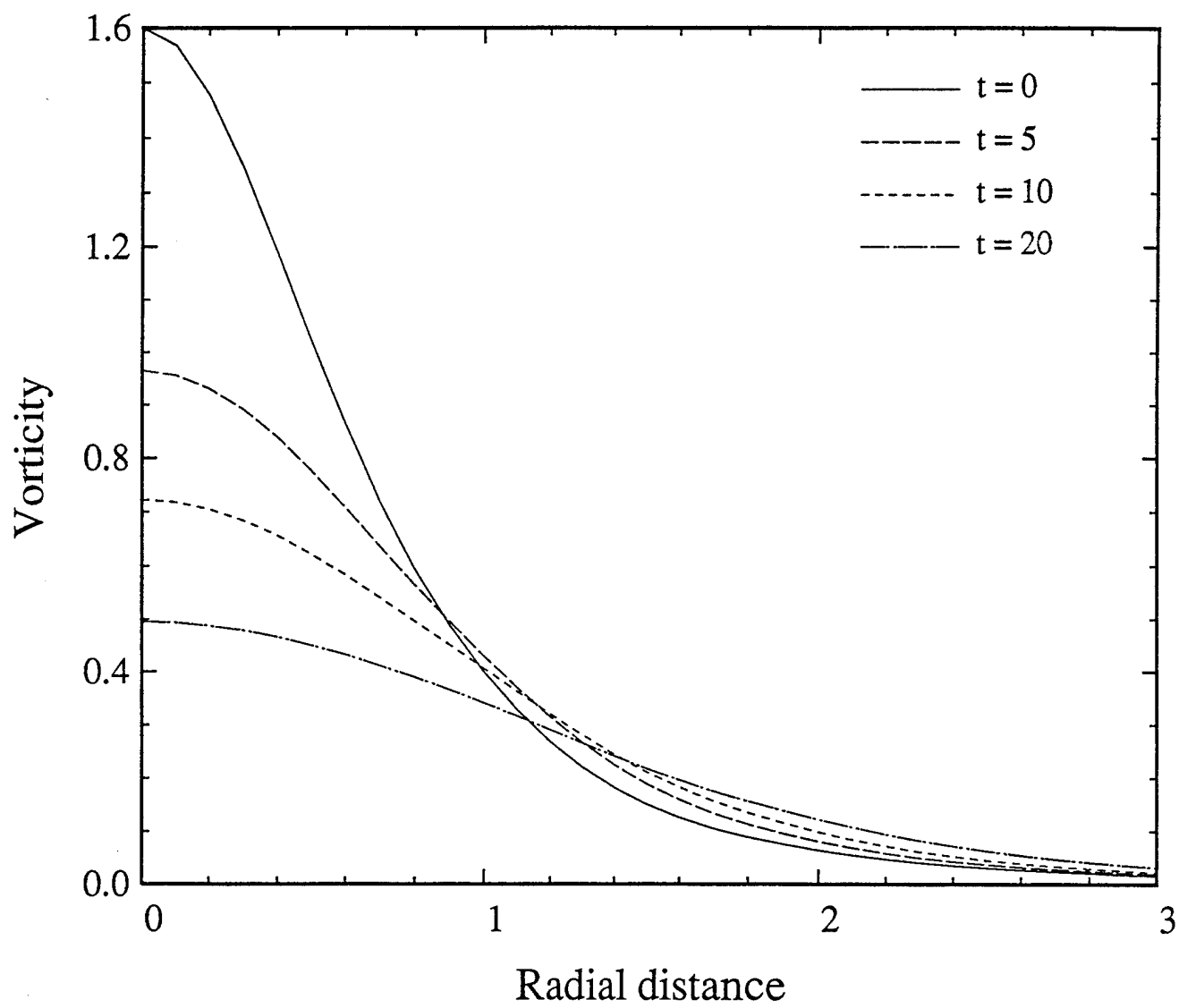
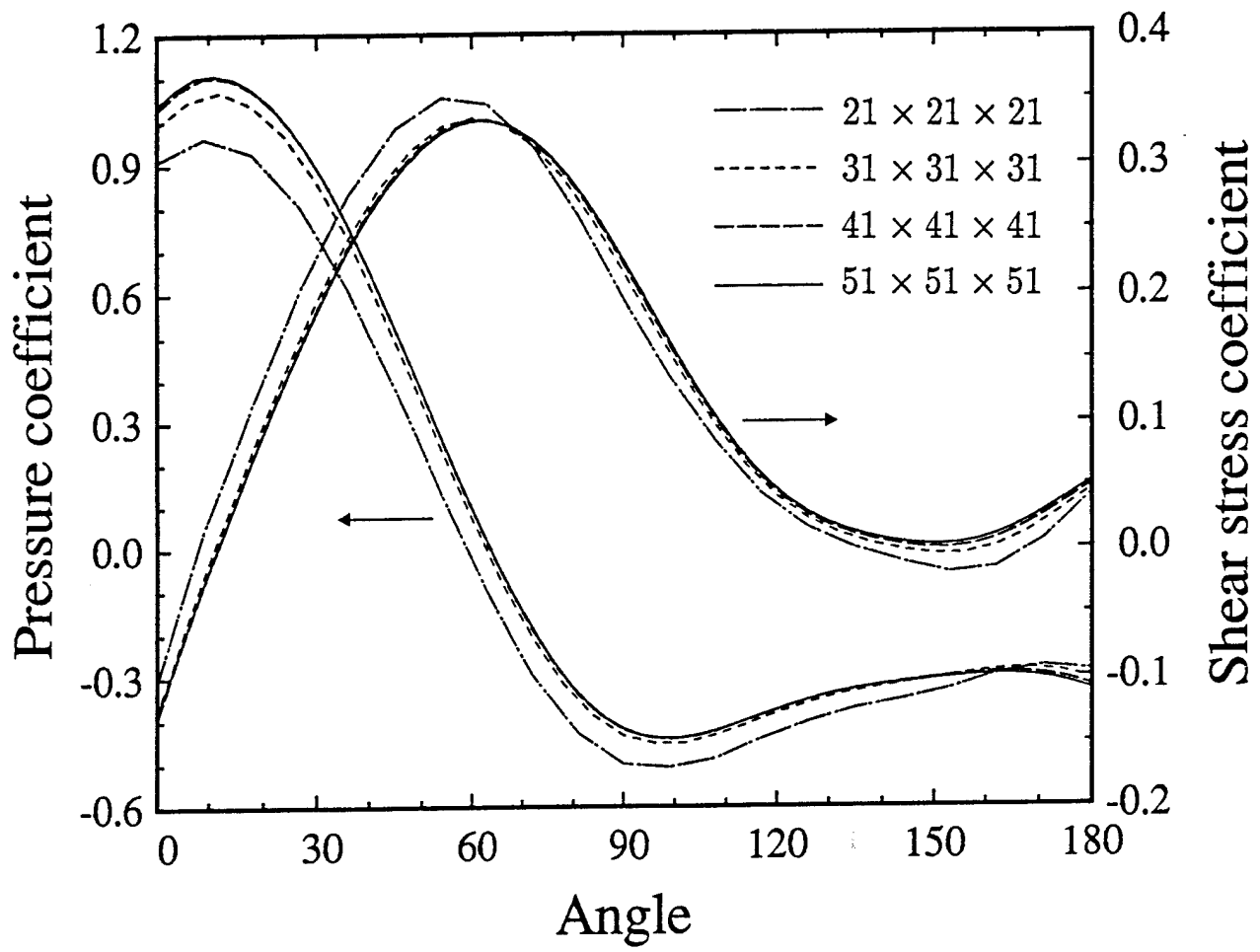
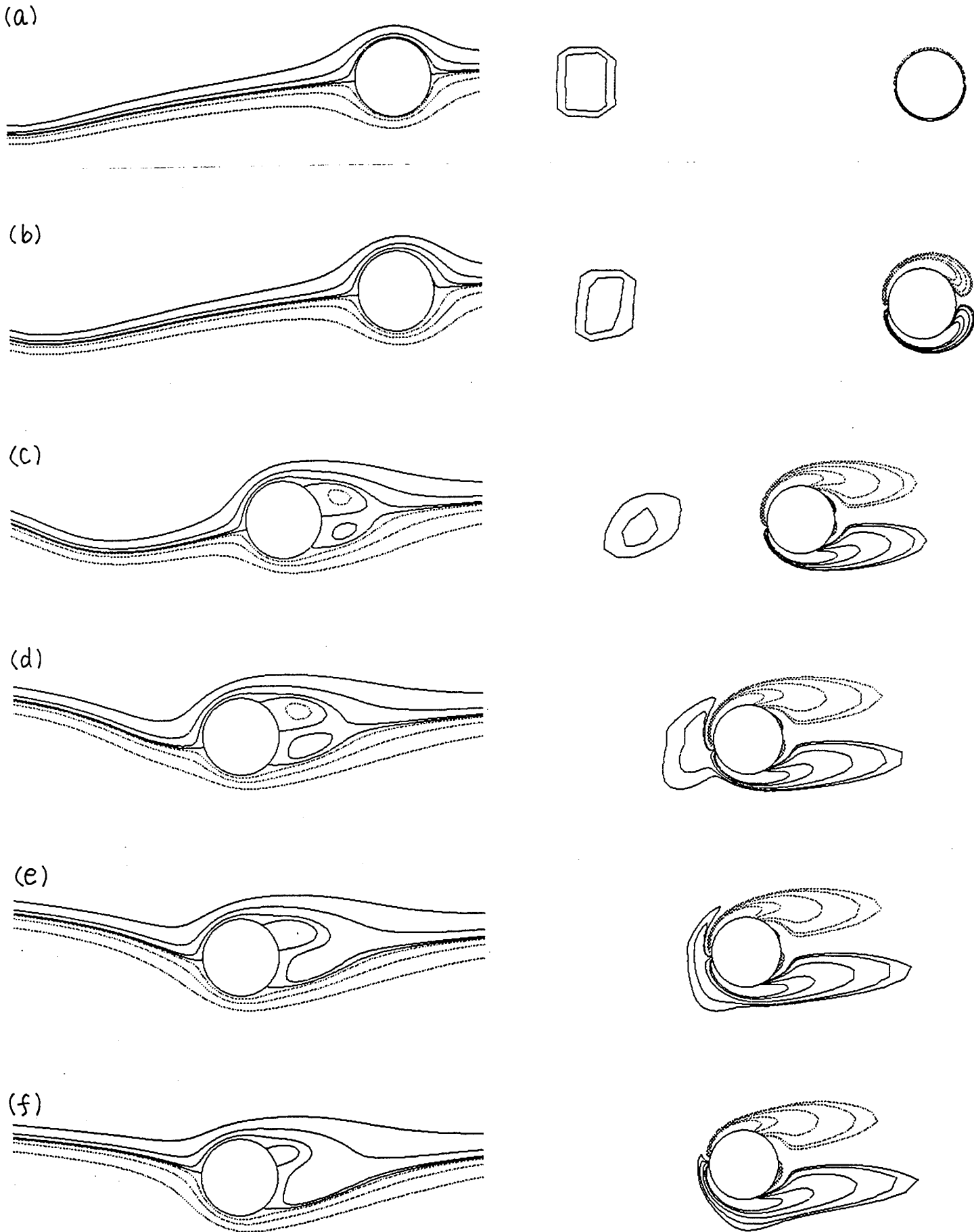
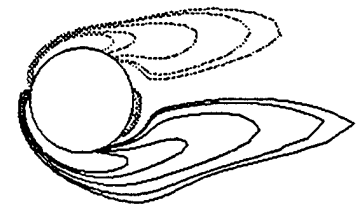
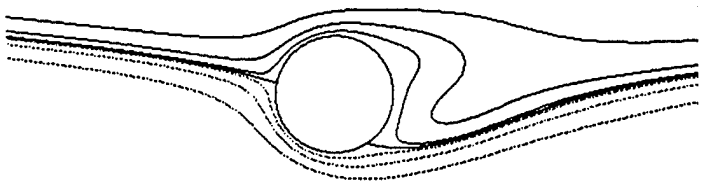


