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Advatech Pacific
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Analysis of Coaxial Injectors Using CFD++

Presenter:

Henry Vu, Ph.D.

*This material is based upon work supported by
AFRL/RZSA, AFRL/RZSE, and AFRL/RZST under Contract Nos.
FA9300-06-D-0002 and FA9300-10-C-4002*



Overview

- Company Background
- Current CFD projects
- Coaxial Jet Flow with Variable Density
- Coaxial Particle Laden Flow



Advatech Pacific, Inc. Background



- An Aerospace Engineering Research & Development Company Founded in 1995 primarily focused on:
 - Aerospace Vehicle Physics-based Modeling, Simulation and Analysis
 - Electronic Communications System Interoperability
 - Aerospace Engineering Design and Analysis Services



Contract Objectives

- Objectives

- Modeling and analysis of fluid flows and heat transfer in support of experiments performed at AFRL Edwards
 - Complement experimental data by providing detailed visualization of fluid flow and thermal distributions inside experiments
 - Supplement experimental data by generating data at test conditions not performed in experiments
- Provide independent verification and validation for CFD++ code development



Contract Objectives



● CFD Projects

- Coaxial Particle laden flow dynamics to assist in design of an experimental apparatus
- Coaxial gas/gas injector flow analysis for rocket fuel injector design
- Mixing of jet in cross-flow for pre-burner studies
- Evaluation of pipe flow conditioning devices
- Conjugate heat transfer studies in pipe flow for experimental design.



COAXIAL JET FLOW WITH VARIABLE DENSITY

Collaborators: S. Alexander Schumaker, Ananda Himansu,
Stephen Danczyk, Malissa Lightfoot



Motivation

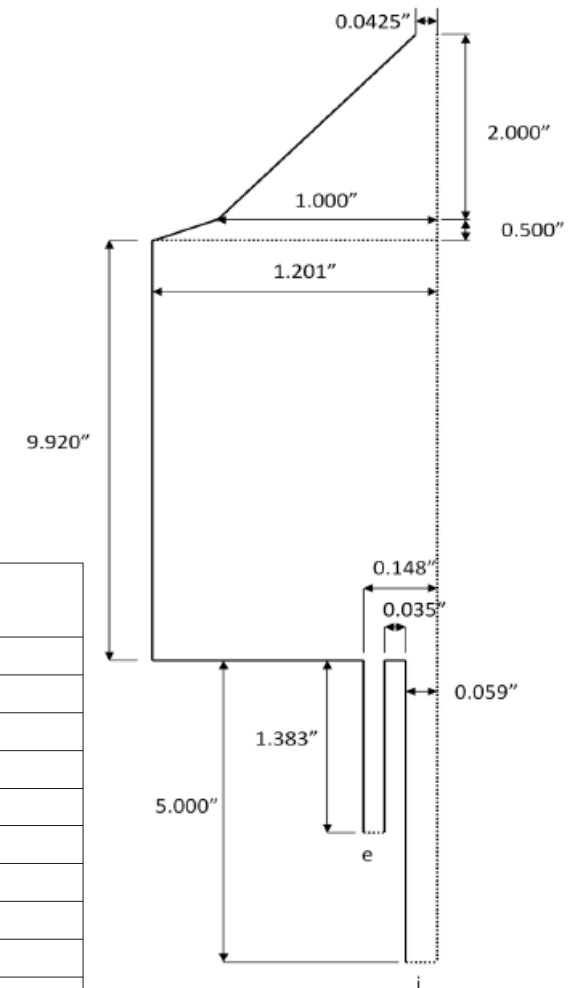
- Liquid propellant rocket engine injectors characterized by low velocity, high density inner jet and high velocity, low density annular jet.
- Conservation equations linked through density variations.
- Need to determine accuracy of RANS predictions in flows with large density differences.
- Validate Metacomp CFD++ tool for relevant flows.



Michigan SEI Case

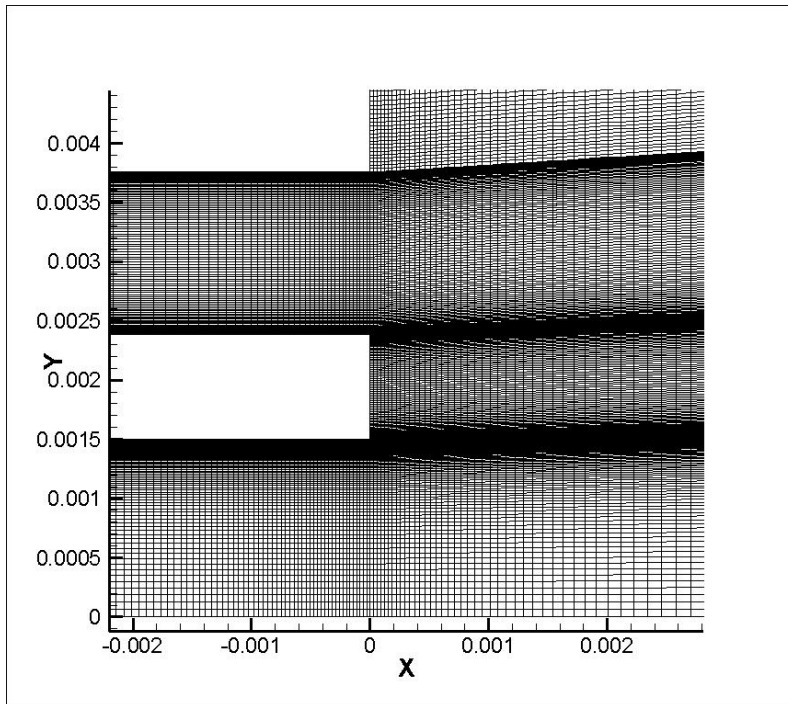
- PLIF concentration data for single element injector
- Low speed flow for ease of modeling
- Confined flow with nozzle at the exit
- Inner fluid (i) is air seeded with acetone
- Outer fluid (e) is helium
- $\rho_i/\rho_e=7.5$

Inner Fluid (i)	Air with 3.35% Acetone by Volume	
Molecular Weight (i)	29.95	Kg/kmol
Molecular Weight (e)	4.003	Kg/kmol
Chamber Pressure	543829.7	Pa
To (e)	293	K
To (i)	293	k
ρ_o (e)	0.894	Kg/m ³
ρ_o (i)	6.685	Kg/m ³
Uavg (e) in tube	63.559	m/s
Uavg (i) in tube	12.587	m/s
Mdot (e)	1.50136E-3	Kg/s
Mdot (i)	5.93253E-4	Kg/s

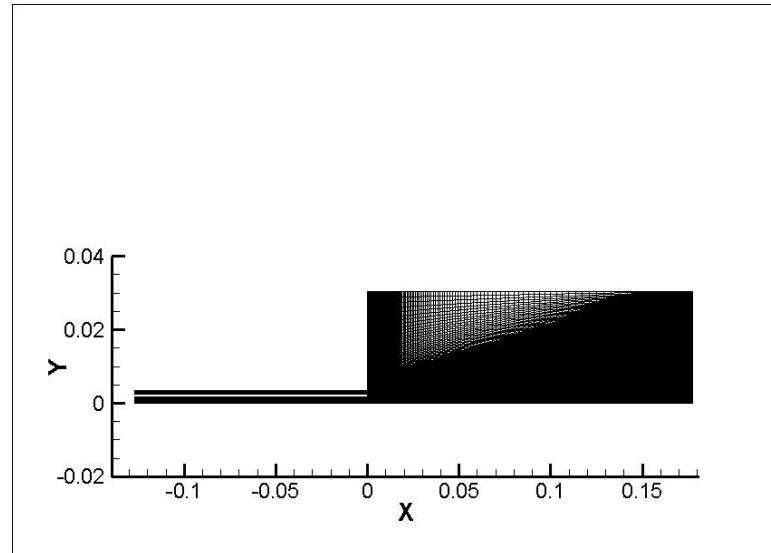




Michigan SEI Case



Mesh:
2D axisymmetric
93386 quadrilateral cells, $y^+=1$



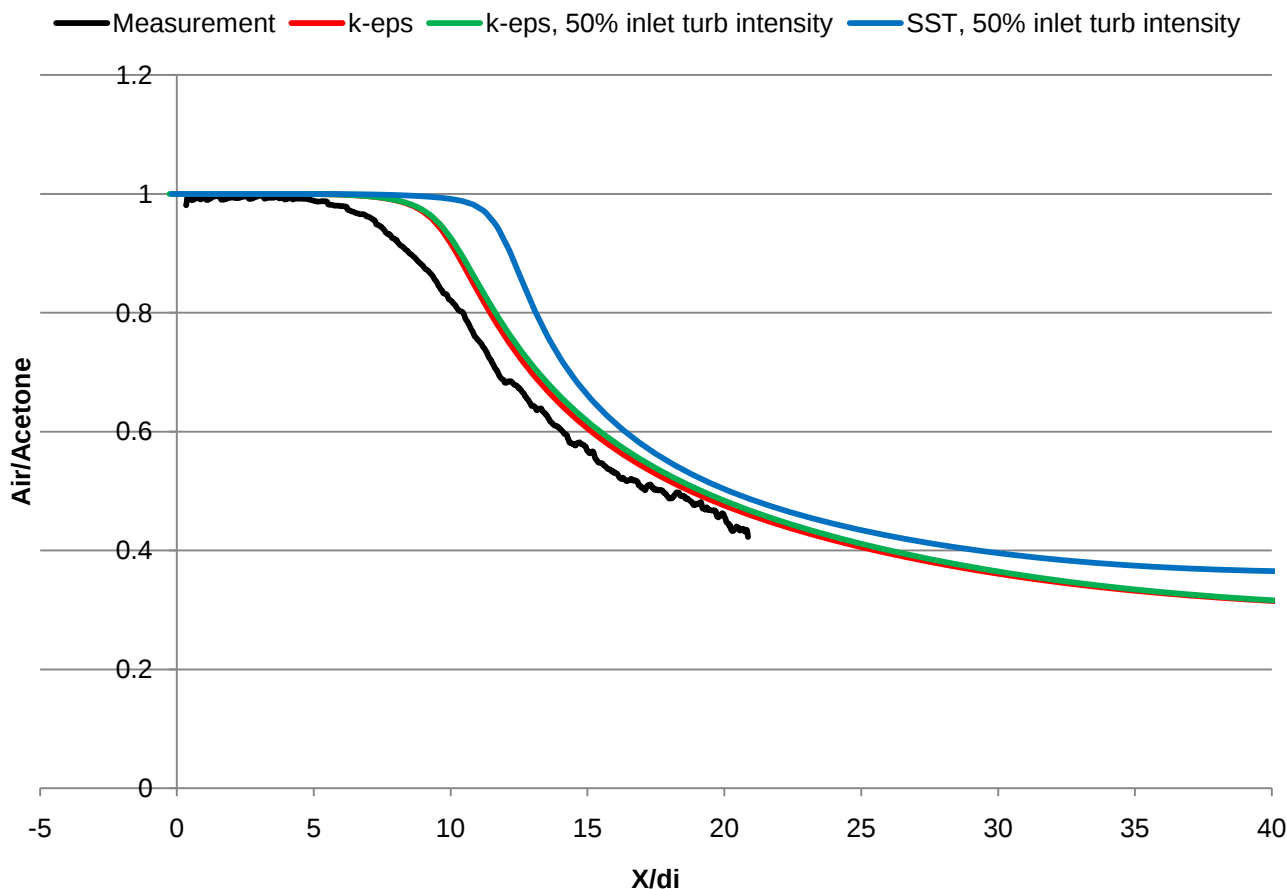
Case Conditions:

