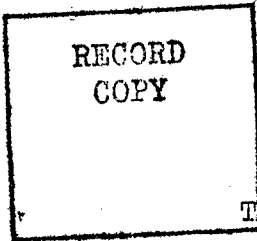


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THE ROLE OF ROUGHNESS IN RUBBER FRICTION

AND THE LAW OF FRICTION

By S. B. Ratner

- USSR -

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THE ROLE OF ROUGHNESS IN RUBBER FRICTION  
AND THE LAW OF FRICTION

-USSR-

[Following is the translation of an article entitled "O role sherckhovatosti pri trenii reziny i o zakone treniya" (English version above) by S. B. Ratner in Doklady Akademii Nauk SSSR (Reports of the Academy of Sciences USSR), Vol XCIII, No 1, Moscow, 1953, pages 47-50.]

(Submitted to Academician P. A. Rebinder, 22 August 1953)

In accordance with the theory of B.V. Deryagin (1), the force of friction depends on normal load  $N$  as per formula

$$F_1 = \mu_1(N + N_0) \quad (1)$$

where  $\mu_1$  is a constant (true) coefficient of friction corresponding to the median value of the angle of micro-elevation, which "depends on the molecular-atomic coarseness of surface" (page 167);  $N_0$  - "the resultant of forces of molecular attraction between both bodies" (page 177), proportional to area of true contact of bodies in friction.

Quantity  $N_0$  must be added to the external load  $N$  in order to take into account the whole of the normal force active on the surfaces of true contact, where the micro-coarseness of bodies in friction is of consequence.

Deryagin's molecular theory correctly describes the role of those forces which come into being on the surfaces of true contact. But it does not take into consideration the role of the direct mechanical traction connected with rough (common) coarseness. However, such micro-coarseness is bound to lead to the additional component

of the friction force

$$F_2 = \mu_2 N, \quad (2)$$

where  $\mu_2$  is the friction coefficient corresponding to the median angle of macro-elevation, which is connected with the rough coarseness (2). This alteration of the area of true contact, called forth [brought about] by changed macro-coarseness, will have a manifest bearing on the variation of quantity  $N_0$  (which may depend on  $N$  as well).

Thus, the full force of friction

$$F = (\mu_1 + \mu_2)N + \mu_1 N_0 \quad (3)$$

$$F = \mu_\infty N + F_0. \quad (4)$$

where\*

$$\mu_\infty = \mu_1 + \mu_2. \quad (5)$$

$$F_0 = \mu_1 N_0. \quad (6)$$

Let us see to what extent this agrees with the experimental findings, the presence of two members in the law of friction (4) being beyond argument in all of the cases (3-7) and only the question of their connecting interrelationship to be considered. The following formula may be employed in this,

$$\mu = \mu_\infty + \frac{F_0}{N}. \quad (7)$$

\* If we consider the effect of macro-coarseness not only on the area of contact (i.e. on  $N_0$ , see above) but also on micro-coarseness (i.e. on  $\mu_1$ ) -- at the expense of the wedging action and the necessity of mutual interaction of contiguous macroprojections, then instead of  $\mu_2$  we may put  $\mu_3 = \mu_2 \mu_1$  in our formulae. Then  $F = \mu_1 [(1 + \mu_2)N + N_0]$  because  $\mu_\infty = \mu_2 (1 + \mu_2)$ . This result, obtained by us jointly with B.V. Deryagin during discussion of this article, will not alter subsequent conclusions which have a qualitative character.

which is the result of substitution (4) in the formula that serves for the determination of the computational coefficient of friction

$$\mu = \frac{F}{N}. \quad (3)$$

In order to prove directly and quantitatively the validity of formula (3) presented by us earlier (3) in the form

$$F = \mu_{\infty} N + \mu_1 N_0. \quad (9)$$

it is necessary to measure  $\mu_{\infty}$  and  $\mu_1$ , and to show that  $\mu_{\infty} > \mu_1$ . Definition of  $\mu_{\infty}$  does not present any difficulties, because it can be discovered as a characteristic of the straight line in the processing of experimental data either in the coordinates  $F - N$ , per formula (4) or in coordinates  $\mu - 1/N$ , according to formula (7).

As a second characteristic of the straight line, quantity  $F_0$  can also be determined. However, knowing  $F_0$ , one can arrive at  $\mu_1$  by formula (7) only if adhesion force  $N_0$  is directly measured under exactly the same conditions under which  $F_0$  was determined. This measurement, which can be conducted by employing the method of crossing lines, presents specific difficulties (4). Therefore we shall turn to qualitative testing of validity of formulae (3) and (9).

In accordance with the presented concepts, increased coarseness of metal must have a dual effect upon the friction coefficient. On one hand,  $\mu_2$  is supposed to increase, on the other - at the expense of diminishing contact -  $N_0$  must diminish.

This effect is essential mainly for soft rubbers, especially at small loads (7); see formula (7). Therefore, when coarseness of steel is increased, we observe (see Table 1) at specific unit load  $P = 0.1 \text{ kgm/cm}^2$  a diminution of  $\mu$  at the expense of decrease of  $N_0$ , and, at  $P = 10 \text{ kgm/cm}^2$ , an increase in  $\mu$  at the expense of increased mechanical traction (i.e.  $\mu_2$ ), which becomes especially apparent when the hardness of rubber is increased. Certain combinations of hardness of rubber and coarseness of steel permit an observation of tendency towards growth on the part of the friction coefficient when load is increased

(see Table 1). An analogous picture appears also when the coarseness of brass and aluminum is increased.