Fig. 4

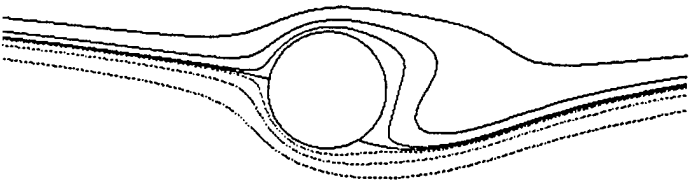




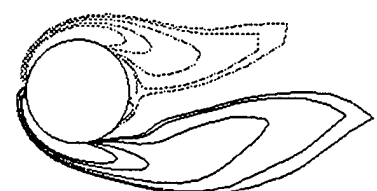
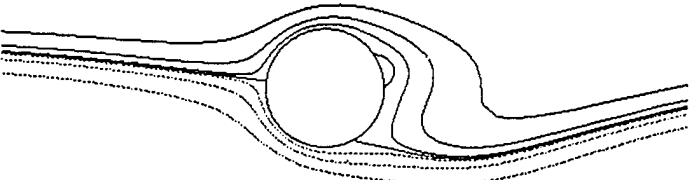
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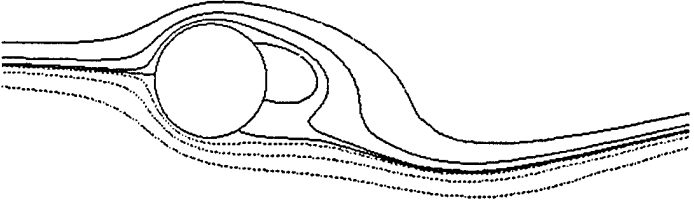
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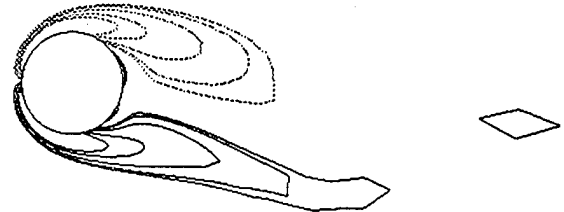
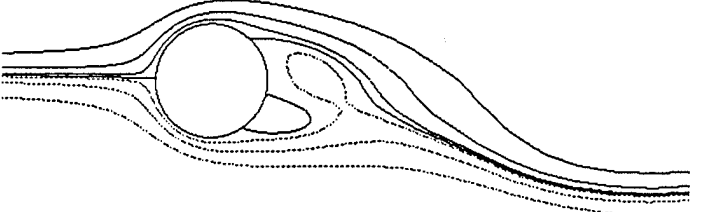
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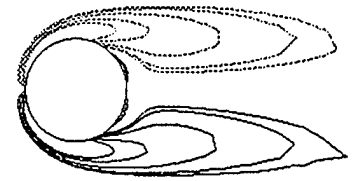
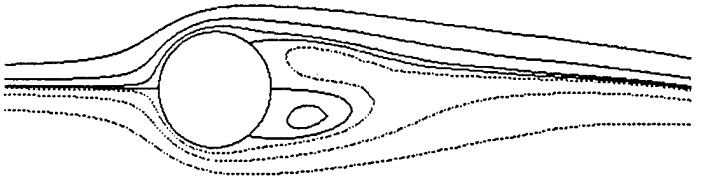
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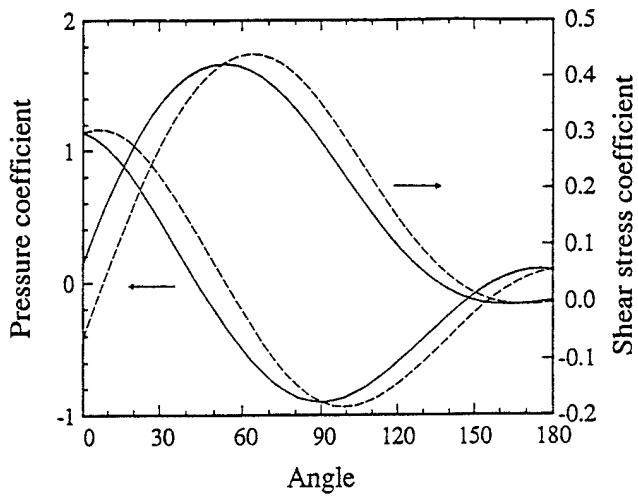
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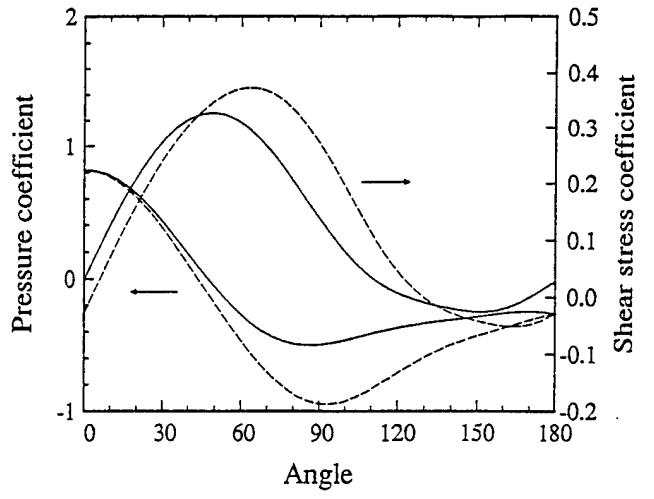
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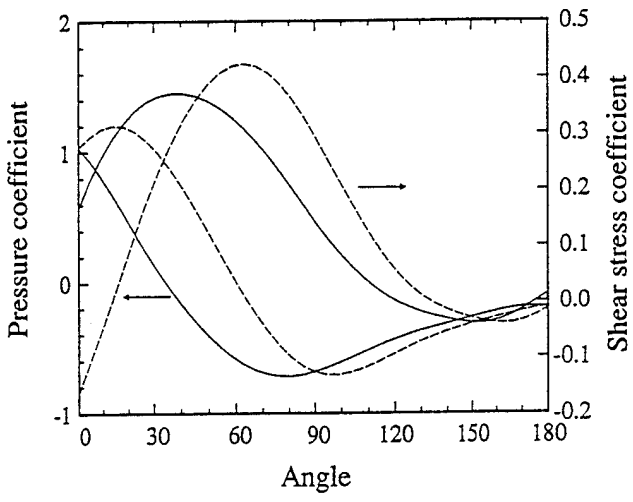
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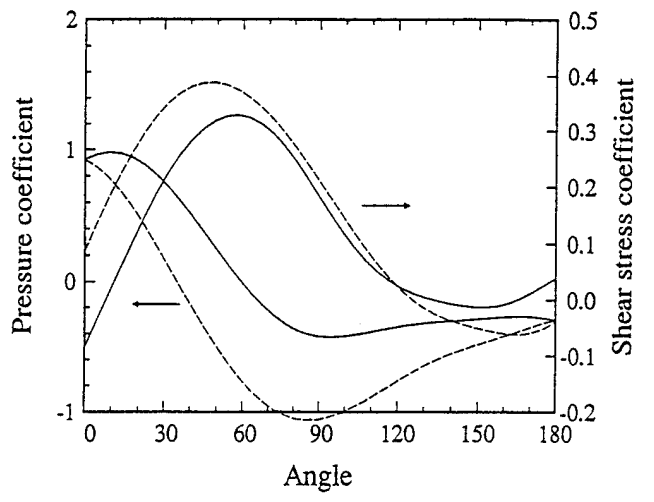
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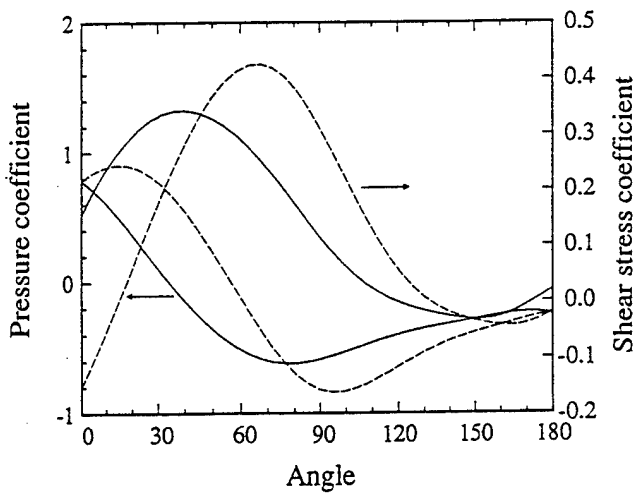
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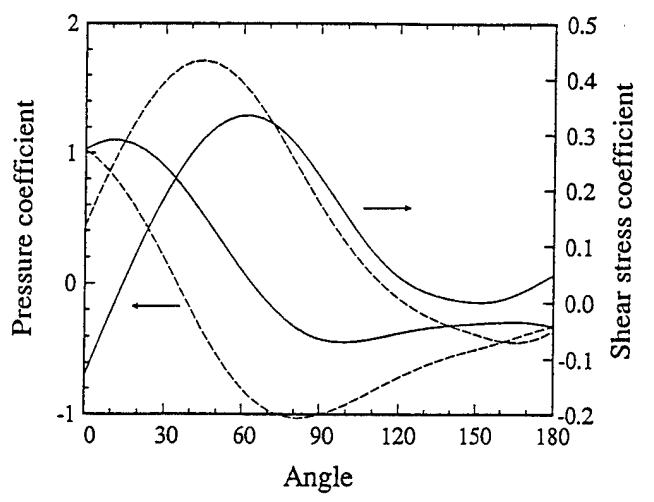
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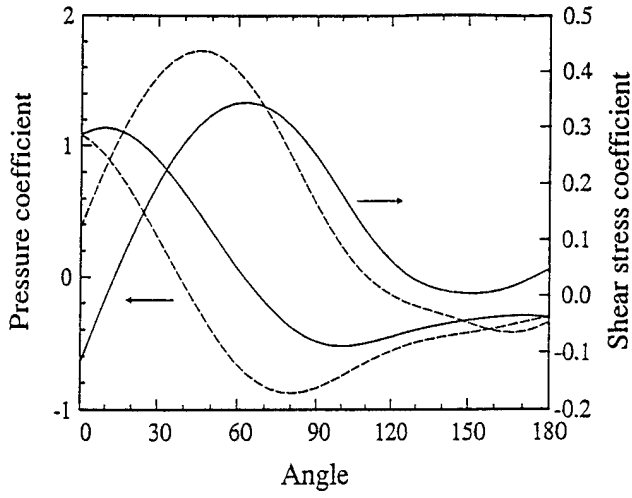
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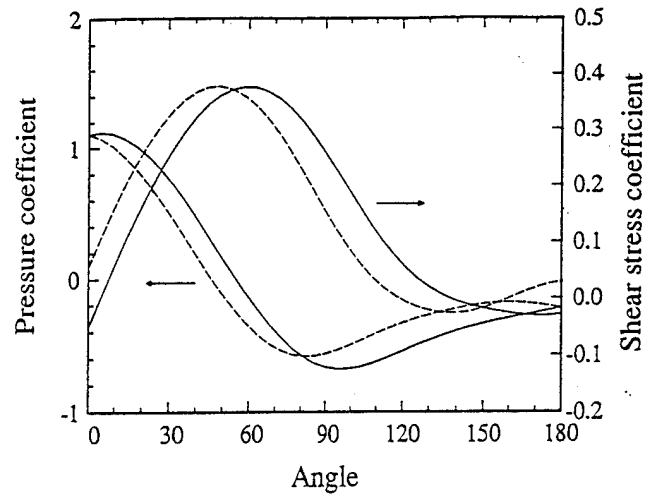
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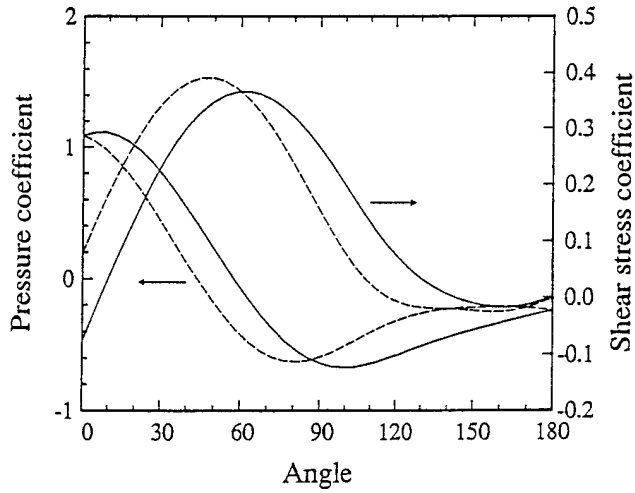
(g)



(i)



(h)



(j)

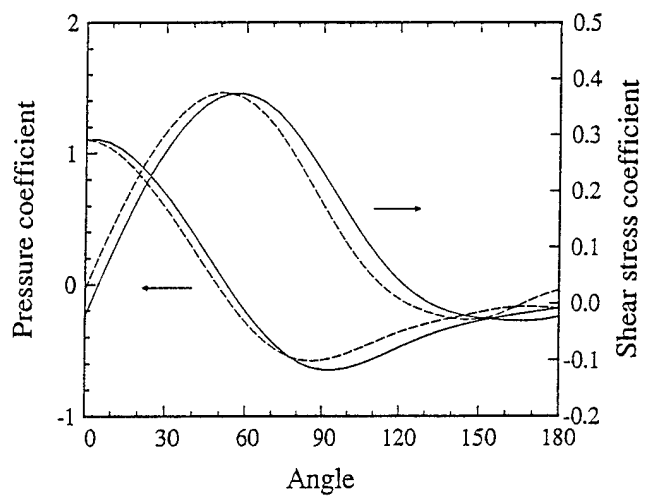


Fig. 7

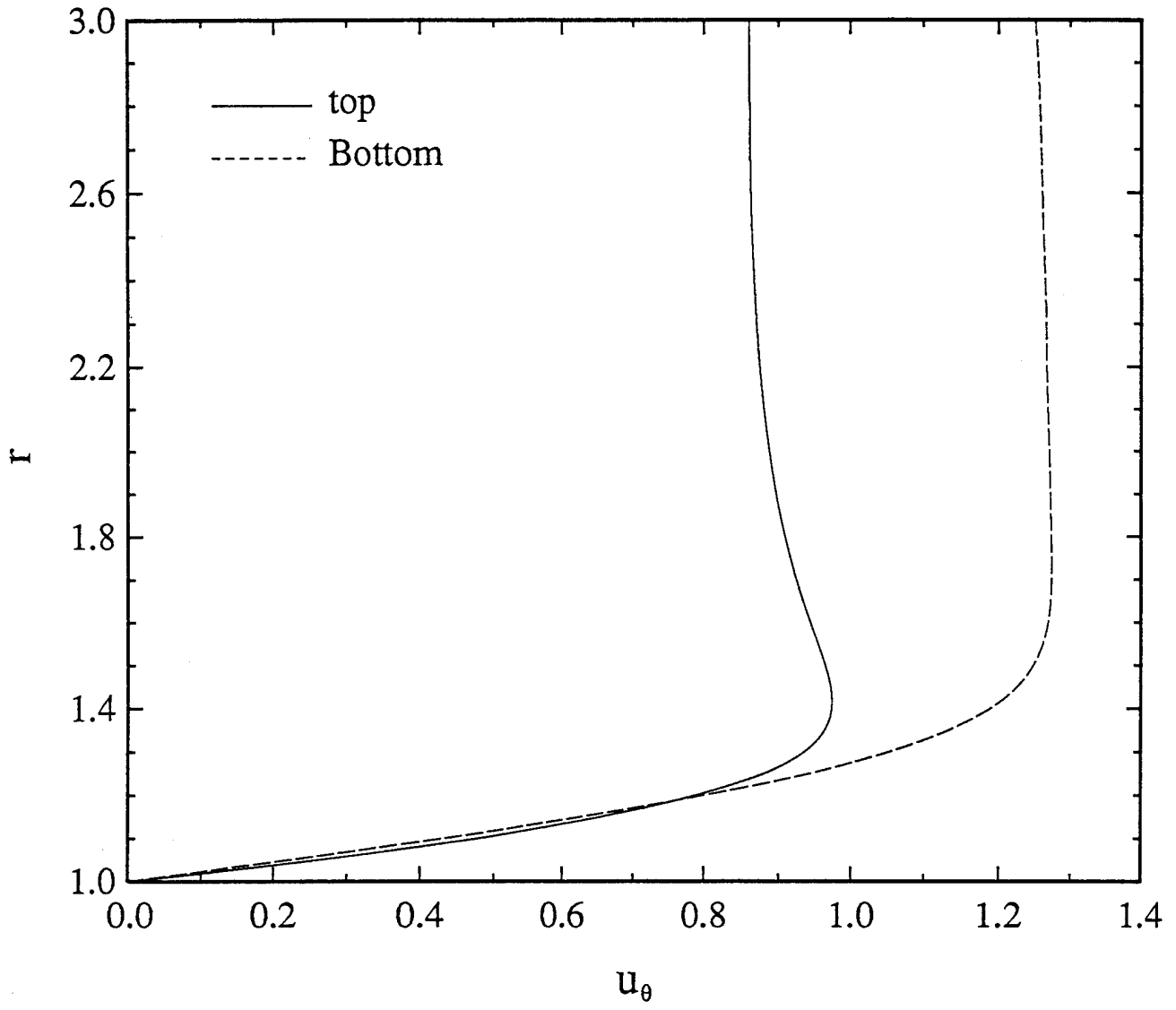


Fig. 8(a)