- Two species (He, Air/Acetone)
- Base Equation Type: Compressible Real Gas Navier-Stokes/Euler
- Equation of State: Ideal Gas
- Turbulence Simulation: RANS, realizable k-eps or SST
- Turbulence Intensity: 2% or 50%
- nozzle geometry at the exit remove and replaced with back pressure imposition



Michigan SEI Case

Centerline Mass Fraction

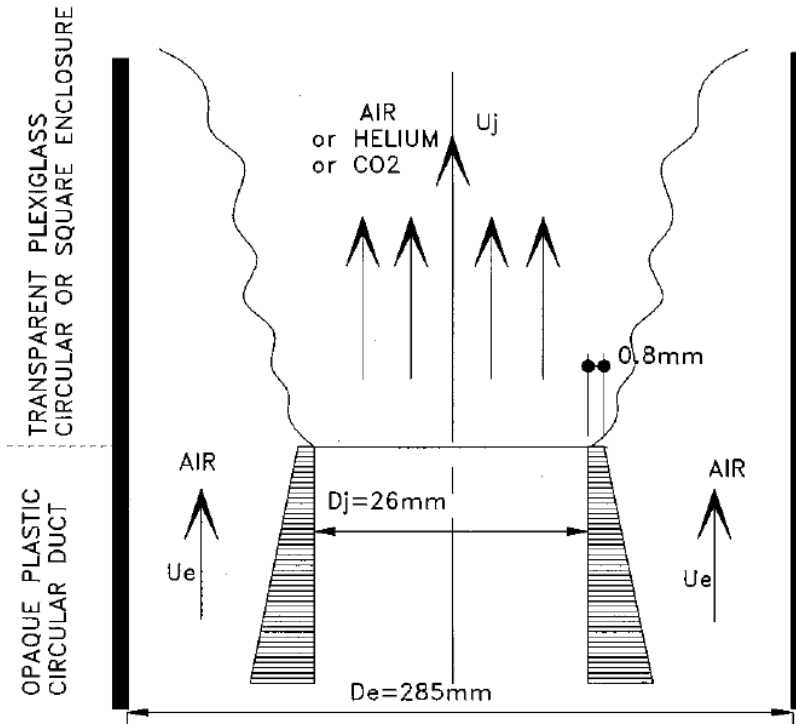


- At least 15% error in concentration along the centerline in the near-field
- No improvement by increase turbulence at inlet
- SST turbulence model makes result worse

Is error due to problem in modeling scalar or velocity field?



Validation Case



- 2D axisymmetric compressible RANS
- Density-based solver
- Realizable k-eps turbulence solved to the wall
- $T=300$ K and $P=101325$ Pa
- Center inlet boundary:
 - He normal velocity = 24.45 m/s
 - Air normal velocity = 10.5 m/s
 - CO_2 normal velocity = 9.0 m/s
- Outer inlet boundary: Air with normal velocity of 0.9 m/s
- Outlet: Simple back pressure outlet of 0 Pa.

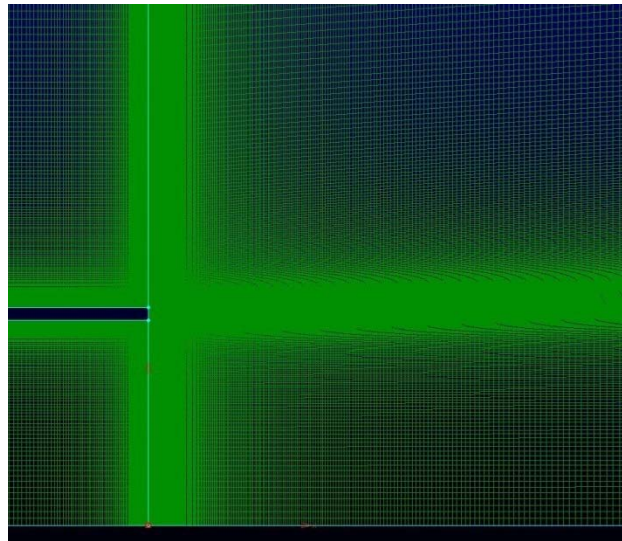
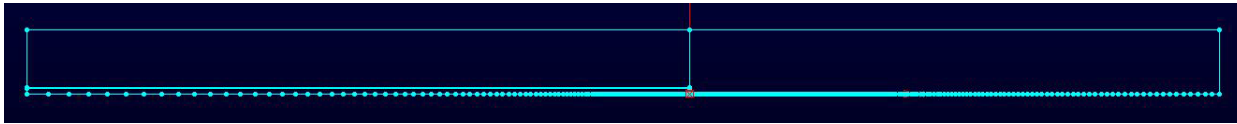
Gas	U_j [m/s] (Expt)	U_j [m/s] (CFD)	Re_j	ρ_j/ρ_e
He	32	31.4	7000	0.14
Air	12	12.68	21000	1
CO_2	10	10.75	32000	1.4

Amielh, M., Djeridane, T., Anselmet, F., & Fulachier, L. (1996). Velocity near-field of variable density turbulent jets. *Int. J. Heat Mass Transfer*, 2149-2164.

Djeridane, T., Amielh, M., Anselmet, F., & Fulachier, L. (1996). Velocity turbulence properties in the near-field region of axisymmetric variable density jets. *Phys. Fluids*, 1614-1630.



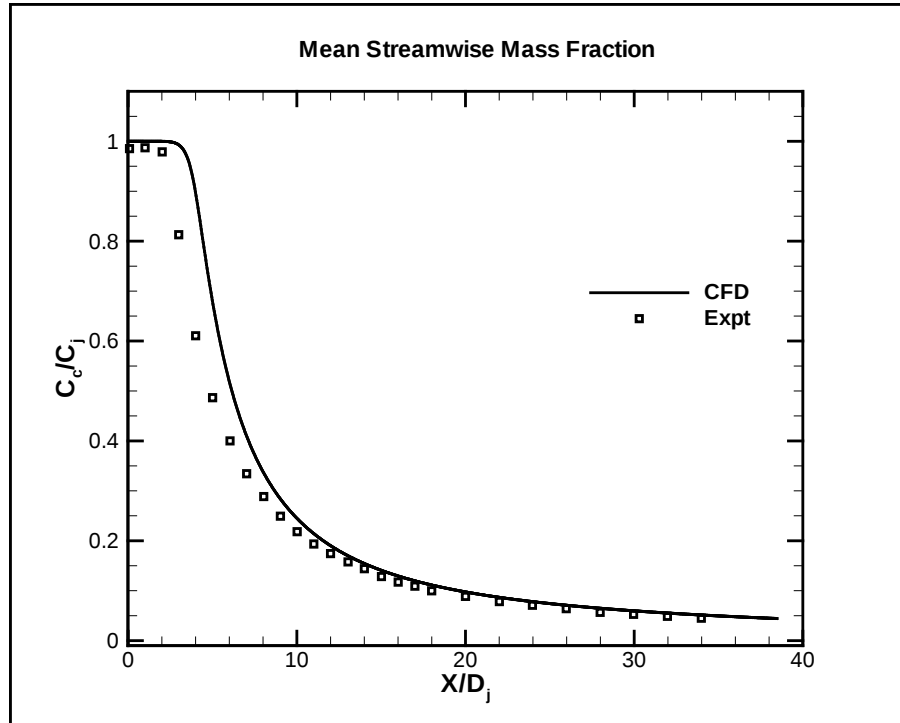
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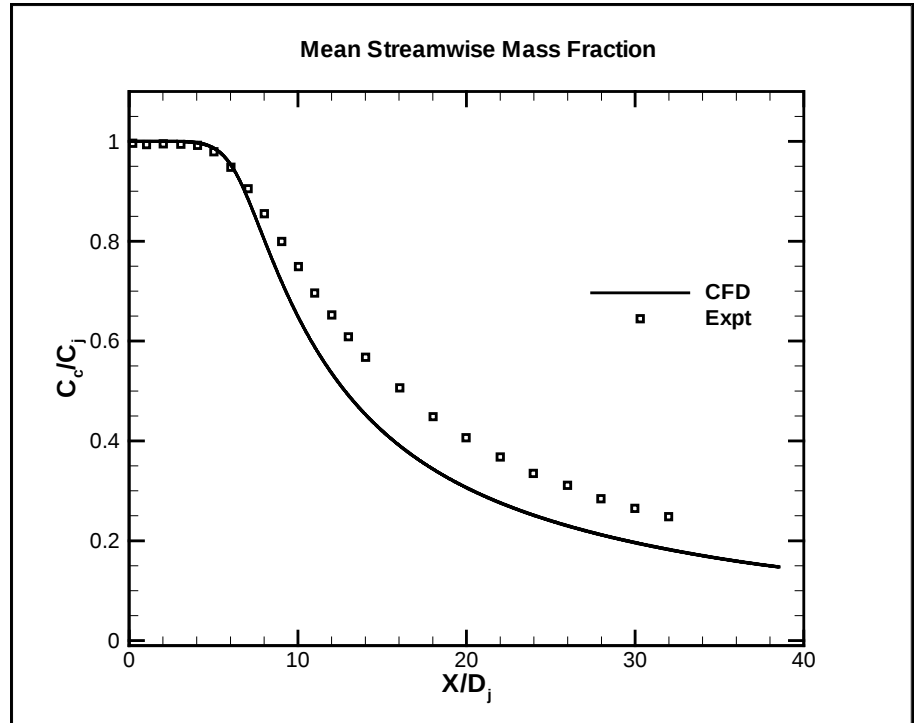
- 234880 quadrilateral cells with a y^+ of 1 or less at all wall locations except the outermost wall.
- Run up length = 1.5 m
- Total domain length = 2.7 m