Further, if formula (1) were applicable in the capacity of law of friction, i.e. if it could endure the whole of the force of friction (and not its part), then  $\mu_2 \cong 0, \mu_{\infty} \cong \mu_1$ , and consequently, there must be a strict proportionality between  $F_0$  and  $\mu_{\infty}$  in accordance with formulae (5) and (6). This can be expected in instances where macro-coarseness is very small - friction with lubrication (stearic acid or oleinic acid) of paraffin against glass (5), the friction of smooth criss-crossing threads of processed quartz - uncoated as well as covered by a thin coat of caoutchouc (4).

Such phenomenon should not be observable at "dry" friction of rubber, when  $\mu_2$  cannot be neglected in comparison with  $\mu_1$ . In instances of friction between identical samples of rubber containing merely different quantities of various fillers, the particles of which become coated with a film of caoutchouc, quantity remained practically constant (7) because  $\mu_2 = const$ , because macro-coarseness is identical and microrelief changes insignificantly, within the limits of compatibility of caoutchouc with the filler (soot, graphite, chalk, bioxide of silicon).

A change of  $\mu_{\infty}$  can be achieved at the effect of a change in surface of metal (a shift to a different variety or sample) or by introduction of such a large amount of filler that it will already overstep the limits of compatibility with caoutchouc and serve as a layer between the pair in friction. Both of these methods, which must lead to change in  $\mu_{\infty}$ , were actually observed (7).

When  $\mu_{\infty}$  changes due to changed  $\mu_2$ , i.e. switch of metal surface (see also Table 1) or even when a switch to plexiglass was made, a practical constancy of  $F_0$  for rubber with a given amount of filler was observed, which speaks of the dominant role of rubber (as the softer component of the working pair) in the creation of microrelief of the friction surface.

$\mu_{\infty}$  can also be altered at the expense of change of  $\mu_1$ , i.e. of the molecular relief, which can be realized when switching from one caoutchouc to another or from one metal to another (3,7).

Let us, at last, consider the role of temperature. According to the above, variations in the temperature of rubber are not supposed to effect macro-coarse-

Table 1

Coefficient of Static Friction of Rubber on Base  
SKN-26 Against Metals With Different Degree of  
Mechanical Finish of Surface

Soot Content in Parts by Weight	Hardness of Rub- ber According to Shor	Specific Unit Load in Kilograms /cm <sup>2</sup>	Steel			Brass		Aluminum	
			▽▽▽	▽▽	▽	▽▽▽	▽▽	▽▽▽	▽▽
10	52	0.1	0.96	0.81	0.80	1.32	1.18	0.99	0.99
		1.0	0.83	0.78	0.68	0.92	0.84	0.78	0.84
		10.0	0.43	0.48	0.54	0.56	0.49	0.49	0.52
45	71	0.1	0.74	0.61	0.64	0.97	0.95	0.60	0.82
		1.0	0.64	0.61	0.60	0.74	0.68	0.57	0.68
		10.0	0.43	0.59	0.67	0.41	0.43	0.57	0.67
120	96	0.1	0.44	0.61	0.42	0.33	0.72	0.57	1.06
		1.0	0.25	0.37	0.33	0.27	0.38	0.32	0.58
		10.0	0.27	0.40	0.52	0.26	0.51	0.33	0.74

ness  $\mu_{\infty}$ , but is of consequence for microrelief  $\mu_1$ , and possibly also for  $N_0$ , when it comes to friction of rubber, the physico-mechanical qualities of which depend on temperature to a great extent. Insomuch as the heat-movement weakens the action of intermolecular forces determining the quantity  $F_0$ , we must expect that, when the temperature is higher,  $F_0$  will be significantly smaller without an equally consequential decrease of  $\mu_{\infty}$ . The experiment (3) confirms this (see Table 2).

Accepting the following relationship, which flows from relaxational concepts about rubber (8)

$$F_0 = A e^{U/RT} \quad (10)$$

we obtain from Table 2 that  $U \approx 5$  kilocal/mole; this corresponds to the usual values of the energy barrier of

van-der-Waals' forces.

Table 2

Effects of Temperature on the Constants of Formula (4) in the Case of Static Friction of Rubber on Base SKBM (Hardness According to Shor - 90) Against Steel (a) and on Alloy Containing Aluminum (b)

Temperature in °	$\mu_{co}$		$F_0$ in kg/cm <sup>2</sup>	
	a	b	a	b
20	0.36	0.5	16	17
50	0.30	---	10	---
80	0.24	0.3	3	3

Such dependence continues only until further climb to higher temperatures leads to a significant softening of rubber, as a result of which the area of true contact gains sharply, and the computational coefficient of friction  $\mu$  increases. Thus, in friction of rubber on steel at  $P = 1 \text{ kgm/cm}^2$ ,  $\mu = 0.25$  at  $80^\circ$  (Table 2),  $\mu = 0.23$  at  $95^\circ$ , and  $\mu = 0.5$  at  $110^\circ$ , i.e., the curve runs through the minimum.

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16 VII 1953

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