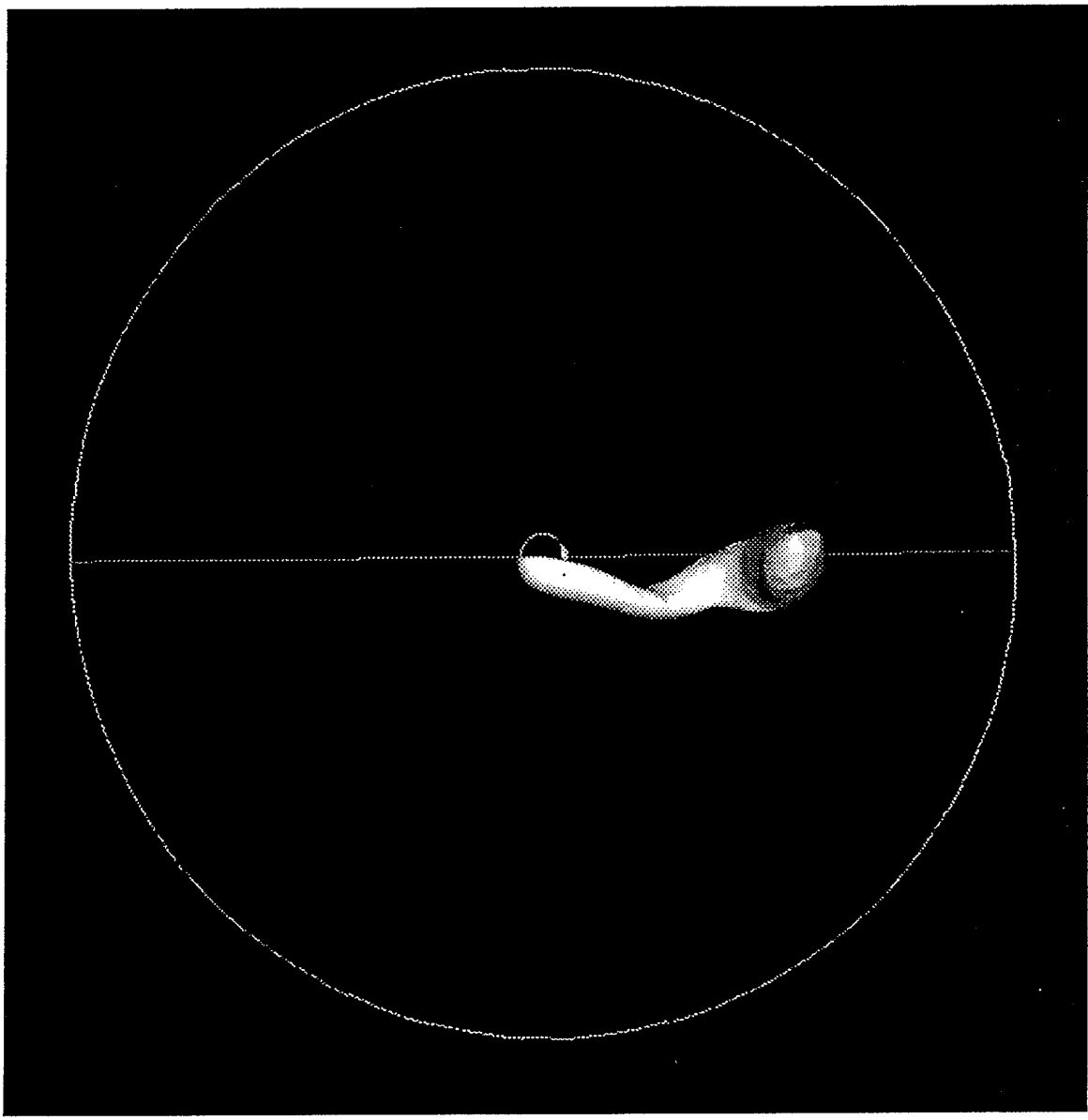
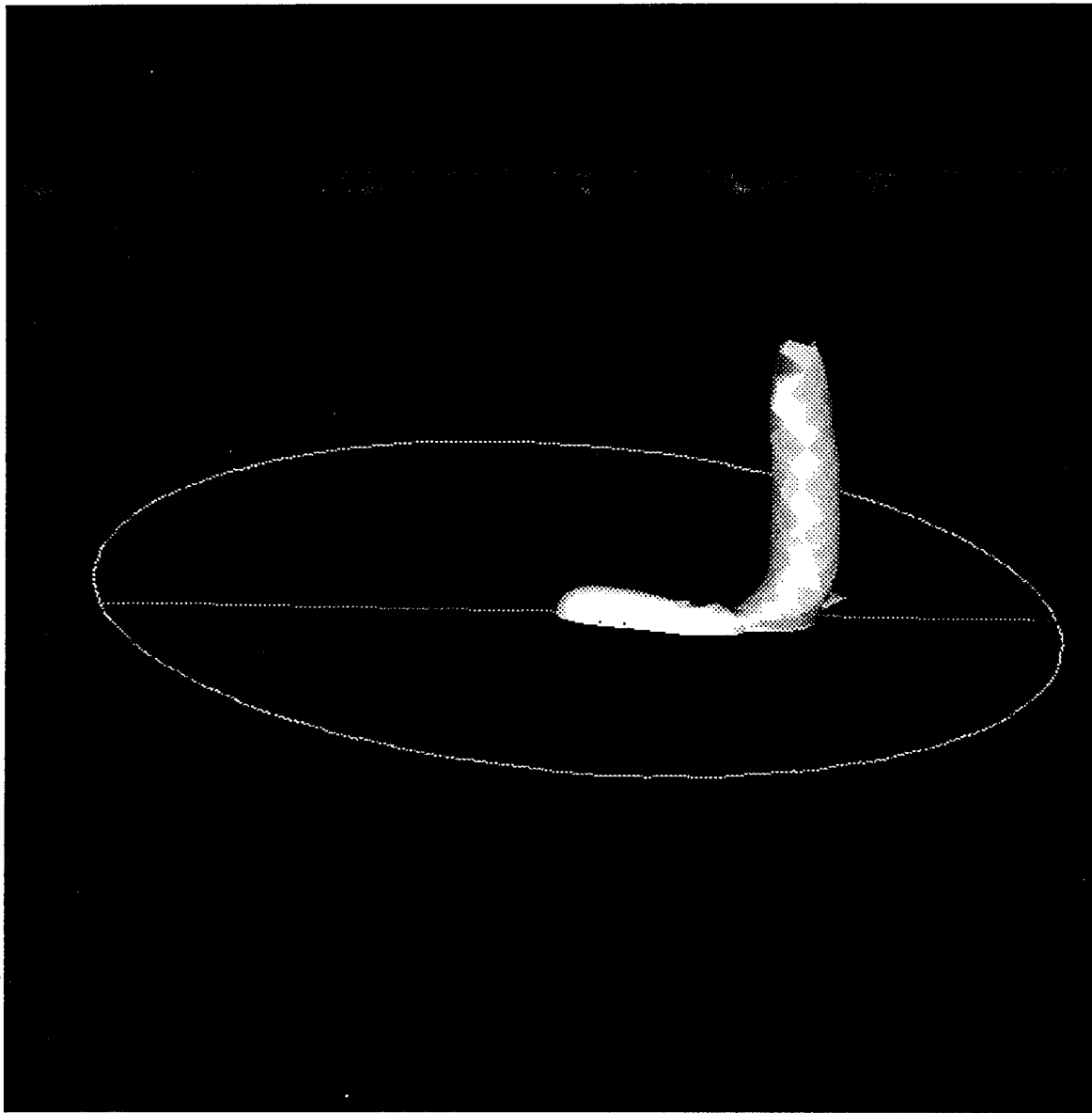
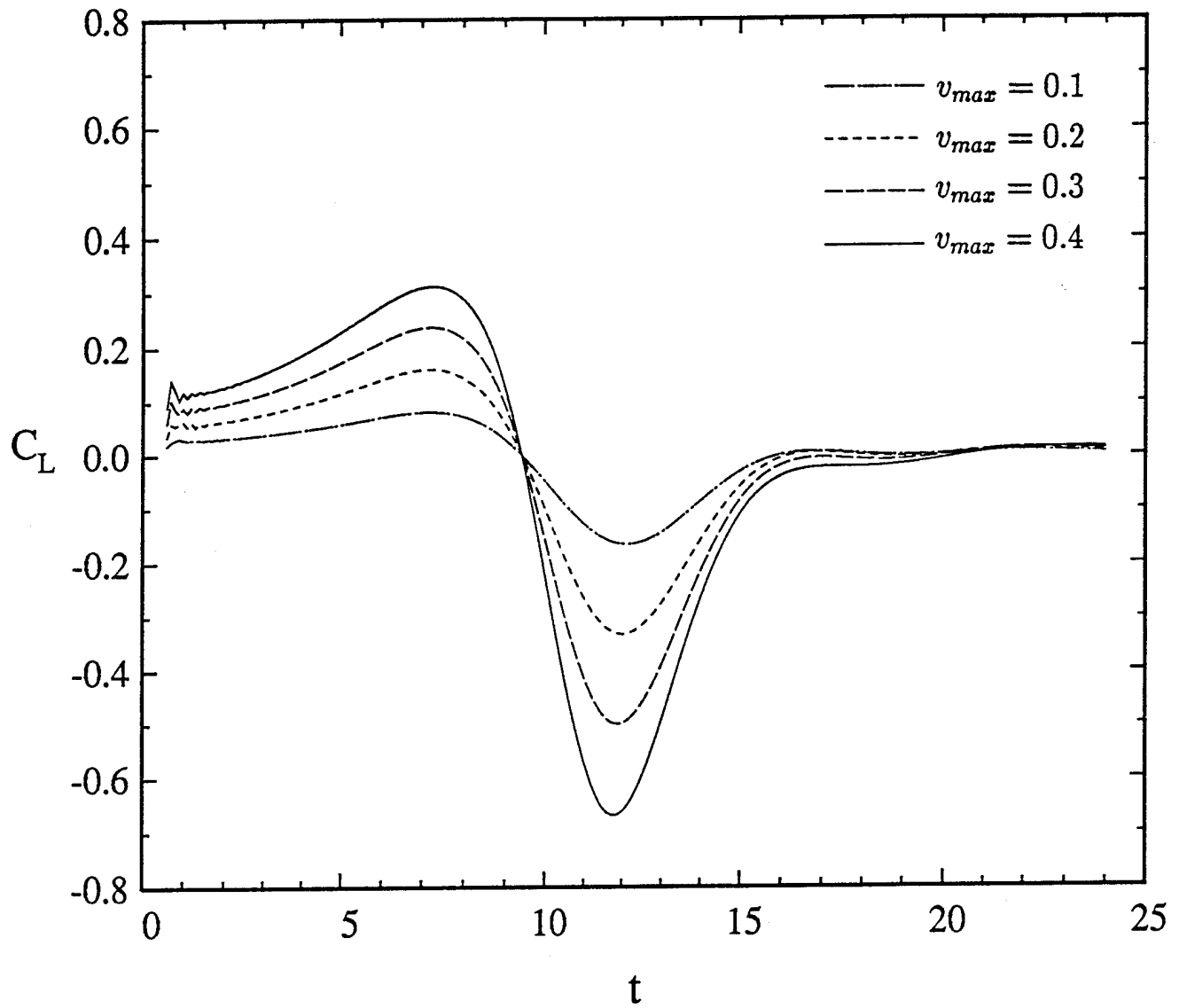
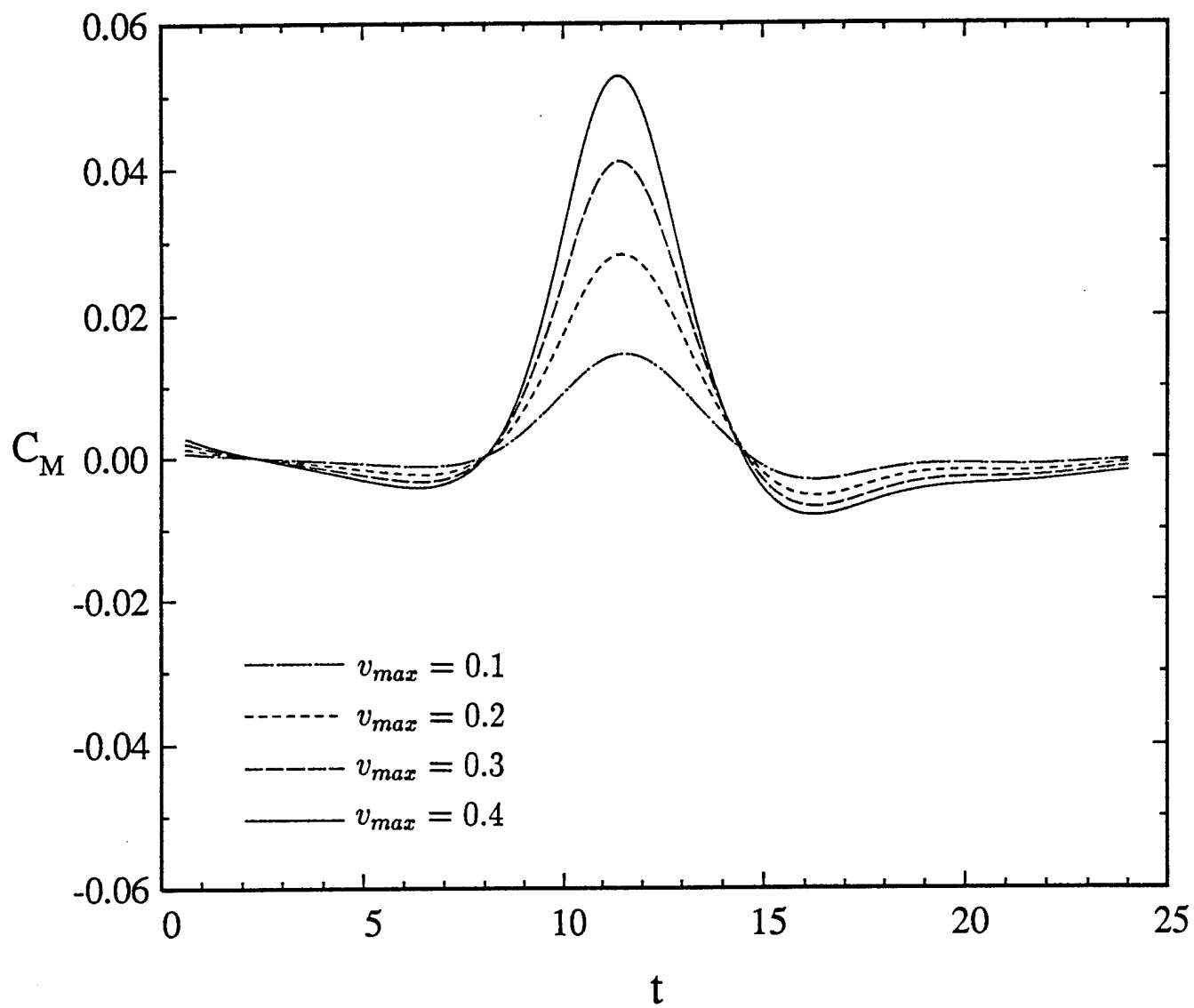
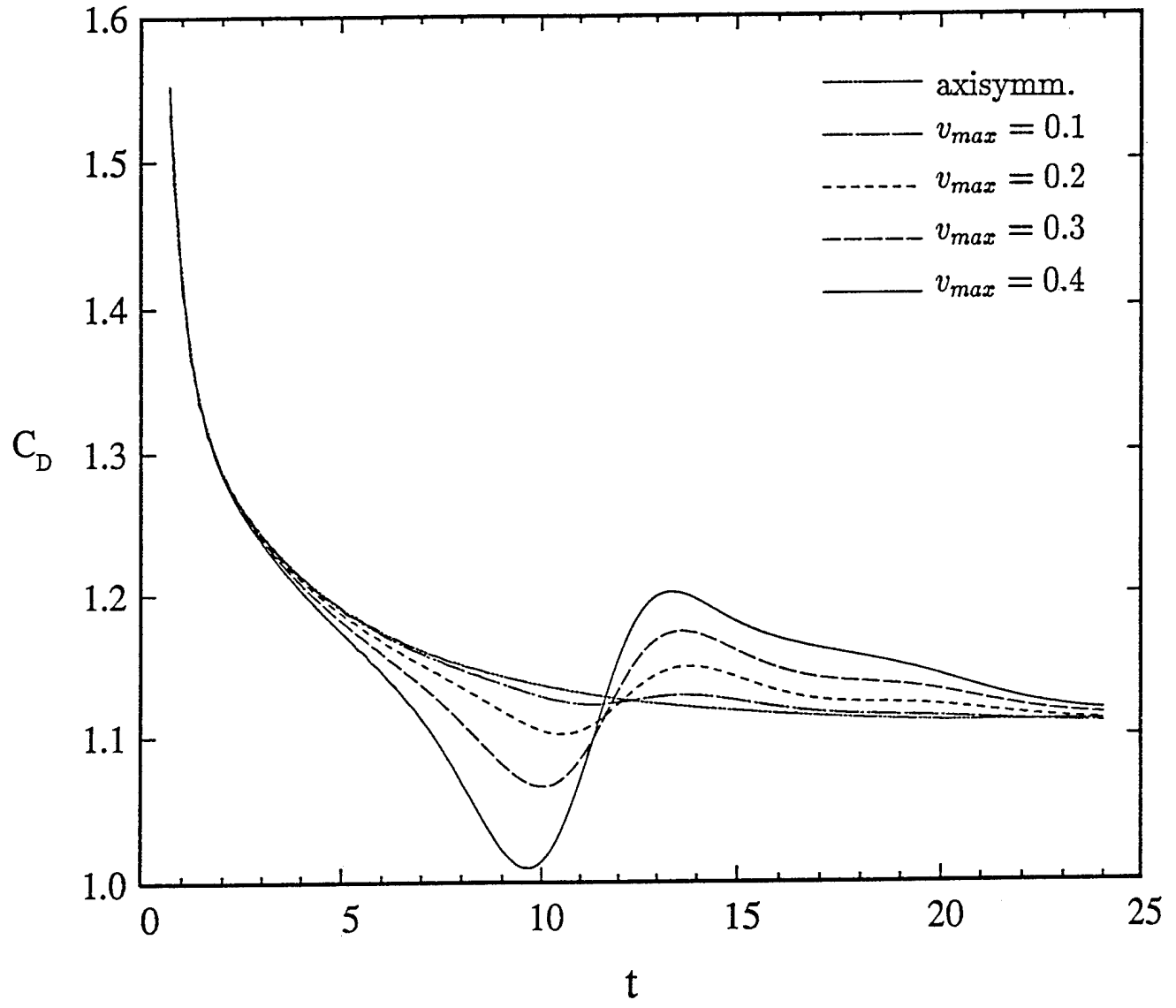


Fig. 8(b)





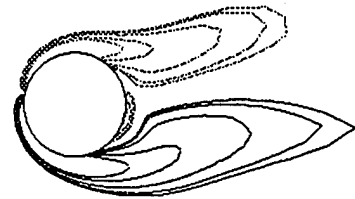




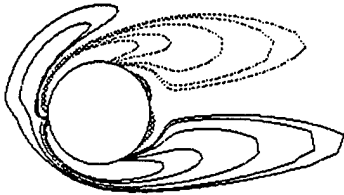
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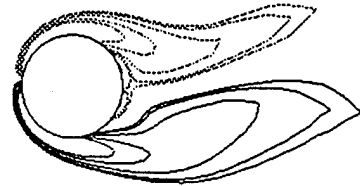
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(b)