Validation Case Results



He

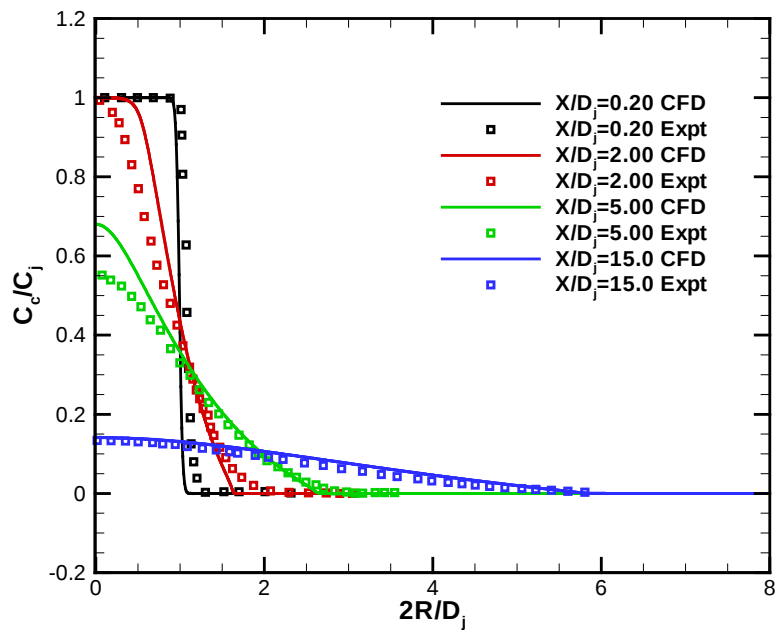


CO₂



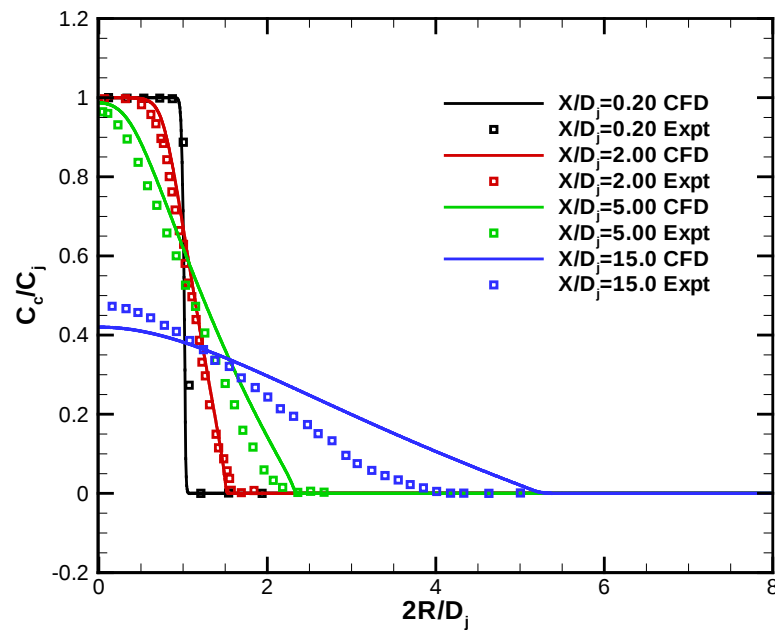
Validation Case Results

Radial Profiles of Mean Mass Fraction



He

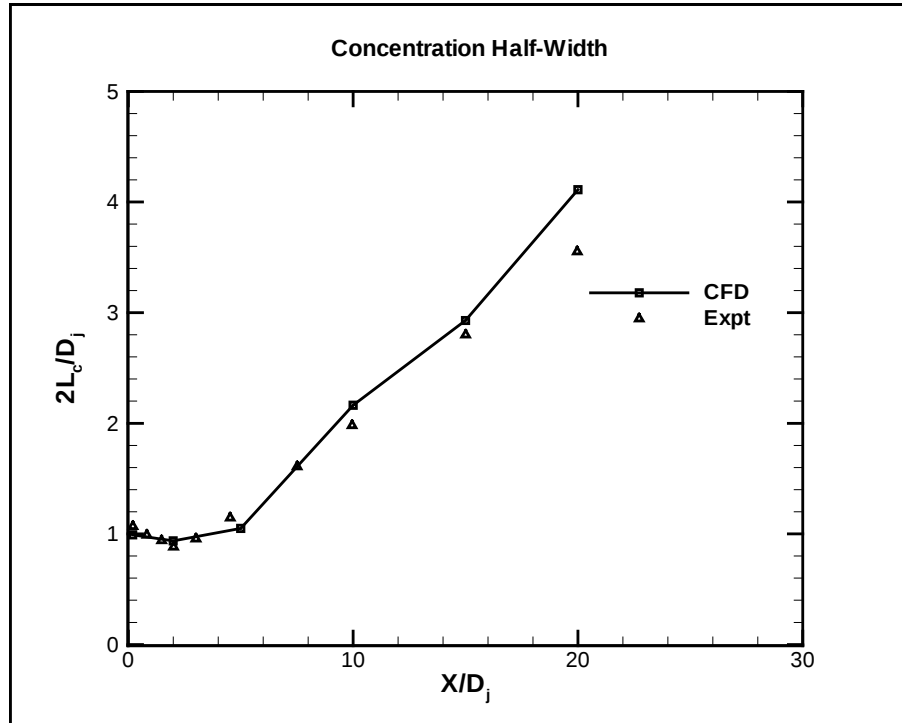
Radial Profiles of Mean Mass Fraction



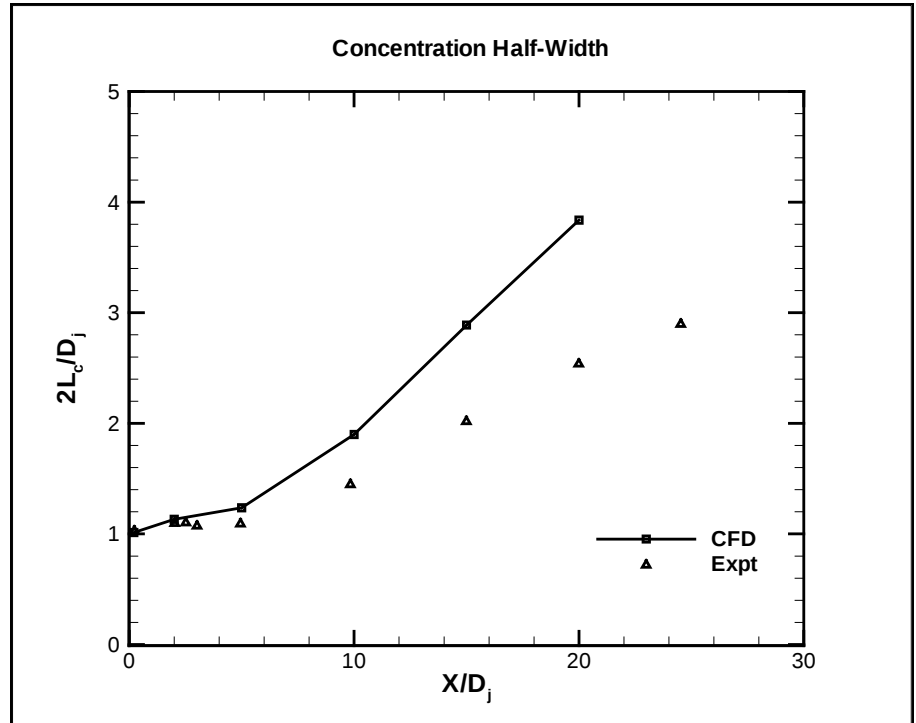
CO₂



Validation Case Results



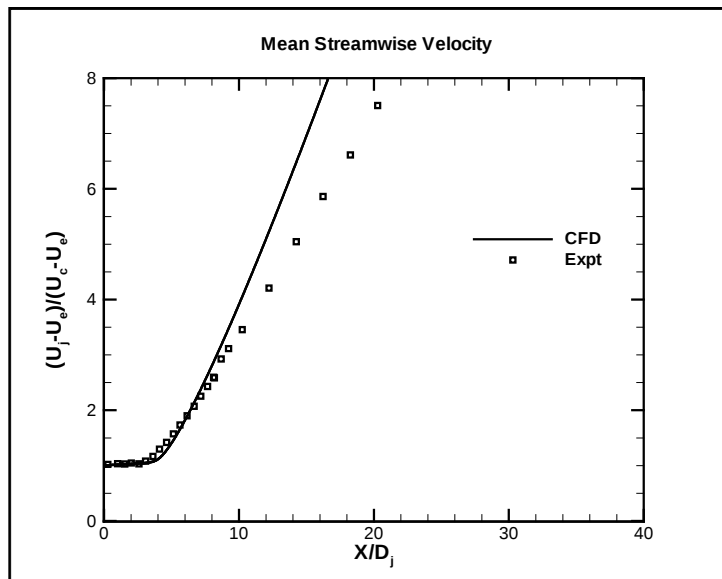
He



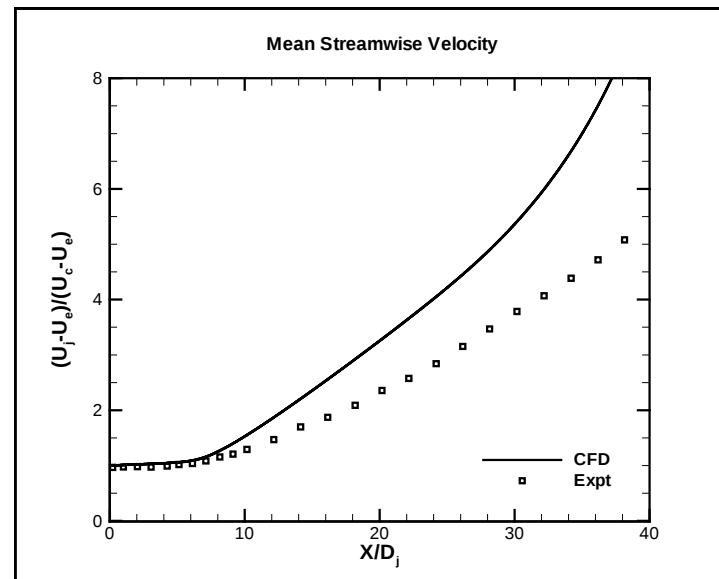
CO₂



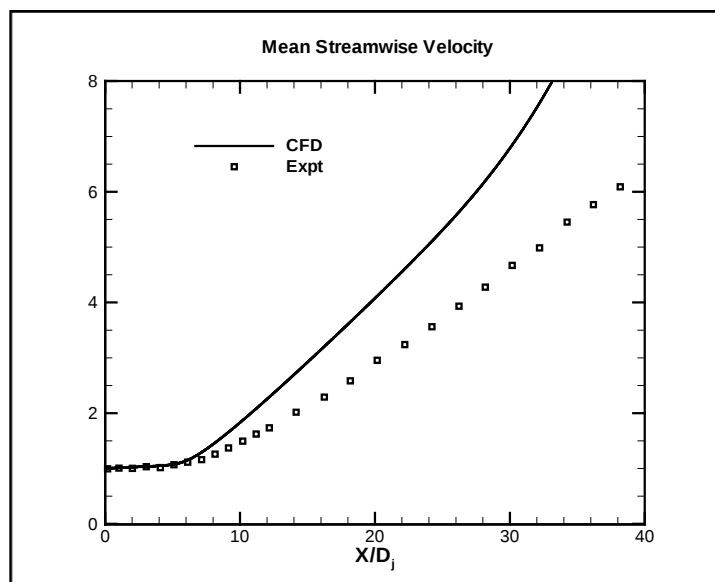
Validation Case Results



He



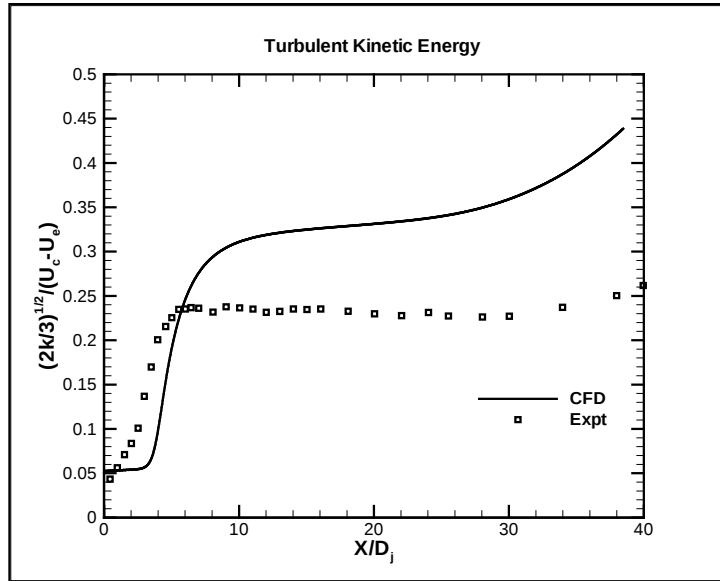
CO₂



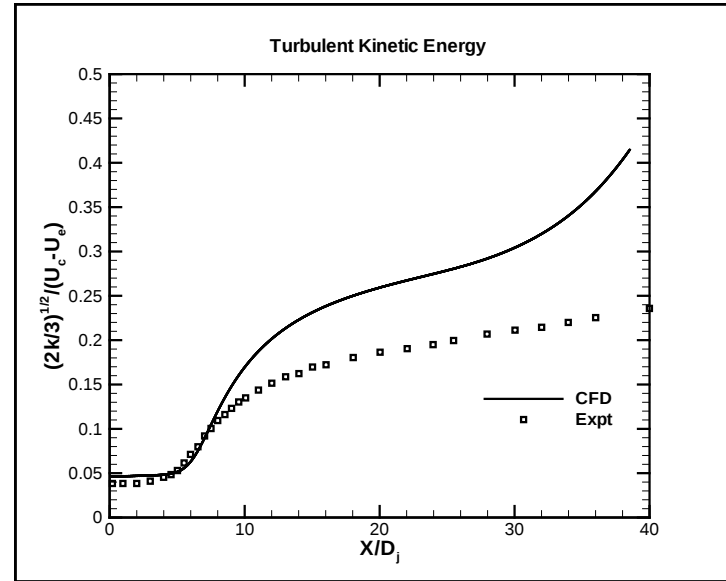