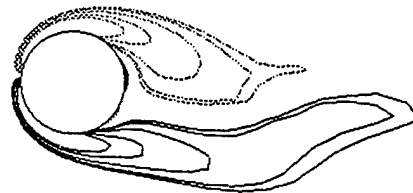
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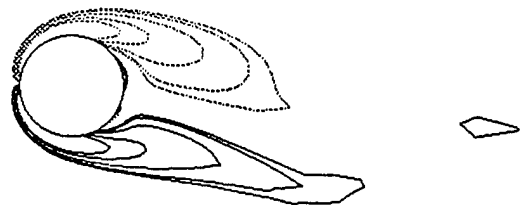
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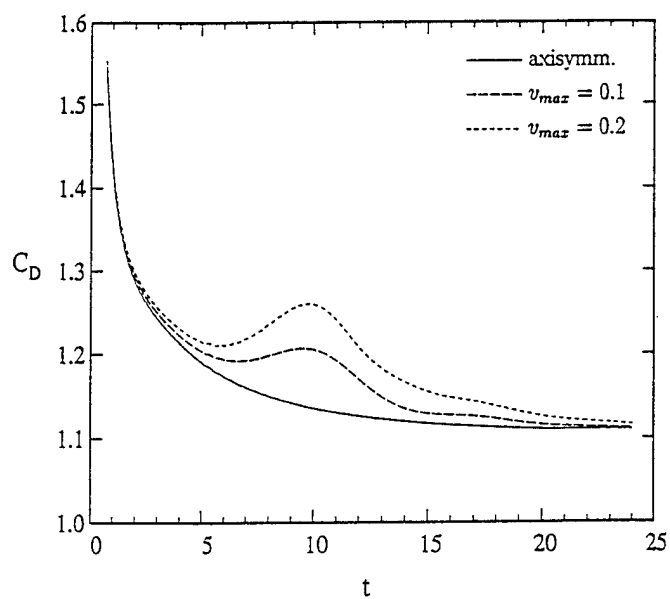
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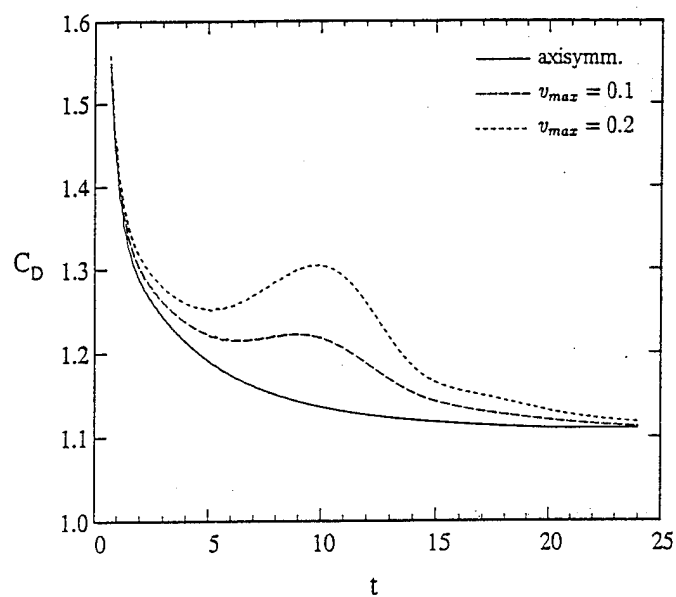
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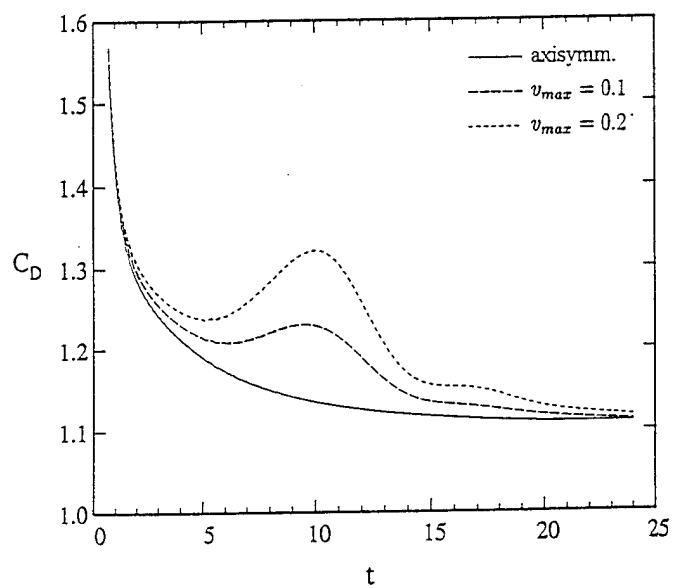
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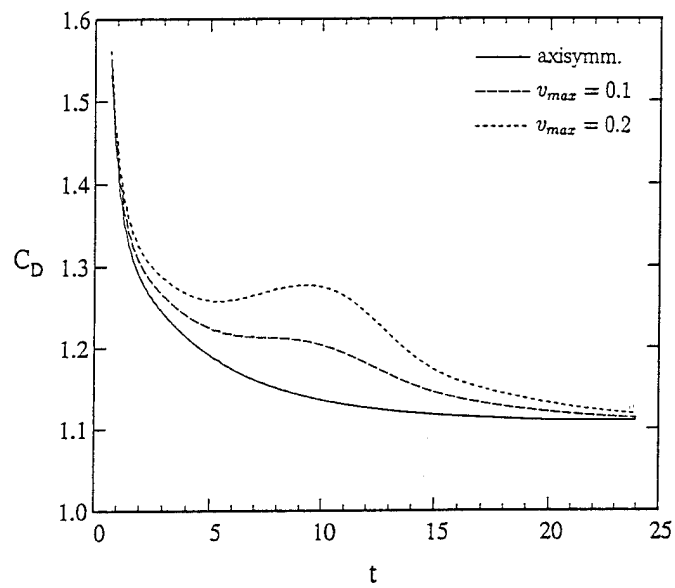
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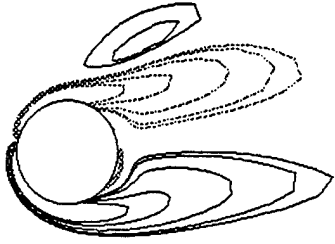
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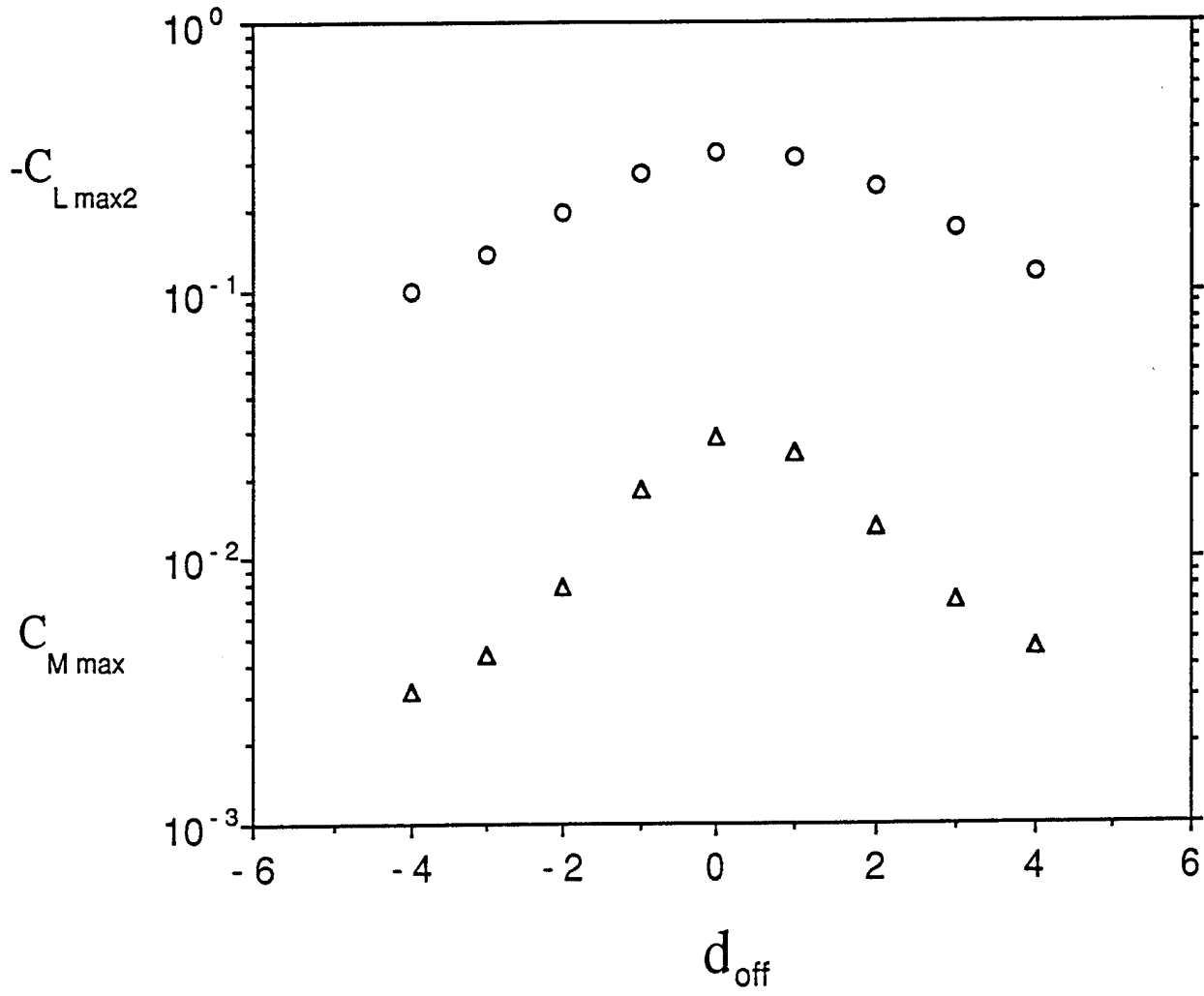


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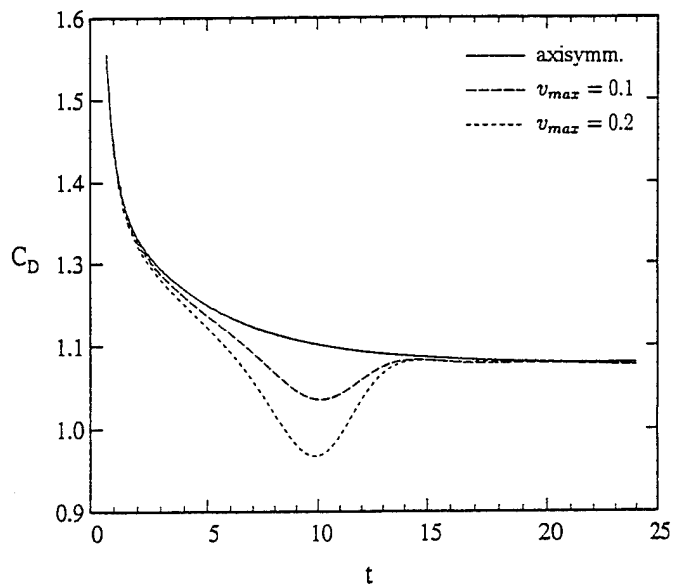


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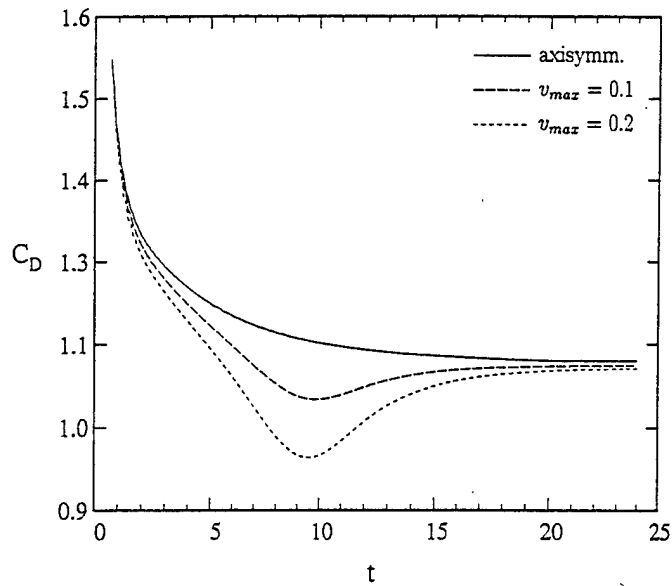




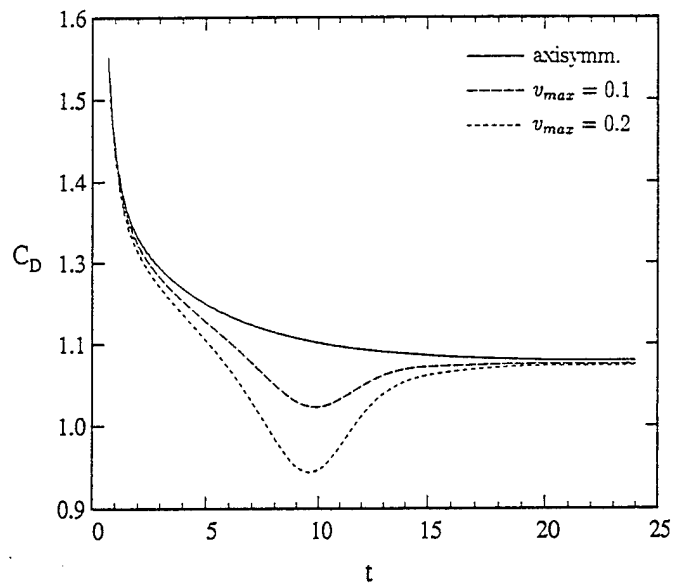
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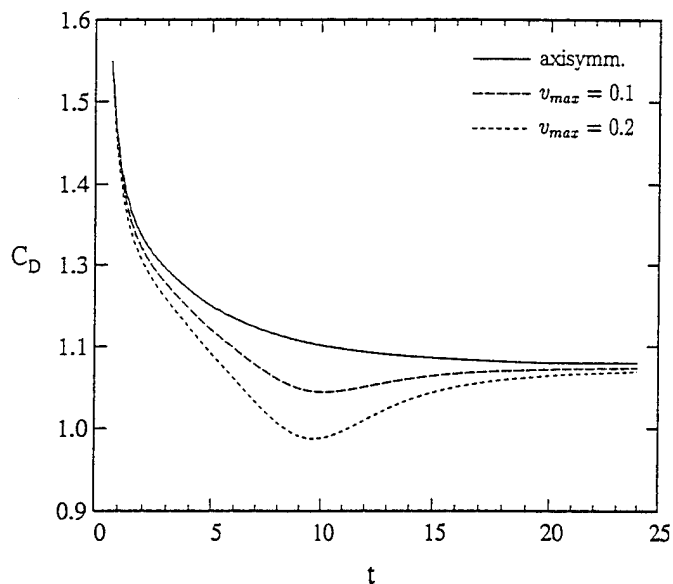
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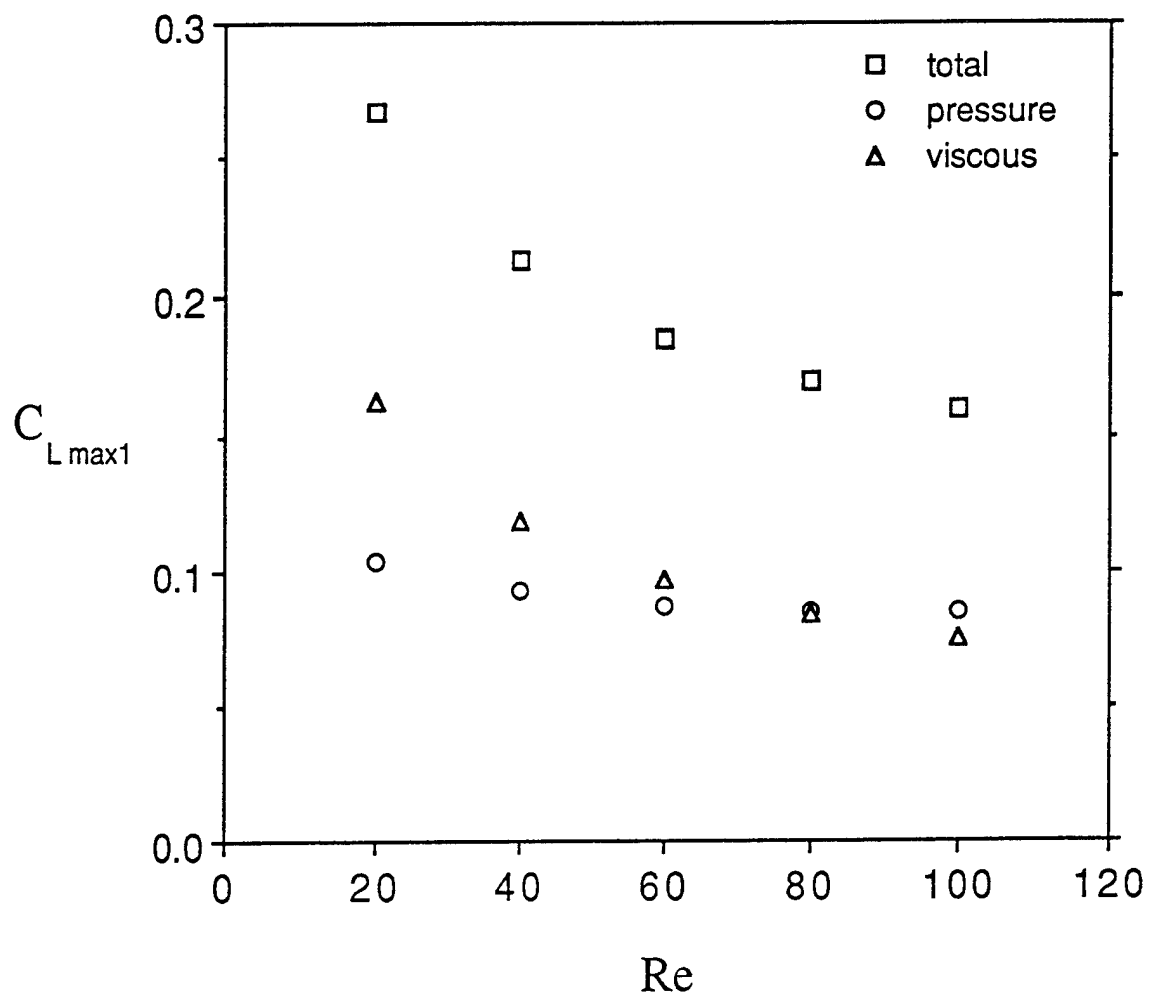


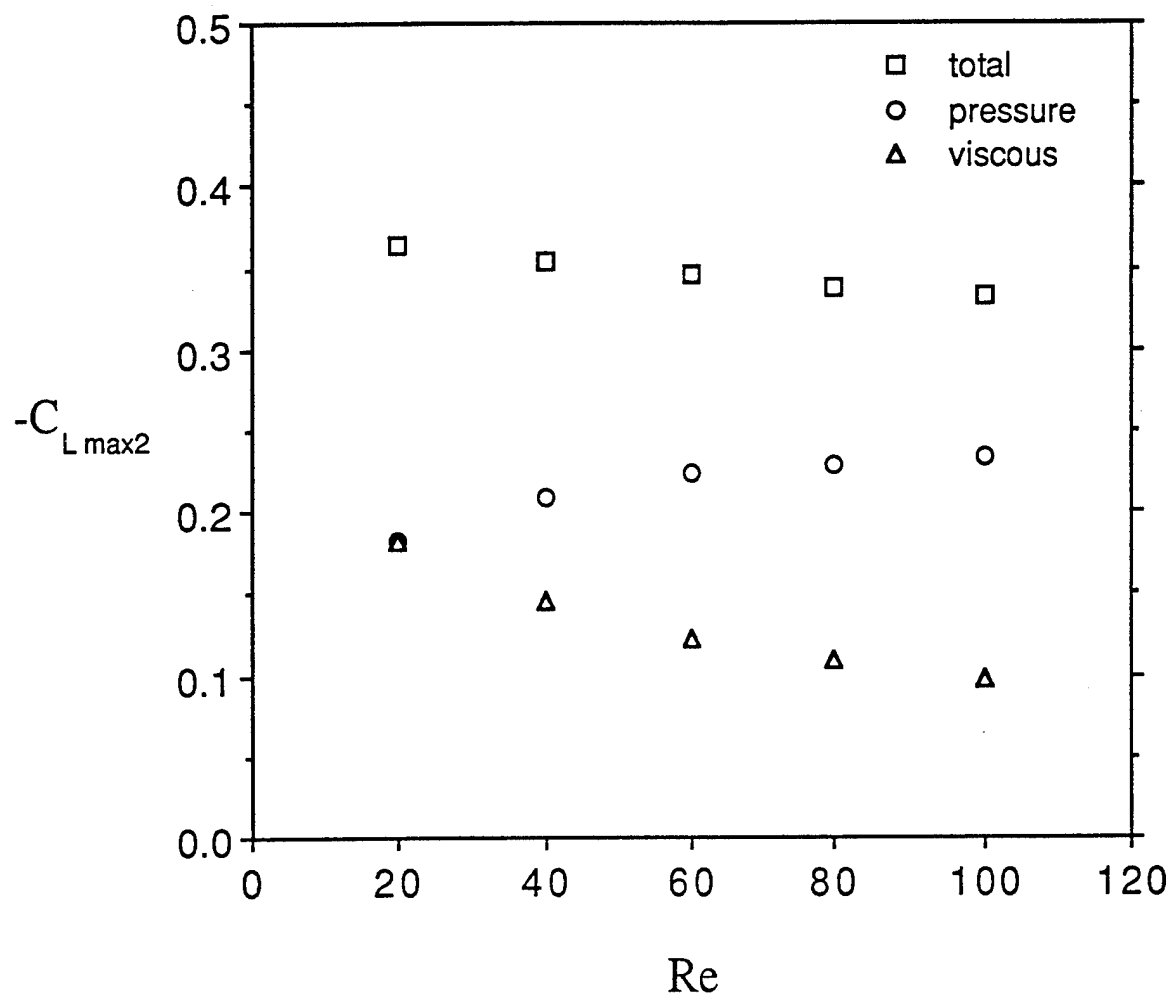
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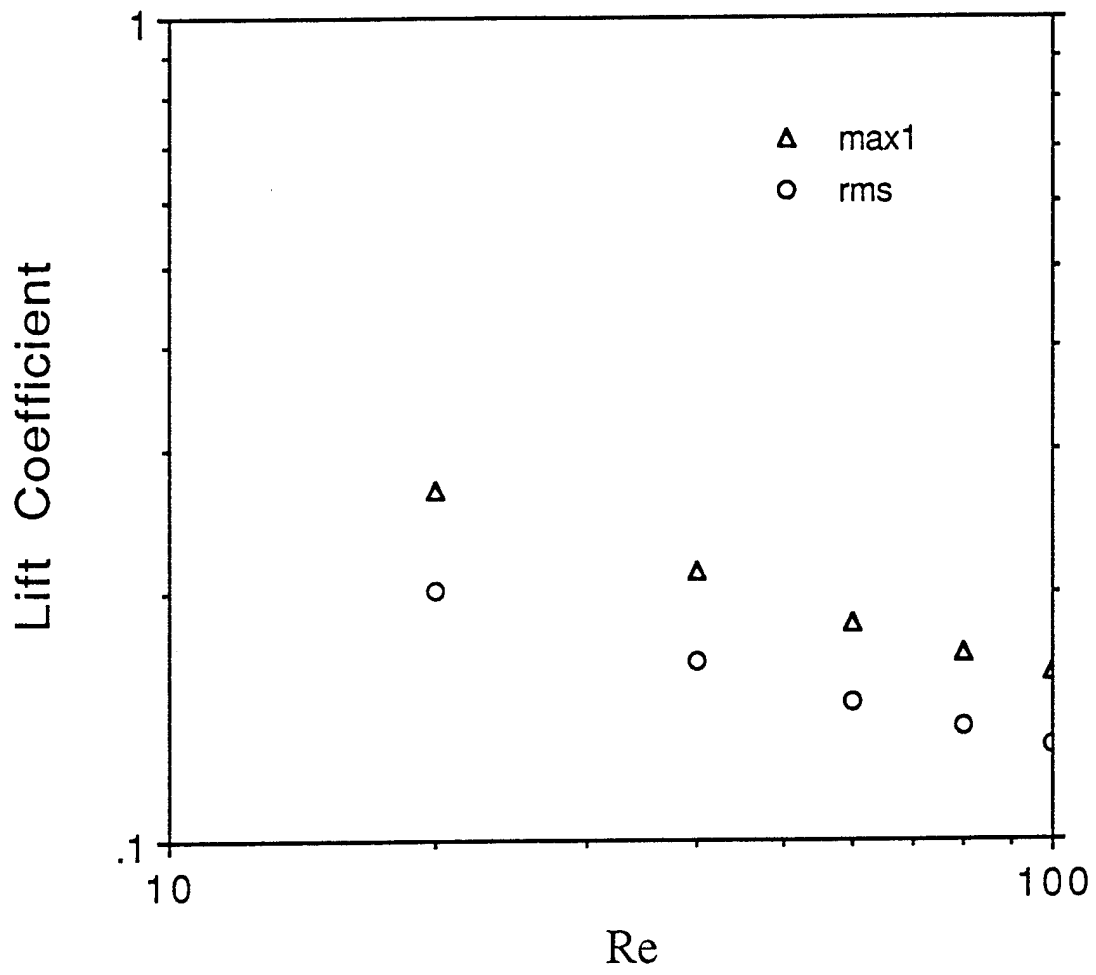