Air



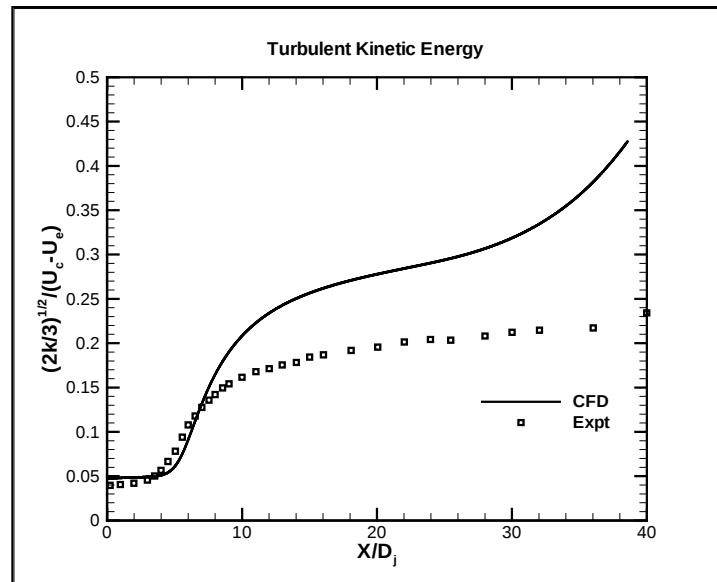
Validation Case Results



He



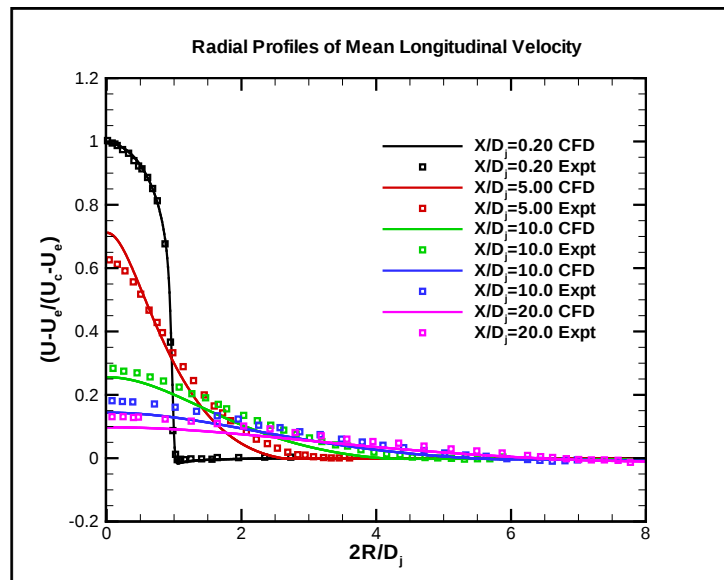
CO₂



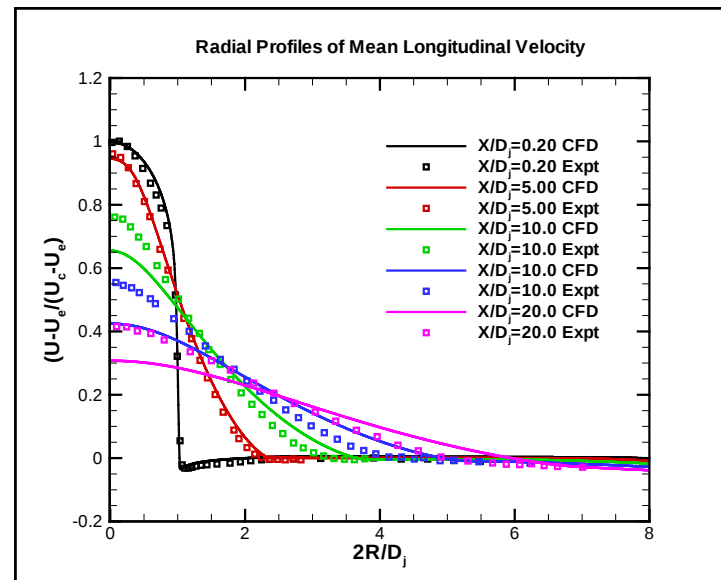
Air



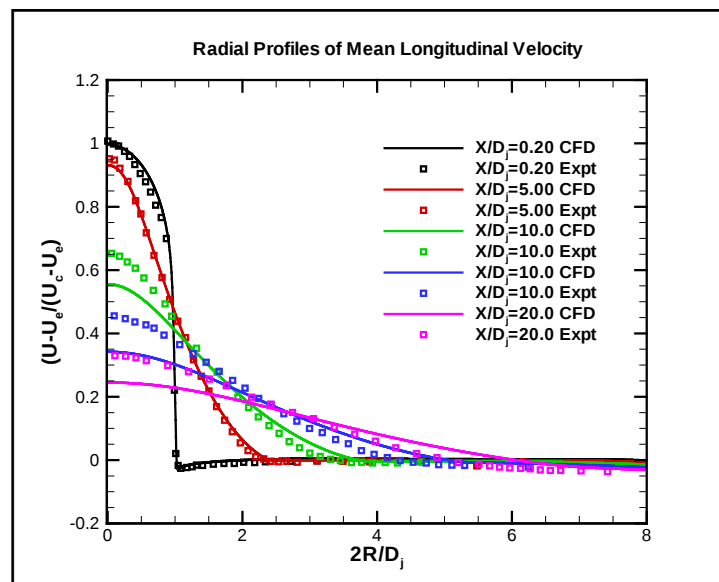
Validation Case Results



He



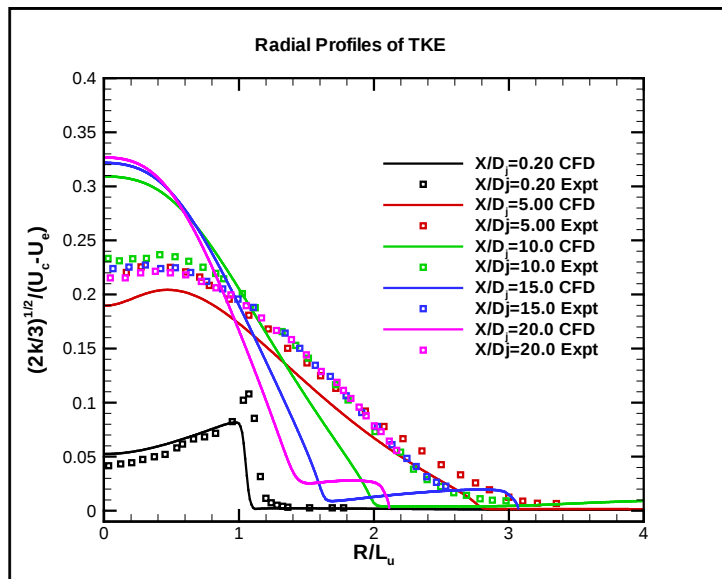
CO₂



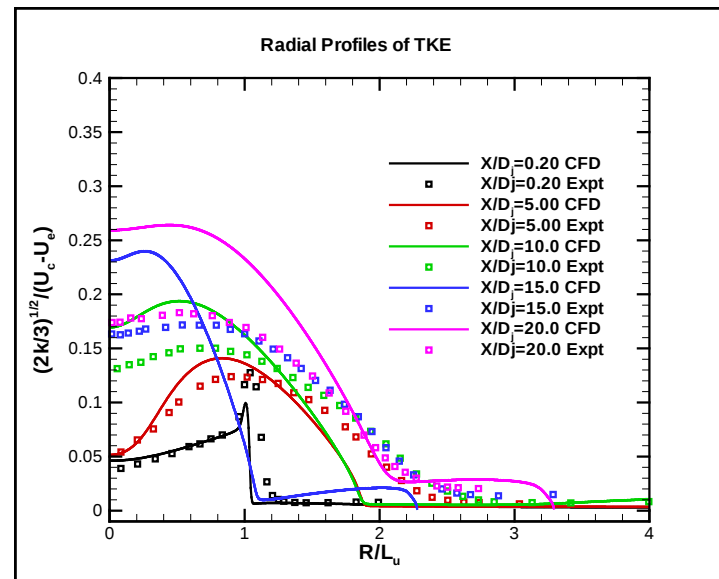
Air



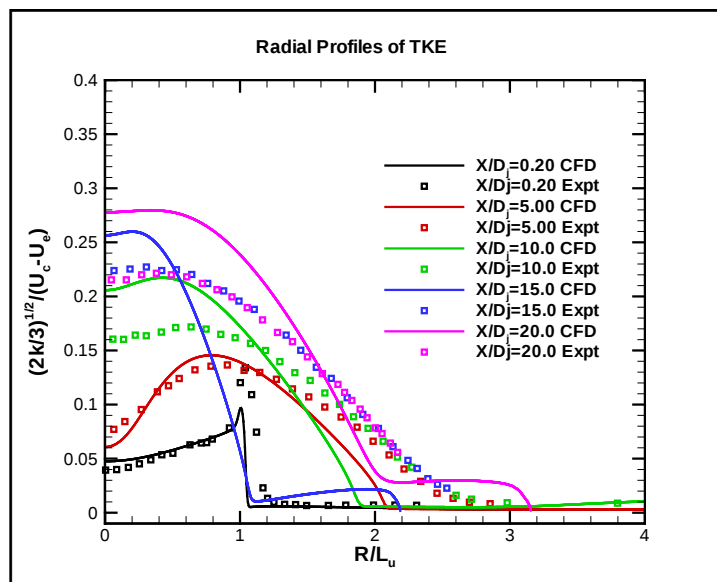
Validation Case Results



He



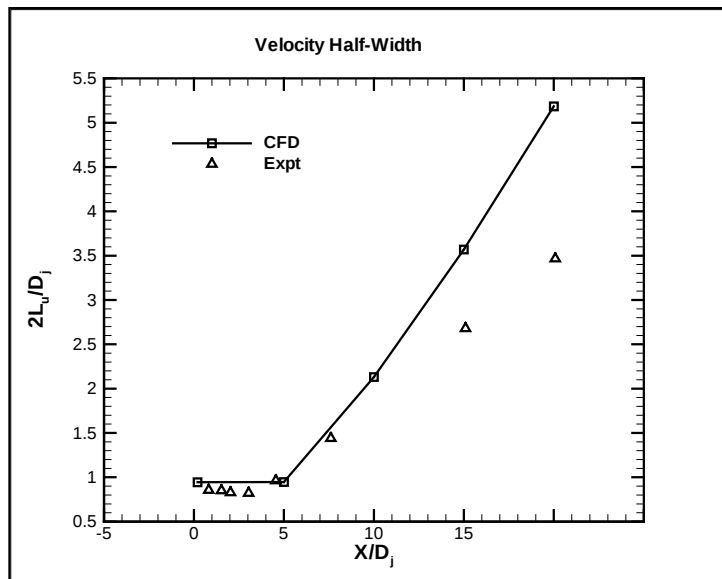
CO₂



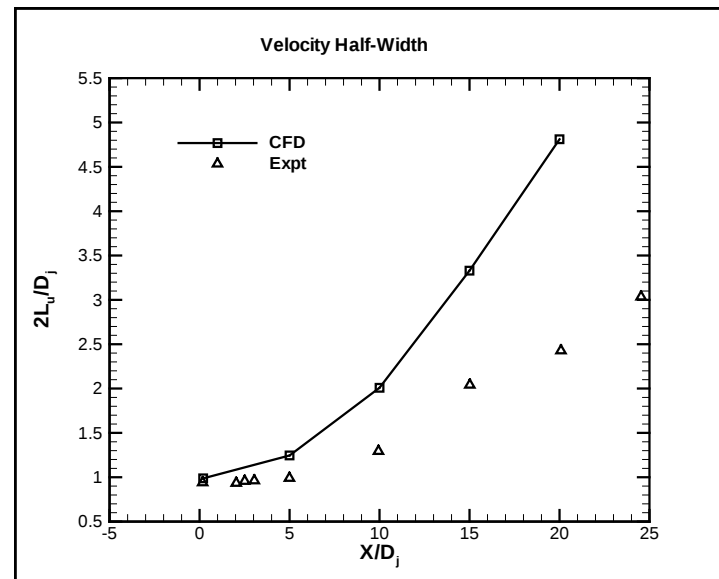
Air



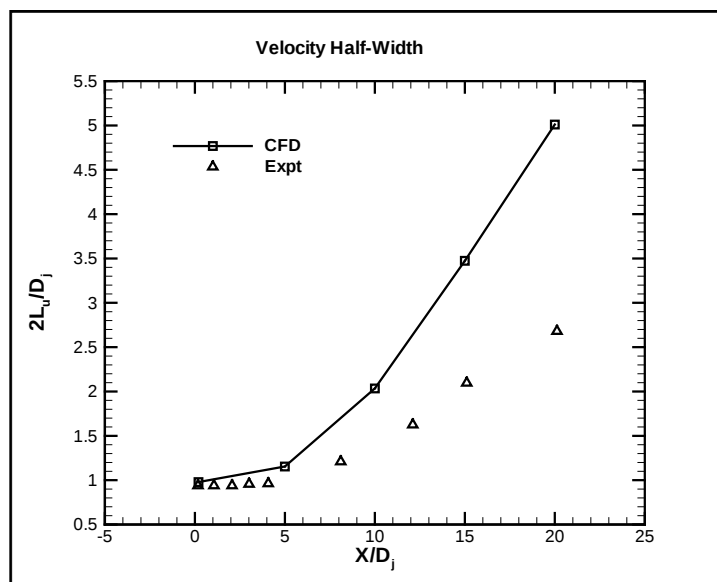
Validation Case Results



He



CO₂



Air



Validation Case Summary

- Case exhibits same deficiency in centerline mixing for He/Air in the near field while improving in the far-field.
- Radial profiles of mean quantities generally good, except at centerline.
- CO₂/Air case shows accurate mixing in the near-field and worsens downstream.
- Both CFD and experiment show low sensitivity of density ratio on jet spreading. Mixing occurring by entrainment of surrounding fluid.
- Turbulence is generally over predicted everywhere except in the near field for He/Air.
- Deficiencies in centerline mixing may be correlated to turbulence.



Future Work

- Try difference turbulence models.
- Modify k-eps to compensate for low near-field turbulence.
- Try Large Eddy Simulation (LES).
- Proceed to more relevant flow conditions for rocket injectors.



COAXIAL PARTICLE LADEN FLOW

Collaborators: Ananda Himansu, Alireza Badakhshan, Stephen Danczyk



Motivation

- Experimental set up in lab to evaluate novel fuel ignition strategies
- USI device with a shroud to induce swirl in coflow
- Use CFD to help design apparatus and flow conditions
- Some earlier CFD work helped in redesign of shroud for better swirling
- A parametric study to test the effects of swirling flow showed some unusual results
- Need to validate dispersed phase modeling features in CFD++ to confirm results and determine important conditions to accurate modeling



Eulerian Dispersed Phase Modeling in CFD++



Dispersed Phase

Mass:
$$\frac{\partial(\rho_{pi})}{\partial t} + \nabla \cdot (\rho_{pi} \vec{u}_{pi}) = \dot{m}_i$$

Momentum:
$$\frac{\partial(\rho_{pi} \vec{u}_{pi})}{\partial t} + \nabla \cdot (\rho_{pi} \vec{u}_{pi} \vec{u}_{pi}) = \vec{F}_{D_i} + \vec{F}_{V.M._i} + \vec{F}_{T.D._i} + \vec{F}_{L_i} + \vec{F}_{P.G._i} + \vec{F}_{B_i} + \dot{m}_i \vec{u}_{pi}$$

Energy:
$$\begin{aligned} \frac{\partial(\rho_{pi} e_{pi}^0)}{\partial t} + \nabla \cdot (\rho_{pi} e_{pi}^0 \vec{u}_{pi}) &= \text{Conduction source term} + \text{Radiation source term} + \dot{m}_i e_{pi}^0 \\ &= \dot{Q}_{pi} + \dot{m}_i e_{pi}^0 \end{aligned}$$

Number Density:
$$\frac{\partial N_i}{\partial t} + \nabla \cdot (N_i \vec{u}_{pi}) = 0$$

Melting Fraction:
$$\frac{\partial(\xi_i N_i)}{\partial t} + \nabla \cdot (\xi_i N_i \vec{u}_{pi}) = \text{Melting source term}$$

Material Density:
$$\tilde{\rho}_{pi} = \tilde{\rho}_{pi_{liquid}} \xi_i + \tilde{\rho}_{pi_{solid}} (1 - \xi_i),$$

Particle radius:
$$r_i = \left(\frac{3\rho_{pi}}{4\pi\tilde{\rho}_{pi}N_i} \right)^{1/3}.$$



Eulerian Dispersed Phase Modeling in CFD++



Continuous Phase

Momentum source terms:

$$\sum_{i=1}^{NP} \left(\vec{F}_{D_i} + \vec{F}_{V.M.i} + \vec{F}_{T.D.i} \right) ,$$

Energy source terms:

$$\sum_{i=1}^{NP} \left[\frac{\rho_{pi} C_{pi} f_{Ni}}{\tau_{Ti}} (T_f - T_{pi}) + \vec{u}_f \cdot (\vec{F}_{D_i} + \vec{F}_{V.M.i} + \vec{F}_{T.D.i}) \right] .$$

Relevant Source Terms

Interphase Drag:

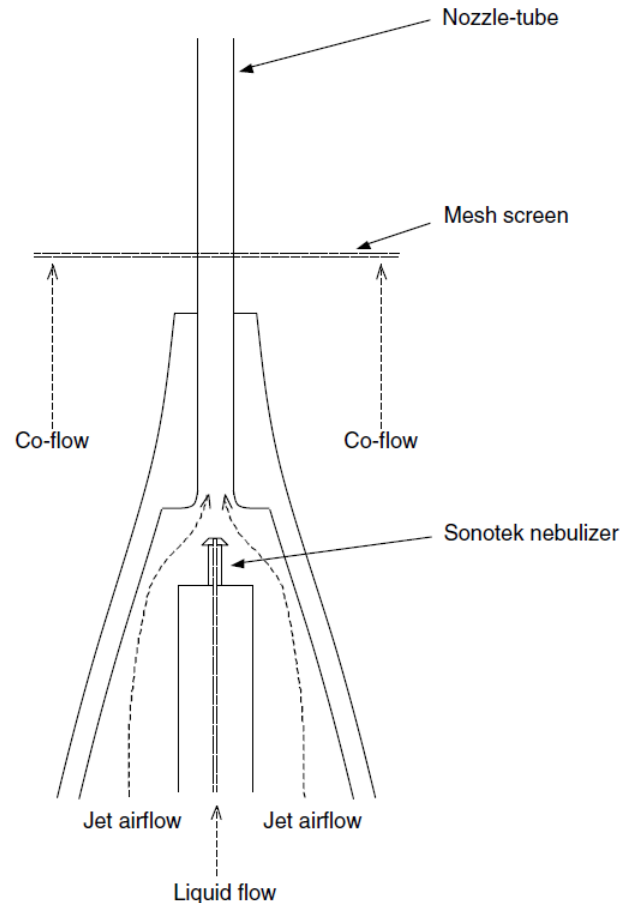
$$\frac{\vec{F}_{D_i}}{\rho_{pi}} = \frac{f_{Di}}{\tau_{ui}} (\vec{u}_f - \vec{u}_{pi}) ,$$

Turbulent Dispersion:

$$\frac{\vec{F}_{T.D.i}}{\rho_{pi}} = -\frac{f_{Di}}{\tau_{ui}} \frac{\nu_t}{Pr_t} \left(\frac{\nabla \eta_{pi}}{\eta_{pi}} - \frac{\nabla \eta_f}{\eta_f} \right) ,$$



Model Validation Case



Poly-dispersed
turpentine droplets

$$d = 1-90 \text{ um}$$

$$U_c = 2.4 \text{ m/s}$$

Low turbulence
intensity of 1.4%

Nijdam, J., Langrish, T., & Fletcher, D. (2008). Assessment of an Eulerian CFD model for prediction of dilute droplet dispersion in a turbulent jet. *Appl. Math. Model.*, 2686-2705.



Model Validation Case

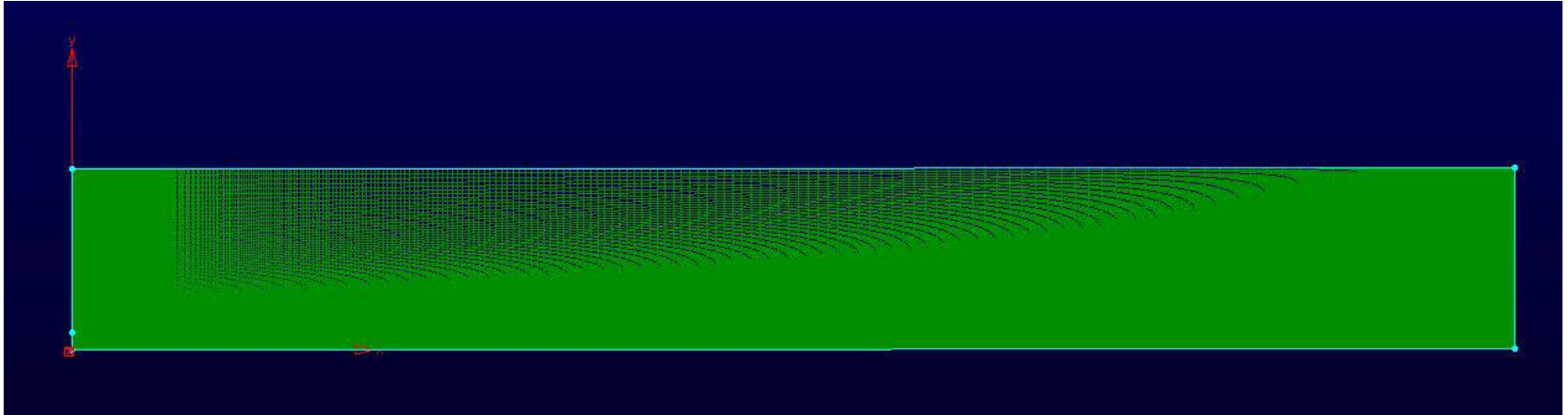
Droplet Inlet Boundary Condition (1D downstream)

Variable	Constants
Excess axial mean velocity (m/s) $U_e = U_{eo} [1 - [\sin((R/R_{1/2U})^{n_2})]^{n_1}]$	(1) Peak excess axial mean velocity, U_{eo} $U_{eo} = -0.05148 d + 21.97494$ where d is droplet diameter (μm) (2) $n_1 = 3.9320$, $n_2 = 1.222$, $R_{1/2U} = 4.978$
Radial mean velocity (m/s) $V = 0$	
Volume fraction $r = r_o \exp[-A(R/R_{1/2VF})^n]$	(1) Peak volume fraction r_o 4.32E-08 (5 μm), 6.91E-07 (15 μm), 5.07E-06 (25 μm), 9.50E-06 (35 μm), 1.47E-05 (45 μm), 2.16E-05 (55 μm), 2.04E-05 (65 μm), 7.43E-06 (75 μm), 2.50E-06 (85 μm) (2) $A = 0.6942$, $n = 2.1543$, $R_{1/2VF} = 3.9480$
Gas turbulent kinetic energy (m^2/s^2) $k = D \exp[-A(R - B)^n] + C$	(1) $A = 1.011$, $B = 4.904$, $C = 1.75$, $D = 8.73$, $n = 1.418$
Gas turbulent energy dissipation (m^2/s^3) $\varepsilon = \frac{k^{1.5}}{0.2D}$	(1) $D = 0.0098 \text{ m}$

Nijdam, J., Langrish, T., & Fletcher, D. (2008). Assessment of an Eulerian CFD model for prediction of dilute droplet dispersion in a turbulent jet. *Appl. Math. Model.*, 2686-2705.