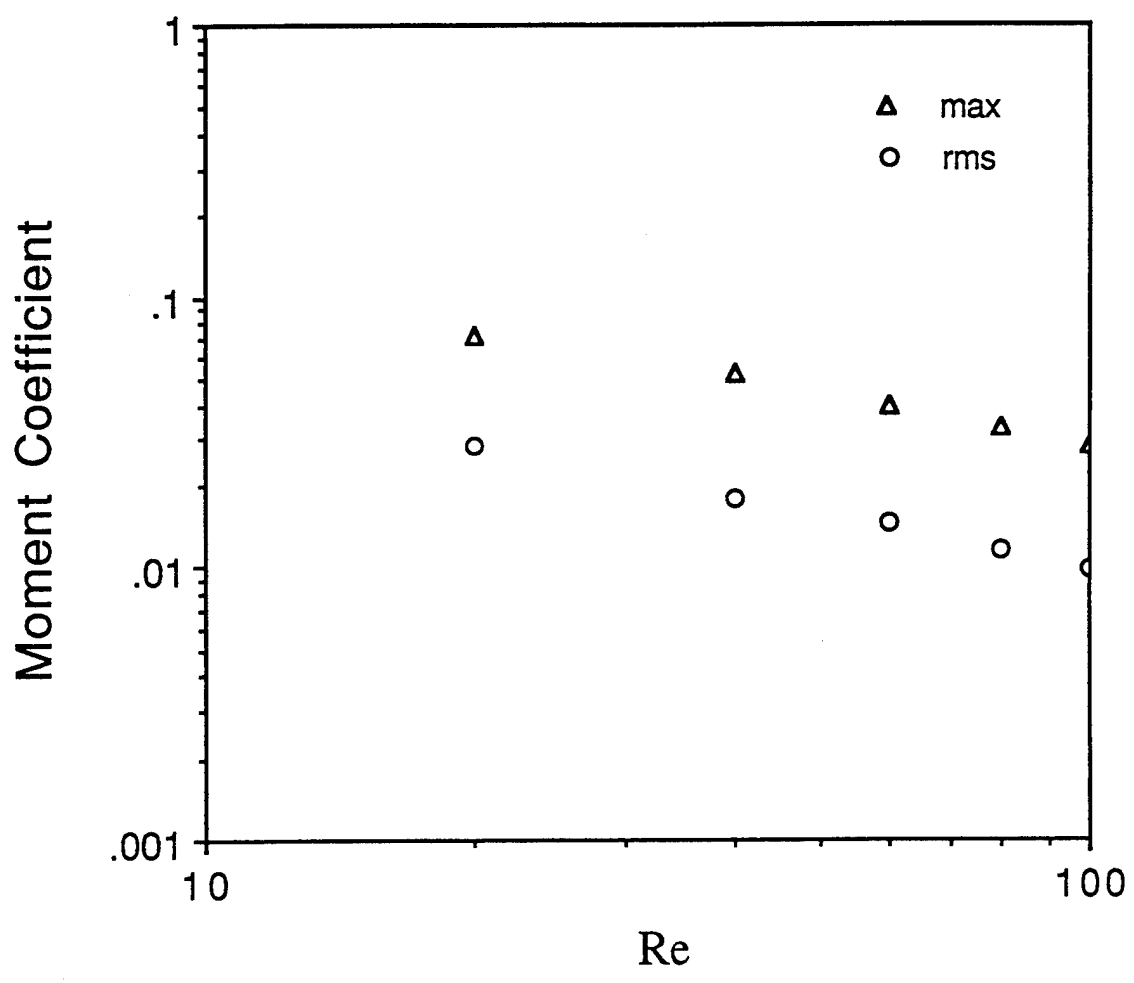
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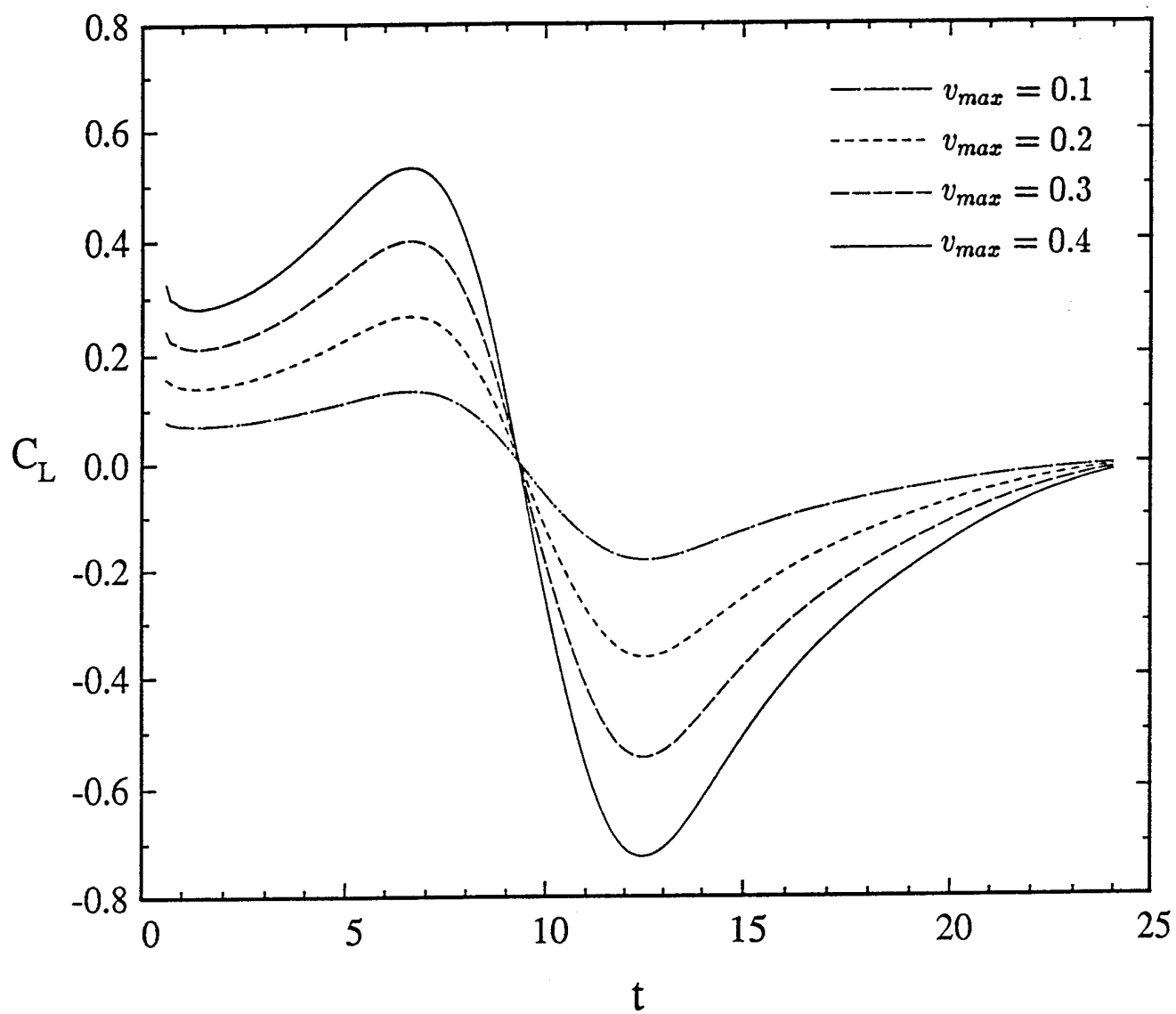


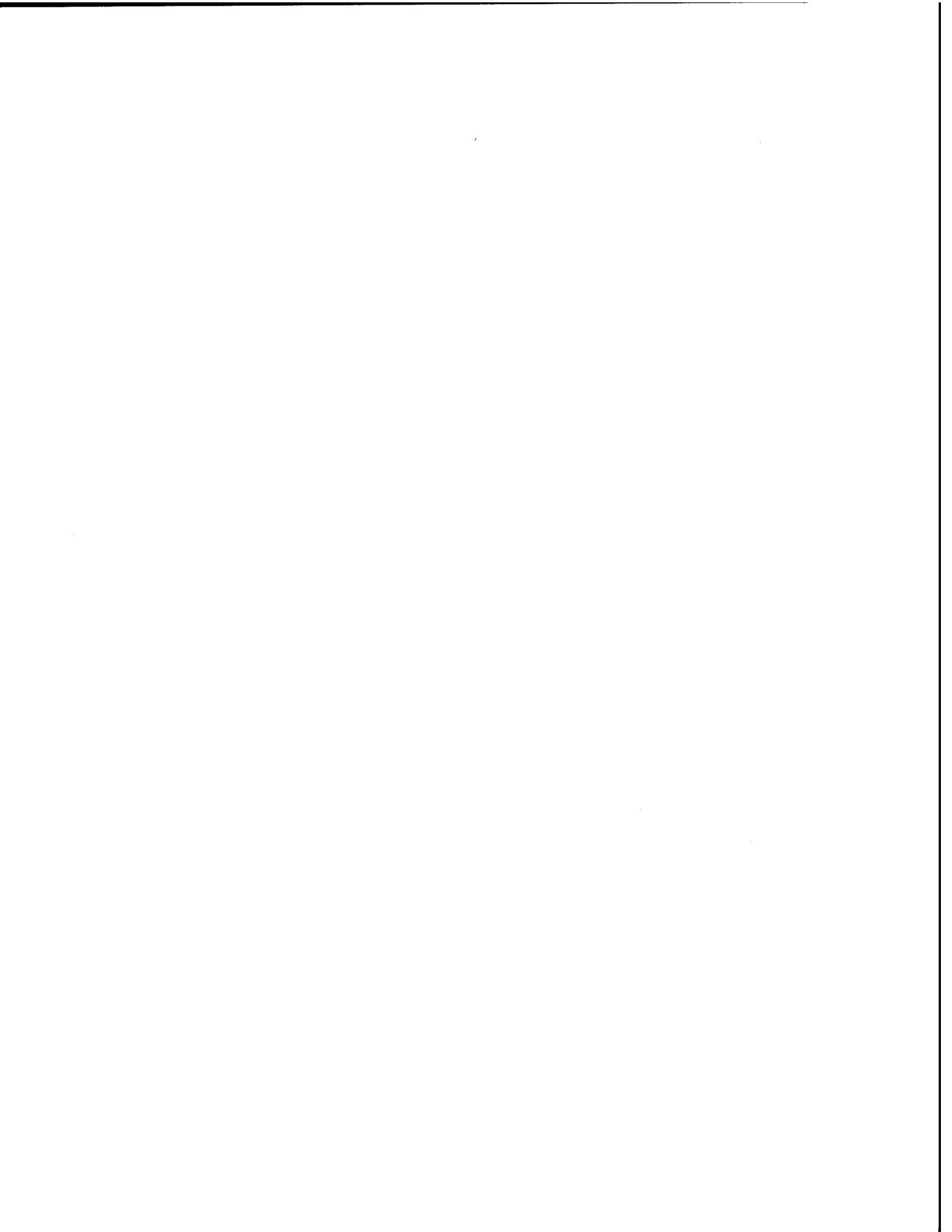














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**Unsteady Flow Interactions Between
a Pair of Advected Cylindrical Vortex Tubes
and a Rigid Sphere**

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University of California
Irvine, CA

**33rd Aerospace Sciences
Meeting and Exhibit
January 9-12, 1995 / Reno, NV**

Unsteady Flow Interactions Between a Pair of Advected Cylindrical Vortex Tubes and a Rigid Sphere *

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Department of Mechanical and Aerospace Engineering
University of California, Irvine

Abstract

This study concerns the detailed three-dimensional, viscous, incompressible interactions of vortical structures with a rigid sphere. A pair of vortical structures (initially cylindrical) advect past a sphere. Navier-Stokes equations describe the unsteady viscous flow field. Finite-difference computations yield flow properties plus temporal behavior of lift, drag, and moment coefficients of the sphere. Lift and moment coefficients are shown to be linearly proportional to the maximum velocity fluctuation. Effects of Reynolds number, initial vortex size, and initial vortex configuration are determined. Lift coefficients are used to estimate spherical particle deflection in turbulent flows; deflection is found to increase slowly as Reynolds number decreases. The moment coefficient is an order of magnitude less than the lift coefficient implying that torque is often negligible.

Nomenclature

a'_o	dimensional sphere radius
$C_{L,max1}$	maximum positive lift coefficient
$C_{L,max2}$	maximum negative lift coefficient
$C_{L,rms}$	rms lift coefficient
d_{off}	initial offset distance normalized by a'_o
N_1, N_2, N_3	numbers of grids in ξ, η, ζ directions
Re	Reynolds number based on sphere diameter
u, v, w	velocities in the x, y, and z direction
v_{max}	maximum fluctuation velocity induced by one vortex tube
v_{max2}	total maximum fluctuation velocity induced by a pair of vortex tubes
U'_∞	dimensional freestream velocity
t	time normalized by a'_o/U'_∞
x, y, z	Cartesian coordinates

Greek symbols

ξ, η, ζ	nonorthogonal generalized coordinates
σ	radius of vortex tube normalized by a'_o
Γ_j	circulation of vortex tube normalized by U'_∞ and a'_o
ω_y	y-component of vorticity vector

Superscript

' dimensional quantity

Subscript

o initial quantity

1. Introduction

Particle-laden turbulent flows are important in many natural and engineering applications such as atmospheric dispersion of pollutants, combustion systems, and transport of suspended substances in slurries or pneumatic systems. The fluid may be either gas or liquid containing particles, droplets, or bubbles. The geometric scale of the flow may be as large as weather patterns in the atmosphere of a planet, or as small as the veins transporting blood cells in the body's circulatory system.

In general, there is no analytical solution to the equation of motion of a single particle in a laminar or turbulent flow due to the nonlinear nature of the equation. This has led to extensive application of numerical approaches to study the particle motion in a turbulent flow. When the concentration of particles in a flow is small, the particle-particle interaction such as collisions or repulsions can be neglected, and the flow-particle interaction is dominated by the time response of the individual particles. This type of flow is classified as a dilute dispersed flow. Extensive research has been performed to characterize a variety of dilute dispersed flow fields. Review articles by Lumley [1], Crowe [2], and Faeth [3] provide a summary of current research results and directions.