Model Validation Case



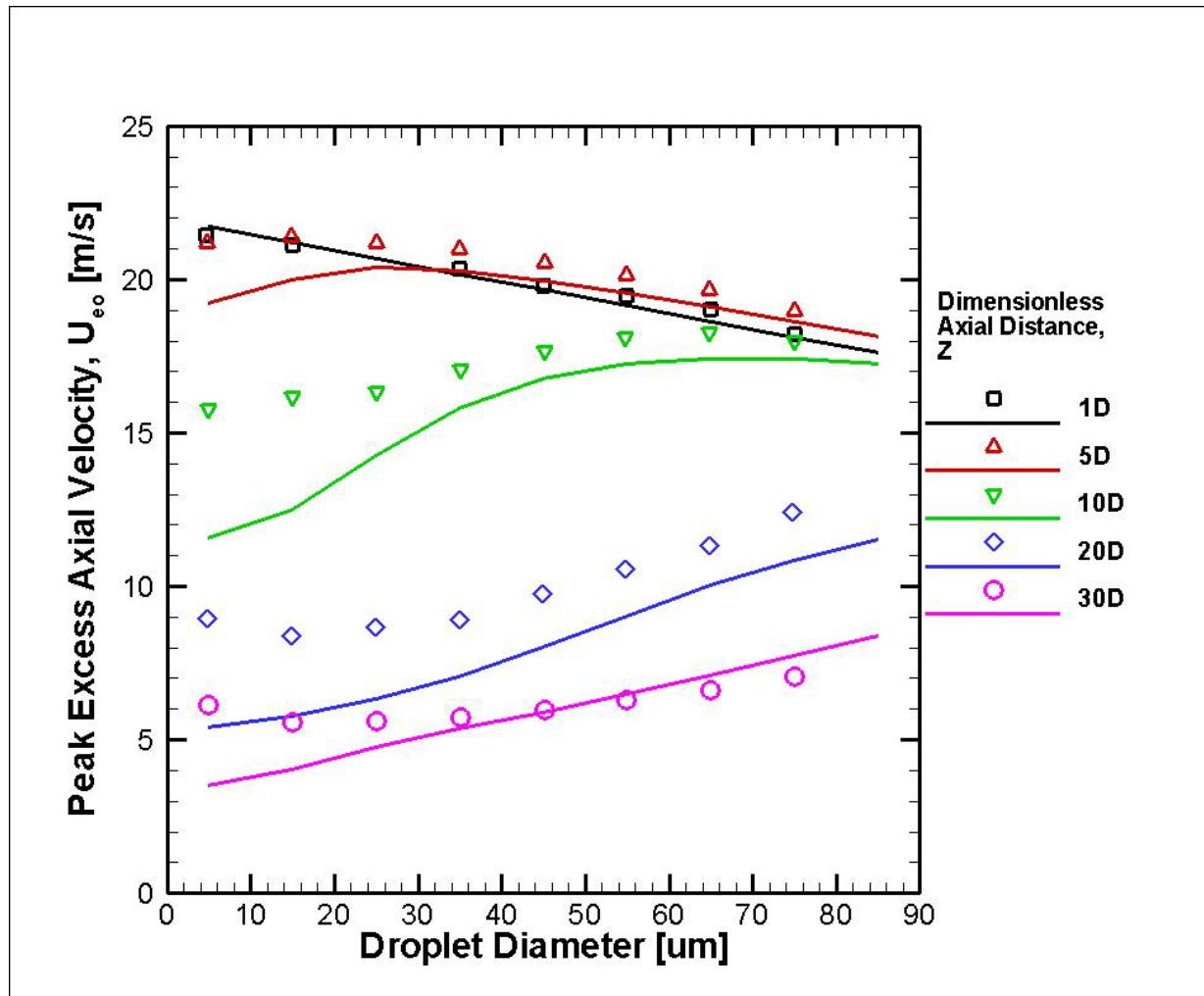
Case Conditions:

- Two species (Air, Turpentine)
- Base Equation Type: Compressible Real Gas Navier-Stokes/Euler
- Equation of State: Ideal Gas
- Turbulence Simulation: RANS, realizable k-eps or SST
- Turbulence Intensity: 1.4%
- 10 um droplet size bins from 5-85 um diameter
- Temperature-based inlet profiles generated with Matlab script

- 79000 quadrilateral cells
- Total domain size = 0.05 m x 0.4 m

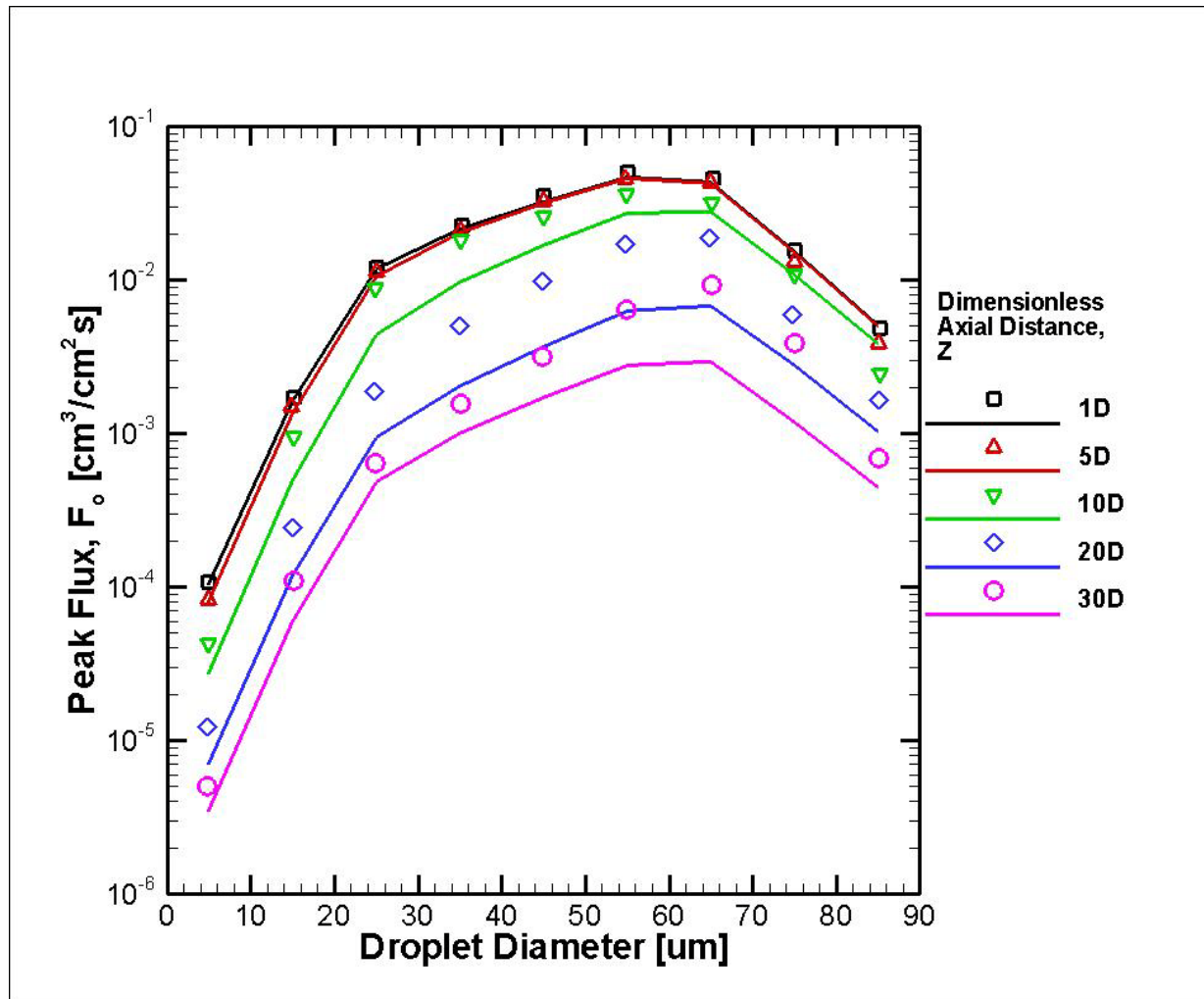


Results



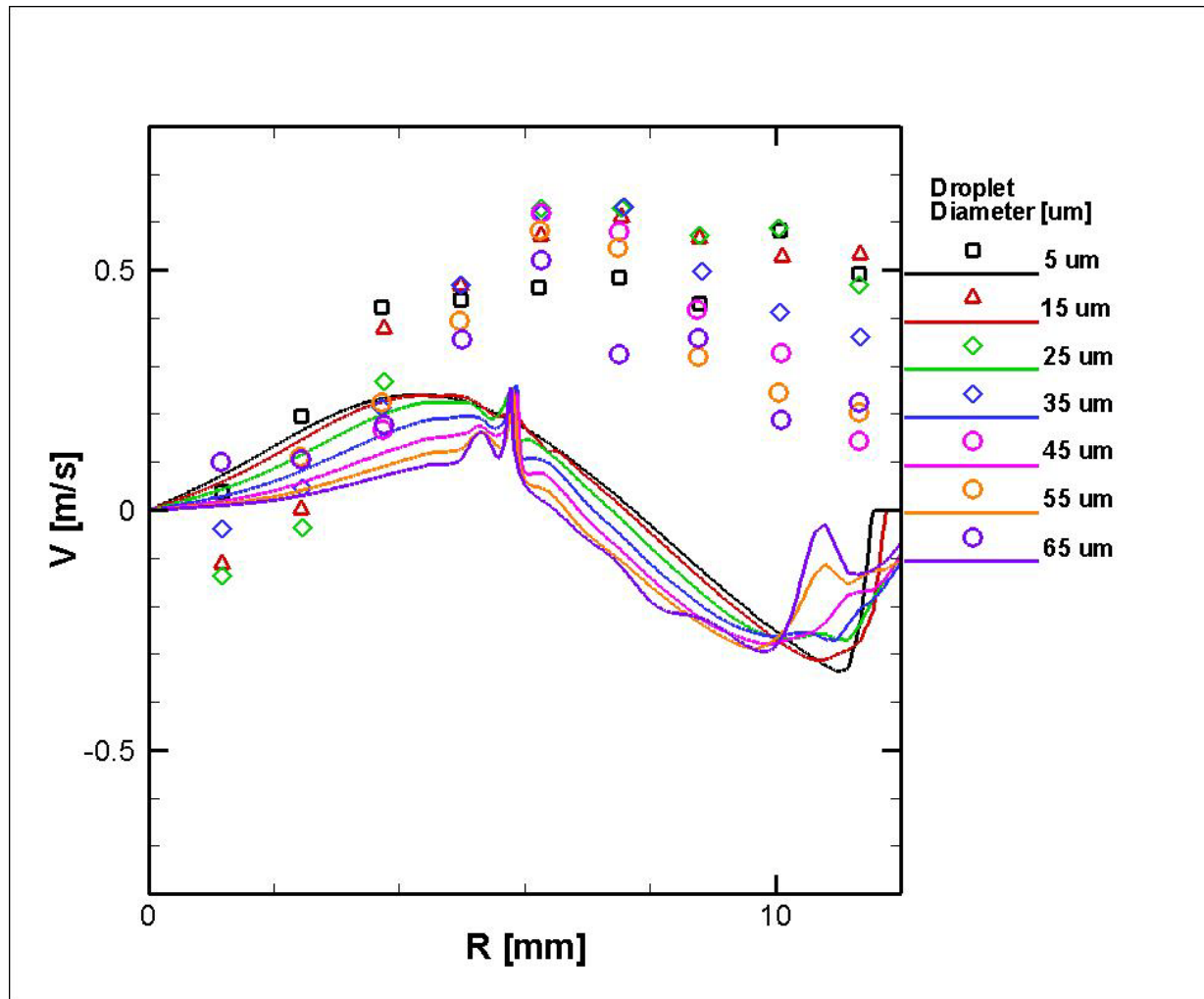


Results





Results





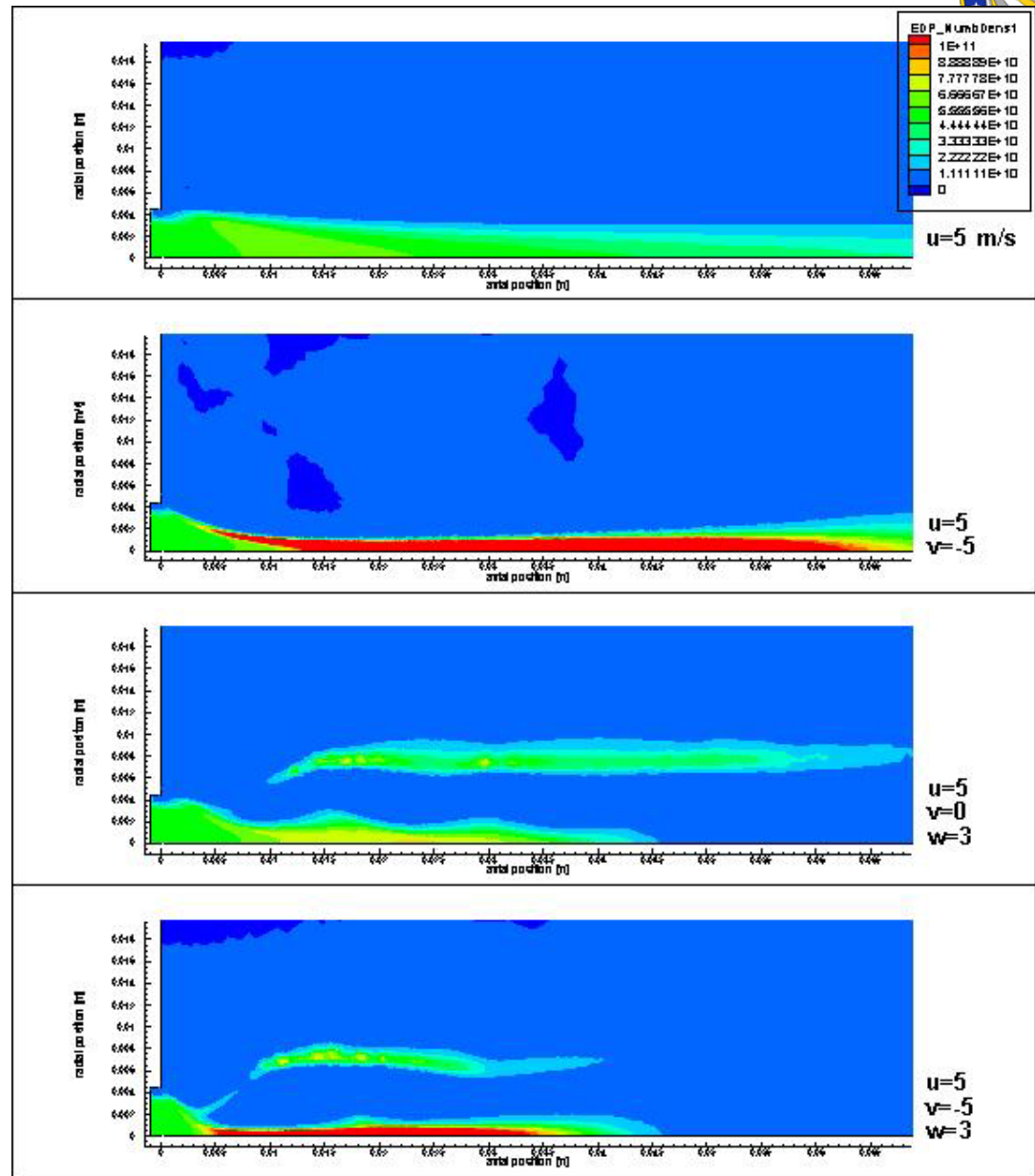
Summary

- Excess axial velocities of smaller droplets up to 50% slower than experimental while larger droplets show better agreement
- Error for peak volumetric fluxes higher for larger droplets—may be due to the fact that they carry more volume
- CFD agreement with radial velocities is poor— may be measurement error
- EDP features available in CFD++ may be missing important physics



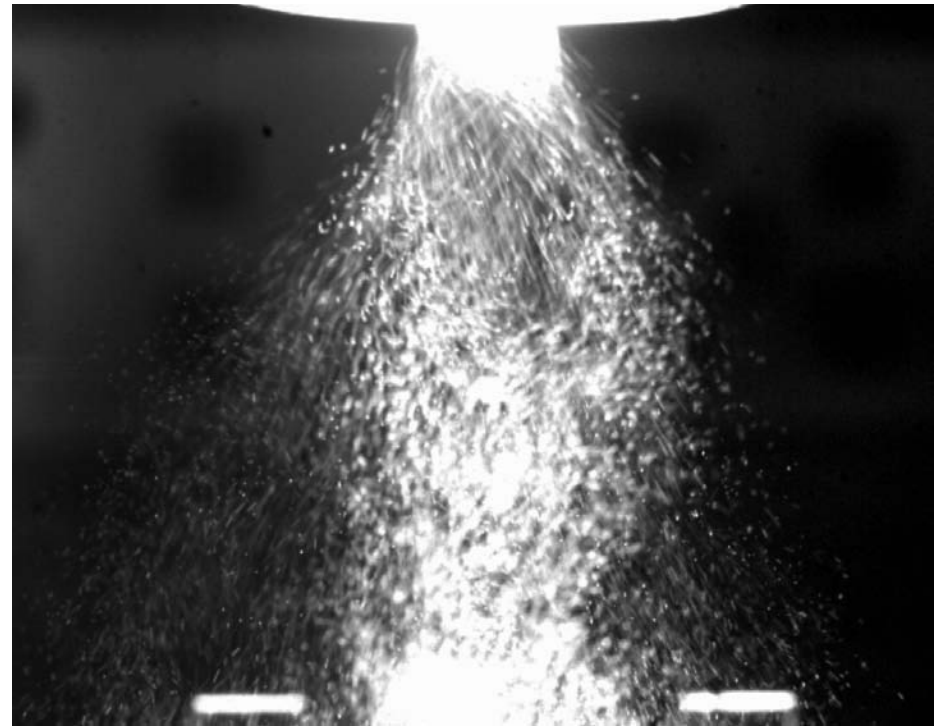
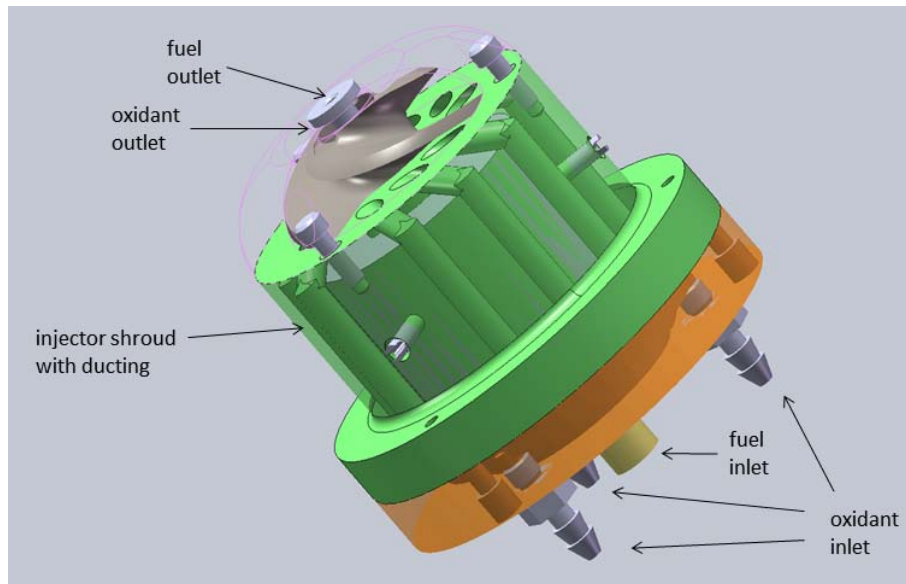
Future Work

Parametric study of effects of co-flow velocity components on droplet mixing





Future Work



Use CFD to design and determine conditions for experimental setup for testing combustion ignition strategies



Questions?