Most studies to predict particle motion in a turbulent flow employ the Eulerian-Lagrangian approach, where particles are considered as point sources in the Navier-Stokes equations. After the local fluid velocities are obtained from the Navier-Stokes equations, the particle motion is obtained by solving the BBO

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(Basset-Boussinesq-Oseen) equation. The drag force in that equation consists of two terms, namely, the quasi-steady Stokes drag and the Basset correction. Strictly, this correction is only correct analytically as a correction to the linear theory developed for Stokes flow and valid only for very low particle Reynolds number. Thus, it is not possible in that approach to obtain the actual force and torque on the particle, heat and mass transfer rate of the particle, and the details of the flow around the particle including near wake due to the turbulent velocity fluctuation in a finite-Reynolds-number-flow. An alternative is to examine the details of the interaction between a particle and its surrounding turbulent flow by solving numerically the full Navier-Stokes equations around an individual particle with the appropriate boundary conditions and employing a simple mathematical description for turbulent velocity fluctuation.

A turbulent flow possesses a wide spectrum of eddy sizes. For example, in a typical gas turbine combustor where the Reynolds number is of the order of 10^5 and the integral length scale is of the order of $0.1m$, the smallest (Kolmogorov) length scale, η , is about $100\mu m$, which is comparable to the size of a typical fuel droplet. Motion at the large (integral) length scale contains most of the turbulent kinetic energy and governs the dispersion of particles (or droplets). On the other hand, fluid motion at the Kolmogorov length scale experiences the largest strain rates and scalar gradients in the flow. The largest scalar gradients control the important phenomena of heat and mass transfer and chemical reaction.

As a first step towards better understanding the interactions of a particle with a turbulent flow, Kim, Elghobashi & Sirignano [4] studied the three-dimensional flow interactions between a vortical (initially cylindrical) structure advected by the free stream and a spherical solid particle held fixed in space. The particle Reynolds number based on the freestream velocity and the particle diameter was in the range $20 \leq Re \leq 100$. The initial size of the cylindrical vortex tube was in the range $0.25 \leq \sigma \leq 4$, where σ is the radius of the vortex tube normalized by that of the particle. They found that the rms lift coefficient of the sphere is *linearly* proportional to the *circulation* of the vortex tube at small values of σ , on the other hand, at large values of σ , the rms lift coefficient is linearly proportional to the maximum fluctuation velocity induced by the vortex tube but *independent* of σ .

We study in this paper the interactions between a pair of vortex tubes and a rigid sphere. Our main goal is to generalize the findings of Kim *et al.* [4] by determining:

1. the details of the interaction of a pair of vortex tubes with each other and with the sphere in a viscous incompressible flow
2. the relationship between the lift coefficient of the

sphere and the maximum fluctuation velocity induced by a pair of vortex tubes

3. the effects of Reynolds number, vortex size, and initial offset distance of the vortex.

The detailed study of the interactions between the particle and the unsteady velocity field provides fundamental information about the flow behavior which can be used in developing mathematical models for two-phase flows.

2. Formulation and numerical solution

2.1 The flow description

Consider the time-dependent, three-dimensional, incompressible, viscous flow interactions between a pair of initially cylindrical vortex tubes and a solid sphere. The vortex tubes are moving with the laminar free stream, and a sphere is suddenly placed and held fixed in space as shown in Figure 1. The initial offset distance, d'_{off} , denotes the shortest distance between the initial vortical axis and the y - z plane which is parallel to the free stream. All the variables are nondimensionalized using the sphere radius a'_0 as the characteristic length and U'_∞ as the characteristic velocity, where the superscript $'$ denotes dimensional quantity. The pair of cylindrical vortex tubes, whose diameters are of the same size and of the order of the sphere diameter, are initially located ten radii upstream from the center of the sphere. The effects of the vortex tube on the sphere are negligible at this initial distance because the magnitude of the initial velocity field induced by the vortex tubes is less than 2 percent of the free stream velocity. Far upstream, the flow is uniform with constant velocity $U'_\infty \mathbf{k}$ parallel to the y - z plane. We have one symmetry plane, the x - z plane, as seen in Figure 1. A second symmetry plane (y - z) exists when the two vortices have opposite rotations. Our general formulation does not take advantage of this second symmetry.

Two coordinate systems are used in our formulation: the Cartesian coordinates (x,y,z) and the nonorthogonal generalized coordinates (ξ, η, ζ) . The origin of the former coincides with the sphere center. ξ is the radial, η is the angular, and ζ is the azimuthal coordinates. The nonorthogonal generalized coordinate system can be easily adapted to three-dimensional arbitrary geometries. In the present study, a spherical domain is used, and the grid reduces to an orthogonal, spherical grid. The grids are denser near the surface of the spherical particle, and the grid density in the radial direction is controlled by the stretching function developed by Vinokur [5]. Due to symmetry, the physical domain is reduced to a half spherical space. The domain of the flow is bounded by $1 \leq \xi \leq N_1$, $1 \leq \eta \leq N_2$, $1 \leq \zeta \leq N_3$, where $\xi = 1$ and N_1 correspond, respectively, to the sphere surface and the farfield boundary surrounding the sphere; $\eta = 1$ and N_2 denote, respectively, the positive z -axis (upstream) and the negative

z-axis (downstream); $\zeta = 1$ and N_3 refer, respectively, to the x-z plane in the positive x-direction and the x-z plane in the negative x-direction. Uniform spacing ($\delta\xi = \delta\eta = \delta\zeta = 1$) is used, for convenience, for the generalized coordinates.

The initial vortex tube has a small core region with a radius σ (normalized by the sphere radius). This core is defined such that the initial velocity induced by the vortex tube approaches zero as the distance from the center of the vortex tube goes to zero, and at distances much greater than σ , the induced velocity approaches that of a point vortex. We use the vortex tube construction of Spalart [6], which has the following stream function:

$$\psi_v(x, z, t = 0) = -\frac{\Gamma_j}{2\pi} \ln[(x - x_j)^2 + (z - z_j)^2 + \sigma^2],$$

where Γ_j is the nondimensional circulation around the vortex tube at radius σ and at the initial time. Γ_j is positive when the vortex tube rotates counterclockwise, and x_j and z_j denote the location of the center of the vortex tube. The circulation around a circular path far away from the center of the vortex is given by $\Gamma_{jt} = 2\Gamma_j$. The stream function for a pair of vortex tubes is given by

$$\psi_v(x, z, t = 0) = -\sum_{j=1}^2 \frac{\Gamma_j}{2\pi} \ln[(x - x_j)^2 + (z - z_j)^2 + \sigma^2] \quad (1)$$

2.2 Governing equations and boundary conditions

The continuity and momentum equations and the initial and boundary conditions are nondimensionalized using the sphere radius a'_0 as the characteristic length and U'_∞ as the characteristic velocity.

$$\nabla \cdot V = 0 \quad (2)$$

$$\frac{\partial V}{\partial t} + \nabla \cdot VV = -\nabla p + \frac{2}{Re} \nabla^2 V. \quad (3)$$

The governing equations (2) and (3) are cast in terms of the generalized coordinates (ξ, η, ζ) to treat a three-dimensional body of arbitrary shape. The numerical integration is performed using a cubic computational mesh with equal spacing ($\delta\xi = \delta\eta = \delta\zeta = 1$).

The velocities on the sphere surface are zero due to the no-slip condition, and the pressure on the sphere is obtained from the momentum equation. The boundary conditions are

$$\frac{\partial p}{\partial n} = \frac{2}{Re} \frac{\partial^2 V_n}{\partial n^2}, \quad u = v = w = 0 \text{ at } \xi = 1, \quad (4)$$

$$p = 0, \quad u = v = 0, \quad w = 1 \text{ at } \xi = N_1, \quad N_{2m} \leq \eta \leq N_2 \quad (5)$$

$$p = 0, \quad \frac{\partial u}{\partial \xi} = \frac{\partial v}{\partial \xi} = \frac{\partial w}{\partial \xi} = 0 \text{ at } \xi = N_1, \quad 1 \leq \eta < N_{2m} \quad (6)$$

$$\frac{\partial p}{\partial \zeta} = \frac{\partial u}{\partial \zeta} = \frac{\partial w}{\partial \zeta} = 0, \quad v = 0 \text{ at } \zeta = 1, \quad \zeta = N_3, \quad (7)$$

where u , v , and w are the velocities in the x, y, and z direction, respectively, V_n is the velocity in the direction normal to the sphere surface, and p is the pressure. n denotes the direction normal to the sphere surface, $\partial/\partial n = \sqrt{\xi_x^2 + \xi_y^2 + \xi_z^2} \partial/\partial \xi$, and $\eta = N_{2m}$ denotes the mid-plane between $\eta = 1$ and N_2 . Equation (7) corresponds to the symmetry conditions and zero v velocity in the x-z symmetry plane. Conditions guaranteeing continuity in the η direction for p , u , v , and w on the axes $\eta = 1$ and $\eta = N_2$ are also imposed.

In order to start the numerical solution of equations (2) and (3), we provide initial velocity by superposing the flow fields due to the uniform stream and the vortex tube and the no-slip condition on the sphere surface:

$$p_o = 0, \quad u_o = -\frac{\partial \psi_v}{\partial z}, \quad v_o = 0, \quad w_o = 1 + \frac{\partial \psi_v}{\partial x} \quad \text{except at } \xi = 1 \quad (8)$$

$$p_o = 0, \quad u_o = v_o = w_o = 0 \quad \text{at } \xi = 1, \quad (9)$$

where ψ_v is given by equation (1).

The only nondimensional groupings appearing in the equations and initial and boundary constraints are the sphere Reynolds number, vortex tube radius, offset distance, and vortex circulation (or vortex Reynolds number).

The drag, lift, and moment coefficients are evaluated in dimensional form as follows.

$$F'_D = \int_S -p' \mathbf{n} \cdot \mathbf{k} \, dS' + \int_S \mathbf{n} \cdot \boldsymbol{\tau}' \cdot \mathbf{k} \, dS' \quad (10)$$

$$F'_L = \int_S -p' \mathbf{n} \cdot \mathbf{i} \, dS' + \int_S \mathbf{n} \cdot \boldsymbol{\tau}' \cdot \mathbf{i} \, dS' \quad (11)$$

$$M' = \int_S \mathbf{r}' \times \boldsymbol{\tau}' \, dS', \quad (12)$$

where S' denotes the surface of the sphere, \mathbf{n} is the outward unit normal vector at the surface, \mathbf{r}' is the position vector from the center of the sphere, and $\boldsymbol{\tau}'$ is the viscous stress tensor. The lift force is assumed positive when it is directed toward the positive x-axis. Due to symmetry, only the y-component of the moment is non-zero and is assumed positive in the counter-clockwise direction.

The nondimensional coefficients of drag, lift, and moment are defined respectively as

$$C_D = \frac{F'_D}{\frac{1}{2}\rho'U_\infty'^2\pi a_o'^2} \quad (13)$$

$$C_L = \frac{F'_L}{\frac{1}{2}\rho'U_\infty'^2\pi a_o'^2} \quad (14)$$

$$C_M = \frac{M' \cdot j}{\frac{1}{2}\rho'U_\infty'^2\pi a_o'^3} \quad (15)$$

2.3 Numerical solution

A three-dimensional, implicit, finite-difference algorithm has been developed to solve simultaneously the set of the discretized partial differential equations. The method is based on an Alternating-Direction-Predictor-Corrector (ADPC) scheme to solve the time-dependent Navier-Stokes equations. ADPC is a slight variation of Alternating-Direction-Implicit (ADI) method and implemented easily when embedded in a large iteration scheme (Patnaik [7]). The control volume formulation is used to develop the finite-difference equations from the governing equations with respect to the generalized coordinates (ξ, η, ζ) . An important part of solving the Navier-Stokes equations in primitive variables is the calculation of the pressure field. In the present work, a pressure correction equation is employed to satisfy indirectly the continuity equation. The pressure correction equation is of the Poisson type and is solved by the Successive-Over-Relaxation (SOR) method.

The overall solution procedure is based on a cyclic series of guess-and-correct operations. The velocity components are first calculated from the momentum equations using the ADPC method, where the pressure field at the previous time step is employed. This estimate improves as the overall iteration continues. The pressure correction is calculated from the pressure correction equation using the SOR method, and new estimates for pressure and velocities are obtained. This process continues until the solution converges at each time step.

3. Results and discussion

3.1 Numerical accuracy

Here we examine the flow generated by an impulsively started solid sphere in a quiescent fluid at two Reynolds numbers: 20 and 100. The time-dependent solution converges asymptotically to a steady-state which is in good agreement with the available experimental data and correlations. Refer the details for numerical data to Kim *et al.* [4]. Although the solution in these test cases are axisymmetric, none of the three velocity components in our formulation becomes identically zero. Therefore, the three-dimensional solution scheme is fully exercised here. The $51 \times 51 \times 51$ grid is used in the following calculations.

3.2 Interactions of a pair of vortex tubes of like rotation and a sphere

Now consider in particular the interactions of a pair of vortex tubes advected by the free stream and a sphere suddenly placed in the flow and held fixed in space. The two vortex tubes are initially of the same size and cylindrical shape rotating counter-clockwise as shown in Figure 1. The two vortex tubes are separated by the same distance from the y-z plane so that the offset distance of one vortex tube is minus the offset distance of the other. The center of the each vortex tube is located at 10 sphere-radii upstream from x-y plane containing the center of the sphere. The base case calculation is that of $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$.

Initially each vortex tube has the maximum fluctuation velocity v_{max} on the edge of the core. Because the velocity and vorticity fields induced by one vortex tube influence those by the other, the total maximum fluctuation velocity, $v_{max,t}$, induced by two vortex tubes depends on the separation distance between them and their size and is in the range $v_{max} \leq v_{max,t} \leq 2v_{max}$ with $2v_{max}$ when $|d_{off}| = 0$ and v_{max} when $|d_{off}| \gg 1$. For example, $v_{max,t}$ is 0.59 for $v_{max} = 0.4$, $d_{off} = \pm 1.5$, and $\sigma = 1$.

The x-z plane of symmetry is defined as the *principal plane*, where the strongest interactions occur between the vortical structure and the sphere. The line connecting the front and rear stagnation points in the standard axisymmetric flow over a single sphere, which is the $x = 0$ line in the principal plane, will be used as a reference line. We refer to the region above the line as the 'upper' region and that below the line as the 'lower' region.

Figures 2(a)-(f) display the contour lines of y-component vorticity in the principal plane at $t = 1, 6, 10, 15, 21,$ and 30 for $Re = 100$, $d_{off} = \pm 1.5$, $\sigma = 1$ with $v_{max,t} = 0.59$. The contour values of the vorticity are $\pm 0.4, \pm 0.8, \pm 1.4, \pm 2$, with the highest magnitude at the sphere surface. The solid and dotted lines in the figures represent positive and negative values, respectively.

The vorticity contours in figures 2(a) and 2(b) show that the vortex tubes not only are advected downstream but also rotate each other. The contour lines of vorticity in the figures also show that the viscous diffusion takes place. Figures 2(c) and 2(d) show that the vortex tubes contact the boundary layer of the sphere and go around the bottom of the sphere. When the vortex tube comes close to the sphere boundary layer, it augments the magnitude of the edge velocity in the lower boundary layer and reduces the edge velocity in the upper boundary layer. The result is a higher strength vorticity in the lower boundary layer than in the upper boundary layer (see the vorticity contours in

figure 2(c)). Consequently, the vorticity in the lower boundary layer induces a velocity in the downward direction at the location of the vortex tube with higher magnitude than that induced by the vorticity in the upper boundary layer. This downward induced velocity advects the vortex tube below the sphere (Kim *et al.* [4]). Figure 2(e) shows that the pairing vortex tubes merge into one vortex due to the attraction of each other and the viscosity.

3.2.1 Lift, moment, and drag coefficients and effect of tube circulation

Figure 3 shows the lift coefficients of the sphere as a function of time for $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$. The lift coefficients are computed for four different total maximum fluctuation velocities $v_{max t}$ induced by the pair of vortex tubes, with magnitudes equal to 0.148, 0.295, 0.443, and 0.590 ($v_{max} = 0.1, 0.2, 0.3, \text{ and } 0.4$). The lift coefficient of the sphere interacting with a single vortex tube as a function of time is also shown as a reference in figure 3 for $Re = 100$, $d_{off} = 0$, and $\sigma = 1$ with $v_{max t} = 0.148$. The connection between the case of a single vortex tube and that of a pair of vortex tubes will be discussed in the next section in detail. Due to the sudden placement of the sphere into the stream, it takes a short time ($0 < t < 0.8$) for the initial flow perturbations to vanish.

When the pair of vortex tubes approach the sphere ($0 \leq t < 9$), they produce upwash resulting in a positive lift force on the sphere. The maximum positive lift coefficient $C_{L,max1}$ occurs at about $t = 6.8$. On the other hand, when the vortex tubes pass the sphere, they produce downwash and higher fluid velocity near the bottom of the sphere than the top due to the shear flow imposed by the vortex tubes resulting in a negative lift force. The magnitude of the negative lift is greater than the positive lift. The maximum negative lift coefficient $C_{L,max2}$ occurs at about $t = 12.2$. $C_{L,max1}$ and $C_{L,max2}$ are linearly proportional to the maximum fluctuation velocity. The maximum positive lift coefficient $C_{L,max1}$ is expressed by

$$C_{L,max1} = c v_{max t}, \quad (16)$$

where the proportionality constant $c = 0.88$. The maximum negative lift coefficient $C_{L,max2}$ is also expressed by equation (16) but with $c = -1.62$. After the lift coefficient reaches its maximum negative value, it decays quickly towards zero because the vortex tube vorticity is diffused in the sphere wake. The time averaged lift coefficient (averaged over a time span between $t = 0.8$ and the maximum time 24.5) for all values of $v_{max t}$ is nearly zero ($O(10^{-2})$). As mentioned earlier, the behavior of $C_L(t)$ during the period $0 < t < 0.8$ is influenced by the initial flow perturbation, and thus its value during this initial period is excluded from the averaging process. The root mean square $C_{L,rms}$ of the

lift coefficient as a function of time is also linearly proportional to $v_{max t}$ with $c = 0.7$.

Figure 4 shows the temporal development of the moment coefficients for the sphere under the same conditions of Figure 3. The moment coefficient of the sphere interacting with a single vortex tube as a function of time is also shown as a reference in figure 4 for $Re = 100$, $d_{off} = 0$, and $\sigma = 1$ with $v_{max t} = 0.148$.

When the vortex tubes pass the sphere, the front stagnation point on the sphere is shifted above the plane $x = 0$ due to the downwash. This causes higher shear stress in the lower left region compared to the upper left region resulting in a positive (counter-clockwise) torque on the sphere. The upward shift of the front stagnation point also causes the shear stress to be higher in the top and upper right regions than in the bottom and lower right regions resulting in a negative torque on the sphere. However, the effect of this negative torque is diminished by the shear flow induced by the vortex tubes across the sphere which produces high shear stress at the bottom of the sphere. As a consequence, a net high positive torque acts on the sphere. The maximum positive moment coefficient $C_{M,max}$ occurs at $t = 11.5$. $C_{M,max}$ is linearly proportional to $v_{max t}$ with $c = 0.11$.

When the vortex tubes approach the sphere or are relatively far away from the sphere, the effect of the shear flow induced by the vortex tubes across the sphere is small, resulting in a net weak torque on the sphere.

The time averaged moment coefficient (averaged over a time span between $t = 0.6$ and 24.5) for all values of $v_{max t}$ is $O(10^{-3})$, and the root mean square $C_{M,rms}$ of the moment coefficient is linearly proportional to $v_{max t}$ with $c = 0.043$. We note that the torque depends only the distribution of the shear stresses ($\tau_{r\theta}$ and $\tau_{r\phi}$) and is relatively small compared to the lift force.

Figure 5 shows the drag coefficients of the sphere as a function of time for the same conditions of Figure 4. The drag coefficients are computed for four different values of $v_{max t}$ as in Figure 4, in addition to $v_{max t} = 0$ which corresponds to the axisymmetric flow without the vortex tubes.

The initially large values of shear stress and pressure on the sphere results in large drag as shown in Figure 5. When the vortex tubes approach the sphere, the pressure at the front stagnation point is lower than that of the axisymmetric flow past a sphere due to the low pressure at the center of the vortex tube. Also, the maximum shear stresses in the upper and lower regions of the sphere are lower than those of the axisymmetric flow. This causes the drag on the sphere to be lower than that of the axisymmetric flow without the vortex tube. As the vortex tubes move around the bottom of the sphere, the front stagnation point is shifted above the plane $x = 0$ due to the downwash. Consequently, high pressure and high shear stress act respectively in the upper left region and the lower left region. This

increases the drag during the period $9 < t \leq 13.4$. For $t > 13.4$, the drag approaches that of the axisymmetric flow as the vortex tube moves further downstream. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of v_{max} is nearly zero ($O(10^{-4})$).

3.2.2 Effects of the size and the offset distance of the vortex tube

The effects of the size of the vortex tube on the flow field are studied by performing computations similar to those in section 3.2.1 for $Re = 100$, $d_{off} = \pm 1.5$, and five different sizes of the vortex tube, $0.25 \leq \sigma \leq 4$ in addition to the base case $\sigma = 1$.

Table 1 shows $C_{L,max1}$, $C_{L,max2}$, $C_{L,rms}$, $C_{M,max}$, and $C_{M,rms}$ as a function of the vortex tube size which covers six different initial radii of the vortex tube, $\sigma = 4, 3, 2, 1, 0.5$, and 0.25 , for $v_{maxt} = 0.1$. Another computation with different v_{maxt} showed that all the lift and moment coefficients are linearly proportional to v_{maxt} at each σ . When $\sigma \geq 2$, $C_{L,max1}$ and $C_{L,rms}$ become independent of σ , but the magnitudes of $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ for $\sigma = 4$ are smaller than those for $\sigma = 2$ and 3 . When σ approaches zero, all the coefficients tend to be proportional to (σv_{maxt}) or (σv_{max}) which is proportional to the circulation of the vortex tube. For example, $C_{L,rms}$ is expressed by

$$\begin{aligned} C_{L,rms} &= c_1 v_{maxt}, \quad 2 \leq \sigma \leq 4 \\ &= c_2 v_{maxt} \sigma^n, \quad 0.25 \leq \sigma < 2, \\ &\quad 0.75 \geq n \geq 0.3, \quad (17) \end{aligned}$$

where the constant $c_1 = 1$ and $c_2 = 0.7$, and n depends on σ and should approach unity as σ reaches zero. $C_{L,max1}$, $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ are also expressed by equation (17) with $c_1 = 1.1$ and $c_2 = 0.88$, $c_1 = -2$ and $c_2 = -1.65$, $c_1 = 0.13$ and $c_2 = 0.11$, and $c_1 = 0.053$ and $c_2 = 0.04$, respectively. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of σ is nearly zero ($O(10^{-4})$).

Note that $C_{L,max2}$, $C_{M,max}$, and $C_{M,rms}$ for $\sigma = 4$ are, respectively, smaller than those for $\sigma = 2$ and 3 , and the reason is explained as follows. When the initial size of the vortex core is considerably larger than the sphere size ($\sigma \geq 4$), the effect of the shear flow (induced by the passage of the vortex tube) across the sphere diminishes. As a result, the magnitude of $C_{L,max2}$ decreases and approaches $C_{L,max1}$ as $\sigma \gg 1$. We explained in section 3.2.1 that the magnitude of $C_{L,max2}$ depends on the combined effect of the downwash and the shear flow across the sphere due to the vortex tube. $C_{M,max}$ and $C_{M,rms}$ also decreases for the same reason. In addition, when the initial size of the vortex core is larger than the sphere size, the effect of the wake behind the sphere on the vortex tube diminishes. As a consequence, the magnitudes of the lift and moment

coefficients decay slowly towards zero after they peak near the time of passage of the pair of vortex tubes by the sphere.

In summary, the maximum positive lift coefficient and the rms lift coefficient depend only on circulation at small values of σ while they depend only on v_{maxt} (and not σ) at large values of σ . For mid-range values of σ , they depend on both σ and v_{maxt} (or equivalently both σ and Γ_j).

Now, the effects of the offset distance on the flow field are investigated by varying d_{off} for $Re = 100$ and $\sigma = 4$. The computation was performed for $d_{off} = \pm 1, \pm 2, \pm 3, \pm 4$, and 0 in addition to the base case $d_{off} = \pm 1.5$. Note that the case of $d_{off} = 0$ corresponds to the interaction between a single vortex tube and a sphere.

It is found that $C_{L,max1}$, $C_{L,max2}$, $C_{L,rms}$, $C_{M,max}$, and $C_{M,rms}$ for each d_{off} are linearly proportional to v_{maxt} as in the case of $d_{off} = \pm 1.5$. The triangular symbols in Figure 6 show $C_{L,rms}$ as a function of $|d_{off}|$ for $Re = 100$ and $\sigma = 4$ while the maximum fluctuation velocity (or the circulation) of each vortex tube is kept as a constant, $v_{max} = 0.2$. The figure shows that $C_{L,rms}$ decays rapidly as $|d_{off}| > 0$ for $v_{max} = 0.2$. On the other hand, the circular symbols in Figure 6 show $C_{L,rms}$ as a function of $|d_{off}|$ for $Re = 100$ and $\sigma = 4$ while the total maximum fluctuation velocity induced by the two vortex tubes is kept as a constant, $v_{maxt} = 0.2$. They also show that the magnitudes of the rms lift coefficients for $d_{off} = \pm 1, \pm 1.5, \pm 2, \pm 3$, and ± 4 are close to that for $d_{off} = 0$. After examining the effect of offset distance for $\sigma = 1$ and 2 , we found that the equation of the rms lift coefficient for a single vortex tube can be applicable to that for a pair of vortex tubes when the separation distance between their centers is less than $2\sqrt{\sigma}$ vortex tube diameter for $Re = 100$ if v_{maxt} is used instead of v_{max} in the equation of the rms lift coefficient. The behaviors of $C_{L,max1}$ and $C_{L,max2}$ as a function of $|d_{off}|$ are similar to that of $C_{L,rms}$.

The magnitude of the rms moment coefficient as a function of $|d_{off}|$ decays faster than that of the rms lift coefficient as $d_{off} > 0$, and the equation of the rms moment coefficient for a single vortex tube can be applicable to that for a pair of vortex tubes when the separation distance between their centers is less than $\sqrt{\sigma}$ vortex tube diameter for $Re = 100$ if v_{maxt} is used instead of v_{max} in the equation of the rms moment coefficient. The behavior of $C_{M,max}$ as a function of d_{off} is similar to that of $C_{M,rms}$.

3.2.3 Effects of Reynolds number

Similar computations to those in section 3.2.2 are made for four different Reynolds numbers in the range of $20 \leq Re \leq 80$, $d_{off} = \pm 1.5$, and $\sigma = 1$ in addition to the base case $Re = 100$.

$C_{L,max1}$, $C_{L,max2}$, $C_{L,rms}$, $C_{M,max}$, and $C_{M,rms}$ are

linearly proportional to v_{maxt} for each Reynolds number as in the case of $Re = 100$. Figure 7 shows $C_{L,max1}$ and $C_{L,rms}$ as a function of Reynolds number for $d_{off} = \pm 1.5$ and $\sigma = 1$ with $v_{maxt} = 0.295$. $C_{L,max1}$ increases with a negative power of Reynolds number as Reynolds number decreases and is expressed by

$$C_{L,max1} = A v_{maxt} Re^P, \quad (18)$$

where the constant $A = 4.6$ and $P = -0.37$.

$C_{L,max2}$ is not sensitive to the change of the Reynolds numbers and slowly increase as Reynolds number decreases. When Reynolds number decreases, the effect of wake behind the sphere decreases, and thus the lift coefficient decays slowly to zero after it attains the maximum negative value. $C_{L,rms}$ increases with a negative power of Reynolds number as Reynolds number decreases and is expressed by equation (18) with $A = 3.5$ and $P = -0.35$. The maximum moment coefficient and the rms moment coefficient are affected by only the viscous effect and increases with a negative power of Reynolds number as Reynolds number decreases. $C_{M,max}$ and $C_{M,rms}$ follow the form of equation (18) with $A = 3.26$ and $P = -0.75$ for the former, and $A = 1.43$ and $P = -0.77$ for the latter for $\sigma = 1$.

The effect of Reynolds number ($20 \leq Re \leq 80$) on the flow field is also investigated for the vortex size larger than $\sigma = 1$ ($2 \leq \sigma \leq 4$) and $d_{off} = \pm 1.5$. $C_{L,max1}$ and $C_{L,rms}$ are linearly proportional only to v_{maxt} and independent of σ when $\sigma \geq 2$ at fixed Reynolds number as in the case of $Re = 100$. $C_{L,max1}$ and $C_{L,rms}$ increase with a negative power of Reynolds number as Reynolds number decreases and follow the form of equation (18) with $A = 8.9$ and $P = -0.45$ for the former, and $A = 8.1$ and $P = -0.45$ for the latter for $2 \leq \sigma \leq 4$. $C_{M,max}$ and $C_{M,rms}$ also increase with a negative power of Reynolds number as Reynolds number decreases and follow the form of equation (18) with $A = 5.5$ and $P = -0.83$ for the former, and $A = 3.1$ and $P = -0.88$ for the latter for $2 \leq \sigma \leq 4$.

Now, the effect of the offset distance is determined for $20 \leq Re \leq 80$, in addition to the base case $Re = 100$.

The triangular symbols in Figure 8 show $C_{L,rms}$ as a function of $|d_{off}|$ for $Re = 20$ and $\sigma = 4$ while the maximum fluctuation velocity (or the circulation) of each vortex tube is kept as a constant, $v_{max} = 0.2$. The figure shows that $C_{L,rms}$ decays fast as $|d_{off}| > 0$ for $v_{max} = 0.2$. On the other hand, the circular symbols in Figure 8 show $C_{L,rms}$ as a function of $|d_{off}|$ for $Re = 20$ and $\sigma = 4$ while the total maximum fluctuation velocity induced by the two vortex tubes is kept as a constant, $v_{maxt} = 0.2$. They also show that the magnitudes of the rms lift coefficients for $d_{off} = \pm 2$ and ± 4 are close to that for $d_{off} = 0$. The results for the range of σ values indicate that the equation of the rms lift coefficient for a single vortex tube can be applicable to that for a pair of vortex tubes when the separation distance between their centers is less than $2\sqrt{\sigma}$ vortex tube diameter for

$Re = 20$ if v_{maxt} is used instead of v_{max} in the equation of the rms lift coefficient. The same result as above was obtained at different Reynolds numbers, $Re = 40, 60$, and 80 . The behaviors of $C_{L,max1}$ and $C_{L,max2}$ as a function of $|d_{off}|$ are similar to that of $C_{L,rms}$.

The magnitude of the rms moment coefficient as a function of $|d_{off}|$ decays faster than that of the rms lift coefficient as d_{off} becomes large, and the rms moment coefficient for a single vortex tube can be applicable to that for a pair of vortex tubes when the separation distance between their centers is less than $\sqrt{\sigma}$ vortex tube diameter if v_{maxt} is used instead of v_{max} in the equation of the rms moment coefficient. The behavior of $C_{M,max}$ as a function of $|d_{off}|$ is similar to that of $C_{M,rms}$.

In summary, comparison of the results from this section with those from the previous section shows that the range of the offset distance for which the equations for a single vortex tube can be applicable to those for a pair of vortex tubes does not change as a function of Reynolds number for $20 \leq Re \leq 100$. $C_{L,rms}$ and $C_{M,rms}$ for a single vortex tube which were obtained by Kim *et al.* are expressed by equation (18) with $A = 8.1$ and $P = -0.45$ for the former, and $A = 3.1$ and $P = -0.88$. The rms lift coefficient is written again here for later use.

$$C_{L,rms} = 8.1 v_{maxt} Re^{-0.45}, \quad 2 \leq \sigma \leq 4 \quad (19)$$

3.3 Interactions of a pair of vortex tubes of opposite rotation and a sphere

We consider the same initial flow geometry and parameters as those in section 3.2 but for a pair of vortex tubes of opposite rotation. The base case calculation is that of $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$. Note that the lift and torque on the sphere are zero due to the flow symmetry in upper and lower regions of the sphere

3.3.1 Tubes of top-positive and bottom-negative circulations

Figure 9 shows the drag coefficients of the sphere as a function of time for $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$. The drag coefficients are obtained with four different total maximum fluctuation velocities due to the vortex tubes, $v_{maxt} = 0.185, 0.369, 0.554$, and 0.738 ($v_{max} = 0.1, 0.2, 0.3$, and 0.4). The temporal behavior of the drag coefficients is different from that of the case of the pair of vortex tubes of like rotation. The time-averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of v_{maxt} is not negligible and increased linearly as v_{maxt} increases. When the top and bottom vortex tubes have positive and negative circulations, respectively, the induced velocity due to the vortex tubes adds its magnitude to the base flow along the stagnation streamline, and so the dynamic pressure ahead of

the front stagnation point becomes higher than that of the axisymmetric flow past a sphere. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be higher than those of the axisymmetric flow past a sphere. As a consequence, the drag is increased.

3.3.2 Tubes of top-negative and bottom-positive circulations

Figure 10 shows the drag coefficients of the sphere as a function of time for the same parameters as used in section 3.3.1. The drag coefficients are obtained with four different total maximum fluctuation velocities due to the vortex tubes, $v_{max,t} = 0.185, 0.369, 0.554,$ and 0.738 ($v_{max} = 0.1, 0.2, 0.3,$ and 0.4). The temporal behavior of the drag coefficients is different from that of the case of the pair of vortex tubes of like rotation. The time averaged value of the deviation of the drag coefficient from that of the axisymmetric flow past a sphere for all values of $v_{max,t}$ is not negligible and decreased linearly as $v_{max,t}$ increases. When the top and bottom vortex tubes have negative and positive circulations, respectively, the induced velocity due to the vortex tubes subtracts its magnitude from the base flow along the stagnation streamline, and so the dynamic pressure ahead of the front stagnation point becomes lower than that of the axisymmetric flow past a sphere. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be lower than those of the axisymmetric flow past a sphere. As a consequence, the drag is decreased.

4. Conclusions

In order to understand better the physics of interaction between a particle and the turbulent carrier flow, the unsteady, three-dimensional, incompressible, viscous flow interactions between a pair of vortex tubes advected by a uniform free stream and a spherical particle suddenly placed and held fixed in space is investigated numerically for a range of particle Reynolds number $20 \leq Re \leq 100$. The counter-clockwise rotating a pair of cylindrical vortex tubes are initially located ten radii upstream from the center of the sphere.

A summary of the findings and their applications is provided as follow.

(i) The effects of the size and the offset distance of the pair of vortex tubes on the flow field are examined for $20 \leq Re \leq 100$. The lift and moment coefficients are found to be linearly proportional to the maximum fluctuation velocity ($v_{max,t}$) induced by the pair of vortex tubes at given size (σ) and offset distance of the vortex tube, and the rms lift coefficient depends only on $v_{max,t}$ but independent of σ when $\sigma \geq 2$. Furthermore, the equations for the lift and moment coefficients of the sphere for a single vortex tube are applicable to those for a pair of vortex tubes when the separation

distance between their centers is less than $2\sqrt{\sigma}$ vortex tube diameter for the lift coefficients and less than $\sqrt{\sigma}$ vortex tube diameter for the moment coefficients if $v_{max,t}$ is used instead of v_{max} in the equations. These separation distances do not vary for a range of Reynolds number $20 \leq Re \leq 100$.

(ii) The results in (i) can be applied to turbulent flows containing small concentration of particles in order to obtain the rms lift force on a particle. A turbulent flow possesses a wide spectrum of eddy sizes. The large eddies contain most of the turbulent kinetic energy and produce high velocity fluctuations, and so they are responsible for the dispersion of particles. The particle size, at the extremes, may be comparable to either the integral length scale or to the Kolmogorov length scale. When the size of particle is comparable to the integral length scale, the rms lift coefficient of the particle is obtained by equation (19). Furthermore, the results tend to support the idea that equation (19) would be applicable to the case of an eddy being much larger than the particle. Thus, when the size of particle is comparable to the Kolmogorov length scale, the rms lift coefficient of the particle can be obtained approximately by equation (19), where $v_{max,t}$ is the maximum velocity fluctuation induced by eddies with the integral length scale. The time during which the particle is influenced by the eddy is of the order of the eddy lift time.

The deflection of the particle path will depend on the magnitude of the rms lift coefficient and the ratio, ρ_r , of the particle density to that of the carrier fluid ($C_L = \frac{8}{3}\rho_r A$, where A is the dimensionless acceleration of the particle due to the lift force). This result provides a simple method to estimate the deflection of particle trajectory in the dilute particle-laden turbulent flow. Equation (19) and the nondimensionalized Newton's second law show that the deflection increases slowly as Reynolds number decreases.

(iii) The magnitude of the rms moment coefficient of the particle is one order of magnitude less than that of the rms lift coefficient when $Re \geq 20$. Furthermore, when the initial size of the vortex core is considerably larger than the sphere size ($\sigma \geq 4$), the effect of the shear flow (induced by the passage of the vortex tube) across the sphere diminishes and the torque on the particle decreases. Thus, the torque on the particle might be negligible in many applications.

(iv) When the top and bottom vortex tubes have positive and negative circulations, respectively, the induced velocity due to the vortex tubes adds its magnitude to the base flow along the stagnation streamline. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be higher than those of the axisymmetric flow past a sphere. As a consequence, the drag is increased. On the other hand, When the top and bottom vortex tubes have negative and positive circulations, respectively, the induced velocity due to the vortex tubes

subtracts its magnitude from the base flow along the stagnation streamline. This causes the pressure at the stagnation point and the shear stresses in the upper and lower left regions to be lower than those of the axisymmetric flow past a sphere. As a consequence, the drag is decreased.

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References

1. Lumley, J. L. 1978 Two-phase and non-Newtonian flows. In P. Bradshaw, editor, *Turbulence* 289-331, Springer-Verlag, second edition.
2. Crowe, C. T. 1982 REVIEW - Numerical models for dilute gas-particle flows. *J. Fluids Eng.* 104 297-303.
3. Faeth, G. M. 1987 Mixing, transport and combustion in sprays. *Prog. in Energy and Combustion Sci.* 13(4) 293-345.
4. Kim, I., Elghobashi S. & Sirignano, W. A. 1995 Unsteady flow interactions between an advected cylindrical vortex tube and a spherical particle. *J. Fluid Mech.* in press.
5. Vinokur, M 1983 On one-dimensional stretching functions for finite-difference calculations. *J. Comput. Phys.* 50, 215-234.
6. Spalart, P. R. 1982 Numerical simulation of separated flows. Ph. D. Thesis, Stanford University.
7. Patnaik, G. 1986 A numerical solution of droplet vaporization with convection. Ph.D. Dissertation, Carnegie-Mellon University.

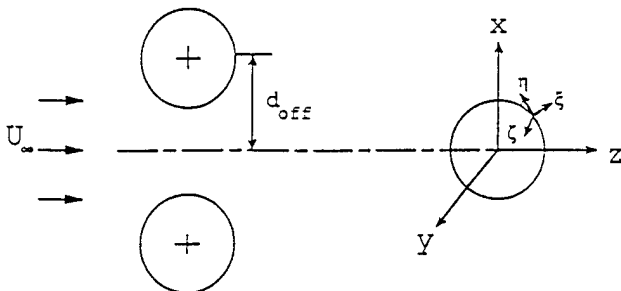


Figure 1. Flow geometry and coordinates

σ	$C_{L,max1}$	$C_{L,max2}$	$C_{L,rms}$	$C_{M,max}$	$C_{M,rms}$
4	0.111	-0.186	0.103	0.011	0.0051
3	0.111	-0.196	0.102	0.013	0.0053
2	0.108	-0.197	0.094	0.013	0.0053
1	0.088	-0.162	0.070	0.011	0.0043
0.5	0.058	-0.116	0.046	0.0065	0.0023
0.25	0.034	-0.070	0.027	0.0034	0.0012

Table 1. Maximum positive and negative lift coefficients, rms lift coefficient, maximum moment coefficient, and rms moment coefficient as a function of the size of vortex tube for $Re = 100$ and $d_{off} = 1.5$ with $v_{max} = 0.1$.

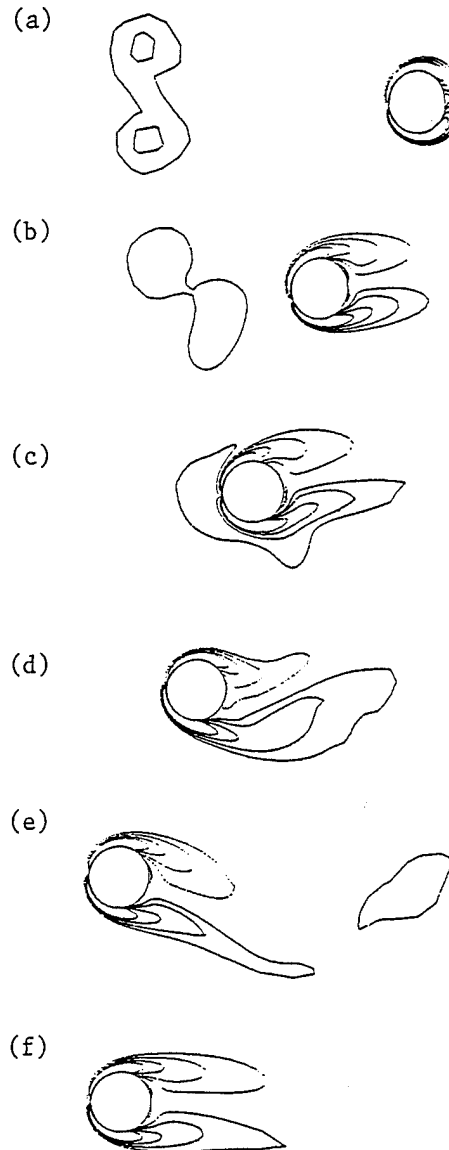


Figure 2. Contour lines of y-component vorticity in the principal plane at (a) $t = 1$, (b) 6, (c) 10, (d) 15, (e) 21, and (f) 30 for $Re = 100$, $d_{off} = 0$ and $\sigma = 1$ with $v_{max} = 0.59$.

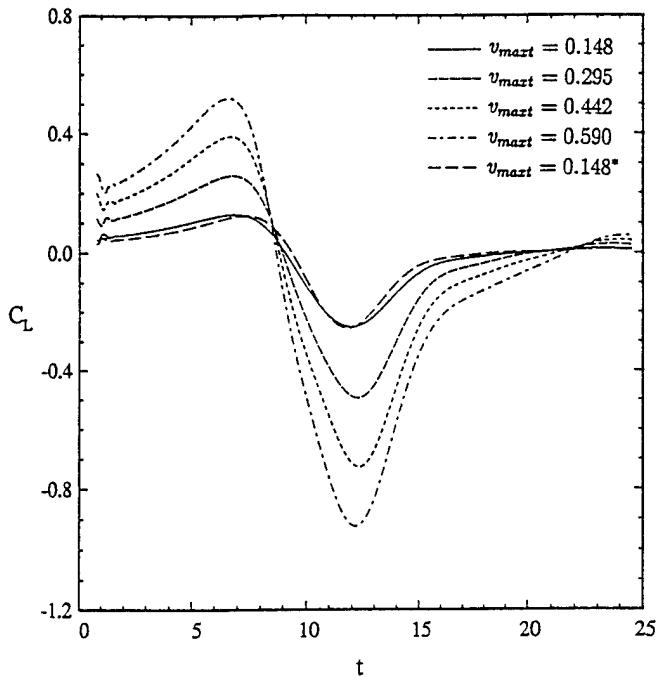


Figure 3. Lift coefficients of the sphere as a function of time and v_{max} for $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$. * denotes the case of a single vortex.

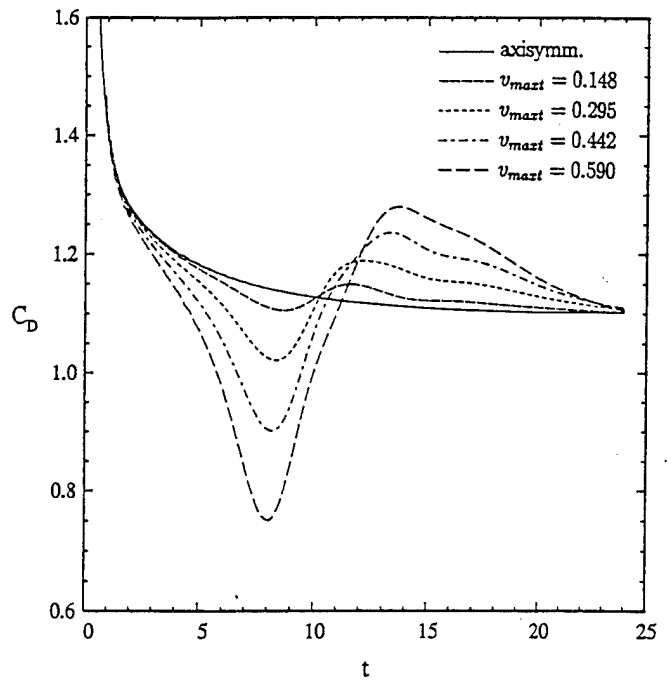


Figure 5. Drag coefficients of the sphere under the same conditions of figure 3.

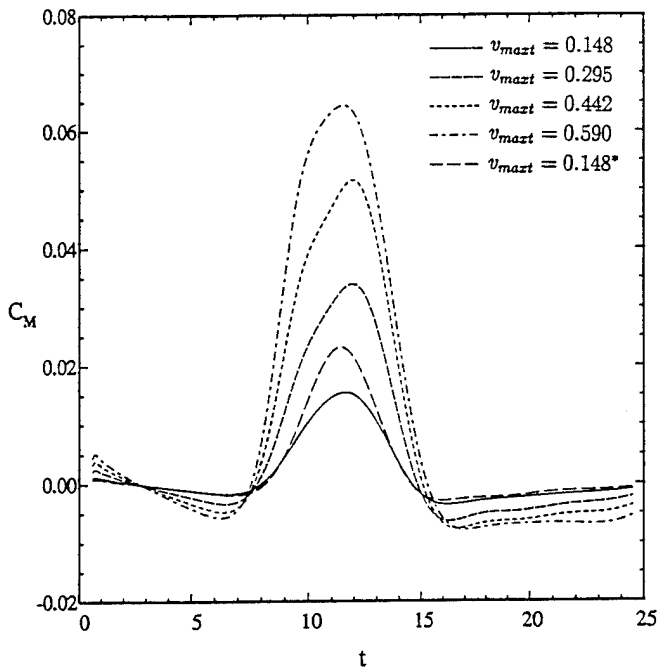


Figure 4. Moment coefficients of the sphere under the same conditions of figure 3. * denotes the case of a single vortex.

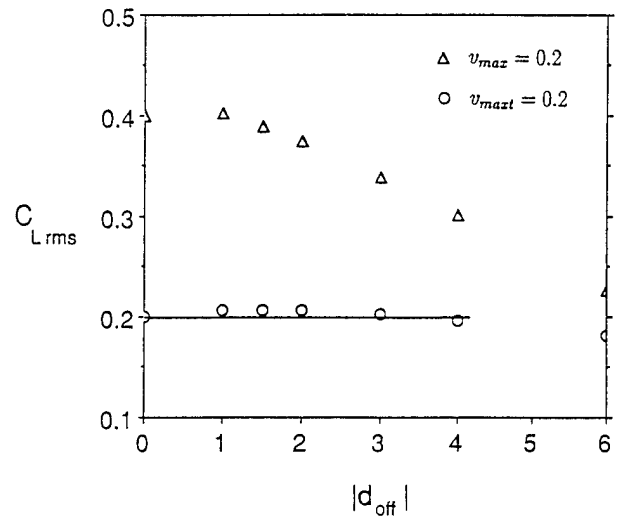


Figure 6. Rms lift coefficients of the sphere as a function of $|d_{off}|$ for $Re = 100$ and $\sigma = 4$.

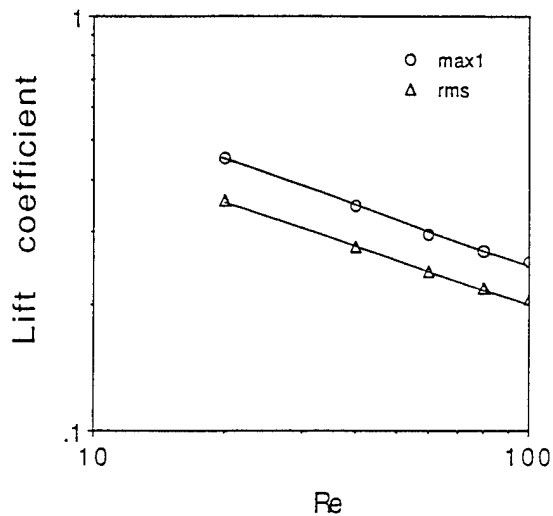


Figure 7. Maximum positive lift coefficient and rms lift coefficient as a function of Reynolds number for $d_{off} = \pm 1.5$ and $\sigma = 1$ with $v_{maxt} = 0.295$.

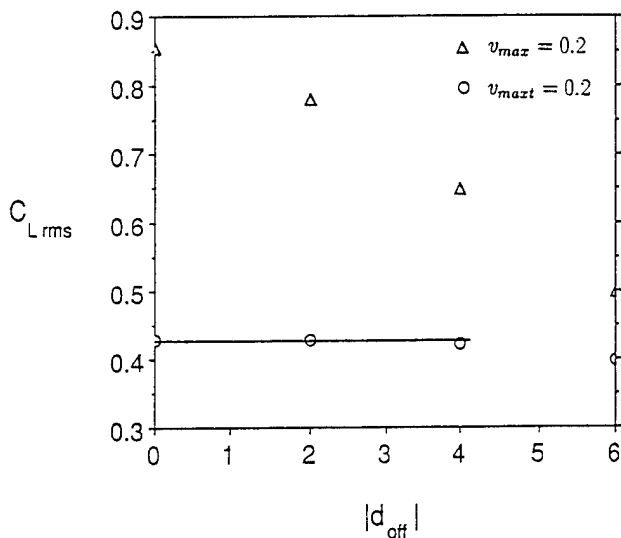


Figure 8. Rms lift coefficients of the sphere as a function of $|d_{off}|$ for $Re = 20$ and $\sigma = 4$.

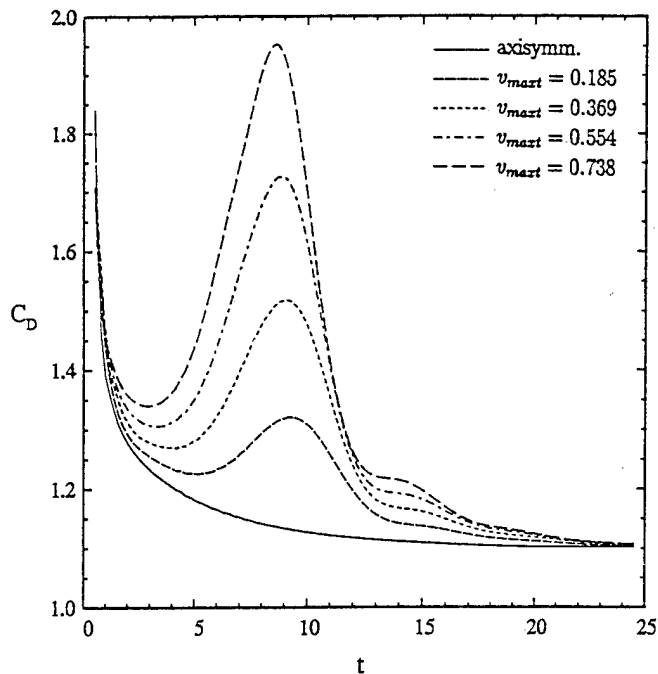


Figure 9. Drag coefficients of the sphere as a function of time and v_{maxt} for $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$ with top-positive and bottom-negative circulations.

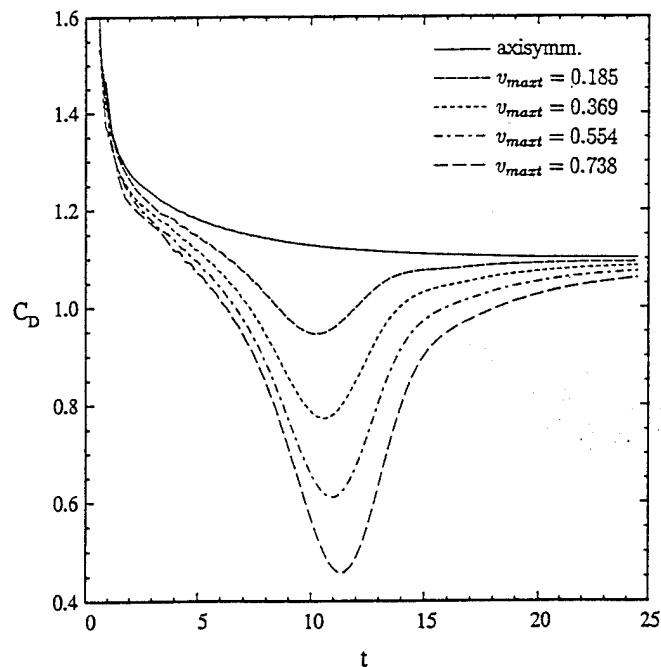


Figure 10. Drag coefficients of the sphere as a function of time and v_{maxt} for $Re = 100$, $d_{off} = \pm 1.5$, and $\sigma = 1$ with top-negative and bottom-positive circulations.